Signal amplification by unidirectional coupling of oscillators

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Abstract

We report our investigation on the input signal amplification in unidirectionally coupled monostable Duffing oscillators in one- and two dimensions with first oscillator alone driven by a weak periodic signal. Applying a perturbation theory, we obtain a set of nonlinear equations for the response amplitude of the coupled oscillators. We identify the conditions for undamped signal propagation with enhanced amplitude through the coupled oscillators. When the number of oscillators increases the response amplitude approaches a limiting value. We determine this limiting value. Also, we analyse the signal amplification in the coupled oscillators in two dimensions with a fraction of oscillators chosen randomly for coupling and forcing.

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(Some figures may appear in colour only in the online journal)

1. Introduction

Over the last three decades or so, a great deal of interest has been paid to exploring the various features of nonlinear systems, particularly, in systems driven by a periodic force. Often, a periodic force is treated as a simple external perturbation to induce oscillatory motion and different nonlinear phenomena in damped nonlinear oscillators. In many physical, engineering and biological systems and in electronic circuits, it is easy not only to introduce a periodic force but also to represent an inherent part of them. For example, a periodic force represents a periodic seasonal variation in ecological systems [1], an injection current in lasers [2], imaging and audio pumping waves in an acoustic field [3], oscillations introduced to mechanical devices by means of electromagnetic vibration generator [4], an input voltage/current source in electronic circuits [4, 5], membrane current in neuronal axons [6, 7], pacemakers’ activity in real-life systems [8], an applied illumination in photolithography-assisted techniques in chemical reactions [9, 10] and so on.

Identification of a weak input periodic signal and enhancement of response of nonlinear systems are of importance. We note that the response of a nonlinear system can be enhanced in different ways. For example, in nonlinear oscillators when the frequency $\omega$ of a periodic force is varied for a fixed small value of amplitude of the force, the response amplitude increases, reaches a maximum value at a particular frequency different from the natural frequency of oscillation of the systems and then decreases. This phenomenon is the well-known nonlinear resonance. In bistable systems, a weak noise can improve the response at the frequency of the input signal. This phenomenon is called stochastic resonance [11]. It has been shown that resonance can occur in the absence of weak periodic force and is termed coherence resonance [12, 13]. That is, noise alone is able to induce or enhance temporal order in the dynamics of certain nonlinear systems, particularly, in excitable systems. Apart from the Gaussian white noise, resonance has been realized due to coloured noise [14] and chaotic signal [15, 16]. Noise-supported wave propagation in the experiments of photosensitive Belousov–Zhabotinsky chemical reaction [17], one- and two-dimensional arrays of two-way coupled bistable oscillators [18], nonlinear cells with two-state threshold elements [19] and bistable oscillators driven by a biharmonic signal [20] and fault-tolerant behaviour in a chain of one-way-coupled bistable oscillators [21] have been realized. On the other hand, in the vibrational resonance approach, the noise is replaced by a high-frequency periodic force [22].

It is possible to enhance the response amplitude by unidirectionally coupling the oscillators. There are few interesting dynamics realized as a result of unidirectional
coupling. One-way coupling was introduced by Visarath In et al. [23, 24] to induce oscillations in undriven, overdamped and bistable systems. The mechanism for the generation of oscillation is described for a three-coupled overdamped Duffing oscillators [25]. The response of three-(one-way)-coupled core magnetometer systems to a periodic magnetic-flux signal applied to all the three units is explored. The response is found to be either synchronized to the signal frequency or to one-third of it [26]. In one-way coupled systems, the propagation of waves of dislocations in equilibria are found [27]. One-way coupling is utilized in electronic sensors and microelectronic circuits [28] and is found to assist the propagation of localized nonlinear waves [29, 30]. Experimental evidences of propagation and annihilation of solitons in a mechanical array of unidirectionally coupled oscillators [31] and an electronic circuit in two dimensions [32] are reported. Improved transmission of low-frequency signal by the combinative action of high-frequency input signal and one-way coupling has been found in coupled overdamped bistable oscillators [33] and in coupled maps [34].

In biology also, unidirectionally coupled structures constitute one of the simplest topologies of practical interest. We wish to cite an example from the field of gene regulatory network. A network of transcriptional regulators of considerable interest is the repressilator model. It is the network consisting of \( n \) genes with the protein product of \( i \)th gene repressing the expression of \((i+1)\)th gene with periodic boundary conditions. Oscillatory dynamics is found in an experimentally constructed repressilator with three cells [35]. The occurrence of monostable, spiral and limit cycle dynamics is analysed in a mathematical model of repressilator with three genes [36, 37]. A repressilator system with an even number of genes is found to exhibit multistability, while with an odd number of genes display a stable limit cycle [38]. The existence of quasi-stable travelling wave periodic solutions is analysed both theoretically and numerically in unidirectionally coupled repressilators [39, 40].

The goal of this work is to investigate the enhancement of output signal amplitude and undamped signal propagation in unidirectionally coupled oscillators in one- and two dimensions. In the coupled systems, the first oscillator (unit) alone subject to a periodic force, while the other oscillators are coupled unidirectionally. The coupling term is linear. For our analysis, we choose the paradigm Duffing oscillator as the reference oscillator. For the coupled oscillators in one dimension applying a perturbation theory, we obtain a set of nonlinear equations for the amplitudes of the periodic oscillations of the oscillators. Below a critical value of the coupling strength \( \delta \), the amplitude \( A_i \) of \( i \)th oscillator decays to zero as \( i \) increases even though \( A_1 \) of the first oscillator is nonzero. This corresponds to a damped signal propagation. When \( A_i = 0 \) the state variables of the \( i \)th oscillator decay to zero as time \( t \to \infty \). Above the critical value of \( \delta \), the response amplitude \( A_i \) increases with \( i \) and attains a saturation leading to an enhanced signal propagation through the coupled oscillators. In this case, \( A_i > A_1 \) for \( i > 1 \) and the state variables of the oscillators do not decay to zero with time; but, the oscillators exhibit periodic motion with the frequency \( \omega \) of the periodic force applied to the first oscillator. We refer this case as an undamped signal propagation. From the nonlinear equations of the amplitudes \( A_i \), we are able to determine the saturate value of response amplitude and the threshold condition on \( \delta \) for undamped signal propagation. The theoretical predictions agree quite well with the numerical simulation. For a fixed value of \( \delta \) when the frequency \( \omega \) of the external periodic force is varied, the oscillators exhibit nonlinear resonance and hysteresis. The width of the hysteresis decreases with increases in the value of \( \delta \) and then it disappears.

For the one-way coupled oscillators in two dimensions with:

(i) the first oscillator alone driven by the periodic force;

(ii) all the oscillators are driven, but certain randomly chosen number of oscillators are alone coupled;

(iii) all the oscillators are coupled, but the periodic force is applied to randomly chosen certain number of oscillators only;

also the theoretical treatment enable us to determine the response amplitudes of the oscillators. The enhancement of response amplitude is realized in the above three cases.

2. Coupled Duffing oscillators in one dimension

The equation of motion of the unidirectionally coupled \( n \)-Duffing oscillators of our interest in one dimension is given by

\[
\ddot{x}_i + \alpha_0^2 x_i + \beta x_i^3 = f \cos \omega t, \quad (1a)
\]

\[
\dot{x}_i + \omega_0^2 x_i + \beta x_i^3 = \delta x_{i-1}, \quad i = 2, 3, \ldots, n. \quad (1b)
\]

We assume that the values of all parameters in equations (1) are \( \geq 0 \). In [26], coupled magnetometer systems were considered, wherein the units form a closed ring (periodic boundary conditions), the coupling is unidirectional and the signal was applied to all the units in the ring. In contrast to these, in system (1), the first oscillator alone is driven by the periodic signal \( f \cos \omega t \). The other oscillators are coupled unidirectionally. The dynamics of the first oscillator is independent of the dynamics of the other oscillators, while that of the \( i \)th oscillator depends on the dynamics of \((i-1)\)th oscillator through the linear coupling. The last oscillator is not connected to the first oscillator. System (1) is capable of showing a variety of nonlinear phenomena including chaotic dynamics. In our present work, we are interested in resonance behaviour and the enhancement of the amplitude of periodic response of the system due to one-way coupling. Therefore, we choose \( f \ll 1 \) so that in equation (1a), periodic oscillation occurs about the equilibrium point.

Applying a perturbation theory to the system (1), we obtain a set of coupled nonlinear equations for the amplitudes of the period-\( T = 2\pi/\omega \) solutions of the oscillators and then analyse the influence of one-way coupling on response amplitudes and resonance.
2.1. Theoretical treatment

We assume the periodic solution of the system (1) as

\[ x_i(t) = a_i(t) \cos \omega t + b_i(t) \sin \omega t, \]

where \( a_i(t) \) and \( b_i(t) \) are slowly varying functions of time \( t \). We write

\[ x_i(t) = \frac{1}{2} \left( a_i^2 + b_i^2 \right) (a_i \cos \omega t + b_i \sin \omega t), \]

(3a)

\[ \dot{x}_i(t) = \dot{a}_i \cos \omega t + \dot{b}_i \sin \omega t - a_i \omega \sin \omega t + b_i \omega \cos \omega t, \]

(3b)

\[ \ddot{x}_i(t) = -2 \dot{a}_i \omega \sin \omega t + 2b_i \omega \cos \omega t - a_i \omega^2 \cos \omega t - b_i \omega^2 \sin \omega t, \]

(3c)

where in equation (3c), we neglected \( \dot{a}_i \) and \( \dot{b}_i \) because of their smallness. Substituting (2)–(3) in (1), neglecting \( \ddot{a}_i \) and \( \ddot{b}_i \) as they are assumed to be small and then equating the coefficients of \( \sin \omega t \) and \( \cos \omega t \) separately to zero, we obtain

\[ \dot{a}_i = -\frac{b_i}{2\omega} \left[ \omega^2 - a_i^2 - \frac{3}{4} \beta (a_i^2 + b_i^2) \right] - \frac{1}{2} k a_i + S_a, \]

(4a)

\[ b_i = \frac{a_i}{2\omega} \left[ \omega^2 - a_i^2 - \frac{3}{4} \beta (a_i^2 + b_i^2) \right] - \frac{1}{2} k b_i + S_b, \]

(4b)

where for \( i = 1 \)

\[ S_a = 0, \quad S_b = \frac{f}{2\omega} \]

and for \( i > 1 \)

\[ S_a = \frac{\delta a_i-1}{2\omega}, \quad S_b = \frac{\delta a_i-1}{2\omega} \]

(5b)

Next, we introduce the transformation

\[ a_i(t) = A_i(t) \cos \theta_i(t), \quad b_i(t) = A_i(t) \sin \theta_i(t) \]

(6)

with \( A_i^2 = a_i^2 + b_i^2 \). Then equations (4) become

\[ \dot{A}_i = -\frac{1}{2} dA_i + S_A, \]

(7a)

\[ A_i \dot{\theta}_i = \frac{A_i}{2\omega} \left[ \omega^2 - a_i^2 - \frac{3}{4} \beta A_i^2 \right] + S_b, \]

(7b)

where for \( i = 1 \)

\[ S_A = \frac{f}{2\omega} \sin \theta_1, \quad S_b = \frac{f}{2\omega} \cos \theta_1 \]

(8a)

and for \( i > 1 \)

\[ S_A = \frac{\delta}{2\omega} A_i-1 \sin(\theta_i - \theta_i-1), \quad S_b = \frac{\delta}{2\omega} A_i-1 \cos(\theta_i - \theta_i-1). \]

(8b)

The response of the system (1) in the long time limit is periodic with period \( T = 2\pi/\omega \) provided \( A_i(t) \) and \( \theta_i(t) \) become constants as \( t \to \infty \) and are the equilibrium points of equations (7). To find the equilibrium points of (7), we set \( \dot{A}_i = \dot{\theta}_i = 0 \), \( A_i(t) = A_i^*, \theta_i(t) = \theta_i^* \) and drop * in \( A_i \) and \( \theta_i \). We obtain

\[ A_i^2 \left[ \omega^2 - \omega^2 + \frac{3}{4} \beta A_i^2 \right]^2 + d^2 \omega^2 A_i^2 = \begin{cases} f^2 & \text{for } i = 1, \\ \delta^2 A_i^2-1 & \text{for } i > 1 \end{cases} \]

(9)

Figure 1. Variation of the response amplitudes \( A_i \) with \( i \), the number of oscillators, for three values of the coupling strength \( \delta \). The continuous lines and solid circles represent theoretical and numerical values of \( A_i \). Numerical \( A_i \)'s are shown for certain selected values of \( i \) for clarity. The values of the parameters are \( d = 0.5, \omega_0^2 = 1, \beta = 1, f = 0.1 \) and \( \omega = 0.7 \).

and

\[ \theta_i = \theta_i-1 + \tan^{-1} \left[ \frac{d\omega}{\omega^2 - \omega^2 + \frac{3}{4} \beta A_i^2} \right], \]

(10)

where \( \theta_0 = 0 \).

When \( i = 1 \), equation (9) is the frequency-response equation of the first oscillator. For \( i > 1 \), equation (9) is independent of \( f \) and depends on the coupling constant \( \delta \) and the response amplitude \( A_i-1 \) of the \( (i-1) \)th oscillator. Equation (9) (for \( i > 1 \)) is the coupling strength-response (amplitude) equation of the \( i \)th oscillator. The amplitude and phase of the \( i \)th oscillator with \( i > 1 \) depend on the amplitude and on the phase, respectively, of the \( (i-1) \)th oscillator. \( A_i \) and \( \theta_i \) can be determined by solving equations (9) and (10) successively.

2.2. Analysis of the effect of one-way coupling

Now, we analyse the influences of the coupling strength on the response amplitudes \( A_i \) and resonance. We fix the values of the parameters as \( d = 0.5, \omega_0^2 = 1, \beta = 1 \) and \( f = 0.1 \). Equation (9) contains even powers of \( A_i \). It can be viewed as a cubic equation in terms of \( A_i^2 \). Explicit analytical expressions for the roots of a cubic equation is given in [41]. Equation (9) can admit either one real root or three real roots. First, we determine the values of \( A_1 \) and then successively calculate \( A_2, A_3, \ldots \). For our analysis we fix \( n = 200 \).

In order to know the validity of the theoretical treatment, we solve equations (1) numerically using the fourth-order Runge–Kutta method. From the numerical solution, we compute the amplitudes \( A_i \). Figure 1 shows both theoretical and numerically computed \( A_i \) versus \( i \) for three fixed values of \( \delta \) with \( \omega = 0.7 \).

The theoretical \( A_i \) very closely match with numerically calculated \( A_i \) even for large values of \( i \). For \( \delta = 0.6 \), as \( i \) increases, the amplitudes \( A_i \) of oscillation of the consecutive oscillators decrease and the distant oscillators do not oscillate but settle to the equilibrium state which is the origin. The coupled oscillators exhibit damped signal propagation.
(\(A_1 \to 0\) as \(i \to \infty\)). When \(\delta = 0.65\) and 0.7 in figure 1, we note that \(A_i\) increases with \(i\) and then reaches a saturation. Although the first oscillator alone is driven by the periodic force the one-way coupling gives rise enhanced signal propagation (\(A_i > 1\) for \(i \gg 1\)) over the coupled oscillators. This is a quintessence feature of one-way coupling. The point is that for the parametric values corresponding to the undamped signal propagation, there exists a critical number of oscillators to realize a maximum value of the response amplitude of the last oscillator. This critical number of oscillators and the maximum amplitude depend on the control parameters \(\omega\) and \(\delta\).

Interestingly, the value of the saturate amplitude denoted as \(A_L\) (limiting value of amplitude in the limit of very large \(i\)) and the condition on \(\delta\) for undamped signal propagation can be obtained from the nonlinear equations of \(A_i\) given by equation (9). \(A_i = 0\) and \(A_L > 1\) for damped and undamped signal propagation, respectively. Equation (9) for \(i > 1\) can be viewed as a nonlinear map specifying the relation between \(A_{i-1}\) and \(A_i\). The stable equilibrium states of equation (9) are the values of \(A_L\). For \(i > 1\), the equilibrium states of equation (9) can be obtained by setting \(A_i = A_{i-1} = A_L\). This gives \(A_L = 0\) and

\[
A_{L\pm} = \left[\frac{4}{3\beta} \left(\omega^2 - \omega_0^2 \pm \sqrt{\delta^2 - d^2 \omega^2}\right)\right]^{1/2}.
\]

Now, we have two cases:

**Case 1.** \(\omega^2 < \omega_0^2\). When \(\omega^2 < \omega_0^2\), \(A_L = 0\) is the only equilibrium state for \(\delta < \delta_c\) where

\[
\delta_c = \sqrt{\left(\omega^2 - \omega_0^2\right)^2 + d^2 \omega^2} = \frac{f}{A_{\text{linear}}},
\]

where \(A_{\text{linear}}\) is the response amplitude of the linear part of equation (10) \((\beta = 0)\). \(A_{L+} = 0\) and \(A_{L\pm}\) are the equilibrium states for \(\delta > \delta_c\).

**Case 2.** \(\omega^2 > \omega_0^2\). \(A_L = 0\) is the only equilibrium state for \(\delta < \delta_0\). For \(\delta_0 < \delta < \delta_c\), the equilibrium states are \(A_L = 0\) and \(A_{L\pm}\). When \(\delta > \delta_c\), the possible equilibrium states are \(A_L = 0\) and \(A_{L+}\). It is noteworthy to mention that \(A_{L\pm}\) are independent of the amplitude \(f\) of the driving force, but depend on the frequency of the driving force. They are inversely proportional to the coefficient \(\beta\) of the nonlinear term.

Next, we determine the condition on \(\delta\) for enhanced and undamped signal propagation. The difference between the nonlinear equations for \(A_1\) and \(A_i\), \(i > 1\) is only in the last term in equation (9). For \(A_1\), the last term is \(f^2\), while it is \(\delta^2 A_{L-1}\) for \(A_i\). Consider equation (9) with \(f^2\) (or \(\delta^2 A_{L-1}\)) replaced by \(\alpha\). Figure 2 shows the variation of the response amplitude \(A_1\) or \(A_i\) for three fixed values of \(\omega\). In all the cases, \(A\) increases monotonically with \(\alpha\). From figure 2, we infer that in order to have \(A_2 > A_1\) the condition is \(\delta^2 A_1^2 > f^2\), that is

\[
\delta > \delta_u = \frac{f}{A_1}.
\]

For \(\omega = 0.7\) and \(f = 0.1\), we find \(A_1 = 0.15777\) and \(\delta_u = 0.634\). That is, undamped signal propagation occurs for \(\delta > 0.634\) and is confirmed in the numerical simulation.

We point out the interplay between the parameters \(d\) and \(\delta\). For fixed values of the parameters, the amplitude \(A_1\) is found to decrease with increase in the damping coefficient \(d\). Consequently, from equation (13), it is clear that \(\delta_u\) increases with increase in the value of \(d\). This is also evident from equation (12). \(A_{L\pm}\) given by equation (11) depend on all the parameters of the system except the amplitude \(f\) of the input periodic force. Further, \(A_{L+}(A_{L-})\) increases (decreases) with increase in the value of \(\delta\). In contrast to this, both \(A_{L+}\) and \(A_{L-}\) decrease with increase in \(d\). For \(\omega = 0.7\) and \(f = 0.1\), the values of \(\delta_u\) for \(d = 0.3, 0.5\) and 0.7 are 0.57275, 0.63402 and 0.71781, respectively. For \(\delta = 0.7\) the values of \(A_1\) for \(d = 0.3, 0.5\) and 0.7 are 0.45863, 0.35818 and 0, respectively.

In figure 3, we plot the dependence of \(\delta_c\) and \(\delta_u\) with \(\omega\). For the values of \(\omega\) and \(\delta\) below the lower curve, \(A_1 = 0\) is the only equilibrium state and is stable. In the regions between the upper and lower curves in addition to \(A_L = 0\), a nontrivial \(A_L\) also exists with \(A_1 \neq 0 < A_1\). The nontrivial state is unstable, while the state \(A_1 = 0\) is stable. In the regions above the upper curve, \(A_1 = 0\) is unstable, while the nontrivial \(A_L > A_1\) is stable leading to an enhanced undamped signal propagation. Undamped signal propagation with \(A_1 \neq 0 < A_1\) is not observed. Figure 4 presents \(A_L\) versus the control parameters \(\delta\) and \(\omega\). For each fixed value of \(\omega\), \(A_L\) increases monotonically with \(\delta\) for \(\delta > \delta_u\).

Figure 5 shows the frequency-response amplitude profile of various oscillators for a few fixed values of \(\delta\). For \(\delta = 0.45\),

![Figure 2](image-url) Variation of the response amplitude \(A\) \((A_1\) or \(A_i, i > 1\)) with the parameter \(\alpha\) \((= f^2\) for \(A_1\) and \(\delta^2 A_{L-1}\) for \(A_i\)) for three values of \(\omega\).

![Figure 3](image-url) Theoretical \(\delta_c\) (dotted curved) and \(\delta_u\) (continuous curve) given by equations (12) and (13), respectively, versus \(\omega\). The solid circles are the numerically computed values of \(\delta_u\).
Figure 4. Variation of the limiting value $A_L$, the value of the response amplitude $A_i$, in the limit of $i \to \infty$, with the parameters $\delta$ and $\omega$. Zero and nonzero values of $A_L$ represent damped and undamped signal propagations, respectively.

Figure 5. Three-dimensional plot of $A_i$ versus $i$ and $\omega$ for three fixed values of $\delta$. $A_i$ is shown only for some selected values of $i$. The thick curve represents $A_1$.

Figure 6. The effect of the coupling strength $\delta$ on the hysteresis phenomenon. In the subplot (a), the thin and the thick curves represent $A_i$ and $A_{100}$, respectively. The arrows indicate the path followed by the response amplitude when $\omega$ is varied in the forward and reverse directions. The values of the parameters are $d = 0.1$, $\omega_0 = 1$, $\beta = 2$ and $f = 0.2$. $A_i$ shown in the subplot (a) is independent of $\delta$ because the first oscillator is unaffected by the other oscillators in the system.

The output signal dampens out as it propagates through the coupled oscillators. When $\delta = 0.5$ in figure 4, we note that $A_L \neq 0$ for a small range of values of $\omega$ implying undamped signal propagation and is evident in figure 5(b). The range of $\omega$ values for which undamped signal propagation occurs increases with increase in $\delta$. This is clearly seen in figures 4 and 5(b), (c). In figure 5(c) where $\delta = 0.7$, for the first certain number of oscillators, the response amplitude $A_i$ increases with $\omega$, reaches a maximum and then decreases smoothly. For distant oscillators after reaching a maximum value $A_i$ suddenly jumps to a lower value. When $\omega$ is increased from a smaller value to a higher value and then varied in the reverse direction, $A_i$ follows the same path. $A_i$ is single valued and there is no hysteresis phenomenon; however, jump phenomenon occurs. Next, we give an example of the case where $A_i$ shows hysteresis. Figure 6 illustrates the effect of one-way coupling on $A_{100}$ for three values of $\delta$, where $d = 0.1$, $\omega_0 = 1$, $\beta = 2$ and $f = 0.2$.

The width of the hysteresis loop decreases with increase in $\delta$. In figure 6(c), for $\delta = 1.9$, there is no hysteresis in $A_{100}$; however, we can clearly notice a sudden jump or a
sharp variation in the value of $A_{100}$ at two values of $\omega$. Unidirectional coupling weakens and suppresses hysteresis.

3. Coupled oscillators in two dimensions

In this section, we extend our study to unidirectionally coupled oscillators in two dimensions. Figure 7 depicts an example of $5 \times 5$ oscillators coupled unidirectionally in two dimensions. The first oscillator alone is driven by the force $f \cos \omega t$. For simplicity, we consider $n \times n$ oscillators. The equations of motion of the oscillators are given by

$$\ddot{x}_{j,1} + \omega_0^2 x_{j,1} + \beta x_{j,1}^3 = f \cos \omega t,$$

(14a)

$$\ddot{x}_{j,i} + \omega_0^2 x_{j,i} + \beta x_{j,i}^3 = A_{j,1} \cos \theta_j,$$

(14b)

$$\ddot{x}_{j,i} + \omega_0^2 x_{j,i} + \beta x_{j,i}^3 = \delta x_{j,i-1},$$

(14c)

$$\ddot{x}_{j,i} + \omega_0^2 x_{j,i} + \beta x_{j,i}^3 = \frac{1}{2} \delta (x_{j,i-1} + x_{j,i+1}),$$

(14d)

where $j = 2, 3, \ldots, n$ and $i = 2, 3, \ldots, n$.

Seeking the periodic solution of equations (14) in the form

$$x_{j,i}(t) = a_{j,i}(t) \cos \omega t + b_{j,i}(t) \sin \omega t,$$

(15a)

where

$$a_{j,i}(t) = A_{j,i}(t) \cos \theta_j, \quad b_{j,i}(t) = A_{j,i}(t) \sin \theta_j,$$

(15b)

with $A_{j,i}^2 = a_{j,i}^2 + b_{j,i}^2$, we obtain the following set of equations:

$$A_{j,1}^2 \left[ \omega_0^2 - \omega^2 + \frac{3}{2} \beta A_{j,1}^2 \right]^2 + A_{j,1}^2 \delta^2 A_{j,1}^2 = F_{j,1},$$

(16a)

where

$$F_{j,1} = f^2, \quad F_{j,i} = \delta^2 A_{j,1}^2,$$

(16b)

$$F_{j,i} = \delta^2 A_{j,i-1}^2.$$

(16c)

with $i, j = 1, 2, \ldots, n$ and

$$\theta_{1,i} = \theta_{1,i-1} + \tan^{-1} \left( \frac{\omega \delta^2 A_{j,1}^2}{\omega_0^2 - \omega^2 + \frac{3}{2} \beta A_{j,1}^2} \right),$$

(16d)

$$\theta_{j,1} = \theta_{j-1,1} + \tan^{-1} \left( \frac{\omega \delta^2 A_{j,1}^2}{\omega_0^2 - \omega^2 + \frac{3}{2} \beta A_{j,1}^2} \right),$$

(16e)

$$\theta_{j,i} = \theta_{j,i-1} + \tan^{-1} \left( \frac{\omega \delta^2 A_{j,1}^2}{\omega_0^2 - \omega^2 + \frac{3}{2} \beta A_{j,1}^2} \right),$$

(16f)

$$\delta \omega A_{j,1} = \frac{1}{2} \delta A_{j-1,1} \sin (\theta_{j,i} - \theta_{j,i-1}) + \frac{1}{2} \delta A_{j,i-1} \sin (\theta_{j,i-1} - \theta_{j,i}),$$

(16g)

where $\theta_{1,0} = \theta_{1,1} = 0$.

For the oscillators numbered as $(j, 1)$ and $(1, i)$, the equations for $A_{j,1}$ and $A_{1,i}$, are independent of $\theta_{1,1}$ and $\theta_{1,i}$, respectively, and are the solutions of the cubic equations of the square of the amplitudes given by equations (16a)–(16c). $\theta_{1,1}$ and $\theta_{1,i}$ are given by equations (16e) and (16f), respectively. For the oscillators numbered as $(j, i)$, $j, i \neq 1$, the equations for $A_{j,i}$ depend on $\theta_{j,i-1}$ and $\theta_{j-1,i}$, but independent of $\theta_{j,1}$. Therefore, $A_{j,i}$ are the solutions of equations (16a) and (16d). Then we compute $\theta_{j,i}$ from equation (16g) using the Newton–Raphson method.

First, we point out the results obtained without solving the above set of equations (16). The set of $n$ oscillators in the first row and first column in figure 7, that is, the oscillators numbered as $(j, i) \rightarrow (1, i)$, $i = 1, 2, \ldots, n$ and $(j, i) \rightarrow (j, 1)$, $j = 1, 2, \ldots, n$, essentially form two identical set of coupled oscillators in one dimension. We note that

$$A_{1,k} = A_{k,1}, \quad \theta_{1,k} = \theta_{k,1}, \quad k = 2, 3, \ldots, n.$$  

(17)

Suppose

$$A_{1,i} \neq A_{1,k}, \quad i = 1, 2, \ldots, k \leq n.$$  

(18a)

$$A_{1,i} = A_{1,k}, \quad i = k + 1, k + 2, \ldots, n.$$  

(18b)

Then, because the coupling term of $(j, i)$th oscillator is $\delta (x_{j,i-1} + x_{j,i})/2$, we realize that for each $l$ where $l = 1, 2, \ldots, k$ all the

$$A_{j,i+1-j}, \quad j = 1, 2, \ldots, k$$  

are same while all the other $A_{j,i}$ become $A_{1,k}$. For example, if $A_{1,i} \neq A_{1,i}$ for $i = 1, 2, 3$ and $A_{1,i} = A_{1,i}$ for $i = 4, 5, \ldots, n$, then

$$A_{1,2} = A_{2,1}, \quad A_{1,3} = A_{2,2} = A_{3,1} \neq A_{1,3}.$$  

(20)

while all other $A_{j,i}$ are $A_{1,k}$. This is depicted in figure 8 where the oscillators with same amplitude but which are different from $A_{1,k}$ are connected by lines, while for the oscillators represented by open circles $A_{j,1} = A_{1,k}$. Because $A_{12} \neq A_{13}$ the two set of oscillators numbered as $(j, i) \rightarrow (1, 2)$ and $(2, 1)$ and $(1, 3), (2, 2)$ and $(3, 1)$ are joined by lines with different thickness. $A_{1}$ of the system (14) is same as that of the system (1).
illustrates the variation of the response

given by equation (21) on δ and ω for the coupled oscillators in (a) two dimensions and (b) one dimension with the first oscillator alone driven by the periodic force.

Figure 8. For the case of $A_{1,i} \neq A_{i}$ for $i = 1, 2, 3$ and $A_{1,j} = A_{1}$ for $i = 4, 5, \ldots, n$ the oscillators represented by solid circles connected by same lines have the same amplitudes but not equal to $A_{i}$ while the oscillators represented by open circles have the same amplitude $A_{1}$. We fix the values of the parameters as $d = 0.5$, $\alpha_{0} = 1$, $\beta = 1$ and $f = 0.1$. For $n \times n$, with $n = 100$, coupled oscillators we calculate $A_{j,j}$ by solving equations (16) and then compute the average gain of the response amplitude given by

$$\bar{G} = \frac{1}{n^{2}A_{1,1}} \sum_{j=1}^{n} \sum_{i=1}^{n} A_{j,i}. \quad (21)$$

Figure 9(a) presents $\bar{G}$ as a function of the coupling strength $\delta$ and the frequency $\omega$ of the driving force. In figure 9(b), the result for the coupled oscillators in one dimension is shown for comparison. Both figures 9(a) and (b) are almost identical.

Figure 10 illustrates the variation of the response amplitude $A_{j,i}$ with $(j, i)$ for three fixed values of $\delta$ with $\omega = 0.7$. For $\delta = 0.6 < \delta_{c}(= f/A_{1})$ the $A_{j,i}$ decays to zero with increase in $j$ and $i$. Enhanced undamped signal propagation with $A_{j,i}$ evolving to a saturation along all the directions can be clearly seen in figures 10(b) and (c) for $\delta = 0.7$ and $1.5 > \delta_{c}$.

4. Randomly coupled and forced oscillators in two dimensions

In this section, we consider two more configurations of coupled oscillators in two dimensions. In the previous section, we analysed the network where the first oscillator alone is driven by a periodic force, while the other oscillators are coupled unidirectionally. Now, we consider the following two types of coupled oscillators:

1. Randomly coupled oscillators. All the oscillators are driven by the periodic force $f \cos \omega t$. For $n \times n$ oscillators, the total number of connections is $2n(n - 1)$. Out of these only, $N_{C} \%$ of total connections chosen randomly are switched on. Figure 11 shows an example of the network with 50\% of the total connections are set on.

2. Randomly forced oscillators. All the oscillators except the first oscillator are coupled unidirectionally. However, periodic force is applied to only $N_{C} \%$ of the total number $(n^{2})$ of oscillators chosen randomly.

With reference to the above types of coupled network systems, it is worth to mention the relevance of such network configurations in neuroscience. In the nervous system, the structure permitting to transmit an information from one neuron to another is the synapse. In neuroscience, synaptic plasticity is the ability of the nervous system to modify the neural network to realize an enhanced or depressed efficacy of synaptic transmission, and experimentally several forms of synaptic plasticity have been measured. Response of a neural network can be modified by changing the value of coupling strength or establishing new connections or breaking certain connections in the network [42, 43]. Such changes in a neural circuitry lead to the recovery of functions after brain injury and mental illness. Synaptic plasticity is found to play important roles in fast network configuration and fast adaptation in free operand learning.

Interestingly, for the randomly coupled oscillators and randomly forced oscillators also, the theoretical approach used in the previous sections can be employed. The equation of motion of the randomly coupled oscillators is

$$\ddot{x}_{1,i} + \alpha_{0}^{2}\dot{x}_{1,i} + \beta x_{1,i} = f \cos \omega t, \quad (22a)$$

$$\ddot{x}_{j,i} + \alpha_{0}^{2}\dot{x}_{j,i} + \beta x_{j,i} = \delta_{i,i-1}x_{1,i-1} + f \cos \omega t, \quad (22b)$$

$$\ddot{x}_{j,1} + \alpha_{0}^{2}\dot{x}_{j,1} + \beta x_{j,1} = \delta_{j-1,j-1}x_{j-1,1} + f \cos \omega t, \quad (22c)$$

\[\]
we plot $G$ of (b) $\delta$ (a) $\delta$ $\omega_j$ $\omega_i$. The values of the other parameters are $d = 0.5, \omega_0^2 = 1, \beta = 1, f = 0.1$ and $\omega = 0.7$.

Figure 10. Three-dimensional plot of the response amplitude $A_{j,i}$ of the system (14) as a function of $i$ and $j$ for three fixed values of the coupling strength $\delta$. The values of the other parameters are $d = 0.5, \omega_0^2 = 1, \beta = 1, f = 0.1$ and $\omega = 0.7$.

Figure 11. An example of $5 \times 5$ two-dimensional oscillators in which 50% of the one-way couplings chosen randomly are set on. The other connections are switched off. All the oscillators are driven by the periodic force $f \cos \omega t$.

\[ x_{j,i} + \alpha x_{j,i} + \omega_0^2 x_{j,i} + \beta x_{j,i}^3 = \frac{1}{4} (\delta_{j,i-1}x_{j,i-1} + \delta_{j-1,i}x_{j-1,i}) + f \cos \omega t. \]  

In equations (22) $\delta_{j,i-1} = \delta$, a constant if the one-way coupling between the oscillators $(j, i)$ and $(j, i-1)$ is set on otherwise $\delta_{j,i-1} = 0$. Similarly, we can set the value $\delta$ or 0 to the quantities $\delta_{k,l}$ in equations (22). For system (22), the equations for the response amplitudes $A_{j,i}$ are given by equation (16a) with

\[ F_{1,1} = f^2, \quad F_{1,i} = \delta_{j,i-1}^2 A_{j,i-1}^2 + f^2, \]  

\[ F_{j,1} = \frac{1}{4} \left( \delta_{j-1,i} A_{j-1,i}^2 + \delta_{j,i-1} A_{j,i-1}^2 \right) + \frac{1}{2} \delta_{j-1,i} \delta_{j,i-1} A_{j-1,i} A_{j,i-1} \cos (\theta - \theta_{j-1,i}), \]  

where $j = 2, 3, \ldots, n, i = 2, 3, \ldots, n$. $\theta_{j,i}$ can be obtained from

\[ \frac{d}{d \omega} A_{j,i} = \frac{1}{4} \delta_{j-1,i} A_{j-1,i} \sin (\theta - \theta_{j-1,i}) + \frac{1}{2} \delta_{j,i-1} A_{j,i-1} \sin (\theta - \theta_{j,i-1}) + f \sin \theta_{j,i}, \]  

where $\theta_{1,i}$ is given by equation (16f), while $\theta_{0,i}$ and $\theta_{i,0}$ are all zero because the starting values of $i$ and $j$ are 1.

For a fixed $N_C$ and for one realization of the coupled oscillators, we calculate $A_{j,i}$ from equations (16a) and (23) and then compute an average gain factor $\overline{G}$ given by equation (21) of the response amplitudes. Repeating this procedure for different realizations of coupled oscillators, we obtain the corresponding $\overline{G}$. Then we compute the average value of $\overline{G}$ and denote it as $\langle \overline{G} \rangle$. We fix $d = 0.5, \omega_0^2 = 1, \beta = 1, f = 0.1$ and $n = 100$. In figure 12 we plot $\langle \overline{G} \rangle$ as a function of $N_R$, the number of different realizations of coupled oscillators, for $N_C = 40\%$, $\delta = 2$ and $\omega = 1.28$. $\langle \overline{G} \rangle$ increases with $N_R$ and becomes almost constant for $N_R > 70$. For $N_R = 100$ we found $\langle \overline{G} \rangle = 4.374$ with standard deviation 0.089. In the case of $N_R = 200$, $\overline{G}$ = 4.370 with standard deviation 0.089. In our further analysis we fix $N_R = 100$. Although $N_R = 100$ is quite small, for each realization of coupled

Figure 12. Variation of $\langle G \rangle$ as a function of $N_R$ for the randomly coupled $100 \times 100$ oscillators in two dimensions with $N_C = 40\%$. The values of the parameters are $d = 0.5, \omega_0^2 = 1, \beta = 1, f = 0.1, \delta = 2$ and $\omega = 1.28$. 

\[ \overline{G} = \frac{1}{N} \sum_{i=1}^{N} \overline{G}_i, \]  

where $\overline{G}_i$ is the average gain factor for the $i$th realization of the coupled oscillators.
oscillators, $\bar{G}$ is calculated averaging over $10^4$ oscillators which is quite large.

Figure 13 shows $\bar{G}$ versus $\delta$ and $\omega$ for four values of $N_C$. In the undamped signal propagation regions of $(\delta, \omega)$, the quantity $\bar{G}$ increases with increase in $N_C$.

Compare figure 13(d) where $N_C = 100\%$ (i.e. all the oscillators are coupled and driven by the periodic force $f \cos \omega t$) with figure 9(a) corresponding to the network in which all the oscillators coupled but the first oscillator alone driven by the periodic force. For a range of values of $\delta$ and $\omega$, the values of $\langle \bar{G} \rangle$ in figure 13(d) are lower than that in figure 9(a). The important result is that driving all the oscillators by the periodic force $f \cos \omega t$ does not increase $\langle \bar{G} \rangle$ compared to the case of the first oscillator alone driven. To illustrate the effect of driving in all the oscillators in figure 14(a), we plot $A_{j,i}$ versus $j$ and $i$ for $N_C = 100\%$. This figure can be compared with figure 10(c) where only the first oscillator is driven. All the oscillators exhibit periodic motion with period $T = 2\pi/\omega$. Therefore, the one-way coupling term can be viewed as a second external periodic force. Figure 14(b) shows the variation of $\theta_{j,i}$ with $j$ and $i$. $\theta_{j,i}$ varies with $j$ and $i$. This is clear from figure 14(c). The motion of the oscillators is periodic (with different amplitude), but they are not synchronized. That is, the irregular variation of $A_{j,i}$ and reduction in their values is due to the phase difference between the applied periodic force and the coupling term in equations (22).

Finally, we consider the randomly forced coupled oscillators. For this system also we apply the theoretical method used for the system (22) and compute $\langle \bar{G} \rangle$. Figure 15 displays $\langle \bar{G} \rangle$ versus $\delta$ and $\omega$ for $N_C = 1$ and $40\%$. We do
not observe any appreciable difference between the cases of $N_C = 1$ and $40\%$.

5. Conclusions

In conclusion, in the unidirectionally coupled Duffing oscillators in both one- and two dimensions with the first oscillator alone driven by a periodic force, we have shown the occurrence of enhanced signal propagation. Above a critical value of the coupling strength, the response amplitude increases with increase in the number of oscillators and reaches a saturation. That is, there exist a critical number of oscillators to have a maximum response and a further increase in the number of oscillators is ineffective in enhancing the response. The applied perturbation method enables us to predict (i) the response amplitude of each oscillator, (ii) the saturation value of the response amplitude and (iii) the critical value of the coupling strength above which undamped signal propagation occurs. Increase in the coupling strength leads to a decrease in the width of the hysteresis and above a critical value of it hysteresis is suppressed; however, jump phenomenon persists. We have shown the applicability of the perturbation method for the cases of the coupled oscillators with fraction of total number of oscillators chosen for coupling and forcing.

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Figure 15. Variation of gain factor response of the unidirectionally coupled oscillators with $N_c$ % of the total number of oscillators chosen randomly driven by the periodic force.
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