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Intensity of singular stress fields of an embedded fibre under pull-out force

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Abstract. Previous experimental studies of fibre pull-out test show two dangerous points on
the interface. Failure usually occurs at the bonded end of the fibre (Point A) or at the entry
point on the surface of the matrix (Point E). Both points have different singular stress fields
which causes crack initiation, crack propagation, and final failure. In this paper, intensity of
singular stress fields (ISSF) at the fibre bonded end A and ISSF at the intersection point E of
the fibre and the surface are discussed. The analysis method focuses on calculating the finite
element method (FEM) stress ratio by using a reference model and an unknown model. In the
unknown model and the reference model, same FEM mesh pattern is applied. To analyse the
ISSF at A, the body force method solution is used as the reference model. To analyse the ISSF
at E, the reciprocal work contour integral method (RWCIM) solution is used as the reference
model. Then, the two ISSFs are compared and discussed by varying the fibre embedded
length \( l_{in} \). When \( l_{in} \) is shorter, the singular stress at A is larger than the singular stress at E.
When \( l_{in} \) is longer, the singular stress at E is larger than the ISSF at A.

1. Introduction
Fibre reinforced composites are very common nowadays in construction, marine industry, aerospace
technology, transportation sector, consumer products and industrial equipment [1-4]. In fibre
reinforced composites, fibres are mainly used to enhance load carrying capacity. Indeed, composites
are produced when two or more materials or phases are used together to give a combination of
properties that cannot be achieved otherwise [4]. The mechanical properties of fibre-reinforced
plastics are governed, to a great extent, by the adhesion of the matrix to the fibre surface, i.e. the
interface bond strength [5]. Many different alternative test set-ups and experimental techniques have
been developed in recent years to gain more insight into the basic mechanisms dominating the
properties of the fibre/matrix interface. One of the most popular is the pull-out test in which a single
fibre or bar is pulled out off the surrounding matrix and the corresponding load versus displacement
relation, \( P(\omega) \), is recorded [6]. However, the inhomogeneous stress distribution on the interface with
stress concentrations at the fibre entry point and at the fibre bonded end cannot be indicated by \( P(\omega) \).
The resistance \( P(\omega) \) of the debonding and pull-out processes depends on the bond properties of the
fibre/matrix interface and the interfacial bond area. Commonly, the failure is assumed to occur when
the maximum load is reached. However, pull-out experiments revealed that the interface crack forms
long before the maximum load is reached [7]. Besides, a typical force-displacement curve from the
pull-out test was shown. The ‘kink’ in the figure indicates crack initiation [8].

S Zhandarov et al found the \( P(\omega) \) curve of pull-out and micro-bond tests is similar, i.e. crack
propagation may start from the fibre entry point E [8-10]. However, K-H Tsai et al observed that crack initiate at the fibre bonded end A during the fibre pull-out test [11,12]. In a rod pull out test that very similar to fibre pull-out test, C. Atkinson, et al. observed crack initiation sometimes occur at point A and sometimes occur at Point E [13].

2-D analytical model for the embedded fibre under pull-out force is shown in figure 1. As mentioned above, stress singularities arise at the fibre entry point E as well as at the fibre bonded end A in the theory of elasticity under idealized geometrical conditions, i.e. at sharp transitions between the two phases, even though no interface crack has formed [7]. The order of singularity arising at Point A and at Point E is not constant, but varies with the angle between the two surfaces as well with the elastic constants of fibre and matrix. The determination of stress intensity factors (i.e. ISSF) remains reserved to the analysis of an interface crack, even though the situation is more complex compared with the homogeneous case [7]. There are two different types of specific singular stress fields at point A as well as at point E. However, little research on these two singular stress fields is found. Therefore, attention is paid to discuss the ISSF by varying the fibre bonded length $l_{in}$.

![Figure 1. Reinforcing fibre under pull-out force.](image)

![Figure 2. (a) Rectangular inclusion in infinite plate and (b) Single lap joint.](image)

As is shown in figure 2(a), a rectangular inclusion is embedded in an infinite matrix under tension. Since the stress concentration zone is very small compared with the fibre diameter, the singular stress field at point A in Figure 1 is similar to that at point $A^\ast$ in figure 2. Therefore, configuration shown in figure 2(a) can be used as the reference problem of the fibre bond end. The order of singularity arising at point $A^\ast$ was discussed by D H Chen et al [14-16]. Singular stress field at point E in figure 1 and at point $E^\ast$ in figure 2 is similar. As is shown in figure 2(b), Point $E^\ast$ is the corner of the interface of a single lap joint, at which ISSF was discussed by Miyazaki et al [17,18]. The singular stress field in this case can be examined by the interaction line integral method (RWCIM method) [19].

2. The two specific singular stress fields

As the order of singularity arising at point A and at point E is different, the singular stress field at point A and at point E has to be calculated separately.

2.1. Singular stress around corner A

On the one hand, stress distribution near corner A can be expressed as follows [14-16,20]:

$$\sigma_{ij,k} = \frac{K_{A}^{A}}{r^{1-\alpha_{A}}} f_{ij,k}^{A}(\theta) \pm \frac{K_{H}^{A}}{r^{1-\alpha_{A}}} f_{ij,k}^{H}(\theta) \quad (ij = r, \theta, r\theta; \ k = M, I) \quad (1)$$

In order to calculate the normal stress $\sigma_{2}^{A}(r_{1})$ on the interface $r_{1}$, $ij = \theta$ and $\theta = 3\pi/4$ are used as shown in figure 1. Similarly, normal stress $\sigma_{2}^{A}(r_{2})$ on the interface $r_{2}$ can be calculated from equation
when $\theta = -3\pi/4$. Here, $f_{i,j}^I(\theta)$ and $f_{i,j}^{II}(\theta)$ can be calculated from equation (2) [14-16]. $K_A^I$ and $K_A^{II}$ are ISSFs, which are used for mode I and Mode II failure separately. $K_A^I = K_{i,j}^I \cdot f_{i,j}^I(\theta)$ and $K_A^{II} = K_{i,j}^{II} \cdot f_{i,j}^{II}(\theta)$.

For matrix $M (-3\pi/4 \leq \theta \leq 3\pi/4)$ [15,16]

$$f_{\theta,M}^I(\theta) = \frac{\lambda_1^I}{\sqrt{2\pi(\alpha-\beta)}} \left\{ [-\lambda_1^I(\alpha-\beta) \cos(\lambda_1^I\pi/2) + (1-\beta) \sin(\lambda_1^I\pi)] \times \cos[(\lambda_1^I + 1)\theta] \\
+ (\lambda_1^I + 1)(\alpha-\beta) \sin(\lambda_1^I\pi/2) \times \cos[(\lambda_1^I - 1)\theta] \right\} \quad (2)$$

$$f_{\theta,M}^{II}(\theta) = \frac{\lambda_2^I}{\sqrt{2\pi(\alpha-\beta)}} \left\{ [-\lambda_2^I(\alpha-\beta) \cos(\lambda_2^I\pi/2) - (1-\beta) \sin(\lambda_2^I\pi)] \times \sin[(\lambda_2^I + 1)\theta] \\
+ (\lambda_2^I + 1)(\alpha-\beta) \sin(\lambda_2^I\pi/2) \times \sin[(\lambda_2^I - 1)\theta] \right\} \quad (2)$$

Here, $\alpha$, $\beta$ denote Dundurs bimaterial parameters, and shear modulus $G_I$ and $G_M$ can be transformed from Young’s Modulus $E_I$, $E_M$ and Poisson’s Ratio $v_I$ and $v_M$.

$$\alpha = \frac{G_M(K_{j=1}^I - G_j(K_{j=1}^M + 1)}{G_M(K_{j=1}^I + G_j(K_{j=1}^M + 1)} \quad \beta = \frac{G_j(K_{j=1}^I - G_j(K_{j=1}^M - 1)}{G_j(K_{j=1}^I + G_j(K_{j=1}^M - 1)}$$

$$\kappa_i = \begin{cases} (3-v_I)/(1+v_I) & \text{(Plain stress)} \\ (3-4v_I) & \text{(Plain strain)} \end{cases} \quad (i = M, I) \quad (3)$$

Singular indexes $\lambda_1^I$, $\lambda_2^I$ around the corner $A$ can be calculated from the following equations (4) [15]. Here, the singular indexes $\lambda_1^I$, $\lambda_2^I$ have real values in the range $0 < \text{Re}(\lambda_i^I) < 1$ if $\beta(\alpha - \beta) > 0$, which are equivalent to the order of singularity. Here, $\gamma = \pi/2$.

$$D_1(\alpha, \beta, \gamma, \lambda) = (\alpha - \beta)^2 \lambda^2 \left[ 1 - \cos(2\gamma) \right] + 2\lambda(\alpha - \beta) \sin(\gamma) \left[ \sin(\lambda\gamma) + \sin[\lambda(2\pi - \gamma)] \right]$$

$$+ 2\lambda(\alpha - \beta) \sin(\gamma) \left[ \sin[\lambda(2\pi - \gamma)] - \sin(\lambda\gamma) \right]$$

$$+ (1 - \alpha^2) - (1 - \beta^2) \cos(2\lambda\pi) + (\alpha^2 - \beta^2) \cos[2\lambda(\gamma - \pi)] = 0 \quad (4)$$

$$D_2(\alpha, \beta, \gamma, \lambda) = (\alpha - \beta)^2 \lambda^2 \left[ 1 - \cos(2\gamma) \right] - 2\lambda(\alpha - \beta) \sin(\gamma) (\sin(\lambda\gamma) + \sin[\lambda(2\pi - \gamma)])$$

$$- 2\lambda(\alpha - \beta) \sin(\gamma) \left[ \sin[\lambda(2\pi - \gamma)] - \sin(\lambda\gamma) \right]$$

$$+ (1 - \alpha^2) - (1 - \beta^2) \cos(2\lambda\pi) + (\alpha^2 - \beta^2) \cos[2\lambda(\gamma - \pi)] = 0 \quad (4)$$

2.2. Singular stress around corner E

On the other hand, singular stress distribution around the corner $E$ along the interface $r_3$, as is shown in figure 1, can be expressed as follows [17,18]:

$$\sigma_x^E(\rho) = \frac{K_A^E}{r^{2-\lambda_1^E}} + \frac{K_A^{II}}{r^{2-\lambda_2^E}} \quad (5)$$

The value of singular index $\lambda_1^E$ and $\lambda_2^E$ can be determined from equation (6) [20-22]. Detail calculation of $K_A^E$ and $K_A^{II}$ in reference problem can be found in [17,18].

$$4\sin^2(\pi\lambda) \left( \sin^2 \left( \frac{\pi\lambda}{2} \right) - \lambda^2 \right) \beta^2 + 4\lambda^2 \sin^2(\pi\lambda) \alpha\beta + \left\{ \sin^2 \left( \frac{\pi\lambda}{2} \right) - \lambda^2 \right\} \alpha^2 - 4\lambda^2 \sin^2(\pi\lambda)\beta$$

$$- 2 \left\{ \alpha^2 \cos(2\pi\lambda) + \sin^2 \left( \frac{\pi\lambda}{2} \right) \cos(\pi\lambda) + \frac{1}{2} \sin^2(\pi\lambda) \right\} \alpha + \sin^2 \left( \frac{3\pi\lambda}{2} \right) - \lambda^2 = 0 \quad (6)$$

2.3. Singular index of different material combinations
Some results of singular index from equation (4) and equation (6) are shown in table 1. It can be verified that the singularity arising at point A and at point E is different.

Table 1. Singular index $\lambda$ value for some material combination.

| $\alpha$ | $\beta$ | $\lambda^A_1$ | $\lambda^A_2$ | $\lambda^E_1$ | $\lambda^E_2$ |
|---------|---------|---------------|---------------|--------------|--------------|
| $-0.90$ | $-0.40$ | 0.752493      | 0.748645      | 0.800102     | 0.997323     |
| $-0.90$ | $-0.20$ | 0.796059      | 0.699861      | 0.657967     | 0.999111     |
| $-0.80$ | $-0.40$ | 0.772894      | 0.847628      | 0.794890     | 0.988598     |
| $-0.80$ | $-0.20$ | 0.809649      | 0.791916      | 0.651598     | 0.996246     |
| $-0.9071$ | $-0.2016$ | 0.763235      | 0.621844      | 0.659165     | 0.999232     |

3. Method of analysis

Finite Element Method (FEM) was widely used for many engineering applications [23-28]. In this analysis software MSC Marc was used. Stress distribution along the interface $(r_1, r_2, r_3)$ were calculated by applying the same mesh pattern to the pull-out model and reference model. Thus stress ratio $[\sigma^A_{FEM}(r)/\sigma^A_{FEM}(r)]$ can be calculated between the pull-out model and the reference model. Similar method was used in [17,18].

As is shown in equation (7), $\sigma^A_{FEM}(r)$ is calculated from the stress distribution $\sigma^A_y(r_1)$ along the interface $r_1$ and $\sigma^A_y(r_2)$ along the interface $r_2$ by using the pull-out model (figure 1). $\sigma^A_{FEM}(r)$ is calculated from the stress distribution $\sigma^A_y(r_1)$ along the interface $r_1$ and $\sigma^A_y(r_2)$ along the interface $r_2$ by using the reference model (figure 2). Material properties are shown in table 2. As mentioned above, Fibre in figure 1 and Inclusion in figure 2(b) are set to be same material properties.

$$2\sigma^A_y(r) = \sigma^A_y(r_1) + \sigma^A_y(r_2), (r = r_1 = r_2);\quad 2\sigma^A_y(r) = \sigma^A_y(r_1) - \sigma^A_y(r_2), (r = r_1 = r_2). \quad (7)$$

Table 2. Material properties.

|                  | Matrix (Epon 828) | Fibre (Inclusion) |
|------------------|-------------------|-------------------|
| Young’s Modulus  | $E_I = 3.3$ GPa   | $E_J = 75$ GPa    |
| Poisson’s Ratio  | 0.35              | 0.17              |

Table 3. FEM Stress Ratio of Symmetrical type.

| $r/e_{min}$ | $\sigma^A_{FEM}(r)$ | $\sigma^A_{FEM}(r)$ | $\sigma^A_{FEM}(r)$ | $\sigma^A_{FEM}(r)$ |
|-------------|---------------------|---------------------|---------------------|---------------------|
| 0           | 13.20               | 1.136               | 0.00                | 16.53               | 1.136               |
| 0.5         | 10.88               | 1.135               | 0.50                | 13.62               | 1.136               |
| 1           | 8.383               | 1.135               | 1.00                | 10.49               | 1.135               |
| 1.5         | 7.741               | 1.134               | 1.50                | 9.681               | 1.135               |
| 2           | 7.603               | 1.134               | 2.00                | 9.507               | 1.135               |
In order to verify that the stress ratio between reference model and pull-out model is independent of mesh size, different mesh size $e_{\text{min}}$ is used. As shown in table 3, for different $e_{\text{min}}$ the value of the symmetrical type stress $\sigma_{l,FEM}^A(r)$ is different, while the ratio of stress distribution $[\sigma_{l,FEM}^A(r)/\sigma_{l,FEM}^B(r)]$ is constant. Same situation happen to $\sigma_{l,FEM}^A(r)$ and $[\sigma_{l,FEM}^A(r)/\sigma_{l,FEM}^B(r)]$, which is known as Skew-symmetrical type.

The normalized stress intensity factor $F_{l1}^*$ and $F_{l2}^*$ can be acquired on the basis of body force method [14]. Here $F_{l1}^*$ and $F_{l2}^*$ of the reference problem can be expressed as follows, in which $l_2 = D/2$.

$$F_{l1}^* = K_{l1A}^*/(\sigma^* \sqrt{\pi} l_2^{1/2} - \lambda_1^A) \quad F_{l2}^* = K_{l2A}^*/(\sigma^* \sqrt{\pi} l_2^{1/2} - \lambda_2^A) \quad (8)$$

The normalized stress intensity factor of the fibre pull-out problem can be defined similarly as follows:

$$F_{l1} = K_{l1A}^*/[(P/2l_2)\sqrt{\pi} l_2^{1/2} - \lambda_1^A] \quad F_{l2} = K_{l2A}^*/[(P/2l_2)\sqrt{\pi} l_2^{1/2} - \lambda_2^A] \quad (9)$$

As is shown in equations (10), by using the FEM stress ratio mentioned above and $F_{l1}^*$ acquired by body force method, $F_{l1}$ for the fibre pull-out problem can be calculated. So is $F_{l2}$.

$$\frac{F_{l1}}{F_{l1}^*} = \frac{\sigma_{l1,FEM}^A(r)}{\sigma_{l1,FEM}^B(r)} \quad \frac{F_{l2}}{F_{l2}^*} = \frac{\sigma_{l2,FEM}^A(r)}{\sigma_{l2,FEM}^B(r)} \quad (10)$$

4. Numerical results and discussion

In short fibre reinforced composite, the aspect ratio $2l_1/D$ of short fibres is about 30 [29]. In this study, we assume the diameter of the fibre, i.e. the width of the inclusion, is $D = 21 \mu m$ in the 2-D model. The total fibre length should be $l = 630 \mu m$. Therefore, the embedded length of the fibre $l_{in}$, from 30 $\mu m$ to 1000 $\mu m$, was considered in the material combination shown in table 2.

4.1. ISSF at point A of different $l_{in}$

The ISSF values $K_{l1A}^A$ and $K_{l2A}^A$ correspond to $\lambda_1^A$ and $\lambda_2^A$, respectively. It is found that, $K_{l1A}^A$ and $K_{l2A}^A$ decrease as the embedded length $l_{in}$ increases.

For example when $l_{in} = 150 \mu m$, $K_{l1A}^A = 0.02334 MPa \cdot m^{1-0.7632}$, whereas when $l_{in} = 300 \mu m$, $K_{l1A}^A = 0.01494 MPa \cdot m^{1-0.7632}$. The value of $K_{l1A}^A$ for $l_{in} = 300 \mu m$ is 36.0% less than that of $l_{in} = 150 \mu m$. Besides, when $l_{in} = 150 \mu m$, $K_{l2A}^A = 0.01519 MPa \cdot m^{1-0.6218}$, whereas when $l_{in} = 300 \mu m$, $K_{l2A}^A = 0.01017 MPa \cdot m^{1-0.6218}$. The value of $K_{l2A}^A$ for $l_{in} = 300 \mu m$ is 33.0% less than that of $l_{in} = 150 \mu m$. That is, the ISSF of the 1/2 embedded case is about 2/3 of that of 1/4 embedded case.

4.2. ISSF at point E

The ISSF values $K_{l1E}^A$ and $K_{l2E}^A$ correspond to $\lambda_1^E$ and $\lambda_2^E$, respectively. It is found that, $K_{l1E}^A$ and $K_{l2E}^A$ decrease as the embedded length $l_{in}$ increases.

For example, when about 1/4 of the length of the fibre is embedded, $l_{in} = 150 \mu m$, when about half of the fibre is embedded $l_{in} = 300 \mu m$, when $l_{in} = 150 \mu m$, $K_{l1E}^A = 0.03689 MPa \cdot m^{1-0.6592}$, whereas when $l_{in} = 300 \mu m$, $K_{l1E}^A = 0.03219 MPa \cdot m^{1-0.6592}$. The value for $l_{in} = 300 \mu m$ is 12.7% less than that of $l_{in} = 150 \mu m$. Besides, when $l_{in} = 150 \mu m$, $K_{l2E}^A = 0.04929 MPa \cdot m^{1-0.9992}$, whereas when $l_{in} = 300 \mu m$, $K_{l2E}^A = 0.05350 MPa \cdot m^{1-0.9992}$. The value for $l_{in} = 300 \mu m$ is 8.54% less than that of $l_{in} = 150 \mu m$. That is, the ISSF of the 1/2 embedded case is about 10% less than that of 1/4 embedded case.
4.3. Compare of ISSF at point A and at point E

As the singular indexes \((\lambda_1^A, \lambda_2^A), (\lambda_1^E, \lambda_2^E)\) are different at point A and at point E, it is not possible to compare ISSF directly. In order to compare the magnitude of the ISSF at point A and at Point E. Before crack initiation the radial tensile stress greatly exceed the shear stresses [7,30]. Therefore, \(\sigma_y^A(r_1)\) and \(\sigma_x^E(r_3)\) is calculated from the ISSFs. Here, \(\sigma_x^E(r_3)\) is the radial stress distribution along interface \(r_3\) in figure 1. As is shown in figure 3 when \(l_{in} = 100\mu m\), the peculiar stress at point A is larger than the specific stress at point E. Therefore, the crack initiation may occur from point A. However, figure 4 shows that when \(l_{in} = 1000\mu m\), radial stress at point A is larger than the specific stress at point E, and crack initiation may occur from point E.

![Figure 3](image3.png)

**Figure 3.** The dependence of \(\sigma_y^A(r_1)\) and \(\sigma_x^E(r_3)\) on \(r\) if \(l_{in} = 100\mu m\)

![Figure 4](image4.png)

**Figure 4.** The dependence of \(\sigma_y^A(r_1)\) and \(\sigma_x^E(r_3)\) on \(r\) if \(l_{in} = 1000\mu m\)

5. Conclusions

In this study, the singular stress fields generated at the entry point and at the bond end of an embedded fibre under pull-out force are discussed. Stress distribution around the two areas was compared, and the following conclusions were obtained.

- The ISSF \(K_{\sigma_\lambda A}^A\) and \(K_{\sigma_\lambda A}^E\) at point A decreases as the embedded length \(l_{in}\) gets longer.
- The ISSF \(K_{\sigma_\lambda E}^E\) and \(K_{\sigma_\lambda E}^E\) at point E decreases as the embedded length \(l_{in}\) gets longer.
- For the selected material combination the singular stress at the entry point is larger than that at the bonded end if \(l_{in} > 300\mu m\). On the other hand, the crack of embedded fibres may occur from bonded end if \(l_{in} < 300\mu m\).

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