Analytical Solutions for Transient and Steady State Beam Loading in Arbitrary Traveling Wave Accelerating Structures

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Abstract

Analytical solutions are derived for both transient and steady state gradient distributions in the travelling wave (TW) accelerating structures with arbitrary variation of parameters over the structure length. The results of the unloaded and beam loaded cases are presented. Finally the exact analytical shape of the RF pulse waveform was found in order to apply the transient beam loading compensation scheme during the structure filling time. The obtained theoretical formulas were crosschecked by direct numerical simulations on the CLIC main linac accelerating structure and demonstrated a good agreement. The proposed methods provide a fast and reliable tool for the initial stage of the TW structure analysis.
1. Introduction

The steady state theory of beam loading in electron linear accelerators was developed in the ‘50s by a number of authors both for constant impedance [1,2,3] and constant gradient [4] accelerating structures. They considered the equation for energy conservation in a volume between any two cross sections; the power gained by the beam or lost in the walls due to the Joule effect results in a reduction of the power flow. Later on, transient behavior was studied following a similar approach, but in this case, in addition to the power dissipated in the walls and gained by the beam, the transient change in the energy stored in the volume contributes to the power flow variation along the structure. Again, only constant impedance [5,6,7] or constant gradient [8,9] accelerating structures were considered.

However, traveling wave accelerating structures with arbitrary (neither constant impedance nor constant gradient) geometrical variations over the length are widely used today in order to optimize the acceleration structure and linac performance [10,11]. The relationships between structure length, input and average accelerating gradients are obtained by solving the energy conservation equation numerically. For the first time an analytical solution of the gradient profile in a loaded arbitrary TW structure was recently proposed in [12] but for the steady-state regime only. The comprehensive numerical analysis of an arbitrary TW structure including the effects of a signal dispersion was recently published in [13] using the circuit model and mode matching technique.

In this paper, generalized analytical solutions of the gradient distribution in the TW accelerating structure with an arbitrary variation of parameters over the structure length are presented for both steady state and transient regimes. It is based on the method
suggested earlier by one of the coauthors [14] and is similar to the classical approach [1−9]. Finally a simple analytical relation is derived that allows the input power ramp needed to create, at the end of the filling time, the field distribution inside the TW structure that coincides to the loaded field distribution in the presence of the beam to be determined. The compact analytical formulas so obtained give us a better understanding of the physics of TW structures and provide a tool for a fast preliminary structure optimization.

The following definitions are used throughout the paper:

\( P \) – Power flow through the structure cross section

\( W * \) – Stored energy per unit length

\( \omega \) – Circular frequency

\( Q * \) – Quality factor

\( G * \) – Loaded accelerating gradient

\( \tilde{G} * \) – Unloaded accelerating gradient

\( I \) – Beam current

\( v_g * \) – Group velocity

\( \rho * \) – Normalized shunt impedance, often called \( R/Q \), where \( R \) is the shunt impedance per unit length

\( z \) – Longitudinal coordinate

where * denotes that continuous parameters are averaged over the structure period and represent the effective values of an individual cell.

The following assumptions are used: a) the structure is perfectly matched at both ends and has no internal reflections, b) all dispersion effects that limit field rise time: \( t_r >> c/\omega v_g \), where \( c \) is the speed of light, are neglected, c) time separation between two neighboring
bunches and time of flight of the beam through the structure are much less than the filling
time of the structure.

2. Steady State Regime

The basic traveling wave structure relations are:

\[ P = W v_g \]  \hspace{1cm} (2.1)

\[ W = \frac{G^2}{\omega \rho} \]  \hspace{1cm} (2.2)

Energy conservation including wall losses and the interaction with the beam gives:

\[ \frac{dP}{dz} = -\frac{W \omega}{Q} - GI \]  \hspace{1cm} (2.3)

Using Eq. (2.2) in the derivation of the power flow Eq. (2.1) yields:

\[ \frac{dP}{dz} = W \frac{dv_g}{dz} + v_g \frac{dW}{dz} = G^2 \frac{dv_g}{\omega \rho \omega} + \frac{v_g}{\omega} \left[ \frac{2G}{\rho} \frac{dG}{dz} - \frac{G^2}{\rho^2} \frac{d\rho}{dz} \right] \]  \hspace{1cm} (2.4)

Substituting Eq. (2.4) into Eq. (2.3) and using Eq. (2.2) results in the first order non-
homogeneous differential equation with variable coefficients:

\[ \frac{dG}{dz} = -G(z) \alpha(z) - \beta(z) \]  \hspace{1cm} (2.5)

where \[ \alpha(z) = \frac{1}{2} \left[ \frac{1}{v_g} \frac{dv_g}{dz} - \frac{1}{\rho} \frac{d\rho}{dz} + \frac{\omega}{v_g Q} \right], \quad \beta(z) = I \frac{\omega \rho}{2v_g} \]. The solution of the non-
homogeneous differential Eq. (2.5), \( G(z) \), can be presented as a product of the solution of
the homogeneous equation \( \tilde{G}(z) \) and a function \( C(z) \):

\[ G(z) = \tilde{G}(z) \cdot C(z) \]  \hspace{1cm} (2.6)

where
\[
\frac{d\tilde{G}}{dz} = -\tilde{G}(z)\alpha(z) \tag{2.7}
\]

Substituting Eq. (2.6) into Eq. (2.5) and using Eq. (2.7) yields:

\[
\frac{dC(z)}{dz} = -\frac{\beta(z)}{G(z)} \tag{2.8}
\]

Integrating Eq. (2.8) gives:

\[
C(z) = -\int_0^{\tilde{z}} \frac{\beta(z')}{G(z')} dz' + C_1,
\]

where the constant \(C_1 = 1\) (taking into account the initial condition \(G(0) = \tilde{G}(0)\)) and \(z'\) is a local integration variable.

Therefore the general solution of Eq. (2.5) is:

\[
G(z) = \tilde{G}(z) \left[ -\int_0^{\tilde{z}} \frac{\beta(z')}{G(z')} dz' + 1 \right] \tag{2.9}
\]

The solution for the homogeneous Eq. (2.7) is:

\[
\tilde{G}(z) = G_0 e^{-\int_0^{\tilde{z}} \alpha(z') dz'} \tag{2.10}
\]

where \(G_0 = G(0)\) is a gradient at the beginning of accelerating structure and can be found from initial conditions:

\[
G_0 = \sqrt{\frac{\alpha \rho(0) P_0}{\nu_g(0)}} \tag{2.11}
\]

where \(P_0\) is input RF power.

The integral of function \(\alpha(z)\) can be simplified using analytical solutions:

\[
\int_0^{\tilde{z}} \alpha(z') dz' = \frac{1}{2} \left[ \ln \left( \frac{v_g(z)}{v_g(0)} \right) - \ln \left( \frac{\rho(z)}{\rho(0)} \right) + \int_0^{\tilde{z}} \frac{\omega}{v_g(z') Q(z')} dz' \right] \tag{2.12}
\]
Finally we can rewrite Eq. (2.10) as:

$$\tilde{G}(z) = G_0 \frac{v_g(0)}{v_g(z)} \sqrt{\frac{\rho(z)}{\rho(0)}} - \frac{1}{2} \int_0^z \frac{\omega}{\nu_g(z')Q(z')} dz' = G_0 g(z) \quad (2.13)$$

Eqs. (2.13) and (2.9) give us an expression for the loaded gradient:

$$G(z) = \tilde{G}(z) \left[ 1 - \int_0^z \frac{I}{G(z')} \frac{\omega \rho(z')}{2 \nu_g(z')} dz' \right] = G_0 g(z) - g(z) \int_0^z \frac{I}{g(z')} \frac{\omega \rho(z')}{2 \nu_k(z')} dz' \quad (2.14)$$

The first term on the right hand side of Eq. (2.14) is the solution of the homogeneous equation for the unloaded gradient obtained above in Eq. (2.13). The second term is the so-called beam induced gradient which is the difference between the loaded and unloaded gradient distributions.

Fig. 1 Individual cell geometry of the CLIC main linac accelerating structure with strong waveguide HOM damping (a), HFSS simulations of the surface electric (b) and magnetic (c) fields are shown.
Parameters of the CLIC main linac accelerating structure are summarized in the Table 1 [11]. They have been used to compare an accurate solution for an arbitrary variation of the TW structure parameters given by Eq. (2.14) to an approximate solution given in [4] where it has been assumed that the shunt impedance and Q-factor are constant in the range over which the group velocity changes and that they are both equal to their respective averages over the structure.

Table 1: Parameters of the CLIC main linac accelerating structure.

| Parameter                                                   | Value                  |
|-------------------------------------------------------------|------------------------|
| Average loaded accelerating gradient                        | 100 MV/m               |
| Frequency                                                   | 12 GHz                 |
| RF phase advance per cell                                   | $2\pi/3$ rad           |
| First, Middle and Last cell group velocity                  | 1.65, 1.2, 0.83 % of c |
| First, Middle and Last cell Q-factor (Cu)                   | 5536, 5635, 5738       |
| First, Middle and Last cell normalized shunt impedance      | 14587, 16220, 17954 $\Omega/m$ |
| Number of regular cells                                     | 26                     |
| Structure length including couplers                          | 230 mm                 |
| Bunch spacing                                               | 0.5 ns                 |
| Bunch population                                            | $3.7 \times 10^9$      |
| Number of bunches in the train                              | 312                    |
| Rise time                                                   | 22 ns                  |
| Filling time                                                | 67 ns                  |
| Peak input power                                            | 61.3 MW                |
Fig. 2 The HFSS simulation of the full CLIC accelerating structure. Electric field profile (a), input ($S_{11}$, red curve) and output ($S_{22}$, blue curve) couplers matching (b), phase advance per cell versus frequency (c) and internal reflections (SWR) in the structure (d) are shown. The phase advance per cell is equal to 120 degree and both coupler are matched better than -30 dB level at operating frequency of 11.994 GHz.

The unloaded gradient has been calculated for a 3D model of the structure using Ansoft HFSS [15], a frequency-domain finite-element code which takes into account internal reflections [11]. First of all the parameters of individual cells were calculated for the given phase advance, shunt impedance, group velocity and maximum EM-field strength on the surface. The result of individual cell optimization is shown in Fig. 1. Next, the input and output RF couplers were designed in order to match the TW structure with
feeding waveguide and RF loads. The detailed procedure of RF coupler design using Ansoft HFSS code is described in [16]. After that we made the simulation of full CLIC main linac accelerating structure and verified RF phase advance per cell and internal reflections using the well-known “Kroll’s” method [17] (see Fig. 2). Finally we derived the secondary values (stored energy and RF power flow per cell) necessary for the unloaded gradient calculation.

![Graph showing loaded and unloaded gradients](image)

**Fig. 3** Loaded (red) and unloaded (blue) gradients calculated accurately (solid) and approximately (dashed) for the CLIC main linac accelerating structure. In blue circles, the unloaded gradient calculated numerically is shown.

Both the loaded and unloaded gradients are shown in Fig. 3 for an input RF power of 61.3 MW which corresponds to an average loaded gradient of 100 MV/m. There is clearly a very good agreement between the accurate analytical solution and the numerical
simulation. In contrast, the approximate solution is quite different from the accurate solution due mainly to a significant (~30%) variation of the shunt impedance along the structure, see Table 1.

3. Transient Regime

The transient regime can also be derived analytically. The instantaneous energy conservation is given by:

\[ \frac{\partial W}{\partial t} = -\frac{dP}{dz} - \frac{W\omega}{Q} - GI \]  \hspace{1cm} (3.1)

Substituting Eqs. (2.1), (2.2) and (2.4) into Eq. (3.1) yields:

\[ \frac{\partial G}{\partial t} = \frac{G}{2} \frac{dv_g}{dz} - v_g \frac{dG}{dz} + \frac{v_g G}{2\rho} \frac{d\rho}{dz} - \frac{\omega G}{2Q} - \frac{\omega \rho}{2} I \]  \hspace{1cm} (3.2)

We assume the following initial conditions:

\[ G(0, t) = G_0(t), \hspace{0.5cm} \text{at} \ z = 0 \]
\[ G(z, 0) = 0, \hspace{0.5cm} \text{at} \ t = 0 \]  \hspace{1cm} (3.3)

Using the Laplace transformation of a function \( G(t) \):

\[ \hat{G}(p) = L[G(t)] = \int_0^\infty e^{-pt} G(t) dt, \hspace{0.5cm} t \geq 0, \text{its differentiation property:} L\left\{\frac{dG}{dt}\right\} = p\hat{G} - G(z, 0) \]

and taking into account Eqs. (3.3) we can write Eq. (3.2) as follows:

\[ p\hat{G} = \frac{\hat{G}}{2} \frac{dv_g}{dz} - v_g \frac{d\hat{G}}{dz} + \frac{v_g \hat{G}}{2\rho} \frac{d\rho}{dz} - \frac{\omega \hat{G}}{2Q} - \frac{\omega \rho}{2} \hat{J} \]  \hspace{1cm} (3.4)

First, we consider the unloaded case \( I = 0 \). In this case Eq. (3.4) becomes a homogeneous differential equation:
\[
\frac{d\hat{G}}{dz} = -\hat{G}(z, p)\hat{\alpha}(z, p) \tag{3.5}
\]

where \(\hat{\alpha}(z, p) = \frac{1}{2} \left[ \frac{1}{v_g} \frac{dv_g}{dz} - \frac{1}{\rho} \frac{d\rho}{dz} + \frac{\omega}{v_g Q} + \frac{2p}{v_g} \right] \). The solution of Eq. (3.5), obtained in a similar manner to the solution of Eq. (2.7), is:

\[
\hat{G}(z, p) = \hat{G}(0, p)e^{-\int_0^z \hat{\alpha}(z', p)dz'} = \hat{G}(0, p)g(z)e^{-p\int_0^z \frac{dz'}{v_g(z')}} \tag{3.7}
\]

where \(g(z)\) is defined in Eq. (2.13). The time-domain solution of Eq. (3.7) is obtained by applying the inverse Laplace transformation and its time shifting property:

\[
L^{-1}\{F(p)e^{-\tau p}\} = f(t-\tau)H(t-\tau), \text{ where } H(t-\tau) \text{ is the Heaviside step function and}
\]

\[
\tau(z) = \int_0^z \frac{dz'}{v_g(z')} \tag{3.8}
\]

is the signal time delay. Thus, the distribution of the unloaded gradient in time-domain along the structure is:

\[
\tilde{G}(z,t) = G_0[\tau(z) - \tau(z)]g(z)H[\tau(z)] \tag{3.9}
\]

or taking into account Eqs. (2.11) and (2.13) it can be expressed as a function of the input RF power:

\[
\tilde{G}(z,t) = \sqrt{P_0[\tau(z)]} \left[ \frac{\omega p(z)}{v_g(z)} \right] e^{-1} \left[ \frac{\omega}{2v_g(z')}Q(z') dz' \right] H[\tau(z)] \tag{3.10}
\]
The solution of non-homogeneous Eq. (3.4) is obtained in a similar manner to the solution of Eq. (2.5) as a product of the solution to the homogeneous equation \( \hat{G}(z, p) \) and a function \( \hat{C}(z, p) \): \( \hat{G}(z, p) = \hat{G}(z, p) \cdot \hat{C}(z, p) \) \hspace{1cm} (3.11)

Then Eq. (3.4) becomes:

\[
\frac{d\hat{G}}{dz} = \frac{d\hat{G}}{dz} \hat{C} + \frac{d\hat{C}}{dz} \hat{G} = -\hat{G}(z, p)\hat{\alpha}(z, p) - \hat{\beta}(z, p)
\] \hspace{1cm} (3.12)

where \( \hat{\beta}(z, p) = \hat{i} \frac{\alpha p}{2v_g} \).

Substituting Eqs. (3.5) and (3.11) into Eq. (3.12) yields:

\[
\hat{G} \frac{d\hat{C}}{dz} = -\hat{\beta}(z, p)
\] \hspace{1cm} (3.13)

and furthermore using Eqs. (3.7) and (3.8)

\[
\hat{G}(0, p)g(z)e^{-p\tau(z)} \frac{d\hat{C}}{dz} = -\hat{\beta}(z, p)
\] \hspace{1cm} (3.14)

The solution of Eq. (3.14) can be obtained by integration in the form:

\[
\hat{G}(0, p)\hat{C}(z, p) = -\int_0^z \frac{\hat{\beta}(z', p)}{g(z')} e^{p\tau(z')} dz' + \hat{C}_1(p)
\] \hspace{1cm} (3.15)

where \( \hat{C}_1(p) = \hat{G}(0, p) \) (taking into account the initial condition \( \hat{G}(0, p) = \hat{G}(0, p) \)). Note, that \( g(z) > 0 \). Finally, the general solution of Eq. (3.4) is derived using Eqs. (3.11 and 3.15):
Thus the time-dependent solution of Eq. (3.1) is obtained by applying the inverse Laplace transform to Eq. (3.16). Here again the time shifting property has been used:
\begin{align*}
G(z, t) &= G_0 [t - \tau(z)] \frac{g(z) H(t - \tau(z))}{\int_0^z \frac{(t' - \tau(z) + \tau(z')) H(t - \tau(z) + \tau(z'))}{g(z')} \omega(p(z')) dz'} \frac{\omega(p(z'))}{2v_g(z')} dz' \\
&= G(z, p) - g(z) \int_0^z \frac{\hat{\beta}(z', p)}{g(z')} \left[ e^{-p(\tau(z) - \tau(z'))} - e^{-p(\tau(z) - \tau(z'))} \right] dz' \\
&= \hat{G}(z, p) - \hat{\beta}(z', p) \frac{e^{-p(\tau(z) - \tau(z'))}}{g(z')} dz' \\
&= \hat{G}(z, p) - \frac{\hat{\beta}(z', p)}{g(z')} \left[ e^{-p(\tau(z) - \tau(z'))} - e^{-p(\tau(z) - \tau(z'))} \right] dz'
\end{align*}
(3.17)

where, \( \tau(z) \) is a function of the coordinate \( z \) and given by Eq. (3.8).

The first term on the right hand side of Eq. (3.17) is the solution of the homogeneous equation for the unloaded gradient obtained above in Eq. (3.9) or Eq. (3.10) in terms of the input power. The second term is the so-called beam induced gradient which is the difference between the loaded and unloaded gradient distributions.

For the CLIC main linac accelerating structure with the parameters from the Table 1, the time-dependent solution given by Eq. (3.17) during the transient related to structure filling and to beam injection is illustrated in Fig. 4 (a) and (b) respectively. In Fig. 5, the corresponding input power and beam-current time dependences are shown together with the unloaded, loaded and beam voltages defined as:
\begin{align*}
V(t) &= \int_0^L G(z, t) dz, \quad \tilde{V}(t) = \int_0^L \tilde{G}(z, t) dz, \quad V_b(t) = V(t) - \tilde{V}(t) \\
\end{align*}
(3.18)
ns $\gg c/\omega v_s$. The sum of the signal rise time $t_r$ and the structure filling time $t_f = \tau(L) = 66.7$ ns form the overall time of 89 ns corresponding to a transient of a cavity excitation. The total beam pulse length $t_b$ is $312 \times 0.5 = 156$ ns and the beam current $I = eN_e f_b = 1.6 \times 10^{-19} \cdot 3.72 \times 10^9 / (0.5 \times 10^{-9})$ is 1.2 A [11].
Fig. 4 The instantaneous unloaded (blue) and loaded (red) gradient distributions along the structure at different moments of time during the transient related to structure filling (a) and to the beam injection (b). The steady state solutions are shown as well (solid lines).

![Image of gradient distributions](image_url)

Fig. 5 The time dependence of the input RF power (blue) with a rise time of 22 ns, beam current (green) and the corresponding unloaded (black), loaded (red) and beam (light blue) voltages are shown.

![Image of time dependence](image_url)

A transient change in the loaded voltage just after the beam injection causes energy spread along a multi-bunch beam train. One possible method of transient beam loading compensation in TW structures is presented in the next section.

4. Compensation of the transient beam loading

The idea of transient beam loading compensation was proposed in 1993 at SLAC (USA) [18], where a linear ramp of the input RF amplitude has been applied to compensate the bunch-to-bunch energy variation to first order. Later a sophisticated
numerical algorithm for beam loading compensation was developed in the framework of the NLC project in order to calculate the precise profile of the RF pulse wavefront [19]. Recently the effectiveness of this method of transient beam loading compensation has been experimentally verified at KEK (Japan) [20, 21].

In this paper, the exact modification of the input power during a filling time $t_f$ needed to set the gradient distribution at the beam injection time equal to the steady-state loaded gradient solution $G(z)$ is calculated. Thus, the loaded voltage remains flat during the time when the beam is on because the transient related to the beam injection is fully compensated by the transient of the cavity excitation itself (at least in the framework of the applied analytical model).

Based on Eq. (3.9) the instantaneous gradient distribution at the moment of injection $t = t_f$ is:

$$G(z,t_f) = \tilde{G}(z,t_f) = G_0[t_f - \tau(z)]g(z)$$  \hspace{1cm} (4.1)

At the same time, the steady state beam loaded solution is expressed by (2.14). Equating Eq. (4.1) and Eq. (2.14) the required time dependence for the input gradient $G_0(t)$ during the filling time is obtained:

$$G_0[t_f - \tau(z)]g(z) = G_0(t_f)g(z) - g(z)\int_0^L \frac{I}{g(z')} \frac{\rho(z')}{2v_g(z')} dz'$$  \hspace{1cm} (4.2)

Where $G_0(t_f)$ is the steady-state value of the input gradient after injection. The input gradient in Eq. (4.2) indirectly depends on time. Introducing the function $z(t)$ as a solution of the following integral equation:

$$t(z) = \int_z^L \frac{dz'}{v_g(z')}$$  \hspace{1cm} (4.3)
Eq. (4.2) becomes an explicit function of time:

\[
G_0(t) = G_0(t_f) - \int_0^{z(t)} \frac{I}{g(z')} \frac{\omega \rho(z')}{2v_g(z')} \, dz'
\]  

(4.4)

An expression for the input RF power is derived using Eq. (2.11):

\[
P_0(t) = P_0(t_f) \left[ 1 - \sqrt{\frac{v_g(0)}{\omega \rho(0) P_0(t_f)}} \int_0^{z(t)} \frac{I}{g(z')} \frac{\omega \rho(z')}{2v_g(z')} \, dz' \right]^2
\]  

(4.5)

where \(P_0(t_f)\) is the steady-state value of the input RF power after injection.

Fig. 6 The input RF pulse profile with 22 ns rise time and ramp-up during the filling time for the transient beam loading compensation is shown in blue. Beam current injected exactly at the end of the ramp is shown in green. The corresponding unloaded, loaded and beam voltages are shown in black, red and light blue, respectively.
Fig. 7 The instantaneous unloaded gradient distribution along the structure at different moments of time is presented in (a). Special correction to the input RF pulse was applied (see Fig.6). In (b), the instantaneous unloaded gradient at different moments of time after beam injection is shown in blue. Solid lines represent the steady state distributions for loaded (red) and unloaded (blue) gradients. The beam injection time is 89 ns.
The solution of Eq. (4.5) is shown in Fig. 6 (blue) together with the beam current (green) injected exactly at the end of the ramp and the corresponding unloaded (black), loaded (red) and beam (light blue) voltages. The gradient distribution at different moments of time is presented for the compensated case in Fig. 7 (a) and (b) for the structure filling transient and the beam injection transient, respectively.

Summary

Analytical expressions for unloaded and loaded gradient distributions in travelling wave structures with arbitrary variation of parameters were derived in steady state and in transient. They were applied to the case of the CLIC main linac accelerating structure. The analytical solution agrees very well with the numerical solution obtained using finite-element code. On the other hand, it differs from the approximate solution obtained using expressions derived earlier in [4]. Finally the exact analytical solution was found for the wavefront of input RF pulse which theoretically provides exact compensation of the beam loading effect. The derived analytical formulas are very useful during the preliminary stages of structure design and later for structure efficiency optimization.

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References

1. K. Johnsen, Heavy Beam Loading in Linear Electron Accelerators, Proc. Phys. Soc. VB 64 (1951) 1062.

2. G. Saxon, Theory of Electron Beam Loading in Linear Accelerators, Proc. Phys. Soc. VB 67 (1954) 705.

3. R. B. Neal, Design of Linear Electron Accelerators with Beam Loading, J. Appl. Phys. 29, (1958), 1019; or M. L. Report no. 379, W. W. Hansen Laboratories, Stanford (1957).

4. R. B. Neal, Theory of the Constant Gradient Linear Electron Accelerator, M. L. Report no. 513, W. W. Hansen Laboratories, Stanford (1958).

5. R. B. Neal, Transient Beam Loading in Linear Electron Accelerators, M. L. Report no. 388. (1957).

6. J. E. Leiss, Transient Beam Loading in Linear electron Accelerators, NBS Internal Report (September 1958).

7. J. E. Leiss, Beam Loading and Transient Behavior in Travelling Wave Electron Linear Accelerators, In Linear Accelerators, ed., A. Septier and P.M. Lapostolle. Amsterdam: North-Holland Publishing Company, 1969,

8. L. Burnod, Regime transitoire dans les accelerateurs lineaires, LAL Report no. 17, Orsay (1961).

9. J. W. Wang, RF Property of Periodic Accelerating Structures for Linear Colliders, Ph.D. Dissertation, SLAC-339, (1989).

10. R. H. Miller et al., A Damped Detuned Structure for the Next Linear Collider, Proc. Linac96, Geneva (1996).
11. A. Grudiev, W. Wuensch, Design of the CLIC main linac accelerating structure for CLIC Conceptual design report, Proc. LINAC10, Tsukuba (2010).

12. G. Guignard and J. Hagel, Phys. Rev. ST Accel. Beams 3, 042001 (2000)

13. Roger M. Jones, Valery A. Dolgashev, and Juwen W. Wang, "Dispersion and energy compensation in high-gradient linacs for lepton colliders", Phys. Rev. ST Accel. Beams 12, 051001 (2009).

14. M. M. Karliner, O. A. Nezhevenko, B. M. Fomel, V. P. Yakovlev, "On Comparison of Accelerating Structures, Operating in the Stored Energy Mode", Preprint INP 86-146, Novosibirsk, 1986, (in Russian).

15. Ansoft HFSS, www.ansoft.com.

16. A. Lounine, T. Higo “Precise Designing of RF Coupler for Accelerator Structure”, Proc. of the 14-th Symposium on Accelerator Science and Technology (SAST2003), Tsukuba, Japan, November 11-13, 2003

17. Norman M. Kroll et al, “Applications of Time Domain Simulations to Coupler Design for Periodical Structure”, LINAC2000, Aug. 21-25, 2000, Monterey, California.

18. K. A. Thompson, R. D. Ruth, Simulation and Compensation of Multibunch Energy Variation in NLC, Proc. PAC93, Washington, D.C. (1993).

19. C. Nantista, C. Adolphsen, Calculating RF profiles for beam loading compensation of arbitrary current profiles, SLAC NLC-Note 25, 1997.

20. M. Satoh, T. Matsumoto et al., Beam loading compensation for acceleration of multi-bunch electron beam train, Nucl. Instrum. Methods Phys. Res., Sect. A 538, 2005.
21. M. Satoh, T. Koseki et al., Development of a new initial-beam-loading compensation system and its application to a free-electron-laser linac, Phys. Rev. ST Accel. and Beams 12, 013501 (2009)