Electrical Motor Current Signal Analysis using a Modulation Signal Bispectrum for the Fault Diagnosis of a Gearbox Downstream

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Abstract. Motor current signal analysis has been an effective way for many years of monitoring electrical machines themselves. However, little work has been carried out in using this technique for monitoring their downstream equipment because of difficulties in extracting small fault components in the measured current signals. This paper investigates the characteristics of electrical current signals for monitoring the faults from a downstream gearbox using a modulation signal bispectrum (MSB), including phase effects in extracting small modulating components in a noisy measurement. An analytical study is firstly performed to understand amplitude, frequency and phase characteristics of current signals due to faults. It then explores the performance of MSB analysis in detecting weak modulating components in current signals. Experimental study based on a 10kw two stage gearbox, driven by a three phase induction motor, shows that MSB peaks at different rotational frequencies can be based to quantify the severity of gear tooth breakage and the degrees of shaft misalignment. In addition, the type and location of a fault can be recognized based on the frequency at which the change of MSB peak is the highest among different frequencies.

Key words: Gearbox, Motor current signature analysis, Modulation signal bispectrum.

1. Introduction
In the last three decades a number of papers have been published on condition monitoring (CM) using MCSA for detection and diagnosis of mechanical defects. Benbouzid [1] reviewed MCSA (spectral analysis of the stator current) as a mechanism for fault detection and showed how MCSA could be used for the detection and localisation of abnormal electrical and mechanical conditions that indicate, or may lead to, a failure of an induction motor. Kar and Mohanty [2] used MCSA to detect gearbox faults instead of conventional vibration techniques. Based on a 7.5kw test rig he found that the MCSA can detect low frequencies using FFT analysis, but that high frequencies were difficult to detect. They also found that the use of MCSA combined with discrete wavelet transform DWT can be a good replacement for conventional vibration mentoring, but that continuous wavelet transforms CWT is not suitable for detecting fault or load fluctuations. In addition, Kia et al [3] developed a signal model for monitoring gearboxes using phase current and demonstrated the effectiveness of the spectrum method in extracting the frequency components relating to a signal stage gearbox driven by a 5.5kw.
To expand the capability of the current signature analysis, Rgeai [4] used the spectrum and conventional bispectrum to analyse phase current signals and demonstrated that these can be used to detect different degrees of tooth breakage of a two stage helical gearbox driven by an 11kW motor. More recently, Naid and Gu et al. [5, 6, 7] proposed a new modulation signal bispectrum (MSB) and showed reliable fault diagnosis based on the current signals from a reciprocating compressor in which the motor was subject to a high oscillatory load. To improve the performance of monitoring gearboxes, this study focuses on applying MSB to current signals from a two stage helical gearbox of 11kW under different incipient faults.

2. Motor Current Responses Due to Faults

2.1. Phase current response under healthy conditions

When a motor drive such as the gearbox system shown in Figure 2 is operating under healthy conditions, the ideal electromagnetic relationship of the driving motor can be examined in phase $A$ for a simple understanding of the fault effect. By neglecting the higher order harmonics and mechanical errors and referring to supply voltage signal, the current signal in phase $A$ for a healthy motor drive can be expressed [8, 9] as

$$i_A = \sqrt{2}I \cos(2\pi f_s t - \alpha_I)$$

Correspondingly, the magnetic flux in the motor stator is

$$\phi_A = \sqrt{2}\phi \cos(2\pi f_s t - \alpha_\phi)$$

The electrical torque produced by the interaction between the current and flux can be expressed as

$$T = 3P\phi I \sin(\alpha_I - \alpha_\phi)$$

where $I$ and $\phi$ denote the root mean squared (RMS) amplitudes of the supply current and linkage flux respectively, $\alpha_I$ and $\alpha_\phi$ are the phases of the current and flux referring to supply voltage, $f_s$ is the fundamental frequency of electrical supply and $P$ is the number of pole pairs.

2.2. Phase current response under faulty conditions

If there is a fault, such as a broken rotor bar, a damaged gear and a misaligned shaft occurring in the gear drive system, there will be an additional mechanical torque component induced by the fault. To balance this torque component, a corresponding electric torque will be generated by the motor. Supposing that the additional torque $\Delta T$ is a sinusoidal wave with a frequency $f_F$ and an associated current wave with an amplitude $I_F$ and phase $\alpha_F$, the oscillatory torque can be expressed

$$\Delta T = 3P\phi I_F \sin[2\pi f_F t - (\alpha_I - \alpha_\phi - \alpha_F)]$$

which causes a corresponding angular oscillation:

$$\Delta\alpha_F = \int \Delta\omega dt = \frac{3P^2\phi I_F}{4\pi^2 f_F^3 J} \sin[2\pi f_F t - (\alpha_\phi - \alpha_F)]$$

where $J$ is the inertia of the rotor system of motor. This angular oscillation modulates the phases of linkage flux in equation (2) and eventually leads to additional motor current signals in conjunction with associated phases to be approximated as [8]:
\[ i^F_A \approx \sqrt{2} I \cos(2\pi f_1 t - \alpha_f) \]
\[ \quad + \sqrt{2} I_s \cos[2\pi(f_s - f_F)t - \alpha_f - \alpha_F - \varphi] \]
\[ \quad - \sqrt{2} I_s \cos[2\pi(f_s + f_F)t - 2\alpha_f + \alpha_f - \alpha_F - \varphi] \quad (6) \]

where \( \varphi \) is the angular displacement of the motor equivalent circuit impedance at the supply frequency, \( I_1 \) and \( I_r \) are the RMS values of the lower sideband component and the upper sideband component, respectively, which are the currents induced by the back-EMF voltages produced by the flux variations at frequencies of \( f_s - f_F \) and \( f_s + f_F \). This simplified expression of current signal is employed widely for motor fault condition monitoring. By checking the amplitude of the sidebands through spectrum analysis, various faults such as rotor bar breakage and eccentricity can be diagnosed with a high degree of accuracy. It is noted in equation (5) that the amplitude of angular oscillation depends inversely on squared frequency and the inertia and hence that sidebands amplitudes also correlate to the frequency and inertia in the same way.

However, conventional spectrum analysis uses amplitude information only and overlooks the phase effect which also contains fault information, as shown in equation (6). The consequence of ignoring phase information may degrade diagnosis performance in the case of incipient faults when the sideband amplitude is very small and is masked by random noise. This is particularly true for diagnosing faults from the downstream mechanical system. Fortunately, bispectrum analysis allows the retention of both the amplitude and the phase information. In addition, it has good noise suppression and identification of nonlinear effects. Therefore, bispectrum can be a more powerful tool to process the current signals for resolving the weak fault signals and for better fault diagnosis.

3. Sideband Extraction using Modulation Signal Bispectrum

3.1. Modulation signal bispectrum

For a discrete time current signal \( x(t) \) its Discrete Fourier Transform \( X(f) \) is defined as:
\[ X(f) = \sum_{t=-\infty}^{\infty} x(t)e^{-2\pi j ft} \quad (7) \]

and the second-order measure (power spectrum) of \( x(t) \), can be computed by the formula
\[ P(f) = E[X(f)X^*(f)] \quad (8) \]

where \( X^*(f) \) is the complex conjugate of \( X(f) \) and \( E[\cdot] \) is the statistical expectation. The power spectrum is a linear transform and is a function of the frequency \( f \). Extending this definition to the measures of order 3 gives rise to the conventional bispectrum:
\[ B(f_1, f_2) = E[X(f_1)X(f_2)X^*(f_1 + f_2)] \quad (9) \]

Conventional bispectrum of equation (8) examines only the presence of quadratic phase coupling (QPC) from the harmonically related frequency components of \( f_1, f_2 \) and \( f_1 + f_2 \). It neglects the possibility that the occurrence of \( f_1 - f_2 \) may be also so due to the nonlinearity between \( f_1 \) and \( f_2 \). Because of this, it is not adequate to describe modulation signals such as the motor current signal in equation (6). Several approaches such as [10, 11, 12, 13] have been studied on the use of bispectrum for modulation signals and showed more compact results in analysing signals. To improve the
performance of the conventional bispectrum in describing the motor current signals, this study uses the signal bispectrum (MSB) suggested in [7]:

$$B_{MS}(f_1, f_2) = E[X(f_2 + f_1)X(f_2 - f_1)X^*(f_2)X^*(f_1)]$$

(10)

Unlike equation (9), this equation considers both \((f_1 + f_2)\) and \((f_1 - f_2)\) concurrently for possible QPC in MS signals. It illustrates that a bispectral peak will be clear at bifrequency \(B_{MS}(f_1, f_2)\) if \((f_1 + f_2)\) and \((f_1 - f_2)\) are both due to QPC between \(f_1\) and \(f_2\). This is more accurate and effective in representing the MS signal features.

The total phase of MS bispectrum in equation (9) is

$$\phi_{MS}(f_1, f_2) = \phi(f_2 + f_1) + \phi(f_2 - f_1) - \phi(f_2) - \phi(f_1)$$

(11)

when two components \(f_1\) and \(f_2\) are in coupling, their phases are related by

$$\phi(f_2 + f_1) = \phi(f_2) + \phi(f_1)$$

$$\phi(f_2 - f_1) = \phi(f_2) - \phi(f_1)$$

(12)

By substituting (11) into (10) the total phase of MS bispectrum will be zero and its amplitude will be the product of the four magnitudes, which is the maximum of the complex product. Therefore, a bispectral peak will appear at \((f_1, f_2)\). Especially, equation (9) now takes into account both \((f_1 + f_2)\) and \((f_1 - f_2)\) simultaneously for measuring the nonlinearity of modulation signals. If \((f_1 + f_2)\) and \((f_1 - f_2)\) are both due to nonlinear effect between \(f_1\) and \(f_2\), a bispectral peak will appear at bifrequency \((f_1, f_2)\). This is more accurate and apparent in representing the sideband characteristics of modulated signals.

For the current signal in equation (5), its MS bispectrum phase can be

$$\phi_{MS}(f_s, f_f) = 2\alpha_p + 2\alpha_f + 2\phi + \pi$$

(13)

Equation (13) shows that the bispectrum phase is connecting to only sideband components i.e. the angular position of faults and motor design and operating parameters, but exclude the phase of the fundamental component. It means that MS bispectrum is independent of the angular position of the rotor or the start point of a signal acquired. This will allow an average of MS bispectrum estimation to be performed using a data set collected or framed at any time. The average in turn will suppress random noise and unrelated components to obtain a reliable estimation of MS bispectrum and of the hidden modulating signal.

3.2. Sideband detection MSB

Because a motor current signal with mechanical faults contains a series of sideband components which appear mainly around the supply component, a bispectrum slice at the supply frequency is sufficient to characterize the sidebands for fault detection. By setting \(f_2\) of equation (9) into a constant frequency values such as \(f_2=50Hz\), MSB at supply frequency can be expressed as:

$$B_{MS}(f_1, f_s) = E\{X(f_s + f_1)X(f_s - f_1)X^*(f_s)X^*(f_1)\}$$

(14)

To extract sidebands only, the magnitude influences of the supply component on the resultant bispectrum amplitude can be removed by normalizing the supply component by its magnitude: \(X_n(f_s) = X(f_s)/|X(f_s)|\), which has unity magnitude but maintains the useful phase for averaging. In this way equation (13) becomes a sideband MSB:

$$B^n_{MS}(f_1, f_s) = E\{X(f_s + f_1)X(f_s - f_1)X^*_n(f_s)X^*_n(f_1)\}$$

(15)
And in the form of magnitude-phase, the sideband MSB is

\[ B^\text{MS} (f_1, f_s) = E \left( \| X(f_s + f_1) \| X(f_s - f_1) e^{j \delta \text{MS}}(f; f_1) \right) \]

(16)

These equations show that the magnitude of normalized MSB is only the products of the 2 sideband components. Under the condition that the amplitude of the sideband is the same, the amplitude from equation (14) or (15) can be directly compared with the sideband amplitude of power spectrum in equation (7). However, the amplitude from equation (14) is obtained by including phase influences. It will suppress any components with phase inconsistency due to random noise. On the other hand, the sideband amplitude includes noise influences because the power spectrum is unable to use phase information.

Corresponding MSB bicoherence for modulation signals only in [7] can be simplified as

\[ b^2_{\text{MS}} (f_1, f_s) = \frac{\left| B^\text{MS} (f_1, f_s) \right|^2}{E \left( \| X(f_s + f_1) X(f_s - f_1) \|^2 \right)} \]

(17)

to check the degree of random noise influences and hence confirm the existence of MSB peaks.

### 3.3. Performance evaluation

To evaluate the performance of the sideband MSB in detecting and diagnosing weak modulating components, a synthetic signal is created to have a carrier frequency of 50Hz, two modulating components of 50±24 Hz and 50±15Hz, an additional component of 10Hz and a high level of Gaussian noise. Figure 1(a) shows the power spectrum of the signal. To highlight the small modulating components, the figure does not display the full amplitude at 50Hz and at 10Hz, but shows the details of the disband changes with and without noise induced. It is clear that the spectrum has a high background level through the whole frequency range due to the high noise induced. Moreover, the amplitudes at the smaller and the larger sidebands are higher than those of the noise free signal. These higher amplitudes obviously will degrade detection performance.

In contrast, the MSB slice at 50Hz shown in Figure 1(b) presents a very clean representation for the signal with noise. It has an almost zero background floor and shows clearly the two frequency components accounting for the sidebands in the spectrum. In addition, these two components are confirmed by high amplitudes of MSB coherence shown in Figure 1(c).

Most importantly, the two amplitudes from MSB are very close to those of the noise-free case. It gives a more accurate estimation of the modulating signals because MSB takes phase information during the average process. In addition, the error of peak estimation is in a similar range, less than 2% for both the larger and smaller modulating signals. On the other hand, the peak estimation of power spectrum produces significantly greater errors and becomes worse when the signal-noise ratio (SNR) is lower for the case of small modulating signal. This shows that the MSB spectrum can produce more accurate detection and diagnosis.

In addition, MSB can also exclude any components that are not from modulation effects. The component at 10Hz has a small peak at 40Hz in MSB due to limited accuracy in numerical calculation. However, because the amplitude of MSB coherence is very low, it is easy to identify that this is not a real modulating component.

This simulation study enhances understandings that MSB can provide a much cleaner representation of the modulating process which creates a reliable basis for reliable detection and accurate estimation of modulating signals.
4. Test Setsups

To evaluate the performance of MSB analysis in diagnosing gearbox faults, a number of electric current signals were measured from a two stage gearbox test rig under different gear fault cases and operating conditions.

4.1. Test rig

Figure 2 shows the gearbox test rig used to test different gear conditions using motor current diagnosis in this research. It consists of a 3-phase induction motor, a two stage helical gearbox, flexible couplings and a DC motor which acts as a mechanical load. The induction motor (11kW, 1465 rpm and four magnetic poles) was supplied by a 3-phase power supply through a variable speed drive which allows different operating conditions to be specified. The schematic of the test rig setup is

Figure 1. Comparison of detection and diagnosis between power spectrum and MSB.
shown in Figure 2. The technical specifications of the helical gearbox are given in Table 1. In general the rig is sufficiently large to represent many industrial applications for evaluating the performance of current signal based condition monitoring. In addition, the 1st stage is configured as a speed increaser to explore detection performance in a wider frequency range.

![Figure 2. Gearbox test rig.](image)

![Figure 3. Schematic layout of the gearbox test rig.](image)

4.2. Fault simulation

The gear box faults examined in this study are tooth breakage and shaft misalignment, which occurs commonly in gear transmission and has been studied extensively in CM in surface vibration based fault detection. Vibration results show tooth breakage creates short transient vibration responses due to discontinuity in mesh stiffness. Obviously, the local mesh stiffness changes will also cause rotational oscillation. As discussed in Section 2, the rotational oscillation will lead to electric current change in a
form of modulation. Thus by detecting the degree of modulation, it is possible to find the level of tooth breakage.

To study the detection performance of MSB, four fault cases were examined. As detailed in table 2, the faults were created with different severity and locations. As both the tooth breakage and shaft misalignment can induce torque oscillations, but at different frequencies and locations, it is possible for MSB to detect and separate between these two types of faults and differing severity.

**Table 1. Specification of two-stage helical gearbox.**

| Description         | First Stage (Input Shaft 1) | Second Stage (Output Shaft 3) |
|---------------------|-----------------------------|------------------------------|
| Reduction ratio     | 0.8103 (speed increase)     | 4.5385 (speed decrease)      |
| Number of teeth     | \(Z_1/Z_2=58/47\)           | \(Z_3/Z_4=13/59\)           |
| Overlap ratio       | 2.890                       | 1.289                        |
| Contact ratio       | 1.450                       | 1.469                        |
| Shaft diameter      | 42mm                        | 40mm                         |
| Shaft frequency     | \(f_{r1}=24.42\)Hz          | \(f_{r2}=30.16\)Hz          |
| Mesh frequency      | 1416.36Hz                   | 391.76Hz                     |

**Table 2. Tested gear sets and fault details.**

| Gear ID | Fault location | Fault description                                         | Fault severity |
|---------|----------------|------------------------------------------------------------|----------------|
| Gear01  | Pinion on shaft 1 | 25% tooth breakage on one pinion tooth                      | small          |
| Gear10  | Pinion on shaft 1 | 50% tooth breakage on one pinion tooth                      | large          |
| Gear07  | Gear on shaft 2  | 25% tooth breakage on one gear tooth                        | small          |
| Gear12  | Pinion on shaft 1 | Healthy but with misalignment on shaft 3                    | 3 grades       |

4.3. Data acquisition

Electrical current signals are measured by a hall-effect sensor with a linear frequency response from 0Hz to 4kHz, which allows the content in a wide frequency range, especially around the supply fundamental of 50Hz, to be measured accurately. To examine the influence of the operating condition on fault diagnosis performance, electrical current signals were measured under loads: 12%, 20%, 30%, 38% and 44% of the full operating load of the AC motor. In addition, to create a sufficient number of averages and obtain reliable results, data acquisition for each gear condition was repeated three times. To benchmark the performance of the current signal based fault detection, vibration from the gearbox casing and instantaneous angular speed were also measured. All of the measurements were sampled simultaneously at a rate of 100 kHz and with a data length of 16 seconds per channel at each acquisition.

5. Results and Discussions

The datasets for the 4 gear sets are processed to obtain their corresponding MSB respectively. To reduce the influences of spectral leakage, the segment size of Fourier Transform (FT) is set to be 402128 points, coving nearly 1000 cycles of the supply components. A Blackman-Harris window is applied to the segment to minimise the sidelopes of FT. In addition, the number of average is 90 for each MSB calculation.
5.1. Characteristics of MSB from a multistage gearbox

To find best detection frequency components, MSB has been explored up to 3000Hz. However, it was found that the most useful information lies in the range below 50Hz where gear shaft related components show high amplitudes and exhibit high correlation with operating conditions. This confirms the fast attenuation modulation with frequency in equation (5). Therefore, the components in the low frequency are focused to develop diagnostic features. Figure 4 shows typical spectra of the current signals from Gear07 under both baseline and faulty cases. Figure 4(a) shows that MSB of the baseline has two primary components at 24.631Hz and 30.396Hz. These components are the shaft frequencies of fr1 and fr2 respectively and hence link to the transmission performance at the 1st stage of the gearbox. As both of them have high amplitudes at the baseline case it shows that they are mainly caused by manufacturing errors such as gear eccentricity on the gear set, although fr1 may have some contribution from the rotor eccentricity of the motor. Nevertheless, the existence of these two components shows that the current signal contains useful information about the conditions of the gear set.

For more detailed understanding, the MSB spectrum is presented with a magnification of the low amplitude components as shown in Figure 4(b). It shows that the MSB has a series of secondary components due to gear transmission in the low amplitude range. Although they have low amplitude in MSB, their MSB coherence of Figure 4(c) are significantly high to support the existence of the components. Firstly, the components at fr3=6.7617Hz and its high order harmonics can be clearly seen, showing that the current signals also contain the information of the transmission at the second stage, even though it is further away from the driving motor. Secondly, two more components due to the interactions of shaft 1 and shaft 2 can be observed at fr2-fr1=5.7648Hz and at 2xfr1-fr2=18.867, which shows again the high errors of the gear set at the 1st stage. Finally, a component at 9.455Hz is found to be the higher order harmonics of rotor bar pass frequency (BPF), which are identified by investigating the frequency variation characteristics with loads in high frequency ranges.

Comparing these identified components between the baseline and the faulty case has found that the amplitudes of the components linking to fr1, fr2 and their interactions: fr2-fr1 and 2xfr1-fr2 has a clear and humongous increase due to more oscillations from the fault. On the other hand, the amplitudes at fr3 and its harmonics show an increase only at 2fr2 whereas a decrease at fr2 and 4xfr2, which are not so consistent with the fault effect. These change characteristics indicate that the faulty tooth creates more oscillation on the first stage and this can be used to determine the presence, the location and severity of the fault. In particular, the most significant changes due to the fault are at fr1 and fr2 and they have high SNR. Therefore, they are taken in the following sections as key features to examine the MSB performance of diagnosing gear tooth breakage at the first stage.

However, the distinctive component at 39.51Hz for the faulty case is not from gear transmission because it has very low coherence amplitude. It has demonstrated again that the MSB analysis has the capability of excluding interference components.

5.2. MSB peaks with load variation

Figure 5 shows MSB peaks and their diagnosis performance under different load conditions based on Gear10 and Gear01. It is clear that from Figure 5 (a) and (b) that MSB peaks at fr1 and fr2 vary with load in a nonlinear behaviour. With load increases they increase in the small load range and then decrease in the high load ranges. Especially, the maximum value of fr1 is at 20% load whereas that of fr2 is at 30% load. These behaviours are not so consistent with the sideband amplitudes decreasing with increase in load due to an effect of load damper in [3]. Moreover, the distribution of amplitudes at fr1, fr2 and fr3 is very different in that MSB peaks are inversely proportional to the square of frequency values, shown in equation (5), providing that the inertia difference and error discrepancy between the three shafts is relatively small. Because of these inconsistencies and very high amplitudes at fr1 and fr2 compared with that of fr3 components, it has found that the gearbox under testing may have possible torsional resonances in the frequency range from 24Hz to 30Hz.
Based on these resonance effects, the nonlinear variation of MSB peaks with loads can be understood as a combined effect of the load damper and torsional resonance. For fr1 component, its peak starts with lower amplitude at 13% load. When the load increases to 30%, the frequency value decreases and moves closer to its resonance, which results in the maximum MSB peak. As the load increases further, the frequency moves even lower and moves away from the resonant point, which results in lower MSB amplitudes. In a similar way the component at fr2 shows the highest at 31% load where a second resonance may exist.

Figure 4. Frequency contents of motor current MSB results.

5.3. Gear tooth breakage diagnosis
From Figure 5 (a) and (b), it can also be seen that baseline MSB amplitudes at fr1 and fr2 are very different between the two gear sets. In particular, the amplitude of Gear10 is several times higher than that of Gear01 due to inherent manufacture errors. However, compared with the baseline for each gear set, MSB peaks at fr1 for corresponding faulty cases and becomes higher under nearly all load conditions. However, peaks at fr2 show an increase in a fewer cases. In addition, the amplitude
distribution with loads also shows a distinctive difference. These show that the faulty cases can be differentiated from their baseline in spite of the high manufacturing error.

However, because the peaks value of Gear10 are much higher than that of Gear01, it is difficult to tell which gear set has a severe fault using peak value directly. Instead, relative changes have to be employed for measuring the severity. Figure 5 (c) and (d) show relative changes with load at fr1 and fr2 respectively. It can be seen that the changes at fr1 are much higher, compared with that at fr2, showing that the fault is located at shaft 1. Moreover the changes of Gear01 at fr1 are higher than that of Gear10, showing that the fault at Gear01 is severe.

Figure 5. Gear tooth breakage diagnosis under different loads.

Figure 6. Gear shaft misalignment diagnosis under different loads.
5.4. Gear shaft misalignment diagnosis

To confirm the diagnostic performance, successive misalignments are induced to the end of output shaft end by adding different thicknesses of washers underneath the base of gearbox. Because of the rigid coupling connections, the misalignment will cause load oscillations at 1X. As shown in Figure 6 (c1) MSB peaks at fr3 increase gradually with the degrees of misalignments. However, the peaks at fr1 and fr2 do not show such a gradual increase change. This may show that the fault is located at shaft 3. This can be further confirmed by the relative changes presented in graphs of the bottom row. Especially, relative changes at fr3 also show the gradual changes but with a much higher amplitude. On the other hand, the changes at fr1 and fr2 are small. This conforms further that the fault is at shaft 3 and that MSB of current measurement is able to produce a good detection and diagnosis of different mechanical faults.

6. Conclusion

Mechanical faults occurring in an electrical motor drive result in a modulation effect to current signals. However, the modulating component is very small and difficult to be estimated accurately by conventional power spectrum analysis.

The modulation signal bispectrum (MSB) suggested in this paper is able to include the phase effects of modulating components and hence suppress noise influences to obtain an accurate estimation of the modulating components.

Experimental study based on a 10kw two stage gearbox driven by a three phase induction motor, shows that MSB peaks at different rotational frequencies can be used to quantify the severity of gear tooth breakage and the degree of shaft misalignment. In addition, the type and location of a fault can be determined based on the frequency at which the relative change of MSB peak is the highest among different rotational frequencies.

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