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Characterizing a lubricant additive for 1,3,4-tri-(2-octyldodecyl) cyclopentane: Computational study and experimental verification

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Abstract: In order to increase the life of spacecraft, it is important to improve the comprehensive lubrication performance. Multiple alkylated cyclopentane (MAC) lubricants are presently gaining wide acceptance for actual space applications; adding extreme pressure additive is a strategy to improve lubrication performance. In this study, taking 1,3,4-tri-(2-octyldodecyl) cyclopentane as base oil, tricresol phosphate (traditional additive) and tri-(2-octyldodecyl) phosphate (developmental additive) have been screened computationally for compatibility, shear film forming and energy dissipation. Theoretical results indicate that (a) tricresol phosphate additive is not suited for addition to 1,3,4-tri-(2-octyldodecyl) cyclopentane lubricant due to limited compatibility; (b) tri-(2-octyldodecyl) phosphate is an excellent lubricant additive due to its perfect compatibility, ease of forming a shear film on the surface of friction pairs, higher strength, and low energy dissipation; and (c) lubrication occurs through the solid-liquid composite lubrication mechanism. These theoretical results were confirmed experimentally.

Keywords: 1,3,4-tri-(2-octyldodecyl) cyclopentane; lubricant additive; computational screening; lubrication mechanism; experimental verification

1 Introduction

Development of space technology has led to an increase in the number of sensitive and active components. Meanwhile, the indispensability of spacecraft has necessitated an increase in their design life [1]. Therefore, it is necessary to improve the comprehensive lubrication performance of the lubricant.

Multiple alkylated cyclopentane (MAC) lubricants are presently gaining wide acceptance for space applications because of their chemical inertness, beneficial viscosity properties, low volatility [2], and low friction coefficients [3–6]. They are synthesized by reacting cyclopentadiene with alcohols in the presence of a strong base. The reaction products are then hydrogenated to produce the final products, which are a mixture of di-, tri-, tetra-, or penta-alkylated cyclopentanes [7]. In order to satisfy the application requirement under special working conditions and environments, extreme pressure additives are added as a strategy to further improve lubrication performance [5, 8–10]. Traditional simulated space environment experiments have proven that additives such as tricresol phosphate [11, 12] and lead naphthenate [13, 14] can significantly improve the wear resistance and other extreme pressure properties of MACs. Lead additives are toxic, so they will not be considered here.

In this study, 1,3,4-tri-(2-octyldodecyl) cyclopentane is taken as the base oil (denoted as L) and a suitable
lubricant additive is selected utilizing a computational protocol. Two different phosphate additives—tricresol phosphate (traditional additive) and tri-(2-octyldodecyl) phosphate (guessed additive based on the similarity of structures)—are taken as research objects. The structures of the base oil and additives are shown in Fig. 1.

2 Computational details

In this study, several elements are carefully explored to research lubrication performance. These are: (1) compatibility of base oil with additive, (2) strength of lubricated thin film on friction pairs, and (3) energy dissipation.

2.1 Compatibility of the additive with oil

The compatibility of the base oil with the additive was calculated using the Blends module \cite{15, 16}, which combines a modified Flory–Huggins model \cite{17} and molecular simulation techniques. The general expression for the energy of mixing of a binary system is shown by Eq. (1) \cite{17},

\[
\frac{\Delta G}{RT} = \frac{\phi_a}{n_a} \ln \phi_a + \frac{\phi_b}{n_b} \ln \phi_b + \chi \phi_a \phi_b
\]  

In the above equation, the free energy of mixing (per mole) is indicated by $\Delta G$, $\phi_a$ and $\phi_b$ correspond to the volume fractions of components $a$ and $b$. $n_a$ and $n_b$ are the degrees of polymerization of components $a$ and $b$. The first two terms represent the combinatorial entropy, and the last term is the free energy contribution due to the interaction. As per standard convention, $T$ is the absolute temperature and $R$ is the gas constant; $\chi$ herein can be defined through Eq. (2),

\[
\chi = \frac{E_{\text{mix}}}{RT}
\]  

$E_{\text{mix}}$ is the free energy difference between the mixed and pure states, computed by Eq. (3),

\[
E_{\text{mix}} = \frac{1}{2}Z(E_{ab} + E_{ba} - E_{ab} - E_{ba})
\]  

$Z$ is the coordination number and it is the intuitional index of compatibility. The larger the value of $Z$, the better compatibility there is. $E_{ab}$ and $E_{ba}$ are the binding energy between $a$ and $b$. For molecules, the binding energy is an ensemble average over molecular configurations, and similar distributions of $E_{ab}$, $E_{ba}$ and $E_{ab}$ combinations indicate an excellent compatibility.

2.2 Interaction between lubricated thin film and friction pairs

The strength of shear thin film on friction pairs can be evaluated from the following.

(1) Concentration profile of atoms nearby the friction pairs. The higher concentration of atoms near the friction pairs indicates the formation of a lubricant film. Herein, the concentration profile is obtained from the confined shear simulation, which is a non-equilibrium molecular dynamic method. The simulations are carried out by Forcite module in Material Studio version 6.0, Accelrys Software Inc. Iron (Fe) is taken as the material for the friction pairs since it is the main component of friction pairs for
most friction test experiments. Before carrying out confined shear simulation, two amorphous cells were constructed and were filled with 1,3,4-tri-(2-octyldodecyl) cyclopentane and tri-(2-octyldodecyl) phosphate, respectively. The densities of 1,3,4-tri-(2-octyldodecyl) cyclopentane and tri-(2-octyldodecyl) phosphate ester are 0.85 g/cm$^3$ and 1.10 g/cm$^3$, respectively. A short energy minimization (to optimize the cell) was initially performed on both amorphous cells, followed by an annealing cycle calculation. The initial temperature was 300 K, the mid-cycle temperature was 500 K, and Nose thermostating algorithm was used. The annealed results are shown in Fig. 2.

The Dreiding force field [18] has been adopted for simulation calculations. The relative sliding velocity of the friction pairs is 0.2 Å/ps, the time step is 1 fs, and the total simulation time is 20 ps. The initial temperature of the friction pairs is 300 K.

(2) Binding energy between lubricant molecules and friction pairs. Binding energy between lubricant molecules and friction pairs can be used to evaluate the strength of the lubricant film. A quench dynamic calculation was carried out to search for the minimum energy configuration of the base oil or additive molecule on friction. Fe (001) (14×14×2) supercells were taken as friction pairs and the binding energy was calculated by Eq. (4),

$$E_b = E_{\text{min}} - E_{\text{molecular}} - E_{\text{Fe}}$$  \hspace{1cm} (4)

$E_b$ is the binding energy of the base oil or additive with Fe (001), $E_{\text{min}}$ is the energy of minimum energy molecule configuration, $E_{\text{molecular}}$ represents the energy of free molecular of base oil or additive, and $E_{\text{Fe}}$ is the energy of Fe (001).

2.3 Energy dissipation

Energy dissipation can be evaluated by shear viscosity and local heating caused by internal friction among molecules. Shear viscosity and local heating will also be obtained by confined shear simulation results from the Forcite module. In the present context, lower local heating and shear viscosity indicates the better shear slip between interlayers, and thus leads to lower energy dissipation. The shear viscosity ($\eta$) was obtained from the numerical integration of the stress autocorrelation function (SAF), and the relation between shear viscosity and SAF can be expressed by Eq. (5) [19],

$$\eta = \frac{V}{kT} \int_0^\infty \langle \sigma_{\alpha\beta}(t)\sigma_{\alpha\beta}(0) \rangle dt$$  \hspace{1cm} (5)

$\sigma_{\alpha\beta}$ denotes the three equivalent off-diagonal elements of the stress tensor, $k$ is the Boltzmann constant, $T$ is already defined earlier, and $V$ is the shear velocity.

3 Results and discussion

3.1 Computational simulation results

Compatibility calculation results are presented first. Figures 3(a) and 3(b) show that the coordination numbers for tri-(2-octyldodecyl) phosphate (denoted as A$_1$) are 6; the distribution of binding energy of tri-(2-octyldodecyl) phosphate with base oil is completely identical. Meanwhile, the coordination numbers of tricresol phosphate (denoted as A$_2$) with base oil are 3, and there is some difference between A$_1$ and A$_2$ in the distribution of binding energy combination. Therefore, we can conclude that tricresol phosphate is not a good additive for base oil, and carrier will be needed to disperse it in base oil. However, tri-(2-octyldodecyl) phosphate has an excellent compatibility with base oil.

The relative concentration of atoms A$_1$ is much higher than L in the region adjacent to the friction pairs (Fig. 4). Thus, we can conclude that A$_1$ preferentially
forms a thin film on the surface of friction pairs compared with L.

The two quench dynamic search results (Figs. 5(a) and 5(b)) indicate that, at the minimum energy configuration of A, the molecule of A lies completely on Fe (001), and the corresponding binding energy is −511.96 (kcal/mol). In contrast, only a fraction of the chain of L lies on Fe (001) and the binding energy between L and Fe (001) is −177.66 (kcal/mol). Therefore, the minimum energy configuration and the higher binding energy of A with the surface of friction pairs demonstrate that A has the higher film strength.

Figure 6 shows that A has relatively low local heating during the confined shear course compared with L. The lower energy dissipation for A indicates a better lubrication performance when compared with L.

Analysis of confined shear simulation results produces shear viscosities of 6.67 and 13.39 centipoise for A and L, respectively. This indicates that A has a better interlay slip corresponding to lower dissipation, and this is also the main reason for the lower local heating during friction for A.

Based on the theoretical results, it can be concluded that tricresol phosphate is not a good additive for the base oil, and a carrier will be needed to disperse it in the base oil. Tri-(2-octyldodecyl) phosphate has excellent compatibility with base oil, strong film forming capability, and low energy dissipation, suggesting that it is an excellent lubricant additive. The difference of film forming capability between base oil and tri-(2-octyldodecyl) phosphate demonstrated that the lubrication mechanism is solid-liquid composite lubrication. Tri-(2-octyldodecyl) phosphate creates a solid-like thin film on the surface of the friction pairs, and 1,3,4-tri-(2-octyldodecyl) cyclopentane acts as the liquid component. Therefore, we can infer that lubrication performance will be improved substantially when compared the pure base oil.

### 3.2 Experiment certification results

In order to verify the conclusions of our calculations, tri-(2-octyldodecyl) phosphate was synthesized as per the literature [20]. Phosphorus oxychloride (Shanghai Chemical Reagent Co., Ltd.) and synthesized 2-octyl-1-dodecanol are the initial reagents. 2-octyl-1-dodecanol was synthesized according to the previously reported method [21]. The base oil tris-(2-octyldodecyl) cyclopentane was also synthesized following the same
procedure reported previously [22]. The chemical structure of the tris-(2-octyldodecyl) phosphate was characterized by IR (Nicolet iS10, Thermo Fisher Scientific) and NMR (Bruker, AVANCE III HD 400MHz) spectroscopy (See Appendix). The friction tests were performed on a ball-on-disk wear tester under high vacuum conditions ($3 \times 10^{-4}$ Pa). The sliding contact was brought by pushing a 9Cr18 steel ball (3 mm in diameter) on a rotating 9Cr18 steel disk ($\Phi$ 24 mm × 7.9 mm) with a load of 3 N. The sliding speed was 300 rpm and the test time was 1 h. The result of the experiment is shown in Fig. 7.

The coefficient of friction decreases substantially by up to 30% due to the addition of tris-(2-octyldodecyl) phosphate, which can powerfully verify the conclusion of our theoretical calculations. Thus we can conclude that the tri-(2-octyldodecyl) phosphate is an excellent additive for 1,3,4-tri-(2-octyldodecyl) cyclopentane lubricant.

4 Conclusions

Tricresol phosphate is not suited as a lubricant additive for 1,3,4-tri-(2-octyldodecyl) cyclopentane lubricant due to its limited compatibility. Tri-(2-octyldodecyl) phosphate is an excellent lubricant additive due to perfect compatibility, ease of forming shear film on friction pairs, high film strength, and low energy dissipation. The mechanism of action is solid-liquid composite lubrication and the theoretical results were verified by experiment. In this study, a new testing
method has been created for screening additives to lubricant oil.

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**Appendix: Characterization of tris-(2-octyldodecyl) phosphate**

![Molecular structure of tri-(2-octyldodecyl) phosphate](image1)

**Fig. A1** The molecular structure of tri-(2-octyldodecyl) phosphate used for analyzing the IR and NMR spectra.

![Infrared spectrum of tri-(2-octyldodecyl) phosphate](image2)

**Fig. A2** The infrared spectrum of tri-(2-octyldodecyl) phosphate. IR (liquid film): 2955.57 cm$^{-1}$ (–CH$_3$), 2923.04 cm$^{-1}$ (–CH$_2$), 2853.57 cm$^{-1}$ (–CH$_2$), 1466.36 cm$^{-1}$ (–CH$_3$, –CH$_2$), 1377.89 cm$^{-1}$ (–CH$_2$), 1244.00 cm$^{-1}$ (–P=O), 1027.76 cm$^{-1}$ (O=P–O–C), 721.53 cm$^{-1}$ (–(CH$_2$)$x$, $x$≥4).
Fig. A3  NMR spectra of tri-(2-octyldecyl) phosphate.
Fig. A3  (Continued)

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