Multi-Field Collaborative of Oil-Immersed Transformer for Distributed Energy Resources Temperature Rise Considering the Influence of Heat Transfer Oil

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The current calculation method of fluid–structure interaction (FSI) has a defect for oil-immersed transformers for distributed energy resources (DERs). In order to solve this problem, a calculation method is proposed for the temperature rise of oil-immersed transformers in this article. The vibration of insulating oil is considered in this method. Different from the temperature field model established by FSI, the effect of insulating oil vibration on the temperature field is considered. The structure field is introduced to establish the insulating oil vibration model. The temperature field correction coefficient is introduced by coupling the insulating oil vibration and the natural convection of the insulating oil. The result shows that compared with FSI, the results of the calculation method in this study are consistent with the experiment, and the temperature field distribution in the oil-immersed transformer can be calculated more accurately.

Keywords: DERs, correction coefficient, insulating oil, multiphysical interaction, oil-immersed transformer, temperature field

INTRODUCTION

With the increasing demand for energy in industrial production and people’s lives, as well as the contradiction between environmental problems and energy development, exploring how to improve the energy utilization rate on the premise of environmental friendliness has become a topic of common concern all over the world. With the development of microgrids and distributed generation technology, more and more clean energy is used in power systems, which puts forward new requirements for the stability of power grids. The operation state of a power transformer is directly related to the stability of the power grid, and in actual operation, insulation and thermal problems are the key factors affecting the operation state of the transformer. The loss will generate heat during the operation of the transformer, making the temperature rise. The presence of insulating oil will cause the heat accumulation above the core and winding, and the core and winding will locally overheat. Operating at high temperatures for a long time will destroy the insulation layer above the transformer, leading to thermal aging of the transformer, causing insulation breakdown and affecting the stability of the power system. Therefore, it is of great importance to obtain the accurate temperature field distribution in the transformer.

In order to balance the interests of IEO and users, a novel Stackelberg game-based optimization framework is proposed for the optimal scheduling of integrated demand response (IDR). IDR-
enabled integrated energy systems with uncertain renewable
generations are proposed in Li et al. (2021). A novel optimal
scheduling model based on chance-constrained programming
(CCP), aimed at minimizing the generation cost, is proposed for a
small-scale integrated energy system (IES) with CHP units,
thermal power units, renewable generations, and representative
auxiliary equipment (Li et al., 2020). The 2D simulation model
was established in Liu et al. (2017) for the simulation analysis of
the overall temperature rise of oil-immersed transformers by
means of mixed numerical calculation. A method for transformer
temperature field calculation was proposed in Liao et al. (2015)
on the basis of the finite element method and finite volume method,
in which a three-phase three-column transformer is calculated
and analyzed by the establishment of the electromagnetic–fluid–temperature field coupling model. The effectiveness of
the short-circuit method and regression analysis method was
verified by Yang et al. (2011). Investigation of the effect of sulfur
corrosion on the characteristics of transformer cellulose
insulation was proposed in Gao et al. (2020), and the results
were compared with those of fluent software to verify the
effectiveness of the method. Xia et al. (2017) proposed a
method for calculating the temperature rise of transformer
winding, which can effectively tackle the problem of low
experimental efficiency of winding temperature rise.

At present, the loss of transformer and the hot spot temperature
rise of winding are analyzed in most articles. However, the
temperature field of an oil-immersed transformer, especially the
calculation of the overall temperature rise, is seldom mentioned
(Wang et al., 2017; Lin et al., 2004). Due to the existence of insulating
oil, convective heat dissipation is the main heat dissipation mode of
an oil-immersed transformer. For coupled heat transfer, the thermal
boundary conditions are dynamically determined by the heat
exchange process, and there is no need to specify them in
advance (Wu et al., 2012; Zhang et al., 2014). However, the FSI
calculation is the distribution of the convective heat transfer
temperature field in the case of bulk rest of the transformer,
tending to focus on the flow velocity in the case of natural
convection or forced convection. Generally, it is suitable enough
to calculate the heat dissipation of dry transformer air. Given that the
insulating oil is much denser than air, the effect of forced vibration of
the insulating oil on the temperature field caused by the vibration of
the transformer body needs to be considered, making the calculation
result closer to the real value.

In order to improve the stability of distributed energy grid
operation and maintain the operation safety of the power
transformer, considering that the density of insulating oil is much
higher than that of air, the influence of forced vibration of insulating
oil on the temperature field caused by the transformer body’s
vibration needs to be considered. Therefore, this article proposes a
calculation method of transformer temperature rise considering
the influence of vibration of insulating oil. An oil-immersed transformer
can be an example: A multiphysical coupling model of an
electromagnetic–structural–fluid–temperature field is established.
The distribution of the temperature field in an oil-immersed
transformer is analyzed by means of multiphysical field coupling
simulation and verified by experiments. The experimental results
show that the error of the model is reduced by 3.35%.

VIBRATION MODEL OF INSULATING OIL

The oil-immersed transformer model and technical parameters
adopted in this study are shown in Figure 1 and Table 1.

As shown in Figure 2, we only consider the convective heat
dissipation of insulating oil based on FSI, namely, the turbulent
layer in Figure 3. In the turbulent layer, the insulating oil carries

![FIGURE 1 | Oil-immersed transformer model.](image1)

![TABLE 1 | Transformer technical parameters.](table1)

| Parameter      | Value | Parameter            | Value |
|----------------|-------|----------------------|-------|
| High voltage (kV) | 5     | No-load current rating (%) | 2.8   |
| Low voltage (V)  | 220   | Short impedance (%)    | 3     |
| Cooling mode     | ONAN  | Capacity (KVA)        | 20    |

![FIGURE 2 | Schematic diagram of insulating oil vibration.](image2)
away heat by flowing, playing the role of heat dissipation. However, the laminar flow layer still exists on the turbulent layer and the transformer’s surface. Due to the viscous effect of the fluid, the flow velocity of the laminar flow layer is close to zero, and the temperature changes rapidly in the normal vector direction of the laminar flow layer. The laminar flow layer will greatly hinder the heat dissipation effect of the turbulent layer. The influence of the vibration of the insulating oil on the temperature field is considered, which is caused by the vibration of the wall surface, improving the heat dissipation effect by disturbing the laminar flow layer while increasing the flow velocity of the turbulent layer. Therefore, the vibration of the insulating oil is first calculated and then explained in this article.

Vibration Analysis of Insulating Oil
Considering the influence of vibration on the fluid field, the fluid travels from the transformer body to the tank wall as a pressure wave, which is manifested as fluid vibration. The surface pressure generated here is the key to coupling the fluid field of insulating oil vibration with the structural field of the transformer body’s vibration. The propagation formula is as follows (Zhang et al., 2018):

$$\nabla^2 \rho = \frac{1}{c_0^2} \frac{\partial^2 \rho}{\partial t^2}, \tag{1}\nabla$$

where $\rho$ is the fluctuating pressure, $c_0$ is the propagation velocity of the vibration wave in oil, $t$ is the time variable, and $\nabla$ is the Laplace operator.

When the vibration wave in the insulating oil is transferred to the surface of the oil tank, the wave impedance of the insulating oil is smaller than that of the tank wall, so the transfer wave will reflect as shown in Figure 3. At the same time, the hydraulic pressure formed by the gravity of the insulating oil itself will also suppress the vibration of the transformer.

It can be seen that the reflected pressure wave is still a longitudinal wave, but this time, the phase difference is 180 degrees, and the subsequently transmitted vibration wave interacts with the disturbance. The core and winding through which the vibration wave passes can no longer remain in phase of zero, indicating that the insulating oil vibration phase is no longer consistent with the vibration source phase. Phase difference of $\Delta \varphi$ makes insulating oil vibration wave produce a reaction force for the surface of the core and winding, and vibration damping effect inhibition of the transformer.

Table 2 shows the physical property parameters of the transformer’s insulating oil. In order to analyze the influence of insulating oil vibration on the suppression effect of the transformer body’s vibration, in line with the analysis of transformer vibration, vibration fluid–solid interaction is adopted to analyze the vibration of the transformer’s insulating oil, as shown in Figure 4. As can be seen, the bottom of the transformer core is based on the fixed constraint, and the vibration of the insulating oil is dominated by the core column and the top one.

As shown in Figure 5, the transformer is separated and the vibration of the winding and the core is measured separately by the acceleration sensor. In this experiment, the self-designed distributed feedback laser (DFB) fiber Bragg grating sensor is used for vibration measurement to avoid electromagnetic interference of the traditional piezoelectric sensor and overvoltage damage of the sensor since the sensor needs to have direct contact with the surface of the transformer.

As shown in Figure 6, the measured result of winding vibration acceleration is smaller than that of the iron core, indicating that the vibration of the transformer is dominated by the magnetostriction of the iron core, and then the vibration calculation is dominated by the iron core under normal circumstances, which is consistent with the results shown in Figure 4. Under normal circumstances, the vibration is dominated by iron core.

Vibration Suppression Model of Insulating Oil
The reaction force will be produced by the vibration of the insulating oil, which can restrain the vibration of the transformer and reduce the amplitude of transformer vibration.

TABLE 2 | Insulating oil property parameters.

| Characteristic parameter       | Fitting formula |
|-------------------------------|-----------------|
| Density (kg/m³)               | 895             |
| Heat conductivity (W/mK)      | $0.13215 - 0.00024 \, T$ |
| Dynamic viscosity (mm/s²)     | $18.4 - 0.16 \, T$ |
| Dilatation coefficient (K)    | 0.00087         |

FIGURE 3 | Insulating oil vibration propagation.
vibration. As the vibration source of the insulating oil, the vibration of the transformer is essential for its accurate calculation. Therefore, the effect of oil damping on vibration is calculated by introducing a percentage coefficient of vibration suppression:

\[ S_{\text{oil}} = (1 - \varepsilon)S, \]  

(2)

where \( S \) represents the vibration displacement of the transformer without considering the vibration suppression of the insulating oil, \( S_{\text{oil}} \) represents the vibration displacement of the transformer with oil damping.
while considering the vibration suppression of the insulating oil, and ε (%) represents the ratio coefficient while considering the damping effect of the insulating oil. The correction coefficient ε can be affected by many factors. Since the vibration and temperature of the transformer vary significantly under different working conditions, it is assumed that the vibration intensity and temperature of the transformer with the correction coefficient ε are the influencing factors, which will be discussed, respectively, through finite element simulation.

As shown in Figure 7, the percentage coefficient δ is used to indicate the intensity of the vibration. It is worth mentioning that when the temperature remains unchanged, the value of the coefficient ε varies little with the change in δ, with the relative error of no more than 5%. It can be assumed to be a constant value by default, which is the case for both temperatures, indicating that the influence of the vibration intensity is weak in vibration suppression.

It should be noted that the ε is significantly affected by temperature. It can be seen in Figure 7 that its value at different temperatures varies greatly because the wave pressure and viscous impedance in the fluid are closely related to the hydrodynamic viscosity. Temperature has a significant effect on the dynamic viscosity of insulating oil and the modulus of silicon steel. Only considering the influence of these factors can help accurately calculate the vibration displacement of the oil-immersed transformer. Therefore, the influence of temperature on vibration model modification should be further considered.

As shown in Figure 8, by simulating the correction coefficient at different temperatures and keeping the vibration intensity constant, it can be noted that there is an approximately linear relationship between the temperature and the correction coefficient ε at 100°C.

Through formula (2), the value of ε can be calculated, and the influence of insulating oil on vibration displacement can be corrected. The ε parameter is set as a function of temperature T:

$$\epsilon = \epsilon_0 + K_{soil}e^{(T/\alpha)}$$

where $K_{soil}$, $\alpha$, and $\lambda$ are the influence coefficients ($K_{soil} = -0.97$; $\epsilon_0 = 19.33$; and $\lambda = 47.56$).

It can be found that considering the insulating oil vibration for the transformer vibration suppression model is also a function of temperature. With the increase in temperature, the vibration suppression effect of the insulating oil on the transformer becomes weaker and weaker due to the change in the physical parameters of the insulating oil with the temperature change. However, based on the calculations, the vibration of the insulating oil itself becomes more intense as the temperature increases. The vibration of the insulating oil can be accurately calculated with the suppression coefficient.

**CALCULATION OF THE INFLUENCE OF INSULATING OIL VIBRATION ON TEMPERATURE FIELD**

The influence of insulating oil vibration on the transformer temperature field is reflected in the influence on the surface heat dissipation coefficient (Yang and Tao, 2006). In general, when calculating convective heat dissipation by fluid–structure interaction, only the natural convective heat transfer generated by viscous volume expansion force is considered, while the fluid vibration acceleration generated by pressure resultant force is ignored. The formula is as follows (Tian et al., 2016):

$$\rho \left( \frac{\partial p}{\partial t} + v \frac{\partial p}{\partial z} + u \frac{\partial p}{\partial r} \right) + \rho \left( \frac{\partial \rho}{\partial z} + \frac{1}{r} \frac{\partial \rho}{\partial r} (ru) \right) = 0,$$

$$\rho \left( \frac{\partial v}{\partial t} + \frac{\partial \rho}{\partial r} + u \frac{\partial \rho}{\partial z} \right) = \left( \frac{\partial^2 u}{\partial r^2} + \frac{1}{r} \frac{\partial u}{\partial r} + u \frac{\partial^2 u}{\partial z^2} + \frac{\partial^2 u}{\partial z^2} \right) - \frac{\partial p}{\partial r},$$

$$\rho \left( \frac{\partial v}{\partial t} + \frac{\partial \rho}{\partial r} + u \frac{\partial \rho}{\partial z} \right) = \left( \frac{\partial^2 v}{\partial r^2} + \frac{1}{r} \frac{\partial v}{\partial r} + u \frac{\partial^2 v}{\partial z^2} + \frac{\partial^2 v}{\partial z^2} \right) - \rho g - \frac{\partial p}{\partial z},$$

$$\rho \left( \frac{\partial T}{\partial t} + \frac{\partial \rho}{\partial r} + u \frac{\partial \rho}{\partial z} \right) = \frac{\lambda}{c_p} \left( \frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} + \frac{\partial^2 T}{\partial z^2} \right),$$

where $t$ is time; $z$ and $r$ are the axial and radial directions of cylindrical coordinates, respectively; $u$ and $v$ are the velocity components in the $z$ direction and the $r$ direction, respectively; $\lambda$ is thermal conductivity; $\rho$ is the insulating oil density; $T$ is temperature; $P$ is liquid pressure; and $C_p$ is the specific heat capacity of the insulating oil.

In this study, the vibration velocity of the insulating oil is coupled with the natural convection velocity, and then the vibration velocity of the insulating oil is coupled with the temperature field on the basis of the natural expansion heat convection. The convective heat transfer coefficient of the transformer surface considering the vibration of the insulating oil can be expressed as follows:
where $k_f$ is the thermal conductivity of the insulating oil; $P_r$ is the Prandtl number; $L$ is the characteristic length; $q_r$ and $q_s$ are the buoyancy velocity and vibration velocity of the insulating oil microaggregate flowing through the transformer’s surface; $C$ is the cosine of the angle between the two velocities; and $k_p$, $m$, and $n$ are constants.

According to formula (4), the heat dissipation coefficient is directly proportional to the temperature rise and fluid flow velocity. The change of fluid velocity will change the temperature field. The fluid flow velocity of the insulating oil is not only composed of the natural buoyancy caused by thermal expansion, but also composed of the vibration velocity caused by the vibration of the insulating oil. According to the above calculation results of vibration of insulating oil, the comparison of two oil flow velocities when the temperature rise reaches 60°C is shown in Figure 9. The velocity of oil disturbed by the insulating oil is obviously lower than that of oil under natural convection.

The vibration of the insulating oil exerts a direct impact on the temperature field, without an impact on the transformer loss directly. However, it is difficult to calculate the overall distribution of the temperature field of the transformer after the coupling structure field, so the loss can be modified by formula (6) to achieve the equivalent correction of the influence of vibration on the temperature field:

$$P_T = (1 - \beta(V, T)) \left[ (k_h(f, B_m, T, V)B_m^2 + k_e(f, B_m, T, V)f^2B_m^2 + k_{ce}(f, B_m, T, V)(fB_m)^{3/2}) + P_{Cu} \right]$$

(6)

Considering the influence of the vibration of the insulating oil on the temperature field, the temperature correction coefficient $\beta$ is directly related to the vibration inhibition coefficient $\epsilon$. The correlation coefficient is obtained by fitting through simulation calculation, and the ambient temperature is 20°C, as shown in Figure 10. It can be seen that the trend of the correction coefficient $\epsilon$ is opposite to that of the temperature field correction coefficient $\beta$. With the increase in temperature, the inhibition of the vibration will be weakened by the insulating oil, while the vibration speed of the insulating oil will increase and the influence on the temperature field will be enhanced. It is shown that the temperature field model of insulating oil vibration is still a function of temperature.

**TRANSFORMER LOSS CALCULATION**

**Calculation Model of Iron Loss and Copper Loss**

The iron loss is divided into three basic loss components: hysteresis, eddy current, and abnormal loss (Yang and Tao,
In most cases, the extra loss is ignored. The classical trinomial formula is usually used to calculate

$$P_{Fe} = k_d B_m^2 + k_f f^2 B_m^2 + k_{ex} (B_m)^{3/2},$$

(7)

where $f$ is the frequency and $B$ is the peak magnetic density.

AC copper loss can be equivalent to the composition of DC winding copper loss and eddy current loss (Xue et al., 2018):

$$P_{ac} = P_{dc} + P_{eddy}$$

(8)

Transformer windings are mostly flat copper wires. The skin effect and proximity effect of the flat copper wire windings are shown in Figure 11.

By ignoring the influence of eddy current redistribution on the electric field distribution, the average loss in conductors per unit length can be estimated (Lou and Li, 2016; Tian et al., 2016):

$$P = \frac{m^2 n^2 \omega^2 B_n^2}{8 \pi \rho_c},$$

(9)

where $\rho_c$ is the resistivity of the conductor; $B_n$ is the peak of flux density; and for angular frequency, $m$ and $n$ are the length and width of the flat copper wire, respectively. The resistance is equivalent to the skin depth of the windings, and then the eddy current loss is corrected:

$$R_S = R \left[1 + \frac{1}{48 \pi^2} \left(\frac{mn}{\eta}\right)^2\right]$$

(10)

The high voltage side is a round copper wire winding, while the low voltage side is a flat copper wire winding. Although the distribution law of current density is the same, the difference in the shape makes the degree of skin effect and proximity effect different, resulting in different increase rates of equivalent resistance. Figure 12 shows the measurement of the winding equivalent resistance using LCR.

The results indicate that there is almost no difference in the growth rate of equivalent resistance between the two wires at low and medium frequencies, and the value slightly increases at high frequencies, but the increase is limited. Therefore, a unified model is adopted to calculate the equivalent resistance of the two wires.

**Temperature Correction Model**

There exists a big error of iron loss calculated by the ordinary trinomial formula. For the trinomial formula, temperature will directly affect the coefficient values of core loss. Notably, there are few studies on the influence of temperature on the loss of silicon steel at home and abroad, and the loss data of silicon steel with the influence of temperature are not perfect. The missing material data of the silicon steel must be obtained directly from experiments.

The temperature model of a silicon steel sheet is introduced to correct the coefficient of the trinomial formula, and the temperature feedback model is used for two-way interaction calculation, thus reducing the loss error (Tian et al., 2016):
\[ k_h(T, f, B_m) = k_{Th}k_{h,T_0}(f, B_m), \quad (11) \]
\[ k_e(T, f, B_m) = k_{Te}k_{e,T_0}(f, B_m), \quad (12) \]

where \( k_{Th} \) and \( k_{Te} \) are the correction coefficients considering temperature and \( k_{h,T_0} \) and \( k_{e,T_0} \) are the coefficients at time \( T_0 \).

The influence of temperature on iron loss can be obtained by measuring two sets of data, which can effectively solve the problem of imperfect material parameters of the silicon steel sheet. The correlation coefficient is obtained by using the no-load test of iron core as shown in Figure 13 (\( k_{Th} = 1.012 \) and \( k_{Te} = 1.004 \)).

**TEMPERATURE RISE CALCULATION**

The temperature simulation results of the electromagnetic–structural–fluid–temperature field of the multiphysical field interaction transformer filled with insulating oil established by considering the vibration correction model of the insulating oil are shown in Figure 14, and the oil temperature at the upper end of the transformer is obviously higher than that at the lower end.

In order to verify the influence of insulating oil vibration on transformer temperature, the basic temperature of the transformer is changed by changing the filling amount of the insulating oil. When the filling amount of the insulating oil is 80 and 100%, the experimental data and simulation data are compared, as shown in Table 3. Experiments for temperature rise to the top of the oil temperature in rated conditions have been measured. The upper insulating oil temperature of 55.2 and 54.75 °C in the model of the electromagnetic–structural–fluid–temperature field interaction correction, only considering the 57.3 and 56.41°C in the model of three physical field electromagnetic–fluid–temperature field of fluid–solid interaction, there is a difference. The temperature distribution does not change, but the difference between the upper insulating oil temperature rises by about 2.1 and 1.6°C. It can be seen that the error of model construction in this study is lower, with a temperature rise difference of 9.2 and 9.5%, which is reduced by 3.5 and 3.35%.

The multiphysical interaction model of the electromagnetic–structural–fluid–temperature field considering the vibration of the insulating oil is closer to the real value than the traditional FSI model. The multiphysical coupling method considering the influence of the vibration of the insulating oil is more reliable. It can be extended to all types of oil-immersed power transformers for the reason proposed in this article: The calculation method considers the influence of insulating oil vibration on the internal temperature field of the oil-immersed transformer as a function of temperature.

**CONCLUSION**

In this article, a more accurate method is proposed to calculate the temperature field of the transformer considering the vibration of the insulating oil. Taking an oil-immersed transformer as an example, the multiphysical coupling model of the electromagnetic–structural–fluid–temperature field considering the vibration of the insulating oil can be used to calculate the
temperature field of the oil-immersed transformer more accurately. The following conclusions are drawn:

1) A modified model of vibration suppression of insulating oil is established, and a temperature-dependent vibration suppression function of insulating oil is established by introducing the correction coefficient \( \varepsilon \).

2) A correction model for the influence of insulating oil vibration on the temperature field is established. As the temperature increases, the influence of heat conduction oil on the temperature field increases. At the same time, the influence of insulating oil vibration on temperature field considered in this study is still a function of temperature through correction coefficients \( \varepsilon \) and \( \beta \).

3) The multiphysical coupling model of the electromagnetic–structural–fluid–temperature field is established to calculate the temperature field of an oil-immersed transformer.

The model of vibration suppression function and the calculation method of temperature field considering the vibration of insulating oil are presented in this article, which is of great significance to prolong the service life of power transformers and maintain the stable operation of distributed energy resource systems.

**DATA AVAILABILITY STATEMENT**

The original contributions presented in the study are included in the article/Supplementary Material. Further inquiries can be directed to the corresponding author.

**AUTHOR CONTRIBUTIONS**

The individual contributions of the authors are as follows: data curation, KH; formal analysis, HL; methodology, WW; supervision, XL; validation, LZ; writing (original draft), AF. All authors have read and agreed to the published version of the manuscript.

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**Conflict of Interest:** KH, HL, and WW were employed by Guangzhou Power Supply Bureau Limited.

The remaining authors declare that the research was conducted in the absence of any commercial or financial relationships that could be construed as a potential conflict of interest.

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