Control of a velocity-sensitive audio-band quantum non-demolition interferometer

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The Sagnac speed meter interferometer topology can potentially provide enhanced sensitivity to gravitational waves in the audio-band compared to equivalent Michelson interferometers. A challenge with the Sagnac speed meter interferometer arises from the intrinsic lack of sensitivity at low frequencies where the velocity-proportional signal is smaller than the noise associated with the sensing of the signal. Using as an example the on-going proof-of-concept Sagnac speed meter experiment in Glasgow, we quantify the problem and present a solution involving the extraction of a small displacement-proportional signal. This displacement signal can be combined with the existing velocity signal to enhance low frequency sensitivity, and we derive optimal filters to accomplish this for different signal strengths. We show that the extraction of the displacement signal for low frequency control purposes can be performed without significantly the quantum non-demolition character of this type of interferometer.

I. INTRODUCTION

The Advanced LIGO gravitational wave observatories [1] made the first detection of gravitational waves from a binary black hole in 2015 [2] at the start of their joint observing run. In the coming years Advanced Virgo [3] in Italy and KAGRA [4] in Japan are expected to join the network, opening up a new window through which to observe the universe. These detectors are, fundamentally, Michelson interferometers with the addition of Fabry-Pérot arm cavities and other optical modifications [1, 5] to achieve exquisite sensitivity to motion of the test masses. Using the Local Lorenz gauge, an incident gravitational wave can be pictured as a change in length between the cavity mirrors in each arm of the interferometer. The primary degree of freedom a gravitational wave would excite is the differential mode of the cavity lengths, \( L_{\pm} \), which can be defined in terms of the position of the end test masses (ETMs) \( x_{TEM}^A \) and \( x_{TEM}^B \) in cavities A and B, respectively, as:

\[
L_{\pm} = \frac{x_{TEM}^A - x_{TEM}^B}{2}.
\]  

Quantum noise, the combination of quantum radiation pressure noise at low frequencies and quantum shot noise at high frequencies, restricts the ultimate sensitivity of classical Michelson interferometers at what is termed the Standard Quantum Limit (SQL) [6]. As quantum noise is expected to limit the sensitivity of the aforementioned second generation detectors over a wide frequency band, particular effort is being paid to its reduction in proposals for third-generation instruments such as the Einstein Telescope facility [7] and LIGO Cosmic Explorer [8].

Reduction of quantum noise below the SQL in a particular optical configuration involves a so-called quantum non-demolition (QND) measurement [6]: either the use of quantum correlations in the outgoing light or the enhancement of the test mass dynamics by means of light [9]. Quantum correlations arise naturally between the otherwise uncorrelated sources of quantum noise due to the interaction between the light and the motional degrees of freedom of the interferometer and can be used to effectively avoid radiation pressure noise contributions to the readout signal. This group of methods includes frequency-dependent squeezing [10], variational readout [10, 12] and speed meters [13, 14]. The latter approach implies the use of light-mirror interactions to create new test mass dynamics in order to increase their response to the gravitational wave action. This includes so-called “optical springs” [17, 21], “optical inertia” [22, 23] and “intracavity schemes” [24, 26].

The method on which we focus employs an interferometer topology intrinsically sensitive to a QND observable [27]. The measurement of velocity, itself approximately proportional to the QND observable of the free test mass momentum, is one way in which a reduction in quantum radiation pressure noise can be achieved. A proof-of-concept experiment is under way at the University of Glasgow to demonstrate an audio-band reduction of quantum radiation pressure noise in a Sagnac speed meter topology over an equivalent Michelson design [28]. This topology is being considered as an alternative to the Einstein Telescope’s Michelson interferometer design [29, 30]. The topology under investigation in the proof-of-concept experiment utilises a zero-area Sagnac inter-
ferometer with the addition of arm cavities based on the concept presented in ref. [15]. By design, this interferometer produces a signal at the beam splitter’s output port proportional to differential arm cavity mirror velocity, in contrast to the displacement-proportional signal sensed in the Michelson topology.

The presence of arm cavities within the proof-of-concept Sagnac speed meter gives rise to challenges not previously encountered in the control of gravitational wave detectors and other experiments involving Michelson or Sagnac interferometers, and this aspect will be addressed in the following work. In Section II we describe in more detail the proof-of-concept Sagnac speed meter experiment and its control requirements. We then describe a control strategy for the Sagnac speed meter’s differential degree of freedom based on that of Michelson designs, and demonstrate the control challenges this approach introduces. In Section III we present an alternative strategy which achieves adequate control of the interferometer to reach its design sensitivity over extended periods, and in Section IV we present a noise budget of the Sagnac speed meter using the alternative control strategy. A summary is provided in Section V.

II. VELOCITY CONTROL

Figure 1(a) shows a simplified optical layout of the proof-of-concept Sagnac speed meter. The main beam splitter (BS) splits the input field towards the two triangular arm cavities where they form counter-propagating modes. One mode from each arm cavity is coupled into the other via the inter-cavity mirror M1, and the other modes recombine at the main beam splitter.

Each arm cavity in the Sagnac speed meter is an independent degree of freedom and so changes to \( L_{(-)} \) lead to frequency-dependent signals at the output port. Motion of an arm cavity mirror imparts signal sidebands upon the counter-propagating modes. These modes have different optical path lengths to the beam splitter and so the signal at the output port contains the superposition of signals representing the mirror’s displacement from different points in time, which is analogous to velocity. At dc frequencies the two modes at the output port contain the same displacement information and the velocity signal is therefore zero (for a more complete description of the Sagnac speed meter’s behaviour, see, for example, Section IIb of [15]). Motion of M1 and BS imprints common phase changes on both counter-propagating modes and are not considered degrees of freedom for signals sensed at the output port.

The error signal (readout) representing \( L_{(-)} \) is sensed at the main beam splitter’s output port by means of a balanced homodyne detector (BHD, see Section II A [31]), as shown in the shaded green area in Figure 1(a). The frequency dependence of the phase quadrature signal at the output port \( s_{\text{BHD}} \) is given by the following relationship, ignoring the effect of losses (a full treatment of the effect of loss is given in ref. [32]):

\[
s_{\text{BHD}}(\Omega) \propto \frac{\Omega}{(\Omega^2 + \gamma_{\text{arm}}^2)} L_{(-)},
\]

for angular frequency \( \Omega \) and with arm cavity half-bandwidth \( \gamma_{\text{arm}} \) defined to be:

\[
\gamma_{\text{arm}} = \frac{c_0 T_{\text{ITM}}}{4 L_{\text{RT}}},
\]

for speed of light \( c_0 \), arm cavity input test mass (ITM) power transmissivity \( T_{\text{ITM}} \) and arm cavity round-trip length \( L_{\text{RT}} \).

Other terms in the response function dependent upon mirror mass, laser power and mechanical modes are not frequency dependent. Note that for \( \Omega \ll \gamma_{\text{arm}} \) the response is proportional to frequency, vanishing towards dc, as described above and shown in Figure 1(b).

In order to maintain peak BHD sensitivity to \( L_{(-)} \) and therefore gravitational waves, the positions of the cavity mirrors are controlled using linear inverting feedback, where an error signal is extracted and applied through a control law to cavity mirror actuators. In the experiment, voice coils and plate capacitor electrostatic drives (ESDs) [33] are used to actuate on the position of the end test masses within each cavity. This feedback maintains the interferometer close to its operating point within the bandwidth of the controller. In order to achieve the required stability, the relative position of the cavity mirrors must be controlled to within \( 3.5 \times 10^{-13} \) m rms (see Appendix A).

A. Balanced homodyne detection

The BHD consists of two high quantum-efficiency photodetectors sensing the reflected and transmitted fields from the BHD’s beam splitter. A local oscillator is incident upon the BHD’s beam splitter to provide gain for the velocity information encoded within the light from the main beam splitter. The difference current is converted to a voltage by an op-amp with transimpedance resistor \( R_T \) before being sent to the data acquisition system (DAQ). An example circuit for the balanced homodyne detector is shown in Figure 1(c). The op-amp introduces its own noise to the output, though a well-chosen op-amp will possess noise significantly lower than the signal representing \( L_{(-)} \) in the intended measurement band. In order for an op-amp to contribute less than 1% of the uncorrelated noise in the measurement, its noise must be at least a factor of 10 below the dominating noise source in the measurement band.

Op-amps used for control in audio-band interferometry typically possess a noise power spectrum inversely proportional to frequency (so-called flicker noise [34] Section 11.2.3) in the low audio band. As the BHD error signal is dependent upon the time derivative of the mirror positions, however, there will necessarily be frequencies
at which the op-amp noise will dominate the BHD error signal. This makes control of slow drifts of the arm cavity mirror positions impossible with the velocity readout, despite the op-amp being well-chosen for a measurement band above 100 Hz. This control problem with relation to the proof-of-concept experiment will be quantified in the following subsections.

B. Op-amp noise

To measure the effect of a suitable op-amp’s noise at low frequency, the output from an applicable BHD circuit was investigated. The circuit shown in Figure 2(a) was housed within a dark enclosure to minimise photocurrent, with one of the two op-amps within a Texas Instruments® OPA2227 integrated circuit being used to amplify the noise from the other by a factor of 100, to a level detectable by the DAQ. This op-amp is suitable for the BHD circuit shown in Figure 1(c) given the intended measurement band around a few hundreds of Hz [28].

The time series in Figure 2(b) shows a drift in the measured op-amp noise over the course of 16 days. An open channel on the DAQ was measured concurrently. A Fourier transform of the measured op-amp noise time series (Figure 2(c)) shows a combination of flicker noise and an additional slope possibly due to resistor current noise below around 1 Hz [35]. DAQ noise dominates above 4 Hz. The “Model (total)” spectral density in Figure 2(c) show the contributions to the measurements from the first op-amp $N_1$’s current and voltage noise and the Johnson-Nyquist noise of its transimpedance resistor $R_T$. This spectral density additionally contains the measured open channel noise summed in quadrature to show the agreement it has with the measurements down to around 1 Hz.

The op-amp noise drift produces an offset upon the BHD error signal which is to be fed back to the cavity mirror actuators, and thus op-amp noise directly contributes to cavity mirror displacement noise, affecting the experiment’s sensitivity to the arm cavity differential mode. Since the signal measured at the BHD represents cavity mirror velocity, it must necessarily drop below the noise at low frequencies where the velocity tends to zero.

C. Noise projection

To reach the desired sensitivity of the interferometer it is crucial to understand the noise characteristics associated with the sensing and control apparatus employed in the experiment. Individual noise sources, for example arising from the BHD op-amp electronics, can be projected into units of differential displacement-equivalent noise using the linear projection technique [36]. The sources of noise can be logically separated into two broad categories: sensing noise and displacement noise. Both sources of noise are fed back to the test masses because in
quantum shot noise, electronic noise including op-amp noise as modelled in Section II B and digitisation noise due to the analogue-to-digital converter (ADC).

Displacement noise sources directly influence the positions of the test mass surfaces being measured by the interferometer and are therefore transformed by the dynamics of the test masses \[32\]. As the readout variable in the BHD is the time derivative of position, the control system measures and actively suppresses these noise sources. Significant sources of displacement noise in the Sagnac speed meter experiment are quantum radiation pressure noise, seismic noise, suspension thermal noise \[37\] and coating brownian noise arising from the dielectric coatings present upon the cavity mirrors \[38\].

The noise projection for \(L(-)\), calculated using the numerical optomechanical simulation tool Optickle \[39\] and the control noise modelling tool Simulink\textsuperscript{Nb} \[40\], is shown in Figure 3(a). The root-mean-square (rms) differential displacement this creates is shown in Figure 3(b) as a function of time. It shows that, as the interferometer is held at its operating point, over a period of several hours the expected drift is large enough for the cavities to become uncontrollable (see Appendix A).

Although for sensing noise we only consider electronic and shot noise, in the experiment there will be other contributing forms of time-varying offset present upon the BHD error signal:

- residual local oscillator light due to temperature-driven imbalances in the BHD beam splitting ratio and photodetector quantum efficiencies;
- common mode arm cavity motion due to imbalanced beam splitting at the main beam splitter \[32\];
- thermo-electric potentials and op-amp drift in pre-amplifier and whitening electronics;
- any other time-varying effects.

As such, the estimated rms displacement shown in Figure 3(b) represents a “best case” scenario where the op-amp’s electronic noise is the dominant effect at low frequencies, and this drift becomes unacceptably large after a few hours. To allow for long term cavity stability it is essential for the error signal to contain a signal significantly above the electronic noise at low frequencies. In the next section we present a strategy for obtaining an error signal of suitable magnitude across the entire control bandwidth.

### III. VELOCITY-DISPLACEMENT CONTROL

Light from each counter-propagating mode is incident upon the inter-cavity steering mirror \(M_1\), and as such this is a natural port in which to separate the modes and sense the motion of each arm cavity (see the shaded blue region of Figure 1(a)). Using RF modulation, for instance via
the Pound-Drever-Hall (PDH) technique \cite{41}, it is possible to obtain a displacement error signal for each cavity that, unlike the velocity signal from the BHD, has flat response at dc, with a similar cavity pole frequency. The individual cavity PDH signals can be mixed to obtain a measurement of \( L_{(-)} \), and the frequency dependence of the signal \( s_{\text{PDH}} \) is, following ref. \cite{10}, given by:

\[
s_{\text{PDH}}(\Omega) \propto \sqrt{\frac{\gamma_{\text{arm}}}{(\Omega^2 + \gamma_{\text{arm}}^2)}} L_{(-)},
\]

(4)

ignoring again the effect of losses and constant terms as with Equation 2. Note that for \( \Omega \ll \gamma_{\text{arm}} \), the response is flat as expected for a displacement measurement and as such the PDH readout offers a suitable signal to sense \( L_{(-)} \) at low frequencies.

### A. Combined filter

The separate velocity and displacement readouts contain the same fundamental information about the position of the mirrors, albeit with different response functions. We can express the signal at output field \( i \) as a function of the \( k^{\text{th}} \) mode of motion, \( \hat{\delta}_{k,i}(\Omega) \), as \cite{10}:

\[
\hat{\delta}_{k,i}(\Omega) = L_k(\Omega) + \frac{\hat{n}_i(\Omega)}{R_{k,i}(\Omega)}
\]

(5)

where \( L_k \) is the position of mode \( k \), \( \hat{n}_i(\Omega) \) is the noise at field \( i \) and \( R_{k,i}(\Omega) \) is the optomechanical transfer function of mode \( k \) to field \( i \). The definition of a field in this case refers to that of a single signal sideband, \( \Omega \). The total time domain signal on a perfect sensor due to the \( k^{\text{th}} \) mode at the location of the output field will see a combination of the upper and lower signal sidebands:

\[
\hat{\delta}_{k,i}(t) = \int_0^\infty \frac{d\Omega}{2\pi} \left( \hat{\delta}_{k,i}(\omega_0 + \Omega) + \hat{\delta}_{k,i}^\dagger(\omega_0 - \Omega) \right) e^{-i\Omega t},
\]

(6)

where \( \omega_0 \) is the angular frequency of the carrier.

Classical noise sources associated with the test mass modes, such as thermal and seismic noise, are implicit in \( L \). The excess noise at each readout port is therefore due to \( \hat{n}_i \), the quantum vacuum entering at open ports within the interferometer. The presence of such vacuum noise limits the sensitivity of the interferometer in the measurement band. For this reason the reflectivity of \( M_1 \) must be chosen to be close to unity, therefore only a small amount of light is available to the displacement readout for use as a low frequency error signal.

By considering the response and noise characteristics of the BHD and PDH readouts it is possible to combine them with a filter in order to maximise the interferometer’s sensitivity across the full intended frequency range. A desirable crossover frequency for this filter is constrained from below by the signal-to-noise ratio of the BHD and from above by the noise introduced onto the feedback signal by the PDH readout. There is a 90° phase difference between the displacement and velocity signals at low frequencies and as such simply combining the PDH and BHD signals with dc gain produces a filter with a stable crossover that can be used as an error signal. The feedback of this combined filter output allows the displacement signal from the PDH to control the cavity mirrors at low frequencies where it is stronger, while letting the BHD signal provide feedback at higher frequencies where it yields the greatest response. The differential displacement-equivalent noise projection for a suitable combined filter is shown in Figure 4(a). The rms displacement in Figure 4(b) shows a clear reduction in residual displacement with respect to the feedback using only the velocity signal.
where $M_{m,i}^f(\Omega)$ represents the transfer function between input field $m$ and output field $i$ for signal sideband $\Omega$. The cross-correlation spectral density for unity noise at the $i$th and $j$th output channels, for the $k$th mode, is then [27]:

$$S_{k,ij}(\Omega) = \sum_{m=1}^{N_p} \left[ |M_{m,i}^{f*}(\Omega)M_{m,j}^{f}(\Omega) + M_{m,j}^{f*}(\Omega)M_{m,i}^{f}(\Omega)|^2 \right]$$

This reduces to the following form for noise entering the same port in which it exits:

$$S_{i,i} = \frac{1}{2} \left[ |M_{i,i}^{f}(\Omega)|^2 + |M_{i,i}^{f*}(\Omega)|^2 \right]$$

Assuming a filter $\alpha(\Omega)$ combines the BHD ($i = 1$) and PDH ($i = 2$) fields, its output for $L_k$ would be:

$$\hat{n}_{k,\text{combined}}(\Omega) = \alpha(\Omega) \hat{n}_{k,1}(\Omega) + (1 - \alpha(\Omega)) \hat{n}_{k,2}(\Omega)$$

The corresponding total noise power spectral density of the combined readout is then:

$$S_{\text{readout}} = |\alpha|^2 S_{n_1,n_1} + |1 - \alpha|^2 S_{n_2,n_2}$$

By considering cross-correlations in the quantum noise at the BHD and PDH readouts, it is possible to produce an optimal filter with which to combine the two in such a way as to minimise the total noise spectral density. The noise at each readout is the sum of the quantum noise inputs at open ports propagated through the interferometer with appropriate transfer functions, so we can rewrite $\hat{n}_i$ in Equation 5 in terms of the quantum noise amplitudes $\hat{q}_m$ entering at $N_p$ open ports:

$$\hat{n}_i(\Omega) = \sum_{m=1}^{N_p} M_{m,i}^f(\Omega) \hat{q}_m(\Omega),$$

where $M_{m,i}^f(\Omega)$ is the time series in (b) shows the root-mean-square differential displacement the combined filter causes. The combined filter provides a displacement reference at low frequencies and allows control of the Sagnac speed meter for significantly longer periods than with only a velocity readout.

### B. Optimal filter

By considering cross-correlations in the quantum noise at the BHD and PDH readouts, it is possible to produce an optimal filter with which to combine the two in such a way as to minimise the total noise spectral density. The noise at each readout is the sum of the quantum noise inputs at open ports propagated through the interferometer with appropriate transfer functions, so we can rewrite $\hat{n}_i$ in Equation 5 in terms of the quantum noise amplitudes $\hat{q}_m$ entering at $N_p$ open ports:

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$$S_{k,ij}(\Omega) = \sum_{m=1}^{N_p} \left[ |M_{m,i}^{f*}(\Omega)M_{m,j}^{f}(\Omega) + M_{m,j}^{f*}(\Omega)M_{m,i}^{f}(\Omega)|^2 \right]$$

This reduces to the following form for noise entering the same port in which it exits:

$$S_{i,i} = \frac{1}{2} \left[ |M_{i,i}^{f}(\Omega)|^2 + |M_{i,i}^{f*}(\Omega)|^2 \right]$$

Assuming a filter $\alpha(\Omega)$ combines the BHD ($i = 1$) and PDH ($i = 2$) fields, its output for $L_k$ would be:

$$\hat{n}_{k,\text{combined}}(\Omega) = \alpha(\Omega) \hat{n}_{k,1}(\Omega) + (1 - \alpha(\Omega)) \hat{n}_{k,2}(\Omega)$$

The corresponding total noise power spectral density of the combined readout is then:

$$S_{\text{readout}} = |\alpha|^2 S_{n_1,n_1} + |1 - \alpha|^2 S_{n_2,n_2}$$

where $S_{n_1,n_1}$ is the noise power spectral density at the BHD port due to vacuum entering at the BHD port, $S_{n_2,n_2}$ is the noise power spectral density at the PDH port due to vacuum entering at the PDH port, and $S_{n_1,n_2}$ and $S_{n_2,n_1}$ are the noise power spectral densities for noise entering at one port and exiting at the other. The optimal filter $\alpha_{\text{opt}}$ can be determined by minimising Equation 11 over $\alpha$:

$$\alpha_{\text{opt}} = \frac{S_{n_1,n_2} - S_{n_1,n_2}^*}{S_{n_1,n_1} + S_{n_2,n_2} - \Re[S_{n_1,n_2} - S_{n_2,n_1}]}$$

The reflectivity of $M_1$ is implicit in both the field-to-field and mode-to-field transfer matrices for each signal sideband, $M^f$ and $R$, respectively, and as such $\alpha_{\text{opt}}$ depends on the value of $M_1$.

The matrices $M^f$ and $R$ are not calculated in Optickle by default, and so some modifications to the code were necessary (see Appendix). The effect of $M_1$’s reflectivity on $\alpha_{\text{opt}}$ is shown in Figure 3. Note that, because it is calculated with precomputed spectral densities and not tested for stability, the filter predicted by Equation 12 is not necessarily realisable. A causal Wiener filter has previously been calculated for single-readout interferometers [43] [44], but a similar calculation for more than
one readout has not yet been investigated. While Equation I2 enables the lowest noise spectral density for the measurement of the motion of the differential mode of the Sagnac speed meter, in the case of the proof-of-concept experiment simply combining the BHD and PDH signals with dc gain, as suggested in Section III A, is close to optimal. The difference in response gradients above the cavity pole frequency prevents the PDH signal from contaminating the QND effect in the intended measurement band. The calculation presented in this section, however, is a general solution for any system with multiple readouts for a single variable and may prove useful for future gravitational wave detectors utilising QND techniques.

IV. NOISE BUDGET

In order to show that quantum noise is reduced with respect to an equivalent Michelson interferometer, the design of the proof-of-concept Sagnac speed meter intends for it to be the limiting noise source in a frequency band in the region of a few hundreds of Hz 28. Using the linear projection technique, each anticipated significant source of noise has been estimated and projected into differential mode displacement-equivalent noise to discover the limiting sources across the control bandwidth, and verify that the experiment will be limited by quantum noise in the intended band. The projection requires that the characteristics of the length sensing and control loop be well defined. A detailed description of the interferometer’s optomechanics, signal extraction, control law and suspension and actuator dynamics is provided in Appendix C, while the computed noise budget for the differential mode is presented in Figure 6.

The sensitivity between 100 Hz and 700 Hz, shaded in blue, is the quantum noise limited measurement band. This band is constrained from below by test mass suspension mechanical mode cross-couplings (not shown) and from above by the first violin mode of the ETM suspensions. Suspension thermal noise is the second highest noise source present in this band and is at most a factor of 2.3 below quantum noise, allowing a careful direct measurement of quantum radiation pressure noise to be made in this region. The contribution to the quantum noise from the PDH feedback is far below the limiting quantum noise, showing that the use of the displacement readout as part of the combined filter presented in Section III A does not significantly affect the sensitivity of the Sagnac speed meter in the desired band.

V. SUMMARY

We have demonstrated that positional drifts of the cavity mirrors in the proof-of-concept Sagnac speed meter at low frequencies due to sensing noise lead to an inability to control the cavity mirrors over time scales longer than a few hours. We have shown that this drift can be suppressed by taking a small amount of light from the path between the arm cavities to provide a displace-
ment readout, and that this does not significantly affect the sensitivity of the main, velocity readout. A combination of the displacement and velocity readouts provides a suitable error signal for the control of the arm cavity differential mode at all relevant frequencies without spoiling the quantum non-demolition effect at higher frequencies, facilitating measurements with arbitrary integration time and allowing the Sagnac speed meter to reach its design sensitivity.

Since the main readout of any interferometer primarily sensitive to velocity will encounter the problem of vanishing signal in the presence of flat or increasing sensing noise at low frequencies, we believe the solution presented in this work is applicable to any audio-band speed-meter.

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Appendix A: Required control

The noise present within the interferometer will produce an unintended dark-fringe offset at the output port of the Sagnac speed meter. Using the parameters listed in Appendix D with the relation linking laser frequency fluctuations $\Delta f$ and cavity length fluctuations $\Delta L_{(\text{cavity})}$,

$$
\frac{\Delta L_{(\text{cavity})}}{L_{\text{RT}}} = \frac{\Delta f}{f_0},
$$

(A1)

with $f_0 = \frac{c}{2L_0}$ representing the laser frequency, the requirement for the Sagnac speed meter is that the residual motion of the mirrors must be less than $3.5 \times 10^{-13}\text{m}$. This uses the assumption that the frequency fluctuations $\Delta f$ fall within 1% of the arm cavity full-width half-maxima—a common heuristic method used to ensure that technical noise sources do not couple strongly to the gravitational wave channel.

As shown in ref. [32], main beam splitter asymmetries introduce common arm cavity mode coupling at the output port, which leads to further unintended dark fringe offset, and so the real requirement is likely to be more stringent. The above requirement, however, is sufficient to highlight the problem of velocity-only control as shown in Section II.

Appendix B: Calculation of field transfer matrix

By default, Optickle will only output the signal and noise on probes defined within the system, where a probe is analogous to a photodetector with unity quantum efficiency. A probe signal is a superposition of the field amplitudes in a given location within the interferometer, where the exact linear combination of field amplitudes is determined by the type of probe. In the process of determining a probe signal, the quadrature sum of the field amplitudes immediately in front of the probe is computed. The phase information contained within these fields is lost in this process. Similarly, transfer functions from drives (test mass modes within the interferometer) to probes are provided, but not transfer functions from drives to fields.

In order to calculate the cross-correlation spectral density required for the calculation of the optimal filter in Section III B, the complex field and drive transfer matrices, $\mathbf{M}^f$ and $\mathbf{R}$, respectively, must be extracted from Optickle indirectly. Optickle’s calculation of the quantum noise at each probe within the interferometer uses field-to-field and drive-to-field matrices, but because the quantum noise and drive excitations are not necessarily unity, these matrices are not transfer matrices. In order to obtain $\mathbf{M}^f$ the code which computes the quantum noise at each probe has to be modified to instead inject quantum noise at open ports with unity amplitude. Similarly, $\mathbf{R}$ can be computed by setting the drive amplitudes to unity. The modified source code is publicly available [15].

Appendix C: Control scheme

The intended control loop schematic for the proof-of-concept experiment is shown in Figure 7. The “fiducial” section is well defined and is based on existing DAQ hardware and software, an implementation of the LIGO control and data system [10]. The bottom section contains the blocks which have been modelled during the course of this work.

All blocks shown are frequency dependent matrices with one or more inputs and outputs. The green blocks represent simple filters possessing single inputs and outputs. Each filter multiplies its input by a transfer function in order to calculate its output. The blue blocks represent transfer matrices, which are multiple-input and multiple-output filters. The grey triangles represent dc gain, and are equivalent to simple filters without frequency-dependent response. The undulating orange arrows represent noise injection.

1. Sensing

The outputs from the interferometer block are probe signals modelled by Optickle for a given set of test mass
FIG. 7. Sagnac speed meter control loop model. The interferometer plant produces signals representing the probes in the interferometer, and sensing noise is added before the signals are sent to the digital controller. Within the controller, the interferometer readouts are combined into an error signal representing \( L_\text{e} \). The error signal is fed through a series of filters and sent to the test mass actuators, with the addition of DAC noise. The suspension blocks transform the feedback signals into test mass displacements, and seismic, coating brownian and suspension thermal noise is injected at the input to the interferometer plant.

Displacement inputs. Quantum vacuum noise, also modelled by Optickle, is injected onto these signals. The BHD signal is calculated by taking the difference between the two homodyne probe signals, BHD A and BHD B. Each signal is passed through a quantum efficiency gain stage to represent the conversion of light power into photocurrent by the photodetector, and then through a transimpedance gain stage to convert the photocurrent into a voltage. The PDH signals receive additional gain due to RF demodulation. At this point, the signals are sent to the DAQ. Whitening filters ensure the signals are far above the noise of the ADCs. The noise associated with the conversion process is injected at this point. To prevent aliasing due to the ADC sample frequency, 65,536 Hz, filters are employed to suppress signal content above around 10 kHz. The conversion from volts to digital counts within the ADCs is represented by a gain block for each channel.

2. Control

In the digital domain, dewhitenning filters reverse the effect of the whitening filters to recover the original signals. At this point, the arm cavity displacement signal can be constructed by taking the difference between the individual PDH signals. This signal is then combined with the BHD’s velocity signal via readout gain stages (as presented in Section 3A) which set the crossover frequency between velocity and displacement measurements. If an optimal filter were to be employed as shown in Section 3B these gain blocks would be filters.

The unity gain frequency of the feedback system is set by the loop gain block. A filter is also present here to compensate the feedback signals for the shape of the interferometer’s frequency response. The output from this filter is split into two signals to be applied differentially to ETMs \( M_{\text{ETM}_1}^A \) and \( M_{\text{ETM}_1}^B \), as shown in Figure 1(a). Actuator range can be further extended by splitting this feedback between the other mirrors in each cavity, but this is not modelled in this work. The signal applied to the \( M_{\text{ETM}_1}^B \) suspension system is inverted in order to move the test masses differentially.

3. Driving

Each test mass’s correction signal is passed through a set of driving filters representing the dynamics of each actuator. The ETMs are suspended from triple pendulums to provide sufficient isolation from seismic noise within the measurement band. In order to provide corrective actuation upon the test masses, voice coil actuators are present on the suspension stage immediately above each test mass in addition to (unsuspended) ESDs situated behind each test mass. The ESDs are sufficiently insensitive...
FIG. 8. Simulated proof-of-principle Sagnac speed meter ETM suspension open loop transfer functions showing the difference in gain between the two test mass actuators. The voice coils provide extensive actuation range but are suppressed at high frequencies by the final stage pendulum. The ESD actuates directly upon the test mass and is therefore capable of providing stronger correction than the voice coils at higher frequencies. Second order low-pass notch filters are present on both actuators at 800 Hz to prevent excitation of the first suspension violin mode.

to seismic motion transverse to the test mass that they need not be suspended [33]. The ESDs’ ability to actuate directly upon the test masses makes them suitable for high frequency corrections whereas the extensive range of the voice coils makes them suitable for the suppression of the dominating seismic noise below the measurement band.

Each correction signal is passed through a gain hierarchy to split the feedback between the voice coil and ESD actuators. The open loop transfer functions for each actuator are shown in Figure 8. The crossover frequency between the two actuators is 18 Hz.

Before being sent to the actuators, the driving filter outputs are converted into the analogue domain using appropriate DAC gain and noise injection stages. The conversion from voltage to force is represented by gain stages at the input to each suspension block.

The suspension block contains three inputs, each representing a different passive filtering stage of the suspension system. These input signals are mapped to a single output representing the displacement of each ETM, and these signals are fed back to the interferometer block, completing the loop.

4. Loop Gain

The open loop gain of the control system is shown in Figure 9. The greatest gain is required in the region below 10 Hz where the test mass suspensions provide little to no isolation from seismic noise. Above this frequency the required gain decreases as other noise sources become dominant. The unity gain frequency is at 350 Hz.

Appendix D: Experimental parameters

The parameters used in the simulations presented in this work are shown in Table I. Unless otherwise stated, the mirrors specified in the figures and simulations are assumed to have unity reflectivity. All listed transmis-sivities represent power, no substrate loss is assumed for any optic and all simulations have been performed using the plane-wave approximation.

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TABLE I. Experimental parameters. The properties for the suspensions and test masses are given in order for the reader to be able to reproduce the suspension Thermal noise spectral density presented in Figure 6.

| Parameter                                      | Fiducial value |
|------------------------------------------------|----------------|
| Laser wavelength $\lambda_0$                   | 1064 nm        |
| Input power                                    | 1.8 W          |
| Round-trip cavity length $L_{RT}$              | 2.83 m         |
| $\mathrm{M}_i$ transmissivity                  | $10^4$ ppm     |
| ITM transmissivity                             | 700 ppm        |
| Arm cavity full-width half-maximum             | 12.2 kHz       |
| Arm cavity finessie                           | 8663           |
| BHD quantum efficiency                         | 0.95 $\text{AW}^{-1}$ |
| PDH quantum efficiency                         | 0.80 $\text{AW}^{-1}$ |
| $R_T$                                          | 10 $\text{k}\Omega$ |
| PDH demodulation gain                          | 21 dB          |
| ADC/DAC quantisation noise                     | 1.8 $\text{VP} \sqrt{\text{Hz}}^{-1}$ |
| ETM mass                                       | 113 g          |
| ETM fibres                                     | 4              |
| ETM fibre diameter                             | 40 $\mu$m      |
| ETM fibre length                               | 200 mm         |
| ITM mass                                       | 0.86 g         |
| ITM fibres                                     | 2              |
| ITM fibre diameter                             | 10 $\mu$m      |
| ITM fibre length                               | 100 mm         |
| Suspension vertical-to-horizontal coupling     | 0.01           |

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