A Review of NDT/Structural Health Monitoring Techniques for Hot Gas Components in Gas Turbines

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Abstract: The need for non-destructive testing/structural health monitoring (SHM) is becoming increasingly important for gas turbine manufacturers. Incipient cracks have to be detected before catastrophic events occur. With respect to condition-based maintenance, the complex and expensive parts should be used as long as their performance or integrity is not compromised. In this study, the main failure modes of turbines are reported. In particular, we focus on the turbine blades, turbine vanes and the transition ducts of the combustion chambers. The existing monitoring techniques for these components, with their own particular advantages and disadvantages, are summarised in this review. In addition to the vibrational approach, tip timing technology is the most used technique for blade monitoring. Several sensor types are appropriate for the extreme conditions in a gas turbine, but besides tip timing, other technologies are also very promising for future NDT/SHM applications. For static parts, like turbine vanes and the transition ducts of the combustion chambers, different monitoring possibilities are identified and discussed.

Keywords: gas turbines; SHM; non-destructive testing (NDT); sensing

1. Introduction

The components of a gas turbine, in direct contact with hot gas, are subject to high loads. These include turbine blades that have to withstand extreme conditions. Monitoring the health of these components is important for two reasons. First, it is important to detect incipient cracks before catastrophic events occur, and second, it is the basis for condition-based maintenance.

Structural health monitoring (SHM) is a further development of non-destructive testing and evaluation (NDT&E). Years ago, it seemed to be sufficient to test components before use or again in a disassembled machine. The vision is moving in the direction of realizing component damage detection in real time. Once the damage occurs, a warning is issued and information about the remaining life of the affected component is displayed. This leads to increased operating safety of the machine and at the same time reduces maintenance costs and time. Therefore, an optimal utilisation of these parts from the customer point of view is highly desirable.

An exemplary SHM system consists of many components. Transducers are needed for actuation and sensing. A signal generator supplies the actuator with diagnostic input while the sensor carries the measured data to the data acquisition system. A NDT/SHM system requires appropriate transducers, its signal processing system to reduce noise, consider environmental conditions and write down the damage signature. In order to implement this practically, knowledge of typical damage experienced by the structures are required. This can be a physical model of the structure, a statistical model or machine learning schemes [1].

Proulx published a book on SHM of rotating machines focused on vibration methods [2]. In [3–5] new sensor technologies and trends in SHM are reported.
This review summarises the different failure modes, which are relevant for NDT/SHM in industrial gas turbines and currently used monitoring technologies. In particular, critical defects are discussed and how to determine the remaining useful life of the turbine components. For the component load it makes a big difference whether the machine is started up once and stays in this state for a long time or whether permanent load changes take place. Depending on these operating modes, the life limiting factors for the highly loaded turbine components are shown.

Furthermore it is indicated, which consequences small cracks on turbine blades can have. The subsequent turbine stages up to the entire machine can be severely damaged. A challenge is certainly the massive, forged turbine discs where the blades are mounted. For example, at overspeed, it must be ensured that the centrifugal force always bursts the blades first before the discs do. Only in this way can it be ensured that no rotor components break through the outer casing of the gas turbine.

We also review the different monitoring techniques for gas turbine blades, vanes and combustion chamber transition ducts (Figure 1). For the static parts (vanes and transition ducts), the same monitoring techniques could be used.

![Figure 1. Gas turbine–Combustion chamber transition duct, turbine vanes and blades.](image)

Strain gauges were used to obtain information about the components during machine operation. However, these have some significant disadvantages. With the tip timing method, sensors are placed opposite the rotating blades in the casing. These can usually determine both the important gap between blade and casing and the dynamic behaviour of the blades in-situ. To implement this technically, there are a number of sensor technologies, which are presented in more detail. This principle is already well established in turbomachinery, but there are some promising alternatives that are based on very different concepts. These monitoring techniques can set the course for a shift away from scheduled-based maintenance.

The physics of the different techniques are described in this review. Also the accuracy, the sensitivity and the bandwidth of the sensors are explained. Due to the limited installation space in the turbine area, the sizes of the sensors are also important and are evaluated. A particular challenge is the high ambient temperature, in which the sensors must work. The limits are shown below. Qualitatively the sensor costs are also evaluated. The hot gas components, in particular, have complex designs and are expensive. The great competition and price pressure in this business segment makes it necessary to offer cost-optimized series machines and signal processing units. Relevant research examples with the general areas of application are shown.

2. Failure Modes

Continuous duty machines are typically applied in power generation. Therefore, an electric generator will be driven at a relatively constant speed.
Cyclic duty means mechanical drive applications where a gas turbine drives another turbomachine (e.g., a compressor). The speed of the gas turbine is adjusted to requirements of the driven unit. Due to this different operating behaviour, different failure modes can occur. The typical failure modes for hot gas parts in a gas turbine in continuous duty and cyclic duty machines are listed in Table 1 [6–8].

| Continuous Duty | Cyclic Duty |
|-----------------|------------|
| Rupture         | TMF        |
| Creep deflection| High-cycle fatigue |
| High-cycle fatigue | Rubbing/wear |
| Oxidation       | Foreign object damage |
| Erosion         | Combined failure mechanism (creep/fatigue, corrosion/fatigue, oxidation/erosion and so on) |
| Corrosion       | Corrosion |
| Rubbing/wear    | Rubbing/wear |
| Foreign object damage | Foreign object damage |
| Combined failure mechanism (creep/fatigue, corrosion/fatigue, oxidation/erosion and so on) | Combined failure mechanism (creep/fatigue, corrosion/fatigue, oxidation/erosion and so on) |

For peaking machines, thermal mechanical fatigue (TMF) is the main life limiting failure mode. For continuous duty machines, creep, oxidation and corrosion are the main life limiters [7].

Low-cycle and high-cycle fatigue are primary influenced by design issues, while foreign object damage and corrosion are load and environment dependent [9].

One typical failure mode for the transition ducts of combustion chambers is wear at the mating surfaces. High temperature gradients, especially at transient operating conditions, allow the parts to rub against each other. Another failure mode is the degradation of the thermal barrier coating (TBC) in the combustion chambers [7]. A missing or degraded coating will lead to overheating of the combustion chamber components. In Figure 2 the typical construction of a single-shaft gas turbine is shown. The air is led via the inlet housing in the compressor section and compressed. In the combustion chambers, the air is heated to 1200 °C. In the turbine section, the hot gas expands and leads to highly loaded hot gas components. Part of the energy is needed to drive the compressor; the other part is fed into the electric generator for power generation.

Figure 2. Typical gas turbine construction.
The overheated turbine vanes are shown in Figures 3 and 4. In Figure 5, an incipient tear in a turbine blade is illustrated, while in Figure 6, cracked turbine blades are displayed.

Figure 3. Overheated turbine vane.

Figure 4. Overheated turbine vane inside machine.

Figure 5. Tear in turbine blade.

Figure 6. Cracked turbine blades.

Examples of failures in turbine blades in military jet engines with the failures analysis are reported in [10–12].
3. Hot Gas Component Monitoring

Historically, a common principle to detect vibration and frequencies in rotating blades is with the use of strain gauges. The application of strain gauges is very complex and not every blade can be equipped with these sensors. Furthermore, they are not suitable for long-term operation. Nowadays, several technical alternatives exist with different advantages and disadvantages.

In [13] a historical summary of the use of strain gauges is reported. Kestner et al. studied the correlations between different installed sensor types on blade vibrations. The authors used acoustic pressure, bearing vibration, tip timing and gas path measurement from an operating machine [14]. Guo et al. developed a new method to identify engine order, amplitude, natural frequency and damping coefficients of the blades using three casing mounted sensors [15].

3.1. Tip Timing Method

The tip timing method allows for the possibility of measuring, in-situ, the dynamic behaviour of a turbine blade. The radial clearance between the sensor and blade tip can be measured and also the lateral movement of the blade tip. Additional information like time of arrival can be evaluated (Equation (1)), which gives information about blade vibrations:

\[ d_i = \frac{2\pi r}{T} \Delta t_i \]  \hspace{1cm} (1)

where \( d_i \) is the deflection of blade \( i \), \( r \) is the turbine radius, \( T \) is the period for one revolution and \( \Delta t_i \) is the difference between theoretical and practical time of arrival for blade \( i \).

In Figure 7 the design of a typical tip timing system is outlined.

![Figure 7. Design of a tip timing system.](image)

One general problem is that the circumference clearances could differ because of the ovalisation of the casing [16].

A significant challenge is the small changes in position of the blades during an engine starting and stopping. This is because a small clearance exists between the fir-tree of the turbine blade and the turbine disc. Therefore, the mechanical repeatability is not high [16]. The blade tips of gas turbine blades, often have a complex geometry, which could be problematic for failure detection with some tip timing sensors [17,18]. Figure 8 shows the tip geometry of a typical turbine blade.

![Figure 8. Turbine blade.](image)
With the tip timing monitoring technologies, the following failure modes could be detected:

- Crack detection
- Creep deflection
- Rubbing/wear

Witos et al. showed a theoretical approach for health monitoring of compressor blades. With the tip timing method they tried to answer why a blade cracked and with the metal magnetic memory method the localisation of the crack position was determined [19,20]. Rokicki et al. analysed bearing failures of a jet engine with tip timing. Experiments with damaged and working bearings were compared [21]. Tamura et al. focused on non-contact vibration measurement in gas turbines and different sensor technologies were used [22]. Dimitriadis et al. developed a mathematical model to simulate blade tip timing tests [23]. Gallego-Garrido et al. compared different analysis methods for tip timing data [24,25]. The uncertainties of blade tip timing were studied by Russhard [26] and Satish et al. [27].

3.1.1. Microwave Probes

Microwaves can be used to measure tip clearance. Microwaves are electromagnetic waves that propagate in the GHz range. A typical microwave tip clearance probe includes the transmittance and a receiving antenna. The microwave sensor sends a continuous microwave signal to the target and measures the reflected signal. This signal is then compared with an existing reference signal and the phase difference with the distance to the blade. The phase differences of the reflected signal are directly proportional to the distance between the sensor and the target [28]. The whole system is shown in Figure 9.

A typical transmitted signal is:

\[ X_0 = A_s \cos(\omega_s t + \varphi_0) \]  

(2)

The received signal is:

\[ Y = A(t) \cos(\omega_s t + \varphi_0 + \varphi_l + \varphi(t)) + A_r \cos(\omega_s t + \varphi_l) \]  

(3)

where \( \varphi_l \) is the summed phase in the transmission path and \( \varphi(t) \) is the phase difference caused by the change in the tip clearance. The term \( A_r \cos(\omega_s t + \varphi_l) \) is the reflected signal from the sensor itself. This should be removed later in the signal processing [29]. Temperature factors of phase \( \varphi_l \) can be obtained by setting the frequency outside the sensor bandwidth. Thus the tip clearance can be determined from \( \varphi(t) \) after the calibration.

![Figure 9. Microwave tip timing system.](image_url)
The blade damage will be detected by measuring dimensional changes reflected in the blade tip clearance [16]. These sensors have a large bandwidth and are able to operate at high temperatures [29,30]. Practically, it was shown that a temperature of 900 °C is feasible [31,32]. Furthermore, the temperature influence was tested, and high temperature measurements are possible [30]. One challenge is the change of phase with temperature, which can be correlated [16]. A validation with optic tip timing systems showed that microwave tip timing systems achieve good accuracy [16,29]. Some failure can develop over a long period of time [29], so accurate sensors are essential.

Repeatability errors are caused by [16]:

- Drifts in electronic component
- Sensor material (change due to temperature effect)
- Contamination of the probe

Compared to eddy current (EC) probes, microwave sensors have a superior resolution and measurement range [30]. Typically the sensors have an outer diameter between 8.5 and 10 mm [17,31]. Many types of damage show just a small change in tip clearance, for example, blade root cracking is around 0.025–0.1 mm (small crack). A sensor for this application needs a sensitivity of at least 0.05 mm, and test conducted showed sensor sensitivities of 0.025 mm [16]. Violetti et al. showed that these sensors are suitable for long-term use [30].

Microwave sensors are a technology that requires further investigation. In particular, the calibration techniques are not fully developed [33]. The distortion of asymmetric turbine casing can be caught with more probes. Typical 4–8 sensors are used [16].

The first application in a gas turbine was reported by Wagner et al. in 1998. For pilot tests a 65 MW gas turbine were used [34]. Zhang et al. proved experimentally the general feasibility of blade tip timing with microwave sensors [29]. Woike et al. especially focused on the turbine blades. 5.8 GHz and 24 GHz sensors were used, to measure successfully the blade tip clearance [33]. Hafner et al. investigated the possibility for online tip clearance measurement with microwave sensors and the immediate visualisation of the results [35]. In field machines no applications of microwave sensors were found. For SHM, this monitoring technology is very promising.

3.1.2. Optical Sensors

A laser Doppler vibrometer (LDV) is a measuring device to determine the vibration frequency and vibration amplitude where, the laser is focused on the surface to be measured. Due to the doppler effect, the frequency of the backscattered laser light shifts as the surface to be measured moves.

If the wavelength of the light is superimposed on that of the vibrating object, the following result is obtained for a 180° scattering angle:

\[ \Delta f_D = f_L - f_O \]  \hspace{1cm} (4)

\[ \Delta f_D = \frac{2V}{\lambda} \]  \hspace{1cm} (5)

where \( \Delta f_D \) is the resulting frequency, \( f_L \) is the frequency of the light, \( f_O \) is the frequency and \( V \) is the velocity of the vibrating object.

In Figure 10, the basic structure of an LDV system is outlined. The reference beam does not leave the LDV. It is directed via a bragg cell onto the photodetector, where it interferes with the reflected measuring beam. The bragg cell is an acousto-optic modulator and shifts the frequency of the reference beam. The result is a frequency-modulated voltage, which is directly proportional to the speed of the measured object. Using fast Fourier transform (FFT), the measurement results can be further analysed.

Laser Doppler vibrometers (LDVs) are non-contact vibration measurement instruments. They can also be used for tip timing.
A scanning LDV (SLDV) allows for scanning of whole surfaces. This technology has high sensitivity, even when the sensor is not placed directly on the blade [36]. Optical sensors have good time accuracy and are small sized with a large bandwidth [29,37]. They have an excellent lateral and spatial resolution, a fast response and are often used because of their reliability and repeatability [38]. SLDVs are a good solution to detecting the dynamic shapes of a structure [39].

Recent studies have shown that optical fibre probes are suitable for development testing [40]. However, outside the laboratory, debris and low tolerances make their application difficult [17,38,41]. The reported shading problems are disadvantageous [42], as are the optical waveguides with limited operating temperatures and the required inspection window to the blade tip that may be polluted within a short period of time. Therefore, optical sensors are not optimal for long-term instrumentation [37]. To counteract these disadvantages, the use of purge air for cooling was proposed. However, this means additional systems with additional weight. Optical techniques are dominant in the available literature [41]. To monitor the dynamic behaviour of blades in development machines, this is an important technical solution, but for long-term use, it is considered challenging for SHM applications.

Reinhardt et al. made experiments to determine vibration frequencies and amplitudes of turbine blades with a laser vibrometer [43]. Lezhin et al. compared different vibration measurement systems with numerical calculations [44]. The investigation of bladed disks with optical laser probes were publicised by several authors [45–49]. In-situ tip clearance measurements with laser doppler method were presented by Büttner et al. [50,51]. Gil-García et al. [52] and Zielinski et al. [53] focused their work on time of arrival measurements on compressor and turbine blades, to determine blade vibration. Overton developed an LDV system to measure tip clearance more accurately, compared to the capacitive method [54]. Sharma et al. [55] and García et al. [56] used laser vibrometry for SHM aspects. To measure the whole rotor deformation during rotation, Günther et al. made experiments with optical sensors [57]. Oberholster et al. explored a new approach to detect blade damages with Eulerian LDV [58]. Pfister et al. investigated the application to measure simultaneously position and velocity on moving rough surfaces with only one sensor [59].

### 3.1.3. Inductive Sensors

Inductive sensors consist of multiple mini-sized planar spiral coils. The sensor measures the tip clearance by measuring the inductance change of planar spiral coils by the passage of the rotor blades. The smaller the tip clearance, the higher the inductance drops, due to the larger EC induced in the blade tip [17]. As the turbine blades passes the magnetic field of the sensor, an eddy current flows...
because of the electromagnetic induction. This increases the induction current flow, whereby the load
of the oscillation circuit increases and the vibration is damped or stopped. The sensor measures this
change (Figure 11).

According to Faraday’s law, Equation (6) is given:

\[
U_i = \oint_c \left( \overrightarrow{v} \times \overrightarrow{B} \right) \cdot d\overrightarrow{l}
\]

where \(U_i\) is the magnitude of the induced voltage, \(B\) the course of the magnetic induction, \(l\) the function
of coil length and \(v\) the resulting speed.

![Principle inductive sensor.](Figure 11)

These sensors typically operate at temperatures below 60 °C. Higher temperatures decrease
the magnetisation of the permanent magnet and increase the impedance of the winding. They are
characterised by long life, high reliability and resistance to contamination [40]. Another advantage is
their simple construction. Clearances are tested from 0 to 5 mm with a resolution of 10 \(\mu\)m at a rotation
speed of up to 80,000 rpm. Inductive sensors are of simple design, low in cost and easy to install. They
have high resolution, high sensitivity and the capability of monitoring a large number of tip clearances
simultaneously. These sensors do not need a penetrating hole through the casing. Therefore, the device
is more sensitive to the relative vibration between the casing and the sensor. However, this only works
if the casing does not contain ferrous material, because the magnetic field, and thus the output signal
amplitude is reduced significantly. The sensors cannot detect variation in tip clearances of less than
50 \(\mu\)m. The main disadvantages are the calibration effort and the limited lateral resolution. These
sensors can only detect one tip clearance at a specific location at the blade tip. Therefore, for advanced
health monitoring techniques, multiple sensors are needed. The low temperature resistance of the
sensors does not allow to be used in the turbine part of a gas turbine. However, for the first compressor
stages of a gas turbine, their use is still conceivable.

Przysowa et al. developed a new inductive sensor for blade health monitoring systems for military
turbofans. Also the resistance to high temperature and contamination were verified [40,60]. Du et al.
developed multiplexed inductive sensors for detection of blade tip clearances [17].

3.1.4. Eddy Current Sensors

EC sensors emit a high frequency electromagnetic field. The passing blade in this field induces
ECs in the blade tip, acting against the existing high frequency field. The following change in coil
impedance can be measured [61].

The general expression is shown in Equation (7):

\[
V(t) = n \frac{d\Phi}{dt}
\]

\[\text{V}(\text{t}) = \int \left( \overrightarrow{v} \times \overrightarrow{B} \right) \cdot d\overrightarrow{l}
\]

\[\overrightarrow{U}_i = \overrightarrow{r} \left( \overrightarrow{v} \times \overrightarrow{B} \right)
\]

\[\Phi = \int_{c} \overrightarrow{v} \times \overrightarrow{B} \cdot d\overrightarrow{l}
\]
where \( V \) is the sensor voltage, \( \Phi \) the magnetic flux passing through the coil and \( n \) is the number of turns on the coil.

This principle is shown in Figure 12. The determination of the change of this phase position is the main difference compared to the inductive sensors.

![Principle of an EC sensor](image)

**Figure 12.** Principle of an EC sensor.

Parts of the sensor start to melt at 93 °C [62]. It was reported that some companies have developed EC sensors that are able to operate at high temperatures up to 1000 °C if an air-cooling system is used [63]. EC sensors have good accuracy [29] and are typically embedded in the casing and the arrival time of the blades will be measured. From this basis, blade deformation can be calculated. They do not work well with non-conductive materials (composites). Furthermore, it is difficult to measure high vibration frequencies without a priori knowledge [64].

Cardwell et al. developed a new EC sensor for blade tip timing of engine fans [65]. Lui et al. proposed a new method to improve the measuring accuracy of EC sensors by considering torsional vibration [66]. Ghana et al. showed new tip timing algorithm for EC sensors which was verified in laboratory and real engines [67]. Tsutomu et al. published their work on the usage of EC sensors for displacement measurements successfully [68]. Przysowa et al. studied health monitoring techniques with EC sensors for military aircrafts [69].

### 3.1.5. Magnetoresistive Sensors

Magnetoresistivity is the effect that describes the change in the electrical resistance of a material by applying an external magnetic field. These include, in particular, the anisotropic magnetoresistive effect (AMR effect), the giant magnetoresistance effect (GMR effect), the colossal magnetoresistance effect (CMR effect), the tunnel magnetoresistance effect (TMR effect) and the planar Hall effect. The use depends on whether the component to be examined is magnetic.

To describe the strength of the respective magnetoresistive effect, the quotient of resistance change and resistance without external field is used:

\[
\frac{\Delta R}{R[\%]} = \frac{R(H) - R(0)}{R(0)} \cdot 100
\]

(8)

where \( R(H) \) is the resistance in dependence of the magnetic field, \( R(0) \) is the resistance without external magnetic field, \( \frac{\Delta R}{R} \) is the characteristic of the magnetoresistive effect.

When the blade passes the sensor, the magnetic field will be distorted. This variation can be measured (Figure 13).

The main advantage of this technology is the relatively small sensor size. Typically, the sensor has a maximum outer diameter of 8 mm. In addition, the simple design lowers the sensor costs [38,42]. They have good tolerances to debris, which is an important for the accuracy and long-term operation. The fast rise time (~20 ns) is also advantageous and their high signal repeatability and time accuracy is comparable to the optical systems and the clearance measurements are in line with capacitive sensors [42].
A new development shows that these sensors have the potential to survive up to 700–800 °C. However, the robustness, durability and accuracy has to be tested [63].

Procházka et al. made fundamental research on this sensor technology in turbomachinery. An online system was developed to monitor the vibrational amplitudes and frequencies of all blades. Possible blade damages could be notified and also blade elongation and blade twisting was shown [70,71]. Brouckaert et al. developed a new magnetoresistive sensor for non-contact blade vibration measurements [72]. Tomassini et al. presented a new sensor design as well [73,74]. The new sensor was successful verified in test benches and in a jet-engine.

3.1.6. Capacitance Sensors

The electrical capacitance, between two electrically conductive materials insulated from one another, is equal to the ratio of the charge quantity \( Q \) and the electrical voltage \( U \) (\( C = \frac{Q}{U} \)). The change in electrical capacitance can be used inter alia to determine distances. In our case capacitance sensors measure the capacitance change between the probe and the blade tip. When the capacitor plate generates an electrostatic field and a turbine blade is present, the capacitance changes so that the oscillator begins to oscillate (Figure 14). Equation (9) shows the relationship of capacitance, sensor geometry and rotor to stator gap:

\[
C_x = \frac{\varepsilon_r \varepsilon_0 S}{d}
\]

where \( C_x \) is the capacitance from sensor to blade, \( \varepsilon_r \) and \( \varepsilon_0 \) are the permittivity of the medium and the vacuum, \( S \) is the area between two sensor plates and \( d \) is the distance from blade to sensor.

![Figure 13. Function of a magnetoresistive sensor.](image1)

![Figure 14. Principle capacitance sensor.](image2)
The use of plastics limits the operating temperature of some sensors to 200 °C [75]. Capacitive sensors are predominant due to the improved temperature robustness [37] compared to the other sensor technologies. However, in a gas turbine, the thermal load of the sensors is much higher, therefore, cooling is required. The sensor diameter is relatively small [75] allowing for easier implementation in the engine.

The main disadvantages are the calibration effort and the limited lateral resolution. Some sensors have limitations in bandwidth [29]. The electronic circuits of these systems have to be placed only a few centimetres away from the probe head. The distance from the probe head to the first amplifier has to be at a maximum of 1 m. The output of the probe is nonlinear. For a high precision, the blade tip geometry must be considered. Therefore a "calibration wheel" can be used. With increasing distance between the probe head and blade tip, the signal-to-noise ratio (SNR) decreases. Also, a minimal clearance exists. These sensor types have low costs and are of a simple design [76]. However, the measured capacitance often not only gives the correct clearance because the dielectric property of air could change, due to variations in pressure and humidity [17].

Capacitance probes have a greater potential than optical probes because it is difficult to measure tip clearance with them [41].

Sarma et al. [77] and later Drumm et al. [78] designed a dual-amplifier circuit configuration for tip clearance measurements. Mönch et al. [79] and Müller et al. [80] developed a tip clearance system for compressor and turbine blades. For the usage in micro gas turbines, Fabian et al. described the special requirement of a capacitive tip clearance measurements system in this application [81]. Lavagnoli et al. studied the implementation of a high frequency capacitive sensor on a large transonic turbine stage [82]. These works show that these sensors can be used in hot gas sections for industrial gas turbines for health monitoring of turbine blades.

### 3.2. Vibrational Monitoring

Vibration sensors, like accelerometers, displacement sensors and velocity sensors, are the widest used techniques for blade fault diagnosis in field conditions.

In industrial gas turbines eddy current proximity transducers are usually used on the bearings to measure the rotor vibrations. The same sensors are also used in the keyphasor to determine the phase angle and the axial position of the rotor [83]. The functional principle is identical to that described in Section 3.1.4. For measuring the casing vibration mostly accelerometers are used. A piezoceramic sensor plate converts dynamic pressure fluctuations into electrical signals. The pressure fluctuation is generated by a seismic mass attached to the piezoceramic and acts on the piezoceramic when the overall system is accelerated (Figure 15).

![Figure 15. Principle accelerometer.](image-url)
These signatures combined can be taken for health prognosis of the blades. For the frequency analysis, the most common technique is the frequency spectrum analysis technique. This means the conversion of the vibration signals from time domain into the frequency domain [84]. By analysing these frequencies, the location and failure types can be detected. These sensors work with a frequency of 10 kHz [85]. This method is effective in detecting severe blade faults (e.g., terminal rubbing), while minor faults (e.g., impending rubbing) are mostly not detectable. Therefore, vibrational analysis is not a reliable tool for SHM in field engines [86–88].

Südmersen et al. showed a combination of pressure measurements, casing vibration measurements and shaft displacement [89]. Lebold et al. presented a work to demonstrate the feasibility of torsional vibration measurements for shaft crack detection. It was mentioned that the same technique may be used for crack detection in turbine blades [90]. In a case study, it was shown that when blade faults are the only failure in the gas turbine, they will often not readily be detected with conventional vibration measurements. The reason is that these failures do not generate enough excitation compared to the other vibration amplitudes in the machine [91]. Alternatively, it was demonstrated that blade related failures could be detected before catastrophic events occur [84]. Ghouti et al. used the shaft torsional vibration signals to extract blade vibration signatures [92].

Zielinski et al. described the configuration of different measurement systems with two different probe types [53]. Zhang et al. studied the start-up vibration signatures of an industrial gas turbine for condition monitoring [93]. Sinha et al. presented a study to reduce the number of vibrational probes by improving the signal processing [83]. Loutas et al. tried the combination of vibration, acoustic emission and oil debris for condition monitoring [94]. Schlagwein et al. focused on mistuning effects on blades [95]. In [96–100] it was shown, how condition based maintenance with machine-learning approaches were realised. To detect blade damages with vibrational analysis following papers were published [23,92,101–116]. Several books were written with focus on vibration of rotating machines [2,117–120].

Despite some disadvantages of this technology, this is a long-proven and cost-effective monitoring technique in gas turbines. This also plays an important role in the performance monitoring (Section 3.11).

3.3. Ultrasound

Ultrasound is a common non-destructive testing method for detection of crack or material changes [121–123]. Ultrasonic waves usually propagate in a band from 20 kHz to about 1 GHz [124].

The ultrasonic harmonic waves presented can be described with the general wave equation:

$$\psi = A \sin(\lambda x - \omega t + \phi)$$

where \(\phi\) is the initial phase angle, \(k\) the wavenumber with the wavelength \(\lambda = 2\pi/k\), the period \(T = 1/f\) and the frequency \(f = \omega/2\pi\). Typically an ultrasonic wave is sent and the reflected signal from the crack can be used for failure prediction. The most commonly used piezoelectric transducers for SHM in gas turbine applications are limited in operating temperature.

The most relevant guided waves for gas turbine are Rayleigh waves and Lamb waves. Many authors are researching these waves for early crack detection and determination of crack sizes [1,125–130].

Rayleigh waves are the simplest forms. The longitudinal and shear motions are linked and propagate at a common velocity. The terms “surface acoustic waves (SAW)” and “Rayleigh waves” are usually used as equivalents.

In addition to the possibilities presented below, surface waves with angled sensors are generated on the component. The disadvantage is that a coupling fluid is needed. Furthermore it should be noted, however, that higher-frequency waves penetrate less in the material than low-frequency waves. For Rayleigh waves, the penetration depth is about one wavelength [131]. These waves are very
promising for NDT&E. Especially in high-temperature components, cracks will develop near the surface. Therefore Rayleigh waves are a good tool for inspection [132]. They can be used to evaluate the elastic properties of samples and coatings. The phase velocity of the SAW depends on elastic tensor of the material. This means that SAW information can determine the mechanical properties of components. The phase velocity is sensitive to see small changes in material density, Young’s modulus and Poisson’s ratio [133].

In a far-field characterization, the spatial resolution is half a wavelength ($\lambda/2$). Therefore, it is important to use very high frequencies to detect micro-cracks. The disadvantage is that at very high frequencies SNR problems occur. Far-field methods are unsuitable for the detection of micro-cracks. In contrast, in the near-field the spatial resolution is much better [134].

Lamb waves are elastic waves that propagate in solid plates whose particle motion lies in the plane that contains the direction of wave propagation and the plate normal (the direction perpendicular to the plate). Basically deflections occur both in the propagation direction (longitudinal) and vertically (transversal). Lamb waves are dispersive in nature. The speed of propagation $c$ depends on the frequency (or wavelength), and elastic constants and density of the material. Therefore, Lamb waves are mixed pressure and shear waves [124]. To determine the relationship between wave velocity and frequency of a sample, dispersion curves can be used. Lamb waves for damage detection are important because they can propagate over long distances and they can show complex phenomena [123,125].

Lamb waves consist of the fundamental symmetric mode $S_0$ and the fundamental antisymmetric mode $A_0$. These modes occur for each excitation frequency. For Lamb waves of short wavelength, several modes of oscillation occur for one wavelength. These are used for symmetrical and antisymmetrical Lamb waves ($S_0$, $S_1$, $S_2$ and so on, respectively $A_0$, $A_1$, $A_2$ and so on). These modes have different wavelengths and propagation speeds. They interact differently with small disturbances and mode conversion could occur in presence of of damage or other changes [135].

Linear ultrasonic technology focuses on the measurement of the velocity of sound, attenuation, transmission coefficients and reflection coefficients for crack detection [1,136–138].

Nonlinear ultrasound techniques provide the ability to detect creep and thermal aging. Generation of the second harmonic or higher order harmonics can be used for accurate flaw detection. The relationship between the amplitude of the second harmonic wave and the fundamental wave is a proven approach [1,139–146].

Berwig et al. focused on examining the blade root of a turbine blade with Rayleigh waves for surface cracks. These blades are made of $\gamma$-titanium aluminide and are brittle. After manufacturing the blade root, this investigation serves the final quality control [147–149].

Lane developed an ultrasonic array system to inspect single-crystal turbine blades in-situ [150]. Ultrasonic phased arrays consist of several ultrasound transducers, where the excitation time and amplitude is controlled individually by computer. This gives the possibility, to focus the ultrasonic waves. This leads to a more accurate statement about the crack position and size. In addition, the SNR is improved [151,152]. Chatillon et al. has dealt with complex geometries to detect errors with the phased array transducer [153].

The next subsections show special applications that are already used for the SHM of gas turbine components or that offer promising solutions for future research.

### 3.3.1. Waveguides

Harold et al. have filed a patent dealing with the condition monitoring of gas turbine components by means of acoustic waveguides (AWG) [154]. The general idea is to send an acoustic wave through waveguides to each vane. The sound waves pass through the vanes and are received by a second acoustic waveguide. A filter removes the lower frequencies below 30 kHz. As the coating on the blade deteriorates the size and/or velocity of the resulting acoustic wave changes. This is an indication that a blade must be serviced. The wave velocity for an intact coating is about 2500 m/s and for a completely deteriorated coating almost 5000 m/s. The principle of this idea is shown in Figure 16.
a wire with a diameter of 1 mm of platinum and platinum/13% rhodium was chosen. They must survive in the environment of hot oxidative gases and high dynamic loads. Typically AWGs use small wires or rods with a diameter of 0.25 to 6.4 mm [155].

Willsch et al. have implemented this concept practically in an industrial gas turbine [156]. Atkinson et al. have studied how the acoustic waves from the transducer are directed into the AWG via a conical transformer [157]. This is an elegant solution to the temperature issue but it is not feasible for industrial applications.

Waveguides are also used for temperature measurement with ultrasound (ultrasound thermometry). The basic idea is to run ultrasonic waves through the waveguide. By increasing the temperature, the waveguide expands. This can be measured in the delayed ultrasonic signal and give conclusions about the temperature to be measured [158–162].

3.3.2. Thermosonic or Ultrasonic Stimulated Thermography

This is a NDT&E method to detect micro-cracks on the surface and sub-surface of test parts. The test object is vibrated ultrasonically with or without contact or air-coupled transducers. As a result, the contact surfaces of the crack rub against each other and generate heat. With an infrared camera, the crack can then be detected (Figure 17).

The patent also describes the possibility of performing measurements without an acoustic input signal. As the rotating blades spin past the vanes, the blade produces a pulsating gas pressure, causing acoustic waves in the vane. This signal can be measured via an acoustical waveguide.

The inventors gave a good description of how the acoustic waveguides can be designed and fastened with the sensors and vanes. The AWGs are connected to the vanes by either a point or a few inches long. When entering the AWG, the waves are converted into longitudinal waves. They must be a good transmitter for high-frequency acoustic signals (20 to 500 kHz) and be weldable. A variety of metals and composites are also possible. For the investigations of this patent, a wire with a diameter of 1 mm of platinum and platinum/13% rhodium was chosen. They must survive in the environment of hot oxidative gases and high dynamic loads. Typically AWGs use small wires or rods with a diameter of 0.25 to 6.4 mm [155].

Dyrwal et al. have demonstrated that with this technique micro-cracks can be detected [172]. Dhital et al. investigated a similar topic, but used air-coupled transducers for damage detection [173].

Figure 16. Schematic setup of an acoustic waveguide sensor.

Figure 17. Thermosonic measurements with air-coupled transducer on a turbine blade.
Dyrwal et al. have demonstrated that with this technique micro-cracks can be detected on the outer shroud of a turbine blade. In this work different techniques were compared. The non-contact nonlinear air-coupled thermosonics (NACT) technique was the most promising. Zhang investigated the physical process of the chaotic excitation of a turbine blade to improve the fault detection capability of thermosonic technology.

Further literature is available for the air-coupled transducers.

3.3.3. Laser Ultrasonic

In laser ultrasound, the ultrasonic wave is transmitted with light. The light of a laser hits a material surface where it is absorbed in a layer near the surface, resulting in heating, thermal expansion and finally emission of an ultrasonic wave (Figure 18). The generated ultrasonic waves propagate perpendicular to the sample surface and are independent of the angle of the laser.

Compared to conventional piezoelectric transducers (PZT), ultrasound generation by laser irradiation offers several advantages. No sensor contact is required; they have high spatial resolution and can work on complex designed surfaces. The acoustic waves can be received with PZT or electromagnetic acoustic transducers (EMAT). Laser interferometers on the receiving side are also possible.

The surface of the component to be examined is not damaged. One disadvantage is the poor SNR with laser ultrasound. Masserey et al. and An et al. worked on crack detection using laser ultrasound with focus on complex designed geometries. Pei et al. studied the inspection of inner cracks. Dhital et al. investigated a similar topic, but used air-coupled transducers for damage detection.

Direct applications of this technology in the turbomachinery sector have not been identified, but this is a promising NDT&E approach. An SHM application is difficult here because laser ultrasonic has the same disadvantages as the optical technique shown above.

3.3.4. High Temperature Transducers

To solve the problem of extremely high temperatures in gas turbines, there are also promising approaches to use high-temperature ultrasonic sensors. On the one hand, it is important to solve the problem that the sensors themselves survive the high temperature, but on the other hand, there must also be a functioning acoustic coupling between the sensor and the test object. \(\text{La}_2\text{Ti}_2\text{O}_7\) (lanthanum titanate) or \(\text{LiNbO}_3\) (lithium niobate) enables operation at higher temperatures (Curie temperature higher than 1000 °C). Other promising materials are also possible. Many of these ideas are still in developmental status and commercially difficult to make available, but as an SHM approach, shows potential good performance.
3.3.5. EMAT

Alternating current is conducted into the induction coil of the sensor. This generates electromagnetic oscillations, which in turn induce eddy currents on the surface of the test object. The eddy current interacts with the permanent magnetic field of the sensor and generates ultrasonic waves through the Lorentz force directly on the surface of the test object (Figure 19).

![Figure 19. Working principle EMAT.](image)

The main advantage is certainly that EMATs work contactless and the sensor is located a few millimetres from the test object. That means that no coupling is needed. In addition, a wide variety of ultrasonic wave types can be generated that would not be possible with PZTs. For example, transverse shear waves (SH) can be transmitted and received. They can also be used at significantly higher temperatures. The disadvantage is that metal particles can be attracted by the magnet. In addition, there is the danger of magnetization of the test object. These sensors are typically quite bulky and are not commonly used in aerospace applications [177].

Edwards et al. and Jian et al. researched Rayleigh waves generated by EMAT for crack detection [178–180]. Dixon et al. studied a hybrid method in order to bring the ultrasonic waves into the component by means of a laser and to detect them using EMAT [181].

3.4. Temperature Measurements

Pyrometry and infrared thermography are good methods to detect creep, low-cycle fatigue, corrosion, erosion and oxidation [87,182]. Phosphor thermometry is able to identify failures such as cracks, erosion, corrosion and wear [183]. These techniques will be presented in detail in the next chapters.

3.4.1. Pyrometry

With this technique, thermal radiation at the target point of the turbine blades can be measured. They are used for non-contact temperature measurement. The emitted thermal radiation of a body can be measured with the help of a pyrometer (Figure 20).
The relationship between the surface temperature and the emitted radiant energy is defined by Planck’s law (Equation (11)):

$$E_b(\lambda, T) = \frac{c_1 \lambda^{-5}}{\exp\left(\frac{c_2}{\lambda T}\right) - 1}$$

(11)

where $E_b$ is the monochromatic radiation, $T$ the temperature, $\lambda$ the wavelength, $c_1$ and $c_2$ the radiation constants.

Pyrometers have the capability to measure ~40 points per blade [184]. To make these sensors work, no upper temperature limit exists. However, a minimum temperature limit of ~500 °C is required. They have a fast response and no physical contact with the turbine blades. Pyrometry is also immune to electromagnetic interference. However, finding a position to install the probe where it can view the blade surfaces is problematic and the optic has to be protected from deposits. The instrumentation is vulnerable to extreme temperatures, vibrations and high pressures. The combustion gases are turbulent with variable densities and high velocities. The sensors in the flow channel have to operate at an oxidising atmosphere. Corrosion is also a failure mode. It could be difficult to gain enough radiation of the small target area of the blades. In addition, records for representative temperature measurements are needed. Difficulties are reported in specifying the emission accurately [182].

The development of a blade temperature management system with pyrometers was reported by different authors [185–189]. Alaruri et al. [190] and Gao et al. [191] analysed the emissivity of different superalloys with TBC and verified the theoretical approach experimentally. Kerr et al. [192] and Daniel et al. [193] presented an overview of measurement and reflection errors associated with pyrometers.

3.4.2. Infrared Thermography

This is an imaging process for displaying the surface temperature of objects. Every body with a temperature above the absolute zero emits heat radiation. An infrared camera converts the infrared radiation, which is invisible to the human eye, into electrical signals. From this, the camera creates a picture. Infrared cameras are used to take infrared pictures of the whole aerofoil that has to be evaluated (Figure 21).

Lemieux from Siemens Westinghouse Power Corporation showed a conceptual design of this technology in gas turbines where two IR cameras were used. These cameras also have a blade positioning sensor allowing tracking of all blades [194]. The automatic comparison of the IR pictures showed temperature increases at specific points of the blade. This could also be used to monitor degradation of the TBC or cracks.
3.4.2. Infrared Thermography

This is an imaging process for displaying the surface temperature of objects. Every body with a temperature above the absolute zero emits heat radiation. An infrared camera converts the infrared radiation into an optical image and provides it to the user (Figure 21).

Bison et al. evaluated especially this topic. They compared pulsed thermography and thermal wave interferometry to estimate ageing effects of the TBC [195,196]. Sun et al. showed a multilayer analysis method to detect change in TBC properties as well [197]. Meola et al. developed a non-destructive testing method to detect small remaining ceramic fragments in the casted core [198].

3.4.3. Phosphor Thermometry

The basis of this technique is a thermal history coating. Active rare earth ions are introduced. This could be integrated in the TBC, which is what the hot gas parts in a gas turbine are mostly coated with. By illuminating with an excitation light after a machine run, phosphorescence gives information about the temperature that the coatings experience. Figure 22 shows a combustion chamber outlet liner, with thermal paints, after service.

A temperature capability of –5 to 1550 °C is given and a precision of ±5 °C was proved. The disadvantage of pyrometry and infrared thermography is the lower susceptibility to background radiation. Feist et al. showed in different publications the industrial application of phosphor thermometry. To develop these special paintings a Rolls-Royce jet engine was used for verification [183,200–204]. The application in an industrial gas turbine was proven as well. The surface temperature of combustion chamber transition ducts, turbine blades and sideplates were measured and used for validation for the CFD calculations [205,206].

3.5. Induction Thermography

In inductively excited thermography, an electrically conductive component is inductively impressed by a near-surface eddy current. The resulting heating of the component can be visualized with an infrared camera (Figure 23). Cracks disturb the current flow and thus also influence the temperature development in the test part. This technique is similar (Section 3.3.2) to thermosonic technique, where the excitation is achieved using ultrasound excitation.
The penetration depth of the electromagnetic field is described by the skin effect:

\[ \delta = \sqrt{\frac{2\rho}{\omega \mu}} \]  

where \( \delta \) describes the skin depth, \( \rho \) is the conductor resistivity, \( \omega \) the angular frequency and \( \mu \) the absolute permittivity. The choice of the excitation frequency determines, inter alia, the depth to be examined into the component. In addition, coil geometry and power are also important test parameters [207].

The method is of particular interest where the crack detection with conventional methods is difficult or impossible. Compared to other excitation techniques, inductive excitation offers the advantage of being less sensitive to radiation or emission differences on a test surface, as the heat is generated directly in the test part [207].

Sensitivity to crack detection on components can be compared to magnetic particle inspection [208]. The advantage of induction thermography compared to magnetic particle testing is that it is non-contact. In addition, the use of chemicals is avoided. Accuracy is limited when components with highly reflective surfaces are tested, which have a low emissivity [208].

Carl et al. developed a system for automatic crack detection of turbine blades. A robotic arm puts the blades in test position. After the measurement was carried out, the automatic evaluation of the blades takes place [207]. Šrajbr et al. used induction thermography for automated crack inspection of aircraft structures [209,210]. Bamberg et al. also introduces a system for inspecting turbomachinery components for cracks using induction thermography [211]. Vrana dealt with active thermography and examined cracks in the foot area of gas turbine blades [212].

Spießberger et al. compared inductive thermography with ultrasonically excited thermography [213]. The detectability of cracks in induction thermography depends on their orientation in the component and in ultrasound-excited thermography, this is largely independent. With ultrasonic excitation, the detection of defects is possible at a greater depth than in the inductively excited thermography, since the heat of fracture generated by ultrasound diffuses from greater depths to the surface. With complex geometries, it is difficult to excite the component surface evenly with induction. It is then often only possible to examine individual component areas. If the entire component is to be tested, the use of ultrasound-excited thermography appears to be more sensitive. For materials with low thermal conductivity, ultrasound-excited thermography is more suitable as a test method [213].

### 3.6. Acoustic Emission (AE)

The basis of AE are transient elastic waves. They are generated due to the release of energy in a material when a crack occurs. The frequency range is between 100 KHz and 1 MHz.

Defects in a machine generate an individual type of crack growth. This phenomenon results in elastic waves, which can be detected with AE sensors (Figure 24) [85].

![Induction Thermography](image-url)
Al-Obaidi et al. provided a good explanation on how AE works: “Instead of supplying energy to the object under examination, AE simply listens for the energy released by the object” [214].

AE is sensitive enough to monitor minor changes in the gas turbine parts. It has the potential to detect failures at an early stage. Furthermore, the crack position can be detected without intrusion. AE sensors were used to observe roller bearings in gas turbines [215]. It was found that AE is a proper tool to detect rubbing from rotor to stator.

Mba et al. showed promising experiments for early rub detection in turbine rotors [216]. AE is very sensitive to incipient cracks compared to vibrational measurements. It was shown that the AE technique is able to detect cracks in blades earlier than vibrational analysis [87]. One drawback is the attenuation of the signal. Therefore, the sensor has to be placed close to the signal source [217]. The accuracy of AE is limited because the propagation of the velocity of the acoustic wave depends on temperature, pressure and the relative speed of the medium [37].

Mba et al. developed a AE technique for monitoring rotating machines as well [217]. The focus was on bearings, pumps, gearboxes, engines and rotating structures. Leahy et al. investigated experimentally rubbing detection between rotor and stator of a steam turbine [218,219]. Nashed et al. explored the sources of fluid generated AE in a running turbine [215]. These findings can be used for blade faults predictions.

AE is a relatively simple and cost effective monitoring technique for health monitoring of hot gas components. These sensors are limited by the high temperatures in the gas turbine, so that use in the hot gas without active cooling is not possible.

3.7. Mm-Waves

Mm-waves are microwaves with wavelengths in the millimetre range between 1 and 10 mm. This corresponds to a frequency band from 30 to 300 GHz. The physical principle is the same as described for tip-timing measurement in Section 3.1.1.

The mm-wave technique is commonly used in motion detection and special radar systems. It can be used to measure movements of turbine blades, rotor-stator clearances and the degradation of TBCs. TBCs typically consist of four layers. Starting from the bottom the metal substrate, the metallic bond coat, the thermally grown oxide (TGO) and the ceramic topcoat. Mm-waves are nearly totally reflected by metal surfaces. TBCs have a high dielectric constant of typically 25. The waves will be partly absorbed and differently reflected. These differently reflected waves could be used to detect the degradation of the coating (Figure 25). An antenna sends the waves to the moving blades and the reflected waves will be evaluated. Tests were done with 2.45 and 10 GHz sensors. Further investigations concerning the electromagnetic losses of TBCs are necessary. Willsch et al. have shown this SHM application in a running gas turbine [156].

The degradation of a TBC is a relevant indicator for health monitoring of the hot gas parts. However, an active cooling strategy for the antenna is needed.
3.8. Pressure Measurements

In gas turbine casings are placed several pressure sensors. The distortion of the pressure fields around the blades can be used for fault detection [220]. An experimental study was completed to find a correlation between compressor casing vibrations and the pressure field around the compressor blades [87]. The pressure field around the compressor blades gives a clearer picture about the blade fault compared to vibrational analysis [88]. It was shown that pressure measurements deliver better results than common vibrational probes [87]. Mathioudakis et al. found different signatures corresponding to the different blade faults [220,221]. Failure modes, such as blade creep, rotor eccentricity and rubbing, were experimentally detected [88]. Forbes et al. presented a concept to combine internal casing pressure measurements with casing vibration measurements. It was proven that blade defects can be detected [222–226].

3.9. Direct-Write Thermal Spray (DWTS) Sensors

DWTS is a spray process that accelerates material to high speeds and hits a substrate. This creates a dense and strongly adherent deposit. Typically, the material is injected in the form of a powder, wire or rod into a high velocity combustion or thermal plasma flame. As a result, thermal and kinetic energy is supplied to the particles (Figure 26). This is a similar process to apply TBC to a component [227]. The DWTS process allows the additive generation of sensor circuits on complex shaped components without pre-masking.
Longtin et al. have studied the use of temperature sensors and strain gauges manufactured by DWTS. K-type thermocouples can measure high operating temperatures. Industry standard K-type thermocouples are made of Chromel (90Ni/10Cr) and Alumel (95Ni/3Mn/2Al/1Si) [227]. A K-type thermocouple consists of a pair of metallic conductors made of these different metals. These are connected at one end and are suitable for temperature measurement due to the thermoelectric effect. In principle, the thermocouple provides electrical energy from heat at a temperature difference along the electrical conductor (Figure 27).

Schönberg et al. has also applied K-type thermocouples with DWTS direct to the surface of a test component. In this way, in-situ temperature measurement data of the turbine blades can be determined [228]. Zhang et al. has developed a system to weld K-type thermocouples directly onto the TBC, which then embeds the sensor through a second ceramic layer [229].

Capacitive strain gauges are well suited for determining the mechanical stress of a component in extreme environmental conditions. Compared to resistive stain gauges, they have a better SNR and are not so sensitive to high temperature and temperature fluctuations. If the fingers of the sensor are displaced relative to each other by the component load, this results in a capacitive change $\Delta C$. This is approximately proportional to the strain $\varepsilon$ [230] (Figure 28).

Li et al. used DWTS and ultrafast laser micromachining to produce the sensor. After spraying a layer, the laser worked out the exact sensor contour [230].

It is advantageous that thermal spray is already used as a standard method for applying a protective layer in gas turbine components. As a result, the sprayed-on layer adheres directly to the component. No adhesive or mechanical connection is necessary. It is also possible to form a capacitor-inductor circuit (LC). This would provide the opportunity to transmit the measurement data passively and wirelessly [230–232].

Chen et al. dealt with micromachining strain gauges using DWTS and precision laser [233]. Hon et al. have published an overview of the various direct write technologies and their state of development [234]. Pique reports different areas of application in his book on direct-writing technologies [235].

This technique is a promising way to determine the condition of the turbine blades in-situ and may be considered in future research. It will certainly be difficult, due to the high temperature, to use the electronic components for data transmission as shown in Figure 26. Passive data transmission systems would be a good solution.
3.10. Uniform Crystal Temperature Sensor (UCTS)

These sensors are particularly suitable for prototype machines to determine the temperature at certain points of the test subject. The crystal sensor is inserted into the test object and a pocket with a diameter of 0.75 mm and 0.75 mm deep must be removed. After inserting the crystal, it is closed again. When the test run is finished, the sensor must be removed for evaluation [236]. The temperature changes the lattice structure of the crystal, allowing an accuracy of $\pm 3.3^\circ C$ [237]. The application temperature is 150-1430 $^\circ C$ [237]. DeVoe et al. have studied their use in gas turbine blades [238,239].

3.11. Performance Monitoring

Performance monitoring processes different input data. The temperature, pressure and speed of the gas turbine are used to calculate the performance of the machine. This method is also able to detect blade faults. This fault changes the aerodynamic behaviour of the blades, and ultimately, the performance of the whole machine. A combination of performance and vibration monitoring is known as a hybrid method. It was shown that blade deformation, blade wearing and blade fouling could be detected [240].

Lattime et al. discussed the degradation of the high pressure turbine (HPT) performance. Especially, wear on blade tip and sealing can account for losses in the HPT performance of 1% and more [241]. Salar et al. developed a method to detect faulty components inside a gas turbine. Therefore, a gas path analysis and extended Kalman filters were used. The main parameters to detect the degradation are efficiency and flow capacity of the compressor and the turbine [242]. Diallo et al. established a new statistical signal processing technique for performance monitoring [243]. A physics-based modelling approach for performance deterioration were shown by Hanachi et al. [244]. Heat loss index and power deficit index were used as indicators. Different methodologies to improve the robustness of performance monitoring against sensor faults were developed [245,246]. Tahan et al. reviewed current gas path performance monitoring techniques. The different failure modes in a gas turbine and the possibilities to detect them with performance monitoring were listed [247].

This technique is very promising for future condition based maintenance concepts. Many sensor technologies are already available as standard monitoring in series machines that can be used for the evaluation. A research focus should be on the further development of modern signal processing techniques.

4. Conclusions

The most relevant failure modes for gas turbine hot gas components have been described. For blade health monitoring, TMF, creep, oxidation and corrosion are the main life limiters depending on the turbine configuration. For the transition ducts of the combustion chambers, wear of the mating surfaces and degradation of TBCs and cracks are problematic.

A review of most relevant NDT/SHM techniques for gas turbine was reported. Tip timing technology allows for measurement of the tip clearance of the blades, time of arrival and also axial deflection. This is a very powerful method for which several sensor technologies could be used. Optical measurements are the best choice for prototype machines because of their accuracy and resolution, but the sensitive optics could be problematic for long-term operation in field machines. Inductive and magnetoresistive sensors are not able to work in the first turbine stage because of their limited temperature. Microwave, EC and capacitive sensors seem to be a good choice for turbine tip timing.

The pyrometry enables the measurement of heat radiation at different points of the blades. IR cameras take infrared pictures of every blade during every rotation. Both monitoring techniques compare previous measurements and pictures. Phosphor thermometry is a new innovative method for temperature detection. However, all of them are rather complex solutions for health monitoring in field machines.
AE is a powerful diagnosis technique. The sensor has to be placed close to the signal source. This is a promising method to detect rubbing; however, limited accuracy and signal attenuation of this technique needs can be a limiting factor.

Millimetre waves are a possibility to detect degradation of TBCs. However, a sensor antenna directly placed in the hot gas disturbs the flow and the sensor has to be actively cooled.

The detection of pressure differences in the flow channel could also be a good indicator for health monitoring of gas turbine components, especially in combination with other technologies.

The vibrational approach is the classical technique, where for example accelerometers on the bearings and on the casings provide important information on the machine behavior; however, it is known that blade failures could not be detected readily with conventional vibration measurements.

The ultrasound methods presented offer promising possibilities in the field of NDT&E and SHM.

The current review reports some current NDT/SHM solutions, able to mostly monitor the components contactless and the results shows that a combination of monitoring technology would provide valuable information about the performance and integrity of gas turbine components.

A summary of the different techniques with the corresponding properties is shown qualitatively in Table 2. Here it is presented which failure modes can be detected with the individual techniques and whether they are primarily suitable for NDT or SHM. In addition, important properties and also the costs are summarized.

| Sensing Technology | Failure Modes | NDT | SHM | Temp. limit | Band-width | Accuracy | Sensitivity | Costs |
|--------------------|---------------|-----|-----|-------------|------------|----------|-------------|-------|
| Microwave probes   | Creep         | X   | +   | +           | +          | 0        | 0           | -     |
| Optical sensor     | Crack         | X   | −   | +           | +          | +        | +           | +     |
| Inductive sensors  | Rubbing       | X   | −   | 0           | +          | 0        | 0           | +     |
| EC sensors         | Wear          | X   | +   | +           | +          | +        | +           | +     |
| Magnetoresistive sensors |       | X   | +   | 0           | 0          | 0        | 0           | +     |
| Capacitance sensors |               | X   | +   | 0           | 0          | 0        | +           | -     |
| Viscous monitoring | Rubbing       | X   | 0   | +           | −          | -        | -           | +     |
| Ultrasonic         | FOD           | X   | 0   | +           | −          | -        | -           | +     |
| Waveguides         | Crack         | X   | +   | 0           | +          | +        | +           | +     |
| Thermosonic        |              | X   | −   | 0           | +          | +        | 0           | −     |
| Laser Ultrasonic   | Crack         | X   | −   | 0           | +          | +        | −           | −     |
| High temperature   |              | X   | +   | 0           | +          | +        | −           | −     |
| transducers        |               | X   | −   | 0           | +          | +        | 0           | +     |
| EMAT               |               | X   | −   | 0           | +          | +        | 0           | +     |
| Pyrometry           | Creep         | X   | X   | +           | −          | 0        | 0           | −     |
| Infrared thermography |               | X   | X   | +           | −          | 0        | 0           | −     |
| Phosphor thermometry |               | X   | +   | −           | 0          | 0        | 0           | −     |
| Induction thermography | Crack       | X   | X   | +           | −          | 0        | 0           | −     |
| AE                 | Creep         | X   | X   | −           | 0          | +        | +           | −     |
| Mm-waves           | Crack         | X   | X   | +           | +          | +        | +           | +     |
| Pressure           | Creep         | X   | X   | +           | +          | +        | +           | +     |
| measurements       | Rubbing       | X   | +   | +           | +          | +        | +           | +     |
| DWTS               | Creep         | X   | X   | +           | +          | 0        | +           | −     |
| UCTS               | Creep         | X   | X   | +           | +          | 0        | +           | −     |
| Performance        | Creep         | X   | X   | +           | +          | 0        | +           | −     |
| monitoring         | Creep         | X   | X   | +           | +          | +        | +           | +     |

"+" means positive and "−" means negative in this context. "0" means neutral.

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References
1. Mitra, M.; Gopalakrishnan, S. Guided wave based structural health monitoring: A review. *Smart Mater. Struct.* **2016**, *25*, 53001. [CrossRef]
2. Proulx, T. Rotating Machinery, Structural Health Monitoring, Shock and Vibration, Volume 5, Proceedings of the 29th IMAC, A Conference on Structural Dynamics 2011; Springer: New York, NY, USA, 2011.
3. Mukhopadhyay, S.C. New Developments in Sensing Technology for Structural Health Monitoring; Springer: Berlin/Heidelberg, Germany, 2011.
4. Niezrecki, C. Structural Health Monitoring & Damage Detection, Volume 7: Proceedings of the 35th IMAC, A Conference and Exposition on Structural Dynamics 2017; Springer: Cham, Switzerland, 2017.
5. Ostachowicz, W. New Trends in Structural Health Monitoring; Springer: London, UK; Wien, NY, USA, 2013.
6. Jansohn, P. (Ed.) Modern Gas Turbine Systems: High Efficiency, Low Emission, Fuel Flexible Power Generation; Woodhead Publishing: Cambridge, UK, 2013.
7. Pallos, K.J. *GER-3957B—Gas Turbine Repair Technology*; GE Energy Services Technology: Atlanta, GA, USA, 2001.
8. Meher-Homji, C.B. Blading Vibration and Failures in Gas Turbines: Part A—Blading Dynamics and the Operating Environment. In *ASME 1995 International Gas Turbine and Aeroengine Congress and Exposition: Volume 4: Heat Transfer; Electric Power; Industrial and Cogeneration, Houston, TX, USA, 5 June 1995*; American Society of Mechanical Engineers: New York, NY, USA, 1995; p. V004T11A014.
9. Jaw, L.C.; Wu, D.N.; Bryg, D.J. Tracking and Control of Gas Turbine Engine Component Damage/Life. In *RTO A VT Symposium on “Aging Mechanisms and Control”*; RTO: Manchester, UK, 2001.
10. Mishra, R.K.; Srinivasan, K. Failure of Low-Pressure Turbine Blades in Military Turbofan Engines: Causes and Remedies. *J. Fail. Anal. Prev.* **2016**, *16*, 622–628. [CrossRef]
11. Mishra, R.K.; Thomas, J.; Srinivasan, K.; Nandi, V.; Bhatt, R.R. Failure analysis of an un-cooled turbine blade in an aero gas turbine engine. *Eng. Fail. Anal.* **2017**, *79*, 836–844. [CrossRef]
12. Park, M.; Hwang, Y.-H.; Choi, Y.-S.; Kim, T.-G. Analysis of a J69-T-25 engine turbine blade fracture. *Eng. Fail. Anal.* **2002**, *9*, 593–601. [CrossRef]
13. Russhard, P. The rise and fall of the rotor blade strain gauge. *Mech. Mach. Sci.* **2015**, *23*, 27–37.
14. Kestner, B.; Lieuwen, T.; Hill, C.; Angello, L.; Barron, J.; Perullo, C.A. Correlation analysis of multiple sensors for industrial gas turbine compressor blade health monitoring. *J. Eng. Gas Turb. Power* **2015**, *137*, 112605. [CrossRef]
15. Guo, H.; Duan, F.; Zhang, J. Blade resonance parameter identification based on tip-timing method without the once-per revolution sensor. *Mech. Syst. Signal Process.* **2016**, *66–67*, 625–639. [CrossRef]
16. Geisheimer, J.L.; Kwapisz, D.; Holst, T.; Hafner, M. Blade Tip Clearance Sensors for Use in Engine Health Monitoring Applications. *SAE Int. J. Aerosp.* **2013**, *6*, 417–423. [CrossRef]
17. Du, L.; Zhu, X.; Zhe, J. A high sensitivity inductive sensor for blade tip clearance measurement. *Smart Mater. Struct.* **2014**, *23*, 065018. [CrossRef]
18. Kwapisz, D.; Hafner, M.; Queloz, S. Calibration and characterization of a CW radar for blade tip clearance measurement. In Proceedings of the 7th European Radar Conference, Paris, France, 30 September–1 October 2010.
19. Witos, M.; Wisniow, M. Compressor Blade Health Monitoring with Use of Tip Timing and Modal Analysis Method. *PAMM 2009*, *9*, 209–212. [CrossRef]
20. Witos, M.; Wachlaczenko, M. Structural Health Monitoring of Compressor and Turbine Blades with the Use of Variable Reluctance Sensor and Tip Timing Method. In Proceedings of the 19th World Conference on Non-Destructive Testing, Munich, Germany, 13–17 June 2016.
21. Rokicki, E.; Majewski, P.; Rzadkowski, R.; Szczepanik, R.; Spychała, J.; Drewczyński, M. Analysis of Middle Bearing Failure in Rotor Jet Engine Using Tip-Timing and Tip-Clearance Technique. In Proceedings of the ASME Turbo Expo: Turbo Technical Conference and Exposition, Düsseldorf, Germany, 16–20 June 2014; American Society of Mechanical Engineers: Düsseldorf, Germany, 2014; p. V07AT31A014.
22. Tamura, K.; Ono, M.; Torii, S.; Morimoto, K. Non-contact Vibration Measurement of the Rotor Blades that Play a Pivotal Role in the Reliability of Gas Turbines. *Mitsubishi Heavy Ind. Tech. Rev.* **2014**, *51*, 10.
23. Dimitriadis, G.; Carrington, I.B.; Wright, J.R.; Cooper, J.E. Blade-tip timing measurement of synchronous vibrations of rotating bladed assemblies. *Mech. Syst. Signal Process.* **2002**, *16*, 599–622. [CrossRef]
24. Gallego-Garrido, J.; Dimitriadis, G.; Wright, J.R. A Class of Methods for the Analysis of Blade Tip Timing Data from Bladed Assemblies Undergoing Simultaneous Resonances—Part I: Theoretical Development. *Int. J. Rotating Mach.* 2007, 1–11. [CrossRef]

25. Gallego-Garrido, J.; Dimitriadis, G.; Carrington, I.B.; Wright, J.R. A Class of Methods for the Analysis of Blade Tip Timing Data from Bladed Assemblies Undergoing Simultaneous Resonances—Part II: Experimental Validation. *Int. J. Rotating Mach.* 2007, 1–10. [CrossRef]

26. Russhard, P. Blade tip timing (BTT) uncertainties. In *AIP Conference Proceedings*; AIP Publishing: Melville, NY, USA, 2016; Volume 1740.

27. Satish, T.N.; Murthy, R.; Singh, A.K. Analysis of uncertainties in measurement of rotor blade tip clearance in gas turbine engine under dynamic condition. *Proc. Inst. Mech. Eng. Part G J. Aerosp. Eng.* 2014, 228, 652–670. [CrossRef]

28. Roeder, J.W.; Bencic, T.J.; Hughes, C.E.; Woike, M.R. Testing of a Microwave Blade Tip Clearance Sensor at the NASA Glenn Research Center—NASA/TM-2009-215589. In *47th Aerospace Sciences Meeting*; Sponsoring Organization; NASA Glenn Research Center: Cleveland, OH, USA, 2009.

29. Zhang, J.; Duan, F.; Niu, G.; Jiang, J.; Li, J. A Blade Tip Timing Method Based on a Microwave Sensor. *Sensors* 2017, 17, 1097. [CrossRef] [PubMed]

30. Violetti, M.; Qin Xu, O.; Hochreutiner, A.K.; Skrivervik, A.K. New microwave sensor for on-line Blade Tip Timing in gas and steam turbines. In Proceedings of the 2012 Asia-Pacific Microwave Conference Proceedings (APMC), Kaohsiung, Taiwan, 4–7 December 2012.

31. Violetti, M.; Skrivervik, A.K.; Qin Xu, M.; Hafner, M. New microwave sensing system for blade tip clearance measurement in gas turbines. In Proceedings of the 2012 IEEE Sensors, Taipei, Taiwan, 28–31 October 2012.

32. Woike, M.; Abdul-Aziz, A.; Oza, N.; Matthews, B. New sensors and techniques for the structural health monitoring of propulsion systems. *Sci. World J.* 2013, 2013, 596506. [CrossRef]

33. Abdul-Aziz, A.; Woike, M.R.; Bencic, T.J. A Microwave Blade Tip Clearance Sensor for Propulsion Health Monitoring—NASA/TM-2010-216736. In *Infotech@Aerospace 2010 Conference*; Sponsoring Organization; NASA Glenn Research Center: Cleveland, OH, USA, 2010.

34. Wagner, M.; Schulze, A.; Vossiek, M.; Stephelbauer, C.; Weigel, R.; Vortmeyer, N.; Heide, P. Novel microwave vibration monitoring system for industrial power generating turbines. In *Proceedings of the IEEE MTT-S International Microwave Symposium Digest*, Baltimore, MD, USA, 7–12 June 1998; Volume 3.

35. Hafner, M.A.; Holst, T.A.; Billington, S.A. Blade tip measurement advanced visualization using a three dimensional representation. In Proceedings of the ASME Turbo Expo 2011, Vancouver, BC, Canada, 6–10 June 2011; American Society of Mechanical Engineers: Vancouver, BC, Canada, 2011; Volume 3.

36. Neri, P.; Peeters, B. Non-Harmonic Fourier Analysis for bladeed wheels damage detection. *J. Sound Vib.* 2015, 356, 181–194. [CrossRef]

37. Schicht, A.; Schwarzer, S.; Schmidt, L.-P. Tip Clearance Measurement Technique for Stationary Gas Turbines Using an Autofocusing Millimeter-Wave Synthetic Aperture Radar. *IEEE Trans. Instrum. Meas.* 2012, 61, 1778–1785. [CrossRef]

38. Tomassini, R.; Rossi, G.; Brouckaert, J-F. On the development of a magneto resistive sensor for blade tip timing and blade tip clearance measurement systems. *Rev. Sci. Instrum.* 2016, 87, 102505. [CrossRef]

39. Gopalakrishnan, S. *Computational Techniques for Structural Health Monitoring*; Springer: London, UK, 2011.

40. Przybylowska, R.; Rokicki, E. Inductive sensors for blade tip-timing in gas turbines. *J. KONBIN* 2015, 36, 147–164. [CrossRef]

41. Lawson, C.P.; Ivey, P.C. Tubomachinery blade vibration amplitude measurement through tip timing with capacitance tip clearance probes. *Sens. Actuators A Phys.* 2005, 118, 14–24. [CrossRef]

42. Tomassini, R.; Rossi, G.; Brouckaert, J-F. On the development of a magneto resistive sensor for blade tip timing and blade tip clearance measurement systems. In *AIP Conference Proceeding*; AIP Publishing: Melville, NY, USA, 2014; Volume 1600.
46. Battiato, G.; Firrone, C.M.; Berruti, T.M. Forced response of rotating bladed disks: Blade Tip-Timing measurements. *Mech. Syst. Signal Process.* 2017, 85, 912–926. [CrossRef]

47. Di Maio, D.; Ewins, D.J. Experimental measurements of out-of-plane vibrations of a simple blisk design using Blade Tip Timing and Scanning LDV measurement methods. *Mech. Syst. Signal Process.* 2017, 85, 912–926. [CrossRef]

48. Chen, Z.; Yang, Y.; Xie, Y.; Guo, B.; Hu, Z. Non-contact crack detection of high-speed blades based on principal component analysis and Euclidian angles using optical-fiber sensors. *Sens. Actuators A Phys.* 2013, 201, 66–72. [CrossRef]

49. Gwashavanhu, B.; Oberholster, A.J.; Heyns, P.S. Rotating blade vibration analysis using photogrammetry and tracking laser Doppler vibrometry. *Mech. Syst. Signal Process.* 2016, 76–77, 174–186. [CrossRef]

50. Büttner, L.; Pfister, T.; Czarske, J. Fiber-optic laser Doppler turbine tip clearance probe. *Opt. Lett.* 2006, 31, 1217–1219. [CrossRef]

51. Pfister, T.; Büttner, L.; Czarske, J.; Krain, H.; Schodl, R. Turbo machine tip clearance and vibration measurements using a fibre optic laser doppler position sensor. *Meas. Sci. Technol.* 2006, 17, 1693–1705. [CrossRef]

52. Gil-Garcia, J.M.; García, I.; Zubia, J.; Aranguren, G. Measurement of blade tip clearance and time of arrival in turbines using an optic sensor. In Proceedings of the 2015 International Conference on Applied Electronics (AE), Pilsen, Czech Republic, 8–9 September 2015.

53. Zielinski, M.; Ziller, G. Noncontact vibration measurements on compressor rotor blades. *Meas. Sci. Technol.* 2000, 11, 847–856. [CrossRef]

54. Overton, G. Fiberoptic probe measures turbine tip clearance. *Laser Focus World* 2006, 42, 20–23.

55. Sharma, V.K.; Hanagud, S.; Ruzzene, M. Damage Index Estimation in Beams and Plates Using Laser Vibrometry. *Asian J.* 2006, 44, 919–923. [CrossRef]

56. García, I.; Zubia, J.; Beloki, A.; Aldalbaketreku, G. Tip timing measurements for Structural Health Monitoring in a low-pressure turbine using three optical sensors. In Proceedings of the 8th European Workshop on Structural Health Monitoring (EWSHM 2016), Spain, Bilbao, 5–8 July 2016.

57. Günter, P.; Dreier, F.; Pfister, T.; Czarske, J.; Haupt, T.; Hufnab, W. Measurement of radial expansion and tumbling motion of a high-speed rotor using an optical sensor system. *Mech. Syst. Signal Process.* 2011, 25, 319–330. [CrossRef]

58. Oberholster, A.J.; Heyns, P.S. Eulerian laser Doppler vibrometry: Online blade damage identification on a multi-blade test rotor. *Mech. Syst. Signal Process.* 2011, 25, 344–359. [CrossRef]

59. Pfister, T.; Büttner, L.; Czarske, J. Laser doppler profile sensor with sub-micrometre position resolution for velocity and absolute radius measurements of rotating objects. *Meas. Sci. Technol.* 2005, 16, 627–641. [CrossRef]

60. Przysowa, R.; Majewski, P.; Rokicki, E. Optimized magnetic sensors to measure speed and position in adverse environments. In Proceedings of the IET & ISA 60th International Instrumentation Symposium 2014, Cavendish Square, London, UK, 24–26 June 2014.

61. Procházká, P. Sensors for noncontact vibration diagnostics in rotating machinery. In *AIP Conference Proceedings*; AIP Publishing: Melville, NY, USA, 2016; Volume 1740.

62. Roeseler, C.; von Flotow, A.; Tappert, P. Monitoring blade passage in turbomachinery through the engine case (no holes). In Proceedings of the IEEE Aerospace Conference Proceedings, Big Sky, MT, USA, 9–16 March 2002.

63. Ghoshal, A.; Le, D.; Kim, H. Technological assessment of high temperature sensing systems under extreme environment. *Sens. Rev.* 2012, 32, 66–71. [CrossRef]

64. Neumann, M.; Dreier, F.; Günter, P.; Wilke, U.; Fischer, A.; Büttner, L.; Holzinger, F.; Schiffer, H.-P.; Czarske, J. A laser-optical sensor system for blade vibration detection of high-speed compressors. *Mech. Syst. Signal Process.* 2015, 64–65, 337–346. [CrossRef]

65. Cardwell, D.N.; Chana, K.S.; Russhard, P. The use of eddy current sensors for the measurement of rotor blade tip timing-sensor development and engine testing. In Proceedings of the ASME Turbo Expo 2008, Power for Land, Sea and Air, Berlin, Germany, 9–13 June 2008; American Society of Mechanical Engineers: Berlin, Germany, 2008; Volume 2.

66. Liu, C.; Jiang, D. Improved Blade Tip Timing in Blade Vibration Monitoring with Torsional Vibration of the Rotor. *J. Phys. Conf. Ser.* 2012, 364, 12136. [CrossRef]
67. Chana, K.S.; Sridhar, V.; Singh, D. The use of eddy current sensors for the measurement of rotor blade tip timing: Development of a new method based on integration. In Proceedings of the ASME Turbo Expo 2016, Turbomachinery Technical Conference and Exposition, Seoul, South Korea, 13–17 June 2016; American Society of Mechanical Engineers: Seoul, South Korea, 2016; Volume 6.
68. Tsutomu, M.; Sho, G.; Kenta, D.; Yoshinori, K.; Yuichi, A.; Shigemi, E.; Hiroki, S. Method for Identifying Type of Eddy-Current Displacement Sensor. IEEE Trans. Magn. 2011, 47, 3554–3557. [CrossRef]
69. Przysowa, R.; Spychała, J. Health Monitoring of Turbomachinery Based on Blade Tip-Timing and Tip-Clearance. Available online: https://www.researchgate.net/profile/Radoslaw_Przysowa/publication/273137276_Health_Monitoring_of_Turbomachinery_Based_on_Blade_Tip-Timing_and_Tip-Clearance/links/54f97ca0cf28d6dca4edd9/Health-Monitoring-of-Turbomachinery-Based-on-Blade-Tip-Timing-and-Tip-Clearance.pdf (accessed on 10 October 2018).
70. Prochážka, P.; Vanek, F. New methods of non-contact sensing of blade vibrations and deflections in turbomachinery. In Proceedings of the 2013 IEEE International Instrumentation and Measurement Technology Conference (I2MTC), Minneapolis, MN, USA, 6–9 May 2013.
71. Prochážka, P.; Vank, F. Contactless diagnostics of turbine blade vibration and damage. J. Phys. Conf. Ser. 2011, 205, 012116. [CrossRef]
72. Brouckaert, J.-F.; Marsili, R.; Rossi, G.; Tomassini, R. Development and experimental characterization of a new non contact sensor for blade tip timing. In AIP Conference Proceedings; AIP Publishing: Melville, NY, USA, 2012; Volume 1457.
73. Tomassini, R.; Rossi, G. Noncontact vibration measurements using magnetoresistive sensing elements. In AIP Conference Proceedings; AIP Publishing: Melville, NY, USA, 2016; Volume 1740.
74. Tomassini, R. Blade Tip Timing and Blade Tip Clearance Measurement System Based on Magnetoresistive Sensors. Ph.D. Thesis, University of Padua, Padua, Italy, 2016.
75. Lawson, C.P. Capacitance Tip Timing Techniques in Gas Turbines. Ph.D. Thesis, Cranfield University, Silsoe, UK, 2003.
76. Steiner, A. Techniques for blade tip clearance measurements with capacitive probes. Meas. Sci. Technol. 2000, 11, 865–869. [CrossRef]
77. Sarma, G.R.; Barranger, J.P. Capacitance-type blade-tip clearance measurement system using a dual amplifier with ramp/DC inputs and integration. IEEE Trans. Instrum. Meas. 1992, 41, 674–678. [CrossRef]
78. Drumm, M.; Hause, W.C. High performance rotor monitoring. In Proceedings of the 19th 2000 Digital Avionics Systems Conference (DASC), Philadelphia, PA, USA, 7–13 October 2000.
79. Mönich, M.; Bailleul, G. Capacitive Measurement System for Blade Clearances. Sens. Rev. 1993, 13, 23–26. [CrossRef]
80. Mueller, D.; Mozumdar, S.; Johann, E. Capacitive measurement of compressor and turbine blade tip to casing running clearance. J. Eng. Gas Turbines Power 1997, 119, 877–884. [CrossRef]
81. Fabian, T.; Sangkyun Kang, F.; Prinz, F. Capacitive blade tip clearance measurements for a micro gas turbine. In Proceedings of the 19th 2000 IEEE IMTC/2002 Instrumentation and Measurement Technology Conference, Anchorage, AK, USA, 21–23 May 2002; Volume 2.
82. Lavagnoli, S.; Paniagua, G.; Tulkens, M.; Steiner, A. High-fidelity rotor gap measurements in a short-duration turbine rig. Mech. Syst. Signal Process. 2012, 27, 590–603. [CrossRef]
83. Sinha, J.K.; Elbhbah, K. A future possibility of vibration based condition monitoring of rotating machines. Mech. Syst. Signal Process. 2013, 34, 231–240. [CrossRef]
84. Abdelrhman, A.M.; Leong, M.S.; Alfatih, S.; Al Obiadi, S. A Review of Vibration Monitoring as a Diagnostic Tool for Turbine Blade Faults. Appl. Mech. Mater. 2012, 229–231, 1459–1463. [CrossRef]
85. Elmaleeh, M.; Saad, N. Development of acoustic emission diagnostic system for condition monitoring of rotating machines. In Proceedings of the IEEE 2nd International Power and Energy Conference (PECon), Johor Bahru, Malaysia, 1–3 December 2008.
86. Lim, M.H.; Leong, M.S. Improved Blade Fault Diagnosis Using Discrete Blade Passing Energy Packet and Rotor Dynamics Wavelet Analysis. In Proceedings of the ASME Turbo Expo 2010: Presented at the 2010 ASME Turbo Expo, Glasgow, UK, 14–18 June 2010; American Society of Mechanical Engineers: New York, NY, USA, 2010; pp. 31–37.
87. Abdelrhman, A.M.; Hee, L.M.; Leong, M.S.; Al-Obaidi, S. Condition Monitoring of Blade in Turbomachinery: A Review. Adv. Mech. Eng. 2015, 6, 210717. [CrossRef]
88. Hee, L.M.; Leong, M.S. Improved blade fault diagnosis using discrete Blade Passing Energy Packet and rotor dynamics wavelet analysis. In Proceedings of the ASME Turbo Expo 2010: Power for Land, Sea and Air, Glasgow, UK, 17–18 June 2010; American Society of Mechanical Engineers: Glasgow, UK, 2010; Volume 3.

89. Südmersen, Ü.; Pietsch, O.; Scheer, C.; Reimche, W.; Bach, F.-W. Condition Monitoring of Steam Turbines by combined vibration and process parameter analysis. In Proceedings of the Conferência Tecnologia de Equipamentos, Salvador, Brazil, 9–12 September 2002.

90. Lebold, M.S.; Maynard, K.; Reichard, K.; Trethewey, M.; Bieryla, D.; Lissenden, C.; Dobbins, D. Using torsional vibration analysis as a synergistic method for crack detection in rotating equipment. In Proceedings of the 2004 IEEE Aerospace Conference, Big Sky, MT, USA, 6–13 March 2004.

91. Leong, M.S. Field Experiences of Gas Turbines Vibrations—A Review and Case Studies. J. Syst. Des.Dyn. 2008, 2, 24–35. [CrossRef]

92. Al-Bedoor, B.O.; Ghouti, L.; Adewusi, S.A.; Al-Nassar, Y.; Abdlsamad, M. Experiments on the extraction of blade vibration signature from the shaft torsional vibration signals. J. Qual. Maint. Eng. 2003, 9, 144–159. [CrossRef]

93. Zhang, Y.; Cruz-Manzo, S.; Latimer, A. (Eds.) Start-Up Vibration Analysis for Novelty Detection on Industrial Gas Turbines. In Proceedings of the 2016 International Symposium on Industrial Electronics (INDEL), Banja Luka, Bosnia Herzegovina, 3–5 November 2016.

94. Loutas, T.H.; Roulias, D.; Pauly, E.; Kostopoulos, V. The combined use of vibration, acoustic emission and oil debris on-line monitoring towards a more effective condition monitoring of rotating machinery. Mech. Syst. Signal Process. 2011, 25, 1339–1352. [CrossRef]

95. Schlagwein, G.; Schaber, U. Non-contact blade vibration measurement analysis using a multi-degree-of-freedom model. Proc. Inst. Mech. Eng. Part A: J. Power Energy 2006, 220, 611–618. [CrossRef]

96. Sendhil, K.S.; Senthil, K.M. Condition monitoring of rotating machinery through vibration analysis. J. Sci. Ind. Res. 2014, 73, 258–261.

97. Yunusa-kaltungo, A.; Sinha, J.K. Effective vibration-based condition monitoring (eVCM) of rotating machines. J. Qual. Maint. Eng. 2017, 23, 279–296. [CrossRef]

98. Matthaiou, I.; Khandelwal, B.; Antoniadou, I. Vibration Monitoring of Gas Turbine Engines: Machine-Learning Approaches and Their Challenges. Front. Built Environ. 2017, 3, 54. [CrossRef]

99. Lifson, A.; Quentin, G.H.; Smalley, A.J.; Knauf, C.L. Assessment of gas turbine vibration monitoring. J. Eng. Gas Turbines Power 1989, 111, 257–263. [CrossRef]

100. Fábry, S.; Češković, M. Aircraft gas turbine engine vibration diagnostics. Mag. Aviat. Dev. 2015, 2017, 7, 24–28. [CrossRef]

101. Madhavan, S.; Jain, R.; Sujatha, C.; Sekhar, A.S. Vibration based damage detection of rotor blades in a gas turbine engine. Eng. Fail. Anal. 2014, 46, 26–39. [CrossRef]

102. Djaidir, B.; Hafaifa, A.; Kouzou, A. Faults detection in gas turbine rotor using vibration analysis under varying conditions. J. Theory Appl. Mech. 2017, 55, 393–406. [CrossRef]

103. Gubran, A.A.; Sinha, J.K. Shaft instantaneous angular speed for blade vibration monitoring. Mech. Syst. Signal Process. 2014, 44, 47–59. [CrossRef]

104. Kumar, S.; Roy, N.; Ganguli, R. Monitoring low cycle fatigue damage in turbine blade using vibration characteristics. Mech. Syst. Signal Process. 2007, 21, 480–501. [CrossRef]

105. Rajagopalan, V.; Behera, A.; Bhattacharya, A.; Prabhu, R.; Badami, V. Estimation of static deflection under operational conditions for blade health monitoring. In Proceedings of the 2012 IEEE Conference on Prognostics and System Health Management (PHM), Beijing, China, 23–25 May 2012.

106. Heath, S.; Imregun, M. A survey of blade tip-timing measurement techniques for turbomachinery vibration. J. Eng. Gas Turbines Power 1998, 120, 784–791. [CrossRef]

107. Hu, Z.; Lin, J.; Chen, Z.-S.; Yang, Y.-M.; Li, X.-J. A Non-Uniformly Under-Sampled Blade Tip-Timing Signal Reconstruction Method for Blade Vibration Monitoring. Sensors 2015, 15, 2419–2437. [CrossRef] [PubMed]

108. Guru, S.S.; Shylaja, S.; Kumar, S.; Murthy, R. Pre-emptive Rotor Blade Damage Identification by Blade Tip Timing Method. J. Eng. Gas Turb. Power 2014, 136, 072503. [CrossRef]

109. Heath, S. A new technique for identifying synchronous resonances using tip-timing. J. Eng. Gas Turb. Power 2000, 122, 219–225. [CrossRef]
110. Pan, M.; Yang, Y.; Guan, F.; Hu, H.; Xu, H. Sparse Representation Based Frequency Detection and Uncertainty Reduction in Blade Tip Timing Measurement for Multi-Mode Blade Vibration Monitoring. *Sensors* 2017, 17, 1745.

111. Wu, F.; Liang, L.; Xing, J.; Wang, L.; Jia, L. Blade tip timing vibration monitoring method based on fiber Bragg grating. *Photonic Sens.* 2014, 4, 188–192. [CrossRef]

112. Rao, R.A.; Dutta, B.K. Vibration analysis for detecting failure of compressor blade. *Eng. Fail. Anal.* 2012, 25, 211–218.

113. Jones, H. Shedding light on vibration. *Mech. Eng.* 1996, 118, 94–98.

114. Kadoya, Y.; Mase, M.; Kaneko, Y.; Umemura, S.; Oda, T.; Johnson, C.M. Noncontact Vibration Measurement Technology of Steam Turbine Blade. *JSME Int. J. Ser. C Dyn. Control Robot. Des. Manuf.* 1995, 38, 486–493. [CrossRef]

115. Satyam, M.; Rao, V.; Devy, C.G. Cepstrum Analysis—An Advanced Technique in Vibration Analysis of Defects in Rotating Machinery. *Def. Sci. J.* 1994, 44, 53–60. [CrossRef]

116. Sinha, J.K.; Hahn, W.; Elbhbah, K.; Tasker, G.; Ullah, I. Vibration investigation for low pressure turbine last stage blade failure in steam turbines of a power plant. In Proceedings of the ASME Turbo Expo 2012, Copenhagen, Denmark, 11–15 June 2012; American Society of Mechanical Engineers: Copenhagen, Denmark, 2012; Volume 7.

117. Sinha, J.K. *Vibration Engineering and Technology of Machinery, Proceedings of VETOMAC X 2014, Held at the University of Manchester, UK, 9–11 September 2014*; Springer: Cham, Switzerland, 2015.

118. Warren, C.; Niezrecki, C.; Avitable, P. Optical non-contacting vibration measurement of rotating turbine blades II. In *Conference Proceedings of the Society for Experimental Mechanics Series*; Springer: New York, NY, USA, 2011; Volume 1.

119. Di Maio, D.; Castellini, P. *Rotating Machinery, Hybrid Test Methods, Vibro-Acoustics & Laser Vibrometry, Volume 8: Proceedings of the 35th IMAC, A Conference and Exposition on Structural Dynamics 2017*; Springer: Cham, Switzerland, 2017.

120. Deraemaeker, A. *New Trends in Vibration Based Structural Health Monitoring*; Springer: Vienna, Austria, 2010.

121. Tagashira, T.; Sugiyama, N.; Matsuda, Y.; Matsuki, M. (Eds.) *Measurement of Blade Tip Clearance Using an Ultrasonic Sensor*; American Institute of Aeronautics and Astronautics: Reston, VA, USA, 1997.

122. Tagashira, T.; Sugiyama, N.; Matsuda, Y.; Matsuki, M. (Eds.) *An Experimental Study of Ultrasonic Tip Clearance Sensor*; American Institute of Aeronautics and Astronautics: Reston, VA, USA, 1998.

123. Giurgiutiu, V. *Structural Health Monitoring with Piezoelectric Wafer Active Sensors*; Elsevier Academic Press: Oxford, UK, 2014.

124. Cheeke, J.D.N. *Fundamentals and Applications of Ultrasonic Waves*; CRC Press: Boca Raton, FL, USA, 2002.

125. Cho, H.; Lissenden, C.J. Structural health monitoring of fatigue crack growth in plate structures with ultrasonic guided waves. *Struct. Health Monit.* 2012, 11, 393–404. [CrossRef]

126. Chan, H.; Masserey, B.; Fromme, P. High frequency guided ultrasonic waves for hidden fatigue crack growth monitoring in multi-layer model aerospace structures. *Smart Mater. Struct.* 2015, 24, 25037. [CrossRef]

127. Masserey, B.; Fromme, P. Analysis of high frequency guided wave scattering at a fastener hole with a view to fatigue crack detection. *Ultrasonics* 2017, 76, 78–86. [CrossRef]

128. Masserey, B.; Fromme, P. In-situ monitoring of fatigue crack growth using high frequency guided waves. *NDT E Int.* 2015, 71, 1–7. [CrossRef]

129. Masserey, B.; Fromme, P. Noncontact monitoring of fatigue crack growth using high frequency guided waves. In *Proceedings of SPIE. Society of Photo-optical Instrumentation Engineers: Bellingham*; Society of Photo-Optical Instrumentation Engineers: Washington, DC, USA, 2014.

130. Masserey, B.; Fromme, P. Fatigue crack growth monitoring using high-frequency guided waves. *Struct. Health Monit.* 2013, 12, 484–493. [CrossRef]

131. Vanlanduit, S.; Guillaume, P.; van der Linden, G. On-line monitoring of fatigue cracks using ultrasonic surface waves. *NDT E Int.* 2013, 36, 601–607. [CrossRef]

132. Masserey, B.; Aebi, L.; Mazza, E. Ultrasonic surface crack characterization on complex geometries using surface waves. *Ultrasonics* 2006, 44 (Suppl. 1), e957–e961. [CrossRef]

133. Lin, B.; Zhang, L.; Dorantes, D.; Li, Y.; Fu, X.; Hu, X. Numerical simulation of surface acoustic wave and detection of surface crack in steel. *Trans. Tianjin Univ.* 2011, 17, 254–258. [CrossRef]
134. Mukhopadhyay, S.C. *New Developments and Applications in Sensing Technology*; Springer: Berlin/Heidelberg, Germany, 2011.

135. Willberg, C.; Duczek, S.; Vivar-Perez, J.M.; Ahmad, Z.A.B. Simulation Methods for Guided Wave-Based Structural Health Monitoring: A Review. *Appl. Mech. Rev.* 2015, 67, 10803. [CrossRef]

136. Rose, J.L.; Soley, L.E. Ultrasonic guided waves for anomaly detection in aircraft components. *Mater. Eval.* 2000, 58, 1080–1086.

137. Achenbach, J.D.; Lin, W.; Keer, L.M. Mathematical modelling of ultrasonic wave scattering by sub-surface cracks. *Ultrasonics* 1986, 24, 207–215. [CrossRef]

138. Rose, J.L.; Pelts, S.P.; Quarry, M.J. A comb transducer model for guided wave NDE. *Ultrasonics* 1998, 36, 163–169. [CrossRef]

139. Fierro, G.P.M. Development of Nonlinear Ultrasonic Techniques for Multidisciplinary Engineering Applications. Ph.D. Thesis, University of Bath, Bath, UK, 2014.

140. Dutta, D.; Sohn, H.; Harries, K.A.; Rizzo, P. A Nonlinear Acoustic Technique for Crack Detection in Metallic Structures. *Struct. Health Monit.* 2009, 8, 251–262. [CrossRef]

141. Friswell, M.I.; Penny, J.E.T. Crack Modeling for Structural Health Monitoring. *Struct. Health Monit.* 2002, 1, 139–148. [CrossRef]

142. Kawashima, K.; Omote, R.; Ito, T.; Fujita, H.; Shima, T. Nonlinear acoustic response through minute surface cracks: FEM simulation and experimentation. *Ultrasonics* 2002, 40, 611–615. [CrossRef]

143. Morris, W.L.; Buck, O.; Inman, R.V. Acoustic harmonic generation due to fatigue damage in high-strength aluminum. *J. Appl. Phys.* 1979, 50, 6737–6741. [CrossRef]

144. Nazarov, V.E.; Sutin, A.M. Nonlinear elastic constants of solids with cracks. *J. Acoust. Soc. Am.* 1997, 102, 3349–3354. [CrossRef]

145. Ostrovsky, L.A. Wave processes in media with strong acoustic nonlinearity. *J. Acoust. Soc. Am.* 1991, 90, 3332–3337. [CrossRef]

146. Barnard, D.J. Monitoring Fatigue Damage Accumulation with Rayleigh Wave Harmonic Generation Measurements. In Proceedings of the AIP Conference Proceedings, Bellingham, WA, USA, 14–19 July 2002; pp. 1393–1400.

147. Berwig, P. Anwendung von Ultraschall-Rayleigh-Wellen zur Rissdetektion an Turbinenschaufeln aus Titanaluminid. Bachelor Thesis, Hochschule für Angewandte Wissenschaften München, Munich, Germany, 2014.

148. Rau, E.; Bamberg, J.; Burchards, J.; Berwig, P.; Voelkl, W. High Resolution Crack Detection on Turbine Blade Roots by the Use of Eddy Current and Ultrasonic Rayleigh Waves. Available online: http://www.ndt.net/article/wendt2016/papers/we3a2.pdf (accessed on 1 March 2018).

149. Rau, E.; Bamberg, J.; Berwig, P. Ultraschallprüfung an Turbinenschaufeln mittels Oberflächenwellen. Available online: https://www.dgzfp.de/Portals/ultraschall2015/BB/8.pdf (accessed on 1 March 2018). (In German)

150. Lane, C. *The Development of a 2D Ultrasonic Array Inspection for Single Crystal Turbine Blades*; Springer: Cham, Switzerland, 2014.

151. Olympus. *Introduction to Phased Array Ultrasonic Technology Applications: Olympus Guideline*; Olympus: Waltham, MA, USA, 2004.

152. Drinkwater, B.W.; Wilcox, P.D. Ultrasonic arrays for non-destructive evaluation: A review. *NDT E Int.* 2006, 39, 525–541. [CrossRef]

153. Chatillon, S.; Cattiaux, G.; Serre, M.; Roy, O. Ultrasonic non-destructive testing of pieces of complex geometry with a flexible phased array transducer. *Ultrasonics* 2000, 38, 131–134. [CrossRef]

154. Harrold, R.T.; Sanjana, Z.N. Acoustic Waveguide Sensing the Condition of Components within Gas Turbines. U.S. Patent 6487909B2, 3 December 2002.

155. Harrold, R.T.; Sanjana, Z.N. Monitoring Thermal Barrier Coating Deterioration via Acoustic Response to Gas Flow, Pressure and Impact. U.S. Patent 7062971B2, 20 June 2006.

156. Willsch, M.; Bosselmann, T.; Theune, N.M. New approaches for the monitoring of gas turbine blades and vanes. In Proceedings of the Sensors 2004 IEEE, Vienna, Austria, 24–27 October 2004.

157. Atkinson, D.; Hayward, G. The generation and detection of longitudinal guided waves in thin fibers using a conical transformer. *IEEE Trans. Ultrason. Ferroelectr. Freq. Control* 2001, 48, 1046–1053. [CrossRef] [PubMed]
158. Wai, Y.; Gao, Y.; Xiao, Z.; Wang, G.; Tian, M.; Liang, H. Ultrasonic Al₂O₃ Ceramic Thermometry in High-Temperature Oxidation Environment. *Sensors* **2016**, *16*, 1905. [CrossRef] [PubMed]

159. Carlson, N.M.; Musick, C.A.; Watkins, A.D.; Tillotson, R.D.; Mullenix, P.D. Evaluation of a novel temperature sensing probe for monitoring and controlling glass temperature in a joule-heated glass melter. In *Proceedings of the Waste management Science Technology in the Ceramic and Nuclear industries (Symbosium I)*, Indianapolis, IN, USA, 25–28 April 1999; American Ceramic Society: Austin, TX, USA, 1999.

160. Periyannan, S.; Rajagopal, P.; Balasubramaniam, K. Ultrasonic bent waveguides approach for distributed temperature measurement. *Ultrasonics* **2017**, *74*, 211–220. [CrossRef]

161. Huang, K.N.; Huang, C.F.; Li, Y.C.; Young, M.S. Temperature measurement system based on ultrasonic phase-shift method. In *Proceedings of the 2003 IEEE EMBS Asian-Pacific Conference on Biomedical Engineering*, Kyoto, Japan, 20–22 October 2003.

162. Tsai, W.-Y.; Chen, H.-C.; Liao, T.-L. An ultrasonic air temperature measurement system with self-correction function for humidity. *Meas. Sci. Technol.* **2005**, *16*, 548–555. [CrossRef]

163. Dyrwal, A.; Meo, M.; Ciampa, F. Nonlinear air-coupled thermosonics for fatigue micro-damage detection and localisation. *NDT E Int.* **2018**, *97*, 59–67. [CrossRef]

164. Zhang, D. Study of Chaotic Ultrasound and Frequency Sweep Excitations in Sonic IR NDE Technology. Ph.D. Thesis, Wayne State University, Detroit, MI, USA, 2014.

165. Chimenti, D.E. Review of air-coupled ultrasonic materials characterization. *Ultrasonics* **2014**, *54*, 1804–1816. [CrossRef]

166. Lim, H.J.; Song, B.; Park, B.; Sohn, H. Noncontact fatigue crack visualization using nonlinear ultrasonic modulation. *NDT E Int.* **2015**, *73*, 8–14. [CrossRef]

167. Thiele, S.; Kim, J.-Y.; Qu, J.; Jacobs, L.J. Air-coupled detection of nonlinear Rayleigh surface waves to assess material nonlinearity. *Ultrasonics* **2014**, *54*, 1470–1475. [CrossRef] [PubMed]

168. Zhou, Z.; Zhang, K.; Zhou, J.; Sun, G.; Wang, J. Application of laser ultrasonic technique for non-contact detection of structural surface-breaking cracks. *Opt. Laser Technol.* **2015**, *73*, 173–178. [CrossRef]

169. Arias, I. Modeling of the Detection of Surface-Breaking Cracks by Laser Ultrasonics. Ph.D. Thesis, Northwestern University, Evanston, IL, USA, 2003.

170. Liu, P.; Nazirah, A.W.; Sohn, H. Numerical simulation of damage detection using laser-generated ultrasound. *Ultrasonics* **2016**, *69*, 248–258. [CrossRef] [PubMed]

171. An, Y.-K.; Kwon, Y.; Sohn, H. Noncontact laser ultrasonic crack detection for plates with additional structural complexities. *Struct. Health Monit.* **2013**, *12*, 522–538. [CrossRef]

172. Pei, C.; Demachi, K.; Fukuchi, T.; Koyama, K.; Uesaka, M. Cracks measurement using fiber-phased array laser ultrasound generation. *J. Appl. Phys.* **2013**, *113*, 163101. [CrossRef]

173. Dhillon, D.; Lee, J. A Fully Non-Contact Ultrasonic Propagation Imaging System for Closed Surface Crack Evaluation. *Exp. Mech.* **2012**, *52*, 1111–1122. [CrossRef]

174. Amini, M.H.; Sinclair, A.N.; Coyle, T.W. High Temperature Ultrasonic Transducer for Real-time Inspection. *Phys. Procedia* **2015**, *70*, 343–347. [CrossRef]

175. Kažys, R.; Voleišis, A.; Voleišienė, B. High temperature ultrasonic transducers. *Ultragarsas* **2008**, *63*, 7–17.

176. Searfoss, C.T.; Tittmann, B.R. High temperature ultrasonic transducer up to 1000 °C using lithium niobate single crystal. *Appl. Phys. Lett.* **2010**, *97*, 232901. [CrossRef]

177. Salas, K.I.; Cesnik, C.E.S. Guided wave excitation by a CLoVER transducer for structural health monitoring: Theory and experiments. *Smart Mater. Struct.* **2009**, *18*, 75005. [CrossRef]

178. Edwards, R.S.; Dixon, S.; Jian, X. Depth gauging of defects using low frequency wideband Rayleigh waves. *Ultrasonics* **2006**, *44*, 93–98. [CrossRef] [PubMed]

179. Jain, X.; Dixon, S.; Guo, N.; Edwards, R. Rayleigh wave interaction with surface-breaking cracks. *J. Appl. Phys.* **2007**, *101*, 064906. [CrossRef]

180. Jain, X.; Dixon, S.; Guo, N.; Edwards, R.S.; Potter, M. Pulsed Rayleigh wave scattered at a surface crack. *Ultrasonics* **2006**, *44* (Suppl. 1), e1131–e1134. [CrossRef] [PubMed]

181. Dixon, S.; Burrows, S.E.; Dutton, B.; Fan, Y. Detection of cracks in metal sheets using pulsed laser generated ultrasound and EMAT detection. *Ultrasonics* **2011**, *51*, 7–16. [CrossRef] [PubMed]

182. Kerr, C.; Ivey, P. Optical pyrometry for gas turbine aeroengines. *Sens. Rev.* **2004**, *24*, 378–386. [CrossRef]
183. Feist, J.P.; Sollazzo, P.Y.; Berthier, S.; Charnley, B.; Wells, J. Precision Temperature Detection Using a Phosphorescence Sensor Coating System on a Rolls-Royce Viper Engine. In Proceedings of the ASME Turbo Expo, Copenhagen, Denmark, 11 June 2012; American Society of Mechanical Engineers: New York, NY, USA, 2012; pp. 917–926.

184. Meher-Homji, C.B. Blading Vibration and Failures in Gas Turbines: Part C—Detection and Troubleshooting. In ASME 1995 International Gas Turbine and Aeroengine Congress and Exposition: Volume 4: Heat Transfer; Electric Power; Industrial and Cogeneration, Houston, TX, USA, 5 June 1995; American Society of Mechanical Engineers: New York, NY, USA, 1995; p. V004T11A016.

185. Rohy, D.A.; Compton, W.A. Radiation Pyrometer for Gas Turbine Blades. SAE Trans. 1972, 81, 720001–720196.

186. Becker, W.J.; Roby, R.J.; O’Brien, W.F.; Bensing, G.K. Dynamic turbine blade temperature measurements. J.Propul. Power 1994, 10, 69–78. [CrossRef]

187. Beynon, T.G.R. (Ed.) Turbine Pyrometry—An Equipment Manufacturer’s View; ASME: New York, NY, USA, 1981.

188. Lucia, M.; Lanfranchi, C. An infrared pyrometry system for monitoring gas turbine blades: Development of a computer model and experimental results. J.Eng. Gas Turb. Power 1994, 116, 172–177. [CrossRef]

189. Sun, J.G.; Tao, N. Multi-spectral temperature measurement method for gas turbine blade. Meas. Sci. Technol. 2016, 27, 105301. [CrossRef]

190. Kerr, C. An overview of the measurement errors associated with gas turbine aeroengine pyrometer systems. Meas. Sci. Technol. 2002, 13, 873–881. [CrossRef]

191. Ketui, D.; Chi, F.; Shan, G. Single wavelength and ratio pyrometry reflection errors in temperature measurement of gas turbine blade. Measurement 2016, 86, 133–140. [CrossRef]

192. Meola, C.; Carlomagno, G.M.; Di Foggia, M.; Natale, O. Infrared thermography to detect residual ceramic in gas turbine blades. Appl. Phys. A 2008, 91, 685–691. [CrossRef]

193. Topal, A.; Catori, C.; Cagan, L.; Uslu, S.; Turan, O.; Piskin, A. One-Dimensional Heat Transfer Analysis and Experimental Investigation of a Gas Turbine Combustor. In Proceedings of the CONV-14: International Symposium on Convective Heat and Mass Transfer, Kusadasi, Turkey, 8–13 June 2014; Begell House: Antalya, Turkey, 2014; p. 15.

194. Feist, J.P.; Sollazzo, P.Y.; Berthier, S.; Charnley, B.; Wells, J. Application of an Industrial Sensor Coating System on a Rolls-Royce Jet Engine for Temperature Detection. In Proceedings of the ASME Turbo Expo, Copenhagen, Denmark, 11–15 June 2012; p. 191.

195. Heyes, A.L.; Rabhiou, A.; Feist, J.P.; Kempf, A.M. Phosphor Based Temperature Indicating Paints. In Proceedings of the ASME Turbo Expo, Copenhagen, Denmark, 11–15 June 2012; p. 927.

196. Pilgrim, C.C.; Sollazzo, P.Y.; Feist, J.P.; Beyrau, F.; Heyes, A.L.; Yañez Gonzalez, A.; Nicholls, J.R. Temperature Sensing inside Thermal Barrier Coatings using Phosphor Thermometry. In Proceedings of the IET & ISA 60th International Instrumentation Symposium, London, UK, 24–26 June 2014; p. 3.1.2.

197. Yañez, G.A.; Pilgrim, C.C.; Feist, J.P.; Sollazzo, P.Y.; Beyrau, F.; Heyes, A.L. On-Line Temperature Measurement Inside a Thermal Barrier Sensor Coating During Engine Operation. J. Turbomach. 2015, 137, 101004. [CrossRef]
204. Khalid, A.H.; Kontis, K.; Behtash, H.-Z. Phosphor thermometry in gas turbines: Consideration factors. *Proc. Inst. Mech. Eng. Part G J. Aerosp. Eng.* **2010**, *224*, 745–755. [CrossRef]

205. Pilgrim, C.C.; González, A.Y.; Saggese, R.; Krewinkel, R.; Blaswich, M.; Feist, J.P.; Orth, U.; Rabs, M.; Frank, D.; Rodriguez, S.A. Surface temperature measurement in an industrial gas turbine using thermal history paints. In Proceedings of the 12th European Conference on Turbomachinery Fluid dynamics & Thermodynamics, Stockholm, Sweden, 3–7 April 2017.

206. Krewinkel, R.; Färber, J.; Lauer, M.; Frank, D.; Orth, U.; Gonzalez, A.Y.; Pilgrim, C.; Feist, J.; Saggese, R.; Berthier, S.; et al. (Eds.) *Validation of Surface Temperature Measurements on a Combustor Liner Under Full-Load Conditions Using a Novel Thermal History Paint*; The American Society of Mechanical Engineers: New York, NY, USA, 2016.

207. Carl, V.; Zenzinger, G. Automatische Rissprüfung mit induktiv angeregter Thermografie. 2005. Available online: https://www.ndt.net/article/dgzfp05/v58.pdf (accessed on 13 July 2018).

208. Netzelmann, U.; Walle, G.; Lugin, S.; Ehlen, A.; Bessert, S.; Valeske, B. Induction thermography: Principle, applications and first steps towards standardization. In Proceedings of the 2015 Asia International Conference on Quantitative InfraRed Thermography, Mahabalipuram, Indian, 6–10 July 2015.

209. Šrajbr, C.; Mehler, S. Induktionsthermografie zur automatisierten Rissprüfung von Flugzeugstrukturen aus Aluminium. 2013. Available online: http://www.dgzfp.de/Portals/thermo2013/BB/vortrag%2012.pdf (accessed on 13 July 2018).

210. Šrajbr, C. Anwendung der Induktionsthermografie im industriellen Umfeld. 2015. Available online: https://www.dgzfp.de/Portals/thermo2015/BB/13.pdf (accessed on 13 July 2018).

211. Bamberg, J.; Erbeck, G.; Zenzinger, G. EddyTherm: Ein Verfahren zur bildgebenden Rißprüfung metallischer Bauteile. 1999. Available online: https://www.ndt.net/article/dgzfp/pdf/zfp68-bamberg.pdf (accessed on 15 July 2018).

212. Vrana, J. Grundlagen und Anwendungen der aktiven Thermographie mit elektromagnetischer Anregung: Induktions- und Konduktionsthermographie. Ph.D. Thesis, Universität des Saarlandes, Saarbrücken, Germany, 2008. (In German)

213. Spießberger, C.; Dillenz, A.; Zwescher, T. Rissprüfung mittels aktiver Thermografieverfahren: Induktions- und Ultraschallanregung im Vergleich. 2011. Available online: http://www.dgzfp.de/Portals/thermo2011/BB/v07.pdf (accessed on 13 July 2018). (In German)

214. Al-Obaidi, S.M.A.; Leong, M.S.; Hamzah, R.R.; Abdelrhman, A.M. A Review of Acoustic Emission Technique for Machinery Condition Monitoring: Defects Detection & Diagnostic. *Appl. Mech. Mater.* **2012**, *229–231*, 1476–1480.

215. Nashed, M.S.; Steel, J.A.; Reuben, R.L. The use of acoustic emission for the condition assessment of gas turbines: Acoustic emission generation from normal running. *Proc. Inst. Mech. Eng. Part E J. Process Mech. Eng.* **2014**, *228*, 286–308. [CrossRef]

216. Mba, D.; Hall, I.D. The transmission of acoustic emission across large-scale turbine rotors. *NDT E Int.* **2002**, *35*, 529–539. [CrossRef]

217. Mba, D. Development of Acoustic Emission Technology for Condition Monitoring and Diagnosis of Rotating Machines: Bearings, Pumps, Gearboxes, Engines, and Rotating Structures. *Shock Vib. Dig.* **2006**, *38*, 3–16. [CrossRef]

218. Leahy, M.; Mba, D.; Cooper, P.; Montgomery, A.; Owen, D. Experimental investigation into the capabilities of acoustic emission for the detection of shaft-to-seal rubbing in large power generation turbines: A case study. *Proc. Inst. Mech. Eng. Part J J. Eng. Tribol.* **2006**, *220*, 607–615. [CrossRef]

219. Mba, D.; Cooke, A.; Roby, D.; Hewitt, G. Detection of shaft-seal rubbing in large-scale power generation turbines with acoustic emissions. Case study. *Proc. Inst. Mech. Eng. Part A J. Power Energy* **2004**, *218*, 71–81. [CrossRef]

220. Mathioudakis, K.; Papathanasiou, A.; Loukis, E.; Papaiiou, K. Fast response wall pressure measurement as a means of gas turbine blade fault identification. *J. Eng. Gas Turb. Power* **1991**, *113*, 269–275. [CrossRef]

221. Mathioudakis, K.; Loukis, E.; Papaiiou, K. Casing Vibration and Gas Turbine Operating Conditions. *J. Eng. Gas Turb. Power* **1990**, *112*, 478–485. [CrossRef]

222. Forbes, G.L.; Randall, R.B. Estimation of turbine blade natural frequencies from casing pressure and vibration measurements. *Mech. Syst. Signal Process.* **2013**, *36*, 549–561. [CrossRef]
223. Forbes, G.L. Non-contact gas turbine blade vibration monitoring using internal pressure and casing response measurements. Ph.D. Thesis, The University of New South Wales, School of Mechanical and Manufacturing Engineering, Sydney, Australia, 2010.

224. Forbes, G.L.; Randall, R.B. *Simulation of Gas Turbine Blade Vibration Measurement from Unsteady Casing Wall Pressure*; Australian Acoustical Society: Adelaide, Australia, 2009.

225. Forbes, G.L.; Randall, R.B. *Simulated Gas Turbine Casing Response to Rotor Blade Pressure Excitation*; Australian Acoustical Society: Adelaide, Australia, 2007.

226. Forbes, G.L.; Randall, R.B. Detection of a Blade Fault from Simulated Gas Turbine Casing Response Measurements; DEStech Publications: Lancaster, PA, USA, 2008.

227. Longtin, J.; Sampath, S.; Tankiewicz, S.; Gambino, R.J.; Greenlaw, R.J. Sensors for harsh environments by direct-write thermal spray. *IEEE Sens. J.* 2004, 4, 118–121. [CrossRef]

228. Schonberg, T.; Ruusuvuori, K.; Christensen, B.H.; Boivie, K.; Berild, J.P.; Ronkainen, H.; Gellein, L.T. Surface embedded temperature sensor on tool part for real time injection moulding process monitoring. In Proceedings of the 2013 Transducers & Eurosensors XXVII: The 17th International Conference on Solid-State Sensors, Actuators and Microsystems (TRANSUDCERS & EUROSENSORS XXVII), Barcelona, Spain, 16–20 June 2013; pp. 116–119.

229. Zhang, Y.; Mack, D.; Mauer, G.; Vaßen, R. Laser Cladding of Embedded Sensors for Thermal Barrier Coating Applications. *Coatings* 2018, 8, 176. [CrossRef]

230. Li, J.; Longtin, J.P.; Tankiewicz, S.; Gouldstone, A.; Sampath, S. Interdigital capacitive strain gauges fabricated by direct-write thermal spray and ultrafast laser micromachining. *Sens. Actuators A Phys.* 2007, 133, 1–8. [CrossRef]

231. Li, P.; Wen, Y.; Zheng, M.; Zhou, Z. Self-powered Piezoelectric Wireless Micro-sensors by Absorbing Electromagnetic Energy. In Proceedings of the 2004 IEEE Sensors, Vienna, Austria, 24–27 October 2004; Vienna University of Technology: Vienna, Austria, 2004.

232. Xia, R.; Farm, C.; Choi, W.; Kim, S.-G. Self-Powered Wireless Sensor System using MEMS Piezoelectric Micro Power Generator. In Proceedings of the 2006 IEEE Sensors, Daegu, Korea, 22–25 October 2006.

233. Chen, Q.; Tong, T.; Longtin, J.P.; Tankiewicz, S.; Sampath, S.; Gambino, R.J. Novel Sensor Fabrication Using Direct-Write Thermal Spray and Precision Laser Micromachining. *J. Manuf. Sci. Eng.* 2004, 126, 830. [CrossRef]

234. Hon, K.K.B.; Li, L.; Hutchings, I.M. Direct writing technology—Advances and developments. *CIRP Ann.* 2008, 57, 601–620. [CrossRef]

235. Pique, A. Direct-Write Technologies for Rapid Prototyping Applications Sensors, Electronics, and Integrated Power Sources; San Diego Academic Press: San Diego, CA, USA, 2002.

236. LG Tech-Link Global. Uniform Crystal Temperature Sensor Technology and Support Services. Available online: http://www.lgtechlink.com (accessed on 15 July 2018).

237. Sheth, A.; Thomas, A. Uniform Crystal Temperature Sensor (UCTS) Application to Validation, Verification and Technical Comparison Processes; UC DRC: Lynn, MA, USA, 2015.

238. DeVoe, J.; Thomas, A.; DeVoe, R.; Ginzbursky, E. Gas temperature measurement in engine conditions using uniform crystal temperature sensors (UCTS): GT2018-76890. In *ASME Turbo Expo 2018: Turbomachinery Technical Conference and Exposition*; American Society of Mechanical Engineers: New York, NY, USA, 2018; p. V005BT13A016.

239. DeVoe, J.; Odom, S.; Ginzbursky, L. Uniform Crystal Temperature Sensor Accuracy Under Transient Conditions: GT2012-68197. In *ASME Turbo Expo 2012*; American Society of Mechanical Engineers: New York, NY, USA, 2012; pp. 79–87.

240. Kubiak, J.A.; Franco, J.M.; Carnero, A.; Rothhirsch, L.A.; Aguirre, R.J. A Diagnosis of Failure of a Compressor Blade. In *Volume 5: Manufacturing Materials and Metallurgy; Ceramics; Structures and Dynamics; Controls, Diagnostics and Instrumentation; Process Industries; General*, Anaheim, CA, USA, 31 May 1987; American Society of Mechanical Engineers: New York, NY, USA, 1987; p. V05BT13A003.

241. Lattime, S.B.; Steinetz, B.M. Turbine Engine Clearance Control Systems: Current Practices and Future Directions. In Proceedings of the 38th AIAA/ASME/SAE/ASEE Joint Propulsion Conference & Exhibit, Indianapolis, IN, USA, 7–10 July 2002.
242. Salar, A.; Hosseini, S.M.; Zangmolk, B.R.; Sedigh, A.K. Improving Model-Based Gas Turbine Fault Diagnosis Using Multi-Operating Point Method. In Proceedings of the 2010 Fourth UKSim European Symposium on Computer Modeling and Simulation (EMS), Pisa, Italy, 17–19 November 2010.

243. Diallo, O.; Mavris, D. A data analytics approach to failure precursor detection of gas turbine. In ASME 2011 Turbo Expo: Turbine Technical Conference and Exposition; American Society of Mechanical Engineers: New York, NY, USA, 2011; Volume 3.

244. Hanachi, H.; Liu, J.; Chen, Y.; Koul, A. A Physics-Based Modeling Approach for Performance Monitoring in Gas Turbine Engines. IEEE Trans. Reliab. 2015, 64, 197–205. [CrossRef]

245. Dewallef, P.; Borguet, S. A Methodology to Improve the Robustness of Gas Turbine Engine Performance Monitoring Against Sensor Faults. J. Eng. Gas Turb. Power 2013, 135, 51601. [CrossRef]

246. Borguet, S.; Léonard, O. Comparison of adaptive filters for gas turbine performance monitoring. J. Comput. Appl. Math. 2010, 234, 2202–2212. [CrossRef]

247. Tahan, M.; Tsoutsanis, E.; Muhammad, M.; Abdul, K.Z.A. Performance-based health monitoring, diagnostics and prognostics for condition-based maintenance of gas turbines: A review. Appl. Energy 2017, 198, 122–144. [CrossRef]

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