Development of small-scale downdraft gasifiers for biomass gasification

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Abstract. Biomass gasification mainly involves a process whereby agricultural residues or biomass are subjected to partial combustion for the biomass to undergo pyrolysis and reduction, thereby releasing its gaseous component such as hydrogen, carbon-dioxide and methane. Past studies have laid emphasis on the need to study the effect of moisture content and biomass types on the rate of gasification and gasifier efficiency. This study focused on the development of a downdraft gasifier for production of syngas using agricultural wastes as raw materials. The design of the Imbert type downdraft gasifier was based on specific gasification rate, called hearth load \(G_h\), nozzle air blast velocities, throat inclination angle, air inlet diameter and size of reduction zone. The developed gasifier consisted of a reactor, cyclone and filtration unit of 0.006, 0.016, and 0.006 m\(^3\) capacities respectively. The testing and evaluation of the designed gasifier showed adequate capacity for biomass treatment by pyrolysis. The fabricated gasifier also showed high efficiency in the pyrolysis of the selected biomasses and the product yields are appreciable. The development and use of such gasifier especially for the treatment of other agricultural wastes is hereby solicited.

1. Introduction

In recent years, biomass energy has gained a lot of attention due to its great potential of contributing to sustainable energy development [1-7]. The continuous recognition of biomass as one of the promising renewable sources of energy results from the gradual depletion of conventional fossil fuels [8,9]. Gasification, a thermo-chemical partial oxidation process, has superb capacity for biomass power recovery, as it is able to convert biomass into syngas commonly known as combustible gases, which can be further converted to electricity and biochar that has a large carbon sequestration capability [10]. Also, biomass gasification is one of the key technologies to convert waste biomass or carbonaceous materials to syngas efficiently and economically. Syngas is the source for high-value chemicals (such as methanol, DME) [11] and liquid carbohydrates [1]. By so doing, the issue of environmental pollution which is ubiquitous especially in the Sub-Saharan Africa, resulting from waste can be combated [12-15]. According to You et al. [16], balancing syngas and biochar production is an economically and environmentally friendly (greenhouse gas mitigation) gasification system can be developed.

Gasifier (a gasification device) design is a crucial step to optimize the economics and green potential of a gasification system to reach cleaner energy production. Gasifiers are the main devices used for gasification; they perform the conversion process of biomass to syngas, consisting mainly of CO, CO\(_2\),
H₂, N₂, ashes, tar and small particulates [17]. Past researches have recorded three principal types of gasifiers, which includes fluidized bed gasifiers [18], entrained flow gasifiers [19] and fixed bed gasifiers [20]. Among the various technologies that can be used for biomass combustion, fluidized beds are gradually emerging as the best due to their flexibility with respect to the type of fuel and high efficiency. One major problem of fluidized beds is defluidization due to bed agglomeration; high amounts of oxygen bypass the reactor bed due to the low level of oxygen dissemination from the gas bubbles, which reduces the efficiency of the gasifier. However, successful solution has been reported for other biomass feedstock [21]. In an entrained flow gasifiers, syngas with a lower tar content are been produced and about 20% more oxygen is required [22]. The oldest and common reactors employed to synthesize syngas are the fixed bed gasifiers. Higher than 10MW (that is, large scale) fixed bed gasifiers are losing the interests of industrial units due to scale-up issues [23], however, small scale fixed bed gasifiers with high thermal efficiency are in use for decentralized energy generation and for thermal applications in many industries [24]. Fixed bed gasifiers can be further divided into updraft [25] or downdraft [26] based on the gas flow direction. Due to its simplicity and tractability, the updraft gasifier is one of the oldest types of gasifier. It is termed “updraft” because the biomass is fed from the top of the gasifier, while air is supplied at the bottom of the gasifier. On the other hand, air move in the downward direction in the lower section of the downdraft gasifier unit.

In downdraft bed gasifiers, tar content is much lower compared to that in an updraft gasifier, but the syngas has less calorific value [22]. Downdraft gasifier is more suitable for small-scale applications [27]. Downdraft gasifiers are the most widely deployed gasification technique in small-scale applications and it was reported that around 75% of gasifier manufacturers in Europe produce downdraft gasifiers [28]. A review of 50 gasifier manufacturers in Europe, United State and Canada showed that 75% of the designs were downdraft fixed beds while fluidized beds, updraft fixed beds and the other designs were 20%, 2.5%, 2.5%, respectively [29]. As far as Nigeria is concerned and because of its vast agro-forestry base, fuels of bio-origin can be considered to be ideal alternative renewable fuels to run the internal combustion engines.

The bio-origin fuels could be modified to bring their properties comparable to fossil fuels. A downdraft gasifier usually consists of four sequential zones, i.e. drying zone, pyrolysis zone, combustion zone, and reduction zone, respectively, among which the reduction zone is responsible for syngas production [22].

2. Materials and Methods

2.1. Design Concept

The concept of this design was fashioned after Imbert type of gasifier design mainly for downdraft gasifiers because of its wide acceptability and use among design engineers. The design of an Imbert type downdraft gasifier is based on specific gasification rate, called hearth load \(G_h\), nozzle air blast velocities, throat inclination angle, air inlet diameter and size of reduction zone. The developed gasifier consisted of a reactor (for gasification), cyclone (for cooling) and filtration (for cleaning) unit of 0.006, 0.016, and 0.006 m³ capacities respectively. This is defined as the ratio of Volume of gas to cross-sectional Area of the throat and it is given by equation 1.

\[
G_H = \frac{V}{A}
\]

Where:

\(V\) = Volume of producer gas,

\(A\) = Cross-sectional area of the throat

The gas volume is calculated at normal pressure and temperature. Its values range from 0.1 to 0.9 m³/ cm². Figure 1b depicts a simple system of Imbert type of gasifier.
2.2. The Gasifier Components

The conceptual gasifier design is made of three main parts which are: reactor chamber, cooling unit and the filtering unit.

2.3. Reactor Section

The reactor chamber is the main unit where gasification process occurs. This is a chamber which comprises of inner and outer cylinders. The inner cylinder is usually made of stainless steel so as to withstand the high temperature it will be subjected to. The outer cylinder can be made of mild steel and it serves as housing for the inner chamber to minimize the heat loss or gain into the surroundings. The biomass is been fed into the reactor chamber where processes like drying, pyrolysis, oxidation and reduction take place before the eventual liberation of the syngas. The various design parameters of the reactor chamber are the diameter of the reactor, height of the reactor, diameter of hearth and throat, air velocity, diameter of gas outlet, length of the pipe for the gas outlet, surface Area of air inlet and throat angle.

2.4. Cooling Section

This unit consists of a cyclone which cools the temperature of the gas as it enters from the reactor chamber. This is the section where the various impurities like tar will be collected and separated from the syngas. It is usually in form of cyclone to allow a free fall of the impurities into the tar collection chamber at the conical base. Also, a cooling fan was attached to the filtering section. The various design parameters for the cyclone unit are; the height of the cyclone, diameter of the cyclone, taper angle at the bottom, diameter of the base, gas inlet diameter and gas outlet diameter

2.5. Filtering Unit

This section is made of cylinder chamber filled with the filtering medium in order to remove other impurities inform of particles from the gases before final use. The various parameters of the cyclone are the height of the filter, diameter of the filter, gas inlet diameter and gas outlet diameter. The composition of various types of gasifiers available during the World War II indicated that the maximum specific gasification rate (or hearth load) is of the order 0.09, 0.03 and 0.9 Nm³/hcm² for no throat, single throat and double throat gasifier respectively. This information is useful in order to have an idea of the range of gasification rate to use in this design.

2.6. Design Criteria for Gasifier Reactor

The following recommendations were given by FAO [30] in 1986 as the guidelines to the design of down-draft gasifier as shown in table 1.

i. Nozzle air-blast velocities should be of the order of 22- 33m/s
ii. Throat inclination angle should be about 45° - 60°.
iii. Hearth diameter at the air inlet should be 0.01m and 0.02m larger than the smallest cross-section (throat) in the case of single and double –throat design respectively.
iv. Reduction zone height should be more than 0.02 m
v. Air inlet nozzle plane should be located more than 0.01m above the throat section.

| Parameters                | Value (m) |
|---------------------------|-----------|
| Height of reactor (H)     | 0.700     |
### 2.7. Gasifier Design Calculations

In order to have a proper design, the idea of the capacity of the engine the gasifier will be able to carry is needed, and in this case, 5.0 KVA engine was considered. A typical 5.0 KVA engine have the following engine parameters which are useful for the design of the gasifier reactor.

- **Rated Output** = 5.0 KW (13 HP)
- **Piston Diameter** = 88.00 mm
- **Stroke length** = 64.0 mm
- **Number of cylinder** = 1 (4 stroke cycle)
- **Engine RPM** = 3,600
- **Operation hours** = 10 hours
- **Volumetric Efficiency** = 85%

### 2.8. Reactor Design Steps

The first step is to find out the required gas production rate.

Since Engine Swept Volume \((V_s)\) equals,

\[
V_s = \frac{1}{2} \times \text{RPM} \times N \times A \times S
\]  

(2)

Where \(N\) = number of cylinder

\(A\) = Area \((m^2)\)

\(S\) = Stroke \((m)\)

Substituting the values given in the 5KVA engine parameters into equation (2) as stated above:

\[
V_s = \frac{1}{2} \times 3600 \times 1 \times 0.064 \times \pi d^2
\]

\(V_s = 42.05 \text{ m}^3/h\)

Using stoichiometric air-fuel (gas) ratio 1.1: 1.0, for the intake volume of gas \((V_g)\), the air-fuel intake will be 2.1.

Equation 9 stated below shows the relationship between the Swept volume, Volume of gas and volumetric efficiency of the engine.

\[
V_g = f \times \frac{V_s}{2.1}
\]  

(3)
Where $V_s$ = Swept Volume of the engine = 42.05 m$^3$/h  
$V_g$ = Volume of the gas (m$^3$/h)  
f = volumetric efficiency = 85 % = 0.85.

Substituting into the values into equation (3) above, the value of $V_g$ is:

$V_g = 17.0$ m$^3$/h

For maximum hearth load $G_{h}$ of 0.9 m$^3$/cm$^2$, the throat Area $A_t$ can be calculated from equation 4

$$A_t = \frac{V_g}{G_{h} \text{max}}$$  \hspace{1cm} (4)

Where $A_t$ = Area of the reactor throat (m$^2$)  
$V_g$ = Volume of the gas (m$^3$/h) = 17.0 m$^3$/h  
$G_{h}$ = Gasification rate = 0.9 m$^3$/cm$^2$

By substituting the values above into equation (3)

$A_t = 18.90$ cm$^2 = 0.00189$ m$^2$

Choosing Hearth Load $G_{h}$ = 0.7 Nm$^3$/hcm$^2$  (As recommended by Imbert)

Substituting into equation (3) above

$A_t = 24.29$ cm$^2 = 0.0024$ m$^2$

For this design, hearth load of 0.7 will be used since is closer to the maximum hearth load (0.9) and it was also recommended by Imbert in his design. Thus, the throat diameter $d_t$ for the circular cross-sectional can be obtained from equation (5).

$$A_t \pi \frac{d_t^2}{4}$$  \hspace{1cm} (5)

Where $D = d_t$ is the diameter of the throat

Where $A_t = 0.0024$ m$^2$

d$_t$ = 0.0556 m

Therefore, the diameter of the throat will be taken approximately 0.056m.

Using the graph of $h$/ $d_t$ as against $d_t$ as prepared by Swedish Academy of Engineering Sciences. As shown in figure 1d below:

At $d_t = 6$cm = 0.060m, on the positive x-axis, on the positive y-axis

Where $h$ = height of nozzle above the throat.

d$_i$ = diameter of the throat
\[
\frac{h}{dt} = 1.3
\]  
(6)

\( h = 78 \, \text{mm} = 0.078 \, \text{m} \)

The diameter of firebox \( d_f \) and the diameter of nozzle top ring \( d_n \) can be determined using the graph of \( \frac{h}{d_f} \) against \( d_f \). Also, graph of \( \frac{h}{d_n} \) against \( d_n \), as obtained from FAO [30] in 1986 shown in Figure 1b.

**Figure 1a.** Height of nozzle plane above throat for various diameters, FAO (1986)
Figure 1b. Nozzle ring diameter as a function of throat diameter Reproduced with permission from FAO
From Figure 1b, the values are 3.5 and 2.5 respectively.

At $d_t = 60\text{mm} = 0.06\text{m}$

And $\frac{dn}{dt} = 3.5$ \hspace{1cm} (7)

Where $d_n = \text{diameter of the inner chamber}, \hspace{0.5cm} d_f = \text{diameter of the outer chamber}$

Substituting the value of the throat diameter then, at $d_t = 0.06\text{m}$:

$d_f = 0.21\text{m}$

$d_n = 0.15\text{ m}$

Assuming that a unit of nozzles is to be used for supporting the required amount of air for gasification such that the ratio of $100 \left( \frac{A_m}{A_t} \right)$ against $d_t$ as shown in the gasifier graph (Figure 6) prepared by Swedish Academy of Engineering Sciences and reproduced by FAO [31] in 1960 at $d_t = 6.0\text{ cm}$.

Where $A_m = \text{Area of ignition port}$.

$dt = 60\text{mm} = 6.0\text{cm}$,

$A_t = \text{Area of the throat}$

$$100 \left( \frac{A_m}{A_t} \right) = 6.0$$ \hspace{1cm} (8)

By substituting the value of $A_t = 24.29\text{cm}^2$ into equation (8)
Am = 6.0 x 24.29

Am = 145.74 cm²

Diameter of the ignition port Air-inlet
dm = 13.6 cm = 0.136 m

Also, the equivalent Air-blast velocity can be determined from Figure 1c as Um = 23 m/s.

Height of outer cylinder can be determined by adding allowance to the height of the inner cylinder. Since the height of the inner cylinder is 0.50m, then 0.10m was added at the bottom and top of the inner cylinder and the height of the outer cylinder will be 0.70m.

2.9. Pipe Fittings

Diameter of the pipe in the design will be taken as 2 inches = 5cm. The reason for chosen 5cm diameter pipe is to give room for the gas to move freely and also give room for the gas to cool down and release some of its heat to the pipe.

2.10. Cyclone Design

As shown in table 2, the cyclone design is to produce some level of gas cooling and cleaning. As the gas leaves the reactor, the temperature is appreciably high and its temperature reduces as it travels through the pipe into the cyclone chamber. The temperature further reduces inside the chamber as the gas molecules spins. The tar spins and fall under gravity as both the gas and particles spin in the cyclone chamber.

| Parameter | Value |
|-----------|-------|
| Length of cyclone body (m) | 0.50 |
| Length of cone (m) | 0.20 |
| Diameter of cyclone (m) | 0.01 |
| Width of inlet duct (m) | 0.05 |
| Diameter of cyclone base | 0.04 |
| Number of spins (Ne) s | 12 |
| Gas residence time (T) s | 0.164 |
| Terminal Velocity ( Vt) m/s | 0.305 |
| Tapered angle cyclone (degrees) | 8° |

2.11. Design Parameters

A typical cyclone is shown in the figure 1d with all the parameters as shown. The design parameters to be considered in the cyclone are:

The general guidelines for the design of cyclone as related to the various geometry dimensions are as specified by Gasifier Design Handbook [30] are as listed below:

i. H < S
ii. W < (D - De)/2
iii. Lb + Lc > 3D
iv. 7° ≤ α < 8°
v. \( 0.4 \leq 0.5 \)
vii. \( = 1 \)

The values for the cyclone parameters are as stated below:
- Length of cyclone body \( L_b = 0.50 \text{m} \) (taken from the height of the inner cylinder in the reactor design).
- Length of cyclone base \( L_c = 0.20 \text{m} \) (taken from the reactor design, base of the reactor).
- Width of inlet duct \( w = 0.05 \text{m} \)
- Cyclone Diameter \( D = 0.10 \text{m} \)
- Height of inlet duct \( H = 0.0508 \text{m} \)
- Diameter of the cyclone base = 0.0508m
- Taper angle at the bottom = \( \alpha \)

\[ \text{Figure 1d. Triangular representation of the design parameters} \]

The triangle ABC as shown above depicts the base of the reactor and the falling angle for the biomass needs to be calculated. The reactor needs a conical base in order to allow the burnt biomass to find its way down to the collecting chamber at the base of the reactor as ash. From Triangle ABC, we can determine the angle of fall of the reactor base:

- Since opposite BC = 20 cm
- Adjacent AB = 3 cm

Then angle BAC can be calculated as:

\[ \tan \Theta = \frac{20}{3} \]
\[ \tan \Theta = 6.67 \]
\[ \Theta = \arctan 6.67 \]
\[ \Theta = 81.47^\circ, \text{ so } \alpha = 8^\circ \]
Figure 1e. A typical cyclone for gas cooling and cleaning with the following:

i. $H_c =$ Height of cyclone
ii. $D_c =$ Diameter of cyclone
iii. $\alpha =$ Taper angle at the bottom
iv. $D_{cb} =$ Diameter of cyclone base
v. $G_{id} =$ Gas inlet diameter
vi. $G_{ii} =$ Gas outlet diameter

2.12. \textit{The Collection Efficiency}

Gas spins through a number of revolutions $N_e$, given by equation 14 below (Cooper and Alley 1986)

\begin{equation}
N_e = \frac{1}{H} \left( \frac{L_b + L_c}{2} \right)
\end{equation}

Where $L_b =$ Length of cyclone body (m)
$L_c =$ Length of cyclone base (m)
$H =$ Height of inlet duct (m)

Substituting values of $L_b$, $L_c$ and $H$

$N_e = 12$ turns.
Consequently, by design the gas enters the cyclone and spins 12 times before reaching the base of the cyclone. The gas molecules falls freely under the influence of gravity, the thicker impurities which happen to be the impurities or the char are collected at the base of the cyclone.

2.13. Gas Residence Time

Gas residence time is the average amount of time that a particle spends in a particular system in this case the cyclone. It can be a probability distribution function that describes the amount of time a fluid element could spend inside a chemical reactor.

The residence Time is given by equation 10 stated below;

$$T = \frac{\pi D Ne}{V_i}$$

Where
- $T$ = the residence time
- $D$ = the cyclone diameter = 10cm
- $Ne$ = the number of spins = 12
- $V_i$ = inlet velocity of the gas = 23 m/s

Substituting the values into the equation 10, then the residence time $T$

$$T = \frac{3.142 \times 0.1 \times 12}{23}$$

$T = 0.164s$

2.14. Terminal Velocity

The terminal velocity $V_t$ of the particles in a radical direction that will just allow a particle initially at distance $W$ away from the wall to the collected air time is given by equation 11:

$$V_t = \frac{W}{Dt}$$

Where, $V_t$ is the terminal velocity i.e. the velocity at which drag force equals the resisting force, $W$ is the width of the inlet duct

Substituting the values into equation 16, then $V_t = 0.305$ m/s

2.15. Filtering Section

The filtering section shown in Figure 1e is to remove the impurities that come with the syn-gas as its leaves the cyclone section. The height of the filtering section and the diameter can be chosen conveniently considering the size of the reactor. At least the dimension of the filtering section to be chosen must be relatively close to that of reactor. So, that account for the values chosen and stated below:

i. Height of filtering cylinder = 0.50 m
ii. Diameter of filter = 0.20 m
2.16. Materials Selection
The gasifier reactor section will be subjected to great temperature, so a stainless steel is needed in order to withstand the temperature of the gasification reaction. Also, mild steel is needed for the outer cylinder. The various materials selected for the construction of the gasifiers various components are as listed in Table 3. The various instrument needed for taken measurement during and after gasifier construction are as listed in table 4.

| S/N | Component                          | Material selected     |
|-----|-----------------------------------|-----------------------|
| 1   | Gasifier Reactor (Inner Cylinder) | Stainless steel       |
| 2   | Gasifier Reactor (Outer Cylinder) | Mild steel            |
| 3   | Gas Outlet Pipe                   | Galvanized pipe       |
| 4   | Cyclone Section                   | Mild Steel            |
| 5   | Cyclone gas inlet and outlet      | Galvanized pipe       |
| 6   | Filtering Section                 | Mild Steel            |
| 7   | Gasifier Stand                    | Angle Iron            |
| 8   | Gas Passage                       | Galvanized pipe       |

| S/N | Instrument   | Function                                           |
|-----|--------------|----------------------------------------------------|
| 1   | Thermocouple | Measures oxidation temperature                     |
| 2   | Thermometer  | Measures Ambient temperature                       |
| 3   | Weighing Scale | To measure the amount of biomass and ash content   |
| 4   | Drying Oven  | To determine the moisture content of the biomass   |
| 5   | Stop Watch   | Measures time taken for complete gasification      |
2.17. Fabrication and Assembly
Fabrication of the various sections and components was done at the Central Workshop of the College of Engineering, Federal University of Agriculture, Abeokuta (FUNAAB) and those fabricated components were assembled together (Figures 2a and b).

2.18. Feedstock
The feedstock used in this study are Saw dust, Wood chips, and Bean chaff. They were selected and experimented on the basis of availability and suitability for renewable energy generation since they are all lignocelluloses [32].

2.19. Biomass preparation
Each biomass was sundried till constant weight was achieved. Wood chips and Bean chaff were then crushed in a hammer mill so that all three biomasses achieved mean diameter of 10 to 30 μm. Mean bulk densities (pb) of all biomasses ranged between 573 and 582 kg m⁻³ while their unit densities (pu) ranged between 1331-1420 kg m⁻³ (EN 15103 - 2009). The lower heating value (LHV) for the biomasses ranged between 11.84 and 12.21 MJ kg⁻¹ for the raw sample while it was between 12.99 and 13.31 MJ kg⁻¹ for the dry sample respectively. Also, the higher heating values (HHV) ranged between 14.42 and 14.53 MJ kg⁻¹ for the raw biomasses while it was between 14.94 and 15.31 MJ kg⁻¹ for dry biomasses (ASTM D 5865-13). In all, the first deformation, softening and hemisphere temperatures (ASTM D 1857/D1857) were 1025, 1125, and 1317 °C, respectively.
2.20. Biomass proximate analyses

As shown in tables 5a, the concentrations of total and volatile solids and ash were determined using the standard method for analysis of water and wastewaters [33]. The same method was used for the determination of Chemical Oxygen Demand (COD) in all the samples. The concentrations of parameters including carbon, phosphorus, phosphates, sulphates, potassium, sodium, magnesium, calcium, nitrates, ammonium, iron, copper, zinc, aluminium and manganese were determined with the aid of an inductively coupled plasma mass spectrometry. For pH measurement, 1 g of ground sample of each biomass was dissolved in 20 cm$^3$ of water and then incubated at 25 °C for 30 min after which the supernatant was used for the measurement with the aid of the HI 2211 pH/ORP Meter electrode (Hanna Instruments, Germany). For total organic carbon (TOC) determination, a SSM-5000A TOC analyzer (Shimadzu, Japan) was used while the Kjeldahl method was employed in analyzing the content of total Kjeldahl nitrogen. Volatile fatty acids (VFAs) concentrations were determined using standard method as described by Monlau et al. [34].

| Table 5a. Proximate composition of used lignocelluloses |
|-----------------------------------------------|-----|-----|-----|
| Parameter                                   | Saw dust | Wood chips | Bean chaff |
| Ash Content (%)                             | 3.6±0.01 | 4.2±0.01 | 3.6±0.03 |
| Moisture Content (%)                        | 45.7±1.01 | 52.2±1.01 | 58.3±3.05 |
| Total Carbon (g/kg TS)                      | 241.5±4.01 | 246.2±5.02 | 319.5±4.01 |
| Total Nitrogen (g/kg TS)                    | 9.3±0.02 | 10.2±0.02 | 45.4±0.05 |
| Uronic acids (% VS)                         | 1.0±0.01 | 1.2±1.10 | 1.47±0.02 |
| Soluble sugars (% VS)                       | 3.6±0.01 | 4.1±1.02 | 1.6±0.01 |
| Phenols (mg L$^{-1}$)                       | - | - | - |
Total Phosphorus (g/kg TS) 3.4±0.01 2.1±0.11 4.3±0.01
Potassium (g/kg TS) 4.6±0.01 5.2±0.01 7.1±0.01
Phosphate (g/g TS) 1.2±0.01 2.01±0.01 1.9±0.01
Sulphate (g/kg TS) 40.4±1.00 46.2±2.00 36.5±0.01
Calcium (g/kg TS) 167.1±2.02 181.4±0.42 81.1±4.01
Magnesium (g/kg TS) 45.3±1.02 50.6±0.02 65.3±2.01
Iron (g/kg TS) 0.2±0.01 0.2±0.01 0.6±0.01
Zinc (g/kg TS) 1.2±0.01 1.1±0.01 2.1±0.01
Aluminium (g/kg TS) 26.7±0.01 25.3±0.02 35.4±1.01

| Parameter          | Saw dust | Wood chips | Bean chaff |
|--------------------|----------|------------|------------|
| pH                 | 7.66±0.01| 7.69±0.10  | 7.63±0.10  |
| Total solids (%) m.m⁻¹ | 83.5±2.01| 87.3±4.01  | 64.2±2.12  |
| Volatile solids (%) m.m⁻¹ | 51.1±3.01| 53.5±1.01  | 57.6±2.01  |
| Total Lignin (%) m.m⁻¹ | 49.5±3.01| 45.3±2.01  | 29.4±0.30  |
| Cellulose (%) m.m⁻¹ | 33.3±2.01| 36.5±1.50  | 17.4±0.01  |
| Hemicellulose (%) m.m⁻¹ | 11.8±0.11| 10.2±2.01  | 5.3±0.01   |
| Arabinan (%) m.m⁻¹ | 2.4±0.01 | 3.1±0.05   | 0.8±0.01   |
| Protein (%) m.m⁻¹ | 2.2±0.01 | 3.0±1.01   | 14.6±0.01  |
| Fixed solids (%) m.m⁻¹ | 2.0±0.01| 2.4±0.03   | 1.1±0.05   |

Values shown in table are means of triplicate analyses with respective standard errors; ND = Not determined

2.21. Structural characterization of biomass

As shown in table 5b, the structural composition (lignin, cellulose, hemicellulose and arabinan) of all the biomasses was determined using a standard method [35]. In order to determine the extractable materials in the biomasses; a Soxhlet extractor was employed for 6 h while samples of each biomass was burnt in a muffle furnace for the determination of fixed solids [35]. For the structural compositions, 0.3 g dried sample of each biomass was treated with 3 mL of 72% sulfuric acid (v.v⁻¹) in a thermostatic bath at 30 °C for 1 h while the resulting filtrate was used for carbohydrate determination [36]. The carbohydrates were analyzed using a liquid chromatography-mass spectrometer (LC–MS) (SHIMADZU, Japan) operated with AMINEX® BIORAD HPX87H column in refractive index detector (DIR-10A). The mobile phase in this analysis was 0.005 mol.L⁻¹ sulfuric acid in an isocratic mode, at 45 °C, with an injection volume of 20 μL and flow of 0.6 mL.min⁻¹. Calibration curves were used to determine the final concentrations of the compounds using specific LC–MS standards [37]. All samples were analyzed in triplicates.

Table 5b. Structural characterization of biomasses

2.22. Biomass gasification

The designed gasifier was employed in the pyrolysis of the three different biomasses using air as the gasification agent. The gasifier was designed to allow working with two different airflow inlets i.e. from both top and the tuyeres. Analyses of the biogas major components (CH₄, CO₂, H₂S and N₂) were carried out using infrared and electrochemical sensors (BIOGASS5000, USA).

3. Results and Discussion

The test results showed that the three biomasses (Saw dust, wood chips, Bean chaff) at 10% moisture level produced the least ash content. The ash contents at 10, 20 and 30% moisture contents for wood chips, sawdust and bean chaff were 0.210, 0.457, 0.750 kg; 0.202, 0.290, 0.651 kg and 0.295, 0.228, 0.394 kg respectively. The gas samples were taken at 10% moisture for analysis because it produced the lowest ash content. Gas produced at 10% moisture content showed that methane (CH₄), Carbon dioxide (CO₂),
Hydrogen sulfide (H₂S) and Nitrogen (N₂) contents for wood chips; sawdust and bean chaff were 60.85, 27.50, 0.44, 10.2%; 62.33, 23.77, 0.87, 8.5% and 63.94, 18.91, 0.58, 10.6% respectively. The values of CO was insignificant. The moisture content of the biomasses significantly (p < 0.05) affected the values of ash content, gasification time and temperature but the effects of biomass types were not significant. Ash content and gasification time increased with increase in moisture level with the least value of 0.210 kg and 61 minutes at 10% moisture content respectively. The gasification temperature decreased as moisture level increased and vice versa. Increase in moisture level increased the ash content and gasification time. The gasification temperature also increased as the gasification time reduced. Gasifier efficiency was also affected by moisture content and biomass types. The best gasifier efficiency was observed at 10% moisture content level with 60, 57 and 75% for sawdust, woodchips and bean chaff respectively. These results are in agreement with previous findings [38-48]. The temperature of the cooling unit after filter was measured to be 40°C with the aid of hand-held digital thermometer.

4. Conclusion

This study focused on the design of an imbert type of gasifier design (Figure 2b). Gas storage facility can also be considered in the other research work. The fabricated gasifier also showed high efficiency in the pyrolysis of the selected biomasses and the product yields are appreciable.

Conflict of Interest

Authors declare that there are no conflicts of interest whatsoever. All authors agrees to this submission

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Notation

APHA = American Public Health Association  
COD = Chemical Oxygen Demand  
FAO = Food and Agricultural Organization  
KW = Kilowatt  
KVA = Kilo Volt-ampere  
LC-MS = Liquid Chromatography- Mass Spectrometer  
RPM = Rotation per Minute  
TOC = Total Organic Carbon  
VFAs = Volatile Fatty Acids

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