Research Article

Tracking Control of a Leg Rehabilitation Machine Driven by Pneumatic Artificial Muscles Using Composite Fuzzy Theory

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Abstract

It is difficult to achieve excellent tracking performance for a two-joint leg rehabilitation machine driven by pneumatic artificial muscles (PAMs) because the system has a coupling effect, highly nonlinear and time-varying behavior associated with gas compression, and the nonlinear elasticity of bladder containers. This paper therefore proposes a T-S fuzzy theory with supervisory control in order to overcome the above problems. The T-S fuzzy theory decomposes the model of a nonlinear system into a set of linear subsystems. In this manner, the controller in the T-S fuzzy model is able to use simple linear control techniques to provide a systematic framework for the design of a state feedback controller. Then the LMI Toolbox of MATLAB can be employed to solve linear matrix inequalities (LMIs) in order to determine controller gains based on the Lyapunov direct method. Moreover, the supervisory control can overcome the coupling effect for a leg rehabilitation machine. Experimental results show that the proposed controller can achieve excellent tracking performance, and guarantee robustness to system parameter uncertainties.

1. Introduction

In cases of traumatic brain injury, bone injury, amputation, or spinal cord injury caused by misfortunes such as traffic accidents and cerebral apoplexy, lower limb rehabilitation machine can help patients recover extremity functions by means of continuous passive motion (CPM). Traditionally, physical therapy for achieving functional rehabilitation is carried out by medical therapists on a person-to-person basis. However, recently many automatic rehabilitation devices have been gradually applied in physical therapy programs. Rehabilitation machines are usually driven by electric motors, which are typically rigid in nature. Because of this, actuators can generate discomfort or pain when interfacing with humans. For this reason, current electromechanical actuation systems should be replaced to ensure adaptability, conformity, and safety. An adequate actuator for a rehabilitation device must provide physically adjustable compliance and safety and ensure soft contact with the patient, similar to the behavior of human muscles. It has been suggested that pneumatic artificial muscles (PAMs) can contribute towards achieving more comfortable devices for interfacing with human limb segments.

PAMs behave in a manner very similar to the muscles that move the skeletons of animals and have many advantages, such as high power to weight ratio [1], high power to volume ratio [2], low maintenance, negligible mechanical wear, low cost, cleanliness, high reliability, flexibility, and compliance for use with humans. For these reasons, PAMs are commonly employed in rehabilitation engineering, nursing, and human-friendly therapeutic machine.

However, PAMs exhibit highly nonlinear and time-varying behavior due to the compression of air and the nonlinear elasticity of bladder containers. This makes it difficult for classical controllers to achieve excellent control performance. In recent years, researchers have developed a wide variety of approaches to overcome these problems. Noritsugu and Tanaka [3] developed four modes of linear motion with impedance control to control force during movement and used an adaptive identification method to estimate the system model. Lilly and Yang [4] applied a sliding mode controller to a planar arm actuated by two PMA groups; simulation results
were consistent with theoretical findings for two different masses. Ahn and Anh [5] adopted an ARNN controller in a PAM manipulator for reducing tracking errors. Shen [6] developed a full nonlinear model that encompassed all the major existing nonlinearities. Based on this model, the standard sliding mode control approach was applied to obtain robust control, even in the event of model uncertainties and disturbances.

Since the inception of fuzzy set theory by Zadeh [7] in 1965, a great deal of research has been focused on fuzzy control systems. Takagi and Sugeno [8] proposed the T-S fuzzy model-based controller in 1985, and the T-S fuzzy model-based controller, a complex dynamic model can be decomposed into a set of local linear subsystems via fuzzy inference. Stability analysis is carried out using the Lyapunov direct method, where the control problem is formulated into an inference. Stability analysis is carried out using the Lyapunov function. The fuzzy theory, which includes T-S fuzzy tracking control and conventional fuzzy control systems. Takagi and Sugeno [8] proposed the T-S fuzzy model-based controller in 1985, and the T-S fuzzy control systems. Takagi and Sugeno [8] proposed the T-S fuzzy model-based controller in 1985, and the T-S fuzzy model-based system subsequently emerged as one of the most active and fruitful areas of fuzzy control. Using a T-S fuzzy model-based controller, a complex dynamic model can be decomposed into a set of local linear subsystems via fuzzy inference. Stability analysis is carried out using the Lyapunov direct method, where the control problem is formulated into an inference. Stability analysis is carried out using the Lyapunov function.

The leg rehabilitation machine driven by PAMs is a two-input, two-output system. This paper proposes composite fuzzy tracking controller and supervisor control in order to improve tracking performance. The proposed approach decomposes the model of a nonlinear system into a set of local subsystems with associated nonlinear weighting functions, enabling the use of simple linear control techniques without the need for complicated nonlinear control strategies, and also provides a systematic framework for the design of a state feedback controller [11]. It has been shown that a composite fuzzy control system can be guaranteed to be asymptotically stable if a common positive definite solution exists for a set of Lyapunov inequalities. In addition, the supervisory control can overcome the coupling effect due to two-joint motion. In view of the above advantages, the proposed controller was applied to the output tracking control of this system, and experimental results verified that the proposed controller is capable of achieving excellent tracking performance.

The remainder of the paper is organized as follows. Section 2 describes the control strategies. Section 3 describes the system. In Section 4, the dynamics of the model are derived. Experimental results for output tracking are shown in Section 5. Finally, conclusions are presented in Section 6.

2. Control Strategies

2.1. Takagi-Sugeno Fuzzy Tracking Controller. Consider a general nonlinear dynamic equation

\[
\dot{x}(t) = f(x(t)) + g(x(t))u(t)
\]

\[
y(t) = q(x(t)),
\]

where \(x \in \mathbb{R}^n\) is the state vector, \(y \in \mathbb{R}^m\) is the controlled output, \(u \in \mathbb{R}^m\) is the control input vector, and \(f(x), g(x), q(x)\) are nonlinear functions with appropriate dimensions. The nonlinear system (1) can then be expressed by the fuzzy system.

Rule i:

IF \(z_1(t)\) is \(F_{i1}\) and \(\cdots\) and \(z_r(t)\) is \(F_{gi}\)

THEN \(\dot{x}(t) = A_i x(t) + B_i u(t), \quad i = 1, 2, \ldots, r,\)

where \(z(t)_1 \sim z(t)_r\) are the premise variables including system states, \(F_i\) denotes the fuzzy sets, \(r\) is the number of fuzzy rules, and \(A_i\) and \(B_i\) are system matrices with appropriate dimensions. For simplicity, this study assumed that the membership function had been normalized; that is, \(\sum_{i=1}^{r} F_{i}(z_i) = 1\). As in (1), let the singleton fuzzifier, product inferred, and weighted defuzzifier, the fuzzy system is inferred as

\[
\dot{x}(t) = \sum_{i=1}^{r} h_i [A_i x(t) + B_i u(t)],
\]

where \(h_i(z(t)) = \Pi_{j=1}^{r} F_{i}(z_j)\). Note that \(\sum_{i=1}^{r} h_i(z(t)) = 1\) for all \(t\), where \(h_i(z(t)) \geq 0\) for \(i = 1, 2, \ldots, r\) is regarded as grade functions.

For output tracking control, the control objective is required to satisfy

\[
\overline{y}(t) - r(t) \rightarrow 0 \quad \text{as} \quad t \rightarrow \infty,
\]

where \(r(t)\) denotes the desired trajectory or reference signal. To convert the output tracking problem into a stabilization problem, a set of virtual desired variables \(x_d(t)\) was introduced, to be tracked by the state variable \(x\). Let \(\bar{x}(t) = x(t) - x_d(t)\) denote the tracking error for the state variables. The time derivative of \(\bar{x}(t)\) yields

\[
\dot{x}(t) = \ddot{x} - \dot{x}_d = \sum_{i=1}^{r} h_i [A_i x(t) + B_i u(t)] - \dot{x}_d(t).
\]

If the control input \(u(t)\) is assumed to satisfy the following equation:

\[
\sum_{i=1}^{r} h_i B_i \tau(t) = \sum_{i=1}^{r} h_i B_i u(t) + \sum_{i=1}^{r} h_i A_i x_d(t) - \dot{x}_d(t),
\]

where \(\tau(t)\) is a new control to be designed, then the tracking error system (5) results in the following form:

\[
\dot{x}(t) = \sum_{i=1}^{r} h_i A_i \bar{x}(t) + \sum_{i=1}^{r} h_i B_i \tau(t).
\]

The design of the new control \(\tau(t)\) is similar to solving a stabilization problem. The purpose is to steer \(\bar{x}(t)\) to zero, which means that state \(x(t)\) tracks \(x_d(t)\). The new fuzzy
controller \( r(t) \) is designed on the basis of parallel distributed compensation (PDC) and is represented as follows.

Rule i:

IF \( z_i(t) \) is \( F_{ii} \) and \( \cdots \) and \( z_g(t) \) is \( F_{gi} \)

THEN \( r(t) = -K_i \bar{x}(t) \),

where \( K_i \) represents feedback gain. The inferred output of the PDC controller is expressed in the following form:

\[
\tau(t) = - \sum_{i=1}^{r} h_i K_i \bar{x}(t) . \tag{9}
\]

Substituting (9) into (7) yields

\[
\dot{\bar{x}}(t) = \sum_{i=1}^{r} \sum_{j=1}^{r} h_i h_j \left( A_i - B_i K_j \right) \bar{x}(t) . \tag{10}
\]

The stability analysis of this tracking system (10) is carried out using the Lyapunov direct method, and the Lyapunov function is defined as

\[
V(\bar{x}(t)) = \bar{x}^T(t) P \bar{x}(t) > 0 , \tag{11}
\]

where \( P \) is a positive symmetric matrix. Taking the derivative of \( V \) with respect to time yields

\[
\dot{V}(\bar{x}(t)) = \bar{x}^T(t) P \bar{x}(t) + \bar{x}^T(t) P \dot{\bar{x}}(t) \]

\[
= \sum_{i=1}^{r} \sum_{j=1}^{r} h_i h_j \left( A_i - B_i K_j \right) \bar{x}(t) \]

\[
= \sum_{i=1}^{r} \sum_{j=1}^{r} h_i h_j \bar{x}^T \left( A_i - B_i K_j \right) \bar{x} \]

\[
= \sum_{i=1}^{r} \sum_{j=1}^{r} h_i h_j \bar{x}^T \left( A_i - B_i K_j \right) P \bar{x} + \sum_{i=1}^{r} h_i B_i u_i(t) \]  \tag{12}

The controller is stable if \( \dot{V} < 0 \). Hence, the LMI form is expressed as follows:

\[
\sum_{i=1}^{r} \sum_{j=1}^{r} h_i h_j \left( A_i - B_i K_j \right) P \leq -Q_i, \quad i = 1, \ldots, r , \tag{13}\]

where \( Q_i \) is a positive definite matrix. Based on this assumption, each subsystem is locally controllable, and a stable feedback gain is obtainable. Intuitively, a common matrix \( P \) that satisfies (17) can be obtained more easily than can one that fulfills the basic stabilization conditions. When the LMI method is applied, conditions (17) can be efficiently verified. If a feasible solution is obtained, the design proceeds to exploit the supervisory control in order to deal with the coupling effects.

Choose the Lyapunov function candidate, \( V_1(x) = \bar{x}^T P \bar{x} \). The time derivative of \( V_1(x) \) is as follows:

\[
\dot{V}_1(x) = \sum_{i=1}^{r} h_i^2 \bar{x}^T \left( G_{gi}^T P + PG_{ij} \right) \bar{x} \]

\[
+ 2 \sum_{i=1}^{r} h_i \bar{x}^T \left( G_{gi}^T P + PG_{ij} \right) \bar{x} + 2 \bar{x}^T P \bar{x} \leq -\sum_{i=1}^{r} h_i^2 \bar{x}^T Q_i \bar{x} \]

\[
+ 2 \sum_{i=1}^{r} h_i \bar{x}^T \left( G_{gi}^T P + PG_{ij} \right) \bar{x} + 2 \bar{x}^T P \bar{x} \] \tag{18}

Given the matrix property, clearly,

\[
\lambda_{\min} \left( G_{gi}^T P + PG_{ij} \right) \| \bar{x} \|^2 \leq \bar{x}^T \left( G_{gi}^T P + PG_{ij} \right) \bar{x} \]

\[
\leq \lambda_{\min} \left( G_{gi}^T P + PG_{ij} \right) \| \bar{x} \|^2 , \tag{19}\]
where \( \lambda_{\text{min}(\max)} \) denotes the smallest (largest) eigenvalue of the matrix. Define

\[
\alpha = \max_{i,j} \lambda_{\text{max}} \left( G_{ij}^T P + PG_{ij} \right) \quad \text{for } 1 \leq i < j \leq r. \tag{20}
\]

A relaxed condition concerning the coupling effect is expressed as

\[
\sum_{i < j} h_i h_j \bar{x}^T (G_{ij}^T P + PG_{ij}) \bar{x} \leq k_i \| \bar{x} \|^2, \quad k_i = \frac{r (r - 1)}{2} \alpha. \tag{21}
\]

Finding the maximum value of \( \sum_{i < j} h_i h_j x^T (G_{ij}^T P + PG_{ij}) \bar{x} \) is equivalent to determining the maximum value of \( \sum_{i < j} h_i h_j \lambda_{\text{max}} (G_{ij}^T P + PG_{ij}) \). This can be presented as a nonlinear programming. The optimal algorithms are employed to seek the best solution. Moreover, the MATLAB Optimization Toolbox consists of functions that minimize or maximize general nonlinear functions. By using the toolbox, the nonlinear programming is expressed in the following form:

\[
\begin{align*}
\max_{i,j} & \quad \sum_{i < j} h_i h_j \lambda_{\text{max}} (G_{ij}^T P + PG_{ij}) \quad 1 \leq i < j \leq r \\
\text{Subject to} & \quad \sum_{i=1}^{r} \mu_i = 1 \quad \mu_i \geq 0 \\
& \quad \sum_{j=1}^{r} \mu_j = 1 \quad \mu_j \geq 0. \tag{22}
\end{align*}
\]

The largest eigenvalue of \( (G_{ij}^T P + PG_{ij}) \) can be obtained in advance, so the maximum value is determined to be

\[
k_2 = \max_{i,j} \sum_{i < j} h_i h_j \lambda_{\text{max}} (G_{ij}^T P + PG_{ij}). \tag{23}
\]

The following supervisory control is chosen:

\[
u_s = \begin{cases} 
\frac{-B^T P x}{\| x^T P b \|}, & \text{if } \| x^T P b \| \neq 0 \\
0, & \text{if } \| x^T P b \| = 0, \tag{24}
\end{cases}
\]

where \( k > k_j, j = 1 \text{ or } 2 \). If \( \| x^T P b \| \neq 0 \), then substituting (24) into (18) gives

\[
\begin{align*}
\dot{V}_1 (x) & \leq - \sum_{i=1}^{r} h_i^2 x^T Q_i \bar{x} + 2k \| \bar{x} \|^2 - 2k \| \bar{x} \|^2 \\
& \leq - \sum_{i=1}^{r} h_i^2 \bar{x}^T Q_i \bar{x} = - V_2 (x), \tag{25}
\end{align*}
\]

where \( V_2 (x) \) is a positive definite function. When \( \| x^T P b \| = 0 \) can give the following form:

\[
\begin{align*}
\dot{V}_1 (x) & \leq - \sum_{i=1}^{r} h_i^2 \bar{x}^T Q_i \bar{x} + \sum_{i < j} h_i h_j [\bar{x}^T (A_i P - PA_i) \bar{x} + \bar{x}^T (A_j^T P - PA_j) \bar{x}] \\
& \leq - \sum_{i=1}^{r} h_i h_j [\bar{x}^T (A_i P - PA_i) \bar{x} + \bar{x}^T (A_j^T P - PA_j) \bar{x}] = -V_3 (x), \tag{26}
\end{align*}
\]

where \( V_3 (x) \) is a positive definite function. Thus, the closed-loop fuzzy system is asymptotically stable.

### 3. System Descriptions

Figure 1 shows the experimental setup, including four PAMs, two rotary potentiometers, four pressure proportional valves, and four pressure transducers. The hardware includes an IBM-compatible personal computer to calculate the control signal, which controls the pressure proportional valve through a D/A card. The angles of the joints are detected using rotary potentiometers, the air pressure of each PAM is measured using pressure transducers, and the measurements are then fed back to the computer through an A/D card. These specifications are listed in Table 1.

Figure 2 presents the operation principle of the leg rehabilitation machine, depicting a two-joint leg. The behavior of the leg manipulated by the rehabilitation machine is similar to that of a human leg. Output angles \( \theta_1 \) and \( \theta_2 \) simulate the knee and ankle joints, and the ranges of the rotary angles \( \theta_1 \) and \( \theta_2 \) are from \(-45^\circ \) to \(-45^\circ \) and from \(-50^\circ \) to \(-50^\circ \), respectively. The link mass \( m_1 = 2.7 \text{ kg} \), \( m_2 = 0.81 \text{ kg} \), and the link length \( l_1 = 0.5 \text{ m} \), \( l_2 = 0.26 \text{ m} \). The rotating torque \( r \) is generated by the difference in pressure \( \Delta p \) between the two opposing PAMs. That is, when \( P_a > P_b \), as in Figure 2, the torque exerted on the joint is counterclockwise and the rotation of the joint is also counterclockwise.

So, a pair of such PAMs is tied together around a pulley with a radius \( r_j \), as in Figure 2. Then, the torque values imparted to the pulley by the PAM pair are [12]

\[
\begin{align*}
t_1 &= (\phi_{1a} - \phi_{1b}) r_1 \\
t_2 &= (\phi_{2a} - \phi_{2b}) r_2, \tag{28}
\end{align*}
\]
Figure 1: The experimental setup.

Figure 2: Operation principle of the leg rehabilitation machine driven by PAMs.
where
\[\phi_{1a} = F_{1a} (p_{1a}) - K_{1a} (p_{1a}) r_1 \dot{\theta}_1 - B_{1a} (p_{1a}) r_1 \theta_1 \]  
\[\phi_{1b} = F_{1b} (p_{1b}) - K_{1b} (p_{1b}) r_1 \dot{\theta}_1 - B_{1b} (p_{1b}) r_1 \dot{\theta}_1 \]  
\[\phi_{2a} = F_{2a} (p_{2a}) - K_{2a} (p_{2a}) r_2 \dot{\theta}_2 - B_{2a} (p_{2a}) r_2 \theta_2 \]  
\[\phi_{2b} = F_{2b} (p_{2b}) - K_{2b} (p_{2b}) r_2 \dot{\theta}_2 - B_{2b} (p_{2b}) r_2 \dot{\theta}_2 , \]

where the spring coefficient \(K(p)\) and the damping coefficient \(B(p)\) are given by Reynolds et al. [13]. The desired input pressures \(P_a = [p_{1a} \ p_{2a}]^T\) and \(P_b = [p_{1b} \ p_{2b}]^T\) for each PAM are generated by the following equation:
\[P_a (t) = P_0 + \Delta P (t) , \quad P_b (t) = P_0 - \Delta P (t) , \]

where \(P_0 = [p_{10} \ p_{20}]^T\) is a nominal constant input PAM pressure and \(\Delta P (t) = [\Delta p_1 \ \Delta p_2]^T\) is the control pressure input with an arbitrary function of time. Because the pressure input \(\Delta P (t) = [\Delta p_1 \ \Delta p_2]^T\), and output \(\dot{\theta}_1 = [\dot{\theta}_1 \ \dot{\theta}_2]^T\), the system can be written as a two-input, two-output (TITO) control system. The control signal \(u = [u_1 \ u_2]^T\) is proportional to \(\Delta P\) based on the pressure proportional valve’s characteristics. That is, \(\Delta P\) can be used instead of \(u\) as a control input.

4. Dynamic Model of a Two-Joint Leg Rehabilitation Machine Driven by PAMs

Figure 2 shows a two-joint leg rehabilitation machine driven by PAMs, and the dynamic equation is given as follows [14]:
\[M (\theta) \ddot{\theta} + C (\theta, \dot{\theta}) \dot{\theta} + G (\theta) = \tau , \]

where
\[M (\theta) = \begin{bmatrix} (m_1 + m_2) l^2 & -m_2 l_1 l_2 (s_1 s_2 + c_1 c_2) \\ -m_2 l_1 l_2 (c_1 s_2 - s_1 c_2) & m_2 l_2^2 \end{bmatrix} \]
\[C (\theta, \dot{\theta}) = \begin{bmatrix} 0 & -m_2 l_1 l_2 (c_1 s_2 - s_1 c_2) \dot{\theta}_1 \\ -m_2 l_1 l_2 (c_1 s_2 - s_1 c_2) \dot{\theta}_1 & 0 \end{bmatrix} \]
\[G (\theta) = \begin{bmatrix} (m_1 + m_2) l_1 g s_1 \\ -m_2 l_2 g s_2 \end{bmatrix} \]

and \(M(\theta)\) is the moment of inertia, \(C(\theta, \dot{\theta})\) includes Coriolis and centripetal force, and \(G(\theta)\) is the gravitational force.

Notation \(s_1 = \sin(\theta_1), s_2 = \sin(\theta_2), c_1 = \cos(\theta_1), \) and \(c_2 = \cos(\theta_2).\) Let \(x_1 = \dot{\theta}_1, x_2 = \dot{\theta}_2, x_3 = \theta_1, \) and \(x_4 = \theta_2;\) then (31) can be written as the following state-space form [14]:
\[\ddot{x}_1 = x_2 \]
\[\ddot{x}_2 = f_1 (x) + g_{11} (x) \tau_1 + g_{12} \tau_2 \]
\[\ddot{x}_3 = x_4 \]
\[\ddot{x}_4 = f_2 (x) + g_{21} (x) + g_{22} \tau_2 , \]

where
\[f_1 (x) = \frac{(s_1 c_2 - c_1 s_2) \left[ m_2 l_1 l_2 (s_1 s_2 + c_1 c_2) x_3^2 - m_2 l_2^2 s_1^2 \right]}{l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} + \frac{\left[ (m_1 - m_2) l_1 g s_1 - m_2 l_2 g s_2 \right] (s_1 s_2 + c_1 c_2)}{l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} \]
\[f_2 (x) = \frac{(s_1 c_2 - c_1 s_2) \left[ - (m_1 + m_2) l_1^2 s_2^2 + m_2 l_1 l_2 (s_1 s_2 + c_1 c_2) x_4^2 \right]}{l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} + \frac{\left[ - (m_1 + m_2) l_1 g s_1 \right] (s_1 s_2 + c_1 c_2) + \left[ (m_1 + m_2) l_1 g s_2 \right]}{l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} \]
\[g_{11} (x) = \frac{m_2 l_2^2}{m_2 l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} \]
\[g_{12} (x) = \frac{-m_2 l_1 l_2 (s_1 s_2 + c_1 c_2)}{m_2 l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} \]
\[g_{21} (x) = \frac{-m_2 l_1 l_2 (s_1 s_2 + c_1 c_2)}{m_2 l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} \]
\[g_{22} (x) = \frac{(m_1 + m_2) l_1^2}{m_2 l_1 l_2 \left[ (m_1 + m_2) - m_2 (s_1 s_2 + c_1 c_2) \right]} . \]

5. Experimental Studies

Figure 3 shows the leg rehabilitation machine using an actual human loading with a 65 kg weight. The automatic device
can help patients to recover lower limb motion function by means of continuous passive motion, such as a sinusoidal wave command, an irregular curve command, and an end-effect tracking command. The experiments include both the proposed approach and PDC for comparison in order to evaluate efficacy and control performance. The controllers were implemented on an Intel Pentium 1.8 GHz PC with a sampling time of 5 ms, and the entire control software was coded in C++.

This study attempts to use as few rules as possible in order to minimize design effort and complexity. The T-S fuzzy model of the system is thus given the following four-rule fuzzy model:

\( R_1 \): IF \( x_1 \) is about \( \frac{\pi}{4} \) and \( x_3 \) is about \( \frac{\pi}{4} \)

THEN \( \dot{x} = A_1 x_1 + B_1 u \)

\( R_2 \): IF \( x_1 \) is about \( \frac{\pi}{4} \) and \( x_3 \) is about \( -\frac{\pi}{4} \)

THEN \( \dot{x} = A_2 x_1 + B_2 u \)

\( R_3 \): IF \( x_1 \) is about \( -\frac{\pi}{4} \) and \( x_3 \) is about \( \frac{\pi}{4} \)

THEN \( \dot{x} = A_3 x_1 + B_3 u \)

\( R_4 \): IF \( x_1 \) is about \( -\frac{\pi}{4} \) and \( x_3 \) is about \( -\frac{\pi}{4} \)

THEN \( \dot{x} = A_4 x_1 + B_4 u \),

where

\[
A_1 = \begin{bmatrix}
0 & 1 & 0 & 0 \\
4.4082 & 0.0059 & 0.6742 & -0.0002 \\
0 & 0 & 0 & 1 \\
-1.4572 & 0.0002 & -3.9722 & 0.0002 \\
\end{bmatrix}
\]

\[
A_2 = \begin{bmatrix}
0 & 1 & 0 & 0 \\
4.6734 & 0.0049 & 0.532 & 0.0001 \\
0 & 0 & 0 & 1 \\
-1.2145 & -0.0001 & -3.563 & 0.0001 \\
\end{bmatrix}
\]

\[
A_3 = \begin{bmatrix}
0 & 1 & 0 & 0 \\
5.4782 & 0.0021 & 0.6876 & -0.0002 \\
0 & 0 & 0 & 1 \\
1.4525 & 0.0002 & 4.6723 & 0.0002 \\
\end{bmatrix}
\]

\[
A_4 = \begin{bmatrix}
0 & 1 & 0 & 0 \\
4.9821 & -0.0044 & 0.6543 & 0.0002 \\
0 & 0 & 0 & 1 \\
-1.1034 & -0.0002 & 3.5631 & 0.0003 \\
\end{bmatrix}
\]

\[
B_1 = B_4 = \begin{bmatrix}
0 & 0 \\
1.4811 & -2.849 \\
0 & 0 \\
-2.849 & 23.7417 \\
\end{bmatrix}
\]

\[
B_2 = B_3 = \begin{bmatrix}
0 & 0 \\
1.1965 & 1.0297 \\
0 & 0 \\
1.0297 & 19.1501 \\
\end{bmatrix}
\]

\[
P = \begin{bmatrix}
16.2602 & -0.6640 & 1.0734 & -0.0095 \\
-0.6640 & 0.3007 & -0.3455 & 0.2871 \\
1.0734 & -0.3455 & 0.4730 & -0.3458 \\
-0.0095 & 0.2871 & -0.3458 & 0.3658 \\
\end{bmatrix}
\]

\[
K_1 = \begin{bmatrix}
-0.4509 & 0.1745 & -0.7182 & 0.0527 \\
-0.2807 & 0.0034 & -0.0471 & 0.2639 \\
-0.4619 & 0.1238 & -0.7812 & 0.0351 \\
-0.2827 & 0.0068 & -0.0851 & 0.1401 \\
\end{bmatrix}
\]

\[
K_2 = \begin{bmatrix}
-0.5902 & 0.2132 & -0.8910 & 0.0531 \\
-0.4107 & 0.0117 & -0.0730 & 0.2989 \\
-0.4891 & 0.2109 & -0.6734 & 0.0414 \\
-0.3180 & 0.0085 & -0.0631 & 0.3893 \\
\end{bmatrix}
\]

which guarantee the stability condition (17). MATLAB Toolbox is used to obtain parameters as \( k_1 = 0.0251 \) and
Table 2: Peak-peak error and phase lag for Figure 5.

| The proposed approach | PDC |
|-----------------------|-----|
| Peak-peak error | Phase lag | Peak-peak error | Phase lag |
| $\theta_1$ | $\theta_2$ | $\theta_1$ | $\theta_2$ | $\theta_1$ | $\theta_2$ |
| 0.7% | 0.35% | 4.9° | 4.5° | 1.3% | 0.65% | 16.2° | 10.8° |

$k_2 = 0.01539$. For comparison with the proposed controller, the PDC feedback gains are designed to be

$$\hat{K}_1 = \begin{bmatrix} -0.3293 & 0.1023 & -0.2538 & 0.02317 \\ -0.1783 & 0.0021 & -0.02672 & 0.4732 \end{bmatrix}$$

$$K_2 = \begin{bmatrix} -0.4278 & 0.1845 & -0.2451 & 0.0731 \\ -0.2185 & 0.0026 & -0.0752 & 0.0923 \end{bmatrix}$$

$$K_3 = \begin{bmatrix} -0.6013 & 0.3461 & -0.5482 & 0.0231 \\ -0.4421 & 0.0093 & -0.0651 & 0.3529 \end{bmatrix}$$

$$K_4 = \begin{bmatrix} -0.3756 & 0.3150 & -0.5391 & 0.0421 \\ -0.4250 & 0.0023 & -0.0531 & 0.3597 \end{bmatrix}.$$

5.1. Sinusoidal Wave Response. Continuous reciprocation is required in order to foster the recovery of extremity function. The sinusoidal wave responses of the proposed approach and PDC for both knee and ankle joints are shown in Figure 4. It is evident that angle trajectories of the proposed approach are close to the command. Figure 5 shows that the proposed approach exhibits less tracking errors than does PDC. The peak-peak error and phase lag are listed in Table 2. Because of the interaction of the two joints, PDC has significant angle errors for $\theta_1$, which will degrade the rehabilitation effect. However, supervisory control can overcome the coupling effect of the two joints to achieve excellent rehabilitation function for patients.

5.2. Irregular Curve Response. In practical applications, it could be expected that the reference command will change with different input frequencies. The desired trajectories for both knee and ankle joints are

$$\theta_1 = 20 \ast 0.33 \left( \sin(2\pi f_1 t) + \sin(2\pi f_2 t) + \sin(2\pi f_3 t) \right)$$

$$\theta_2 = 15 \ast 0.33 \left( \sin(2\pi f_1 t) + \sin(2\pi f_2 t) + \sin(2\pi f_3 t) \right)$$

with $f_1 = 0.05$ Hz, $f_2 = 0.1$ Hz, and $f_3 = 0.066$ Hz.

Figure 6 shows the tracking responses of irregular curves obtained using both the proposed approach and PDC. Tracking errors for the knee and ankle joints are shown in Figure 7. Clearly, the angle error of the proposed approach is average maintained within 2°. However, the proposed approach is capable of adapting to different frequencies.
5.3. Elliptic Response. The desired end-effect or trajectory is given by

\[ x_d(t) = 0.614 - 0.015 \cdot \cos(0.2\pi \cdot t - \pi) \]
\[ y_d(t) = -0.1 \cdot \sin(0.2\pi \cdot t - 2\pi) , \]  

where 0 ≤ t ≤ 20 seconds.

The end-effect tracking responses in the x, y coordinate for both the proposed approach and PDC are shown in Figure 8, and the end-effect position tracking errors are displayed in Figure 9. It is evident that tracking behavior of the proposed approach is better than that of the PDC. As can be seen, the tracking errors of the proposed approach are
within 0.03 m. On the other hand, angle tracking errors of knee and ankle joints are shown in Figure 10.

Moreover, it is difficult to enhance end-effector tracking performance using the PDC algorithm because the PDC cannot overcome the nonlinearity of PAMs and the structural interaction. However, the proposed approach overcomes successfully the coupling effect and parameter uncertainties of the system. As seen in the experimental results, the proposed approach can attain excellent end-effector tracking performance in rehabilitation function.

6. Conclusions

In this study, a novel composite fuzzy control is proposed and applied in the two-joint leg rehabilitation device driven by PAMs. The proposed controller is not only capable of decomposing nonlinear systems into a set of linear subsystems, but is also capable of simplifying a complex nonlinear system using linear control techniques, with the control gains determined using MATLAB’s LMI Toolbox based on the Lyapunov stability theorem. Moreover, the supervisory control can overcome
the coupling effect for a leg rehabilitation machine. Experimental results show that the system response of the proposed approach was in good agreement with that of the reference input and guarantee robustness to system parameter uncertainties.

**Conflict of Interests**

The author declares that there is no conflict of interests regarding the publication of this paper.

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