Structural–parametric model electroelastic actuator nano– and microdisplacement of mechatronics systems for nanotechnology and ecology research

Abstract
The structural–parametric model, the decision of the wave equation, the parametric structural schematic diagram, the transfer functions of the electroelastic actuator of the mechatronics system for the nanotechnology and the ecology research are obtained. Effects of geometric and physical parameters of the piezoactuator and the external load on its dynamic characteristics are determined. The parametric structural schematic diagram and the transfer functions of the piezoactuator for the transverse, longitudinal, shift piezoelectric effects are obtained from the structural–parametric model of the piezoactuator. For calculation of the mechatronics systems for the nanotechnology with the piezoactuator it’s the parametric structural schematic diagram and the transfer functions are determined. The generalized parametric structural schematic diagram of the electroelastic actuator is constructed.

Keywords: electroelastic actuator, piezoactuator, deformation, structural–parametric model, parametric structural schematic diagram, decision wave equations, transfer functions

Introduction
For the nanotechnology, the ecology research, the nanobiology, the power engineering, the microelectronics, the astronomy for the large compound telescopes, the antennas satellite telescopes and the adaptive optics equipment is promising for use the mechatronics system with the actuator based on the piezoelectricity for the piezoelectric or the electrostriction effects. The piezoactuator is the piezomechanical device intended for the actuation of mechanisms, systems or the management based on the piezoelectric effect, the converts electrical signals into the mechanical movement or the force. In the present work is solving the problem of building the structural parametric model of the electroelastic actuator in contrast Cady and Mason electrical equivalent circuits for calculation of piezoelectric transmitter and receiver. The structural–parametric model of the piezoactuator describes the structure and conversion the energy electric field into the mechanical energy and the corresponding displacements and forces at its the faces. The structural–parametric model of the electroelastic actuator of the mechatronics system is determined by using the method of the mathematical physics. The transfer functions and the parametric structural schematic diagrams of the electroelastic actuator are obtained from its structural–parametric model. The piezoactuator for the nano– and microdisplacement of the mechatronics system operates based on the inverse piezoeffect. The displacement is achieved due to deformation of the piezoactuator when the external electric voltage is applied to it. The piezoactuator for the drives of nano– and micrometric movements provide a movement range from several nanometers to tens of micrometers, a sensitivity of up to 10 nm/V, a loading capacity of up to 1000 N, a transmission band of up to 100 Hz. The piezoactuator provides high speed and force, its return to the initial state when switched off. The use of the piezoactuator solves the problems of the precise alignment and the compensation of the temperature and gravitational deformations in the nanotechnology and the ecology research. The piezoactuator is used in the majority mechatronic systems for the nanotechnology, the ecology research in the scanning tunneling microscopes and the atomic force microscopes.

Decision wave equation and structural parametric model of electroelastic actuator
The deformation of the electroelastic actuator corresponds to its stressed state. In the piezoactuator there are six stress components \( T_{11}, T_{12}, T_{13}, T_{22}, T_{23}, T_{33} \) where the components \( T_{13}, T_{23} \) are related to extension–compression stresses and the components \( T_{11}, T_{22}, T_{33} \) to shear stresses. The matrix state equations connecting the electric and elastic variables for the polarized piezoceramics have the following form:

\[
D = d T + e^T E , \quad (1)
\]
\[
S = s^E T + d^E E , \quad (2)
\]

where the first equation describes the direct piezoelectric effect, and the second characterizes the inverse piezoelectric effect; \( D \) is the column matrix of electric induction along the coordinate axes; \( S \) is the column matrix of the relative deformations; \( T \) is the column matrix of the mechanical stresses; \( E \) is the column matrix of the electric field strength along the coordinate axes; \( s^E \) is the elastic compliance matrix for \( E = \text{const} \); \( d^E \) is the matrix of dielectric constants for \( T = \text{const} \); \( d^E \) is the transposed matrix of the piezoelectric modules.

In polarized piezoceramics from lead zirconate titanate PZT for the piezoactuator on Figure 1 there are five independent components \( s_{11}^E, s_{12}^E, s_{31}^E, s_{32}^E, s_{33}^E \) in the elastic compliance matrix, three independent components \( d_{31}, d_{32}, d_{33} \) in the transposed matrix of the piezoelectric modules and three independent components \( d_{11}, d_{12}, d_{13} \) in the transposed matrix of the piezoelectric modules and three independent components \( d_{11}, d_{12}, d_{13} \) in the transposed matrix of the piezoelectric modules and three independent components.  

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\[ d_{ij}^2 \delta^2 \frac{d^2}{dx^2} \] in the matrix of dielectric constants.

\[ \Xi(x, p) = \Xi(p) e^{\gamma x} \]  

(7)

With its solution being the function

\[ \Xi(x, p) = Ce^{\gamma x} + Be^{-\gamma x} \]  

(8)

Where \( \Xi(x, p) \) is the Laplace transform of the displacement of the section of the piezoelectric actuator, \( \gamma = \rho/\epsilon \) is the propagation coefficient.

We denote for the faces of the piezoelectric actuator

\[ \Xi(0, p) = \Xi(p) \]  

for \( x = 0 \),

(9)

\[ \Xi(\delta, p) = \Xi(p) \]  

for \( x = \delta \).

Then we get the coefficients \( C \) and \( B \)

\[ C = \left[ \Xi e^{-\delta p} - \Xi(p) \right] \left[ 2 \sinh(\delta p) \right] \]  

and \[ B = \left[ \Xi - \Xi e^{-\delta p} \right] \left[ 2 \sinh(\delta p) \right] \]  

(10)

The solution (7) can be written as

\[ \Xi(x, p) = \left[ \Xi(p) \sinh(\delta x) + \Xi(0) \cosh(\delta x) \right] / \sinh(\delta x) \]  

(11)

The equations for the forces on the faces of the piezoelectric actuator

\[ T(0, p) S_0 = F(p) + M \delta^2 \Xi(0, p) \]  

for \( x = 0 \),

(12)

\[ T(\delta, p) S_0 = -F(p) - M \delta^2 \Xi(\delta, p) \]  

for \( x = \delta \).

Where \( T(0, p) \) and \( T(\delta, p) \) are determined from the equation of the inverse piezoelectric effect. For \( x = 0 \) and \( x = \delta \), we obtain the set of equations for determining stresses in the piezoelectric actuator:

\[ T(0, p) = \frac{1}{s_{33}} \frac{d \Xi(x, p)}{dx} \bigg|_{x=0} - \frac{d \Xi(0, p)}{dx} E(p) \]  

(13)

\[ T(\delta, p) = \frac{1}{s_{33}} \frac{d \Xi(x, p)}{dx} \bigg|_{x=\delta} - \frac{d \Xi(\delta, p)}{dx} E(p) \]  

The set of equations (13) yield the set of the equations for the structural–parametric model of the piezoelectric actuator and the parametric structural schematic diagram of the voltage-controlled piezoelectric actuator for the longitudinal piezoelectric effect on Figure 2.

\[ \Xi(p) = \left[ \Xi(0) \sinh(\delta x) + \Xi(\delta, p) \cosh(\delta x) \right] / \sinh(\delta x) \]  

(14)

\[ \Xi_0(p) = \left[ \Xi(0) \sinh(\delta x) + \Xi(\delta, p) \cosh(\delta x) \right] / \sinh(\delta x) \]  

(15)

Where \( \delta \) is the displacement of the section, \( x \) is the coordinate, \( t \) is the time, \( c^2 \) is the sound speed for \( E = \text{const.} \), \( \alpha \) is the damping coefficient.

We can reduce the original problem for the partial differential hyperbolic equation of type (5) using Laplace transform to a simpler problem for the linear ordinary differential equation [10, 12]. Applying the Laplace transform to the wave equation (5):

\[ \Xi(x, p) = L[\zeta(x, t)] = \frac{\gamma}{\delta t} \zeta(x, t) e^{-\delta t} \]  

(6)

Setting the zero initial conditions we obtain the linear ordinary second-order differential equation with the parameter \( p \) in the form

\[ \frac{d^2 \Xi(x, p)}{dx^2} - \gamma^2 \Xi(x, p) = 0 \]  

(7)

Figure 1 Piezoelectric actuator.

Let us consider the piezoelectric actuator for the longitudinal piezoelectric effect, where \( \delta \) is thickness and the electrodes deposited on its faces perpendicular to axis 3, the area of which is equal \( S \). The direction of the polarization axis \( P \), i.e., the direction along which polarization was performed, is usually taken as the direction of axis 3. The equation of the inverse longitudinal piezoelectric effect has the form:

\[ S = d_{33} E(t) + s_{11} T(x, t) \]  

(3)

where \( S = \partial \zeta(x, t) / \partial x \) is the relative displacement of the cross section of the piezoelectric actuator, \( d \) is the piezomodule for the longitudinal piezoelectric effect, \( E(t) \) is the electric field strength, \( U(t) \) c is the voltage between the electrodes of actuator, \( \delta \) is the thickness, \( s_{11} \) is the elastic compliance along axis 3, and \( T_3 \) is the mechanical stress along axis 3.

The equation of equilibrium for the force acting on the piezoelectric actuator in the form

\[ TS_0 = F + M \delta^2 \Xi(0, p) \]  

(4)

Where \( F \) is the external force applied to the piezoelectric actuator, \( S_0 \) is the cross section area and \( M \) is the displaced mass.

Figure 2 Parametric structural schematic diagram of a voltage-controlled piezoelectric actuator for longitudinal piezoelectric.

\[ \Xi_0(p) = \left[ \Xi(0) \sinh(\delta x) + \Xi(\delta, p) \cosh(\delta x) \right] / \sinh(\delta x) \]  

(15)

Where \( \delta \) is the displacement of the section, \( x \) is the coordinate, \( t \) is the time, \( c^2 \) is the sound speed for \( E = \text{const.} \), \( \alpha \) is the damping coefficient.
From (2), (3), (14) we obtain the system of the equations describing the generalized structural–parametric model of the electroelastic actuator
\[ \xi_1(p) + \xi_2(p) + \xi_3(p) = \left[ d_{mm} \xi_m(p) \right] + \left[ \frac{1}{2} \mu \frac{E}{h \varepsilon_0} \right] + \left[ \frac{1}{2} \mu \frac{E}{h \varepsilon_0} \right] = \xi_m(p), \]
(15)
where
\[ d_{mm} = \left[ \begin{array}{cccc}
d_{11} & d_{12} & d_{13} & d_{14} \\
d_{21} & d_{22} & d_{23} & d_{24} \\
d_{31} & d_{32} & d_{33} & d_{34} \\
d_{41} & d_{42} & d_{43} & d_{44}
\end{array} \right] , \]
and
\[ \xi_m(p) = \left[ \begin{array}{c}
\xi_1(p) \\
\xi_2(p) \\
\xi_3(p)
\end{array} \right] . \]

Then the parameter \( \Psi \) of the control parameter for the electroelastic actuator has the form
\[ \Xi_1(p) = \left[ W_{11}(p) \Psi_m(p) + W_{12}(p) F_1(p) + W_{13}(p) p \right] \]
(16)
where the generalized transfer functions of the electroelastic actuator are
\[ W_{11}(p) = \left[ M \right]_{ij} \left[ \begin{array}{c}
W_{11}(p) \\
W_{12}(p) \\
W_{13}(p)
\end{array} \right] = \left[ \begin{array}{c}
W_{11}(p) \\
W_{12}(p) \\
W_{13}(p)
\end{array} \right] . \]

From the set (15) of the equations we obtain the generalized matrix equation for the electroelastic actuator
\[ \left[ \begin{array}{c}
\xi_1(p) \\
\xi_2(p) \\
\xi_3(p)
\end{array} \right] = \left[ \begin{array}{c}
W_{11}(p) \\
W_{12}(p) \\
W_{13}(p)
\end{array} \right] \Psi_m(p) \]
(17)
where
\[ \xi_1(p) = \left[ \begin{array}{c}
\xi_1(p) \\
\xi_2(p) \\
\xi_3(p)
\end{array} \right] , \quad \Psi_m(p) = \left[ \begin{array}{c}
\Psi_1(p) \\
\Psi_2(p) \\
\Psi_3(p)
\end{array} \right] . \]

Let us find the displacement of the faces for the electroelastic actuator in the stationary regime for the inertial load at \( \Phi(t) = \Phi_{m0} \).

Then we get the static displacement of the faces for the electroelastic actuator
\[ \xi_1(\infty) = \lim_{t \to \infty} \xi_1(t) = \lim_{p \to \infty} p W_{11}(p) \Psi_m(p) = d_{mm} \Psi_m(p) = d_{mm} \Psi_m(p), \]
(18)
the transfer functions in matrix form. Then the transfer functions are the ratio of the Laplace transform of the displacement of the face for the electroelastic actuator and the Laplace transform of the corresponding control parameter or force at zero initial conditions. The system of the equations of the displacements of the faces for the electroelastic actuator has the form
\[ \Xi_1(p) = W_{11}(p) \Psi_m(p) + W_{12}(p) F_1(p) + W_{13}(p) p, \]
where the generalized transfer functions of the electroelastic actuator are
From (21), (22) we obtain the static displacements of the faces of the piezoactuator under the transverse piezoeffect at $m \ll M_1$, $m \ll M_2$ in the form

\[ \xi_e(x) = \lim_{t \to \infty} \xi_e(t) = \lim_{p \to 0^+} \frac{p W_1(p)}{U(p)} \left( U_{\delta\delta} \right) p = d_{33}(\delta) U_0 M_3 / (M_1 + M_2), \]

(24)

\[ \xi_\delta(x) = \lim_{t \to \infty} \xi_\delta(t) = \lim_{p \to 0^+} \frac{p W_2(p)}{U(p)} \left( U_{\delta\delta} \right) p = d_{33}(\delta) U_4 M_1 / (M_1 + M_2). \]

(25)

Let us consider the static characteristics of the piezoactuator from piezoceramics PZT under the transverse piezoelectric effect at $m \ll M_1$ and $m \ll M_2$. For $d_{33} = 2 \cdot 10^{-10} \text{m/V}$, $\delta = 4 \cdot 10^{-2} \text{m}$, $\delta = 2 \cdot 10^{-2} \text{m}$, $U = 50 \text{V}$, $M_1 = 2 \text{kg}$ and $M_2 = 8 \text{kg}$ we obtain the static displacement of the faces of the piezoactuator $\Gamma_1(x) = 160 \text{nm}$, $i_1(x) = 40 \text{nm}$, $i_1(x) + i_2(x) = 200 \text{nm}$.

From (16) we obtain the transfer functions of the piezoactuator with the fixed end and the elastic inertial load so that $M_1 \to \infty$ and $m \ll M_2$ in the following form

\[ W_2(p) = \frac{\Xi_2(p)}{U(p)} = \frac{d_{33}}{1 + C_r/C_0} \left( T_2^2 r^2 + 2 T_2 \xi_1 p + 1 \right), \]

(26)

where the time constant $T_i$ and the damping coefficient $\xi_i$ are determined by the formulas

\[ T_i = \sqrt{M_i / \left( C_r + C_0 \right)}, \quad \xi_i = a \delta C_0 / \left( 3c_0 M \left( C_r + C_0 \right) \right). \]

Let us consider the operation of the piezoactuator from piezoceramics PZT with one face rigidly fixed and the elastic inertial load so that $M_1 \to \infty$ and $m \ll M_2$ for $M_2 = 10 \text{kg}$, $C_{11} = 2.1 \cdot 10^{10} \text{N/m}$, $C_{33} = 0.4 \cdot 10^8 \text{N/m}$ we obtain $T_2 = 2 \cdot 10^{-3} \text{s}$. The experimental and calculated values for the piezoactuator are in agreement to an accuracy of 5%.

**Conclusion**

The structural–parametric models, the decision of the wave equation, the parametric structural schematic diagram, the transfer functions of the electroelastic actuator are obtained using Laplace transform. The parametric structural schematic diagram and the transfer functions of the piezoactuator for the transverse, longitudinal, shift piezoelectric effects are determined from the structural–parametric model of the electroelastic actuator. The transfer functions in matrix form are describes deformations of the piezoactuator during its operation as part of the mechatronics system for the nanotechnology and the ecology research. From the decision of the electroelasticity equation, the wave equation and the features of the deformations along the coordinate axes we obtain the generalized structural–parametric model and the parametric structural schematic diagram of the electroelastic actuator for the mechatronics system and its dynamic and static properties.

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**Conflict of interest**

The author declares there is no conflict of interest.

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