Cutting-Edge Turbulence Simulation Methods for Wind Energy and Aerospace Problems

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Abstract: The availability of reliable and efficient turbulent flow simulation methods is highly beneficial for wind energy and aerospace developments. However, existing simulation methods suffer from significant shortcomings. In particular, the most promising methods (hybrid RANS-LES methods) face divergent developments over decades, there is a significant waste of resources and opportunities. It is very likely that this development will continue as long as there is little awareness of conceptional differences of hybrid methods and their implications. The main purpose of this paper is to contribute to such clarification by identifying a basic requirement for the proper functioning of hybrid RANS-LES methods: a physically correct communication of RANS and LES modes. The state of the art of continuous eddy simulations (CES) methods (which include the required mode communication) is described and requirements for further developments are presented.

Keywords: turbulent flow simulations; Reynolds-averaged Navier–Stokes (RANS) methods; large eddy simulation (LES); hybrid RANS-LES methods; continuous eddy simulations (CES) methods

1. Introduction

Reliable and efficient turbulent flow simulation methods are an essential requirement for future wind energy and aerospace developments. Corresponding developments of simulation methods in time are illustrated in Figure 1. The striking feature is the seemingly slow progress. After developing Reynolds-averaged Navier–Stokes (RANS) equations [1–4] it became clear that this approach has significant disadvantages under conditions where it is impossible to fully explain mechanisms of turbulent flows by pure modeling (which is usually the case for wind energy and aerospace problems involving separated flow). Pure large eddy simulation (LES) methods [1,5–8], aiming at flow resolution in contrast to flow modeling, were developed as an alternative. However, it became clear that these methods result in unaffordable computational cost with respect to most wall-bounded turbulent flows of practical relevance. The conclusion was the need to develop hybrid RANS-LES [5,9–15], which are denoted by RANS+LES in Figure 1 (by taking reference to their conventional design strategy to combine RANS and LES equation elements). Novel hybrid RANS-LES methods, referred to as continuous eddy simulation (CES) methods, were presented recently [15–19]. The latter methods seem to offer major advantages regarding the serious problems of conventional hybrid RANS-LES methods and other methods [20–23].

We see diverging developments of computational methods, in particular, of hybrid RANS-LES methods over decades [15]. It is not difficult to predict that these developments will continue for a long time as long as there is little awareness of very significant differences of various hybrid methods: researchers will pick computational methods for applications...
according to their simplicity and availability of codes. Given the variety of available methods and model options, systematic evaluations of the suitability of using different model strategies are simply not feasible. To overcome this stagnation of the development of computational methods for turbulent flow predictions and related consequences for wind energy and aerospace applications, it needs clarification on which conceptual features are required to ensure the proper functioning of hybrid RANS-LES methods.

Figure 1. An illustration of the development of computational methods for turbulent flows in time. Here, RANS+LES refers to conventional hybrid RANS-LES methods and CES refers to the CES hybrid RANS-LES methods described here. On the right-hand side (in red), there are requirements for further clarification of advantages of CES methods in relation to previously developed methods (RANS, LES, and RANS+LES).

The latter is the main question addressed in this paper. In particular, the goal is to compare two options as illustrated in Figure 2: the option to continue with diverging developments of hybrid RANS-LES methods without paying much attention to conceptual differences, and the option to overcome conceptual problems of existing RANS-LES methods. As a basis for further discussions, typical hybrid RANS-LES methods will be described in Section 2. Then, requirements for hybrid RANS-LES methods arising from theory and applications will be pointed out in Sections 3 and 4. Remaining challenges are described in Section 5, and conclusions will be presented in Section 6.

Figure 2. In extension of Figure 1, an illustration of options regarding the future use of hybrid RANS-LES methods including characteristic advantages (+) and disadvantages (−).

2. Typical Hybrid RANS-LES Methods

Let us consider four hybrid RANS-LES models to prepare the comparison of conceptual features of CES models with three typical hybrid RANS-LES models in following sections.
Several turbulence model structures can be considered for that. Here, the $k - \epsilon$ model will be considered as a frame. The model is given by the incompressible continuity equation $\partial \tilde{U}_i / \partial x_i = 0$ (the sum convention is used throughout this paper) and momentum equation

$$\frac{D \tilde{U}_i}{Dt} = -\frac{\partial (\tilde{p}/\rho + 2k/3)}{\partial x_i} + 2\frac{\partial (\nu + \nu_t) \tilde{S}_{ij}}{\partial x_j}.$$  

(1)

$D / Dt = \partial / \partial t + \tilde{U}_i \partial / \partial x_i$ denotes the filtered Lagrangian time derivative. $\tilde{U}_i$ refers to the $i$-th component of the spatially filtered velocity. We have the filtered pressure $\tilde{p}, \rho$ is the constant mass density, $k$ is the modeled energy, $\nu$ is the constant kinematic viscosity, and $\tilde{S}_{ij} = (\partial \tilde{U}_i / \partial x_j + \partial \tilde{U}_j / \partial x_i) / 2$ is the rate-of-strain tensor. The modeled viscosity is given by $\nu_t = C_\mu k \tau = C_\mu k^2 / \epsilon$. Here, $\epsilon$ is the modeled dissipation rate of modeled energy $k$, $\tau = k / \epsilon$ is the dissipation time scale, and $C_\mu$ is a model parameter. For $k$ and $\epsilon$, we consider the transport equations

$$\frac{D k}{Dt} = P - \epsilon + D_k, \quad \frac{D \epsilon}{Dt} = C_{\epsilon_1} \frac{e^2}{\epsilon} \left( \frac{P}{\epsilon - \alpha} \right) + D_\epsilon.$$  

(2)

The diffusion terms are given by $D_k = \partial [\nu (v + \nu_t) \partial k / \partial x_i ] / \partial x_i, D_\epsilon = \partial [\nu (v + \nu_t) \partial \epsilon / \partial x_i ] / \partial x_i$, and $P = \nu_S S^2$ is the production of $k$, where $S = (2\tilde{S}_{mn} \tilde{S}_{mn})^{1/2}$ is the characteristic shear rate. $C_{\epsilon_1}$ is a constant with standard value $C_{\epsilon_1} = 1.44$, and $\sigma_\epsilon = 1.3$. For the RANS case considered first, the parameter $\alpha = C_{\epsilon_1}/C_\epsilon$, where $C_{\epsilon_1} = 1.92$ [2] implies $\alpha = 1.33$.

One of the most popular hybrid RANS-LES models (preferred because of its original simplicity) is detached eddy simulation (DES) [13,14,24–36]. Reviews of DES were provided by Spalart [31] and Mockett et al. [14]. DES-type hybridizations can be applied to many turbulence model structures. Here, it is used in regard to the $k - \epsilon$ turbulence model structures. Here, it is used in regard to the model equation of Equation (2),

$$\frac{D k}{Dt} = P - \psi_{DES} \epsilon + D_k, \quad \psi_{DES} = \max \left( 1, \frac{k^{3/2}}{C_{DES} \Delta \epsilon} \right).$$  

(3)

The $k$ equation is modified here via introducing $\psi_{DES}$. $C_{DES}$ is a constant which is often considered to be $C_{DES} = 0.65$, and $\Delta$ refers to the filter width used in LES mode. The model switches between length scales, the RANS length scale $L = k^{3/2} / \epsilon$ and the LES length scale $\Delta$. The mechanism of switching to the LES mode is to increase the dissipation of $k$, leading to lower $k$ values and a reduced modeled viscosity. The DES concept is very attractive: it is simple, also regarding its implementation, and there is only one control term that enables the simulation of resolved motions.

Different ways to hybridize the equations considered are given by partially averaged Navier–Stokes (PANS) [37–45], and partially integrated transport modeling (PITM) methods [20–23,46–52]. These methods focus on the $\epsilon$ equation in Equation (2),

$$\frac{D \epsilon}{Dt} = C_{\epsilon_1} \frac{e^2}{\epsilon} \left( \frac{P}{\epsilon - \alpha^*} \right) + D_\epsilon, \quad \alpha^* = 1 + (\alpha - 1) k_+,$$  

(4)

where $\alpha^*$ is parameterized in terms of the ratio $k_+ = k / k_{tot}$ of modeled ($k$) to total ($k_{tot}$) kinetic energy. Within PANS, $k_+$ is provided by a prescribed value, whereas in PITM, $k_+$ is parameterized [32] involving the filter width $\Delta$, the total turbulence length scale $L_{tot}$ (see below), and the Kolmogorov constant $1.53$. The mechanism of this approach is to reduce the dissipation of $\epsilon$ in LES mode leading to higher $\epsilon$ and a reduced modeled viscosity.

Another way of hybridizing the equations considered is given by the CES approach [15–18]. This concept implies the equations (see details given in the Appendix A)

$$\frac{D \epsilon}{Dt} = C_{\epsilon_1} \frac{e^2}{\epsilon} \left( \frac{P}{\epsilon - \alpha^*} \right) + D_\epsilon, \quad \alpha^* = 1 + (\alpha - 1) L_+^{2},$$  

(5)

$L_+ = L / L_{tot}$ refers to the turbulence length scale resolution ratio involving modeled ($L$) and total contributions ($L_{tot}$). The modeled contribution is calculated by $L = \langle k \rangle^{3/2} / \langle \epsilon \rangle$ (the brackets refer to averaging in time). The total length scale is calculated correspondingly by
The model determines the modeled viscosity \( \nu \), which controls the amount of resolved motion (it needs to be sufficiently small such that fluctuations can develop). The model receives information about the amount of resolved motion via the resolution indicator \( L_+ \). Accordingly, the model modifies its contribution to the simulation, see the illustration in Figure 3. On the other hand, most other hybrid methods (like DES or PANS described above) do not include such a communication mechanism between modeled and resolved motions, this means these methods suffer from the serious problems described in the preceding paragraph.

It is worth noting that the availability of the mode communication mechanism is not the only requirement. This mechanism needs to be designed in line with the physics of non-homogeneous turbulent flows, i.e., concepts applicable to homogeneous flows may be inapplicable to non-homogeneous flows. CES methods have this property, whereas PITM methods apply the mode communication concept for homogeneous flows. The latter implies obvious shortcomings in regard to applications to non-homogeneous flows.

\[
L_{tot} = \frac{k_{tot}^3}{\epsilon_{tot}}. \quad \text{Here, } k_{tot} = \langle k \rangle + k_{res} \text{ is the sum of modeled and resolved contributions, where the resolved contribution is calculated by } k_{res} = \langle (U_i U_i) - \langle U_i \rangle \langle U_i \rangle \rangle / 2. \quad \text{Correspondingly, } \epsilon_{tot} = \langle \epsilon \rangle + \epsilon_{res}, \text{ where } \epsilon_{res} = v \left( \frac{\partial U_i}{\partial x_j} \frac{\partial U_i}{\partial x_j} \right) \left( \frac{\partial U_i}{\partial x_j} \right) \left( \frac{\partial U_i}{\partial x_j} \right) \right). \quad \text{The functioning of this approach is similar to the functioning of PITM methods, although there is the relevant difference given by the } a^* \text{ applied. It is worth noting that CES methods can be applied in a variety of ways. It is possible to use these methods in regard to several turbulence models, and there are different ways to hybridize the equations (the hybridization can be accomplished by focusing on the } k \text{ equation, as considered within the DES concept).}
\]
example, there are discrepancies between the imposed resolution and the actual resolution seen in simulations, and resolution indicators show an unphysical behavior near walls.

Figure 3. An illustration of the interplay of modeled motion (indicated by ellipses of different size) and resolved motion (indicated by blue and red turbulent fluctuations). The mode coupling mechanism is illustrated by red lines on the right. The grid is illustrated by vertical and horizontal black lines. On the left, there is a case with high flow resolution, on the right there is a case with almost no flow resolution.

The conclusion of the discussion in this section is that (in contrast to other methods) CES methods can properly deal with requirement R1 described in Figure 1. CES methods also can properly deal with the requirements R2 and R3. Resolution requirements of LES are avoided by the fact that the LES filter width is not explicitly involved, which constrains LES to the use of sufficiently fine grids. In contrast to pure RANS, the inclusion of $\alpha^* = 1 + (\alpha - 1)L^2_2$ in the equations considered enables the generation of resolved motion in RANS, which is known to be highly beneficial for separated flows.

4. Application Requirements: CES Versus Other Methods

In general, the validation of hybrid RANS-LES methods represents a daunting task given the variety of flow configurations, grid configurations, turbulence models, and hybrid versions that should be considered [15,18]. However, there are differences in this regard between conventional hybrid RANS-LES and CES methods. After addressing these differences in Section 4.1 we present in Sections 4.2 and 4.2 CES application results and related conclusions with respect to the requirements R1–R3 described in the last paragraph of Section 3 and Figure 1.

4.1. Application Requirements: CES Versus Other Methods

Obviously, the most relevant requirement for hybrid RANS-LES methods is evidence for their proper performance in applications. With respect to conventional hybrid RANS-LES (which do not apply any mode communication), this question is usually addressed by model evaluations based on experimental or resolved LES data, often combined with the demonstration of advantages compared to RANS or under-resolved LES. Such validation studies are demanding: they need to be performed for a variety of grids (e.g., because of the known grid sensitivity of DES), and usually they involve the consideration of several flow configurations. In addition, such studies may suffer from questions about the applicability of LES and experiments used for validation for high Re complex flows. The basic validation concept may be seen as validation by examples (application to different flow configurations).

The validation of a hybrid RANS-LES model with ability to properly deal with the mode communication described above is different from the validation of models that do not address the mode communication due to the model’s inherent validation ingredient given by using the model in resolving mode [16]. Every hybrid simulation can be validated by using the model in resolving LES mode. This sort of validation may be seen as validation by theory, this means the concept is much more general than the validation by examples concept that needs to be used for existing approaches. Therefore, in a strict sense, model
applications serve as illustrations of the proper functioning of the concept in contrast to their role regarding the evaluation of existing hybrid models: the demonstration of the model’s ability to properly work for a few benchmark cases (to deal with the requirements R1–R3 described in Figure 1) appears to be appropriate to support further model applications. In addition to the reduced number of cases that need to be considered in applications, it is worth noting that the requirements for grid sensitivity studies is also reduced for the models considered here because of the model’s inherent stability with respect to grid variations (in contrast to DES).

Based on the facts described in the preceding paragraph, we will demonstrate the proper functioning of CES methods by applications to periodic hill flows [17] compared to available LES results. Key results of such applications will be described next. Although not shown here, very similar findings were found with respect to applications of the models considered to the NASA hump flow [15,55]. A relevant conclusion obtained by these periodic hill flow investigations was the equivalence of hybrid models: as long as the same analysis is applied to a turbulence model considered, different hybrid models were found to perform equivalently. The latter will be confirmed in the following by considering three hybrid $k − \omega$ two-equation models presented in Table 1. These models can be obtained by the same analysis as used in the Appendix A.

Table 1. Overview of CES models considered in [17]. CES-KO refers to CES performed with the $k − \omega$ model. In particular, -KOS (-KOK, -KOKU) refer to CES-KO performed by modifying the scale equation (the $k$ equation, the turbulent time scales in the $k$ equation according to unified RANS-LES).

| Model   | CES Hybrid Equations | Mode Control |
|---------|----------------------|--------------|
| CES-KOS | $\frac{Dk}{Dt} = P - \epsilon + D_k$ | $\frac{D\omega}{Dt} = C_{\omega 1} \omega^2 \left( \frac{P}{\epsilon} - \beta^* \right) + D_\omega + D_{\omega c}$ | $\beta^* = 1 + L^1 \beta - 1$ |
| CES-KOK | $\frac{Dk}{Dt} = P - \psi \beta \epsilon + D_k$ | $\frac{D\omega}{Dt} = C_{\omega 1} \omega^2 \left( \frac{P}{\epsilon} - \beta \right) + D_\omega + D_{\omega c}$ | $\psi_\beta = 1 - \beta - \beta^*$ |
| CES-KOKU | $\frac{Dk}{Dt} = \psi_\beta^{-1} \left( P - \psi_\beta \beta \epsilon + D_k \right)$, $\frac{D\omega}{Dt} = C_{\omega 1} \omega^2 \left( \frac{P}{\epsilon} - \beta \right) + D_\omega + D_{\omega c}$ | $\psi_\beta = \psi_\beta$ |

4.2. Periodic Hill Flow Simulations

The CES approach described above was used to simulate periodic hill flows [17] on several grids at different Reynolds numbers in comparison to measurements of Rapp and Manhart [56], which are available for $Re = 37$ K. The Reynolds number is based on the hill height $h$ and bulk velocity $U_b$ above the hill crest, which is located at $(x/h, y/h) = (0, 1)$. The height $h$ and $U_b$ were used for the definition of non-dimensional variables presented. The flow configuration is illustrated in Figure 4, essential simulation data are given in Table 2.

Figure 4. Streamwise velocity fluctuations $u$ (xz-plane, $y/h = 0.5$) using the CES-KOKU model on the 500 K grid: $Re = 37$ K (left), 100 K (middle), 500 K (right).
Table 2. Periodic hill flow simulation set-up data.

| Simulation Setting | Setting Applied |
|--------------------|-----------------|
| domain size        | $L_x = 9\ h, L_y = 3.035\ h, L_z = 4.5\ h$ in streamwise $x$, wall normal $y$, and spanwise $z$ directions, where $h$ is the height of the hill |
| boundary conditions| bottom and top: solid walls, no-slip and impermeability boundary conditions; streamwise and spanwise directions: periodic boundary conditions |
| time step          | so that maximum CFL number is 0.5, averaged CFL number is about 0.1 |
| filter width       | $\Delta_{vol} = (\Delta_x \Delta_y \Delta_z)^{1/3}$; $\Delta_x, \Delta_y, \Delta_z$ represent the filter width in $x, y, z$ directions |
| averaging          | after 20 flow-through times (FTT), averaging over 240 FTT, averaging along $z$ |
| grids              | 500 K cells: $N_x \times N_y \times N_z = 128 \times 80 \times 48$; 250 K cells: $N_x \times N_y \times N_z = 102 \times 64 \times 38$; 120 K cells: $N_x \times N_y \times N_z = 80 \times 50 \times 30$ |
| Reynolds numbers   | $Re = (37, 100, 500)\ K$ |

All the simulations have been performed by using the OpenFOAM CFD Toolbox [57]. The combinations of $Re$ cases and grids enabled the analysis of cases where there was an almost complete resolution (as given for $Re = 37\ K$ using the finest 500 K grid) and a minimum of flow resolution (as given for $Re = 500\ K$ using the coarsest 120 K grid). Figure 5 shows the influence of $Re$ variations on the 500 K grid. The case of almost complete flow resolution is on the left. The increase in $Re$ implies a decreasing flow resolution, which is reflected by less details regarding the representation of instantaneous turbulent flow structures.

Figure 5. The hill flow geometry.

Figure 6 addresses the suitability of three CES models presented in Table 1 with respect to the almost resolving, $Re = 37\ K$ (500 K grid) case. It may be seen that the flow variables $\langle U \rangle / U_b$, $\langle uu \rangle / U_b^2$, and $\langle uv \rangle / U_b^2$ obtained by these models agree very well with the measurements (the measured stresses are known to be slightly under-predicted [17,58,59]). In particular, all three models perform almost equally. This observation is clearly relevant. It means that rather different models are capable to realize about the same flow resolution as long as the mode communication mechanism is set up in the same, physically correct way. The flow resolution indicators $k_+, L_+$, and $\epsilon_+$, the corresponding modeled to total variables, are related to each other by $L_+ = k_+^{3/2} / \epsilon_+$. It may be seen that the assumption $\epsilon_+ = 1$, which is often applied in PANS and PITM modeling approaches, represents a reasonable approximation for the well resolving case considered with the exception of the most relevant near-wall regions. It is remarkable to see the relatively uniform variation of $k_+$, $L_+$, and $\epsilon_+$ in space, showing a relatively uniform flow resolution over most of the domain. As required, $k_+$ and $L_+$ increase in the near-wall region because of the increasing amount of flow modeling (the $k_+$ and $L_+$ values indicate that the models work approximately in LES mode away from walls and in almost RANS mode close to walls). Minor differences between the three CES models considered can be observed. The latter is plausible because of the fact that different model equations are applied. However, based on the flow variable profiles we conclude that the effect of such very minor variations of resolution indicators on the actual flow simulation is negligible.
Figure 6. Profiles of flow variables and resolution indicators obtained by periodic hill flow simulations for the $Re = 37 K$ (500 K grid case) at $x/h = 4$. CES models applied are specified in the plots. The first row shows the mean velocity, streamwise normal, and Reynolds shear stress. The resolution indicators $k_+^+, L_+^+, \text{ and } \epsilon_+^+$ are shown in the second row.

Figure 7 addresses the core problem: the functioning of the mode communication to accomplish a proper redistribution of modeled and resolved motions. The expectation is simple: a coarser grid and a higher $Re$ need to imply an increased amount of resolved motion. Reynolds number effects in consistency with Figure 5 cases are considered in the first row. The experimental data involved only apply to the $Re = 37 K$ case. It may be seen that the mean streamwise velocity tends toward a more uniform spatial distribution. The $L_+^+$ plots show the essential model feature. As required, an increased $Re$ leads to higher $L_+^+$ values (i.e., a decreased amount of flow resolution). The latter also applies to the near-wall region. Grid effects are considered in the second and third row. It is essential to note that grid effects have little influence on the flow simulation (again, experimental data only apply to the $Re = 37 K$ case). The almost complete modeling case is involved here for $Re = 500 K$, the coarsest 120 K grid. It is remarkable that such RANS-type simulations imply mean streamwise velocity profiles that hardly differ from higher resolved cases. The $L_+^+$ plots show the required model behavior, $L_+^+$ increases (the amount of flow modeling increases) if the grid becomes coarser, as may be seen very well in the zoomed-in plots close to the upper wall.
4.3. Summary

Based on the results presented in Section 4.2, the following conclusions can be drawn regarding the requirements R1–R3 described in Figure 1.

R1. Most importantly, based on an exact solution to the problem considered [16–18], it is shown that the CES method can deal with the required mode balance with respect to both, significant Reynolds number and grid variations. The model performance is hardly affected by redistributions of resolved and modeled motions. The hybridization mechanism works almost equivalently for different turbulence model structures;

R2. The simulations include an almost complete resolution for $Re = 37$ K (500 K grid). An excellent model performance in comparison to measurements is found. Hence, RANS-type equation can provide flow resolution without involving the LES length scale $\Delta$;

R3. The simulations also include the limit of almost no resolution by the $Re = 500$ K (120 K grid) case. Because of significant and stable fluctuations that do not get extinguished under such very coarse grid conditions, the model still correctly reflects the physics of flow separation.

Figure 7. Profiles of $\langle U \rangle / U_b$, overall and upper wall $L_+$ obtained by CES-KOKU simulations at $x/h = 4$. $Re$ and grids applied are specified in the plots. First row: $Re$ effects using the 500 K grid. Second and third row: grid effects for $Re = (37, 500)$ K. Reprinted with permission from Ref. [17]. Copyright 2020 AIP Publishing.
5. Some Challenges

The theoretical and computational features of CES methods reported above are very promising. However, current computational experience is limited to the analysis of periodic hill flows reported in Section 4 and confirmation that these models also properly work for the more complex NASA hump flow \[15,55\] (not shown). Obviously, applications to different and in particular more complex flows would be highly beneficial to build further confidence in the ability of these hybrid methods. Although complex applications like comprehensive wind farm simulations are finally the goal, it is fair to note that applications of intermediate complexity represent a very valuable step toward wind farm simulations \[60–65\]. The consideration of non-periodic flows around realistic obstacles (e.g., airfoil-type simulations) involving flow separation and a broad range of turbulence regimes appears to be well appropriate to serve this purpose. Another type of equally important applications is given by the demonstration that the CES methods considered are capable of providing a basic solution for the so called Terra Incognita problem: see \[19\] and the references therein.

There are specific challenges in extension of the requirements R1–R3 described in Figure 1 and beyond these requirements that need to be addressed in such additional applications.

C1. The core component of CES hybrid methods, the stable functioning of the mode communication mechanism, needs further investigations with respect to grid variations. Applications involving abrupt grid changes inside domains (LES-type regions embedded by RANS-type regions) are beneficial to attain a more comprehensive understanding of the stability of the generation mechanism of turbulent fluctuations. Such simulations represent the core component of atmospheric mesoscale to microscale couplings (which are closely related to the Terra Incognita problem \[19\]);

C2. Under many circumstances LES have to be performed on relatively coarse grids, leading to the question of how well resolving the LES actually is \[54\]. In this regard, the use of CES methods as resolving methods is highly attractive because these methods are not constrained by strict LES resolution requirements (the use of grids which ensure that the LES filter width is smaller than typical large-scale turbulence structures). It needs specific comparisons between relatively well resolving LES and CES methods in this regard;

C3. Their computational efficiency makes RANS-type simulations very attractive, for example for ABL simulations involving wind turbines. However, the inability of RANS to deal with flow separation hampers such simulations significantly. Existing periodic hill flow simulations indicate that CES used on the same grids as RANS has the potential to deal with this problem in a much more appropriate way, the question is whether the same conclusion can be drawn for more complex flows. Due to the low level of flow resolution under such conditions, it appears to be beneficial to implement a dynamic calculation of model parameters in CES methods \[58,59,66–70\], which is currently not accomplished;

C4. CES methods were currently only applied to turbulent velocity fields. The extension to scalar field simulations is clearly desirable to deal, for example, with atmospheric chemistry problems. The theoretical extension of CES methods to passive scalar field simulations was presented recently in \[18\]. It turned out that the analysis technique applied for velocity fields is capable to also provide a corresponding hybridization of scalar field simulations (which is the same as long the ratio of turbulence and scalar time scales is constant). However, applications of these methods do not exist so far, they are required to build confidence in the treatment of scalar transport in this way;
C5. The structure of CES methods applied so far corresponds to the structure of eddy viscosity type models, which are known to have deficiencies for complex turbulent flow applications. As shown in [18], it is possible to generalize such eddy viscosity type formulations to corresponding Reynolds stress transport models and even their underlying probability density function transport equations (the latter applies to both turbulent velocities and scalars). The use of eddy viscosity type methods for the periodic hill flows considered did not reveal any need for more complex turbulence models. However, there is the question of whether the same conclusion is obtained with regard to more complex flow simulations;

C6. The core motivation of developing hybrid RANS-LES methods is to build the reliable capability to predict turbulent flows at very high Reynolds numbers that cannot be studied or correctly predicted otherwise. The challenge is to provide convincing evidence that the CES methods considered are able to fulfill this task. This has to be completed in comparison to usually applied hybrid RANS-LES (for example, in comparison to DES) in order to demonstrate the different capabilities of methods.

6. Conclusions

Arguably, DES and its improved versions are regularly applied for turbulent flow simulations by researchers if there is the need to use a hybrid RANS-LES method (if RANS simulations are known to fail). The basis of using DES or similar hybrid methods is the availability of DES implementations (the simplicity of its implementation) combined with the understanding that the inclusion of resolved motion is often helpful. There is, however, a price to pay for this strategy. There is a high grid dependence of DES and similar hybrid methods, which results in the need for validation data and in the need to search for the grid appropriate for the application considered. In general, the concept for justifying conventional RANS-LES (validation by examples, i.e., validation by applications to different flow configurations) is expensive and without any guarantee to produce reliable results for very high $Re$ complex turbulent flows.

The reason for these serious problems of usually applied hybrid RANS-LES was identified here: it is the missing mode communication mechanism in these methods. It was also shown that CES methods do not suffer from this problem. Both theoretical features and existing applications support the suitability of this approach. Relevant advantages are given by the opportunities to perform reliable (LES-type) resolved simulations and computationally cheap RANS-type simulations that still stably involve unsteadiness, which is known to be very beneficial for separated turbulent flows simulations. With respect to routine applications there is the minor disadvantage that the mode communication needs to be implemented (via a corresponding extension of RANS codes). The very significant advantage is the stable grid dependence, which is the key for reliable predictions of complex turbulent flows at very high $Re$.

Requirements for further developments and applications of CES methods were pointed out. In regard to reliable predictions of very high $Re$ complex turbulent flows (under conditions where other techniques like experiments and resolved LES are inapplicable because of their resolution requirements), there is the simple question of what would be a reasonable alternative to further developments of CES methods.

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Appendix A. CES Derivation

To hybridize Equation (2), we consider a variable $\alpha^*$ (instead of a constant $\alpha$) combined with an appropriate calculation of $\alpha^*$. Then, we consider variations of model parameters ($\alpha^*$) and related variations of model variables (like $k$ and $\epsilon$): the question is which model coefficient satisfies variation equations implied by the turbulence model considered. The approach applied below follows a recently presented approach [16]. The technical framework applied to derive these results was provided by an analysis of Friess et al. [32]. The significant difference to the latter findings is that Friess et al. focused on a different question: for given PANS/PITM-type relations between model coefficients and resolution indicators, they determined equivalence criteria for hybrid methods based on other turbulence models.

By replacing $P$ in the $\epsilon$ Equation (2) according to $DK/Dt = P - \epsilon + D_k$ and taking the variation (denoted by $\delta$) of this $\epsilon$ equation we obtain the exact relation

$$\delta \alpha^* = Q + \frac{D_k}{\epsilon} [\delta D_k / D_k - \delta \frac{\epsilon}{k} + \frac{\delta k}{k} - \frac{\delta \epsilon}{\epsilon}] + (\alpha^* - 1) \left[ \frac{\delta D_k}{D_k} + \frac{\delta k}{k} - \frac{2}{\epsilon} \left( \frac{\delta \epsilon}{\epsilon} \right) \right],$$

(A1)

where the $\epsilon$ Equation (2) is applied to replace $D_\epsilon$. Here, we introduced the abbreviation

$$Q = \frac{1}{\epsilon} \frac{D_k}{D} \left[ \frac{\delta \epsilon}{\epsilon} - \frac{\delta k}{k} - \frac{\delta D_k}{D_k} \right] + \frac{1}{\epsilon} \frac{D_k}{C_{\epsilon l}^2} \left[ \frac{\delta D_k}{D_k} - \frac{\delta \epsilon}{\epsilon} \frac{D_\epsilon}{D} \right].$$

(A2)

The coefficient $\alpha^*$ calculated via Equation (A1) should not depend on $Dk/Dt$ or $D\epsilon/Dt$. This corresponds to the neglect of $Q$, i.e., $Q = 0$. Additionally, the coefficient $\alpha^*$ should not be proportional to the turbulent transport term $D_k$, which does not reflect a systematic influence on the relationship between $\alpha^*$ and variables characterizing the flow resolution.

Thus, the second term in Equation (A1) should be equal to zero. It turns out that this is the case if we follow Friess et al. [32,71], we assume that variations of $\delta k/k$ and $\delta \epsilon/\epsilon$ in space can be neglected. This means we look for model coefficient variations that produce a uniform relative variation of the energy partition over the domain. In the first order of approximation, we find $\delta D_k / D_k = 3\delta k/k = \delta \epsilon/\epsilon$ and $\delta D_\epsilon / D_\epsilon = 2\delta k/k$. In contrast to corresponding relations reported by Friess et al. [32], these expressions include $\epsilon$ variations. Hence, the second term in Equation (A1) disappears. We find then

$$\delta \alpha^* = (\alpha^* - 1) \left[ 3 \frac{\delta k}{k} - 2 \frac{\delta \epsilon}{\epsilon} \right].$$

(A3)

This relation is equivalent to

$$\frac{\delta \alpha^*}{\alpha^* - 1} = 2 \frac{\delta L}{L},$$

(A4)

which involves $L = k^{3/2} / \epsilon$. This equation can be integrated from the RANS state to a state with a certain level of resolved motion, $\int_0^{\alpha^*} dx / (x - 1) = 2 \int_{\nu_{\text{rid}}}^{L} dy / y$. We obtain in this way

$$\alpha^* = 1 + L^2 \left( \alpha - 1 \right).$$

(A5)

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