Research Article

Energy, Exergy Analysis, and Optimizations of Collector Cover Thickness of a Solar Still in El Oued Climate, Algeria

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Researches in many laboratories on solar still desalination are concerned with increasing efficiency using only solar energy. One of the techniques is the difference in the thickness of the glass cover of the distiller. In order to see the influence of this parameter on efficiency, three similar stills with three different glass coverings were investigated. The flow of heat goes through the cover, and higher glass temperature leads to solar still productivity becoming lower. This paper presents an optimization of glass thickness (Gt) of a conventional solar still (CSS) in El Oued climate, Algeria. Based on the experimental results, the distilled water production rate, energy, and energy efficiency of the CSS have been discussed. The results showed that the suitable Gt of the CSS was 3 mm. The distilled water of around 3.15, 2.02, and 1.13 kg was produced by the CSS at energy efficiency of 30.71, 19.02, and 11.44% with the Gt of 3, 5, and 6 mm, respectively. The daily average exergy efficiency of 2.46, 1.38, and 0.84% was calculated for the CSS at Gt of 3, 5, and 6 mm, respectively.

1. Introduction

Solar distillation is one of the simplest, economical, and environmental solutions with no emission of toxic or dangerous gas [1–3]. This technique is used in many countries around the world, especially in the isolated areas. As its name indicates, it is based on solar energy which is renewable and free energy [4–7]. The solar distiller works according to the laws of heat and mass transfer [8]. Improving the productivity of pure water from a conventional solar distiller is the subject of research in several laboratories [9, 10]. This improvement is based on two main parameters, solar radiation and the gradient of temperature between water and glazing. In addition, exergy, energy, and technoeconomic assessments have been widely carried out to monitor the performance of solar-based conversion systems such as solar distillers to demonstrate their feasibility [11–18]. To prove the first parameter, an experiment was performed in the summer season and in the winter season to see the effect of solar radiation on traditional distillation production; the results show 89.44% increase in productivity during the summer season [19]. Another experiment used reflectors to focus the solar radiation that passes through the still’s glass; the rate of improvement was approximately 30% [20].

The lenses were also used to increase a traditional solar still’s performance, and the result obtained was 638% [21].
To prove the second parameter, double glazing was used in a solar still to reduce temperature gradient, and the results show 55.7% reduction in productivity [22]. There have also been many studies on the glazing itself, for example, a study which tested the cooling of glass evaluated the productivity of the solar stills. The experimental results revealed that the pure water productivity is 10,06 L/m²-day [23]. Another study shows that efficiency of solar distiller glass with water cooling was 11.81% more efficient than the solar still without water cooling [24].

According to an experimental study, cooling of glass covers can also be achieved by flowing air over the glass cover [25]. The use of partial shading on a CSS gave 12% improvement in productivity, a result confirmed by an experimental study carried out in southern Algeria [26]. Some researchers have tried to adjust the glazing angle to maximize pure water production; they have found that the best angles vary between 20° and 30°. The best glazing angle was 30° with an output of 3517 kg/m² in the autumn season and 3633 kg/m² in the winter; the best inclination was 20° with a yield of 5224 kg/m² in the spring season and 4527 kg/m² in the summer season [27].

The variation of 3 different glazing thicknesses (4, 6, and 8 mm) was published by Panchal and Shah [28]. The outcome of the study indicates that the best pure water production corresponds to the 4 mm thickness. The results of an experiment revealed that while the thickness of the glass cover of a solar still reduces to 3.5 mm, the average output of pure water from the still was 31.13% higher than that from a solar still with a 4 mm thick glass cover [29]. Several experiments were reported on the effect of different glass covers thickness on a solar still performance [30–32].

A theoretical and experimental study was done on three different cover thicknesses to determine the optimum thickness of the solar still glass cover. The results obtained show that a glass cover thickness of 4 mm and 5 mm has 27 and 12% higher production as compared to a thickness of 6 mm [33]. Two solar stills with two glass lids with different thicknesses of 4 mm and 6 mm were tested; the best productivity of 1.60 liters from the still with the 4 mm glazing and 1.55 liters from the still with the 6 mm glazing was reported [34, 35]. The heat flow through the glass of a CSS has been studied recently [36]. In another study, 5 glass covers were tested (2, 6, 10, 14, and 18 mm); the result indicates that optimum thickness that offers the highest pure water productivity is 6 mm [37]. The influence of a large glazing thickness on the performance of a solar still was studied by Khechekhouche et al. [38]. The cover consists of two 3 mm thick glass plates separated by a 100 mm air layer, forming an air chamber isolated from the sides. The results show that this technique minimizes the efficiency of the distiller by 56.52% compared to the CSS [38].

From the detailed literature, it has been identified that only few works have been reported on the effect of glass thickness on energy and exergy efficiencies of a solar still. Hence, the present work consists of making an experiment with three conventional solar stills, and each still has a glazing with a thickness that differs from the other distillers for the climatic condition of the El Oued climate, Algeria. The objective is to optimize the thickness in order to obtain the best productivity in pure water. In this work, calculation of energy, exergy efficiency, and fractional exergy of evaporation and convection between $T_{sw}$ and $T_{cc}$ is a novelty of the present study.

### 2. Experimental Set-Up and Methodology

#### 2.1 Protocol of the Experiment.

The city of El Oued southeast of Algeria’s geographical coordinates is 33.3676° north latitude and 6.8516° longitude. The experiment was carried out several times, and since the results obtained are the same, one sunny day (25th May 2018) was considered. K-1 type thermocouples were placed on solar stills and precisely in the water, below and above the glass cover, and in the open place to measure the ambient temperature. The glass side of the two stills faces south for maximum exposure to solar radiation. The quantities of distilled water produced were measured manually every hour by a graduated cup. Temperatures were measured every hour from 9:00 AM to 4:00 PM. The meteorological condition of the experimental day is shown in Table 1.

| Sun lever | 5:41 |
| Sunset | 19:19 |
| Ambient temperature | 35 (°C) |
| Humidity | 25% |

|  |  |
|---|---|
| **Table 1: Meteorological conditions.** |  |

#### 2.2 Solar Distiller Description.

Figure 1 shows the schematic of the CSS with location of thermocouples. This system helps to purify polluted water and restore it to pure water. It is a very simple technique which uses energy from solar radiation. Nevertheless, this type of system is distinguished from other desalination devices by its relatively low efficiency. This distiller has a very simple construction with a low investment cost. The distiller used in our experiment consists basically of a square wooden box of 0.5 × 0.5 m, a transparent glass cover of 0.55 × 0.55 m, a thickness of 0.003 m, a PVC recovery tube of 0.60 m in length and 0.025 m in diameter situated in the lower part of the distillery, and finally a pure-water storage tank.

#### 2.3 Treatment Data.

The experiment was performed in the climate of El Oued region, in southeast Algeria, on 25th May 2018. Three separate solar stills have the same size with the same inclination angle (20° with the horizontal). At the same time, all the three distillers (CSS with the G, of 3, 5, and 6 mm) are exposed to the sun on the same experimental day with similar water quality and quantity. Each distiller has a thickness covering that differs from other distillers as shown in Figure 2.

### 3. Results and Discussion

#### 3.1 Hourly Variations of Solar Intensity and Ambient Temperature.

Figure 3 shows the hourly variations of solar
intensity and ambient temperature. From the graph, it is clear that both the values increase linearly and reached the maximum value at 1 PM, and after 1 PM, it has decreased. The maximum solar intensity and an ambient temperature of 1005 W/m² and 36°C has been recorded. The daily average solar intensity and ambient temperature during the testing have been recorded as 754.38 W/m² and 32.25°C, respectively. During the testing, the sky was very clear and there were no clouds.

3.2. Hourly Variations of Water Temperature. Figure 4 shows the hourly variations of water temperature ($T_W$) for the CSS at $G_t$ of 3, 5, and 6 mm. $T_W$ has a similar curve like solar intensity and ambient temperature since the water temperature is directly proportional to the solar intensity. The maximum $T_W$ of 60, 58, and 55°C has been recorded for the CSS at $G_t$ of 3, 5, and 6 mm, respectively. The daily average $T_W$ of 49.5, 47.88, and 46°C has been calculated for the CSS at $G_t$ of 3, 5, and 6 mm, respectively. From the experimentation of the CSS at different $G_t$, the maximum $T_W$ obtained for the $G_t$ of 3 mm is found. The CSS at $G_t$ of 5 and 6 mm reduces the daily average $T_W$ by 3.28% and 7.07%, respectively, as compared to the CSS at $G_t$ of 3 mm. Compared to the other two stills,
the water temperature in the CSS at $G_t$ of 3 mm is highest, and it is known that any rise in the water temperature causes an increase in evaporation.

### 3.3. Hourly Variations of Glass Temperature

Figure 5 shows the hourly variations of glass temperature ($T_G$) for the CSS at $G_t$ of 3, 5, and 6 mm. $T_G$ also has a similar curve like solar intensity. The maximum $T_G$ of 49, 51, and 54°C has been recorded for the CSS at $G_t$ of 3, 5, and 6 mm, respectively. The daily average $T_G$ of 42.71, 48.76, and 51.81°C has been calculated for the CSS at $G_t$ of 3, 5, and 6 mm, respectively. From the experimentation of the CSS at different $G_t$, it is found that the CSS at $G_t$ of 3 mm has minimum $T_G$ as compared to the CSS at $G_t$ of 5 and 6 mm. The CSS at $G_t$ of 5 and 6 mm increases the daily average $T_G$ by 4.52% and 8.4%, respectively, as compared to the CSS at the $G_t$ of 3 mm. From graphs 3 and 4, it is found that the CSS at $G_t$ of 3 mm has the maximum $T_W$ and minimum $T_G$ which produces the higher difference between $T_W$ and $T_G$.

### 3.4. Hourly Variations of the Yield

The value of EHTC is directly proportional to the $T_W$ so it has a similar curve like a $T_W$ curve. The maximum EHTC of 46.28, 37, and 28.34 W/m²k have been recorded for the CSS at $G_t$ of 3, 5, and 6 mm, respectively. The daily average EHTC of 26.71, 18.76, and 9.81 W/m²k have been calculated for the CSS at $G_t$ of 3, 5, and 6 mm, respectively. The CSS at $G_t$ of 5 and 6 mm reduces the daily average EHTC by 29.74% and 63.27%, respectively, as compared to the CSS at $G_t$ of 3 mm. Figure 6 shows the hourly variations of the yield produced from the CSS at $G_t$ of 3, 5, and 6 mm. The yield produced from the CSS at different $G_t$ has a similar curve like the EHTC because yield is directly proportional to the EHTC. The maximum hourly yield produced from the CSS at $G_t$ of 3, 5, and 6 mm is 0.76, 0.45, and 0.31 kg, respectively. The daily average yield produced from the CSS at $G_t$ of 3, 5, and 6 mm is 3.15, 1.9, and 1.1 kg, respectively. The CSS at $G_t$ of 5 and 6 mm reduces the daily yield by 40.48% and 65.15%, respectively, as compared to the CSS at $G_t$ of 3 mm. From the yield production, it is clear that the CSS at $G_t$ of 3 mm has maximum yield because of the higher difference between $T_W$ and $T_G$. This difference between $T_W$ and $T_G$ is called the gradient of temperature. Due to higher temperature difference; the CSS at $G_t$ of 3 mm produced more yield than the CSS at $G_t$ of 5 and 6 mm.

### 3.5. Thermal Efficiency

Figure 7 shows the thermal efficiency of the CSS at $G_t$ of 3, 5, and 6 mm. The thermal efficiency of the CSS at different $G_t$ has the similar curve like the yield. The maximum thermal efficiency of the CSS at $G_t$ of 3, 5, and 6 mm is 47.33, 32.86, and 23.87%, respectively. The daily average thermal efficiency of the CSS at $G_t$ of 3, 5, and 6 mm is 30.71, 18.35, and 11.25%, respectively. The CSS at $G_t$ of 5 and 6 mm reduces the daily average thermal efficiency by 40.25% and 63.37% as compared to the CSS at $G_t$ of 3 mm.
The thermal efficiency of the system mostly depends on the yield produced from the CSS. In this experiment, the yield produced from the CSS is maximum in the case of \(G_t\) of 3 mm, so thermal efficiency of the system is higher in the case of \(G_t\) of 3 mm.

The thermal efficiency of the CSS is given by

\[
\eta_{\text{passive}} = \frac{\sum m_{\text{ev}} L}{\sum I(t)A_s \times 3600} \times 100.
\]

(1)

### 3.6. Exergy Efficiency

Figure 8 shows the exergy efficiency of the CSS at \(G_t\) of 3, 5, and 6 mm. The exergy efficiency of the CSS at different \(G_t\) has the similar curve like the thermal efficiency. The maximum hourly exergy efficiency of the CSS at \(G_t\) of 3, 5, and 6 mm is 2.46, 1.38, and 0.84%, respectively. The CSS at \(G_t\) of 5 and 6 mm reduces the daily average exergy efficiency by 44.01% and 65.81%, respectively, as compared to the CSS at \(G_t\) of 3 mm. The exergy efficiency of the system mostly depends on the yield produced from the CSS. In this research, the maximum exergy efficiency was calculated for the CSS at \(G_t\) of 3 mm. The performance of the CSS at \(G_t\) of 3 mm is higher, so exergy efficiency is higher at the CSS at \(G_t\) of 3 mm.

The daily average exergy efficiency of the CSS at \(G_t\) of 3, 5, and 6 mm is 2.46, 1.38, and 0.84%, respectively. The CSS at \(G_t\) of 5 and 6 mm reduces the daily average exergy efficiency by 44.01% and 65.81%, respectively, as compared to the CSS at \(G_t\) of 3 mm. The exergy efficiency of the system mostly depends on the yield produced from the CSS. In this research, the maximum exergy efficiency was calculated for the CSS at \(G_t\) of 3 mm. The performance of the CSS at \(G_t\) of 3 mm is higher, so exergy efficiency is higher at the CSS at \(G_t\) of 3 mm.

The time-wise fractional exergy of evaporation and convection between \(T_{s_w}\) and \(T_{c_c}\) for the CSS at \(G_t\) of 3, 5, and 6 mm is calculated using equations (10) and (11). It is found that fractional exergy of evaporation of the CSS at \(G_t\) of 3 mm is higher than that of the CSS at \(G_t\) of 5 and 6 mm. The maximum hourly fractional exergy of evaporation of 96.1, 96, and 95.9% has been calculated for the CSS at \(G_t\) of 3, 5, and 6 mm, respectively. During the experimentation, it is found that the evaporation rate and water temperature of the CSS at \(G_t\) of 3 mm are higher as compared to those of the CSS at \(G_t\) of 5 and 6 mm. So time-wise fractional exergy of evaporation of the CSS at \(G_t\) of 3 mm is higher than that of CSS at \(G_t\) of 5 and 6 mm. It is also found that fractional exergy of convection was calculated to be 9.4, 8.6, and 8.4% for the CSS at \(G_t\) of 3, 5, and 6 mm, respectively. The reason for maximum fractional exergy of convection for the CSS at \(G_t\) of 3 mm is the operating temperature of the \(G_t\).

The EHTC starting from basin of the CSS to the glass cover is computed by

\[
h_{t_w-g} = 16.273 \times 10^{-3} \times h_{t_w-g} \left[ \frac{P_w - P_{g_i}}{T_w - T_{g_i}} \right].
\]

(2)
The convective heat transfer coefficient starting from the basin of the CSS to the glass cover is computed by [39–41]

\[ h_{c,w,g} = 0.884 \left( T_w - T_{gi} \right) + \frac{(P_w - P_{gi}) (T_w + 273)}{(268.9 \times 10^3 - P_w)} \]  \( . \)  \( (3) \)

Partial vapour pressure at the basin \( T_w \) is computed by [39–41]

\[ P_w = \exp \left( 25.317 - \frac{5144}{273 + T_w} \right) . \]  \( (4) \)

Partial vapour pressure at the \( T_{gi} \) is computed by [39–41]

\[ P_{gi} = \exp \left( 25.317 - \frac{5144}{273 + T_{gi}} \right) . \]  \( (5) \)

The thermal efficiency of the CSS is computed as [39–41]

\[ \eta_{\text{passive}} = \frac{\sum \dot{m} w L_a}{\sum I(t) A_j \times 3600} \times 100. \]  \( (6) \)

The exergy efficiency of the CSS is computed by [39–41]

\[ \eta_{\text{overall,exe}} = \frac{\sum \dot{E}_{\text{output}}}{\sum \dot{E}_{\text{input}}} . \]  \( (7) \)

The hourly exergy output is computed by [39–41]

\[ \dot{E}_{\text{output}} = \frac{m_{w} L_{fg}}{3600} \times \left[ 1 - \frac{T_{a}}{T_w} \right] . \]  \( (8) \)

The hourly exergy input is computed by [39–41]

\[ \dot{E}_{\text{input}} = A_w I'(t) \times \left[ 1 - \frac{4}{3} \left( \frac{T_{a}}{T_{s}} \right) + \frac{1}{3} \left( \frac{T_{a}}{T_{s}} \right)^4 \right] . \]  \( (9) \)

The fractional exergies for evaporation and convection are calculated using equations (10) and (11) and are as follows [42]:

\[ F_{e, bw-ig} = \frac{\dot{E}_{e, bw-ig}}{\dot{E}_{i}}, \]  \( (10) \)

\[ F_{c, bw-ig} = \frac{\dot{E}_{c, bw-ig}}{\dot{E}_{i}} . \]  \( (11) \)

3.7. Comparisons of Present Result and Similar Studies. Table 2 gives the several studies previously carried out by researchers on the thickness of the glass cover of solar stills. It also shows the results obtained on the productivity of pure water and the thermal efficiency of the stills tested, and it was noticed that energy efficiency varies between 17 and 42%, and it is found that the present results vary between 11.25 and 30.71%. We draw the attention that some research does not calculate either energy efficiency or exergy efficiency. Comparing the efficiencies, it is found that the CSS with 3 mm gives a good value in both types of efficiencies [38].

| Ref. no. | Glass cover thickness | Energy efficiency | Exergy efficiency |
|----------|-----------------------|-------------------|------------------|
| [28]     | 4 mm                  | -                 | -                |
|          | 8 mm                  | -                 | -                |
|          | 12 mm                 | -                 | -                |
| [29]     | 3.5 mm                | 21.42%            | -                |
|          | 4 mm                  | 28.09%            | -                |
| [30]     | 3.18 mm               | 26.22%            | -                |
|          | 4.76 mm               | 29.28%            | -                |
|          | 6.35 mm               | 18.95%            | -                |
| [31]     | 3 mm                  | 32-34%            | -                |
|          | 4 mm                  | 34-36%            | -                |
|          | 5 mm                  | 30-32%            | -                |
|          | 6 mm                  | 26-28%            | -                |
| [32]     | >8 mm + air film      | -                 | -                |
|          | 12 mm                 | -                 | -                |
|          | >12 mm + air film     | -                 | -                |
| [33]     | 4 mm                  | 27%               | -                |
|          | 5 mm                  | 22%               | -                |
|          | 6 mm                  | 17%               | -                |
| [34]     | 2 mm                  | 45-50%            | -                |
|          | 3 mm                  | 40-45%            | -                |
|          | 4 mm                  | 30-35%            | -                |
|          | 5 mm                  | 30%               | -                |
|          | 6 mm                  | 25-30%            | -                |
|          | 7 mm                  | 20-25%            | -                |
|          | 8 mm                  | 15-20%            | -                |
| [35]     | 4 mm                  | -                 | -                |
|          | 5 mm                  | -                 | -                |
|          | 6 mm                  | -                 | -                |
| [37]     | 2 mm                  | -                 | -                |
|          | 6 mm                  | -                 | -                |
|          | 10 mm                 | -                 | -                |
|          | 14 mm                 | -                 | -                |
|          | 18 mm                 | -                 | -                |
| [38]     | 4 mm + air film       | -52.7%            | -                |
|          | 8 mm + air film       | -                | -                |

Our current work

|               | Glass cover thickness | Energy efficiency | Exergy efficiency |
|---------------|-----------------------|-------------------|------------------|
| 3 mm          |                       | 30.71%            | 2.46%            |
| 5 mm          |                       | 18.35%            | 1.38%            |
| 6 mm          |                       | 11.25%            | 0.84%            |
4. Conclusion

Based on the investigational outcome on the CSS, the following conclusions are arrived at.

(i) The CSS at $G_t$ of 3, 5, and 6 mm has a yield of 3.15, 1.9, and 1.1 kg, respectively. The CSS at $G_t$ of 3 mm produced the 40.48% and 65.15% higher yield than the CSS at $G_t$ of 5 and 6 mm, respectively.

(ii) The thermal and exergy efficiencies of 30.71, 18.35, and 11.25% and 2.46, 1.38, and 0.84% have been obtained for the CSS at $G_t$ of 3, 5, and 6 mm, respectively.

(iii) The thermal efficiency of the CSS is higher in the case of CSS at $G_t$ of 3 mm. The thermal efficiency of the CSS at $G_t$ of 3 mm is 40.25% and 63.37% higher as compared to that of the CSS at $G_t$ of 5 and 6 mm, respectively.

(iv) The exergy efficiency of the CSS is higher in the case of CSS at $G_t$ of 3 mm. The exergy efficiency of the CSS at $G_t$ of 3 mm is 44.01% and 65.81% higher as compared to that of the CSS at $G_t$ of 5 and 6 mm, respectively.

Nomenclature

$G_t$: Glass thickness  
CSS: Conventional solar still  
$T_W$: Water temperature  
$T_G$: Glass temperature  
EHTC: Evaporative heat transfer coefficient.

Data Availability

The data used to support the findings of this study are included within the article.

Conflicts of Interest

The authors declare that they have no conflicts of interest.

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