CHEMICAL ENGINEERING | RESEARCH ARTICLE

Fractional filter IMC-PID controller design for second order plus time delay processes

R. Ranganayakulu¹, G. Uday Bhaskar Babu*¹ and A. Seshagiri Rao¹

Abstract: This article presents a simple method of designing a fractional filter PID controller for second order plus time delay (SOPTD) processes using internal model control (IMC) scheme. There has been limited number of tuning rules for SOPTD processes developed using direct synthesis method and IMC method. The proposed IMC-PID controller using fractional IMC filter results in a controller structure composed of PID controller cascaded with fractional filter. Simulations have been performed on several second order lag dominant and delay significant processes. Robustness checks are performed for variations in the process parameters and robustness analysis is carried out using sensitivity functions. The proposed controller results in an enhanced control performance for nominal process parameters and with parameter variations. In addition, the effect of measurement noise is also studied for set point tracking and load disturbance variations.

Subjects: Systems & Control Engineering; Control Engineering; Dynamical Control Systems; Process Control - Chemical Engineering; Reaction Engineering; Distillation

Keywords: PID controller; fractional filter; internal model control; measurement noise; robustness

1. Introduction

Proportional Integral Derivative (PID) controller has been the main choice for industrial sector applications from centuries. In spite of the advancements in control, PID controller is still being used by 90% of the applications due to its ability to control wide range of industrial processes (Shamsuzzoha, *Corresponding author: G. Uday Bhaskar Babu, Department of Chemical Engineering, National Institute of Technology, Warangal 506004, India E-mails: udaybhaskar@nitw.ac.in, chemuday.iitkgp@gmail.com

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PUBLIC INTEREST STATEMENT

Each process needs a precise control in our day to day life to produce a desired and accurate output. On a broader scale, control has become a major and important part of all types of industries that produces different products. The ultimate aim of production is that the products should meet the predefined specifications to satisfy the needs of end user. Hence, a control system with proper control structure needs to be incorporated to control different aspects of the processes. This article proposes one such structure of the controller designed with internal model control using fractional filter. The design is validated on several industrial processes modeled as second order systems to produce desired output for different types of inputs acting on them. Further, the robustness of the design is verified for different plant conditions. These analyses help engineers in identifying the best control structure to produce desired output.
Moreover, the simple structure and the availability of tuning rules contribute to the wider use of PID controller (Astrom & Hagglund, 1995; Skogestad, 2003). However, the controller tuning methods are still evolving to ensure improved closed loop performance of the processes as they are associated with time delays, external perturbations and non-linearities (Jeng, Tseng, & Chiu, 2014; Lee, Cho, & Edgar, 2013; Ranganayakulu, Uday Bhaskar Babu, & Seshagiri Rao, 2016; Silva, Datta, & Bhattacharyya, 2007; Vilanova & Visioli, 2012). The real processes need to be approximated as lower order models for the application of PID tuning rules. The SOPTD models represent better dynamics of the processes than first order plus time delay (FOPTD) models. The PID tuning rules for SOPTD processes are less (Panda, Yu, & Huang, 2004; Weigand & Kegerreis, 1972) in number compared to FOPTD processes. Hence, the current work is carried out on SOPTD processes.

The direct synthesis (DS) method and internal model control (IMC) schemes are mostly used for designing the controller for SOPTD processes (Chen & Seborg, 2002; Lee, Park, Lee, & Brosilow, 1998; Panda et al., 2004). DS controllers are designed for the desired trajectory of the closed loop transfer function. But, the design not necessarily results in a PID form of controller. Direct synthesis controllers are suitable for set point tracking and don’t give satisfactory performance for disturbance rejection. The IMC based design results in a PID controller structure by proper approximation of the process model. Several controller tuning methods based on IMC-PID method have been proposed for SOPTD processes. The design was based on the selection of optimum IMC filter and controller structure (Shamsuzzoha & Lee, 2007, 2008). An analytical method of designing IMC-PID controller was proposed for all kinds of time delay systems (Shamsuzzoha, 2015). For improved disturbance rejection a PID controller cascaded with lead-lag compensator has been used (Rao, Rao, & Chidambaram, 2009; Shamsuzzoha & Lee, 2008). Further, the overshoot in servo response was reduced by utilizing set point weighting. An optimal tuning method for SOPTD processes by optimizing integral of absolute error (IAE) was also proposed (Madhuranthakam, Elkamel, & Budman, 2008). A graphical method of obtaining PID controller parameters for SOPTD processes through dominant pole placement approach with assured gain margin and phase margin was also proposed (Srivastava & Pandit, 2016).

Some of the design methods presented above are suitable for set point tracking while others were exclusively designed for disturbance rejection. A few of the design procedures don’t guarantee the PID form of controller structure which is widely used in industries while still providing better closed loop performance. Though, the IMC based methods had only one tuning parameter they have used set point weighting and set point filter along with controller to suppress the overshoot. So, there was a need to choose the weighting factor or filter parameter along with the controller settings. Recently, an IMC-PID controller was proposed using pole zero conversion with an unified IMC filter structure (Wang, Lu, & Pan, 2016). The design also used derivative coefficient weighting along with pole zero conversion to obtain the controller settings for SOPTD processes. Also, the overshoot was minimized with set point weighting technique. Majority of the controllers designed for SOPTD processes were based on the use of higher order IMC filter (second to fourth order) and by using optimization. The current work focuses on the design of a simple IMC-PID controller using fractional IMC filter for SOPTD processes. The resulting controller structure consists of a PID in series with fractional filter. The performance metrics like integral of square error (ISE), integral of absolute error (IAE), percentage overshoot (%OS) and maximum sensitivity (Ms) are used to estimate the closed loop performance of SOPTD processes. The effectiveness of the present method is illustrated with simulations carried out on over damped and critically damped SOPTD processes using MATLAB and Simulink. In addition, the simulation is carried out on a higher order process reduced to SOPTD process. The effects on process output performance in presence of parametric uncertainties and output noise have also been discussed.

### 2. Fractional filter IMC-PID controller design

The IMC scheme and feedback control loop with internal blocks are shown in Figure 1, where \(G(s)\), \(\tilde{G}(s)\), \(G_{\text{proc}}(s)\) and \(G_{\text{c}}(s)\) representing process, process model, IMC controller and transfer function of
the traditional controller. Let $r$, $y$, $u$ and $d$ be the set point, controlled variable, control input and disturbance respectively. The controller design using IMC method is given as follows:

1. Decompose the process model into non-invertible and invertible parts

$$\hat{G}(s) = \hat{G}^+(s)\hat{G}^-(s)$$

where $\hat{G}^+(s)$ is non-invertible contains all time delays and unstable zeros, $\hat{G}^-(s)$ is invertible and contains minimum phase elements.

2. The IMC controller is given by

$$G_{IMC}(s) = \frac{f(s)}{\hat{G}^-(s)}$$

where $f(s)$ is the IMC filter

3. The equivalent feedback controller is

$$G_C(s) = \frac{G_{IMC}(s)}{1 - G_{IMC}(s)\hat{G}(s)}$$

2.1. Proposed IMC-PID controller design

The proposed controller has the structure

$$G_C(s) = \left(\text{fractional filter}\right)\left[K_c\left(1 + \frac{1}{T_1s} + \tau_d s\right)\right]$$

Consider a SOPTD model

$$G(s) = \frac{Ke^{-Ls}}{(T_1s + 1)(T_2s + 1)}$$

where $K$-the system gain, $L$-delay and $T_1$, $T_2$ are the process time constants.

The fractional IMC filter used is

$$f(s) = \frac{1}{\gamma s^p + 1}$$

where $\gamma$ is the fractional filter time constant and $p$ is the fractional order. Now, the IMC controller according to Equation (2) is
Finally, the fractional filter IMC-PID controller from (3), (5) and (7) is

\[ G_{\text{IMC}}(s) = \frac{(T_1 s + 1)(T_2 s + 1)}{K} \frac{1}{\gamma s^p + 1} \]  

(7)

The delay \( e^{-Ls} \) expressed as a first order fraction according to Pade’s rule is

\[ G_c(s) = \frac{\left[ \frac{(T_1 s + 1)(T_2 s + 1)}{K \gamma s^p + 1} \right]}{1 - \left[ \frac{(T_1 s + 1)(T_2 s + 1)}{K \gamma s^p + 1} \right] K e^{-Ls}} \]  

(8)

\[ \Rightarrow G_c(s) = \frac{(T_1 s + 1)(T_2 s + 1)}{K \left[ \gamma s^p + 1 - e^{-Ls} \right]} \]  

(9)

The delay \( e^{-Ls} \) expressed as a first order fraction according to Pade’s rule is

\[ e^{-Ls} = \frac{1 - 0.5Ls}{1 + 0.5Ls} \]  

(10)

Now, the controller becomes

\[ G_c(s) = \frac{(T_1 s + 1)(T_2 s + 1)}{K \left[ \gamma s^p + 1 - \left( \frac{1 - 0.5Ls}{1 + 0.5Ls} \right) \right]} \]  

(11)

The above equation can be written as

\[ G_c(s) = \frac{0.5Ls + 1}{0.5Ls + 0.5s^p + 1} \left( \frac{T_1 + T_2}{K} \right) \left[ 1 + \frac{1}{(T_1 + T_2)s} + \left( \frac{T_1 T_2}{T_1 + T_2} \right) s \right] \]  

(12)

Comparing Equations (4) and (12), the controller settings are

\[ K_c = \frac{T_1 + T_2}{K}; \tau_i = T_1 + T_2; \tau_d = \frac{T_1 T_2}{T_1 + T_2} \]  

(13)

and the fractional filter is

\[ \text{filter} = \frac{0.5Ls + 1}{0.5Ls + 0.5s^p + 1} \]  

(14)

2.2. Robustness analysis

The closed loop stability of system should be verified in presence of model uncertainties for robustness of the designed controller which is derived under nominal process conditions. The designed controller should be able to provide better closed loop performance (good servo response and regulatory response) irrespective of the perturbations in process parameters which are common in practice. The controller that ensures good response characteristics for perturbations in system gain, time delay and time constant is said to be robust. This robust stability of closed loop system can be verified with small gain theorem (Maciejowski, 1989; Morari & Zafiriou, 1989). According to this theorem, the closed loop system will be robustly stable if and only if

\[ \| T(s) l_m(s) \| < 1 \]  

(15)

where \( T(s) \) and \( l_m(s) \) are the complementary sensitivity function and the bound on multiplicative uncertainty. They are defined as

\[ T(s) = \frac{G(s)G_c(s)}{1 + G(s)G_c(s)} \]  

(16)
\[ I_{m}(s) = \frac{G(s) - \hat{G}(s)}{\hat{G}(s)} \]  
(17)

where \( G(s) \) is the real process representing the nominal model (Equation (5)); \( \hat{G}(s) \) is the actual model of the process.

In addition to the condition for robust stability in Equation (15), the following inequality constraint must hold good to ensure robust closed loop performance.
\[
\| T(s)I_{m}(s) + S(s)w_{m}(s) \| < 1
\]  
(18)

where \( S(s) \) is the sensitivity function which can be found from \( S(s) = 1 - T(s) \) and \( w_{m}(s) \) is the multiplicative uncertainty bound on the sensitivity function.

### 3. Results and discussion

The performance of different SOPTD processes with the designed controller is analyzed and was compared with the control performance obtained from Wang et al. (2016) tuning method. Several SOPTD processes representing lag dominant and balanced/delay significant process dynamics are considered for simulation. In addition, the simulations are performed on a higher order process reduced to SOPTD model. The performance metrics used for comparison are: integral of square error (ISE), integral of absolute error (IAE), percentage overshoot (%OS) and maximum sensitivity (\( M_s \)). The \( \delta \) performance indices are defined as follows:

\[
\text{ISE} = \int_{0}^{\infty} e^2(t)dt
\]  
(19)

\[
\text{IAE} = \int_{0}^{\infty} |e(t)|dt
\]  
(20)

\[
M_s = \max_{\omega} \left| \frac{1}{1 + G(j\omega)G_C(j\omega)} \right|
\]  
(21)

\[
TV = \sum_{i=0}^{\infty} |u_{i+1} - u_i|
\]  
(22)

The simulation scheme is shown in Figure 2. The closed loop response was observed for a step set point changes of unit magnitude with load disturbance. The quality of response was verified by introducing a white noise in the output. It is to be noted that the fractional term in fractional filter of the controller is approximated using Oustaloup method. The frequency range used for approximation is 0.01–100 rad/s with an approximation order of 5.

**Example 1**  
Consider the delay significant SOPTD process as studied in (Lee et al., 2013)
\[
G_1 = \frac{e^{-2s}}{(s + 1)(0.7s + 1)}
\]  
(23)

The controller obtained for the above process with proposed method is given in Equation (24). The filter time constant was chosen as \( \gamma = 2 \) and the fractional order \( p = 1.02 \).
\[
G_C = \left( \frac{s + 1}{2s^{1.02} + 2s^{0.02} + 2} \right) 1.7 \left( 1 + \frac{1}{1.7s + 0.4117s} \right)
\]  
(24)
The controller settings as proposed in Wang et al. (2016) are $K_c = 0.435$; $\tau_i = 1.653$ and $\tau_d = 0.4$. The weighting factor used to decrease the overshoot is 0.4. Figure 3 shows the servo response with step disturbance change of magnitude $-0.5$ applied at $t = 50s$. The proposed controller results in a lower overshoot in the response without set point weighting which was used in Wang et al. (2016) method. Lower values of performance metrics ISE and IAE are observed with the proposed controller which are shown in Table 1. Figure 4 shows the closed loop response for perturbations of $+10\%$ in time delay and process gain. The effect of noise in the measurement is studied by introducing a white noise of zero mean and a variance of 0.0001. This is illustrated in Figure 5. The performance indices for noise rejection case are given in Table 2. Note that the TV value indicating the control effort is small with the proposed controller for the output mixed with noise. The values of $M_s$ from Table 1 confirms the robustness of closed loop system for model uncertainties. Further, the robust stability of closed loop system is evaluated with a magnitude plot comprising complementary sensitivity function and an uncertainty of $+10\%$ in time delay. The robust stability characteristics are shown in Figure 6 and it confirms the stability condition given in Equation (15) making the system robustly stable.

**Example 2** The second example considered for performance comparison is as follows (Srivastava, Misra, Thakur, & Pandit, 2016):

$$G_2 = \frac{e^{-1.64s}}{s^3 + 3s + 2}$$

(25)

The proposed controller for this process is given in Equation (26) for a filter time constant of 1 and fractional order $p$ is equal to 1.02.

$$G_c = \left( \frac{0.82s + 1}{0.82s^{1.02} + s^{0.02} + 1.64} \right) \left( 1 + \frac{1}{1.5s + 0.3333s} \right)$$

(26)
The controller used for comparison is having the settings: $K_c = 1.1069$; $\tau_i = 1.4995$ and $\tau_d = 0.3332$.

The servo response of the closed loop system is evaluated for step set point changes of unit magnitude and for step change in load applied at $t = 40s$ having a magnitude of $-1$. Figure 7 shows the response characteristics of the SOPTD process in (25) with the two controllers. The performance metrics are provided in Table 1 and an improved performance is resulted with the proposed controller. There is a reduction in the overshoot, ISE and IAE values which is clear from Table 1. Figure 8 shows the robust performance of the proposed controller for $+10\%$ change in time delay and process gain. An important observation here is that both the controllers give robust control performance as the $M_s$ values are less than 2 which is evident from Table 1.

The impact of white noise having a mean value of zero and a variance of 0.0001 in the measured output was well rejected by both the controllers but the control effort is comparatively small with the proposed controller structure. Figure 9 and Table 2 shows the superior performance of the designed controller which is clear from smaller values of ISE and IAE.

**Table 1. Comparison of closed loop performance of the examples**

| Process | Method          | Perfect case |              | Perturbed case |              |
|---------|----------------|--------------|--------------|----------------|--------------|
|         |                | ISE          | IAE          | %OS            | ISE          | IAE          | %OS            |
| $G_1$   | Proposed       | 3.588        | 5.91         | 0.5            | 4.096        | 6.674        | 11.798         | 1.56           |
|         | Wang et al. (2016) | 4.768        | 7.328        | 5.8            | 5.407        | 9.151        | 24.375         | 1.6            |
| $G_2$   | Proposed       | 2.563        | 3.991        | 0.5            | 2.787        | 4.508        | 8.152          | 1.77           |
|         | Wang et al. (2016) | 3.025        | 5.034        | 0.55           | 3.323        | 5.893        | 19.88          | 1.787          |
| $G_3$   | Proposed       | 0.7197       | 1.427        | 0.52           | 0.68         | 1.322        | 0.505          | 1.156          |
|         | Wang et al. (2016) | 0.8204       | 1.678        | $-0.18$        | 0.7706       | 1.551        | 0.502          | 1.138          |
| $G_4$   | Proposed       | 5.638        | 8.255        | 0.64           | 6.305        | 10.16        | 11.992         | 2              |
|         | Wang et al. (2016) | 6.822        | 11.43        | 24.37          | 7.919        | 14.67        | 18.476         | 2.16           |
Figure 5. Closed loop response in presence of measurement noise for $G_1$: Proposed method (red solid line), Wang et al. (2016) method (blue dotted line).

Table 2. Performance comparison with measurement noise

| Process | Method         | ISE  | IAE   | TV            |
|---------|----------------|------|-------|---------------|
| $G_1$   | Proposed       | 3.592| 6.478 | 311.2247      |
|         | Wang et al. (2016) | 4.771| 7.903 | $8.766 \times 10^4$ |
| $G_2$   | Proposed       | 2.572| 4.484 | 721.7699      |
|         | Wang et al. (2016) | 3.034| 5.793 | 749.3172      |
| $G_3$   | Proposed       | 0.7201| 1.658 | 408.6515      |
|         | Wang et al. (2016) | 0.8198| 1.9  | 757.4881      |
| $G_4$   | Proposed       | 5.649| 8.955 | 1,263.2       |
|         | Wang et al. (2016) | 6.825| 11.91| 1,745.5       |

Figure 6. Magnitude plot for $G_1$: Complementary sensitivity function (red solid line), +10% uncertainty in time delay (green dotted line).
In addition, the servo responses with the proposed method at different values of $p$ are compared and the characteristics are shown in Figure 10. The ISE and IAE values at each value of $p$ are given in Table 3 which are less than ISE and IAE values used for comparison. Hence, the proposed method has flexibility that the fractional order $p$ can be varied over a range producing lower values of performance metrics. The trend of complementary sensitivity function satisfying the robustness bound is shown in Figure 11. The characteristics confirm the robust closed loop stability of the simulated system.

**Example 3** Consider the second order lag dominant linear process (Dey & Mudi, 2009) given by

$$G_3 = \frac{e^{-0.2s}}{(s + 1)^2} \quad (27)$$

The controller designed according to the design rules in Wang et al. (2016) is used for comparison. The controller settings as per their method are: $K_c = 1.4247; \tau_i = 1.9924$ and $\tau_d = 0.498$. The proposed controller with the filter parameters $\gamma = 1$ and $p = 1.01$ is
The set point tracking of closed loop system with a negative step disturbance of magnitude 0.2 applied at \( t = 20 \text{s} \) is shown in Figure 12 while the performance metrics are given in Table 1. There is negligible overshoot in the response for both the controllers but the error values are lesser with the proposed controller. The similar response holds even after introducing +10\% parameter variations in time delay and process gain which is evident from Figure 13 and Table 1. The \( M_s \) values for both methods are in the standard range of 1 to 2 which ensure robust control performance.

The response of closed loop system with added noise is illustrated in Figure 14. The white noise used has a mean of zero and a variance of 0.0001. The control effort is significantly less with the proposed method against the method used for comparison. The ISE, IAE and TV values for this case are given by:

\[
G_c = \left( \frac{0.1s + \frac{1}{0.1s^{0.01} + 0.01} + 0.2}{2 \left( 1 + \frac{0.1s}{2s} + 0.5s \right)} \right)
\]

(28)

The response of closed loop system with added noise is illustrated in Figure 14. The white noise used has a mean of zero and a variance of 0.0001. The control effort is significantly less with the proposed method against the method used for comparison. The ISE, IAE and TV values for this case are given by:

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G_c = \left( \frac{0.1s + \frac{1}{0.1s^{0.01} + 0.01} + 0.2}{2 \left( 1 + \frac{0.1s}{2s} + 0.5s \right)} \right)
\]

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G_c = \left( \frac{0.1s + \frac{1}{0.1s^{0.01} + 0.01} + 0.2}{2 \left( 1 + \frac{0.1s}{2s} + 0.5s \right)} \right)
\]

(28)
Figure 10. Comparison of closed loop response for process $G_2$ at different values of $p$.

![Figure 10](image)

Table 4. Performance comparison for $G_2$ with proposed method at different values of fractional order ($p$)

| $p$  | ISE  | IAE  |
|------|------|------|
| 1.01 | 5.501| 6.729|
| 1.02 | 5.5  | 6.76 |
| 1.03 | 5.5  | 6.796|
| 1.04 | 5.501| 6.833|
| 1.05 | 5.502| 6.872|
| 1.06 | 5.503| 6.911|
| 1.07 | 5.506| 6.951|
| 1.08 | 5.509| 6.991|
| 1.09 | 5.512| 7.032|
| 1.1  | 5.516| 7.072|

Figure 11. Magnitude plot for $G_2$: Complementary sensitivity function (red solid line) +10% uncertainty in time delay (green dotted line).

![Figure 11](image)
are given in Table 2. The closed loop robust stability satisfying the stability condition in Equation (15) is shown in Figure 15. It is clear from the magnitude plot that the system is closed loop stable for parametric uncertainties.

Example 4  Consider a higher order process (Astrom & Hagglund, 1995):

\[ G_u = \frac{1}{(s + 1)^6} \]  \hspace{1cm} (29)

The second order approximated model for the above process is

\[ G_u = \frac{0.336e^{-4.3s}}{s^2 + 1.3878s + 0.336} = \frac{1.0002e^{-4.3s}}{(3.201s + 1)(0.9299s + 1)} \]  \hspace{1cm} (30)

The proposed controller for this process with \( \gamma = 2 \) and \( p = 1.1 \) is
The settings of the controller used for comparison are: 
\[ K_c = 0.7086; \quad \tau_i = 3.9987 \text{ and } \tau_d = 0.7136. \]

Figure 16 shows the servo response for step set point change of unit magnitude and step change in disturbance applied at \( t = 70s \) having a magnitude of \(-0.2\). The response for perturbed process model is shown in Figure 17 with +10% mismatch in time delay and process gain. The performance metrics are provided in Table 1. Clearly, the proposed controller outperforms the other one used for comparison. There is a drastic reduction in the overshoot.

Figure 18 shows the closed loop response with white noise in the measurement. The TV value of the proposed controller with added noise is low compared to the other value. Table 2 confirms the lower values of ISE, IAE and TV. The \( M_s \) value of closed loop system with the proposed controller is 2 where as it is 2.17 for the other method. The value of 2 is the higher limit in the standard \( M_s \) range. It means that both the controllers are likely to respond to model uncertainties and their impact could be seen the response.

\[
G_C = \left( \frac{2.15s + 1}{4.3s^{1.1} + 2s^{0.3}} \right) 4.13 \left( 1 + \frac{1}{4.1309s} + 0.7205s \right) \tag{31}
\]

The settings of the controller used for comparison are: \( K_c = 0.7086; \tau_i = 3.9987 \text{ and } \tau_d = 0.7136. \) Figure 16 shows the servo response for step set point change of unit magnitude and step change in disturbance applied at \( t = 70s \) having a magnitude of \(-0.2\). The response for perturbed process model is shown in Figure 17 with +10% mismatch in time delay and process gain. The performance metrics are provided in Table 1. Clearly, the proposed controller outperforms the other one used for comparison. There is a drastic reduction in the overshoot.

Figure 18 shows the closed loop response with white noise in the measurement. The TV value of the proposed controller with added noise is low compared to the other value. Table 2 confirms the lower values of ISE, IAE and TV. The \( M_s \) value of closed loop system with the proposed controller is 2 where as it is 2.17 for the other method. The value of 2 is the higher limit in the standard \( M_s \) range. It means that both the controllers are likely to respond to model uncertainties and their impact could be seen the response.

Figure 15. Magnitude plot for \( G_3 \): Complementary sensitivity function (red solid line), +10% uncertainty in time delay (green dotted line).
Figure 19 shows the servo response of the process in (30) for the values of $p = 1.01, 1.02, \ldots, 1.1$. The performance metrics ISE and IAE are provided in Table 4. Better performance is observed at every value of $p$. Also, the magnitude plot in Figure 20 tells that the complementary sensitivity function with $+10\%$ uncertainty in time delay obeys the robust stability condition. Hence, the closed loop system ensures robust and stable performance even with uncertainty in the process parameters.

The effect of fractional filter on the robustness of closed loop control system is explained with the help of parametric uncertainties. A better closed loop response is obtained compared to Wang et al. (2016) method for uncertainty in process gain and time delay for all the four examples which proves that the closed loop system gives robust performance for uncertainties with fractional filter PID controller. Further, the stability of closed loop system with the proposed controller is investigated for parametric uncertainties. It is proved that all the four systems used in the current work are satisfying
robust stability condition and the illustrations are shown in Figures 6, 11, 15, 20. In addition, the closed loop system is robust for noise inputs and the proposed controller successfully attenuates the noise at high frequencies. This is clear from the magnitude of complementary sensitivity function which is approaching zero (Figures 6, 11, 15, 20) with increase in frequency for all the four systems.

The proposed method may be extended for finite time systems that lead to finite time control. The stability of the closed loop system with the proposed method may be checked over a finite time for boundedness of the system response (Lazarević, 2006; Xu, Zhang, Zhou, & Tong, 2017; Xu, Zhou, Fang, Xie, & Tong, 2016).
4. Conclusion

A simple IMC-PID controller for SOPTD processes using fractional IMC filter is proposed. Different case studies were analyzed for servo response and regulatory control. It can be concluded from the results that a good closed loop performance is obtained with nominal model and actual model of the process with the proposed controller. The proposed controller resulted in a minimum overshoot in the response in absence of set point weighting. It was also shown that the effort of proposed controller is less for noise corrupted measurements. The robustness for parametric uncertainties is proved with robustness analysis using sensitivity functions. The work is in progress to extend the proposed method for unstable and non-minimum phase systems.

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