One pendulum to run them all

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Received 30 April 2013
Published 24 May 2013
Online at stacks.iop.org/EJP/34/1049

Abstract
The analytical solution for the three-dimensional linear pendulum in a rotating frame of reference is obtained, including Coriolis and centrifugal accelerations, and expressed in terms of initial conditions. This result offers the possibility of treating Foucault and Bravais pendula as trajectories of the same system of equations, each of them with particular initial conditions. We compare them with the common two-dimensional approximations in textbooks. A previously unnoticed pattern in the three-dimensional Foucault pendulum attractor is presented.

(Some figures may appear in colour only in the online journal)

1. Introduction

During the first year(s) of undergraduate study students are constantly warned to work in an inertial frame, as Newton’s second law can only be applied safely in this case. Since it rotates around its axes (even ignoring its motion around the Sun), a reference frame fixed to the Earth is definitively non-inertial. However, for most mechanics-related results this fact can be overlooked, as this rotation is not fast. But there are certain circumstances when non-inertial reference frames, especially rotating frames, look like an interesting thing to use. This is especially so when trying to study movement on the Earth’s surface or when looking to prove or measure the Earth’s motion itself through a mechanical experiment.

The first proof of the Earth’s rotation, and perhaps the most simple and elegant, is the Foucault pendulum experiment [1] performed in 1851. A handwritten annotation by Foucault found among his papers reads Mercredi, 8 janvier, 2 heures du matin: le pendule a tourné dans le sens du mouvement diurne de la sphère céleste [2]. Namely, the experiment succeeded 8 January at 2 am. It was carried out in his home’s cellar, d’Assas street in Paris, using a 2 m long pendulum [3]. Afterward, in February, the experiment was repeated at a larger scale with an 11 m long pendulum in the Paris Observatory [1] with the permission of Arago. It is worth noting the profound acknowledgement Foucault gives his assistant Froment.

The public exhibition of the pendulum was held in March, at the request and with the backing of the prince–president of the Republic, and it is now widely known. Gracefully
suspended from the ceiling of the dome of the Panthéon in Paris, it consisted of a 28 kg brass-coated lead bob attached to a 67 m long steel aircraft cable. The plane of the pendulum’s swing appeared to rotate clockwise 264° every day (11° per hour), sweeping a circle of 16 m diameter. Since no rotational forces act on the pendulum, it is obvious that it is the Earth beneath that is actually rotating. However, to the naked sizeable effects take several hours to become apparent. We will return to this point later.

In the same year (and amazingly also in Paris) Bravais proposed an alternative way to observe the rotation of the Earth, also using pendula [4, 5]. His approach was, however, different and has been reviewed recently in this journal [6]. He explored the difference in the period of oscillations of right- and left-handed pendula. This fact, as we are going to show later, opens the door to observing the rotation of the Earth in a few minutes.

However, despite not receiving a unified treatment, Foucault and Bravais pendula share the same equations of motion, differing only in initial conditions. Therefore they may be considered, in all senses, as two particular cases of a large variety of oscillation modes of a pendulum in a non-inertial reference frame.

Foucault-type oscillations are obtained when the bob is moved away from the equilibrium position and then released with zero initial velocity. In an inertial reference frame the pendulum would oscillate in a fixed plane. Due to the Earth’s rotation, we observe this plane to rotate. Thus the trajectory followed by the bob produces the well-known beautiful patterns, which are the proof of the Earth’s rotation. The Bravais pendulum instead refers to conical oscillations of the bob. To this end, the bob is released from a non-equilibrium point with a precise tangential speed in such a way that the projection of the trajectory on the horizontal plane describes a circumference simply. What makes the Bravais pendulum interesting is that the trajectories are not invariant under sign reversal in the initial tangential velocities, as a consequence of the Earth’s rotation. It is just this phenomenon that led Bravais to propose the pendulum as an experimental demonstration of the Earth’s rotation.

There being just two particular sets of initial conditions for the same system of equations, it becomes obvious that there is a whole plethora of systems that present interesting features worth looking at. Extremely precise initial conditions can be quite challenging to achieve from an experimental point of view.

One of the purposes of this paper is to give an explicit solution for the linear pendulum in terms of initial conditions. We find it to be of particular interest for students whose training in solving linear systems of differential equations is concurrent with their course in mechanics.

The structure of the paper is as follows. In section 2 we collect the equations of motion. In section 3 we review some common approximate solutions presented in textbooks. Next, in section 4 the analytical exact solution in three dimensions is obtained. Section 5 connects our results with an almost forgotten theorem by Chevilliet. In section 6 a comparison of two- and three-dimensional treatments of the Foucault pendulum is made. A previously unnoticed pattern in the attractor is reported. Finally, section 7 contains a discussion of our results.

2. Equations of motion for the pendulum for small amplitudes

We follow the careful exposition in [7] to obtain the equations of motion for a pendulum with small oscillations, referred to a rotating reference frame. The starting point is the Newton equation for the bob of the pendulum

\[ m \ddot{r} = \vec{T} - mg \hat{k}, \]  

where the dot stands for the time derivative, \( \vec{r} \) is the position vector of the bob of mass \( m \) with respect to an inertial system of reference and \( \vec{T} \) is the tension of the wire. In this instance, \( \hat{k} \)
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Figure 1. Reference system attached to the rotating Earth used to study the pendulum.

is the unit vector directed vertically upward, which follows the radial direction of the Earth, assumed to have spherical symmetry. This remark will be of interest momentarily. Finally, \( g \) is the corresponding intensity of gravity.

The analysis of the Foucault pendulum is carried out in a rotating reference system fixed to the Earth’s surface. We will consider that the Earth turns with constant angular velocity

\[
\mathbf{\Omega} = -\Omega \mathbf{i} \cos \beta + \mathbf{k} \Omega \sin \beta,
\]

in terms of the latitude \( \beta \), and \( \Omega = 2\pi / 86400 \) rad s\(^{-1}\) = 0.000073 rad s\(^{-1}\). The unit vector \( \mathbf{i} \) is tangent to the local meridian and points toward the equator, in the northern hemisphere. The third unit vector, \( \mathbf{j} \), is directed eastward (see figure 1).

In a rotating reference system with origin located at the center of the Earth, time derivatives have to be replaced with more involved expressions. In particular, if \( \mathbf{\Omega} = 0 \), the acceleration \( \ddot{\mathbf{r}} \) becomes \( \ddot{\mathbf{r}} + 2\mathbf{\Omega} \times \dot{\mathbf{r}} + \mathbf{\Omega} \times (\mathbf{\Omega} \times \mathbf{r}) \). The term \( 2\mathbf{\Omega} \times \dot{\mathbf{r}} \) is the Coriolis acceleration, and \( \mathbf{\Omega} \times (\mathbf{\Omega} \times \mathbf{r}) \) stands for the centrifugal acceleration. Note that both contributions are first and second order in \( \Omega \), respectively, and since \( \Omega \ll 1 \), they can be considered small corrections.

If we move to a frame of reference fixed on the surface of the Earth at position \( \mathbf{R} \), then we have to replace \( \ddot{\mathbf{r}} \) with \( \ddot{\mathbf{r}} + \ddot{\mathbf{R}} \), in the transformation above. The equation of motion in the rotating system then reads

\[
m\ddot{\mathbf{r}} = \ddot{\mathbf{T}} - mgk - m\mathbf{\Omega} \times (\mathbf{\Omega} \times \mathbf{R}) - 2m\mathbf{\Omega} \times \dot{\mathbf{r}} - m\mathbf{\Omega} \times (\mathbf{\Omega} \times \mathbf{r}),
\]

where \( \ddot{\mathbf{T}} \) is now referred to the frame of reference on the surface of the Earth. The term \( \ddot{\mathbf{g}}_e \equiv gk + m\mathbf{\Omega} \times (\mathbf{\Omega} \times \mathbf{R}) \) defines the so-called effective gravity, which takes into account the
Figure 2. For a simple pendulum with small amplitude approximations give $T_i = -T_i/l$, with $i = x, y, l - z$ and $T \simeq mg$.

The linear approximation for the pendulum equations corresponds to the assumption that the tension vector is given by (see figure 2)

$$\vec{T} = -\frac{mg}{l} [x\hat{i} + y\hat{j} + (l - z)\hat{k}],$$

valid for small amplitudes. Here, $l$ is the length of the pendulum, and the origin of coordinates is at its supporting point. Introducing the notation $\omega \equiv \sqrt{g/l}$, we get from (4) and (5)

$$\ddot{r} = \omega^2 \hat{r} - \vec{\Omega} \times \dot{\vec{r}} - \vec{\Omega} \times (\vec{\Omega} \times \hat{r}),$$

or, explicitly in components

$$\ddot{x} = -\omega^2 x + 2s\Omega\dot{y} + s^2\Omega^2 x + sc\Omega^2 z,$$
$$\ddot{y} = -\omega^2 y - 2s\Omega\dot{x} - 2c\Omega\dot{z} + \Omega^2 y,$$
$$\ddot{z} = -\omega^2 z + 2c\Omega\dot{y} + cs\Omega^2 x + c^2\Omega^2 z,$$

where we have used the notation

$$s \equiv \sin \beta, \quad c \equiv \cos \beta.$$

We remind the reader that in these equations the $z$ axis is vertically upward, the $x$ axis is north to south, and the $y$ axis is west to east.
The approximation in (5) is crucial from a practical point of view, for it leads to the linear differential system (7), which is homogeneous with constant coefficients, and therefore solvable. One standard solution method consists of transforming (7) into a linear differential equation of sixth order. Or one can alternatively transform (7) into a system of first order and dimension 6. Both procedures are indeed cumbersome and, to the best of our knowledge, no explicit exact solution is found in textbooks.

3. Approximate trajectories in two dimensions

In addition to the assumption of small amplitude oscillations implicit in (5), two further approximations are usually carried out in order to obtain approximate trajectories. Firstly, one can assume \( \Omega \ll \omega \), and then ignore the centrifugal acceleration, namely the quadratic terms \( \Omega^2 \) in (7). This provides a remarkable simplification. A further approximation considers that the pendulum length \( l \) is large enough for the motion of the pendulum to take place in a plane. Thus \( z, \dot{z}, \ddot{z} \) are dropped in (7) and the equations of motion of the planar pendulum in the small amplitude approximation (without centrifugal acceleration) reduce to the two-dimensional system

\[
\begin{align*}
\dot{x} &= -\omega^2 x + 2s\Omega y, \\
\dot{y} &= -\omega^2 y - 2s\Omega \dot{x}.
\end{align*}
\]

Due to the symmetry of these equations a convenient way to solve them is to introduce the complex representation \( Z = x + iy \). Then the system (9) may be written in terms of a sole complex equation

\[
\dot{Z} + 2i\Omega \dot{Z} + \omega^2 Z = 0,
\]

with \( \Omega \equiv s\Omega \). This is a linear homogeneous differential equation of second degree whose solution reads \( Z(t) = A \exp(\lambda_+ t) + B \exp(\lambda_- t) \); and \( A, B \) are complex arbitrary constants. Here \( \lambda_\pm \equiv -i(\Omega \pm \xi) \), \( \xi \equiv \sqrt{\omega^2 + \Omega^2} \), are the (complex) roots of the second degree equation \( \lambda^2 + 2i\Omega \lambda + \omega^2 = 0 \). The complex solution becomes

\[
Z(t) = \exp(-i\Omega t)[A \exp(i\xi t) + B \exp(-i\xi t)].
\]

The four real quantities contained in the two complex constants \( A, B \) are determined so that \( x(t), y(t) \) satisfy the given initial conditions \( x(0), \dot{x}(0), y(0), \dot{y}(0) \). This requires some further algebra, but just to interpret the structure of the solutions let us think of initial conditions such that \( A \) and \( B \) are complex conjugates of each other. In that case the terms in square brackets stand for a real function and the solutions \( x(t), y(t) \) may be written simply as

\[
\begin{align*}
x(t) &= a \cos(\Omega t) \cos(\omega t + \phi), \\
y(t) &= a \sin(\Omega t) \cos(\omega t + \phi),
\end{align*}
\]

in terms of an amplitude \( a \), a phase \( \phi \), and where, in addition, we have approximated \( \xi \simeq \omega \).

Thus the trajectories as a function of \( t \) are the product of a fast oscillation of frequency \( \omega \) describing the natural motion of a simple pendulum, and a slow oscillation of frequency \( \Omega \) which originates a rotation of the pendulum oscillation plane.

By using elementary matrix methods, the general solution of (9) may be expressed in terms of initial conditions. To get rid of the complex representation in (11) we write \( A \equiv a_r + ia_i, B \equiv b_r + ib_i \), in terms of new four real parameters \( a_r, a_i, b_r, b_i \). Next, using the Euler formula in \( Z(t) \) and \( \dot{Z}(t) \), after some algebra equation (11) yields

\[
\begin{align*}
\begin{pmatrix} x(t) \\ \dot{x}(t) \\ y(t) \\ \dot{y}(t) \end{pmatrix} &= \begin{pmatrix} \cos \sigma t & -\sin \sigma t & \cos \rho t & \sin \rho t \\ -\sigma \sin \sigma t & -\rho \cos \sigma t & -\rho \sin \rho t & \rho \cos \rho t \\ \sin \sigma t & \cos \sigma t & -\sin \rho t & \cos \rho t \\ \sigma \cos \sigma t & -\sigma \sin \sigma t & -\rho \cos \rho t & -\rho \sin \rho t \end{pmatrix} \begin{pmatrix} a_r \\ a_i \\ b_r \\ b_i \end{pmatrix} = M(t) \begin{pmatrix} a_r \\ a_i \\ b_r \\ b_i \end{pmatrix},
\end{align*}
\]

\( \sigma, \rho \) are dropped in (7). This provides a remarkable simplification.
where \( \sigma \equiv \xi - \tilde{\Omega}, \rho \equiv \xi + \tilde{\Omega} \), with \( \xi \) already defined above. Equation (13) defines the time dependent matrix \( M(t) \). At time \( t = 0 \) we have

\[
\begin{pmatrix}
  x(0) \\
  \dot{x}(0) \\
  y(0) \\
  \dot{y}(0)
\end{pmatrix} = M(0) \begin{pmatrix}
  a_r \\
  a_i \\
  b_r \\
  b_i
\end{pmatrix}.
\] (14)

Therefore, we can now give the trajectories in terms of initial conditions

\[
\begin{pmatrix}
  x(t) \\
  \dot{x}(t) \\
  y(t) \\
  \dot{y}(t)
\end{pmatrix} = M(t)M^{-1}(0) \begin{pmatrix}
  x(0) \\
  \dot{x}(0) \\
  y(0) \\
  \dot{y}(0)
\end{pmatrix}.
\] (15)

A straightforward computation yields

\[
M^{-1}(0) = \begin{pmatrix}
  \rho & 0 & 0 & 1 \\
  0 & -1 & \rho & 0 \\
  \sigma & 0 & 0 & -1 \\
  0 & 1 & \sigma & 0
\end{pmatrix}.
\] (16)

The explicit solution for the planar, small amplitude pendulum, with Coriolis and without centrifugal accelerations then reads

\[
x(t) = \frac{1}{\xi}[ (\xi \cos \Omega t \cos \xi t + \tilde{\Omega} \sin \tilde{\Omega} \sin \xi t)x_0 + (\cos \tilde{\Omega} t \sin \xi t)x_0 \\
\quad + (\xi \sin \Omega t \cos \xi t - \tilde{\Omega} \cos \tilde{\Omega} t \sin \xi t)y_0 + (\sin \tilde{\Omega} t \sin \xi t)y_0],
\] (17)

\[
y(t) = -\frac{1}{\xi}[ (\xi \sin \Omega t \cos \xi t + \tilde{\Omega} \cos \tilde{\Omega} t \sin \xi t)x_0 - (\sin \tilde{\Omega} t \sin \xi t)x_0 \\
\quad + (\xi \cos \Omega t \cos \xi t + \tilde{\Omega} \sin \tilde{\Omega} t \sin \xi t)y_0 - (\cos \tilde{\Omega} t \sin \xi t)y_0],
\] (18)

where we use the notation \( x(0) = x_0, y(0) = y_0, \dot{x}(0) = \dot{x}_0, \dot{y}(0) = \dot{y}_0 \).

Equations (17) and (18) allow us to illustrate the shape of trajectories for different sets of initial conditions, and in particular those that give rise to the Foucault and Bravais pendula. The Java applet\(^3\) allows an easy visualization of the trajectories for arbitrary values of the parameters.

Next we illustrate the pendulum trajectories with three different choices of initial conditions. We have chosen the following parameter values: length of the wire \( l = 100 \text{ m} \), which gives the angular velocity \( \omega = \sqrt{g/l} = 0.3162 \text{ rad s}^{-1} \); rotating reference system velocity \( \Omega = \omega/10 \); and latitude \( \beta = \pi/4 \). The Earth’s rotation velocity is artificially increased for the sake of clarity in the figures. In all the figures, the solid dot stands for the initial position. The arrow establishes the direction of turn.

### 3.1. Foucault pendulum in two dimensions

The Foucault pendulum corresponds to the simplest possible choice of initial conditions. It is obtained when we release the bob with vanishing initial velocity (in the Earth attached system) from, say, the point \( (x_0 \neq 0, y_0 = 0) \). We illustrate in figure 3, left panel, a trajectory in the plane \( xy \). The initial values are: \( x_0 = 1, y_0 = 0, \dot{x}_0 = \dot{y}_0 = 0 \). Also, in the right panel of figure 3 we plot the corresponding trajectory in the plane of velocities \( \dot{x} \dot{y} \).

\(^3\) www.uv.es/oteo/foucaultbravais.
This case plainly and elegantly combines the motion of a single, planar pendulum with the precession of its plane of oscillation. The deflection of the plane of oscillation originates in the Coriolis acceleration. Thus, the term $2\vec{\Omega} \times \vec{r}$ deflects the trajectory to the right of the velocity vector in the northern hemisphere. Hence, every half oscillation of the pendulum produces an incremental effect. The drawback of this set-up is that, as stated earlier, the effects take several hours to become evident. For a Foucault pendulum located at the latitude of Paris, for example, it takes about $32 \text{ h}$ to complete a precession cycle. To plot neat trajectory patterns we have used a much more greater angular velocity than that corresponding to the value of Earth’s rotation.

### 3.2. Bravais pendulum in two dimensions

The Bravais, or conical, pendulum is the second simplest choice with regard to initial conditions. It is obtained when we release the bob from a point $(x_0 \neq 0, y_0 = 0)$ with an initial tangential velocity $(\dot{x}_0 = 0, \dot{y}_0 = x_0 (\omega - \tilde{\Omega}))$. Then the pendulum describes conical oscillations and the planar trajectory is a circumference.

If we reverse the initial velocity, $(\dot{x}_0 = 0, \dot{y}_0 = -x_0 (\omega - \tilde{\Omega}))$, the trajectory is only conical-like. The planar projection is no longer a circumference, but is rather an open trajectory. This is due to the effect of the rotation of the Earth on the motion of the pendulum bob.

This is illustrated in figure 4. The black solid line corresponds to the conical pendulum released at $t = 0$ with the initial conditions: $x_0 = 1, y_0 = 0, \dot{x}_0 = \dot{y}_0 = 0.13142$. The red dotted line describes the trajectory started with reversed initial velocity. In figure 4 the left panel corresponds to the trajectory in the plane and in the right panel we plot the corresponding trajectory in the plane of velocities $\dot{x}\dot{y}$.

Due to the Earth’s rotation, the left- and right-handed pendula take slightly different times to complete one turn, besides the slight difference in the shape of their paths. The delay is cumulative and their mutual phase difference becomes sizeable after completing a moderate number of turns. It therefore provides a quick and easy proof of the Earth’s rotation, at least in an ideal experiment with two simultaneous pendula.

*Figure 3.* Foucault pendulum in two dimensions. Trajectory in phase plane (left panel) and in the velocity phase plane (right panel). The solid dot stands for the initial position. For the numerical values of the parameters see the main text.
3.3. General pendulum in two dimensions

It is also interesting to observe the trajectory when the initial velocity has two non-vanishing components. In figure 5 (left panel) we plot the trajectory corresponding to initial conditions \(x_0 = 1, y_0 = 0, \dot{x}_0 = -x_0(\omega - \Omega \sin \beta), \dot{y}_0 = x_0(\omega - \Omega \sin \beta)\). In the right panel of figure 5 we represent the corresponding trajectory in the plane of velocities \(\dot{x}, \dot{y}\).
4. The linear pendulum in three dimensions: exact solution

Let us consider the equations of motion (4) of the pendulum in three dimensions without any further approximation, namely, we keep both Coriolis and centrifugal accelerations

\[ \ddot{\vec{r}} + 2\vec{\Omega} \times \dot{\vec{r}} + \vec{\Omega} \times (\vec{\Omega} \times \vec{r}) + \omega^2 \vec{r} = 0. \]  
(19)

The vector products above, being linear operations, admit a matrix representation. For instance

\[ \vec{\Omega} \times \vec{r} = \Omega \begin{pmatrix} 0 & -\sin \beta & 0 \\ \sin \beta & 0 & \cos \beta \\ 0 & -\cos \beta & 0 \end{pmatrix} \vec{r} = \Omega \Lambda \vec{r}. \]  
(20)

Equation (20) defines the matrix \( \Lambda \) of the cross product associated to the unit vector \( \hat{\Omega} \). By the same token, we have \( \vec{\Omega} \times \dot{\vec{r}} = \Omega \Lambda \dot{\vec{r}} \) and \( \vec{\Omega} \times (\vec{\Omega} \times \vec{r}) = \Omega^2 \Lambda^2 \vec{r} \). Now (19) may be written in matrix form

\[ \ddot{\vec{r}} + 2\Omega \Lambda \dot{\vec{r}} + [\omega^2 + \Omega^2 \Lambda^2] \vec{r} = 0, \]  
(21)

which we solve next.

We commence with a common undergraduate exercise for students on ordinary differential equation courses. They are asked to transform the scalar second order differential equation

\[ \frac{d^2 y}{dx^2} + a(x) \frac{dy}{dx} + b(x) y = 0, \]  
(22)

into its canonical form, namely the one where the coefficient of the first order derivative vanishes. To this end one proposes a factorization of the dependent variable \( y(x) \) as \( y(x) = u(x) v(x) \), with \( v(x) \) the new dependent variable and \( u(x) \) to be determined later. Substitution in (22) yields

\[ \frac{d^2 v}{dx^2} + \left( \frac{2}{u} \frac{du}{dx} + a \right) \frac{dv}{dx} + \frac{1}{u} \left( \frac{d^2 u}{dx^2} + a \frac{dy}{dx} + bu \right) = 0. \]  
(23)

If we choose

\[ u(x) = \exp \left[ -\frac{1}{2} \int a(s) \, ds \right], \]  
(24)

then we eventually get the canonical form

\[ \frac{d^2 v}{dx^2} + \left[ b(x) - \frac{1}{4} (a(x))^2 - \frac{1}{2} \frac{d}{dx} a(x) \right] v = 0. \]  
(25)

We now put forward the question of finding the canonical form of matrix equation (21). Following a similar procedure to that above, we propose the substitution

\[ \vec{r}(t) = U(t) \vec{q}(t), \]  
(26)

in (21), where \( U \) is a square real matrix function of dimension 3. Then, the canonical matrix differential equation, analogous to (23), reads

\[ \frac{d}{dt} U + \Omega \Lambda U = 0, \]  
(27)

Its solution yields

\[ U(t) = C \exp(-\Omega \Lambda t), \quad C \in \mathbb{R}^{3 \times 3}. \]  
(28)

Since the particular value of the constant matrix \( C \) is irrelevant for our purposes, we choose the simplest option, namely \( C = I \). As \( U(t) \) is not singular we get, after some algebra, the canonical form

\[ \frac{d^2 \vec{q}}{dt^2} + \omega^2 \vec{q} = 0. \]  
(29)
Its solution in terms of initial conditions reads
\[ \ddot{q}(t) = \cos(\omega t) \ddot{q}(0) + \frac{1}{\omega} \sin(\omega t) \dot{q}(0). \] (30)

It thus turns out that the idea of finding the canonical form of the equations of motion has led to an uncoupled problem. Moreover, equation (29) has an important dynamical interpretation. It tells us that \( U(t) \) transforms the dynamics into three free uncoupled oscillators. We know that this picture corresponds to an analysis in the inertial coordinate system fixed in space (i.e., not turning with the Earth). In this reference system the motion of the pendulum is observed as the composition of three free harmonic oscillators along the axes \( x, y, z \).

At this point a comment about the rhs of (28) is in order. The exponential of a square matrix \( B \) is defined by the Taylor expansion:
\[ \exp B = I + B + \frac{1}{2!} B^2 + \frac{1}{3!} B^3 \ldots \] (31)

In general, the problem of finding the exponential of a matrix may be involved, but efficient methods do exist [8, 9]. However, due to the symmetry of the matrix \( \Lambda \), in our case the computation may be carried out easily by inspection of the powers \( \Lambda^n \). It is easy to verify that
\[ \Lambda^3 = -\Lambda, \quad \Lambda^4 = -\Lambda^2. \] (32)

So, recurrent substitution in the Taylor expansion yields
\[ \exp(\Omega \Lambda t) = I - \sin(\Omega t) \Lambda + (1 - \cos \Omega t) \Lambda^2, \] (33)

which provides us with an explicit matrix representation of \( U(t) \).

To recover the solution in the rotating reference system fixed to the Earth we use equation (26)
\[ \vec{r}(t) = U(t)\vec{q}(t) = e^{-\Omega \Lambda t} \left[ \cos(\omega t) \ddot{q}(0) + \frac{1}{\omega} \sin(\omega t) \dot{q}(0) \right]. \] (34)

We still have to establish the relationship between the initial conditions in the inertial reference system: \( \ddot{q}(0), \dot{q}(0) \); and those in the rotating reference system fixed to the Earth: \( \vec{r}(0), \dot{r}(0) \). Equation (34) and its derivative at time \( t = 0 \) give
\[ \vec{r}(0) = \ddot{q}(0), \quad \dot{r}(0) = \dot{q}(0) + \Omega \Lambda \ddot{q}(0). \] (35)

The exact solution in terms of initial conditions, in the system of reference that rotates with the Earth, after some algebra is found to be
\[ \vec{r}(t) = \frac{1}{\omega} \left[ \omega (1 - \cos \Omega t \cos \omega t - \Omega \sin \omega t \sin \Omega t) \Lambda^2 \ddot{q}(0) + (1 - \cos \Omega t \sin \omega t) \Lambda^2 \dot{r}(0) \right. \]
\[ + \left[ \Omega \cos \Omega t \sin \omega t - \omega \sin \Omega t \cos \omega t \right] \Lambda \ddot{r}(0) - \sin \Omega t \sin \omega t \Lambda \ddot{r}(0) \]
\[ + \omega \cos \Omega t \dot{r}(0) + \sin \omega t \dot{r}(0). \] (36)

Taking into account trigonometric identities for the product of sin and cos functions we can conclude that the motion is given by the superposition of three frequencies: \( \omega \pm \Omega \) and \( \omega \). This is at variance with the approximate trajectories in two dimensions (17) and (18), where only the two frequencies, \( \xi \pm \Omega \approx \omega \pm s \Omega \), are present. In the case of the Earth’s rotation, it must not be relevant, for we should be able to resolve between two and three peaks in the power spectrum of, say, \( x(t) \), where \( \Omega \ll \omega \).

Since that \( \Lambda \) is the matrix representation of the vector product associated with the unit vector \( \hat{\Omega} \), we can also write
\[ \vec{r}(t) = \frac{1}{\omega} \left[ \omega (1 - \cos \Omega t \cos \omega t - \Omega \sin \omega t \sin \Omega t) \hat{\Omega} \times (\hat{\Omega} \times \ddot{q}(0)) \right. \]
\[ + \left[ (1 - \cos \Omega t) \sin \omega t \right] \hat{\Omega} \times (\hat{\Omega} \times \dot{r}(0)) \]
\[ + \left[ \Omega \cos \Omega t \sin \omega t - \omega \sin \Omega t \cos \omega t \right] \hat{\Omega} \times \dot{r}(0) \]
\[ - \sin \Omega t \sin \omega t \hat{\Omega} \times \ddot{r}(0) + \omega \cos \Omega t \ddot{r}(0) + \sin \omega t \dot{r}(0). \] (37)
Note that in the limit $\Omega \to 0$ the equation above gives the expected result $\vec{r}(t) = \cos \omega t \; \vec{r}(0) + \frac{1}{\omega} \sin \omega t \; \vec{r}(0)$.

In terms of components the exact solution reads

$$x(t) = \left[ \left\{ c^2 + \cos(\Omega t) x^2 \right\} \cos(\omega t) + \frac{\Omega}{\omega} \sin(\Omega t) \sin(\omega t) x^2 \right] x(0) - \left[ \frac{\Omega}{\omega} \cos(\Omega t) \sin(\omega t) - \sin(\Omega t) \cos(\omega t) \right] y(0) + \left[ \frac{\Omega}{\omega} \sin(\Omega t) \sin(\omega t) - 1 - \cos(\Omega t) \right] \cos(\omega t) \right] s c z(0) + \frac{\sin \omega t}{\omega} \left[ \left[ c^2 + \cos(\Omega t) x^2 \right] x(0) + \sin(\Omega t) s y(0) - \left[ 1 - \cos(\Omega t) \right] s c z(0) \right]$$

$$y(t) = \left[ \frac{\Omega}{\omega} \cos(\Omega t) \sin(\omega t) - \sin(\Omega t) \cos(\omega t) \right] \left[ s x(0) + c z(0) \right] + \left[ \frac{\Omega}{\omega} \sin(\Omega t) \sin(\omega t) + \cos(\Omega t) \cos(\omega t) \right] y(0) - \frac{\sin \omega t}{\omega} \left[ \sin(\Omega t) s x(0) - \cos(\Omega t) y(0) + \sin(\Omega t) c z(0) \right]$$

$$z(t) = \left[ \frac{\Omega}{\omega} \sin(\Omega t) \sin(\omega t) - 1 - \cos(\Omega t) \right] \cos(\omega t) \right] s c x(0) + \left[ \sin(\Omega t) \cos(\omega t) - \frac{\Omega}{\omega} \cos(\Omega t) \sin(\omega t) \right] c y(0) + \left[ \left[ c^2 + \cos(\Omega t) c^2 \right] \cos(\omega t) + \frac{\Omega}{\omega} \sin(\Omega t) \sin(\omega t) c^2 \right] z(0) - \frac{\sin \omega t}{\omega} \left[ \left[ 1 - \cos(\Omega t) \right] s c z(0) - \sin(\Omega t) c y(0) - \left[ c^2 + \cos(\Omega t) c^2 \right] z(0) \right]$$

where $s, c$ have been defined in (8).

We have used the expressions above to plot the trajectories corresponding to the very same initial conditions as in the figures of section 3. The paths are rather different.

### 4.1. Foucault pendulum in three dimensions

In figure 6 we represent the exact trajectory corresponding to the Foucault pendulum with the same initial conditions as in figure 3. The left panel presents the $xy$ projection and can be compared with the pattern in figure 3 (left panel). Note the difference in symmetry. The right panel gives the $xz$ projection. In the two-dimensional case it is simply $z(t) = 0$.

### 4.2. Bravais pendulum in three dimensions

The Bravais pendulum trajectories are represented in figure 7. The solid black line corresponds to a turn parallel to $\vec{\Omega}$, and the dotted red line is the trajectory with the initial tangential velocity reversed. We have used the same condition for the initial velocity $\dot{y}(0)$ as for the planar case. It turns out that after one cycle the oscillation is no longer conical-like. The time delay between both turning modes is also apparent, as in the two-dimensional case.
4.3. The general pendulum in three dimensions

The more general mode of oscillation of figure 5 is replicated in figure 8 for the three-dimensional pendulum. The left and right panels give the $xy$ and $xz$ projections, respectively. As in the two preceding subsections, the differences in the patterns are noteworthy, and these are discussed in the final section.
5. Chevilliet’s theorem

Let us consider the case where we release the pendulum bob at $\vec{r}(0) = (\epsilon, 0, \lambda)^\top$, from rest: $\vec{r}(0) = 0$, with $\lambda \equiv l - \sqrt{l^2 - \epsilon^2}$, and $\epsilon \ll l$. It is then interesting to study the trajectory as it is seen from the inertial reference system fixed in space. According to (35), the corresponding initial conditions are $\vec{q}(0) = (\epsilon, 0, \lambda)^\top$, $\dot{q}(0) = (0, -\Omega [s\epsilon + c\lambda], 0)^\top$, where $s, c$ have been defined in (8). Then (30) gives

$$
q_x(t) = \epsilon \cos(\omega t),
q_y(t) = -\frac{\Omega}{\omega} \left[ s\epsilon + c\lambda \right] \sin(\omega t),
q_z(t) = \lambda \cos(\omega t).
$$

(41)

It is straightforward to see that, in three dimensions, the observed trajectory takes place on the ellipsoid

$$
\frac{q_x^2}{\epsilon^2 + \lambda^2} + \frac{q_y^2}{(s\epsilon + c\lambda)^2} \cdot \frac{\omega^2}{\Omega^2} = 1.
$$

(42)

The intersection with the plane $q_z = 0$ is the ellipse

$$
\frac{q_x^2}{\epsilon^2 + \lambda^2} + \frac{q_y^2}{(s\epsilon + c\lambda)^2} \cdot \frac{\omega^2}{\Omega^2} = 1,
$$

(43)

whose ratio of axes in the limit $\epsilon \ll l$ can be approximated by

$$
\frac{\Omega}{\omega} \left( s + c \frac{\epsilon}{2l} \right),
$$

(44)

since $\lambda \approx \epsilon^2/2l$. Thus, the planar projection of the pendulum motion is an ellipse as a consequence of the non-vanishing initial (tangential) velocity provided by the rotation of the Earth on the bob. Otherwise the trajectory would be on a straight segment.
Had we resolved the same initial value problem with the approximated pendulum equations in two dimensions of section 3, we would have found that the trajectory in the fixed reference system describes the ellipse

$$\frac{q_x^2}{\epsilon_x^2} + \frac{q_y^2}{\epsilon_y^2} \left( \frac{\omega}{s \Omega} \right)^2 = 1,$$

(45)

whose axes ratio, $s \Omega/\omega$, is determined simply by the ratio of the Earth rotation frequency at latitude $\beta$ to that of the natural frequency of the pendulum. This is an almost forgotten result, known in older texts as the Chevilliet theorem [10–12]. Interestingly, we can also recover this theorem by performing a further approximation in (43). Indeed, the drastic approximation $\lambda = 0$ immediately leads to the Chevilliet theorem. Thus, the solution in three dimensions (44) provides a small correction to Chevilliet’s ellipse (45). We are now ready to accept that the convoluted patterns we observe in figures such as figure 3 are a direct consequence of observing the motion on a Chevilliet ellipse from a rotating reference system.

We can buttress the idea of the geometric origin of the convoluted patterns in the phase plane as follows. Consider, in the exact solution in (37), the subtraction

$$\vec{S}(t) \equiv \vec{r}(t) - \left[ \omega \cos \omega t \vec{r}(0) + \frac{1}{\omega} \sin \omega t \dot{\vec{r}}(0) \right].$$

(46)

We can interpret $\vec{S}$ as the remaining line of motion that is left once the elliptic piece has been removed (i.e, the non-perturbed or inertial part) from the exact trajectory. Next, due to the mathematical properties of the cross and scalar products, we get for that remainder $\hat{\Omega} \cdot \vec{S}(t) = 0, \forall t$. Hence, every additional contribution to the trajectory originated in Coriolis and centrifugal acceleration terms takes place in a plane orthogonal to $\hat{\Omega}$. There is no correction to the trajectory in the very direction of the Earth’s angular velocity vector $\vec{\Omega}$.

The flowery nature of the patterns of the pendulum trajectories emerges, therefore, from the non-inertial character of the rotating frame. Everything should appear as an ellipse in the phase plane if we are able to choose the proper reference system, so to speak.

Finally, neither the solution in two nor that in three dimensions preserves the constance of the length of the pendulum. This is not a consequence of the non-vanishing angular velocity of the reference system, but of the approximations made to get a linear system of differential equations. It is the price paid when replacing a non-linear problem (the physical pendulum) with a linear one (the simple pendulum) with a linear one (the simple pendulum). Namely, the constraint $x^2 + y^2 + (l - z)^2 = l^2$ (or, equivalently, $q_x^2 + q_y^2 + (l - q_z)^2 = l^2$) which represents a sphere of radius $l$, centered in $(0, 0, l)$, is not preserved along a trajectory. Instead of a sphere we have the ellipsoid (42). The same drawback occurs in the approximate treatment in two dimensions. There the constraint is $x^2 + y^2 = 0$ (or, $q_x^2 + q_y^2 = 0$), which is by no means preserved but is replaced with the Chevilliet ellipse constraint (45). Figure 9 illustrates this discrepancy in two (left panel) and three (right panel) dimensions as a function of time. Both functions are periodic but with different periods. For a pendulum length $l = 100$ m in the two-dimensional model the period is 20 s and the maximum relative deviation is 1/100. In three dimensions the period is 200 s and the maximal relative deviation is 14/100.

6. Foucault pendulum: comparison of the two- and three-dimensional cases

At this point we have already realized that the trajectories obtained from the two-dimensional and the $xy$ phase plane from the three-dimensional system of equations differ. Next we study whether these differences are relevant to analysis of Foucault pendula placed on the Earth.

In his original papers [2] Foucault explained how in Paris the pendulum took *circa* 30 h to complete one turn, corresponding to $1^\circ$ in 5 min. That is to say that in a museum ordinary
visitors observe, at most, that the pendulum bob describes a small arc of circumference during their visit. In the original set-up at the Panthéon there was a pile of damp sand disposed in circular shape around the pendulum equilibrium point that a stylus attached to the underside of the bob plowed, showing the cumulative deflection of the trajectory. Modern museum set-ups instead use falling rods or balls.

The solutions (17) and (18) for the two-dimensional model show that the bob sweeps a circular area after a complete turn. This is represented in the left panel of figure 10. The black dots stand for the location of the bob, sampled every 0.4 s (there is no need to seek for any
particular pattern inside). The pendulum length is 100 m and $\Omega = 0.000073$ rad s$^{-1}$ is the real angular velocity for the Earth. The bob is released from rest (in the Earth attached reference system) at $x(0) = 1$ m, $y(0) = 0$. In addition, the right panel in figure 10 represents the corresponding three-dimensional case from (38) and (39). Here the red solid line corresponds to the pattern border in the preceding panel, and it is included for the sake of comparison. This dual cardioid-like shape is certainly not the pattern observed in museums or reported by Foucault.

There is an untold piece in this story that provides a reasonable explanation for the disagreement. Air resistance gradually damps the oscillation of the pendulum. Foucault wrote [2] that every five or six hours (i.e., a deflection of 60°–70°) the experiment must be stopped in order to restart it from that very precise position with the original oscillation amplitude restored. The blue arrow in figure 10 points out that event and, without further information, the receding of the black border could be erroneously ascribed to damping. Of course, the air resistance effect is incremental.

The course of the experiment is then punctuated by restarting and, as a consequence, what one sees is the juxtaposition of several copies of the first 60°, an (almost) circular sector of the pattern in the right panel of figure 10. The overall composition mimics the circular pattern.

Thus damping prevents experimental observation of the dual cardioid-like pattern. The decrease of the oscillation amplitude is caused by the constructive combination of damping and the three-dimensional nature of the motion. Indeed, Foucault’s experiment was not designed to resolve these two contributions, but to exhibit the deflection of the rotation plane of the pendulum. What we argue is that, in practical realizations, both panels in figure 10 are not in conflict.

In 1855 a Foucault pendulum was established as a permanent exhibit at Ars et Métiers during the Universal Exhibition held in Paris. At this time an ingenious electromagnetic device was added, so that the experiment needed no restarting. An electromagnet placed on the ground exerted attraction on the bob only when this was approaching the equilibrium point. When it crossed the vertical and the bob was receding the action of the electromagnet stopped. A mechanical synchronization device restarted the current once the bob was again in the approaching phase.

Modern pendula exhibitions also incorporate a driving apparatus. For instance, the Museum of Arts and Sciences, Valencia, Spain also has a permanent exhibit. The pendulum is driven electromagnetically through a system located in its upper part. The device actions are controlled by an electronic automata. The question arises then—to what extent is the attractor of the system modified? We will not enter into this subject here and will limit ourselves to pointing out that the solvable, linear, homogeneous, second order system of differential equations with constant coefficients in (7) becomes a different class of equation: non-constant coefficients, non-homogeneous or non–linear, depending on the type of driving. In general, they become non-solvable.

7. Final comments

The Foucault pendulum and its generalizations are not only beautiful museum experiments and excellent tools from a pedagogical point of view (as they contain an ideal admixture of mechanics and mathematics), they have far-reaching applications. Foucault-pendulum-like motion can be viewed as a relative orbital motion under a central force, like the relative motion of celestial bodies subject to the influence of the same gravitational field, or terminal orbital rendezvous and as such the subject is still an active topic and has actual relevance.
The full richness of these systems is sometimes disguised when they are studied as separate entities. The unified treatment we propose here manifestly shows the crucial role played by initial conditions in determining the specific features of the system. Here, as in spontaneous symmetry breaking, where the Lagrangian possesses a symmetry not respected by the vacuum, the equations of motion are identical in every case, the selection is done by the initial conditions.

Acknowledgments

This work was supported by the Spanish MEC and FEDER (EC) under grant no. FPA2011-23596, by Generalitat Valenciana under grant no. PROMETEO/2008/004 (GB) and by MICINN (Spain) under grant no. AYA2010-22111-C03-02 (JAO).

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