Modeling and semigroup formulation of charge or current-controlled active constrained layer (ACL) beams; electrostatic, quasi-static, and fully-dynamic assumptions

Ahmet Özkan Özer

Abstract

A three-layer active constrained layer (ACL) beam model, consisting of a piezoelectric elastic layer, a stiff layer, and a constrained viscoelastic layer, is obtained for cantilevered boundary conditions by using the reduced Rao-Nakra sandwich beam assumptions through a consistent variational approach. The Rao-Nakra sandwich beam assumptions keeps the longitudinal and rotational inertia terms. We consider electrostatic, quasi-static and fully dynamic assumptions due to Maxwell’s equations. For that reason, we first include all magnetic effects for the piezoelectric layer. Two PDE models are obtained; one for the charge-controlled case and one for the current-controlled case. These two cases are considered separately since the underlying control operators are very different in nature. For both cases, the semigroup formulations are presented, and the corresponding Cauchy problems are shown to be well-posed in the natural energy space.

Keywords: current actuation, charge actuation, Rao-Nakra smart sandwich beam, piezoelectric beam, cantilevered active constrained layer beam, Maxwell’s equations.

1. Introduction

A three-layer actively constrained layer (ACL) beams is an elastic beam consisting of a stiff elastic beam and a piezoelectric beam constraining a viscoelastic beam which creates passive damping in the structure. The piezoelectric beam itself an elastic beam with electrodes at its top and bottom surfaces, insulated at the edges (to prevent fringing effects), and connected to an external electric circuit (see Fig. 1). These structures are widely used in in civil, aeronautic and space space structures due to their small size and high power density. They convert mechanical energy to electro-magnetic energy, and vice versa. ACL
composites involve piezoelectric layers and utilizes the benefits of them. Modeling these composites requires understanding the piezoelectric modeling since the ACL composites are generally actuated through their piezoelectric layers. There are mainly three ways to electrically actuate piezoelectric materials: voltage, current or charge. Piezoelectric materials have been traditionally activated by a voltage source [2, 26, 27], and the references therein. However it has been observed that the type of actuation changes the controllability characteristics of the host structure. The main difference between three types of actuation can be classified in three different groups:

(i) **Electrical nonlinearity (hysteresis):** Piezoelectric actuators actuated by a voltage source display nonlinear behavior at the higher drives. This is so-called hysteresis which results from the residual misalignment of crystal grains in the poled ceramic [15]. For this reason, voltage-actuated piezoelectric beams are only actuated at low drives. This prevents them being used to their full potential. The hysteresis also causes a tracking error of 15% of the displacement stroke, see [2] the references therein. It is observed that as the actuation is chosen charge or current instead of voltage, there is a substantial decrease in hysteresis, i.e., [2, 4, 12, 13, 15], and the references therein.

(ii) **The bounded-ness of the control operator:** Another difference is whether the underlying control operator in the state-space formulation of the control system is bounded in the natural energy space. In the case of voltage and charge actuation, the underlying control operator is unbounded in the natural energy space, $\delta(\cdot)$ operators appear at the right side of the equations depending on the locations of sensors/actuators [9, 16]. However, the control operator is bounded in the case of current actuation, which is recently observed in [17] by using the potential formulation of the full set of Maxwell’s equations in the modeling process.

(iii) **Exact controllability & Uniform stabilizability:** In the case of a single piezoelectric beam, there is no way to exponentially stabilize the current controlled model with the magnetic effects [17]. Only strong stabilizability may be achieved for an infinitely many combinations of material parameters [16]. In the case of voltage actuation, the exponential [16] and polynomial [18] stabilizability can be obtained for certain choices of parameters. This is the motivation behind [17] to derive charge or current-actuated distributed parameter models by using a thorough variational approach.

Accurately modeling an ACL beam also requires understanding of how sandwich structures are modeled and how the interaction of the elastic layers are established. A three-layer sandwich beam consists of stiff outer layers and a viscoelastic core layer. The core layer is supposed to deform only in transverse shear. The bending is uniform for the whole composite. Many sandwich beam models have been proposed in the literature, i.e., see [5, 14, 25, 29, 31, 32], and the references therein. These models mostly differ by the assumptions on the viscoelastic layer. As well, depending on the inclusion of the effects of longitudinal and rotational inertias, there are essentially two well-accepted models. The Mead-Marcus type models [14] disregard these effects whereas the Rao-Nakra type models preserve them since it is noted that these inertia effects are ex-
Figure 1: For a current or charge-actuated ACL beam, when $i_s(t)$ or $\sigma_s(t)$ is supplied to the electrodes of the piezoelectric layer, an electric field is created between the electrodes, and therefore the piezoelectric beam either shrinks or extends, and this causes the whole composite stretch and bend.

The models for the ACL beams proposed in the literature mostly use the above-mentioned sandwich beam assumptions for the interactions of the layers, i.e. [28, 31], and references therein. The massive majority of these models are actuated by a voltage source, i.e. see [1, 8], and the references therein. Moreover, the longitudinal vibrations were not taken into account in those papers; only the bending of the whole composite is studied. In this paper, we use the reduced Rao-Nakra approach for which the longitudinal and rotational inertia terms are kept. We even obtain a reduced Rao-Nakra model by letting the weight and the stiffness of the middle layer go to zero [10]. Thus, a coupled but reduced PDE model is obtained for charge and current actuation. We also consider the assumptions of electrostatic, quasi-static and fully-dynamic in the Maxwell’s equations. The biggest advantage of our models is being able to study controllability and stabilizability problems for ACL beams in the infinite dimensional setting including all mechanical, electrical, and magnetic effects. We show that the proposed models with different types of actuation can be written in semigroup formulation, and they are shown to be well-posed in the natural energy space.

2. Modeling Active-Constrained Layer (ACL) beams

The ACL beam considered in this paper is a composite consisting of three layers that occupy the region $\Omega = \Omega_{xy} \times (0, h) = [0, L] \times [-b, b] \times (0, h)$ at equilibrium where $\Omega_{xy}$ is a smooth bounded domain in the plane. The total thickness $h$ is assumed to be small in comparison to the dimensions of $\Omega_{xy}$. The beam consists of a stiff layer, a compliant layer, and a piezoelectric layer, see Figure 1.
Let \( 0 = z_0 < z_1 < z_2 < z_3 = h \), with
\[
h_i = z_i - z_{i-1}, \quad i = 1, 2, 3.
\]
We use the rectangular coordinates \((x, y)\) to denote points in \(\Omega_{xy}\), and \((X, z)\) to denote points in \(\Omega = \Omega^s \cup \Omega^{ve} \cup \Omega^p\), where \(\Omega^s, \Omega^{ve},\) and \(\Omega^p\) are the reference configurations of the stiff, viscoelastic, and piezoelectric layers, respectively, and they are defined by
\[
\Omega^s = \Omega_{xy} \times (z_0, z_1), \quad \Omega^{ve} = \Omega_{xy} \times (z_1, z_2), \quad \Omega^p = \Omega_{xy} \times (z_2, z_3).
\]
Define
\[
\vec{\psi} = [\psi^1, \psi^2, \psi^3]^T, \quad \vec{\phi} = [\phi^1, \phi^2, \phi^3]^T, \quad \vec{v} = [v^1, v^2, v^3]^T
\]
where
\[
\psi^i = \frac{u^i - u^{i-1}}{h_i}, \quad \phi^i = \psi^i + w_x, \quad v^i = \frac{u^{i-1} + u^i}{2}, \quad i = 1, 2, 3, \quad (1)
\]
\[
\phi^1 = \phi^3 = 0, \quad \psi^1 = \psi^3 = -w_x, \quad \phi^2 = \psi^2 + w_x \quad (2)
\]
Let \(T_{ij}\) and \(S_{ij}\) denote the stress and strain tensors for \(i, j = 1, 2, 3\), respectively. The constitutive equations for the piezoelectric layers are
\[
T^{(3)}_{11} = \alpha^3 S^{(3)}_{11} - \gamma D_3, \quad D_1 = \varepsilon_{11} E_1, \quad D_3 = \gamma S^{(3)}_{11} + \varepsilon_{33} E_3, \quad (3)
\]
where \(D, E, \alpha, \gamma, \) and \(\varepsilon\) are electrical displacement vector, electric field intensity vector, elastic stiffness coefficient, piezoelectric coefficient, permittivity coefficient, and and for the middle and the stiff layers are given as the following:
\[
T^{(i)}_{11} = \alpha^i S^{(i)}_{11}, \quad T^{(i)}_{13} = 2G_2 S^{(i)}_{13}, \quad i = 1, 2 \quad (4)
\]
where \(G_2\) is the shear modulus of the second layer. Since we don’t allow shear in the stiff layer we indeed have \(T^{(i)}_{13} = 0, \quad i = 1, 3\). The strain components for the middle layer are
\[
S^{(2)}_{11} = \frac{\partial v^2}{\partial x} - (z - z_i) \frac{\partial \psi^2}{\partial x}, \quad S^{(2)}_{13} = \frac{1}{2} (\psi^2 + w_x) = \frac{1}{2} \phi^2, \quad (5)
\]
and for the piezoelectric and stiff layers are given by
\[
S^{(i)}_{11} = \frac{\partial u^i}{\partial x} - (z - z_i) \frac{\partial \phi^i}{\partial x}, \quad S^{(i)}_{13} = 0, \quad i = 1, 3. \quad (6)
\]

**Magnetic effects and dynamic modeling:** The full set of Maxwell’s equations is (for instance see [6, Page 332]):
\[
\begin{align*}
\nabla \cdot D &= \sigma_b \quad \text{in} \quad \Omega^p \times \mathbb{R}^+ \quad \text{(Electric Gauss’s law)} \\
\nabla \cdot B &= 0 \quad \text{in} \quad \Omega^p \times \mathbb{R}^+ \quad \text{(Gauss’s law of magnetism)} \\
\n\nabla \times E &= - \dot{B} \quad \text{in} \quad \Omega^p \times \mathbb{R}^+ \quad \text{(Faraday’s law)} \\
\frac{1}{\mu} (\nabla \times B) &= i_b + \dot{D} \quad \text{in} \quad \Omega^p \times \mathbb{R}^+ \quad \text{(Ampère-Maxwell law)}
\end{align*}
\]

4
with one of the essential electric boundary conditions prescribed on the electrodes of the piezoelectric layer

\[
\begin{aligned}
-D \cdot n &= \sigma_s \quad \text{on} \quad \partial \Omega^p \times \mathbb{R}^+ \quad \text{(Charge)} \\
\frac{1}{n} (B \times n) &= i_s \quad \text{on} \quad \partial \Omega^p \times \mathbb{R}^+ \quad \text{(Current)} \\
\phi &= V \quad \text{on} \quad \partial \Omega^p \times \mathbb{R}^+ \quad \text{(Voltage)}
\end{aligned}
\]

and appropriate mechanical boundary conditions at the edges of the beam (the beam is clamped, hinged, free, etc.). Here \( B \) denotes the magnetic field vector, and \( \sigma, i, \sigma_s, i_s, V, \mu, n \) denote body charge density, body current density, surface charge density, surface current density, voltage, magnetic permeability, and unit normal vector to the surface \( \partial \Omega^p \), respectively. In this paper we consider only current and charge-driven electrodes (i.e. we ignore the voltage boundary condition in (8)). The voltage-driven electrode case is handled in details in \([19]\).

By (7), there exists a scalar electric potential \( \phi \) and a vector magnetic potential \( \hat{A} \) such that

\[
B = \nabla \times \hat{A}, \quad E = -\dot{\hat{A}} - \nabla \varphi.
\]

where \( \dot{\hat{A}} \) stands for the induced electric field due to the time-varying magnetic effects. In modeling piezoelectric beams, there are mainly three approaches including electric and magnetic effects \([30]\): Electrostatic, quasi-static, and fully dynamic electric field. The electrostatic case assumes \( B = 0 \), and therefore \( \hat{A} = 0 \). The quasi-static approach ignores \( \hat{A} \) and \( \dot{\hat{A}} \) since \( \hat{A}, \dot{\hat{A}} \ll \varphi \). With this assumption \( \dot{D} \) may be non-zero. However, in the fully dynamic case, \( \hat{A} \) and \( \dot{\hat{A}} \) are left in the model. Depending on the type of material, body charge density \( \sigma_b \) and body current density \( i_b \) can also be non-zero. Note that even though the displacement current \( \dot{D} \) is assumed to be non-zero in both quasi-static and fully dynamic approaches, the term \( \dot{D} \) is zero in quasi-static approach since \( \hat{A} = 0 \).

Henceforth, to simplify the notation, \( x = x_1 \) and \( z = x_3 \).

**Electromagnetic assumptions.** The linear through-thickness assumption of the electric potential \( \varphi(x, z) = \varphi^0(x) + z\varphi^1(x) + \frac{z^2}{2}\varphi^2(x) \) (10)

Since we are in the beam theory, we assume that the magnetic vector potential \( A \) has nonzero components only in \( x \) and \( z \) directions, i.e. \( A_2 \equiv 0 \). To keep the consistency with \( \phi \), we assume that \( A \) is quadratic through-thickness as well:

\[
A_i(x, z) = A_i^0(x) + zA_i^1(x) + \frac{z^2}{2}A_i^2(x), \quad i = 1, 3
\]

By (9)

\[
E_1 = -\left( \dot{A}_0^0 + z\dot{A}_1^1 + \frac{z^2}{2}\dot{A}_2^2 \right) - \left( (\varphi^0)_x + z(\varphi^1)_x + \frac{z^2}{2}(\varphi^2)_x \right),
\]

\[
E_3 = -\left( \dot{A}_3^0 + z\dot{A}_3^1 + \frac{z^2}{2}\dot{A}_3^2 \right) - (\varphi^1 + z\varphi^2).
\]
Work done by the external forces The work done by the electrical external forces (as in [11, 17]) is

\[ W_e = \int_{\Omega^p} \left( -\phi \partial_t \phi + \vec{i}_b \cdot \vec{A} \right) \, dX + \int_{\partial \Omega^p} \left( -\sigma_s \phi + \vec{i}_s \cdot \vec{A} \right) \, dX \]

\[ = -\int_0^L \int_{-h_3/2}^{h_3/2} \sigma_b \left( \phi^0(x) + z \phi^1(x) + \frac{z^2}{2} \phi^2(x) \right) \, dz \, dx \]

\[ + \int_0^L \int_{-h_3/2}^{h_3/2} i_b^1 \left( A^0_1(x) + z A^1_1(x) + \frac{z^2}{2} A^2_1 \right) \, dz \, dx \]

\[ + \int_0^L \left( -\sigma_s (\phi(h_3/2) - \phi(-h_3/2)) + i_s^3 \left( A_1(h_3/2) - A_1(-h_3/2) \right) \right) \, dx \]

\[ = \int_0^L \left( -\sigma_b \left( \phi^0 + \frac{h_3^2}{24} \phi^2 \right) - \sigma_s + i_b^1 \left( A^0_1 + \frac{h_3^2}{24} A^2_1 \right) - \sigma_s \phi^3 + i_s^3 A^3_1 \right) \, dx \] (13)

where \( i_s(x,t) = (i_s^1(x,t), 0, 0) \), and \( i_b(x,t) = (i_b^1(x,t), 0, i_b^3(x,t)) \). In the above \( i_s \) has only one nonzero component since \( i_s \perp B \), and \( i_s \perp n \) by [8]. Moreover, \( i_b \) has only one nonzero component since we assumed that there is no force acting in the \( x_2 \) and \( x_3 \) directions. Several remarks are in order:

**Remark 2.1.** (i) We choose either surface charge \( \sigma_s \) or \( i_s \) to be non-zero, or \( i_b \) or \( \sigma_b \) to be non-zero, depending on the type of actuation. For consistency, we keep them both in deriving PDEs for the rest of the paper; in the case of only surface current actuation \( \sigma_s \neq 0, \sigma_s = i_b = \sigma \equiv 0 \), and in the case of surface charge actuation \( \sigma_s \neq 0, i_s = i_b = \sigma_b \equiv 0 \).

(ii) Since the piezoelectric materials are not perfectly insulated, the electric field \( E \) causes currents to flow when conductivity occurs. Therefore the time-dependent equation of the continuity of electric charge must be employed:

\[ \partial_t \sigma_b + \nabla \cdot i_b = 0 \quad \text{in} \quad \Omega^p \]

\[ \partial_t \sigma_s - i_b^3 = 0, \quad \text{or,} \quad \frac{di_s}{dx} - i_b^3 = 0 \quad \text{on} \quad \partial \Omega^p. \] (14)

For more details, the reader can refer to [14, Section 3.9].

(iii) Note that if the magnetic effects are neglected, a variational approach cannot be used since \( A \equiv 0 \) and so the surface current may not be involved in \( W_e \). This is very different from the charge and voltage actuation cases since the charge and voltage terms are not affected by \( A \) in \( W_e \).

Assume that the beam is subject to a distribution of forces \( (\vec{g}^1, \vec{g}^3, \vec{g}) \) along its edge \( x = L \). In parallel to [10], then the total work done by the mechanical external forces is

\[ W_m = g^1 v^1(L) + g^3 v^3(L) +gw(L) - Mw_x(L). \]
Now we use the constitutive equations (3) - (6), and (10) - (12) to find

\[ P = P^1 + P^2 = \frac{1}{2} \int_{\Gamma} \sum_{i=1,2} \left( T_{11}^{(i)} S_{11}^{(i)} + T_{13}^{(i)} S_{13}^{(i)} \right) dX \]

\[ = \frac{1}{2} \int_0^L \left[ \alpha^1 h_1 \left( (v_x^1)^2 + \frac{h_1^2}{12} w_{xx}^1 \right) + \alpha^2 h_2 \left( (v_x^2)^2 + \frac{h_2^2}{12} (\phi_x^2)^2 \right) + G_2 h_2 (\phi^2)^2 \right] dx, \]

\[ \mathbf{E}^3 - \mathbf{P}^3 = \frac{1}{2} \int_{\Gamma_P} \left( D_1 E_1 + D_3 E_3 - T_{11}^{(3)} S_{11}^{(3)} - T_{13}^{(3)} S_{13}^{(3)} \right) dX \]

\[ = \frac{1}{2} \int_0^L \left[ -\alpha^3 h_3 \left( (v_x^3)^2 + \frac{h_3^2}{12} (w_{xx})^2 \right) - 2\gamma h_3 \left( (\phi^3 + \dot{A}_0^3 + \frac{h_3^2}{24} \dot{A}_3^3)^2 v_x^3 - \frac{h_3^2}{12} w_{xx} (\phi^2 + \dot{A}_3^3) \right) \right. \]

\[ + \varepsilon_1 h_3 \left( (\phi_{x}^0)^2 + \frac{h_3^2}{12} (\phi_{x})^2 + \frac{h_3^4}{320} (\phi^2_x)^2 + (\dot{A}_0^3)^2 + \frac{h_3^2}{12} (\dot{A}_3^3)^2 + \frac{h_3^4}{320} (\dot{A}_3^3)^2 \right) \]

\[ + 2\epsilon_1 h_3 \left( (\phi^0_x A_0^0 + \frac{h_3^2}{24} (\phi^0)_{x} (\phi^2_x)_{x} + \frac{h_3^2}{24} (\phi^0)^2 A_1^1 + \frac{h_3^2}{24} (\phi^0_{x})_{x} A_1^1 \right) \]

\[ + \frac{h_3^2}{24} (\phi^2_{x} A_0^0 + \frac{h_3^2}{12} (\phi^0)^2 A_1^1 + \frac{h_3^4}{320} (\phi^2_{x}) A_1^1 \right) \]

\[ + 2\varepsilon_3 h_3 \left( \phi^1 A_0^0 + \frac{h_3^2}{24} (\phi^0)^2 A_3^3 + \frac{h_3^2}{12} (\phi^0)_{x} A_3^3 + \frac{h_3^4}{320} (\phi^2_{x}) A_3^3 \right) \] \( dx, \) \( \tag{15} \)

and \( \mathbf{K} = \mathbf{K}^1 + \mathbf{K}^2 + \mathbf{K}^3 \) where

\[ \mathbf{K}^1 = \frac{\mu h_1}{2} \int_0^L \left( (\ddot{v}^1)^2 + \ddot{w}^1 + \frac{h_1^2}{12} \dddot{w}_x^1 \right) dx, \quad i = 1, 3, \] \( \tag{16} \)

\[ \mathbf{K}^2 = \frac{\rho h_2}{2} \int_0^L \left( (\ddot{\phi}^2)^2 + (\ddot{\phi}^2)^2 + \ddot{\phi}^2 \right) dx, \] \( \tag{17} \)

Finally, the magnetic energy of the piezoelectric beam is

\[ \mathbf{B}^3 = \frac{\mu h_3}{2} \int_{\Gamma_P} \left( \nabla \times \mathbf{A} \right) \cdot \left( \nabla \times \mathbf{A} \right) dX \]

\[ = \frac{\mu h_3}{2} \int_0^L \left[ (A_1^1)^2 + \frac{(h_3)^2}{12} (A_3^1)^2 + ((A_3^0)^2)_{x}^2 + \frac{(h_3)^2}{12} ((A_1^0)^2)_{x}^2 + \frac{(h_3)^2}{320} ((A_3^0)^2)_{x}^2 \right. \]

\[ - 2 \left( 1^1 (A_0^1)_{x} + \frac{(h_3)^2}{24} (A_0^1)^2 - \frac{(h_3)^2}{12} (A_1^1)^2 - \frac{(h_3)^2}{24} (A_5^0)_{x} (A_5^1)_{x} \right) \] \( \tag{18} \)

2.1. Variational Principle & Equations of Motion

By using (1)-(2), the variables \( \{v^2, \phi^2, \psi^2\} \) can be written as the functions of \( \{v^1, v^3, p\} \). As well, we know that the variables \( \{\phi^0, \phi^2, A_1^1, A_0^0, A_3^3\} \) have nothing
to do with the stretching equations as the single piezoelectric beam is actuated through the charge or the current source [17]. For that reason, we choose only \{v^1, w, \phi^1, A_1, A_0, A_2\} as the state variables.

To model charge or current-actuated ACL beams with magnetic effects, we use the following Lagrangian [11, 17]

\[
L = \int_0^T \left[ K - (P - E + B) + W \right] \, dt \tag{19}
\]

where \( P - E + B = P^1 + P^2 + P^3 - E^3 + B^3 \) is called electrical enthalpy where \( B^1 = B^2 = E^1 = E^2 = 0 \), and \( W = W^m + W^e \) is the total work done by the mechanical and electrical external forces, respectively. Note that in modeling piezoelectric beams by voltage-actuated electrodes we use a modified Lagrangian [16]. Let \( H = \frac{h_1 + 2h_2 + h_3}{2} \). We assume that the piezoelectric beam is clamped at \( x = 0 \) and free at \( x = L \).

The application of Hamilton’s principle, setting the variation of Lagrangian \( L \) in (19) with respect to the all kinematically admissible displacements of the chosen state variables to zero, yields a strongly coupled equations for the longitudinal and transverse dynamics together with the magnetic and electrical equations. It is not easy to study the controllability/stabilizability properties. For this reason we study the following reduced model.

### 2.2. Gauge condition and Rao-Nakra model assumptions

The magnetic potential vector \( A \) and the electric potential \( \phi \) are not uniquely defined in (9). In fact, the Lagrangian \( L \) in (19) is invariant under a large class of transformations.

**Theorem 2.1.** For any scalar \( C^1 \) function \( \chi(x,z,t) \), the Lagrangian \( L \) is invariant under the transformation

\[
A \mapsto \tilde{A} := A + \nabla \chi, \quad \phi \mapsto \tilde{\phi} := \phi - \dot{\chi}. \tag{20}
\]

**Proof:** See [17].

An additional condition can be added to remove the ambiguity as in [17]. The additional condition is generally known as a gauge and it can be chosen to decouple the electrical potential equation from the equations of the magnetic potential. Define

\[
\eta := A_0^3 + \frac{h_3^3}{24} A_3^2, \quad \theta := A_1, \quad \xi := \frac{\varepsilon_1 h_3^3}{12 \varepsilon_3}.
\]

In conjunction to [17], the appropriate gauge condition to decouple the magnetic and electric equations is chosen

\[
-\xi \theta_x + \eta = 0, \quad \theta(0) = \theta(L) = 0. \tag{22}
\]

In this section we derive the Rao-Nakra type ACL beam model using the three-layer version of the Rao-Nakra sandwich beam model developed in [11]
by letting $\rho_2, \alpha^2 \to 0$. This approximation retains the potential energy of shear and transverse kinetic energy. This together with the gauge condition \(^{22}\), we obtain the following model

$$\begin{cases}
\dot{m} \ddot{w} - K_1 \dddot{w}_x + K_2 \dddot{w}_{xxx} - H G_2 \phi_x^2 = 0 \\
\rho_1 \dot{h}_1 \dot{v}_1^1 - \alpha^1 \dot{h}_1 v_1^{11} - G_2 \phi^2 = 0, \\
\rho_3 \dot{h}_3 \dot{v}_3^3 - \alpha^3 \dot{h}_3 v_3^{33} + G_2 \phi^2 - \gamma \delta h_3 \left((\phi^1)_x + \eta_x\right) = 0, \\
-\xi (\phi^1)_{xx} + \phi^1 - \frac{2}{3} \psi_3^3 = \frac{\sigma}{3} (t) \quad \xi \xi \dot{h}_3 \dot{\theta} + \mu h_3 (\theta - \eta_x) + \xi \xi \dot{h}_3 (\phi^1)_x = \phi_1^3 (t) \\
\epsilon \epsilon \dot{h}_3 \dot{\eta} + \mu h_3 (\theta - \eta_x) + \epsilon \epsilon h_3 \phi^1 - \gamma h_3 v_3^3 = 0 \\
\phi^2 = \frac{1}{h_2} (-v^1 + v^3) + \frac{m}{h_2} w_x.
\end{cases}$$

(23)

with the natural boundary conditions at $x = 0, L$

$$\begin{align*}
\{v^1, v^3, w, w_x\}_{x=0} &= 0,  \{\alpha^1 h_1 v_1^1\}_{x=L} = g^1,  \\
\{\alpha^3 h_3 v_3^3 + \gamma h_3 (\phi^1 + \eta)\}_{x=L} = g^3,  \{\phi^1, \theta, \eta_x\}_{x=0, L} &= 0,  \\
\{K_2 w_{xxx}\}_{x=L} &= -M,  \{K_1 \ddot{w} - K_2 w_{xxx} + G_2 H \phi^2\}_{x=L} = g
\end{align*}$$

(24)

where $m = \rho_1 h_1 + \rho_3 h_3$, $K_1 = \frac{\alpha^1 h_1^3}{12} + \frac{\alpha^3 h_3^3}{12}$, and $K_2 = \frac{\alpha^1 h_1^3}{12} + \frac{\alpha^3 h_3^3}{12}$.

3. Well-posedness of the fully dynamic model

In this section, we consider the existence and uniqueness of solutions to \(^{26,24}\). First we eliminate the variable $\varphi^1$ by solving the elliptic equation \(^{27}\) with the associated boundary conditions. Defining $D_x^2 \phi = \phi_{xx}$ and its domain

$$\text{Dom}(D_x^2) = \{\phi \in H^2(0, L), \quad \phi_x(0) = \phi_x(L) = 0\},$$

and the operator $P_\xi$ by

$$P_\xi := (-\xi D_x^2 + I)^{-1}. \quad (25)$$

It is well-known that $P_\xi$ is a non-negative and a compact operator on $L_2(0, L)$ \(^{17}\). Therefore, the elliptic equation in \(^{27}\) has the solution

$$\phi^1 = \begin{cases}
\frac{2}{\xi} P_\xi v_3^3,  & \sigma_s(t) \equiv 0, i_3^3(x, t) \neq 0,  \\
\frac{2}{\xi} P_\xi v_3^3 + \frac{\sigma_s(t)}{\xi h_3} \chi_{[0, L]}(x) + K,  & \sigma_s(t) \neq 0, i_3^3(x, t) \equiv 0.
\end{cases}$$

(26)

where $\chi_{[0, L]}(x)$ is the characteristic equation of the interval $[0, L]$, and $K$ is an arbitrary constant ($\dot{K} = 0$ due to \(^{14}\) and Lemma \(^3.2\)). Using \(^{26}\), the system
(23)-(24) is simplified to
\[
\begin{cases}
\rho_1 h_1 \ddot{v}^1 - \alpha_1 h_1 v_{xx}^1 - G_2 \phi^2 = f^1, \\
\rho_3 h_3 \ddot{v}^3 - \alpha_3 h_3 v_{xx}^3 + G_2 \phi^2 - \gamma h_3 \left( \frac{2}{\varepsilon_3} (P_\xi v_x^3) + \eta \right) \\
= \frac{2\sigma_3}{\varepsilon_3} (\delta(x) - \delta(x - L)) + f^3, \\
m \ddot{w} - K_1 \dddot{w}_{xx} + K_2 w_{xxxx} - HG_2 \phi^2 = f \\
\xi h_3 \varepsilon_3 \dddot{\theta} + \mu h_3 (\theta - \eta_x) + \gamma \xi h_3 (P_\xi v_x^3) = i^1(t) \\
\varepsilon_3 h_3 \dddot{\eta} + \mu h_3 (\theta - \eta_x) + \gamma h_3 (P_\xi - I) \dot{v}_x^3 = 0 \\
\phi^2 = \frac{1}{h_2} (-v^1 + v^3) + \frac{h_2}{h_2} w_x.
\end{cases}
\]
with the initial and boundary conditions
\[
\begin{align*}
\{v^1, v^3, w, w_x\}_{x=0} &= 0, \quad \{\alpha_1 h_1 v_x^1\}_{x=L} = g^1, \\
\{\alpha_3 h_3 v_x^3 + \gamma h_3 (\frac{2}{\varepsilon_3} P_\xi v_x^3 + \eta)\}_{x=L} &= g^3, \quad \{\theta, \eta_x\}_{x=0,L} = 0 \\
\{K_2 w_{xx}\}_{x=L} &= -M, \quad \{K_1 \dddot{w} - K_2 w_{xxxx} + G_2 H \phi^2\}_{x=L} = g \\
(v^1, v^3, w, \theta, \eta, v^1, \dot{v}^3, \ddot{w}, \dot{\theta}, \dot{\eta})(x, 0) &= (v_0^1, v_0^3, w_0, \theta_0, \eta_0, v_1^1, v_1^3, w_1, \theta_1, \eta_1)(28)
\end{align*}
\]
Note that the external forces $g^3$ and $\sigma_3$ control the longitudinal vibrations in the third layer in a similar fashion (too many controllers). At this moment we assume that $g_3 \equiv 0$ since the third layer is a piezoelectric layer and it is more appropriate to control this layer by an electrical external force.

Define $\mathbb{X} = L^2(0, L)$,
\[
\begin{align*}
H^1_L(0, L) &= \{ f \in H^1(0, L) : f(0) = 0 \}, \\
H^2_L(0, L) &= \{ f \in H^2(0, L) \cap H^1_L(0, L) : f_x(L) = 0 \}, \\
V &= \{ y \in (H^1_L(0, L))^2 \times H^2_L(0, L) \times H^3_0(0, L) \times H^1(0, L) \\
&\quad -\xi(y_4) + y_5 = -\xi(y_9) + y_{10} = 0 \}
\end{align*}
\]
and the complex linear spaces
\[\mathcal{H} = V \times \mathbb{H}, \quad \mathbb{H} = (L^2(0, L))^2 \times H^1_L(0, L) \times (L^2(0, L))^2.\]

The natural energy associated with (27) is
\[
E(t) = \frac{1}{2} \int_0^L \left\{ \rho_1 h_1 |\dot{v}_x^1|^2 + \rho_3 h_3 |\dot{v}_x^3|^2 + m |\dot{w}|^2 + K_1 |\dddot{w}_x|^2 + \xi h_3 \varepsilon_3 |\dot{\theta}|^2 \\
+ h_3 \varepsilon_3 |\dddot{\eta}|^2 + \alpha_1 h_1 |v_x^1|^2 + \alpha_3 h_3 |v_x^3|^2 + \frac{\gamma^2}{\varepsilon_3} (P_\xi v_x^3) v_x^3 + K_2 |w_{xx}|^2 \\
+ G_2 |v_1 + v^3 + H w_x|^2 + \mu h_3 |\theta - \eta_x|^2 \right\} \, dx.
\]
This motivates the definition of the inner product on \( \mathcal{H} \):

\[
\left\langle \begin{bmatrix} u_1 \\ \vdots \\ u_{10} \end{bmatrix}, \begin{bmatrix} v_1 \\ \vdots \\ v_{10} \end{bmatrix} \right\rangle_{\mathcal{H}} = \left\langle \begin{bmatrix} u_6 \\ \vdots \\ u_{10} \end{bmatrix}, \begin{bmatrix} v_6 \\ \vdots \\ v_{10} \end{bmatrix} \right\rangle_{\mathcal{H}} + \left\langle \begin{bmatrix} u_1 \\ \vdots \\ u_{5} \end{bmatrix}, \begin{bmatrix} v_1 \\ \vdots \\ v_{5} \end{bmatrix} \right\rangle_{V}
\]

\[
= \int_0^L \left\{ \rho_1 h_1 u_6 \ddot{v}_6 + \rho_3 h_3 u_7 \ddot{v}_7 + m u_8 \ddot{v}_8 + K_1 (u_8)(\dddot{v}_8)_x + \xi h_3 \ddot{v}_9 \right. \\
\left. + \varepsilon_0 \dddot{v}_9 + \alpha_1 h_1 (u_1)(\dddot{v}_1)_x + \alpha_3 h_3 (u_2)(\dddot{v}_2)_x + \frac{\gamma^2}{\varepsilon_3} (P_\xi (u_2)_x)(\dddot{v}_2)_x \\
+ K_2 (u_3)(\dddot{v}_3)_x + G \left( -u_1 + u_2 + H (u_3)_x (-\dddot{v}_1 + \dddot{v}_2 + H (\dddot{v}_3)_x) \right) \\
+ \mu h_3 (u_4 - (u_5)_x)(\dddot{v}_4 - (\dddot{v}_5)_x) \right\} \, dx.
\]

Note that \( P_\xi \) is a nonnegative operator, and \( u_4 - (u_5)_x = u_4 + \xi (u_4)_{xx} \) and \( \dddot{v}_4 - (\dddot{v}_5)_x = \dddot{v}_4 + \xi (\dddot{v}_4)_{xx} \) by the definition of \( \mathcal{H} \). As well, the term \( \frac{\gamma^2}{\varepsilon_3} (P_\xi (u_2)_x)(\dddot{v}_2)_x \) is coercive and continuous in \( \mathcal{H} \), i.e., see \[10\]. Therefore (31) is a valid inner product on \( \mathcal{H} \) and so defines a norm. It is straightforward to verify that \( E \) as defined in (30) is the norm induced by (31). It can also easily be shown that \( \mathcal{H} \) with this norm is complete. This follows from the fact that the gauge constraints in \( \mathcal{H} \) are satisfied weakly \[17\].

Let \( \tilde{y} = (v^1, v^3, w, \theta, \eta) \) be the smooth solution of the system of \(27\).\(-\,28\). Multiplying the equations in \(27\) by \( \tilde{y}_1, \tilde{y}_2 \in H^1_0(0, L), \tilde{y}_3 \in H^1_0(0, L), \tilde{y}_4 \in H^1_0(0, L), \) and \( \tilde{y}_5 \in H^1(0, L) \), respectively, where \( \xi (\tilde{y}_4) = \tilde{y}_5 \), and integrating by parts yields

\[
\int_0^L \left\{ \rho_1 h_1 \ddot{v}_1 \dot{\tilde{y}}_1 + \alpha_1 h_1 v^1_x (\dddot{y}_1)_x - G_2 \phi^2 \dddot{y}_1 \right\} \, dx - g_1(t) \tilde{y}_1(L) = 0,
\]

\[
\int_0^L \left\{ \rho_3 h_3 \ddot{v}^3 \dddot{y}_2 + \alpha_3 h_3 v^3_x \dddot{y}_2_x + G_2 \phi^2 \dddot{y}_2 + \gamma h_3 \left( \frac{\gamma}{\varepsilon_3} (P_\xi v^3_x) + \eta \right)(\dddot{y}_2)_x \right\} \, dx
\]

\[
= -\frac{\gamma}{\varepsilon_3} \sigma_3(t) \dddot{y}_2(L),
\]

\[
\int_0^L \left\{ m \dddot{w} \dddot{y}_3 + K_1 \dddot{w}_x (\dddot{y}_3)_x + K_2 \dddot{w}_xx (\dddot{y}_3)_x + H G_2 \phi^2 (\dddot{y}_3)_x \right\} \, dx
\]

\[
- \dddot{g}(t) (\dddot{y}_3)_x(L) + K_2 M(t)(\dddot{y}_3)_x(L) = 0
\]

\[
\int_0^L \left\{ \xi \dddot{h}_3 \dddot{y}_4 + \mu h_3 (\theta - \eta_x)(\dddot{y}_4)_x - \gamma h_3 (P_\xi \dddot{y}_4)_x \right\} \, dx
\]

\[
= \int_0^L \delta(t) \dddot{y}_4 \, dx,
\]

\[
\int_0^L \left\{ \xi \dddot{h}_3 \dddot{y}_5 + \mu h_3 (\theta - \eta_x)(\dddot{y}_5)_x + \gamma h_3 (P_\xi - I) \dddot{y}_5 \right\} \, dx = 0
\]

Note that, the term \( \delta(x) \sigma_3(t) \) in \(27\) is lost in the weak formulation due to the boundary condition \( \tilde{y}_2(0) = 0 \). This explains that the charge actuation only forces boundary condition at \( x = L \) as in the case of voltage actuation \[19\].

Let \( \mathcal{M} : H^1_0(0, L) \to (H^1_0(0, L))^\prime \) and \( \mathcal{D} : H^1_0(0, L) \to (H^1_0(0, L))^\prime \) be a linear operators defined by

\[
\left\langle \mathcal{M} \psi, \tilde{\psi} \right\rangle_{(H^1_0(0, L))^\prime, H^1_0(0, L)} = \int_0^L \left( m \dddot{\psi} \dddot{\tilde{\psi}} + K_1 \dddot{\psi}_x \dddot{\tilde{\psi}}_x \right) \, dx.
\]

\[
\left\langle \mathcal{D} \psi, \tilde{\psi} \right\rangle_{V', V} = \gamma h_3 \int_0^L \left( (\dddot{\psi}_5)(\dddot{\tilde{\psi}})_x - (\dddot{\psi})_x(\dddot{\tilde{\psi}})_x \right) \, dx.
\]

For all \( \psi = (\psi_1, \psi_2, \psi_3, \psi_4, \psi_5)^T, \tilde{\psi} = (\tilde{\psi}_1, \tilde{\psi}_2, \tilde{\psi}_3, \tilde{\psi}_4, \tilde{\psi}_5)^T \in V \), define the linear operator \( A : V \to V' \) by \( \left\langle \mathcal{A} \psi, \tilde{\psi} \right\rangle_{V', V} = \left\langle \psi, \tilde{\psi} \right\rangle_V \). From the Lax-Milgram theo-
rem \( \mathcal{M} \) and \( A \) are canonical isomorphisms from \( H^1_L(0, L) \) onto \( (H^1_L(0, L))^\prime \) and from \( V \) onto \( V' \), respectively.

Next we introduce the linear unbounded operator by \( A : \text{Dom}(A) \times V \subset \mathcal{H} \to \mathcal{H} \) where \( A = \begin{bmatrix} O_{5 \times 5} & I_{5 \times 5} \\ M^{-1}A & M^{-1}D \end{bmatrix} \), and

\[
\text{Dom}(A) = \{(\tilde{z}, \tilde{\varphi}) \in V \times V, A\tilde{z} + D\tilde{\varphi} \in V'\}. \tag{34}
\]

Let \( \text{Dom}(A)' \) is the dual of \( \text{Dom}(A) \) pivoted with respect to \( \mathcal{H} \). Define the control operator \( B \), and \( B_0 \in \mathcal{L}(\mathbb{C}, V') \),

\[
B = \begin{bmatrix} 0_{5 \times 8} \\ B_0 \end{bmatrix}
\]

We have \( V' \) which is the dual of \( V \) pivoted with respect \( \mathcal{H} \). The dual operators of \( B_0 \) and \( B \) are \( B_0^* \in \mathcal{L}(V, \mathbb{C}) \) and

\[
B_0^* \psi = \begin{bmatrix} \psi_1(L), \int_0^L \psi_1(x)dx, -\frac{\gamma}{\varepsilon_3} \psi_2(L), \int_0^L \psi_2(x)dx, \psi_3(L), -K_2 \psi_3(L), \\ \int_0^L \psi_3(x)dx, \frac{1}{\varepsilon_3} \int_0^L \psi_4(x)dx \end{bmatrix}^T
\]

with \( B^* \Phi = (0_{8 \times 5} B_0^*)^T \Phi \). Let \( M = \text{diag}(\rho_1 h_1 I, \rho_3 h_3 I, \mathcal{M} \xi \varepsilon_3 h_3 I, \varepsilon_3 h_3 I) \). \( M \) is clearly an isomorphism from \( \mathcal{H} \) onto \( \mathcal{H}' \).

Writing \( \varphi = [v^1, v^2, w, \theta, \eta, \dot{v}^1, \dot{w}, \dot{\theta}, \dot{\eta}]^T \), the control system (27)-(28) with all controllers \( F = [g^1, g^2, \sigma, f^3, g, M, f, i_5]^T \) can be put into the state-space form

\[
\begin{aligned}
\dot{\varphi} + \begin{bmatrix} 0 & -I_{5 \times 5} \\ M^{-1}A & M^{-1}D \end{bmatrix} \varphi &= \begin{bmatrix} 0_{5 \times 8} \\ M^{-1}B_0 \end{bmatrix} F(t), \\
\varphi(x, 0) &= \varphi^0.
\end{aligned} \tag{36a}
\]

Note that the term \(-\gamma \xi h_3 (P \dot{v}_5^3)(\dot{y}_4) \) in (32) is the same as \(-\gamma h_3 (P \dot{v}_5^3)\dot{y}_5 \) in the last equation due to the Coulomb gauge (22).

**Lemma 3.1.** The operator \( A \) defined by (34) is maximal monotone in the energy space \( \mathcal{H} \), i.e.,

\[
\langle A\tilde{z}, \tilde{\varphi} \rangle \geq 0, \quad \text{for all } \tilde{z} \in \text{Dom}(A),
\]

and \( \text{Range}(I + A) = \mathcal{H} \).
Proposition 3.1. Let $\bar{z} \in \text{Dom}(A)$. A simple calculation using integration by parts and the boundary conditions yields

$$
\left\langle A \begin{bmatrix} \bar{z}_1 \\ \bar{z}_2 \end{bmatrix}, \begin{bmatrix} \bar{z}_1 \\ \bar{z}_2 \end{bmatrix} \right\rangle_{\mathcal{H} \times \mathcal{H}} = \left\langle \begin{bmatrix} -\bar{z}_2 \\ M^{-1}A\bar{z}_1 + M^{-1}D\bar{z}_2 \end{bmatrix}, \begin{bmatrix} \bar{z}_1 \\ \bar{z}_2 \end{bmatrix} \right\rangle_{\mathcal{H}} = (A\bar{z}_1 + D\bar{z}_2, \bar{z}_2)_{H',V \times V} + (A\bar{z}_1 + D\bar{z}_2, \bar{z}_2)_{H',H}.
$$

Since $\bar{z} = \begin{bmatrix} \bar{z}_1 \\ \bar{z}_2 \end{bmatrix} \in \text{Dom}(A)$, then $A\bar{z}_1 + D\bar{z}_2 \in V'$ and $\bar{z}_2 \in V$ so that

$$
(A\bar{z}_1 + D\bar{z}_2, \bar{z}_2)_{H',H} = (A\bar{z}_1 + D\bar{z}_2, \bar{z}_2)_{H',V \times V}.
$$

Since $\text{Re} \langle D\bar{z}_2, \bar{z}_2 \rangle_{V',V \times V} = 0$ by (33), $\text{Re} \langle A\bar{z}, \bar{z} \rangle_{H,H} = 0$.

We next verify the range condition. Let $\bar{z} = \begin{bmatrix} \bar{z}_1 \\ \bar{z}_2 \end{bmatrix} \in \mathcal{H}$. We prove that there exists a $\tilde{y} = \begin{bmatrix} \tilde{y}_1 \\ \tilde{y}_2 \end{bmatrix} \in \text{Dom}(A)$ such that $(I + A)\tilde{y} = \bar{z}$. A simple computation shows that proving this is equivalent to proving $\text{Range}(M + A + D) = H'$, i.e., for every $\tilde{f} \in H'$ there exists a unique solution $\bar{z} \in H$ such that $(M + A + D)\bar{z} = \tilde{f}$. This obviously follows from the Lax–Milgram’s theorem.

Proof: Let $\bar{u} = (\tilde{y}, \bar{z}) \in \mathcal{H}$. Then $\bar{u} \in \text{Dom}(A)$ if and only if the following conditions hold:

$$
\begin{align*}
\tilde{y} &\in (H^2(0, L) \cap H^1_0(0, L))^2 \times (H^3(0, L) \cap H^2_0(0, L)) \\
\times & (H^2(0, L) \cap H^1_0(0, L)) \times H^2(0, L), \text{ and } \bar{z} \in V, \\
\xi(y_4)_x = y_5, \xi(z_4)_x = z_5 \text{ such that } (y_5)_x(0) = (y_5)_x(L) = 0, \text{ and} \\
(y_1)_x = \alpha^3 h_3(y_2)_x + \gamma h_3 \left( \frac{\gamma}{\varepsilon_3} P(\bar{z})_{x} + z_5 \right) = (y_3)_x |_{x=L} = 0. \quad (37)
\end{align*}

Moreover, the resolvent of $A$ is compact in the energy space $\mathcal{H}$.

Proof: Let $\tilde{u} = \begin{bmatrix} \tilde{y} \\ \bar{z} \end{bmatrix} \in \mathcal{H}$ and $\bar{u} = \begin{bmatrix} \bar{y} \\ \bar{z} \end{bmatrix} \in \text{Dom}(A)$ such that $A\bar{u} = \bar{u}$.

Then we have

$$
\begin{align*}
-\bar{z} &\in V, \\
A\bar{y} + D\bar{z} &\in M\bar{z},
\end{align*}
$$

and therefore,

$$
\langle \bar{y}, \varphi \rangle_V + (Dz, \varphi)_{V',V} = \langle \bar{z}, \varphi \rangle_H \text{ for all } \varphi \in V. \quad (39)
$$

Let $\psi = [\psi_1, \ldots, \psi_5]^T \in (C_0^\infty(0, L))^4$. We define $\varphi_i = \psi_i$ for $i = 1, 2, 4, 5$, and $\varphi_3 = \int_0^s \psi_3(s)ds$. Since $\varphi \in V$, inserting $\varphi$ into (32) and using the gauge
condition yield
\[
\int_0^L \left\{ -\alpha^1 h_1 v_{x} \tilde{\psi}_{1} - \alpha^2 h_3 v_{x} \tilde{\psi}_{2} - \frac{(\gamma^3)^2}{\varepsilon^3} (P^\tau v^3)_{x} \tilde{\psi}_{2} - K_2 w_{x} \tilde{\psi}_{3} + G_2 h_2 \phi^2 \tilde{\psi}^2, \\
+ \mu h_3 (\theta - \xi \theta x) (\psi_{4}) + \mu h_3 \left( \frac{1}{2} \eta - \eta x \right) \psi_{5} \right\} \, dx = \int_0^L \left\{ \left( \int_1^L \left( \int_1^T (3z_1 \xi_{x} \tilde{\psi}_{3} - \gamma h_3 (z_3)_{x} \psi_{5} - \gamma h_3 (z_3)_{x} \tilde{\psi}_{2} \right) \, dx \right) \right\} \, dx = 0
\]
for all \( \tilde{\psi} \in (C_c^\infty(0, L))^4 \). Therefore it follows that \( \tilde{y} \in (H^2(0, L) \cap H^1_L(0, L))^2 \times (H^3(0, L) \cap H^2_L(0, L)) \times (H^2(0, L) \cap H^1_L(0, L))^2 \). Next let \( \tilde{\psi} \in \mathcal{H} \). We define
\[
\varphi_i = \int_0^x \psi_i(s) ds, \quad i = 1, \ldots, 4.
\]
Obviously \( \varphi \in \mathcal{V} \). Then plugging (11) into (9) yields
\[
\begin{align*}
\alpha^1 h_1 (y_1)_x (L) \psi_1 (L) &+ \left( \alpha^3 h_3 (y_2)_x + \frac{(\gamma^3)^2}{\varepsilon^3} (P^\tau v^3)_{x} (L) + \gamma^3 h_3 (z_3) (L) \right) \psi_2 (L) \\
+ K_2 (y_3)_{xx} (L) \psi_3 (L) + y_4 (L) \psi_4 (L) + (y_5)_x (L) \psi_5 (L) &= 0
\end{align*}
\]
for all \( \psi \in \mathcal{H} \). Hence,
\[
(y_4)_{xx} (L) = (y_1)_x (L) = (y_2)_x (L) = (y_3)_x (L) = 0.
\]

We have the following well-posedness theorem for (36).

**Theorem 3.1.** Let \( T > 0 \), and \( F \in (L^2(0, T))^8 \). For any \( \varphi^0 \in \mathcal{H} \), \( \varphi \in C([0, T]; \mathcal{H}) \), and there exists a positive constants \( c_1(T) \) such that (30) satisfies
\[
\| \varphi(T) \|_H^2 \leq c_1(T) \left\{ \| \varphi^0 \|_H^2 + \| F \|_{L^2(0, T))^8}^2 \right\}.
\]

**Proof:** The operator \( A : \text{Dom}(A) \rightarrow \mathcal{H} \) is a maximal monotone operator by Lemma 5.1 and Proposition 5.1. Therefore it is an infinitesimal generator of \( C_0 \)-semigroup of contractions by Hille-Yosida theorem. Since
\[
\langle M^{-1} BF, \tilde{\psi} \rangle_{\mathcal{V}, \mathcal{V}} = \langle BF, \tilde{\psi} \rangle_{\mathcal{V}', \mathcal{V}} < \infty
\]
by the Sobolev’s trace inequality, \( B \) is an admissible control operator for the semigroup \( \{ e^{At} \}_{t \geq 0} \) corresponding to (36), and hence the conclusion (42) follows. \( \square \)

**Remark 3.1.** Note that the underlying control operators in each actuation differ substantially. In the case of current actuation, the control operator is bounded in \( \mathcal{H} \) whereas in the case of charge actuation it is unbounded. Therefore the charge actuation case is more similar to the voltage actuation case in this manner.

**Quasi-static model:** The quasi-static assumes that \( \dot{A} = 0 \) but \( A \) may be nonzero, i.e. \( E = -\nabla \phi \), and \( \theta, \eta \) may be nonzero. The Coulomb gauge (22) is
still valid. Running the Hamilton's principle by taking this into account together with the Rao-Nakra assumptions, i.e., $\phi_2, \alpha^2 \to 0$ and using the definition of $\xi$, yields the following initial-boundary value problem:

$$
\begin{align*}
\rho_1 h_1 \ddot{v}_1 - \alpha h_1 \dot{v}_1 v_{xx} - G_2 \dot{\phi}^2 &= 0, \\
\rho_3 h_3 \ddot{v}_3 - \alpha h_3 v_{xx}^3 + G_2 \dot{\phi}^2 - \frac{\gamma}{\varepsilon_3} (P_\eta \dot{v}_3)_x &= -\frac{\gamma \sigma_\varepsilon(t)}{\varepsilon_3} \delta(x - L), \\
m \ddot{w} - K_1 \dot{w}_{xx} + K_2 w_{xxxx} - H G_2 \dot{\phi}^2 &= 0 \\
\mu \dot{h}_3 (\theta - \eta_x) + \gamma v_3 (P_\eta \dot{v}_3)_x &= \dot{i}_s(t) \\
\mu h_3 (\theta - \eta_x)_x + \gamma h_3 (P_\eta - I) \dot{v}_3 &= 0 \\
\phi^2 &= \frac{1}{\h_2} (-v^1 + v^3) + \frac{H}{\h_2} w_x.
\end{align*}
$$

(43)

with the initial and boundary conditions

$$
\begin{align*}
\{v_1, v_3, w_x\}_{x=0} &= 0, & \{\alpha h_1 v_1^1\}_{x=L} &= g^1, \\
\{\alpha h_3 v_3^3 + \frac{\gamma^2}{\varepsilon_3} P_\eta v_3^3\}_{x=L} &= 0, & \{\eta, \eta_x\}_{x=0, L} &= 0, \\
K_2 w_{xx} &= M, & \{K_1 \dot{w} - K_2 w_{xxx} + G_2 H \dot{\phi}^2\}_{x=L} &= g \\
(v^1, v^3, w, \theta, \eta, \dot{v}_3, \dot{w})_t(x, 0) &= (v_0^1, v_0^3, w_0, \theta_0, \eta_0, v_1^3, w_1).
\end{align*}
$$

(44)

**Lemma 3.2.** [17] Lemma 2.1 Let $\text{Dom}(D^2_\xi) = \{w \in H^2(0, L) : w_x(L) = 0\}$. Define the operator $J = \frac{1}{2} (P_\xi - I)$. Then $J$ is continuous, self-adjoint and non-positive on $L_2(0, L)$. Moreover, for all $w \in \text{Dom}(P_\xi)$,

$$
Jw = D^2_\xi P_\xi = D^2_\xi (I - \xi D^2_\xi)^{-1} w .
$$

Note that the last equation can easily be derived as the first derivative of the $\theta$-equation is taken by using Lemma 3.2, the surface continuity condition (14), and the gauge condition (22) in a weak sense. This way the last two equations can be also written as second order elliptic equations as

$$
\begin{align*}
\{\mu h_3 (\theta - \xi \eta_x) + \gamma \xi h_3 (P_\eta \dot{v}_3)_x &= \dot{i}_s(t) \\
\mu h_3 (\eta - \xi \eta_x) + \gamma \xi h_3 (P_\eta - I) \dot{v}_3 &= 0
\end{align*}
$$

(45)

Since the bending and stretching equations is not coupled to $\theta$ or $\eta$ equation directly, this case should be handled delicately in the case of current actuation. The well-posedness of this model can be shown analogously.

**Electrostatic model:** The electrostatic assumption fully ignores magnetic effects, i.e. $A = \bar{A} = 0$. Note that the surface current density $i_s(t)$ cannot survive in the fully dynamic approach in (13). For this reason, current surface density may have to come into the play through the circuit equations. However, the surface charge density $\sigma_s(t)$ survives in (13). If we run the Hamilton’s principle by taking this into account together with the reduced Rao-Nakra assumptions, i.e. $\phi_2, \alpha^2 \to 0$, and using the definition of $\xi$, we obtain the following initial-boundary value problem:

$$
\begin{align*}
\rho_1 h_1 \ddot{v}_1 - \alpha h_1 \dot{v}_1 v_{xx} - G_2 \dot{\phi}^2 &= 0, \\
\rho_3 h_3 \ddot{v}_3 - \alpha h_3 v_{xx}^3 + G_2 \dot{\phi}^2 - \frac{\gamma}{\varepsilon_3} (P_\eta \dot{v}_3)_x &= -\frac{\gamma \sigma_s(t)}{\varepsilon_3} \delta(x - L), \\
m \ddot{w} - K_1 \dot{w}_{xx} + K_2 w_{xxxx} - H G_2 \dot{\phi}^2 &= 0 \\
\phi^2 &= \frac{1}{\h_2} (-v^1 + v^3) + \frac{H}{\h_2} w_x.
\end{align*}
$$

(46)
with the natural boundary conditions at \( x = 0, L \) and initial conditions
\[
\begin{align*}
\{v^1, v^3, w, w_x\} & \big|_{x=0} = 0, \\
\{\alpha^1 h_1 v^1_x\} & \big|_{x=L} = g^1, \\
\{\alpha^3 h_3 v^3_x + \frac{\varepsilon_s}{\varepsilon_0} P_\xi v^3\} & \big|_{x=L} = 0, \\
\{K_2 w_{xx}\} & \big|_{x=L} = -M, \\
\{K_1 \ddot{w} - K_2 w_{xxx} + G_2 H\phi^2\} & \big|_{x=L} = g \\
(v^1, v^3, w, \dot{v}^1, \dot{v}^3, \dot{w})(x, 0) & = (v^1_0, v^3_0, w_0, v^1_1, v^3_1, w_1).
\end{align*}
\] (47)

Note that this case is not much different from the voltage-actuation case \[19, 20\]. The difference arises from the term involving the \( P_\xi \) term in (46). This is only because the voltage actuated model ignores the induced effect of the electric field, i.e. see (10). We account for this effect by choosing quadratic-through-thickness assumption. These effects, in fact, turns out to make the piezoelectric beam more stiff since \( P_\xi \) is a positive operator. This was first observed in \[9\]. The well-posedness of this model can be shown analogously.

4. Future Research

A relevant problem is the ACL-patch problem on a stiff elastic layer, see Figure . The model corresponding to this problem is obtained in a similar fashion by following the methodologies in this paper and \[23\]. However, the exact controllability and stabilizability problem is a bit more challenging relying on the location of the patch and the material parameters. This is currently under investigation.

The three-layer ACL beam model obtained in this paper can be analogously extended to the multilayer ACL beam model with the same assumptions as in \[10\]. The piezoelectric layers can be even actuated by charge, voltage, or current sources simultaneously. This is the subject of the future research.

References

[1] A. Baz, Boundary Control of Beams Using Active Constrained Layer Damping, J. Vib. Acoust. 119-2 (1997), 166–172.
[2] Y. Cao, X.B. Chen, A Survey of Modeling and Control Issues for Piezoelectric Actuators, *Journal of Dynamic Systems, Measurement, and Control*, 137-1 (2014), pp. 014001.

[3] C.Y.K. Chee, L. Tong, and G.P. Steven, A review on the modelling of piezoelectric sensors and actuators incorporated in intelligent structures, *J. Intell. Mater. Syst. Struct.* 9 (1998), 3-19.

[4] S Devasia, E. Eleftheriou, S. O. Reza Moheimani, A Survey of Control Issues in Nanopositioning, *IEEE Trabsactions on Control Systems Technology*, 15-5 (2007), pp. 802–823.

[5] R.A. DiTaranto, Theory of vibratory bending for elastic and viscoelastic layered finitethickness beams, *J. Appl. Mech.* 32 (1965), 881–886.

[6] G. Duvaut, J.L. Lions, *Inequalities in Mechanics and Physics*, (Springer-Verlag-1976).

[7] A.C. Eringen and G.A. Maugin, *Electrodynamics of Continua I, Foundations and Solid Media*, (Springer-Verlag, Berlin-1990).

[8] R.H. Fabiano, S.W. Hansen, Modeling and analysis of a three–layer damped sandwich beam, Dynamical systems and differential equations, *Discrete Contin. Dynam. Systems* Added Volume (2001), 143–155.

[9] S.W. Hansen, Analysis of a Plate with a Localized Piezoelectric Patch, *The Proceedings of the IEEE Conference on Decision & Control*, Tampa, Florida (1998), 2952-2957.

[10] S.W. Hansen, Several Related Models for Multilayer Sandwich Plates, *Mathematical Models & Methods in Applied Sciences* 14-8 (2004), 1103-1132.

[11] P.C.Y. Lee, A variational principle for the equations of piezoelectromagnetism in elastic dielectric crystals, *Journal of Applied Physics*, (69-11) (1991), pp. 7470–7473.

[12] J.A. Main and E. Garcia, Design impact of piezoelectric actuator nonlinearities, *Journal of Guidance, Control, and Dynamics*, 20-2 (1997), pp. 327Ü-332.

[13] J.A. Main, E. Garcia and D.V. Newton, Precision position control of piezoelectric actuators using charge feedback, *Journal of Guidance, Control, and Dynamics*, 18-5 (1995), pp. 1068Ü-73.

[14] D.J. Mead and S. Markus, The forced vibration of a three-layer, damped sandwich beam with arbitrary boundary conditions, *J. Sound Vibr.* 10 (1969), 163–175.

[15] S.O.R. Moheimani, A.J. Fleming, *Piezoelectric transducers for vibration control and damping*, (Springer-Verlag-2006).
[16] K.A. Morris, A.Ö. Özer, Modeling and stabilizability of voltage-actuated piezoelectric beams with magnetic effects, *SIAM J. Control Optim.* **52–4** (2014), 2371–2398.

[17] K.A. Morris, A.Ö. Özer, Comparison of stabilization of current-actuated and voltage-actuated piezoelectric beams, *the 53rd Proceedings of the IEEE Conf. on Decision & Control*, Los Angeles, California, USA (2014), 571–576.

[18] A.Ö. Özer, Further stabilization and exact observability results for voltage-actuated piezoelectric beams with magnetic effects, *Mathematics of Control, Signals, and Systems* **27-2** (2015), 219–244.

[19] A.Ö. Özer, Modeling and well-posedness results for active constrained layered (ACL) beams with/without magnetic effects, submitted.

[20] A.Ö. Özer, Modeling and control results for an active constrained layered (ACL) beam actuated by two voltage sources with/without magnetic effects, submitted, arXiv: 1511.05907v2.

[21] A.Ö. Özer, and S.W. Hansen, Exact boundary controllability results for a multilayer Rao-Nakra sandwich beam, *SIAM J. Control Optim.* **52-2**, 1314–1337.

[22] A.Ö. Özer, and S.W. Hansen, Uniform stabilization of a multi-layer Rao-Nakra sandwich beam, *Evolution Equations and Control Theory* **2-4** (2013), 195–210.

[23] A.Ö. Özer, and K.A. Morris, Modeling an elastic beam with piezoelectric patches by including magnetic effects, The Proceedings of the American Control Conference, Portland, USA (2014), 1045-1050.

[24] B. Rao, A compact perturbation method for the boundary stabilization of the Raleigh beam equation, *Appl. Math. Optim.* **3-33** (1996), 253–264.

[25] Y.V.K.S. Rao and B.C. Nakra, Vibrations of unsymmetrical sandwich beams and plates with viscoelastic cores, *J. Sound Vibr.* **34-3** (1974), 309–326.

[26] N. Rogacheva, *The Theory of Piezoelectric Shells and Plates*, (Boca Raton, FL: CRC Press, 1994).

[27] R.C. Smith, *Smart Material Systems*, (Society for Industrial and Applied Mathematics, 2005).

[28] R. Stanway, J.A. Rongong, N.D. Sims, Active constrained-layer damping: a state-of-the-art review, *Automation & Control Systems*, **217-6** (2003), 437–456.

[29] C.T. Sun and Y.P. Lu, *Vibration Damping of Structural Elements*, (Prentice Hall, 1995).
[30] H.F. Tiersten, *Linear Piezoelectric Plate Vibrations*, (New York: Plenum Press, 1969).

[31] M. Trindade and A. Benjendou, Hybrid Active-Passive Damping Treatments Using Viscoelastic and Piezoelectric Materials: Review and Assessment, *Journal of Vibration and Control* 8-6 (2002), pp. 699–745.

[32] M.-J. Yan and E.H. Dowell, Governing equations for vibrating constrained-layer damping sandwich plates and beams, *J. Appl. Mech.* 39 (1972), 1041–1046.