A study of elastico-viscous fluid flow by a revolving disk with heat dissipation effects using HAM based package BVPh 2.0

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Von Kármán problem of infinite disk is re-examined when fluid under consideration is elastico-viscous, satisfying the constitutive relations of Walters-B model. Main target here is to demonstrate how the presence of elasticity alters heat transfer phenomenon for the said problem especially when heat dissipation term is included in the analysis. We assume a self-similarity solution that results in a system of coupled non-linear equations. An easy to use package BVPh 2.0 based on the homotopy analysis method is used to present series solutions for values of elastico-viscous fluid parameter ($K$) in the range $0 \leq K \leq 1$. Residuals are evaluated numerically at various order of approximations which depict that obtained solutions converge to the exact solutions. Boundary layer is substantially suppressed due to the consideration of elastico-viscous fluid assumption. Furthermore, velocity of the entrained fluid is inversely proportional to the parameter $K$. The results predict a substantial drop in heat transfer rate whenever elasticity effects are present. A considerable role of heat dissipation towards thickening of thermal boundary layer is apparent from the findings.

Research interest in fluid motion triggered by the action of revolving disk has been ever growing since its discovery by Von Kármán. Such an interest is attributed to its occurrence in some technological processes including electrochemistry (which involves rotating-disk electrodes), cooling of computer storage devices and food processing industries. Dynamics of rotating flows, their fundamental equations and practical applications were summarized in a book by Childs. An excellent monograph was published by Shevchuk which includes formulation of convective heat transfer in various rotating flow configurations. Integral methods, self-similar techniques and analytical approaches to deal with rotating disk were also described briefly in this monograph. In the past, many novel results associated with the Von Kármán system have been introduced. We summarize here only a few (significant) problems for the sake of brevity. Stuart modeled the suction/blowing effects in rotating-disk system and found that radial and circumferential flows are decelerated whenever notion of suction is present. Heat transfer process in Von Kármán’s model was assessed in early studies (Refs. ). In , by assuming a linear relationship between viscosity and temperature, the authors obtained series solutions for both small and large Prandtl numbers. Benton introduced time-dependency in Von Kármán’s model by assuming that disk rotation sets up impulsively from rest. Also, more accurate solution for steady-state case was found. Later, Watson and Wang considered more realistic model in which disk angular velocity was assumed to decay as time from the initiation of motion progresses. They concluded that the disk can be made stress free by assigning an appropriate value to a parameter measuring the disk rotation rate. Miclavcic and Wang examined Von Kármán flow when the boundary (or rotating disk) admits partial slip characteristic. It was noticed that velocity slip contributes to a growth in minimum torque needed to keep disk in steady rotation. In articles and , Shevchuk considered a power-law surface temperature distribution (proportional to $r^n$, where $r$ is the radial and $n$ is power-law index). He noticed that heat transfer rate of the infinite disk increases with power law index $n$ in case of positive radial temperature gradient while opposite trend is found for negative radial temperature gradient. Similar model and conclusions were also depicted in another paper by Shevchuk and Buschmann. Xu and Liao put forward a novel analytic solution for the unsteady flow driven by a disk which undergoes impulsive rotation from rest. Later, Fang and Tao modeled fluid flow by a stretching and revolving disk with deceleration. Their analysis depicts that resisting wall shear vanishes by selecting a suitable ratio of disk stretching rate to the rotation rate. Von Kármán flow was also examined by Turkyilmazoglu when the disk was assumed to shrink in the radial
direction. It was found that structure of solutions in shrinking case widely differ from those obtained in usual stretching case. Thermal transport in nanofluid flow triggered by a revolving disk was elucidated by Khan et al.\(^1^6\) by assuming a realistic zero mass flux condition. Muthamilselvan and Renuka\(^1^7\) analyzed flow situation between rotating and stretchable disks in nanofluid. Application of a non-Fourier heat flux theory for micropolar fluid flow occurring between revolving surfaces was presented by Doh et al.\(^1^8\). A rigorous analysis for swirling flow between parallel disks, one undergoing uniform rotation and other stretching/shrinking, was made by Abbas et al.\(^1^9\). A novel model of homogeneous-heterogeneous reaction for nanofluid flow triggered between revolving disks was analyzed by Renuka et al.\(^2^0\). Using homotopy analysis method, entropy growth in nanofluid flow contained between spinning stretchable surface was elucidated by Renuka et al.\(^2^1\).

The frequent and broad occurrence of non-Newtonian behavior in diverse applications (both in nature and technology) is well established. Hoyt\(^2^2\) briefly summarized how non-Newtonian fluid flow is beneficial in some industrial processes including fluid friction reduction, surfactant applications for cooling/heating of large buildings and use of polymer additives to improve flow in petroleum pipe lines. Non-Newtonian behavior is also met in mining industry which treats slurries and muds, and in applications such as lubrication and biomedical flows. In most of the industrial processes, the Newtonian fluid assumption stands invalid and a complex non-Newtonian response needs to be modeled. In the past, various contributions featuring non-Newtonian fluid flow occurring above a disk that undergoes uniform rotation have been reported. The first ever attempt was made by Elliot\(^2^3\) who revisited Von-Karman’s analysis by taking into account constitutive relations due to Walters-B model. Decades ago, Ariel\(^2^4\) used a robust approach to describe the viscoelastic fluid flow due to revolving disk placed in a second-grade fluid. He was able to construct approximate series solutions for small and large values of second grade fluid parameter. Andersson and De Korte\(^2^5\) numerically addressed axial magnetic field effects on rotating disk induced flow of power-law fluid. Their solution was based on generalized von Kármán transformations, which was valid even for highly shear-thickening fluids. Ariel\(^2^6\) later revisited Elliot’s work with a view to obtain accurate numerical results for large elasticity fluid parameter. Von Kármán flow analysis for viscoplastic fluid was made by Osalusi et al.\(^2^7\) using the well-accepted Bingham model. The articles published by Attia\(^2^8\) and Sahoo\(^2^9\) examined different physical characteristics associated with Reiner-Rivlin fluid flow caused by a rotating disk. Other contributions put forward in this domain include the works of Ahmadpour and Sadeghy\(^3^0\), Griffiths\(^3^1\), Guha and Sengupta\(^3^2\), Doh and Muthamilselvan\(^3^3\), Tabassum and Mustafa\(^3^4\), Imtiaz et al.\(^3^5\), Sahoo and Shevchuk\(^3^6\) and Mustafa et al.\(^3^7\).

Our foremost interest is to formulate heat transfer in Von Kármán of Walters-B fluid under viscous dissipation effect. It will be shown later, that viscous dissipation effect yields several additional terms in the energy equation. Notably, viscous dissipation terms should be retained in situations where either fluid has high viscosity coefficient or its average velocity is high. Our second goal is to furnish series solutions for velocity and temperature by optimal homotopy method using package BVPh 2.0. Averaged squared residuals for the system are worked out that support the series solutions obtained. The results indicate that contribution of elasticity combined with viscous dissipation term is significant in the analysis of resisting torque, wall shear, entrained flow and heat transfer rate.

**Problem formulation**

Suppose that an electrically conducting elastico-viscous fluid flows due to steady rotation of an infinite plane surface. Fluid is exposed to axial magnetic field with uniform magnetic flux density \(B_0\). In a cylindrical coordinate system \((r, \varphi, z)\), the disk taken along \(z = 0\) is made to rotate steadily about the axis \(r = 0\) (see Fig. 1). Fluid...
motion takes place in the semi-infinite region $z \geq 0$ and $z = 0$ is the only boundary. Let $u$, $v$ and $w$ symbolize velocity vector projections along $r$-, $\phi$- and $z$-directions respectively. Assuming that electric field is absent and magnetic Reynolds number is small enough so that induced magnetic field is negligible, components of Lorentz force vector are $F_r = -\sigma \mathbf{B}_0^2 \mathbf{u}$, $F_\phi = -\sigma \mathbf{B}_0^2 \mathbf{v}$ and $F_z = 0$, where $\sigma$ is the fluid electrical conductivity. Since the problem is symmetric about the vertical axis, one can neglect variation in velocities in $\phi$-direction, that is, $\partial / \partial \phi \equiv 0$. Relevant equations embodying fluid flow about a rotating disk are\(^{34}\):

$$\frac{\partial u}{\partial r} + \frac{u}{r} + \frac{\partial w}{\partial z} = 0,$$

(1)

$$\rho \left( \frac{\partial u}{\partial r} + w \frac{\partial u}{\partial z} - \frac{v^2}{r} \right) = \frac{\partial S_{rr}}{\partial r} + \frac{\partial S_{rz}}{\partial z} + \frac{S_{rr} - S_{\phi \phi}}{r} - \sigma \mathbf{B}_0^2 u,$$

(2)

$$\rho \left( \frac{\partial v}{\partial r} + w \frac{\partial v}{\partial z} + uv \right) = \frac{\partial S_{\phi \phi}}{\partial r} + \frac{\partial S_{rz}}{\partial z} + \frac{2S_{\phi \phi}}{r} - \sigma \mathbf{B}_0^2 v,$$

(3)

$$\rho \left( \frac{\partial w}{\partial r} + w \frac{\partial w}{\partial z} \right) = -\frac{\partial P}{\partial r} + \frac{\partial S_{zz}}{\partial z} + \frac{S_{zz}}{r},$$

(4)

where $S_{ij} (i,j = 1 - 3)$ are components of stress tensor $\mathbf{S}$. Beard and Walters\(^{38}\) proposed the following stress tensor $\mathbf{S}$ for elastico-viscous liquids:

$$\mathbf{S} = -P\mathbf{I} + \eta_0 \mathbf{A}_1 - \kappa_0 \frac{DA_1}{Dt},$$

(5)

where $\mathbf{A}_1 = \nabla \mathbf{v} + (\nabla \mathbf{v})^t$ is known as first Rivlin-Ericksen tensor, $\eta_0$ stands for apparent viscosity, $\kappa_0$ is termed material fluid parameter, $P$ stands for pressure, $\mathbf{I}$ is the identity tensor, and $DA_1/Dt$ is the upper-convected time derivative defined below:

$$\frac{DA_1}{Dt} = \frac{dA_1}{dt} - (\nabla \mathbf{v}) \mathbf{A}_1 - \mathbf{A}_1 (\nabla \mathbf{v})^t.$$

(6)

Above equations are to be solved for the following constraints:

$$at z = 0 : u = 0, v = r\omega, w = 0,$$

(7a)

$$as z \rightarrow \infty : u \rightarrow 0, v \rightarrow 0.$$

(7b)

The components of stress tensor $\mathbf{S}$ are obtained from Eq. (5) as follows:

$$S_{rr} = -P + 2\eta_0 \frac{\partial u}{\partial r} - 2\kappa_0 \left( \frac{u \partial^2 u}{\partial r^2} + w \frac{\partial^2 u}{\partial r \partial z} + 2 \left( \frac{\partial u}{\partial r} \right)^2 - \frac{\partial u}{\partial r} \left( \frac{\partial u}{\partial r} + \frac{\partial w}{\partial r} \right) \right),$$

(8)

$$S_{\phi \phi} = -P + 2\eta_0 \frac{u}{r} - 2\kappa_0 \left( \frac{u \partial u}{r \partial r} - \frac{u^2}{r^2} + w \frac{\partial u}{\partial z} - \left( \frac{\partial u}{\partial r} \right)^2 - \left( \frac{\partial u}{\partial r} - \frac{v}{r} \right)^2 \right),$$

(9)

$$S_{zz} = -P + 2\eta_0 \frac{\partial w}{\partial z} - 2\kappa_0 \left( \frac{u \partial^2 w}{\partial r \partial z} + w \frac{\partial^2 w}{\partial z^2} - \frac{\partial w}{\partial r} \left( \frac{\partial u}{\partial r} + \frac{\partial w}{\partial r} \right) - 2 \left( \frac{\partial w}{\partial z} \right)^2 \right),$$

(10)

$$S_{rz} = \eta_0 \left( \frac{\partial v}{\partial r} - \frac{v}{r} \right) - \kappa_0 \left( \frac{\partial^2 v}{\partial r \partial z} - \frac{3 \partial v}{\partial r} + 2 \frac{u}{r} + w \frac{\partial v}{\partial z} \right),$$

(11)

$$S_{\phi z} = \eta_0 \frac{\partial v}{\partial z} - \kappa_0 \left[ \frac{u \partial^2 v}{\partial r \partial z} + w \frac{\partial^2 v}{\partial z^2} - \frac{\partial v}{\partial r} - \frac{v}{r} \left( \frac{\partial u}{\partial r} + 2 \frac{\partial w}{\partial r} \right) - \frac{u \partial v}{r \partial r} - 3 \frac{\partial v}{\partial z} \frac{\partial w}{\partial z} \right],$$

(12)

$$S_{\phi z} = \eta_0 \left( \frac{\partial u}{\partial z} + \frac{\partial w}{\partial r} \right) - \kappa_0 \left[ \left( \frac{u \partial^{2}\phi}{\partial r^{2}} + w \frac{\partial^{2}\phi}{\partial z^{2}} \right) \frac{\partial u}{\partial r} + \frac{\partial w}{\partial r} \right] - \frac{\partial u}{\partial r} \frac{\partial u}{\partial z} + \frac{\partial w}{\partial r} \frac{\partial w}{\partial z} - 3 \left( \frac{\partial u}{\partial z} \frac{\partial w}{\partial z} + \frac{\partial u}{\partial r} \frac{\partial w}{\partial r} \right).$$

(13)

Accounting Eqs. (8)–(13) and the boundary layer approximations, Eqs. (2) and (3) become:
\[ u \frac{\partial u}{\partial r} + w \frac{\partial u}{\partial z} - \frac{v^2}{r} = \frac{\eta_0}{\rho} \frac{\partial^2 u}{\partial z^2} - \frac{\kappa_0}{\rho} \left\{ \frac{u}{\rho} \frac{\partial^3 u}{\partial z^3} + w^2 \frac{\partial^2 u}{\partial z^2} - 2 \frac{du}{\partial z} \frac{\partial u}{\partial z} - \frac{2}{7} \left( \frac{\partial u}{\partial z} \right)^2 \right\}, \]

\[ \frac{\partial v}{\partial r} + w \frac{\partial v}{\partial z} + \frac{uv}{r} = \frac{\eta_0}{\rho} \frac{\partial^2 v}{\partial z^2} - \frac{\kappa_0}{\rho} \left\{ \frac{-u^2 + w^2}{\rho} + \frac{w}{\rho} \frac{\partial^2 u}{\partial z^2} - \frac{2}{7} \left( \frac{\partial u}{\partial z} \right)^2 - \frac{2}{7} \left( \frac{\partial u}{\partial z} \right)^2 - \frac{2}{7} \frac{\partial u}{\partial z} \frac{\partial u}{\partial z} \right\}, \]

while Eq. (4) vanishes identically. For the solution of Eqs. (1), (14) and (15), we use the transformations:

\[ u = \rho_0 F(\eta), \quad v = \rho_0 G(\eta), \quad w = -2 \sqrt{\omega_0} F(\eta), \]

with \( \eta = \frac{z(\omega/\eta)^{1/2}}{2} \) as similarity variable.

With the aid of Eq. (16), the mass balance Eq. (1) is fulfilled whereas Eqs. (14) and (15) give rise to the following ODEs:

\[ F'''' + G'' + 2FG'' - F^2 - K \left( 4F'F'''' + 2G^2 - 2FG'''' \right) - MF'' = 0, \]

\[ G'' = 2F'G + 2FG' - K \left( 4F'G'' + 2FG'' - 2FG''' - 2F''G' \right) - MG = 0, \]

and boundary conditions (7a) and (7b) are transformed as follows:

\[ F(0) = 0, F'(0) = 0, G(0) = 1, \]

\[ F' \to 0, G \to 0 as \eta \to \infty, \]

where \( K = \kappa_0 \omega_0 / \rho_0 \) is the elasticity parameter and \( M = \sigma B_0^2 / \rho_0 \omega_0 \) is termed magnetic interaction parameter.

**Heat transfer analysis**

The difference between surface temperature and that of the ambient fluid serves as driving potential for heat flow from the disk to the fluid. In absence of heat generation/absorption, energy equation can be expressed in the following form:

\[ \rho C_p \left( \frac{\partial T}{\partial r} + w \frac{\partial T}{\partial z} \right) = \kappa \left\{ \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial T}{\partial r} \right) + \frac{\partial^2 T}{\partial z^2} \right\} + \Phi, \]

where \( \kappa \) stands for fluid thermal conductivity, \( C_p \) symbolizes specific heat capacity and \( \Phi \) shows viscous dissipation term given by

\[ \Phi = S_{tr} \frac{\partial u}{\partial r} + S_{rw} \frac{u}{r} + S_{wz} \frac{\partial w}{\partial z} + S_{wr} \left( \frac{\partial v}{\partial r} - \frac{v}{r} \right) + S_{zz} \left( \frac{\partial v}{\partial z} + \frac{\partial v}{\partial z} \right). \]

Using (21) in (20) and then simplifying the resulting expression using boundary layer assumptions, one obtains:

\[ \frac{u}{\rho} \frac{\partial T}{\partial r} + w \frac{\partial T}{\partial z} = \frac{\kappa}{\rho C_p} \frac{\partial^2 T}{\partial z^2} + \frac{\eta_0}{\rho C_p} \left\{ \left( \frac{\partial u}{\partial z} \right)^2 + \left( \frac{\partial v}{\partial z} \right)^2 - \frac{\kappa_0}{\rho C_p} \left\{ \frac{u^2 + w^2}{\rho} + \frac{w}{\rho} \frac{\partial^2 u}{\partial z^2} + \frac{w}{\rho} \frac{\partial^2 u}{\partial z^2} + \frac{2}{7} \left( \frac{\partial u}{\partial z} \right)^2 \right\} \right\}. \]

We substitute \( T = T_{\infty} + (T_w - T_{\infty}) \theta(\eta) \), where \( \theta(\eta) \) is non-dimensional temperature and wall temperature \( T_w \) has the form \( T_w = T_{\infty} + \beta r^2 \), in which \( \beta > 0 \) is a constant. Equation (22) yields the following ODE:

\[ \frac{1}{Pr} \theta'''' - 2F'\theta'' + 2F'' - Ec \left( F'' + G^2 \right) - KEc \left\{ 4F \left( F'' + G^2 \right) - 2F \left( F'' + G^2 \right) \right\} = 0, \]

and boundary conditions for \( \theta \) are given below:

\[ \theta(0) = 1 and \ \theta \to 0 as \eta \to \infty. \]

In Eq. (23), \( Pr = \eta_0 C_p / \kappa \) gives the Prandtl number an \( Ec = \omega_0^2 / BC_p \) defines the Eckert number.

**Skin friction coefficients, local Nusselt number and volumetric flow rate**

In examining Von Kármán boundary layer, an important characteristic is the shear stress experienced at the disk. We define the radial and tangential skin friction coefficients as follows:

\[ C_{fr} = \frac{S_{rz} \big|_{z=0}}{\rho (\omega_0)^2}, \quad C_{t\theta} = \frac{S_{r\theta} \big|_{z=0}}{\rho (\omega_0)^2}. \]
Upon utilizing Eq. (16) and boundary conditions (19a) in Eq. (25), one arrives at:

\[ Re^{1/2} C_F = F''(0), \quad Re^{1/2} C_\theta = G'(0). \]  

(26)

Another important concept is the Nusselt number defined as \( Nu = -rk(\partial T/\partial z)_{z=0}/\Delta T \). It can be expressed as:

\[ Re^{-1/2} Nu = -\theta'(0). \]  

(27)

Entrainment velocity \( w(\infty) \) can be used to determine the amount of fluid sucked towards the disk of radius \( R \) as follows:

\[ Q = \int_0^R -w(\infty)2\pi r dr = 2\sqrt{\nu\omega}F(\infty)\pi R^2. \]  

(28)

**Series solutions using optimal homotopy analysis method (OHAM)**

An improved version of the well-known homotopy analysis method (HAM) was developed by Liao with an aim to tackle strongly non-linear problems. The concept was based on computing the best possible value of the so-called auxiliary parameter that eventually accelerates the convergence of HAM solutions. Afterwards, Liao et al. came up with a user-friendly MATHEMATICA package BVPh 2.0 based on the HAM, which is freely accessible online at http://numericaltank.sjtu.edu.cn/BVPh.htm. Using basic idea of HAM, the unknown functions \( f, \xi \) and \( \theta \) are expressed as under:

\[ F(\eta) = \sum_{k=0}^{+\infty} F_k(\eta), \quad G(\eta) = \sum_{k=0}^{+\infty} G_k(\eta), \quad \theta(\eta) = \sum_{k=0}^{+\infty} \theta_k(\eta), \]  

(29)

in which \( F_k, G_k \) and \( \theta_k \) can be obtained from by formulating \( k \)-th-order deformation equations corresponding to Eqs. (17)–(19b), (23) and (24). The initial guesses of the system conforming with the so called rule of solution expression and the boundary conditions (19a), (19b) and (24) are chosen as:

\[ F_0(\eta) = 0, \quad G_0(\eta) = e^{-\eta}, \quad \theta_0(\eta) = e^{-\eta}, \]  

(30)

and auxiliary linear operators \( \mathcal{L}_F, \mathcal{L}_G, \mathcal{L}_\theta \) for the system of Eqs. (17), (18) and (23) are selected as follows:

\[ \mathcal{L}_F \equiv \frac{\partial^3}{\partial \eta^3} - \frac{\partial}{\partial \eta} \mathcal{L}_G \equiv \frac{\partial^2}{\partial \eta^2} - 1, \mathcal{L}_\theta \equiv \frac{\partial^2}{\partial \eta^2} - 1. \]  

(31)

Obviously, the above operators satisfy the following conditions:

\[ \mathcal{L}_F [c_1 + c_2 e^{-\eta} + c_3 e^\eta] = 0, \]  

(32)

\[ \mathcal{L}_G [c_4 e^{-\eta} + c_5 e^\eta] = 0, \]  

(33)

\[ \mathcal{L}_F [c_6 e^{-\eta} + c_7 e^\eta] = 0, \]  

(34)

where \( c_1 - c_7 \) are unknown constants to be determined.

It is customary to mention that the resulting solutions by the HAM contain auxiliary parameters \( c_{\nu F}, c_{\nu G} \) and \( c_0^\theta \) which play essential part in ensuring and accelerating convergence of solution. Here we are primarily interested to estimate optimal values of such parameters which correspond to minimum squared residual of the system. We begin by defining the total squared residual \( E_{T,k} \) in the interval \([a, b]\) as follows:

\[ E_{T,k} = E_{F,k} + E_{G,k} + E_{\theta,k}. \]  

(35)

where \( E_{F,k}, E_{G,k} \) and \( E_{\theta,k} \) are average squared residuals of Eqs. (17), (18) and (23) defined as (see Liao for details):

\[ E_{F,k} = \frac{1}{b-a} \int_a^b \left( N_F[F_k(\eta), G_k(\eta)] \right)^2 d\eta, \]  

(36)

\[ E_{G,k} = \frac{1}{b-a} \int_a^b \left( N_G[F_k(\eta), G_k(\eta)] \right)^2 d\eta, \]  

(37)

\[ E_{\theta,k} = \frac{1}{b-a} \int_a^b \left( N_\theta[F_k(\eta), G_k(\eta), \theta_k(\eta)] \right)^2 d\eta, \]  

(38)

where \( N_F, N_G \) and \( N_\theta \) are the associated non-linear differential operators. The optimal values of \( c_{\nu F}, c_{\nu G} \) and \( c_0^\theta \) are determined by command “GetOptiVar” of BVPh 2.0 (see for details).
**Results and discussion**

In order to ascertain that BVPh 2.0 code is working fine, we computed the total average squared residual (defined in Eqs. (35)) at different values of $K$, the elasticity parameter (see Fig. 2a–d). It is apparent that $E_{T,k}$ decreases monotonically as we increase $k$, the order of approximations. This confirms that series solutions given by Eq. (29) converge to the exact solutions as $k \to \infty$. For a further check, numerical results of $F^\prime(0)$, $G(0)$ and $\theta^\prime(0)$ are compared with the numerical data of previous studies in limiting sense and found in complete agreement (see Table 1). Having validated the employed method, we now turn to foresee the role of different controlling parameters on the considered model.

The disk surface temperature is assumed to vary quadratically with radial distance $r$. Such an assumption is necessary for the governing problem to exhibit self-similar solutions. Figure 3a–d include velocity curves and temperature profile for varying choices of elasticity parameter $K$. Note that radial velocity ($u = r\omega F^\prime$) and entrained flow are linked in such a way that the radially outflow, produced by centrifugal force, is responsible for drawing the fluid downwards towards the disk. Boundary layer is substantially thinned for increasing $K$-values. Reduction in radially driven flow by increasing $K$ is noted in Fig. 3a. This in turn leads to decelerate the axial fluid motion and thus the volumetric flow rate. No overshoot in similarity profiles is detected for the considered range of $K$. Circumferential flow is also predicted to slow down whenever elastic effects are considered (see Fig. 3c). On the contrary, fluid temperature rises for increasing $K$-values.

To see how present flow model is influenced by the presence of magnetic field, we prepared Fig. 4a–d showing velocity and temperature curves for a variety of $M$-values. It is noted that asymptotic value of $F$, that is $F(\infty)$, decreases for increasing values of $M$. Also, it takes shorter distances from the disk for the velocity profiles to attain their respective asymptotic values as $M$ is increased. Moreover, $u$-velocity profile ($u = r\omega F^\prime$) becomes flatter for higher values of $M$. Furthermore, the resistance offered to fluid motion by the Lorentz force leads to enhancement in temperature profile as apparent from Fig. 4d.

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**Figure 2.** Total residual error ($E_{T,k}$) versus order of approximations ($k$) at (a) $K = 0$, (b) $K = 0.25$, (c) $K = 0.5$ and (d) $K = 0.8$. 

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**Figure 3.** Velocity curves and temperature profile for varying choices of elasticity parameter $K$.

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**Figure 4.** Velocity and temperature curves for a variety of $M$-values.
Figure 5a shows the change in temperature distribution by varying Pr, the Prandtl number. It takes smaller distances from the disk for temperature curve to reach η-axis for increasing Pr-values. Moreover, the effect of Eckert number Ec is seen to be typical of fluid gaining temperature (due to the loss of heat energy from the disk) (see Fig. 5b).

### Table 1.
A comparison of 45th order HAM results with those obtained by Ariel24,41 for different values of M when K = 0. *Shows results given in Ariel41. bOur results.

| M   | F''(0)a | -G'(0)a | F''(0)b | -G'(0)b |
|-----|---------|---------|---------|---------|
| 0.2 | 0.453141| 0.708795| 0.453129| 0.708793|
| 0.4 | 0.405576| 0.802376| 0.405576| 0.802376|
| 0.6 | 0.366986| 0.894476| 0.366986| 0.894476|
| 0.8 | 0.335092| 0.983607| 0.335090| 0.983607|
| 1.0 | 0.309258| 1.069053| 0.309258| 1.069053|
| 1.2 | 0.287916| 1.150635| 0.287915| 1.150635|
| 1.4 | 0.270049| 1.228466| 0.270049| 1.228466|
| 1.6 | 0.254892| 1.302793| 0.254892| 1.302793|
| 2.0 | 0.230559| 1.442094| 0.230559| 1.442094|

Figure 3. Curves of velocity components (F, F', G) and temperature (θ) for different values of elasticity parameter K.

Figure 5a shows the change in temperature distribution by varying Pr, the Prandtl number. It takes smaller distances from the disk for temperature curve to reach η-axis for increasing Pr-values. Moreover, the effect of Eckert number Ec is seen to be typical of fluid gaining temperature (due to the loss of heat energy from the disk) (see Fig. 5b).
**Figure 4.** Curves of velocity components $(F, F', G)$ and temperature $(\theta)$ for different values of magnetic interaction parameter $M$.

**Figure 5.** Temperature curves for different values of (a) Prandtl number $Pr$ and (b) Eckert number $Ec$. 
In Fig. 6a–d, we present the graphs of $\sqrt{\text{Re}} C_{f\alpha}$, $\sqrt{\text{Re}} C_{f\phi}$, $\sqrt{\text{Re}} -\nu \partial T / \partial z$ (in Eq. (9)), measuring heat convection, reduces when $K$ is enhanced. As a result, Nusselt number is seen to lower substantially when $K$ enlarges. Similar conclusion can be made for the influence of magnetic force on Nusselt number.

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Concluding remarks

In this framework, we discussed elastico-viscous fluid flow bounded by a rotating disk with heat dissipation effects. The analysis is based on a quadratic surface temperature distribution which is a prerequisite for achieving self-similar solution. The developed system of equations is treated via package BVPh 2.0 of MATHEMATICA based on the HAM. The specific conclusions of the present study are outlined as follows:

- Using the package BVPh 2.0, the averaged squared residual of the governing system is computed which reflects that series solutions converge to the exact solutions as $k$ (order of approximation) tends to infinity.
- Akin to earlier works (see, for instance\(^{23,26}\)), an increase in elasticity parameter $K$ has a retarding effect on the boundary layer flow. The entrained volume of the fluid upon the disk also decreases with an enhancement in elasticity.
- The effect of elasticity is such that radial wall stress decreases as values of $K$ are incremented. However, the resisting torque first decreases to a minimum and then increases for increasing $K$-values. Notably, for sufficiently higher values of $M$, resisting torque is monotonically increasing function of $K$.
- An expansion in thermal boundary layer is found for increasing values of $K$. Such increase accompanies with reduced heat transfer rate from the rotating surface.
- The existence of axial magnetic field opposes radially outward flow initiated by the centrifugal force. Such opposition restricts the amount of fluid drawn vertically thereby providing an expansion in temperature profile.
- As we increase the Eckert number, a relative decrease in enthalpy is noticed which in turn leads to an enhancement in the temperature profile.

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| $K$ | $M$ | $F(\infty)$ | $Re^\frac{1}{2}C_p$ | $Re^\frac{1}{2}C_p$ |
|-----|-----|-------------|-----------------|-----------------|
| 0.2 | 0.5 | 0.175165    | 0.279125        | −0.857647       |
| 0.3 | 0.5 | 0.147619    | 0.226650        | −0.850164       |
| 0.4 | 0.5 | 0.122389    | 0.178885        | −0.835553       |
| 0.5 | 0.5 | 0.097984    | 0.136315        | −0.816793       |
| 0.2 | 0.2 | 0.295671    | 0.368621        | −0.728808       |
| 0.5 | 0.2 | 0.175165    | 0.279125        | −0.857647       |
| 0.8 | 0.2 | 0.104394    | 0.209732        | −0.984472       |
| 1.2 | 0.2 | 0.052581    | 0.139514        | −1.144154       |

Table 2. Computational results of skin friction coefficients at different values of $K$ and $M$ at 20th-order of approximations.

| $K$ | $M$ | $Pr$ | $Ec$ | $−\theta'(0)$ |
|-----|-----|-----|-----|---------------|
| 0   | 0.5 | 1.25| 0.5  | 0.295399      |
| 0.2 | 0.5 | 1.25| 0.5  | 0.228719      |
| 0.4 | 0.5 | 1.25| 0.5  | 0.145372      |
| 0.6 | 0.5 | 1.25| 0.5  | 0.057209      |
| 0.2 | 0.2 | 0.8  | 0.380507      |
| 0.4 | 0.2 | 0.8  | 0.280761      |
| 0.6 | 0.2 | 0.8  | 0.176379      |
| 1.0 | 0.2 | 0.8  | 0.068257      |
| 0.5 | 0.5 | 0.5  | 0.167541      |
| 0.75| 0.5 | 0.5  | 0.193142      |
| 1.00| 0.5 | 0.5  | 0.213866      |
| 1.25| 0   | 0.5  | 0.451032      |
| 0.30| 0   | 0.7  | 0.316919      |
| 0.70| 0   | 0.7  | 0.139779      |
| 1.00| 0   | 0.7  | 0.006359      |

Table 3. Results of Nusselt number at different values of $K$, $M$, $Pr$, and $Ec$ at 20th-order of approximations.
References

1. Von Kármán, T. Über laminare und turbulente Reibung. *Zeitschrift für Angew. Math. Mech* ZAAM 1, 233–252 (1921).
2. Childs, P. R. N. *Rotating Flow* (Elsevier, Amsterdam, 2011).
3. Shevchuk, I. V. Modelising of Convective Heat and Mass Transfer in Rotating Flows (Springer, Berlin, 2016).
4. Stuart, J. T. On the effects of uniform suction on the steady flow due to a rotating disk. *Q. J. Mech. Appl. Math.* 7, 446–457 (1954).
5. Millipas, K. & Pohlhausen, K. Heat transfer by laminar flow from a rotating plate. *J. Aeronaut. Sci.* 19, 120–126 (1952).
6. Riley, N. The heat transfer from a rotating disk. *Q. J. Mech. Appl. Math.* 17, 331–339 (1964).
7. Benton, E. R. On the flow due to a rotating disk. *J. Fluid Mech.* 24, 781–800 (1966).
8. Watson, L. T. & Wang, C. Y. Deceleration of a rotating disk in a viscous fluid. *Phys. Fluids* https://doi.org/10.1063/1.862535 (1979).
9. Miklavcic, M. & Wang, C. Y. The flow due to a rough rotating disk. *Z. Angew. Math. Phys.* 54, 1–12 (2004).
10. Shevchuk, I. V. Turbulent heat transfer of rotating disk at constant temperature or density of heat flux to the wall. *High Temp. 38*, 499–501 (2000).
11. Shevchuk, I. V. Effect of the wall temperature on laminar heat transfer in a rotating disk: an approximate analytical solution. *High Temp. 39*, 682–685 (2001).
12. Shevchuk, I. V. & Buschmann, M. H. Rotating disk heat transfer in a fluid swirling as a forced vortex. *Heat Mass Transf.* 41, 1112–1121 (2005).
13. Xu, H. & Liao, S. A series solution of the unsteady von Kármán swirling viscous flows. *Acta Appl. Math.* 94, 215–231 (2006).
14. Fang, T. & Tao, H. Unsteady viscous flow over a rotating stretchable disk with deceleration. *Commun. Nonlinear Sci. Numer. Simul.* 17, 5064–5072 (2012).
15. Turkylımağızolu, M. MHD fluid flow and heat transfer due to a shrinking rotating disk. *Comput. Fluids* 90, 51–56 (2014).
16. Khan, J. A., Mustafa, M., Hayat, T. & Alsaedi, A. A revised model to study the MHD nanofluid flow and heat transfer due to rotating disk: numerical solutions. *Neural Comput. Appl.* 30, 957–964 (2018).
17. Muthtamilselvan, M. & Renuka, A. Nanofluid flow and heat simultaneously induced by two stretchable rotating disks using Buongiorno’s model. *Multidiscip. Model. Mater. Struct.* 14, 1115–1128 (2018).
18. Doh, D. H., Cho, G. R., Rama, E. & Muthtamilselvan, M. Cattaneo–Christov heat flux model for inclined MHD micropolar fluid flow past a non-linearly stretchable rotating disk. *Case Stud. Therm. Eng.* 14, Article ID: 100496 (2019).
19. Abbas, Z., Jafar, M. A. & Hasnain, J. Asymptotic analysis of MHD viscous fluid flow due to a rotating disc and a radially stretching-shrinking disc with Navier slip condition. *Phys. Scr.* 95, ID: 025204 (2019).
20. Renuka, A., Muthtamilselvan, M., Doh, D. H. & Cho, G. R. Effects of homogeneous-heterogeneous reactions in flow of nanofluid between two stretchable rotating disks. *Eur. Phys. J. Spec. Top.* 228, 2661–2676 (2019).
21. Renuka, A., Muthtamilselvan, M., Doh, D. H. & Cho, G. R. Entropy analysis and nanofluid past a double stretchable spinning disk using Homotopy Analysis Method. *Math. Comput. Simul.* 171, 152–169 (2020).
22. Hoyt, J. W. Some applications of non-newtonian fluid flow. *Rheol. Acta* 8, 797–826 (1999).
23. Elliott, I. Elastico-viscous flow near a rotating disk. *Phys. Fluids* 14, 1086–1090 (1971).
24. Ariel, P. D. Computation of flow of a second grade fluid near a rotating disk. *Phys. Fluids* 22, 1030–1040 (2007).
25. Attia, H. A. The effect of ion-slip on the flow of Reiner–Rivlin fluid due to a rotating disk with heat transfer. *J. Mech. Sci. Technol.* 21, 174–183 (2007).
26. Sahoo, B. Effects of partial slip, viscous dissipation and Joule heating on von Kármán flow and heat transfer of an electrically conducting non-Newtonian fluid. *Commun. Nonlinear Sci. Numer. Simul.* 14, 2982–2988 (2009).
27. Ahmadpour, A. & Sadeghy, K. Swirling flow of Bingham fluids above a rotating disk: An exact solution. *J. Non-Newtonian Fluid Mech.* 197, 41–47 (2013).
28. Griffiths, P. T. Flow of a generalized Newtonian fluid due to a rotating disk. *J. Non-Newtonian Fluid Mech.* 221, 9–17 (2015).
29. Guha, A. & Sengupta, S. Analysis of von Kármán’s swirling flow on a rotating disc in Bingham fluids. *Phys. Fluids* https://doi.org/10.1063/1.4917590 (2016).
30. Doh, D. H. & Muthtamilselvan, M. Thermophoretic particle deposition on magnetohydrodynamic flow of micropolar fluid due to a rotating disk. *Int. J. Mech. Sci.* 130, 350–359 (2017).
31. Tabassum, M. & Mustafa, M. A numerical technique for partial slip flow and heat transfer of non-Newtonian Reiner–Rivlin fluid due to rotating disk. *Int. J. Heat Mass Transf.* 123, 979–987 (2018).
32. Imtiaz, M., Mabood, F., Hayat, T. & Alsaedi, A. Homogeneous-heterogeneous reactions in MHD radiative flow of second grade fluid due to a curved stretching surface. *Int. J. Heat Mass Transf.* 145, 118781 (2019).
33. Sahoo, B. & Shevchuk, I. V. Heat transfer due to revolving flow of Reiner-Rivlin fluid over a stretchable surface. *Therm. Sci. Eng. Prog.* 10, 327–336 (2019).
34. Mustafa, M., Tabassum, M. & Rahi, M. Second law analysis of heat transfer in swirling flow of Binghamfluid by a rotating disk subjected to suction effect. *Therm. Sci. https://doi.org/10.2298/TSCI180722162M* (2019).
35. Beard, D. W. & Walters, K. Elastico-viscous boundary layer flows. Part I. Two-dimensional flow near a stagnation point. *Proc. Camb. Philos. Soc.* 60, 667–674 (1964).
36. Liao, S. J. An optimal homotopy analysis approach for strong nonlinear differential equations. *Commun. Nonlinear Sci. Numer. Simul.* 15, 2003–2016 (2010).
37. Farooq, U., Zhao, Y. L., Hayat, T., Alsaedi, A. & Liao, S. J. Application of the HAM-based Mathematica package BVP 2.0 on MHD Falkner–Skwan flow of Nano-fluid. *Comput. Fluids* 111, 69–75 (2015).
38. Ariel, P. D. On computation of MHD flow near a rotating disk. *J. Appl. Math. Mech.* 82, 235–246 (2002).

Author contributions

M.B.J. has formulated and solved the problem. M.B.J. and M.M. have analyzed the results and written the discussion section. M. Mustafa also managed the Introduction of the manuscript.

Competing interests

The authors declare no competing interests.

Additional information

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