Variable Dual Auxeticity of the Hierarchical Mechanical Meta-
material composed of Re-entrant Structural Motifs

Krzysztof K. Dudek* Julio A. Iglesias Martínez Muamer Kadic

K. K. Dudek
Address: Institute of Physics, University of Zielona Gora, ul. Szafrana 4a, Zielona Gora, 65-069 Poland
Email Address: k.dudek@if.uz.zgora.pl

K. K. Dudek, J. A. Iglesias Martínez, M. Kadic
Address: Institut FEMTO-ST, CNRS, Université Bourgogne Franche-Comté, Besançon, 25030 France

Keywords: auxetic, mechanical metamaterials, re-entrant metamaterials

In this work, a novel hierarchical mechanical metamaterial is proposed that is composed of re-entrant truss-lattice elements. It is shown that this system can deform very differently and can exhibit a versatile extent of the auxetic behaviour depending on a small change in the thickness of its hinges. In addition, depending on which hierarchical level is deforming, the whole structure can exhibit a different type of auxetic behaviour that corresponds to a unique deformation mechanism. This results in a dual auxetic structure where the interplay between the two auxetic mechanisms determines the evolution of the system. It is also shown that depending on the specific deformation pattern, it is possible to observe a very different behaviour of the structure in terms of frequencies of waves that can be transmitted through the system. In fact, it is demonstrated that even a very small change in the parametric design of the system may result in a significantly different band gap formation that can be useful in the design of tunable vibration dampers or sensors. The possibility of controlling the extent of the auxeticity also makes the proposed metamaterial to be very appealing from the point of view of protective and biomedical devices.

1 Introduction

Mechanical metamaterials [1, 2, 3, 4, 5, 6, 7, 8, 9, 10, 11, 12, 13] are rationally-designed structures capable of exhibiting a plethora of atypical mechanical properties that are rarely observed in the case of naturally-occurring materials. Some of the most commonly studied of such properties are negative Poisson’s ratio [14, 15, 16, 17, 18, 19, 20, 21, 22, 23, 24, 25, 26, 27, 28, 29] (auxetic behaviour), negative stiffness [30, 31, 32, 33, 34] and negative compressibility [35, 36]. Over the last thirty years, it has been demonstrated that devices utilising materials exhibiting such characteristics can be used in the case of a variety of applications including protective devices [37, 38], sports equipment [39] as well as biomedical [40, 41] and vibration damping devices [42, 43, 44]. One of the most studied of these unusual mechanical properties seems to be auxetic behaviour where the keen interest of researchers in this property stems from the fact that auxetic materials often exhibit high indentation resistance [45], wave attenuation [46, 47] and many other features that are useful in the case of various applications. The commercial appeal of auxetic mechanical metamaterials resulted in a broad range of studies focused on different types of mechanical structures. One of the most commonly investigated classes of such systems are truss-lattice-based auxetic mechanical metamaterials.

In recent years, auxetic mechanical metamaterials in the form of truss-lattice structures have been a subject of numerous studies [48, 49, 50, 51, 52, 53] with a particular emphasis on the design of efficient protective materials. This stems from the fact that in addition to their ability to exhibit the negative Poisson’s ratio, such systems are also very light in comparison to many other auxetic mechanical metamaterials. This, in turn, is of great significance when designing lightweight protective devices that can exhibit significant energy absorption similarly to their heavier counterparts while at the same time they do not significantly increase the mass of the vehicle/object that they protect. One of the earliest and most famous examples of such structures corresponds to the so-called arrow-head mechanical metamaterial that can be used both as a 2D [49] and 3D [50, 51, 52, 53] system characterised by the negative Poisson’s ratio. Thanks to the success of this and other similar re-entrant mechanical metamaterials [54, 55, 56, 57, 58], over the years, it has been possible to observe a large number of truss-lattice auxetic mechanical metamaterials that are typically working based on a similar hinging mechanism of mutually-
Hierarchical mechanical metamaterials [59, 60, 61, 62, 63, 64, 65, 66] are a class of structures composed of multiple structural levels having their own geometry that typically can deform irrespective of each other. This, in some cases, allows the system to exhibit quantitatively very different mechanical properties without the need of being reconstructed. This, in turn, significantly enhances the applicability of hierarchical structures. In principle, the concept of hierarchical mechanical metamaterials is not limited to a specific type of mechanical metamaterials and can be also applied in the case of truss-lattice systems. In the literature, there are several examples of such structures that are based on re-entrant honeycombs [63] as well as other structural motifs [48]. Nevertheless, this direction of studies is still in its infancy and many new studies have to be conducted in order to better utilise the potential of these structures as well as to fully understand the underlying physics. One of such relatively unexplored aspects related to this class of systems corresponds to the possibility of controlling the band gap formation [65, 67] that would allow to construct tunable vibration insulators or sensors for different ranges of frequencies. Another aspect corresponds to the control over the mechanical properties (e.g. auxeticity) of the hierarchical structure. In this case, despite the success of several projects and their commercial appeal, there are still relatively few known efficient hierarchical mechanisms in comparison to standard non-hierarchical mechanical metamaterials.

In this work, a novel hierarchical mechanical metamaterial composed of truss-lattice elements is proposed and analysed from the point of view of its mechanical properties. More specifically, it is demonstrated that depending on the thickness of different groups of its hinges, the considered structure can follow different deformation patterns. In addition, depending on which hierarchical level is deforming, the whole structure can exhibit a different type of auxetic behaviour that corresponds to a unique deformation mechanism. This results in a dual auxetic structure where the interplay between the two auxetic mechanisms determines the evolution of the system. Finally, it is also presented that depending on which hierarchical level of the structure is deforming, one can observe a drastically different phonon dispersion of the system that also leads to a very different band gap formation.
Figure 1: The unit-cell of the structure analysed in this work from the point of view of its mechanical properties. 

(a) A diagram of the unit-cell of the system with the definition of all of the dimensions and geometric parameters of the structure.

(b) Different deformation patterns exhibited by the considered system in the case of the isolated deformation of the level 0 or level 1 elements of the hierarchical structure.
arrow-head-based structural elements are known in the literature and, as shown in diagrams presented in Fig. 1(b), can exhibit a strong auxetic behaviour. This means that both level 0 and level 1 of the considered hierarchical system correspond to very different deformation mechanisms that can lead to the entire system exhibiting a negative Poisson’s ratio. For the considered model, such observation is of great significance since the two hierarchical levels can deform independently. This means that level 1 blocks can rotate with respect to each other while retaining their shape. Thus, such a process can occur without hinging of truss-lattice beams corresponding to level 0 elements. Conversely, it is possible to observe hinging of level 0 elements without the rotation of level 1 blocks. In the case of this model, the hinging of level 0 elements is quantified by a change in the value of $\theta$ (see Fig. 1(a)). On the other hand, the rotation of level 1 blocks is quantified by a variation in the value of $\gamma$. As a result, once the structure is manufactured and specific geometric dimensions are well-defined, the magnitudes of angles $\theta$ and $\gamma$ provide full information about the configuration assumed by the structure assuming no flexure of truss-lattice elements.

The factor that determines the extent of the deformation of level 0 and level 1 of the structure is the relative thickness of hinges connecting structural elements corresponding to different hierarchical levels. These hinges are denoted as $d_1$-$d_5$ in Fig. 1(a) and in general can assume very different values. As demonstrated schematically in Fig. 1(b), it is expected that upon changing the relative thickness of hinges connecting truss-beams corresponding to level 0 and level 1 elements, it should be possible to increase the extent of deformation of one hierarchical level over another. In the extreme case, if the difference in the thickness of the two groups of hinges was very considerable, it could be even possible to observe an isolated deformation either of level 0 or level 1 of the system. Such possibility is shown in Fig. 1(b) where in the first scenario one can see the deformation of the structure corresponding solely to the rotation of level 1 blocks while in the second case the deformation corresponds exclusively to hinging of level 0 elements. However, in many situations, it can be expected that the mixture of the two deformation mechanisms could be observed.

At this point, it should be also noted that the two types of truss-lattice beams present within the system have a very specific shape. Namely, both ends of such beams having the length of either $a$ or $b$ (see Fig. 1(a)) are significantly narrower than their central parts. This particular aspect of the design of the considered system allows ensuring that its deformation will occur primarily via the relative hinging of truss-lattice beams and that their flexure will be negligible.

### 2.2 Simulations

To assess the mechanical properties of the considered system, all models are deformed via the compression along the $y$-axis as schematically indicated by means of black arrows in Fig. 1(b). Furthermore, since only a single unit-cell is taken into account, in order to describe the behaviour of the system, it is assumed that periodic boundary conditions are imposed [69] on the left and right edge of the unit-cell. These boundary conditions ensure that the two edges retain the form of a straight line throughout the deformation process. In order to simulate the behaviour of the structure, Finite Element Method (FEM) simulations were conducted through the use of the COMSOL Multiphysics software. In order to obtain realistic results, the nonlinear geometry feature was enabled while it was assumed that the material itself was isotropic. More specifically, the respective properties of the material used for the considered model were set to be the following for all conducted simulations. Young’s modulus: 4 GPa, Poisson’s ratio: 0.4 and density: 1200 kg m$^{-3}$. These material properties correspond to some of the resins [70] that could be used in order to 3D print the considered structure. The remaining geometric parameters used in the simulations were set to be the following: $a = 3.5$ cm, $b = 1.5$ cm, $\theta(t = 0) = 46.24^\circ$ (initial value), $\gamma(t = 0) = 10^\circ$ (initial value), $l_1 = 5.5$ mm, $l_2 = 4$ mm, $l_3 = 2$ mm.

At this point, it should be mentioned that in this work, only three variations of the considered system are being studied instead of a hypothetical larger number of possible scenarios. The reason for this is...
3 Results and Discussion

In order to assess the ability of the considered system to exhibit versatile deformation patterns and different values of the negative Poisson’s ratio, three different configurations of the considered model were selected. These configurations named case 1, case 2, and case 3 are almost identical with the only difference between them being the thickness of hinges $d_1$ - $d_6$. Specific values of these parameters are provided in Table 1.

| Case | $d_1$ [mm] | $d_2$ [mm] | $d_3$ [mm] | $d_4$ [mm] | $d_5$ [mm] | $d_6$ [mm] |
|------|-------------|-------------|-------------|-------------|-------------|-------------|
| Case 1 | 1.6         | 0.6         | 8.0         | 4.0         | 0.6         | 4.0         |
| Case 2 | 0.2         | 2.5         | 0.4         | 0.2         | 0.4         | 0.2         |
| Case 3 | 0.8         | 1.6         | 1.6         | 1.4         | 1.6         | 0.8         |

Table 1: The thickness of hinges corresponding to three types of configurations of the considered model.

the fact that there are only three types of the behaviour of the analysed structure that are significantly different from each other. Namely, the two deformation processes corresponding solely to the hinging of level 0 elements or the rotation of level 1 blocks as well as the hybrid case associated with the combination of these two deformation mechanisms. Of course, one could consider a larger number of scenarios but all of them would exhibit the behaviour that is qualitatively the same as for the already considered cases.

3.1 Controllable auxetic behaviour

As shown in Fig. 2, the three different versions of the considered system exhibit very distinct deformation patterns. According to Fig. 2(a), the structure named case 1 deforms primarily via the rotation of level 1 blocks while level 0 elements do not undergo the hinging process. This stems from the fact that as specified in Table 1, hinges within the arrow-head-shaped structural elements (in particular the hinge having its thickness defined as $d_1$) are in this case very thick in comparison to hinges connecting level 1 blocks (primarily hinges having their thickness defined as $d_2$ and $d_5$ although all hinges with the exception for the hinge having the thickness of $d_1$ affect the rotation of level 1 elements). A very different behaviour can be observed in the case of the structure named case 2 where level 1 blocks seemingly do not rotate while level 0 elements undergo a hinging process to a significant extent. This very different deformation pattern can be explained by the fact that in this scenario, some of the hinges connecting level 1 blocks are very thick. On the other hand, hinges connecting truss beams within arrow-head structural elements are very thin. Finally, it is also important to note that as demonstrated in the case of the deformation of the structure referred to as case 3, it is possible to observe the simultaneous rotation of level 1 elements and hinging of level 0 structural motifs. For better clarity, all three of these deformation mechanisms are schematically shown in Fig. 2(c). At this point, it is worth mentioning that in the hypothetical scenario where all of the hinges present within the system would have the same thickness, the behaviour of the structure would resemble that of the system referred as case 3 (see Supplementary Information). It is also worth noting that the three cases considered in this work represent the only types of the mechanical behaviour that are qualitatively different from each other. Any other combination of the values $d_1$-$d_6$ than those already considered will result with one of the three deformation mechanisms described in the main text.

In addition to the possibility of conducting the qualitative comparison of the bahaviour of the three types of the considered model, it is also possible to conduct the quantitative analysis where the variation in specific geometric parameters would be analysed. As shown in Fig. 3, the above qualitative discussion matches the variation in angles $\theta$ and $\gamma$ that quantify the deformation of level 0 and level 1 respectively. More specifically, one can note that angle $\theta$ changes significantly for case 2 and case 3 structures (see Fig. 3(a)) where both of these cases correspond to a significant hinging of level 0 elements. On the other
Figure 2: Deformation patterns and Poisson’s ratio of the three versions of the considered model. **a)** Deformation of the three analysed cases subjected to the compression along the $y$-axis. For each of the examples, the black outline in the background corresponds to the initial shape of the structure. The extent of mechanical deformation is graphically enhanced by a factor of 2 to visually emphasise the difference between different deformation mechanisms. The deformation processes corresponding to structures named case 1, case 2 and case 3 are also presented as animations provided in the form of the **Supplementary Video 1**, **Supplementary Video 2** and **Supplementary Video 3**. **b)** Engineering Poisson’s ratio for all types of the system. **c)** Conceptual diagrams showing differences between the three types of possible deformation patterns.
3.2 Vibrational properties

In addition to the possibility of controlling the deformation pattern and the extent of the exhibited auxeticity, it is also interesting to check whether different types of the considered model allow observing some differences in terms of the band gap formation and frequencies of vibrations that can propagate through the system. To achieve it, FEM simulations were conducted by means of the COMSOL Multiphysics software. In the case of each of the structures, to generate the results, Floquet periodic boundary conditions were implemented in the $x$- and $y$-directions assuming two-dimensional unit-cells. All of the design parameters were set to be identical as in the case of mechanical testing.

Based on Fig. 4, one can note that the phonon dispersion graphs corresponding to systems called case 1 and case 2 are very different. Most importantly, in the case of the first system, within the presented

Figure 3: Variation in the angles describing the deformation of level 0 and level 1 of the considered hierarchical structure.
range of frequencies, there are no band gaps. On the other hand, the phonon dispersion associated with the case 2 structure corresponds to a band gap for frequencies approximately in the range between 0.5 kHz and 1.3 kHz. This, in turn, is very interesting as it indicates that a small change in the thickness of the hinges may cause the appearance of a considerable band gap where its size can be further modified by adjusting the relative thickness of the hinges. This means that the considered system can in general undergo a transition from the structure with no band gaps to the system working as a vibration damper where specific frequencies are not transmitted. Of course, in the current version of the proposed model such transition cannot occur in an active manner since the change in the thickness of hinges would require reconstructing the system or subjecting it to a post-manufacturing treatment that would alter some of its dimensions. However, this is not the only approach making it possible to change the behaviour of the considered system from the characteristic typical for the case 1 system to the behaviour of the case 2 structure and vice versa. This can be achieved for example by constructing hinges from another material corresponding to a very different thermal expansion coefficient than the rest of the system. Such a solution could allow changing the behaviour of the structure at different temperatures. Another approach could correspond to the use of electromagnets / magnets [53] embedded in specific parts of the unit-cell. The mutual interaction of such inclusions could adjust the effective stiffness of specific hinges within the system and as a result, would allow for the active transition of the structure. At this point, we would like to also emphasise the fact that according to the results presented in Fig. 4, the band gap formation can be controlled by the relative thickness of hinges within the system with similar results being observed for other hierarchical mechanical metamaterials known in the literature [67]. However, even though this topic does not belong to the scope of this work, the band gap formation could also be potentially influenced by the deformation of the system. Nevertheless, this aspect of the behaviour of the considered structure would have to be thoroughly analysed before reaching any conclusions.

In this work, we decided to focus on two specific configurations that deform via two different deformation mechanisms, i.e. the hinging of level 0 elements and the rotation of level 1 blocks. As demonstrated in Fig. 4, these two cases result in drastically different phonon dispersion relationship, where for the structure referred to as case 2, a significant band gap can be observed. At the same time, for the structure referred to as case 1, no band gap can be seen within the considered range of frequencies. If one was to consider the hybrid scenario that incorporates the two types of the deformation mechanisms, i.e. the system called case 3, the size of the band gap would be reduced relative to the structure referred to as case 2. This stems from the fact that as demonstrated in Fig. 4(a), the rotation of level 1 blocks does...
not lead to the formation of band gaps.

All of this is very important since, in this work, it is shown that the proposed model can exhibit versatile deformation patterns depending on the interplay between the two auxetic mechanisms present within the system. It is also shown that a modification in the thickness of different groups of hinges can significantly change the extent of the auxetic behaviour exhibited by the system. Both of these results are not commonly observed for standard mechanical metamaterials and can have multiple applications. More specifically, if one was to construct the active version of the considered system (e.g. by using mutually-interacting inclusions in the form of electromagnets to modify the effective stiffness of hinges), then it would be possible to significantly change auxeticity of the structure without the need of reconstructing the system. This would allow to construct highly-efficient protective devices that could adjust the extent of their auxeticity to a specific cause of the deformation. The concept presented in this work could be also very interesting from the point of view of biomedical devices. More specifically, if one was to construct the structure composed of multiple unit-cells corresponding to a different deformation pattern, then in general, it would be possible to arbitrarily modify the shape of the entire structure. If the mismatch in the Poisson’s ratio between different parts of the system would be significant, it would be possible to observe considerable shape-morphing similarly to the concept proposed in [66] (in the aforementioned study, it was demonstrated that an initial shape of the metamaterial subjected to the mechanical deformation can be significantly modified depending on the ratio in the Poisson’s ratio associated with different parts of the structure). The possibility of exhibiting controllable shape-morphing could be used in the design of stents in order to better support specific parts of the blood vessel that are particularly weak. Finally, it is important to emphasise the fact that the results presented in this work are not limited to a specific scale. In fact, the proposed model can be even constructed at the microscale.

4 Conclusion

It is shown that the considered hierarchical structure can follow very different deformation patterns depending on the variation in the thickness of its hinges. This means that the proposed system can utilise very different deformation mechanisms which is very atypical and highly beneficial in comparison to typical mechanical metamaterials. Furthermore, in this work, it is presented that the considered model can exhibit a very broad range of the negative Poisson’s ratio that can be controlled by adjusting the relative stiffness of hinges. This result, which originates from the interplay of two independent auxetic mechanisms present within the structure, indicates that in theory, the auxeticity of the system can be adjusted depending on the specific application. This, in turn, could prove to be very useful in the design of highly-efficient protective devices. Finally, it should be emphasised that different configurations of the considered system may result in a very distinct behaviour of the structure in terms of frequencies of waves that can be transmitted through the analysed metamaterial. In fact, it is demonstrated that even a very small change in the parametric design of the system may lead to a significantly different band gap formation that can be useful in the design of tunable vibration dampers or sensors.

Acknowledgements
K.K.D. acknowledges the support of the Polish National Science Centre (NCN) in the form of the grant awarded as a part of the SONATINA 5 program, project No. 2021/40/C/ST5/00007 under the name “Programmable magneto-mechanical metamaterials guided by the magnetic field”. This research was funded by the Polish Minister of Education and Science under the program “Regional Initiative of Excellence” in 2019–2023, project No. 003/RID/2018/19, funding amount PLN 11 936 596.10.

Conflict of Interest
The authors declare no conflict of interest.

Data Availability Statement
The data that support the findings of this study are available from the corresponding author upon reasonable request.
References

[1] B. Florijn, C. Coulais, M. van Hecke, Phys. Rev. Lett. 2014, 113 175503.
[2] C. Coulais, E. Teomy, K. de Reus, Y. Shokef, M. van Hecke, Nature 2016, 535 529.
[3] G. W. Milton, J. Mech. Phys. Solids 2013, 61 1543.
[4] C. Coulais, A. Sabbadini, F. Vink, M. van Hecke, Nature 2018, 561 512.
[5] J. Christensen, M. Kadic, O. Kraft, M. Wegener, Mrs Commun. 2015, 5, 3 453.
[6] R. M. Neville, F. Scarpa, A. Pirrera, Sci. Rep. 2016, 6 31067.
[7] H. Jiang, L. L. Barbenchon, B. A. Bednarcyk, F. Scarpa, Y. Chen, Addit. Manuf. 2020, 36 101430.
[8] T. Frenzel, M. Kadic, M. Wegener, Science 2017, 358, 6366 1072.
[9] M. Kadic, G. W. Milton, M. van Hecke, M. Wegener, Nat. Rev. Phys. 2019, 1, 3 198.
[10] Z. Zhang, A. O. Krushynska, APL Mater. 2022, 10 080701.
[11] L. Mizzi, A. Spaggiari, Int. J. Solids Struct. 2022, 238 111428.
[12] L. Mizzi, E. Salvati, A. Spaggiari, J.-C. Tan, A. M. Korsunsky, Appl. Mater. Today 2020, 20 100780.
[13] K. K. Dudek, M. Marc, W. Wolak, A. Drzewinski, M. R. Dudek, Phys. Status Solidi B 2021, 258 2100162.
[14] R. F. Almgren, J. Elasticity 1985, 15 427.
[15] K. W. Wojciechowski, Phys. Lett. A 1989, 137 60.
[16] K. W. Wojciechowski, A. C. Branka, Phys. Rev. A 1989, 40 7222.
[17] R. Lakes, Science 1987, 235 1038.
[18] K. E. Evans, A. Alderson, Adv. Mater. 2000, 12 617.
[19] S. Babaee, J. Shim, J. C. Weaver, E. R. Chen, N. Patel, K. Bertoldi, Adv. Mater. 2013, 25 5044.
[20] T.-C. Lim, Compos. Struct. 2019, 209 34.
[21] N. Novak, L. Biaisetto, P. Rebesan, F. Zanini, S. Carmignato, L. Krstulović-Opara, M. Vesenjak, Z. Ren, Addit. Manuf. 2021, 45 102022.
[22] W. Jiang, X. Ren, S. L. Wang, X. G. Zhang, X. Y. Zhang, C. Luo, Y. M. Xie, F. Scarpa, A. Alderson, K. E. Evans, Compos. B. Eng., 2022, 235 109733.
[23] J. N. Grima, K. E. Evans, J. Mater. Sci. Lett. 2000, 19 1563.
[24] N. Novak, O. Duncan, T. Allen, A. Alderson, M. Vesenjak, Z. Ren, Mech. Mater. 2021, 157 103818.
[25] J. W. Narojczyk, K. V. Tretiakov, K. W. Wojciechowski, Phys. Status Solidi B 2022, 259 2200006.
[26] J. W. Narojczyk, M. Bilski, J. N. Grima, P. Kedziora, D. Morozow, M. Rucki, K. W. Wojciechowski, Materials 2022, 15 1134.
[27] O. Duncan, G. Leslie, D. Sawtell, T. Allen, Smart Mater. Struct. 2022, 31 074002.
[28] K. V. Tretiakov, K. W. Wojciechowski, Phys. Status Solidi B 2020, 257 2000194.
[29] W. G. Hoover, C. G. Hoover, Phys. Status Solidi B 2005, 242 585.
[30] K. Bertoldi, V. Vitelli, J. Christensen, M. van Hecke, Nat. Rev. Mater. 2017, 2 17066.
[31] T. A. M. Hewage, K. L. Alderson, A. Alderson, F. Scarpa, Adv. Mater. 2016, 28 10323.
[32] S. Chen, X. Tan, J. Hu, S. Zhu, B. Wang, L. Wang, Y. Jin, L. Wu, Compos. B. Eng. 2021, 215 108745.
[33] K. K. Dudek, R. Gatt, M. R. Dudek, J. N. Grima, Proc. R. Soc. A 2018, 474 20180003.
[34] X. Tan, L. Wang, S. Zhu, S. Chen, B. Wang, M. Kadic, Eur. J. Mech. A Solids 2022, 96 104702.
[35] R. Lakes, K. W. Wojciechowski, Phys. Status Solidi B 2008, 245 545.
[36] K. K. Dudek, D. Attard, R. Caruana-Gauci, K. W. Wojciechowski, J. N. Grima, Smart Mater. Struct. 2016, 25 025009.
[37] G. Imbalzano, S. Linforth, T. D. Ngo, P. V. S. Lee, P. Tran, Compos. Struct. 2018, 183 242.
[38] M. Miniaci, A. Krushynska, F. Bosia, N. M. Pugno, New J. Phys. 2016, 18 083041.
[39] O. Duncan, T. Shepherd, C. Moroney, L. Foster, P. D. Venkatraman, K. Winwood, T. Allen, A. Alderson, Appl. Sci. 2018, 8 941.
[40] H. M. A. Kolken, S. Janbaz, S. M. A. Leeflang, K. Lietaert, H. H. Weinans, A. A. Zadpoor, Mater. Horiz. 2018, 5 28.
[41] T. van Manen, S. Janbaz, K. M. B. Jansen, A. A. Zadpoor, Commun. Mater. 2021, 2 56.
[42] D. E. Quadrelli, R. Craster, M. Kadic, F. Braghin, Extreme Mech. Lett. 2021, 44 101262.
[43] S. Guenneau, A. Movchan, G. Petursson, S. A. Ramakrishna, New J. Phys. 2007, 9 399.
[44] R. Fleury, D. L. Sounas, C. F. Sieck, M. R. Haberman, A. Alì, Science 2014, 343, 6170 516.
[45] T. Li, F. Liu, L. Wang, Compos. B. Eng. 2020, 198 108229.
[46] M. Ruzzene, F. Scarpa, Phys. Status Solidi B 2005, 242 665.
[47] D. Qi, H. Yu, W. Hu, C. He, W. Wu, Y. Ma, Extreme Mech. Lett. 2019, 28 58.
[48] L. Mizzi, A. Spaggiari, Smart Mater. Struct. 2020, 29 105036.
[49] U. Larsen, O. Sigmund, S. Bouwsta, J. Microelectromech. Syst. 1997, 6 99.
[50] K. K. Dudek, D. Attard, R. Gatt, J. N. Grima-Cornish, J. N. Grima, Materials 2020, 13 2193.
[51] M.-F. Guo, H. Yang, L. Ma, Compos. Struct. 2022, 290 115484.
[52] T.-C. Lim, Phys. Status Solidi B 2016, 253 1252.
[53] K. K. Dudek, R. Gatt, J. N. Grima, Mater. Des. 2020, 187 108403.
[54] L. Yang, O. Harrysson, H. West, D. Cormier, Int. J. Solids Struct. 2015, 69-70 475.
[55] W. Liu, N. Wang, T. Luo, Z. Lin, Mater. Des. 2016, 100 84.
[56] K. K. Dudek, A. Drzewinski, M. Kadic, Proc. R. Soc. A. 2021, 477 20200825.
[57] M. H. Fu, O. Xu, L. Hu, T. Yu, Int. J. Solids Struct. 2016, 80 284.
[58] X. Tan, J. A. I. Martinez, G. Ulliac, B. Wang, L. Wu, J. Moughames, M. Raschetti, V. Laude, M. Kadic, Small 2022, 18 2202128.
[59] R. Lakes, Nature 1993, 361 511.
[60] Y. Tang, G. Lin, L. Han, S. Qiu, S. Yang, J. Yin, *Adv. Mater.* 2015, 27 7181.

[61] R. Gatt, L. Mizzi, J. I. Azzopardi, K. M. Azzopardi, D. Attard, A. Casha, J. Briffa, J. N. Grima, *Sci. Rep.* 2015, 5 8395.

[62] N. An, A. G. Domel, J. Zhou, A. Rafsanjani, K. Bertoldi, *Adv. Funct. Mater.* 2020, 30 1906711.

[63] R. Oftadeh, B. Haghpanah, D. Vella, A. Boudaoud, A. Vaziri, *Phys. Rev. Lett.* 2014, 113 104301.

[64] K. K. Dudek, R. Gatt, M. R. Dudek, J. N. Grima, *Materials* 2021, 14 758.

[65] Y. Cho, J.-H. Shin, A. Costa, V. K. T. A. Kim, J. Li, S. Y. Lee, S. Yang, H. N. Han, I.-S. Choi, D. J. Srolovitz, *Proc. Natl. Acad. Sci.* 2014, 111 17390.

[66] K. K. Dudek, J. A. Iglesias Martínez, G. Ulliac, M. Kadic, *Adv. Mater.* 2022, 34 2110115.

[67] V. Kunin, S. Yang, Y. Cho, P. Deymier, D. J. Srolovitz, *Extreme Mech. Lett.* 2016, 6 103.

[68] J. N. Grima, P.-S. Farrugia, R. Gatt, D. Attard, *Phys. Status Solidi B* 2008, 245 521.

[69] L. Mizzi, D. Attard, R. Gatt, K. K. Dudek, B. Ellul, J. N. Grima, *Eng. Comput.* 2021, 37 1765.

[70] F. Ugarak, G. Ulliac, J. A. Iglesias Martínez, J. Moughames, V. Laude, M. Kadic, A. Mosset, *Materials* 2022, 15 4070.