Molecular dynamics simulation of non-azeotropic refrigerants separation in auto-cascading refrigeration

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Abstract: The core problem of auto-cascading refrigeration is the separation of non-azeotropic refrigerants. The separation effect of mixed refrigerant directly determines the refrigeration effect of auto-cascading refrigeration system and the value of COP. The microscopic molecular dynamics simulation method is applied to the refrigeration system, and the properties of the pure substance and the mixed substance are simulated from the microscopic aspects to study the law of molecular condensation cooling, and the separation rate is simulated to obtain a more optimized scheme.

1 Introduction

Auto-cascading refrigeration is one of the main ways to achieve a temperature range of -60°C ~ -150°C. It has very important applications in the fields of national defense, military, modern industry, bio medicine and scientific research. Since the cascade system is in a dynamic stable state, the internal pressure is large, and the movement law of the material cannot be directly detected, the calculation of the system separation rate has been unable to be solved. In this paper, molecular dynamics simulation is used to simulate the process of refrigerant separation in the automatic cascade system to solve this problem.

2 Auto-cascading refrigeration system

As shown in Figure 2.1, the cycle diagram of the non-azeotropic mixed refrigerant auto-cascading refrigeration system used in this study. The specific process is described as follows: a certain proportion of non-azeotropic mixed working fluid is compressed into a high temperature and high pressure state in the compressor A, and then condensed in the condenser B. Most R22 is condensed into liquid, while most medium boiling point working medium R23 and low boiling point working medium R14 are still gases. The mixture flows through a liquid-storage separator E1, in which it is separated into gas-liquid phase. R22 components of liquid from the bottom of the separator, and the temperature in the separation tank E2 is controlled at about -30°C, most of the R23 gas in the tank is condensed into liquid, again by gravity make R23 liquid from the bottom of the tank E2, the temperature control for tank E2 is around -70°C. R14 components is the same process, finally, the evaporator temperature reached to -90°C, the evaporator of the air back through the plate heat exchanger in turn F1, F2 and F3 arriving at gas-liquid separator into the compressor suction mouth, finishing a complete refrigeration cycle [1].

3 Molecular dynamics simulation process

3.1 Potential function and molecular model

The potential function used is the most widely used potential energy function Lennard-Jones (L-J) potential, which can be used to describe the interaction between molecules of inert gases and can also be used to describe some metals such as chromium, molybdenum [2]. The equation of the potential function is:

$$U_{ij}(r_{ij}) = A_m \left( \frac{d}{r_{ij}} \right)^m - B_n \left( \frac{d}{r_{ij}} \right)^n $$

Figure 2.1 Flow chart of non-azeotropic mixed refrigerant auto-cascading refrigeration cycle

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3.2 Periodic boundary conditions and Berendsen’s thermal bath method

The Periodic Boundary Conditions reflect how boundary conditions can be used to replace selected parts (systems) from the surrounding environment\cite{3}. It can be seen as the condition that if the surrounding environment is removed and the system is kept unchanged, it can also be regarded as part of the nature to promote the expression of the global nature. For this problem, the periodic boundary conditions can greatly improve the computational efficiency and increase the accuracy of the simulation calculations is fully applicable\cite{4-5}.

The Berendsen thermal bath method is based on the principle that the system is in contact with an infinitely large temperature bath to maintain its temperature stability\cite{6}. The equation for temperature change is:

\[ \Delta T = \frac{\delta T}{\tau} (T_{\text{bath}} - T(t)) \]  

(3-2)

calibration factor is:

\[ \lambda^2 = 1 + \frac{\delta T}{\tau} \left( \frac{T_{\text{bath}}}{T(t)} - 1 \right) \]  

(3-3)

This method allows the system temperature to float up and down at the desired temperature.

3.3 Simulation process

In order to test the accuracy of the model, we simulated the R23 and R14 pure substances separately. The simulation uses the GROMACS simulation software package to simulate refrigerants R23 and R14 under periodic boundary conditions. The pure material and mixture were then simulated under the control of a thermal bath at Berendsen with a cut-off radius of 11 Å and a time step of 0.001ps. The simulated control conditions are: pressure 3.5MPa, temperature decreased from 281K to 250K, and the equilibrium simulation time is 3ns. 1300 R23 and 137 R14 molecules to be placed in a mixed box according to the same ratio, and the simulation time was 3ns.

The initial distribution of molecules in the box of the size of refrigerant R23 at the beginning of MD simulation is shown in the figure below:

![Figure 3.2 Distribution of pure material R23 initial position (t=0s)](image)

The size of the box when the mixed working fluids R23 and R14 start to simulate at the beginning of the MD, the initial distribution of molecules in the box as shown:

![Figure 3.3 Initial position distribution of the mixture (R23-left, R14-right) (t=0s)](image)

3.3.1 Pure material simulation

According to figure 3.4, we can see that the density value increases with time, and increases to a constant value of 156.86kg/m$^3$(temperature $T=281K$) and 1215.73kg/m$^3$(temperature $T=250K$). With the continuous extension of simulation time, this constant value almost remains unchanged, indicating that the simulation system has entered an equilibrium state. The simulation results are close to the data of NIST database 148.76kg/m$^3$(temperature $T=281K$) and 1197.6kg/m$^3$(temperature $T=250K$), indicating that the molecular and potential functions of R23 simulated by MD are consistent with the actual substance and can meet the calculation needs.

![Figure 3.4 R23 density versus time graph](image)
The solid line in the figure represents the MD simulation results, and the dotted line represents the NIST database data. From the figure, we can find that the simulated data is slightly different from the standard data, but it can be used to describe the properties of materials. It can be used in MD mixture simulation.

Figure 3.5 R14 density versus time graph

Figure 3.5 describes the relationship between the density and time of refrigerant R14 at a constant pressure of 281K and 250K.

3.3.2 Simulation of mixed substances

According to the equal mass ratio of the two refrigerants, 104 R23 particles and 83 R14 particles were selected to mix to make the simulation run 3ns under NPT conditions. The density changes with time at temperatures of 281K and 250K were obtained as shown in figure 3.6.

![Figure 3.6 Mixture density versus time](image)

According to this figure, the density increases with time and remains almost unchanged, indicating that the simulation system has entered an equilibrium state. Figure 3.7 shows the particle position distribution of 1300 R23 particles and 137 R14 particles at t=3ns. The temperature (a) is 281K and the temperature (b) is 250K. In order to distinguish the two particle distributions, we use red and blue to represent R23 and R14, respectively. By comparing the figures (a) and (b), we can find that the mixture volume decreases as the temperature decreases, which indicates that the distance between molecules decreases. But since the system is at rest, it is difficult to accurately judge the separation of the two substances.

Figure 3.7 (a) 281K particle position distribution (b) 250K particle position distribution

According to the equal volume ratio of the two refrigerants, 1300 R23 particles and 137 R14 particles are mixed. Under the control of NVT conditions, the simulation time is 3 ns, and the vapor-liquid separation of the mixture is studied according to the diffusion principle. Figure 3.8 depicts the positional distribution of particles at (a) t = 0 ns and (b) t = 3 ns at a temperature of 250K. Observing this figure, it is found that the interface between the two substances is fused because the diffusion of the two substances occurs at rest. A part of the gas diffuses into the liquid, and a part of the liquid also diffuses into the gas, thus forming a vapor-liquid mixing interface. According to the diffusion principle, the separation rate of the mixture can be solved.

![Figure 3.8 Particle position distribution diagram of the system](image)

Figure 3.8 shows the density distribution in the z-axis direction. With the increase of the z-axis, the density of the system increases to a certain density value of 1082.46 kg/m³, and then the density value begins to rapidly decline to a constant value of 63.45 kg/m³. The critical values of the simulated interface are 978.63 kg/m³ and 71.42 kg/m³ respectively. The dark blue dotted line in the figure indicates the location of the interface.
We use the distribution of particles in the interface to calculate the separation of matter. The separation rate of a substance in the mixture is calculated as follows:

\[ \eta = \frac{n \cdot M}{\rho \cdot N_A \cdot v} \times 100\% \quad (3-4) \]

According to the mass density distribution of the refrigerants R23 and R14, the separation rates of R23 and R14 were calculated to be 21.36% and 32.86%, respectively, using the separation ratio (3-4). Empirically, the separation rate is 10%-20% for better separation. In order to find the optimum separation rate, the cooling amount that provides condensation and the ratio of R23/R14 were changed and simulated again.

When the condensing temperature is changed, the temperature of R23/R14 changes when R22 provides cooling capacity, and the separation rate is 22.73% and 35.42% when the temperature is 281K-250K. The ratio of different R23/R14 was used for simulation, and the ratio separation rate of 1:1 was 35.42%, which was the maximum.

4 Conclusion

For the ternary non-azeotropic mixture in the auto-cascading refrigeration system, when the cooling capacity of the condensing is changed and the ratio of the refrigerant is changed, the separation effect is changed. By analyzing the molecular dynamics simulation results, when R22 provides cooling capacity, the separation rates of R23/R14 changes accordingly, and the separation rate of temperature from 281K to 250K is 22.73% and 35.42%. The simulation was carried out with different ratios of R23/R14, and the ratio of separation ratio obtained at 1:1 was 35.42%. The more cooling is provided and the more R23 is condensed out. Similarly, R23 condenses into the lower stage for final cooling. The R14 refrigerant is larger and the system COP is also increased.

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