Sound Pressure Level Analysis of a Liquid-Fueled Lean Premixed Swirl Burner with Various Quarls

Gergely I. Novotni and Viktor Józsa *

Budapest University of Technology and Economics, Faculty of Mechanical Engineering, Department of Energy Engineering, 1111 Budapest, Műegyetem rkp. 3., Hungary; novotni@energia.bme.hu
* Correspondence: jozsa@energia.bme.hu; Tel.: +36-1-463-2596

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Abstract: Swirl burners are widely used in numerous practical applications since they are characterized by low pollutant emission and a wide operating range. Besides reliable operation, a burner must fulfill noise emission regulations, which is often a sound pressure level in dB(A) when people are affected. Therefore, the present paper evaluates the overall sound pressure level (OASPL) variation of a 15-kW liquid-fueled turbulent atmospheric swirl burner at various setups. Firstly, the combustion air flow rate was adjusted, which induced a swirl number modification due to the fixed swirl vanes. Secondly, the atomizing pressure of the plain-jet airblast atomizer was modified, which also affected the swirl number. High atomizing air jets notably increased combustion noise by intensifying the shear layer. Thirdly, a geometrical modification was performed; 0°–60° half cone angle quarls in 15° steps were installed on the lip of the baseline burner for extended flame stability. By filtering the OASPL to the V-shaped flames, a linearly decreasing trend was observed as a function of swirl number. The derivative also has a linearly decreasing characteristic as a function of the atomizing pressure.

Keywords: swirl combustion; sound pressure level; combustion noise; liquid fuel combustion; quarl; lean combustion; turbulent combustion

1. Introduction

Premixed swirl combustion in V-shaped mode is widely used in steady-operating industrial combustion chambers to fulfill emission standards [1,2]. The leaner the flame, the lower is the emission of nitrogen oxides (NOx); thus, the operation of the combustion chamber lies close to the flame blowout by design [3,4]. Consequently, a high understanding of combustion dynamics, especially thermo-acoustic oscillations [5], is required to prevent operational failures [6]. There are numerous papers available on the investigation of blowout using various time series analyses [7–9]. Typically, OH* and/or acoustic measurements are performed to detect oscillations at a few hundred Hz, which characterizes the flame in V-shaped mode. Their relative intensity is monitored, which shows an increase prior to blowout [6,10]. The majority of acoustical studies focus on common fossil fuels; however, it was shown earlier that stability limits are significantly affected by fuel type and combustion conditions [11]. By installing a quarl on the burner, the stable operating regime can be significantly extended [12]; hence, burner design is investigated in great detail in this paper.

Besides pollutant emission regulations, acoustic emission is also significant [13,14]. Power plants and airports that were built outside of major cities a couple of decades ago belong to suburban regions now [15]. Therefore, operational noise has to be minimized [16,17]. In the case of aviation engines, the contribution of combustion noise is notable at full thrust during takeoff [18,19]. Even though advanced airfoils with porous trailing edges can reduce airfoil noise [20], combustion is still a notable part of the acoustic emission [21]. Combustion noise usually consists of a broadband spectrum without sharp peaks. However, V-shaped flames are characterized by frequencies in the range of a
A few hundred Hz, which tend to decrease as the operation becomes leaner [6]. The theoretical interaction between turbulence and combustion was described by Clavin and Siggia [22]. It is an important observation that as the A-weighting [23] provides a negative gain up to 1 kHz, a positive gain follows it in the 1–6 kHz range. Above 6 kHz, the gain is also negative. Even though the gain of the human ear is slightly different from A-weighting [24], the agreement is close enough to use it in international noise regulations [25–28]. Interestingly, the threshold values in standards and directives are given in the overall sound pressure level (OASPL), the spectral composition of the source is omitted. The OASPL is the total energy contained in the spectrum, usually normalized to a reference pressure of 20 mPa, which is also used in the present study. Since liquid fuel was combusted presently and a plain-jet airblast atomizer was used, the high-velocity shear is a governing factor in the resulting combustion noise. Acoustical characteristics of a similar free jet were discussed by Freund [29] under cold conditions. The directional dependence of such an acoustic source was described by Bailly et al. [30]. Combustion acoustics and its modeling was excellently reviewed by Candel et al. [31].

To date, numerical prediction of combustion noise is available, however, it is limited to simple fuels and burner configurations and is computationally expensive as it requires direct numerical simulation [32,33]. Speth and Ghoniem [34] discussed the OASPL of syngas combustion in the composition range of 20%–80% by CO under lean conditions to evaluate the operation modes of their swirl-stabilized burner. Methane and ethane fuel-rich non-swirl flames diluted with hydrogen were investigated by Singh et al. [35] with the result of linear correlation between OASPL-Mach number and air-to-fuel equivalence ratio-Mach number trends. It is known that fuel-rich mixtures are characterized by increased noise compared to lean combustion, and the OASPL peak is located near the stoichiometric condition [36]. Swirl vanes are widely used in combustion technology; however, they are also excellent for noise suppression of supersonic free jets [37,38]. Apart from pure hydrocarbon flames, Shanbhogue et al. [39] investigated the extinction strain of various methane-hydrogen mixtures. The elevated hydrogen content allowed a wider stability range and resulted in a slightly reduced combustion noise. The OASPL was also found to be a good indicator of flame stability of a propane step burner with hydrogen dilution [40,41].

The present paper investigates the OASPL of a lean premixed swirl burner equipped with an airblast atomizer. Since atomization is governed by atomizing air pressure, it allows a semi-flexible operation as it can be adjusted almost freely to provide the desired spray quality with low dependence on the fuel flow rate. This was the reason why airblast atomizers were developed [42]. In order to compare burner geometries, various quarls were applied on the burner, which alters the blowout stability of the flame. The OASPL trends were evaluated by using a wide parameter matrix, focusing principally on V-shaped flames.

2. Materials and Methods

Firstly, the combustion test rig is discussed, emphasizing on the acoustic setup to provide a general overview of the measurement system. Secondly, the estimated atomization characteristics and fuel properties are presented, highlighting the atomizing free jet, which greatly influences the swirling flow in the mixing tube (discussed in the last subsection).

2.1. Experimental Setup

Figure 1 shows the atmospheric test rig with the installed burner, which is detailed in Subsection 2.2. The combustion air was delivered by a frequency inverter-controlled fan, and flow rate measured by a rotameter. Then, a PID-controlled preheater (HAGA Kft., Budapest, Hungary) increased the air temperature to 400 °C before entering the burner. Over the course of the measurements, the combustion air flow rate increased from 11.9 kg/h until blowout in 2.38 kg/h steps in all setups, which meant 2 m³/h steps in rotameter reading. Since the atomizing air flow rate can be calculated using an adiabatic expansion at the atomizer nozzle [43], the second rotameter was used only for validation.
The air-to-fuel equivalence ratio, $\lambda$, was varied in the range of 0.7–2. Note that it refers to primary $\lambda$, and the rest of the air was used from the atmosphere for complete combustion. $\lambda$ is defined by Eq. (1):

$$\lambda = \frac{m_{\text{air}}}{m_{\text{air,sto}}},$$

where $m_{\text{air}}$ is the actual sum of combustion air plus atomizing air flow rates and $m_{\text{air,sto}}$ refers to the air mass flow rate required for stoichiometric combustion. The burner without a quarl is referred to as the baseline. When quarls were applied, they were put directly to the burner lip, detailed in Subsection 2.2. All variants had 16 mm slant height, while the half cone angle varied from 0° to 60° in 15° steps. The setups mentioned later are referred by their quarl half cone angle. The corresponding pollutant emission analysis was discussed in a previous paper [12]; consequently, it is omitted now.

A SVAN 971-type sound analyzer (SVANTEK, Warsaw, Poland) was used for the acoustic measurement and placed 1 m from the burner axis at the same height as the burner lip. The data acquisition frequency was 12 kHz, which is enough for capturing the typical combustion-related noise in rich detail [44]. The sensitivity of the prepolarized 1/2", 1st class ACO 7052E condenser microphone was 28.74 mV/Pa. The device was automatically calibrated with an SV33 calibrator (114 dB at 1 kHz, complies with the IEC 60942:2003 standard, SVANTEK). For data evaluation, its dedicated software, the SvanPC++ (version 3.1.5, SVANTEK), was used. The software calculations were verified by Matlab scripts at several randomly selected measurement points, using raw sound files. The recording was continuous from ignition until blowout. However, at least 30 s combustion noise was recorded at each measurement condition after reaching the steady-state operation of the corresponding setup. The relative standard deviation of the OASPL over 30 s varied between 1% and 5.6%, depending on the operational parameters. Fourteen percent relative standard deviation was calculated at two measurement points of the 0° quarl near the critical swirl number, which is discussed later in this section. For the spectral analysis, a window size of 1024 samples was used with no overlap. All the 30 s acoustic measurements took place after at least 30 s warm-up time upon setting the desired conditions. This time frame was adequate for the burner to reach its corresponding steady-state temperature. However, the OASPL was principally governed by the highly intense shear generated by the high-velocity atomizing annular jet.

The host environment of the test rig was a heat engine laboratory where this device shared the room with gas turbines and boilers which did not operate during the acoustic measurements. The walls contained several utility pipes, including electricity, water, natural gas, and the exhaust system. The stairs and several cabinets made the acoustic environment even more complex. Consequently, neither single frequency-amplitude pairs nor accurate OASPL measurement was claimed in this paper.
2.2. Burner Design and Atomization

The investigated lean premixed prevaporizing-type burner is shown in Figure 2, featuring its swirl vanes and the plain-jet airblast atomizer, and the general design of the used quarls. It has a central \( d_0 = 0.4 \text{ mm} \) inner diameter fuel pipe, and the atomizing air is accelerated in a concentric annular nozzle (0.8 mm inner and 1.6 mm outer diameter). The mixing tube is 75.5 mm long, measured from the atomizer tip, and its inner radius is \( R = 13.4 \text{ mm} \). In this study, the atomizing gauge pressure, \( p_g \), was varied from 0.3 to 1.6 bar. The uncertainty of pressure measurement was below 1 kPa. The burner was used in a Capstone C30 micro gas turbine while the quarls were manufactured in the local workshop.

\[ SMD = 0.66 \cdot d_0 \cdot We_A^{-0.5} (1 + 1/AFR), \]  
where \( AFR \) is the air-to-fuel mass flow ratio of the atomizer and \( We_A \) is the Weber number of the atomizing air as follows:

\[ We_A = \rho_A \cdot \frac{w_A^2}{\sigma} \cdot d_0, \]
where \( \rho_A \) is the density of atomizing air at the nozzle, \( w_A \) is the atomizing air discharge velocity, and \( \sigma \) is the surface tension of the utilized standard diesel oil (EN 590:2014). Table 1 summarizes the principal parameters of atomization, considering a constant fuel flow rate since the combustion power was uniformly 15 kW for all the cases. \( Ma = a/w_A \) is the Mach number, where \( a \) is the speed of sound of the fully expanded jet. The notable fuel properties are summarized in Table 2.

| \( p_g \) (bar) | \( SMD \) (\( \mu \text{m} \)) | \( We_A \) (-) | \( AFR \) (-) | \( \rho_A \) (kg/m\(^3\)) | \( T_A \) (K) | \( Ma \) (-) |
|----------------|----------------|-------------|-------------|----------------|------------|---------|
| 0.3            | 21.6           | 779         | 0.778       | 1.27           | 277        | 0.62    |
| 0.45           | 16.2           | 1121        | 0.948       | 1.31           | 268        | 0.75    |
| 0.6            | 13.4           | 1438        | 1.089       | 1.34           | 261        | 0.85    |
| 0.8            | 11.1           | 1830        | 1.249       | 1.39           | 252        | 0.95    |
| 1.1            | 9.17           | 2363        | 1.451       | 1.45           | 241        | 1.08    |
| 1.6            | 7.44           | 3143        | 1.725       | 1.54           | 227        | 1.25    |
2.3. Swirl Characterization and Observed Flames

The burner has four radial, and fifteen fixed 45° slots; hence, swirling flow is generated at the inlet. The corresponding dimensionless quantity is the swirl number, introduced by Beér and Chigier [49]:

\[ S = \frac{G_φ}{(G_x R)}, \]  

where \( G_φ \) is the axial flux of the angular momentum, \( G_x \) is the axial thrust, and \( R \) is the radius of the mixing tube. By assuming the conservation of momentum, the geometric swirl number is usually calculated by Eq. (5), based on the inlet conditions of the burner [49] since the spatial distribution of the pressure and velocity of the burner is rarely available in practice:

\[ S' = \frac{G_φ}{(G'_x R)}, \]  

where \( S' \) is the geometric swirl number, \( G'_x \) is the thrust at the inlet; hence, the pressure field can be omitted, and average velocity can be used for \( S' \) estimation. This calculation method of the geometric swirl number was criticized by, e.g., Galley et al. [50], and Durox et al. [9]. Nevertheless, the cited results are limited for a different burner, and its generalization to the present burner is not possible. Hence, Eq. (5) is used as the definition of the swirl number. Since the swirl vanes were fixed, the swirl number varied with the combustion air flow rate. The axial thrust is significantly increased by the increasing atomizing air flow rate at elevated \( p_o \), leading to low \( S' \). \( G'_x \) was calculated by Equation (6):

\[ G'_x = \dot{m}_C \cdot w_C + \dot{m}_A \cdot w_A + \dot{m}_F \cdot w_F, \]  

where subscripts \( C \) and \( F \) refer to combustion air and fuel, respectively. \( G_φ \) is estimated as:

\[ G_φ = \frac{1}{1-\psi} \cdot \frac{\tan(a)}{1+\tan(a)\tan(\pi/z)} \cdot \frac{\dot{m}_φ}{\rho C 2\pi R'}, \]  

where \( a = 45° \) is the swirl vane angle, \( z = 15 \) is the number of swirl slots, \( B = 8.8 \) mm is the height of the swirl slots, \( \rho C = 0.52 \) kg/m3 is the density of combustion air, and \( \dot{m}_φ = A \cdot \dot{m}_C \) is the mass flow rate passing through the swirl vanes. \( A \) is the area ratio of the two inlet types since the pressure drop across them was negligible. Hence, \( A = h \cdot B \cdot z / (h \cdot B \cdot z + y \cdot D^2 \cdot \pi/4) = 0.44 \), where \( h = 1.2 \) mm is the width of the swirl slots, \( y = 4 \) is the number of circular slots with \( D = 8 \) mm diameter. The blockage factor, \( \psi \), is calculated as follows:

\[ \psi = \frac{z s}{2\pi R \cos(\alpha)} \]  

where \( s = 3.3 \) mm is the slot-to-slot distance in the circumference, understood as the width of the metal blockage.

Depending on the swirl number, two regimes and a transitory zone between them can be distinguished. At low \( S' \), the flame is straight. At \( S' \sim 0.52 \), the flame enters a transitory regime where both straight and V-shape are observable with a random alteration between them. Above \( S' \sim 0.81 \), a stable V-shaped flame develops. The mentioned \( S' \) values for the geometric swirl number correspond to the baseline burner, which is only slightly affected by the quarls. The mentioned flame shapes are presented in Figure 3. Even though there are visible flares, the majority of the droplets evaporate...
inside the mixing tube, according to our previous work [43]. Notable flares were only observed in the \( \lambda < 1 \) range. The straight flame has a helical flow structure, shown by the twisted flares. Both the Inner Recirculation Zone (IRZ) and the Outer Recirculation Zone (ORZ) were indicated in the case of V-shaped flames. A significant portion of the ORZ is occupied and hence substituted by the 45° quarl on the right image.

![Flame structures](image)

**Figure 3.** Straight flame at the end of the transitory regime (left, \( p_g = 0.3 \) bar, \( \lambda = 1.1, \) and \( S' = 0.82 \)), stable V-shaped flame (middle, \( p_g = 0.3 \) bar, \( \lambda = 1.23, \) and \( S' = 0.91 \)), and V-shaped flame with a 45° quarl (right, \( p_g = 0.3 \) bar, \( \lambda = 1.25, \) and \( S' = 0.92 \)).

The Reynolds number, \( Re \), of the fuel-air mixture was calculated by Eq. (9), considering the vaporization of the liquid fuel droplets and its mixing with the expanded atomizing air and the preheated combustion air. It was assumed that there is no flashback, which would otherwise increase \( Re \).

\[
Re = w_x \cdot 2 \cdot R \cdot \rho_x / \mu_x, \tag{9}
\]

where \( w_x \) is the mixture discharge velocity at the mixing tube outlet, \( \rho_x \) is the density of the mixture, and \( \mu_x \) is its dynamic viscosity. The Reynolds number values varied between 6800 and 18680; therefore, the combustion was always in the turbulent regime.

### 3. Results and Discussion

This section starts with the discussion of the spectral characteristics of the investigated burner to understand the results of the OASPL analysis better, including the typical noise pattern of the various flame shapes. Secondly, the OASPL is discussed at \( p_g = 0.3 \) bar, highlighting the characteristic effect of the combustion air flow rate on the combustion noise. Thirdly, the effect of the combustion air flow rate on OASPL with Z-weighting is analyzed, emphasizing the V-shaped flames. Lastly, the trends in the previous results are combined to conclude a general noise emission characteristic of the investigated swirl burner.

#### 3.1. Spectral Analysis of the Flame

Initially, the combustion air flow rate was low, which resulted in a straight flame at \( p_g = 0.3 \) bar with \( S' = 0.48 \). The increase in combustion air flow rate, hence, in tangential momentum due to the fixed 45° swirl vanes, led to a random breakdown and merging of the precessing vortex core, called the transitory regime. A stable, V-shaped flame was then observable until blowout, shown in Figure 4. Note that certain setups reached neither the transitory state nor the subsequent V-shaped operation. They were the ones with high atomizing pressures and quarls, which provided no or low effect on flame stability. The spectrum of the straight flame was dominated by frequencies between 3 and 4 kHz. This band starts to fade as the flame approaches the transitory regime with the increasing combustion air flow rate, and hence \( S' \). By increasing the air flow rate further, the average occurrence ratio of the two flame shapes starts to tend to the V-shaped flame; therefore, the amplitude at > 3 kHz decreases and low-frequency bands start to emerge, especially at 500 and 220 Hz. The V-shaped
regime is characterized by lower amplitudes, and the energy density of the reacting flow is concentrated into the sub-1 kHz regime until blowout and contains no peak in the 3–4 kHz band.

![Figure 4. Spectrogram of a single measurement at $p_g = 0.8$ bar, using a 15° quarl. The shaded areas indicate the changeover between two points with distinct flame characteristics. Blowout is indicated by the vertical black line, beyond which the spectrum of the cold flow noise is visible.]

Since the V-shaped flame will be evaluated in the following subsections, it was crucial to define where exactly the V-shaped regime starts. Presently, the mentioned high and low-frequency bands were checked. When only high-frequency components were present in the 30 s signal, the flame was classified as a straight flame; if a mixture of the bands was found in the spectrum, it was the transitory regime. With solely low-frequency components, the flame is called V-shaped. Note that there was no M-shaped flame observed. Singh et al. [44,51] report that the combustion-related noise is concentrated below 1.5 kHz, which is in line with the present spectrum in the V-shaped regime, compared to the spectrum of the blowout. However, this statement fails for the straight flames in the present case. Due to the increase in combustion air flow rate, and hence in $\lambda$, the amplitudes of characteristic frequency bands start to decrease, while the rest of the spectrum remains similar. Considering the A-weighting function, shown in Figure 5, it becomes evident that the shift in characteristic frequencies alone results in lower OASPL, if straight and V-shaped flames are compared. Also, the leaner operation further reduces combustion noise. As a consequence, lean V-shaped flames mean a decreased physiological exposure.

![Figure 5. A and Z weighting functions.]

3.2. OASPL at $p_g = 0.3$ bar

Figure 6 shows the variation of the OASPL as a function of $S'$ and air-to-fuel equivalence ratio, $\lambda$, in four different setups: baseline configuration and with 0°, 15°, and 30° quarls equipped. Here, $p_g$
was uniformly set to 0.3 bar. The results are presented for both A- and Z-weighting functions, indicating the various flame shapes by using different marker types. The combined expanded uncertainty ranges from 3% to 7.5% for $S'$ at 95% level of significance, which is 5%–6.5% for $\lambda$. The higher values characterize low flow rates, while the lower values are typically close to the blowout.

The discussed uncertainty ranges incorporate all cases, including the later discussed ones. Hence, the corresponding error bars are omitted from all figures containing measurement data for better visibility.

**Figure 6.** (a), (c) OASPL with Z- and (b), (d) A-weighting functions using the baseline burner and various quarls as a function of $S'$ (a) and (b) and $\lambda$ (c) and (d) all at $p_g = 0.3$ bar. Symbols: straight flame (○), transitory flame (×), and V-shaped flame (◊).

As was shown in Figure 4, the dominant frequency band of straight flames is in the regime of the slightly positive gain of the A-weighting function. Therefore, the results are close to the values calculated by Z-weighting. As the flame enters the transitory regime, the OASPL with A-weighting decreases compared to that by Z-weighting since the $<1$ kHz regime becomes increasingly dominant. The transitory flame is characterized by a slightly higher OASPL than the straight flame by Z-weighting, which is due to the random alteration between the two stable flame shapes. The noise of the V-shaped flames decreases as the combustion air flow rate increases. This trend is closely linear for all cases and has a higher slope with A-weighting since there is a slight shift towards lower frequencies in the sub-1 kHz frequency regime, in accordance with the swirl combustion literature [6]. The linearity is indicated by the coefficient of determination of 0.9 and above for all cases. Even though the baseline configuration seems to be characterized by the lowest noise emission, the difference is only a few dB at all measurement points in Figures 6a and 6c. This difference is slightly amplified by the A-weighting function in Figures 6b and 6d. However, the improved flame stability performance of both 15° and 30° diffusers is remarkable. Overall, the lowest OASPL was measured
by using a 15° quarl at $\lambda = 2$ or $S' = 1.23$. In the following subsection, the effect of quarls is emphasized as a function of $S'$.

### 3.3. *Effect of Quarls*

Besides the emphasized quarls in subsection 3.2, 45° and 60° quarls are also discussed in the present subsection. In addition, the analysis covers several additional $p_s$s. Hence, data of various quarls are shown in separate diagrams, evaluating only the $OASPL$ with Z-weighting as a function of $S'$.

Before proceeding into the results, the fluid dynamical behavior of the quarls is briefly summarized. Compared to the baseline burner, 0° means only a 16 mm longer mixing tube section. In addition, this quarl adds thermal inertia to the system, which facilitates flame stabilization; however, the two configurations practically showed similar results from this point of view. The next two quarls, 15° and 30°, were the best in flame stabilization for the present burner configuration, allowing a significant extension of the stable operating regime at all $p_2$ [12]. At the initial combustion air flow rates, the flow was governed by the atomizing air jet at the center; hence, the flow was attached to the quarl wall later. The flow separation here was acceptable since the operation was far from the blowout. As the combustion air flow rate was increased, the increasing $S'$ resulted in flow attachment to the wall, allowing flame stabilization through the appearing boundary layer. The 45° quarl provided a sufficient flow attachment at lower $p_{2s}$ however, its half cone angle was too large for higher $p_s$ where the central regime was notably affected by the atomizing air jet. Therefore, the detached flow resulted in similar flame blowout characteristics as the baseline burner had. The flow was always separated from the 60° quarl; it only provided some protection against cold air entrainment. The blowout characteristics were close to the baseline and 0° configurations.

Figure 7 shows the results of the $OASPL$ measurement of all configurations. While increasing the swirl number at the straight flame shape, combustion noise also increases by few decibels or less. The transitory flame shape is characterized by decreasing noise; however, only V-shaped flames show a clear tendency of decreasing $OASPL$, which finally might exceed −10 dB compared to the noisiest operation of the given measurement series.

Interestingly, Figs. 7e and f show an increasing $OASPL$ when the flame is V-shaped, while the transitory regime indicates a decreasing trend. This property might be attributed to the not well-attached flow to the quarl walls, shown in Figure 8. Stable, V-shaped flame was achieved up to 0.8 and 0.6 bar for the 45° and 60° quarls, respectively. The V-shaped operation was limited to $p_s = 0.3$ bar for the baseline burner configuration and up to 0.6 bar for the 0° quarl. Therefore, further analyses are limited to the mentioned configurations where a stable V-shaped flame was achieved for at least two consequent measurement points, excluding 45° and 60° quarls. Figures 7e and d showed stable V-shaped operation for all $p_{2s}$, showing a nearly linear decrease in $OASPL$ while increasing $S'$.

The straight flame is characterized by similar $OASPL$ for all quarls and a given $p_s$. In this state, the higher $p_s$ results in higher $OASPL$. The intersection of the trends is due to the enhanced thrust, generated by the elevated $p_{2s}$, hence, the denominator of Equation (5) was high. Therefore, the increased combustion air flow rate caused a smaller increase in $S'$ while the $OASPL$ notably decreased in the case of V-shaped flames. As for straight flames, the higher $p_s$ leads to higher $OASPL$ for all cases due to the enhanced shear induced by the atomizing jet. The noise under hot conditions was about 20 dB higher due to the following reason. The fuel first mixes with atomizing air; hence, the conditions of ignition could be satisfied here, which lead to a rapid expansion inside the shearing flow structure, significantly enhancing the generated noise by the high-velocity atomizing air jet.
Figure 7. OASPL of the (a) baseline burner and (b)–(f) 0°–60° quarls at six psig as a function of $S'$, using Z-weighting. Symbols: straight flame ($\circ$), transitory state ($\times$), and V-shaped flame ($\Diamond$).

Figure 8 shows four flame images with three different quarls to highlight the flame attachment to the quarl wall, which was mentioned above. Since the burner operates in an open atmosphere, fully attached flames could entrain the least amount of cold ambient air.
3.4. Linearity Analysis of the OASPL trends of V-shaped Flames

It was shown previously that the quarls provide excellent blowout stability; therefore, several operating points at V-shaped flame were investigated beyond the capabilities of the baseline burner. The extended regime allowed the collection of data, especially at elevated atomizing pressures. The straight flame and transitory regimes both have a relatively low effect on OASPL and showed varying trends. However, the V-shaped flame always showed a close linear decrease in OASPL, which proposes the analysis of linearity.

Table 3. The coefficient of determination of linear fit lines for A- and Z-weighting functions as a function of $S'$ and $\lambda$.

| Quarl | $p_b$ (bar) | $R^2$ ($S'$, dB(Z)) | $R^2$ ($S'$, dB(A)) | $R^2$ ($\lambda$, dB(Z)) | $R^2$ ($\lambda$, dBA) |
|-------|-------------|----------------------|----------------------|--------------------------|------------------------|
| Baseline | 0.3 | 0.944 | 0.970 | 0.971 | 0.988 |
| 0° | 0.3 | 0.885 | 0.924 | 0.926 | 0.955 |
| 0.3 | 0.963 | 0.996 | 0.993 | 0.993 | 0.993 |
| 15° | 0.45 | 0.989 | 0.997 | 0.995 | 0.987 |
| 0.6 | 0.988 | 0.965 | 0.979 | 0.947 | 0.947 |
| 0.8 | 0.974 | 0.964 | 0.966 | 0.955 | 0.955 |
| 1.1 | 0.999 | 0.999 | 0.998 | 0.998 | 0.998 |
| 1.5 | 1.000 | 1.000 | 1.000 | 1.000 | 1.000 |
| 0.3 | 0.802 | 0.934 | 0.853 | 0.967 | 0.967 |
| 15° | 0.45 | 0.763 | 0.722 | 0.793 | 0.739 |
| 0.6 | 0.983 | 0.980 | 0.970 | 0.966 | 0.966 |
| 0.8 | 0.987 | 0.994 | 0.978 | 0.989 | 0.989 |
| 1.1 | 0.993 | 0.998 | 0.989 | 0.995 | 0.995 |
| 1.5 | 0.994 | 0.990 | 0.992 | 0.987 | 0.987 |

All the V-shaped operation points were filtered from the above data for the baseline burner and the 0°-30° quarls. Then, the fitting two parameters were determined for all data sets. This procedure was performed by using both A- and Z-weighting functions, analyzing the variation as a function of
S’ and air-to-fuel equivalence ratio, λ as well. The resulting coefficient of determinations was listed in Table 3. This procedure required at least two subsequent measurement points, which, hence, may result in a coefficient of determination of unity. Most values well exceed 0.9, except the 30° quarl at $p_g = 0.45$ bar.

The derivative of the fitted linear trend lines as a function of the atomizing pressure. (a) geometric swirl number, (b) air-to-fuel equivalence ratio.

4. Conclusions

A 15-kW lean premixed prevaporized swirl burner was investigated in the present paper under atmospheric conditions, fueled by standard diesel oil. The noise emission of its operation at various atomizing pressures, swirl numbers, air-to-fuel ratios (λ) were evaluated in terms of sound pressure level. As for geometrical modification, quarls with 0°–60° half cone angles in 15° steps were installed on the burner lip beside the baseline burner. All measurements started from a fixed combustion air flow rate, and it was increased equidistantly until blowout. Straight flame was observed initially in all cases, while the transitory and then the V-shaped flames were limited to a lower number of setups. The overall sound pressure level (OASPL) was evaluated for all setups, using at least a 30 s noise signal at steady operation, using 12 kHz sampling frequency. Based on the results, the following conclusions were derived:

Increasing atomizing pressure results in elevated combustion noise due to the intensifying induced shear. As a consequence, the swirl number decreases as the axial thrust increases.

V-shaped flame of the baseline burner and the 0°–30° quarls showed a linear decrease in OASPL as the swirl number and hence the air-to-fuel ratio increased. The 45° and the 60° quarls resulted in a slightly increasing OASPL at V-shaped flame. However, the flame was not attached to the quarl wall at higher $p_g$. 

The use of concluded decreasing derivatives with elevated $p_g$ might contribute to advanced combustion chamber designs for decreased noise emission and a healthier environment for the operators and affected personnel.
By fitting a line to the V-shaped operation, OASPL data of the baseline burner and 0°–30° quarls showed a linearly decreasing trend in their derivative by both $\lambda$ and $S'$ with the increasing atomizing pressure, by using either A- or Z-weighting functions.

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**Nomenclature**

**Latin letters**

- $a$ (m/s): Speed of sound
- $AFR$ (-): Air-to-fuel mass flow ratio
- $B$ (mm): Height of the swirl slots
- $D$ (mm): Diameter of the circular slots
- $d_0$ (mm): Fuel pipe inner diameter
- $G_a$ (Nm): Axial flux of the angular momentum
- $G$ (N): Axial thrust
- $G'$ (N): Axial thrust for the geometric swirl number calculation
- $h$ (mm): Width of the swirl slots
- $\dot{m}$ (kg/s): Mass flow rate
- $Ma$ (-): Mach number
- $p_a$ (bar): Atomizing gauge pressure
- $R$ (m): Mixing tube inner radius
- $Re$ (-): Reynolds number
- $S$ (-): Swirl number
- $S'$ (-): Geometric (estimated) swirl number
- $s$ (mm): Slot-to-slot distance in the circumference of the mixing tube
- $SMD$ ($\mu$m): Sauter Mean Diameter
- $OASPL$ (dB): Overall Sound Pressure Level
- $T$ (K): Temperature
- $w$ (m/s): Velocity
- $We$ (-): Weber number
- $y$ (piece): Number of circular slots in the cylindrical surface of the mixing tube
- $z$ (piece): Number of swirl slots in the cylindrical surface of the mixing tube

**Greek letters**

- $\alpha$ (°): Swirl vane angle
- $\lambda$ (-): Air-to-fuel equivalence ratio
- $\mu$ (Pa·s): Dynamic viscosity
- $\psi$ (-): Blockage factor
- $\rho$ (kg/m³): Density
- $\sigma$ (N/m): Surface tension

**Subscripts**

- $A$: Atomizing air
- $air$: Sum of air
- $C$: Combustion air
- $F$: Fuel
$S$  
Air passing through the swirl vanes

$sto$  
Stoichiometric

$x$  
Mixture

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