Model-Predictive-Control-Based Time-Optimal Trajectory Planning of the Distributed Actuation Mechanism Augmented by the Maximum Performance Evaluation

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Abstract: Trajectory planning for a redundant manipulator is a classic problem. However, because it is difficult to precisely evaluate its maximum performance, an optimization method has been typically used. In this study, a novel time-optimal trajectory planning method for a redundant manipulator is proposed using the model predictive control (MPC) augmented by the maximum performance evaluation (MPE). First, the optimization formulation is expressed to evaluate the maximum performance of the distributed-actuation-mechanism-based three-revolute-joint manipulator (DAM-3R), which has a high level of redundancy, and the joint-actuation-mechanism-based three-revolute-joint manipulator (JAM-3R) for comparison. The optimization is conducted by linking the multibody dynamics analysis module and the optimization module. For time-optimal trajectory planning, the MPC problem is then formulated using mathematical performance models for the DAM-3R and JAM-3R based on the MPE results, which are considered as the upper bound of the manipulator performance at each end-effector position. To verify the proposed method, a point-to-point task with no predefined path is investigated. The simulation results show that the working time of the DAM-3R is 19.1% less than that of the JAM-3R. Moreover, the energy consumption for the DAM-3R is 45.0% lower than that for the JAM-3R by optimally utilizing the higher redundancy of the DAM-3R. Thus, it can be concluded that the proposed method is effective for time-optimal trajectory planning for redundant manipulators.

Keywords: time-optimal trajectory planning; distributed actuation mechanism; maximum performance evaluation; model predictive control

1. Introduction

Trajectory planning is an ever-challenging issue in robotics. A typical category of trajectory planning problems is time-optimal trajectory planning [1–3]. Another popular category is to minimize energy consumption [4–8]. In addition, limiting the joint speed and acceleration [9] and minimizing the actuation force [10], joint torques [11], and joint wear [12] have been widely studied. However, most studies have been limitedly conducted under a predefined fixed path.

To overcome the above limitation, model predictive control (MPC) has been recently used to solve time-optimal trajectory problems with given initial and end points only [13,14]. This approach can create both the optimal path (i.e., the locus of points in the task space) and the optimal trajectory (i.e., a path on which a timing law is specified) for a minimal operation time. Note that this is essential for point-to-point motion control with no predefined path. However, implementing MPC requires an explicit mathematical model that can precisely express the dynamic performance of the system. Thus, there exists a technical hurdle to implementing MPC-based trajectory planning for redundant manipulators because it is difficult to determine their mathematical performance model.
In robotics, redundant manipulators are widely used because they provide an increased level of dexterity by virtue of having more degrees of freedom (DOFs) in the joint space than those required to execute a given task. The additional DOFs in the joint space can be used to avoid obstacles [15,16], joint limits [17], and collision with other robots [18], and also to minimize joint velocity and acceleration [19] and joint torque [20]. They also provide an advantage in performing maintenance and inspection tasks [21]. Among various redundant manipulators, the distributed actuation mechanism (DAM) can enhance the fingertip force through relocating a redundant actuation point of a slider along the link without changing the posture [22,23]. However, such redundancy inevitably leads to difficulty in precisely evaluating its performance and thereby determining an optimal trajectory for a given task.

It is interesting to note that the bioinspired variable gearing of the three-link DAM has been recently investigated to maximize the manipulation performance [24]. Because the positions of the six redundant sliders, which actuate three joints, function as a gear ratio, the DAM can achieve better performance than the power-equivalent joint actuation mechanism (JAM). A gradient-based optimization algorithm was implemented to precisely conduct maximum performance evaluation (MPE) under redundant DOFs.

This paper proposes a novel time-optimal trajectory planning method for redundant manipulators by conducting MPC using the MPE results. First, the DAM-based three-revolute-joint manipulator (DAM-3R) was analyzed to obtain the MPE results using gradient-based optimization. The MPE results were used to construct an explicit performance model for the redundant DAM-3R as the upper bound of the manipulator performance at each end-effector position. Then, MPC was conducted to determine a time-optimal trajectory using the explicit performance model. For comparison, the JAM-based three-revolute-joint manipulator (JAM-3R), which is also a redundant manipulator, was constructed and analyzed to have equivalent power and geometric specifications.

This study is organized as follows. In Section 2, based on a brief explanation on the concept of the MPE, the MPE results for the DAM-3R and JAM-3R are obtained using gradient-based optimization. Section 3 describes the MPC formulation to determine a time-optimal trajectory for the DAM-3R. The JAM-3R is compared to demonstrate the validity and significance of the proposed method. The conclusions follow in Section 4.

2. Maximum Performance Evaluation

2.1. Optimization Formulation

The dexterity motion of a human finger is not achieved by lumped actuation, but by spatially distributed actuation of opponens pollicis (Figure 1a). Based on the understanding of such spatially distributed actuation, the DAM was designed to actuate one joint by thrusting two sliders along a link [22] (Figure 1b). Because its joint angular velocity and torque vary depending on the position of the sliders (i.e., the application point of force), the DAM is a redundantly actuated system.

![Figure 1. Actuation mechanism inspired by finger movement: (a) Spatially distributed actuation of a finger [24]; (b) A distributed actuation mechanism [22].](image-url)
In principle, the optimization formulation can be expressed to precisely evaluate the maximum performance of a manipulator [25], as follows:

Find \( x_1, x_2, \ldots, x_n \)

To maximize \( f(x) = |v_e(x)| \)

Subject to \( h = d_k \cdot v_e = |d_k||v_e| \quad k = 1, \ldots, K \)

\( g_v(x) \leq 0 \quad \nu = 1, \ldots, N \)

\( x_i^{(l)} \leq x_i \leq x_i^{(u)} \quad i = 1, \ldots, n \)

where the superscripts \((l)\) and \((u)\) denote the lower and upper bounds of the design variable \(x_i\); the objective function \(f\) is the norm of the end-effector velocity; \(h\) is the equality constraint that matches the direction of the end-effector velocity \(v_e\) with the base direction \(d_k\); and \(g_v\) is the inequality constraint that represents motor performance.

Figure 2 shows a DAM-based three-revolute-joint manipulator (DAM-3R), which was investigated in this study. The position and velocity of the sliders can be treated as control variables that determine the end-effector velocity \(v_e\) with the base direction \(d_k\); and \(g_v\) is the inequality constraint function that matches the direction of the end-effector velocity \(v_e\) with the base direction \(d_k\); and \(g_v\) is the inequality constraint that represents motor performance. For comparison, a joint actuation mechanism (JAM)-based three-revolute-joint manipulator (JAM-3R) was also considered with power-equivalent motors and the same geometric specifications (Figure 3 and Table 1). When the end-effector is free to rotate, the inequality constraint function \(g_v\) was formulated as follows:

\[
\begin{align*}
\mathcal{g}_1 & = F_j \pm \frac{F_{\text{max}}}{s_{\text{max}}} s_j \pm F_{\text{max}} \leq 0 \quad j = 1, 2, 3 \\
\mathcal{g}_2 & = F_j^b \pm \frac{F_{\text{max}}}{s_{\text{max}}} s_j^b \pm F_{\text{max}} \leq 0 \quad j = 1, 2, 3
\end{align*}
\]

where the subscript \(j\) indicates the joint number; \(F_j\) and \(F_j^b\) are the thrusting force of the front slider and back slider, respectively; \(F_{\text{max}}\) is the maximum thrusting force of the front and back sliders; and \(s_{\text{max}}\) is the maximum thrusting speed of the front and back sliders. In this study, gravitational force was applied as an external force to reflect inertia loads. In this study, the base direction \(d_k\) was defined as follows:

\[
d_k = [\cos \alpha, \sin \alpha]^T
\]

where \(\alpha\) is at intervals of 22.5 degrees from 0 to 360 degrees. This generated a total of 16 base directions (i.e., \(K = 16\) in (1)). Because the speed and force of the sliders must operate within motor characteristics curves in four quadrants, \(g_v\) was formulated as follows:

\[
\mathcal{g}_1 = F_j \pm \frac{F_{\text{max}}}{s_{\text{max}}} s_j \pm F_{\text{max}} \leq 0 \quad j = 1, 2, 3
\]

\[
\mathcal{g}_2 = F_j^b \pm \frac{F_{\text{max}}}{s_{\text{max}}} s_j^b \pm F_{\text{max}} \leq 0 \quad j = 1, 2, 3
\]

where the subscript \(j\) indicates the joint number; \(F_j\) and \(F_j^b\) are the thrusting force of the front slider and back slider, respectively; \(F_{\text{max}}\) is the maximum thrusting force of the front and back sliders; and \(s_{\text{max}}\) is the maximum thrusting speed of the front and back sliders. In this study, gravitational force was applied as an external force to reflect inertia loads. In this study, gravitational force was applied as an external force to reflect inertia loads.
In principle, the optimization formulation can be expressed to determine the end-effector velocity, and control variables that determine the end-effector velocity, was investigated in this study. The position and velocity of the sliders can be treated as geometric specifications (Figure 3 and Table 1). When the end-effector is free to rotate, the constraint that matches the direction of the end-effector velocity was defined as follows:

\[ \mathbf{F}_g \leq \mathbf{F}_s \]

\[ \mathbf{F}_j \leq \mathbf{F}_s \]

where the superscripts (\(g\), \(s\), \(j\)) denote the lower and upper bounds of the design variables.

Here, the design variable vector became \( \mathbf{x} \) (1). Because the speed and force of the sliders must operate under the effects of gravity [24].

\[ \dot{\mathbf{x}} = \mathbf{A} \mathbf{x} + \mathbf{B} \mathbf{u} \]

\[ \dot{\mathbf{u}} = \mathbf{C} \mathbf{x} + \mathbf{D} \mathbf{u} \]

Figure 2. A three-link planar manipulator equipped with the distributed actuation mechanism under the effects of gravity [24].

Figure 3. A three-link planar manipulator equipped with a conventional joint actuation mechanism under the effects of gravity.

Table 1. Specifications of the commercial motors selected for the DAM-3R and JAM-3R.

| Motor                  | DAM-3R  | JAM-3R  |
|------------------------|---------|---------|
| Model                  | PGM12-1230 | DCX 16 S |
| \( \omega_0 \) [rpm]   | 12,500  | 6340    |
| \( \tau_s \) [mNm]     | 3.12    | 12.5    |
| \( \omega_s \tau \) area [W] | 2.04    | 4.15    |
| Weight [g]             | 13      | 26      |
Table 2. Design and control parameters of the DAM-3R.

| Design Variables | Joint 1 | Joint 2 | Joint 3 |
|------------------|---------|---------|---------|
|                  | Lower Bound | Upper Bound | Lower Bound | Upper Bound | Lower Bound | Upper Bound |
| $\theta_1$ [°]   | $\theta_1^{(l)}$ | $\theta_1^{(u)}$ | $\theta_2^{(l)}$ | $\theta_2^{(u)}$ | $\theta_3^{(l)}$ | $\theta_3^{(u)}$ |
| $s_j$ [mm]       | 37.0 | 77.0 | 37.0 | 77.0 | 37.0 | 77.0 |
| $\dot{s}_j$ [mm/s] | -1.8 | 7.1 | -7.1 | 7.1 | -7.1 | 7.1 |
| $\dot{s}_jb$ [mm/s] | -1.8 | 7.1 | -7.1 | 7.1 | -7.1 | 7.1 |

Table 3. Design and control parameters of the JAM-3R.

| Design Variables | Joint 1 | Joint 2 | Joint 3 |
|------------------|---------|---------|---------|
|                  | Lower Bound | Upper Bound | Lower Bound | Upper Bound | Lower Bound | Upper Bound |
| $\theta_1$ [°]   | $\theta_1^{(l)}$ | $\theta_1^{(u)}$ | $\theta_2^{(l)}$ | $\theta_2^{(u)}$ | $\theta_3^{(l)}$ | $\theta_3^{(u)}$ |
| $\dot{\theta}_1$ [°/s] | -3.3 | 3.3 | -13.2 | 13.2 | -13.2 | 13.2 |

2.2. Numerical Results

In this study, RecurDyn (FunctionBay, Inc., Seongnam, Korea, commercial multibody dynamics software) was used to evaluate the dynamic behavior of the DAM-3R and JAM-3R. To solve (1), the optimization framework was constructed based on the analysis module (RecurDyn) and the optimization module (in-house code written in MATLAB) (Figure 4). As shown in Figure 4, RecurDyn imports the CAD model of a target manipulator to create an initial model for the multibody dynamics analysis. The updated design variables are transferred to RecurDyn to translate and/or rotate the model. Then, RecurDyn performs multibody dynamics analysis and sends the simulation results to the optimization module. The optimization module evaluates the objective and constraint function values (end-effector velocity and thrusting force, respectively) and then calculates the sensitivity of each design variable by using the finite difference method (FDM). If the convergence criterion is satisfied, the optimization process is terminated. Otherwise, the above process continues. Sequential quadratic programming in the MATLAB optimization toolbox [26] was selected as the optimization algorithm. The computing environment for the simulation was as follows: AMD Ryzen 7 5800X 8-Core Processor (CPU), NVIDIA GeForce GTX 1080 Ti (GPU), and DDR4 64GB (RAM). The parameters used in the simulation are summarized in Tables 2 and 3.
Figure 4. Flowchart of the optimization framework using the analysis module (RecurDyn) and the optimization module (MATLAB).

The MPE results at the end-effector position $p_e = (200 \text{ mm}, -150 \text{ mm})$ can be represented in the form of polygons (the so-called allowable polygons in [25]), as shown in Figure 5. The performance results of the DAM-3R and JAM-3R are expressed in solid and dotted lines, respectively. It can be clearly seen that the end-effector velocity of the DAM-3R was greater for all 16 base directions than that of the JAM-3R with no payload. As explained in our previous work [24], this is due to an appropriate change in the gear ratio depending on the position of the sliders.

Figure 5. The maximum end-effector velocity evaluated for the DAM-3R (red solid line) and JAM-3R (magenta dotted line) at the end-effector position $(200 \text{ mm}, -150 \text{ mm})$.

Figure 6 shows the MPE results for both the DAM-3R and JAM-3R at nine target positions equidistantly distributed in the workspace (Figure 6). Although the DAM-3R used a motor with 1.7% less power (Table 1), the maximum end-effector velocity was up to 31.9% higher at $(200 \text{ mm}, -150 \text{ mm})$. 

![Flowchart](image.png)

![Figure 5](image.png)

![Figure 6](image.png)
The MPE results obtained above can be considered as the upper bounds of the maximum end-effector velocity at each end-effector position. Under such velocity constraints, the dynamics of the end-effector position can be expressed by using a first-order model in MPC (see Section 3.2 for more details), which can significantly simplify the MPC formulation for time-optimal trajectory planning without a significant loss of accuracy. We note that it is difficult to precisely evaluate the maximum performance of a redundant manipulator such as a DAM without the aid of the MPE.

3. Time-Optimal Trajectory Planning Based on the MPE and MPC

In an ideal case, the allowable velocity polygon at each end-effector position can be expressed in the form of piecewise linear functions, as shown in Figure 6. However, such polygons should be formulated at all positions in the workspace to conduct MPC. To overcome the above computing issue, it is necessary to approximate the allowable velocity polygon in the entire workspace in a more efficient way.

3.1. Approximation of the MPE Results

For practical purposes, the maximum end-effector velocity polygons at the end-effector position $p_e$ can be approximated as an ellipse, as follows:

$$V(v_e, p_e) = \left( \frac{v_x \cos c + v_y \sin c}{a} \right)^2 + \left( \frac{v_x \sin c - v_y \cos c}{b} \right)^2 - 1 = 0,$$

where $a$ and $b$ are the radii of the major and minor axes of the ellipse, respectively, and $c$ is the angle between the major axis and $v_x$-axis. Note that the parameters $a$, $b$, and $c$ are functions of $p_e$ because the shape of an ellipse varies depending on the end-effector position.
As shown in Figure 5. After these parameters \((a, b, \text{ and } c)\) were determined at the nine target positions by using the least squares method, second-order polynomials for \(a, b, \text{ and } c\) were constructed to cover the entire workspace via the least squares method. Consequently, the MPE results were approximated by a mathematical form for the end-effector velocity, i.e., \(V(\mathbf{v}_e, \mathbf{p}_e) \leq 0\).

3.2. Problem Formulation

Time-optimal trajectory planning can be formulated in the MPC framework [13] as follows:

\[
\begin{align*}
\text{Find} & \quad \Delta T(l) \text{ and } \mathbf{v}_e(l) \text{ for } l = 0, 1, \cdots, N_p - 1 \\
\text{to minimize} & \quad \sum_{l=0}^{N_p-1} \Delta T(l) \\
\text{subject to} & \quad \mathbf{p}_e(l + 1) = \mathbf{p}_e(l) + \Delta T(l) \mathbf{v}_e(l) \\
& \quad V(\mathbf{v}_e(l), \mathbf{p}_e(l)) \leq 0 \\
& \quad \Delta T(l) > 0 \\
& \quad \mathbf{p}_e(N_p) = \mathbf{p}_e^{\text{tar}}
\end{align*}
\]

where \(l\) denotes a time step; \(\Delta T(l)\) denotes the time interval of each time step; \(N_p\) denotes the prediction horizon length; and \(\mathbf{p}_e^{\text{tar}}\) denotes the target end point of the end-effector position.

By solving (6), the optimal trajectories of \(\Delta T\) and \(\mathbf{v}_e\) (i.e., \(\Delta T^*\) and \(\mathbf{v}_e^*\)) can be obtained to approach the target \(\mathbf{p}_e^{\text{tar}}\) in minimum time within the prediction horizon \(N_p\). Only the optimal velocity at the first time step (i.e., \(\mathbf{v}_e^*(0)\)) was used to calculate the reference \(\mathbf{p}_e^*\) for the end-effector position controller, as follows:

\[
\mathbf{p}_e^* \leftarrow \mathbf{p}_e^* + T_s \mathbf{v}_e^*(0),
\]

where \(T_s\) is the control period of the end-effector position controller. In [7], the initial values of \(\mathbf{p}_e^*\) were selected based on the MPE results of the DAM-3R and JAM-3R. \(T_s\) was used instead of the optimal time interval \(\Delta T^*(0)\), which could be greater than \(T_s\), so that the reference \(\mathbf{p}_e^*\) could be tracked by the controller at each control step.

It should again be noted that the time-optimal MPC formulation in (6) requires constraints on the maximum performance of the end-effector (i.e., \(V(\cdot) \leq 0\)). It is practically difficult to solve such an MPC problem without having explicit and accurate performance models of the manipulator. In this study, the mathematical performance models for the DAM-3R and JAM-3R were generated based on the MPE results. We note that, in this study, the dynamics of the end-effector position were simplified by using a first-order model in the MPC \((\mathbf{p}_e(l + 1) = \mathbf{p}_e(l) + \Delta T(l) \mathbf{v}_e(l)\) in (6)) thanks to the MPE-based performance models \((V(\mathbf{v}_e(l), \mathbf{p}_e(l)) \leq 0\) in (6)). This led to a fast and accurate simulation.

3.3. Numerical Results

A point-to-point task (start position \(\mathbf{p}_4\) and end position \(\mathbf{p}_6\) in Figure 6) was set with no predetermined path. To solve (6) for the DAM-3R and JAM-3R, a simulation framework was constructed using MATLAB Simulink for time-optimal trajectory planning and RecurDyn for multibody dynamics analysis (Figure 7). The time-optimal MPC was solved using the Model Predictive Control Toolbox in MATLAB 2020a [27]. The prediction horizon length \(N_p\) was 10, and the control period \(T_s\) was 1 ms.

Figure 8 shows the time-optimal trajectory results for the end-effector reference position (dashed line) and actually controlled position through the controller (solid line). The arrival time of the JAM-3R was 1.57 s, whereas that of the DAM-3R was 1.27 s (19.1% faster). It is interesting to note that the optimal trajectory in the \(y\)-axis for the DAM-3R and JAM-3R slightly descended by 6.2 mm and 6.1 mm, respectively, at the center of the path, \(\mathbf{p}_5 = (150 \text{ mm, } -150 \text{ mm})\). This is because the direction of the time-optimal trajectory was
well aligned with that of the maximum allowable velocities at $p_4$, $p_5$, and $p_6$ (yellow highlighted arrows in Figure 6).

**Figure 7.** Proposed time-optimal trajectory planning using the MPE results and MPC method.

![MPC diagram](image)

**Figure 8.** Time-optimal trajectory results for the end-effector reference position (dotted line) and actually controlled position through the controller (solid line).

Figure 9 shows the detailed simulation results regarding the actuating forces and output motion in the joint space. Figure 9a,c represents the resultant actuating force and slider position, respectively, for the DAM-3R. Figure 9b,d also represents the resultant actuating torque and joint angle, respectively, for the JAM-3R. The total energy consumption of the DAM-3R was reduced by 45.0% compared with that of the JAM-3R (316.5 mJ for the DAM vs. 575.3 mJ for the JAM in Table 4). The proposed MPE-augmented MPC can fully and optimally utilize a high level of redundancy (additional design variables or control degrees of freedom) so as to move in the quickest manner. Particularly, for the DAM-3R, which has higher redundancy than the JAM-3R, the maximum velocity was achieved by less actuating of Joints 1 and 2, which have larger inertia and therefore require larger thrusting torque. This led to faster motion and less energy consumption for the DAM-3R compared with the JAM-3R. Thus, it is clear that, by virtue of conducting MPC augmented by MPE results (explicit and accurate performance models), a redundant manipulator can achieve higher performance in terms of working time and energy consumption for trajectory planning.
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Table 4. Energy consumption in the joint space for the DAM-3R and JAM-3R.

|      | Joint 1 | Joint 2 | Joint 3 | Total  |
|------|---------|---------|---------|--------|
| JAM-3R | 295.6   | 265.9   | 13.9    | 575.3  |
| DAM-3R  | 72.9    | 103.5   | 80.6    | 53.0   |
|        | s1      | s1<sup>b</sup> | s2      | s2<sup>b</sup> | s3      | s3<sup>b</sup> | Total  |

Units are in mJ.

4. Conclusions

In this study, we proposed a novel concept of time-optimal trajectory planning for a redundant DAM based on MPE and MPC. This optimization technique for the DAM-3R and JAM-3R can precisely evaluate the maximum performance (velocity in this study) and, at the same time, determine the optimal control parameters. The MPC for time-optimal trajectory planning was then formulated with the mathematical performance models based on the MPE results obtained. To verify the proposed method, a multibody dynamics module and control analysis module were co-simulated for the DAM-3R and JAM-3R. The simulation results clearly demonstrate that the proposed method can successfully provide the time-optimal trajectory for redundant manipulators. In particular, the DAM-3R, which has higher redundancy than the JAM-3R, outperforms the JAM-3R in terms of working time and energy consumption.

Although we formulated a time-optimal optimization method for MPC, it could be expanded to various performance measures (or tasks), such as minimizing total energy consumption, minimizing the joint torques, and maximizing the end-effector force with limited resources. Therefore, this study could be extended to broad areas of trajectory planning, such as energy minimization, joint torque minimization, and end-effector force maximization with limited resources. Also, in future work, if the proposed method is
implemented in a more powerful computing environment such as a parallel multicore system, the computing time could be significantly shortened.

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