Modal analysis of masonry structures

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Abstract

This paper presents a new numerical tool for evaluating the vibration frequencies and mode shapes of masonry buildings in the presence of cracks. The algorithm has been implemented within the NOSA-ITACA code, which models masonry as a nonlinear elastic material with zero tensile strength. Some case studies are reported, and the differences between linear and nonlinear behaviour highlighted.

Key words: masonry–like materials; modal analysis; numerical methods; nonlinear dynamics.

1. Introduction

The worldwide architectural heritage is in pressing need of maintenance and restoration. Old structures are, in fact, threatened by numerous environmental and anthropogenic actions. In Italy, where seismic actions affect most of the territory, safeguarding ancient masonry buildings and monuments is a crucial issue for institutions and individuals alike. In this context, the use of non–destructive techniques able to evaluate the structural health of old buildings has a vital role to play.

Traditional building monitoring protocols generally involve visual inspections and the measurement of some quantities (displacements, local stresses and crack width, etc.) and their variation in time, while the mechanical properties of the constituent materials are generally assessed locally through destructive or non–destructive tests. More recently ambient vibration tests, first adopted in civil engineering to assess the structural health of bridges and tall buildings, have assumed an increasingly important place in the fields of conservation and restoration. This is for the most part due to the availability
of very sensitive measurement devices and ever more powerful techniques for numerical simulation. These tests allow measuring structures’ dynamic properties, such as natural frequencies, mode shapes and damping ratios and, if coupled with a finite–element model, can provide important information on the mechanical properties of a building’s constituent materials and boundary conditions. Moreover, long–term monitoring protocols allow measuring any variation in the dynamic behaviour of the monitored structures: once the influence of environmental factors such as temperature and humidity has been accounted for, this variation can reveal the presence of any structural damage. Particular attention must be devoted in the monitoring of old masonry buildings, since their make–up and behaviour are profoundly different from modern structures. In fact, masonry is unable to withstand tensile stresses or large compressive stresses. For this reason, old buildings generally present damage scenarios, with cracking induced by the permanent loads or caused by some accidental events occurred during their long history. The dynamic behaviour of these structures should be analyzed taking into account the existing damage.

Numerous examples can be found in the literature of methods for detecting the effects of damage on the dynamic properties of structures. A number of analytical models have been formulated for predicting the effects of cracks on the vibration frequencies of beams and rotors [1]. A review of methods for detecting structural damage by measuring the changes over time of the natural frequencies is presented in [2]. Changes in mode shapes are effective damage indicators [3], [4] and allow better localization of the damage location within the structure. In particular, modal curvatures seem to be very sensitive to local damage [5], though their measurement requires the use of a large number of sensors. Wide variations in modal parameters can be observed when damage occurs in zones with high modal curvatures. With regard to the monitoring of old masonry structures, a number of papers have recently been devoted to the analysis of data from long–term monitoring protocols; these have highlighted changes in the modal characteristics due to environmental factors [6], [7], [8], or, in some cases, structural damage [9], [10], [11], [12].

Much effort has also been devoted to numerically simulating the effects of damage on the vibrations of the structures. In [13] and [14] a damage identification procedure is presented to detect and localize damage in beam structures. A beam is discretized into finite–elements and the damage to it simulated by reducing the stiffness matrix of some selected elements via
a damage index. The equations obtained enable obtaining the beam’s frequencies and mode shapes when the damage indices are fixed or, vice versa, calculating the damage indices when frequencies and mode shapes are measured on real beams. Model updating procedures are employed in [15] in order to fit experimental data with finite-element models; the parameters to be updated are the Young’s moduli of the constituent materials. With regard to masonry buildings, a common approach consists of simulating the existing damage actually observed on the structure, by reducing the stiffness of those elements of the finite element model belonging to the cracked or damaged parts [16], [10]. In [17] and [18] the dynamic properties of some masonry structures at different damage levels are investigated via the discrete element method.

The work presented in [19] formulates an explicit expression linking the fundamental frequency of a masonry-like beam to its maximum transverse displacement. The masonry-like constitutive equation models masonry as a non-linear elastic material with different strengths under tension and compression [20], [21]. This constitutive equation has been implemented in the finite-element code NOSA for the static analysis of masonry bodies [22]. The NOSA code and its updated version NOSA–ITACA 1.0 [23], have been applied to the static and dynamic analysis of many important monuments in Italy. NOSA–ITACA 1.0 is freely downloadable at [24]. The non-linear equilibrium equations of masonry-like bodies are solved via the Newton–Raphson method, by taking into account an explicit expression of the derivative of the stress with respect to the strain [22], which allows calculation of the system’s tangent stiffness matrix.

This paper presents a numerical procedure, implemented in a newly updated version 1.2 of the NOSA–ITACA code, which can evaluate the natural frequencies and mode shapes of masonry buildings in the presence of cracks. Once the initial loads and boundary conditions have been applied to the finite-element model, the resulting non-linear equilibrium problem is solved through an iterative scheme. Then, a modal analysis is performed, by using the tangent stiffness matrix calculated by the code in the last iteration before convergence is reached. The incremental approach used by NOSA–ITACA 1.2 allows the user to perform the modal analysis at the initial step, before application of any load to the structure, and then for different loading steps. Following the prestressed modal analysis applied to problems with geometric nonlinearity [25] and [26], the proposed procedure allows the user to automatically take into account the stress distribution on the system’s stiffness
matrix, thereby evaluating the effects of the presence of cracked and crushed material on the structure’s dynamic properties.

Section 2 briefly recalls the constitutive equation of masonry–like materials. The explicit expressions for the stress function and its derivative, obtained in [22], are summarized, and the numerical procedure for solving static and dynamic problems of masonry structures, implemented in the NOSA–ITACA 1.0 code, are sketched out. Finally, the modal analysis of linear elastic structures is addressed and the procedures implemented in the NOSA–ITACA 1.0 code to solve the constrained generalized eigenvalue problem resulting from application of the finite-element method is outlined [27].

Section 3 presents the new procedure implemented in the NOSA–ITACA 1.2 code. The code allows for calculating the natural frequencies and mode shapes of a masonry body subjected to prescribed boundary conditions and loads. The new procedure takes into account the non–linearity of the constitutive equation of masonry–like materials as well as the presence of cracks due to the applied loads. In general, the results of the modal analysis conducted after application of the assigned loads differ from those obtained via the standard modal analysis and can be used to assess the presence of damaged zones in the structure. Section 4 presents some applications of the method. The first two cases deal with a masonry beam and an arch on piers. Both structures are subjected to incremental loads and analyzed with the NOSA–ITACA code. The aim is to compare the natural frequencies and mode shapes in the linear elastic case with those calculated in the presence of the crack distribution induced by the increasing loads. The third case deals with an actual structure, the bell tower of San Frediano in Lucca. The tower has been instrumented with four high–sensitivity triaxial seismometric stations [28], and its first five natural frequencies have been determined via OMA techniques [29]. Then the NOSA–ITACA code, together with model updating techniques, has been employed in order to fit the experimental results. The model updating in the linear elastic case is compared to the model updating applied to the tower subjected to its own weight by taking into account the crack distribution induced by the loads.

2. The masonry–like constitutive equation and the NOSA-ITACA 1.0 code

Let Lin be the set of all second-order tensors with the scalar product $A \cdot B = \text{tr}(A^T B)$ for any $A, B \in \text{Lin}$, with $A^T$ the transpose of $A$. For
Sym the subspace of symmetric tensors, $Sym^-$ and $Sym^+$ are the sets of all negative-semidefinite and positive-semidefinite elements of $Sym$. Given the symmetric tensors $A$ and $B$, we denote by $A \otimes B$ the fourth-order tensor defined by $A \otimes B[H] = (B \cdot H)A$ for $H \in \text{Lin}$ and by $I_{sym}$ the fourth-order identity tensor on $Sym$. For $a$ and $b$ vectors, the dyad $a \otimes b$ is defined by $a \otimes b h = (b \cdot h)a$, for any vector $h$, with $\cdot$ the scalar product in the space of vectors.

Let $C$ be the isotropic fourth-order tensor of elastic constants

$$C = 2\mu I_{sym} + \lambda I \otimes I,$$  \hspace{1cm} (1)

where $I \in Sym$ is the identity tensor and $\mu$ and $\lambda$ are the Lamé moduli of the material satisfying the conditions

$$\mu > 0, \quad \lambda \geq 0.$$  \hspace{1cm} (2)

$C$ is symmetric,

$$A \cdot C[B] = B \cdot C[A], \quad \text{for all } A, B \in Sym,$$  \hspace{1cm} (3)

and in view of (2) is positive-definite on $Sym$,

$$A \cdot C[A] > 0 \quad \text{for all } A \in Sym, \quad A \neq 0.$$  \hspace{1cm} (4)

A masonry-like material is a nonlinear elastic material [22] characterized by the fact that, for $E \in Sym$, there exists a unique triplet $(T, E^e, E^f)$ of elements of $Sym$ such that

$$E = E^e + E^f,$$  \hspace{1cm} (5)

$$T = C[E^e],$$  \hspace{1cm} (6)

$$T \in Sym^-, \quad E^f \in Sym^+,$$  \hspace{1cm} (7)

$$T \cdot E^f = 0.$$  \hspace{1cm} (8)

$T$ is the Cauchy stress corresponding to strain $E$. Tensors $E^e$ and $E^f$ are respectively the elastic and inelastic part of $E$; $E^f$ is also called the fracture strain. The stress function $T : Sym \rightarrow Sym$ is given by

$$T(E) = T, \quad \text{for any } E \in Sym,$$  \hspace{1cm} (9)
with $\mathbf{T}$ satisfying (5)–(8). The explicit expression for the stress function $\mathbf{T}$, calculated in [22], is recalled in the following.

For $\mathbf{E} \in \text{Sym}$, let $e_1 \leq e_2 \leq e_3$ be its ordered eigenvalues and $\mathbf{q}_1, \mathbf{q}_2, \mathbf{q}_3$ the corresponding eigenvectors. We introduce the orthonormal basis of $\text{Sym}$ (with respect to the scalar product $\cdot$)

$$ O_{11} = \mathbf{q}_1 \otimes \mathbf{q}_1, \quad O_{22} = \mathbf{q}_2 \otimes \mathbf{q}_2, \quad O_{33} = \mathbf{q}_3 \otimes \mathbf{q}_3, $$

$$ O_{12} = \frac{1}{\sqrt{2}}(\mathbf{q}_1 \otimes \mathbf{q}_2 + \mathbf{q}_2 \otimes \mathbf{q}_1), \quad O_{13} = \frac{1}{\sqrt{2}}(\mathbf{q}_1 \otimes \mathbf{q}_3 + \mathbf{q}_3 \otimes \mathbf{q}_1), $$

$$ O_{23} = \frac{1}{\sqrt{2}}(\mathbf{q}_2 \otimes \mathbf{q}_3 + \mathbf{q}_3 \otimes \mathbf{q}_2). $$

Given $\mathbf{E}$, the corresponding stress $\mathbf{T}$ satisfying the constitutive equation of masonry-like materials (5)–(8) is given by

if $\mathbf{E} \in V_0$ then $\mathbf{T} = \mathbf{0}$,

$$ (11) $$

if $\mathbf{E} \in V_1$ then $\mathbf{T} = \mathbf{E} e_1 O_{11}$,

$$ (12) $$

if $\mathbf{E} \in V_2$ then $\mathbf{T} = \frac{2\mu}{2+\alpha} \{[2(1+\alpha)e_1 + \alpha e_2]O_{11} + [\alpha e_1 + 2(1+\alpha)e_2]O_{22}\}$,

$$ (13) $$

if $\mathbf{E} \in V_3$ then $\mathbf{T} = \mathbb{C}[\mathbf{E}]$,

$$ (14) $$

where the sets $V_k$ are

$$ V_0 = \{ \mathbf{E} \in \text{Sym} : e_1 \geq 0 \}, $$

$$ (15) $$

$$ V_1 = \{ \mathbf{E} \in \text{Sym} : e_1 \leq 0, \ \alpha e_1 + 2(1+\alpha) e_2 \geq 0 \}, $$

$$ (16) $$

$$ V_2 = \{ \mathbf{E} \in \text{Sym} : \alpha e_1 + 2(1+\alpha)e_2 \leq 0, \ 2e_3 + \alpha tr \mathbf{E} \geq 0 \}, $$

$$ (17) $$

$$ V_3 = \{ \mathbf{E} \in \text{Sym} : 2e_3 + \alpha tr \mathbf{E} \leq 0 \}, $$

$$ (18) $$
with \( \alpha = \lambda / \mu \) and \( E = \mu(2\mu + 3\lambda)/(\mu + \lambda) \) the Young’s modulus.

As for the fracture strain,

\[
\text{if } \mathbf{E} \in V_0 \text{ then } \mathbf{E}^f = \mathbf{E}, \quad (19)
\]

\[
\text{if } \mathbf{E} \in V_1 \text{ then } \mathbf{E}^f = \left( e_2 + \frac{\alpha}{2(1 + \alpha)} e_1 \right) \mathbf{O}_{22} + \left( e_3 + \frac{\alpha}{2(1 + \alpha)} e_1 \right) \mathbf{O}_{33}, \quad (20)
\]

\[
\text{if } \mathbf{E} \in V_2 \text{ then } \mathbf{E}^f = \left[ e_3 + \frac{\alpha}{2 + \alpha} (e_1 + e_2) \right] \mathbf{O}_{33}, \quad (21)
\]

\[
\text{if } \mathbf{E} \in V_3 \text{ then } \mathbf{E}^f = 0. \quad (22)
\]

For \( W_k \) the interior of \( V_k \), function \( T \) turns out to be differentiable in

\[ W = \bigcup_{i=0}^{3} W_i \] \text{[22],[30]. Thus, for } \mathbf{E} \in W \text{ denoting by } D_E T(\mathbf{E}) \text{ the derivative of } T(\mathbf{E}) \text{ with respect to } \mathbf{E}, \text{ we have}

\[
T(\mathbf{E} + \mathbf{H}) = T(\mathbf{E}) + D_E T(\mathbf{E})[\mathbf{H}] + o(\mathbf{H}), \quad \mathbf{H} \in Sym, \quad \mathbf{H} \to 0. \quad (23)
\]

\( D_E T(\mathbf{E}) \) is a symmetric fourth–order tensor from \( Sym \) into itself and has the following expressions in the regions \( W_i \) [22].

If \( \mathbf{E} \in W_0 \) then \( D_E T(\mathbf{E}) = \mathcal{O}, \quad (24) \)

where \( \mathcal{O} \) is the null fourth-order tensor,

\[
\text{if } \mathbf{E} \in W_1 \text{ then } D_E T(\mathbf{E}) = E \begin{pmatrix} O_{11} \otimes O_{11} - \frac{e_1}{e_2 - e_1} O_{12} \otimes O_{12} \\ - \frac{e_1}{e_3 - e_1} O_{13} \otimes O_{13} \end{pmatrix}, \quad (25)
\]

\[
\text{if } \mathbf{E} \in W_2 \text{ then } D_E T(\mathbf{E}) = 2\mu O_{12} \otimes O_{12} - \frac{2\mu}{2 + \alpha} \frac{2(1 + \alpha)e_1 + \alpha e_2}{e_3 - e_1} O_{13} \otimes O_{13}
\]
\[
\begin{align*}
&\frac{2\mu}{2 + \alpha} \left( \alpha e_1 + 2(1 + \alpha)e_2 \right)O_{23} \otimes O_{23} \\
&\frac{2\mu(2 + 3\alpha)}{2 + \alpha} \left( \frac{O_{11} + O_{22}}{\sqrt{2}} \right) \otimes \left( \frac{O_{11} + O_{22}}{\sqrt{2}} \right) \\
&+ 2\mu \left( \frac{O_{11} - O_{22}}{\sqrt{2}} \right) \otimes \left( \frac{O_{11} - O_{22}}{\sqrt{2}} \right),
\end{align*}
\]

(26)

if \( \mathbf{E} \in W_3 \) then \( D_E \mathbf{T}(\mathbf{E}) = \mathbf{C} \).

(27)

In order to study real problems, the equilibrium problem of masonry structures can be solved via the finite element method. To this end, suitable numerical techniques have been developed \[22\] based on the Newton-Raphson method for solving the nonlinear system obtained by discretising the structure into finite elements. Their application is based on the explicit expression for the derivative of the stress with respect to the strain, which is needed in order to calculate the tangent stiffness matrix. The numerical method studied and the constitutive equation of masonry-like materials described above have therefore been implemented into the finite element code NOSA \[22\].

As far as the numerical solution of dynamic problems is concerned, direct integration of the equations of motion is required \[31\]. In fact, due to the nonlinearity of the adopted constitutive equation, the mode-superposition method is meaningless. With the aim of solving such problems, we have instead implemented the Newmark method \[32\] within NOSA in order to perform the integration with respect to time of the system of ordinary differential equations obtained by discretising the structure into finite elements. Moreover, the Newton-Raphson scheme, needed to solve the nonlinear algebraic system obtained at each time step, has been adapted to the dynamic case. The NOSA code, devoted to static and dynamic analyses of ancient masonry constructions, enables determining the stress state and the presence of any cracking. Moreover, the effects of thermal variations and the effectiveness of various strengthening interventions, such as the application of tie rods and retaining structures, can be evaluated, and those with the least environmental and visual impact identified.

The code has been successfully applied to the analysis of arches and vaults \[33\], and in a number of studies of buildings of historical and architectural interest \[22\], \[34\].
Within the framework of the project "Tools for modelling and assessing the structural behaviour of ancient constructions" [24] funded by the Region of Tuscany (2011-2013), the NOSA code has been integrated in the open source graphic platform SALOME [35], used both to define the geometry of the structure under examination and to visualise the results of the structural analysis. The result of this integration is the freeware code NOSA-ITACA 1.0 [23], [24]. NOSA-ITACA 1.0 has been applied to several studies commissioned by both private and public bodies of the static and dynamic behaviour of historical masonry buildings [36]. In many cases, such studies have also provided important information on the structure’s seismic vulnerability, which can be assessed with respect to current Italian and European regulations [37].

An efficient implementation of the numerical methods for constrained eigenvalue problems for modal analysis of linear elastic structures has been embedded in NOSA-ITACA 1.0 [27]. Such implementation takes into account both the sparsity of the matrices and the features of master-slave constraints (tying or multipoint constraints). It is based on the open-source packages SPARSKIT [38], for managing matrices in sparse format (storage, matrix-vector products), and ARPACK [39], which implements a method based on Lanczos factorization combined with spectral techniques that improve convergence. In particular in [27] the authors have implemented a procedure for solving the constrained generalized eigenvalue problem

\[ K \phi = \omega^2 M \phi, \]
\[ T \phi = 0, \]

with \( T \in \mathbb{R}^{m \times n} \) and \( m \ll n \). Equation (28) is derived from the equation

\[ M \ddot{u} + Ku = 0, \]

governing the free vibrations of a linear elastic structure discretized into finite elements. In equation (30) \( u \) is the displacement vector, which belongs to \( \mathbb{R}^n \) and depends on time \( t \), \( \ddot{u} \) is the second-derivative of \( u \) with respect to \( t \), and \( K \) and \( M \in \mathbb{R}^{n \times n} \) are the stiffness and mass matrices of the finite-element assemblage. \( K \) is symmetric and positive-semidefinite, \( M \) is symmetric and positive-definite, and both are banded with bandwidth depending on the numbering of the finite-element nodal points. Displacements \( u_i \) are also called degrees of freedom; the integer \( n \) is the total number of degrees of freedom.
of the system and is generally very large, since it depends on the level of
discretization of the problem. By assuming that
\[ u = \phi \sin(\omega t), \]  
with \( \phi \) a vector of \( \mathbb{R}^n \) and \( \omega \) a real scalar, and applying the modal super-
position \[32\], equation \[30\] is transformed into the generalized eigenvalue
problem \[28\]. Condition \[29\] expresses the fixed constraints and the master-
slave relations assigned to displacement \( u \), written in terms of vector \( \phi \). The
restriction of the matrix \( K \) to the null subspace of \( \mathbb{R}^n \) defined by \[29\] is
positive-definite.

Although the constitutive equation \[5\]–\[8\] adopted for masonry is nonlin-
ear, the modal analysis, which is based on the assumption that the materials
constituting the construction are linear elastic, is widely used in applications
and furnishes important qualitative information on the dynamic behavior of
masonry structures and thereby allows for assessing their seismic vulnera-
ibility in light of Italian and European regulations \[37\]. On the other hand,
traditional modal analysis does not take into account the influence that both
the nonlinear behaviour of the masonry material and the presence of cracked
regions can have on the natural frequencies of masonry structures. The ap-
proach described in the next section and implemented in NOSA–ITACA 1.2
allows for calculating the natural frequencies and modal shapes of a masonry
body subjected to given system of loads and exhibiting a crack distribution
due to these loads.

3. A new procedure for the modal analysis of masonry–like struc-
tures

Let us consider a body\[1\] \( \mathcal{B} \) whose boundary \( \partial \mathcal{B} \) is composed of two com-
plementary and disjoined portions \( \partial \mathcal{B}_1 \) and \( \partial \mathcal{B}_2 \). \( \mathcal{B} \) is made of a masonry-like
material with constitutive equation \[5\]–\[8\] and mass density \( \rho \). Given the
loads \( (\mathbf{b}, \mathbf{s}_0) \), with the body force \( \mathbf{b} \) defined over \( \mathcal{B} \) and the surface force
\( \mathbf{s}_0 \) defined over \( \partial \mathcal{B}_2 \), let \( (\tilde{\mathbf{u}}, \tilde{\mathbf{E}}, \tilde{T}) \) a triple consisting of one vector and two
tensor fields defined over \( \mathcal{B} \) satisfying the following equilibrium problem \[22\]
\[ \text{div} \tilde{T} + \mathbf{b} = 0 \quad \text{on } \mathcal{B}, \quad (32) \]

\[1\] A body is a regular region of the three-dimensional Euclidean space having boundary
\( \partial \mathcal{B} \), with outward unit normal \( \mathbf{n} \) \[40\].
\[ \widetilde{\mathbf{E}} = \frac{\nabla \mathbf{u} + \nabla \mathbf{u}^T}{2} \quad \text{on } \mathcal{B}, \quad (33) \]

\[ \widetilde{\mathbf{T}} = \mathbf{T}(\widetilde{\mathbf{E}}) \quad \text{on } \mathcal{B}, \quad (34) \]

\[ \mathbf{u} = 0 \quad \text{on } \partial \mathcal{B}_1, \quad (35) \]

\[ \widetilde{\mathbf{T}}_{\mathbf{n}} = s_0 \quad \text{on } \partial \mathcal{B}_2. \quad (36) \]

The equilibrium problem (32)-(36) is dealt with in [22], where the uniqueness of its solution in terms of stress is proved and the iterative procedure implemented in the NOSA code to calculate a numerical solution is described in detail.

Let \((0, t_0)\) denote a fixed interval of time; a motion of \(\mathcal{B}\) is a vector field \(\mathbf{u}\) defined on \(\mathcal{B} \times (0, t_0)\). The vector \(\mathbf{u}(\mathbf{x}, t)\) is the displacement of \(\mathbf{x}\) at time \(t\) and \(\frac{\partial^2 \mathbf{u}}{\partial t^2}\) its acceleration. Let us consider small motions \(\mathbf{u}\) such that their gradient \(\nabla \mathbf{u}\) is small, and \(\mathbf{u} = 0\) on \(\partial \mathcal{B}_1\).

For the displacement field \(\mathbf{t} + \mathbf{u}\), the strain field \(\widetilde{\mathbf{E}} + \mathbf{E}\), with \(\mathbf{E} = (\nabla \mathbf{u} + \nabla \mathbf{u}^T)/2\) and the stress field \(\mathbf{T}(\widetilde{\mathbf{E}} + \mathbf{E})\), the equation of motion reads

\[ \text{div}(\mathbf{T}(\widetilde{\mathbf{E}} + \mathbf{E})) + b = \rho \frac{\partial^2 (\mathbf{t} + \mathbf{u})}{\partial t^2} \quad \text{on } \mathcal{B}. \quad (37) \]

From (37), in view of (23) and (34), bearing in mind the equilibrium equation (32) and neglecting terms of order \(o(\nabla \mathbf{u})\), we get the linearized equation of motion

\[ \text{div}(D_E \mathbf{T}(\widetilde{\mathbf{E}}) \left[ \frac{\nabla \mathbf{u} + \nabla \mathbf{u}^T}{2} \right]) = \rho \frac{\partial^2 \mathbf{u}}{\partial t^2} \quad \text{on } \mathcal{B}, \quad (38) \]

where the fourth-order tensor \(D_E \mathbf{T}(\widetilde{\mathbf{E}})\) depends on \(\mathbf{x} \in \mathcal{B}\).

Equation (38), governing the undamped free vibrations of \(\mathcal{B}\) about the equilibrium state \((\mathbf{t}, \mathbf{E}, \mathbf{T})\), is linear and via the application of the finite element method can be transformed into the equation

\[ M \ddot{\mathbf{u}} + \dot{K} \mathbf{u} = 0, \quad (39) \]

which is analogous to (30), though here the elastic stiffness matrix \(K\), calculated using the elastic tensor \(\mathbf{C}\), has been replaced by the damaged stiffness
matrix \( \tilde{K} \), calculated using \( D_E \tilde{T}(\tilde{\mathbf{E}}) \), which takes into account the presence of cracks in body \( B \).

Thus, a new numerical method for the modal analysis of masonry constructions has been implemented in an updated version 1.2 of the NOSA-ITACA code. Given the structure under examination, discretized into finite elements, and given the mechanical properties of the masonry-like material constituting the structure, together with the kinematic constraints and loads acting on it, the procedure consists of the following steps.

Step 1. A preliminary modal analysis is conducted by assuming the structure’s constituent material to be linear elastic. The generalized eigenvalue problem (28)-(29) is then solved and the natural frequencies \( f_i = \omega_i / 2\pi \) and mode shapes \( \phi_i \) calculated.

Step 2. The equilibrium problem (32)-(36) is solved and its solution, \( (\tilde{\mathbf{u}}, \tilde{\mathbf{E}}, \tilde{\mathbf{T}}) \), calculated. The fourth-order tensor \( D_E \tilde{T}(\tilde{\mathbf{E}}) \) needed to calculate the damaged stiffness matrix \( \tilde{K} \) to be used in the next step is evaluated by using (24)-(27).

Step 3. The generalized eigenvalue problem (28)-(29), with matrix \( \tilde{K} \) in place of the elastic stiffness matrix \( K \), is solved and the natural frequencies \( f_i = \omega_i / 2\pi \) and mode shapes \( \phi_i \) of the damaged structure calculated.

4. Case studies

4.1. The masonry beam

Let us consider the rectilinear beam illustrated in Figure 1 with length \( l = 6 \) m, rectangular cross section of \( 0.4 \) m \( \times \) 1 m, subjected to the concentrated vertical load \( P = 10^6 \) N and a uniform lateral load. The beam, simply supported at its ends and forced to move in the \( y-z \) plane, is discretized with 60 beam elements [23]. The goal of the finite-element analysis, conducted with the NOSA–ITACA 1.2 code is to compare the natural frequencies and mode shapes of the beam in the linear elastic case with those in the presence of the damage induced by the increasing lateral load. We conducted a preliminary modal analysis by assuming the beam to be made of a linear elastic material with Young’s modulus \( E = 3 \cdot 10^8 \) Pa, Poisson’s ratio \( \nu = 0.2 \) and mass density \( \rho = 1800 \) kg/m³, for which we calculated the corresponding natural frequencies \( f_i^l \) and mode shapes \( \phi_i^l \). Then, by following the procedure outlined in Section 3 we incremented the lateral load and, at each increment, calculated the frequencies \( f_i \) and modes \( \phi_i \). The
value of the lateral load applied to the beam was increased through seven increments from $9 \times 10^3$ N (which was able to induce the first crack in the beam’s midsection) to $15.75 \times 10^3$ N. Table 1 reports the value of the lateral load and the corresponding natural frequencies. The first row summarizes the results of the linear elastic modal analysis. As outlined in Table 1, as the lateral load increases, the beam’s fundamental frequency decreases from the linear elastic value of 6.47 Hz to 3.71 Hz, about one half its initial value. The other frequencies fall by up to thirty per cent. The frequencies ratio $f_i/f_1$ is reported in Figure 2. The Figure clearly shows that when the lateral load is applied, the fundamental frequency value falls faster than the other frequency values. This is due to the chosen lateral load distribution, which induces a deformation in the beam similar to the first mode shape. Table 1 also reports the values of the effective modal masses [41] calculated for the first six mode shapes. The modes are also shown in Figure 1, where the black line stands for the modes $\phi_i^l$ calculated for the linear elastic case, while the cyan line represents the modes $\phi_i$ calculated at the last increment of the analysis. It is worth noting that the first, second, fifth and sixth modes substantially maintain their shape, while some changes occur between the third and fourth mode: their frequency values are very similar, but in the linear elastic analysis the third mode has an axial direction, while the fourth mode has transverse direction. The nonlinear case shows that the masses of the third mode pass from the axial to the transverse direction, while for the fourth mode diverts from transverse to axial [42]. A small part of the total mass in the axial direction migrates towards the higher modes and is not shown in the Table.

In order to compare the mode shapes $\phi_i^l$ and $\phi_j$, we introduce the quantity

$$\text{MAC–M}(\phi_i^l, \phi_j) = \frac{|\phi_i^l \cdot M \phi_j|}{\sqrt{\phi_i^l \cdot M \phi_i^l \phi_j \cdot M \phi_j}},$$

which measures the correlation between the $i$–th linear elastic mode shape and the $j$ – th mode shape of the damaged structure. More precisely, if the MAC–M value is close to unity, then the vectors $\phi_i^l$ and $\phi_j$ are nearly parallel with respect to the scalar product in $\mathbb{R}^n$ induced by matrix $M$. Table 2 reports the quantities MAC–M($\phi_i^l, \phi_j$) calculated for $i, j = 1, 2, ..., 6$, with $\phi_j$ the damaged mode shape calculated at the last load increment, corresponding to the lateral load $15.75 \times 10^3$ N. As the lateral load increases, the quantities MAC–M($\phi_3^l, \phi_3$) and MAC–M($\phi_3^l, \phi_4$) drawn in Figure 3 in
blue and green lines, respectively, pass from 1 to 0.4 and from 0 to 0.88 (see Table 2), thus showing that mode shape $\phi_3$, initially coincident with $\phi_l^3$, tends to become $M$-orthogonal to it and $\phi_4$ initially $M$-orthogonal to $\phi_l^3$ tends to become $M$-parallel to it. Finally, Figure 4 shows the distributions of the fractured areas in the beam at the last load increment, together with the ratio between the cracked area $A_f$ and total area $A$ calculated in every section of the beam.

4.2. The masonry arch on piers

Let us consider the masonry arch on piers shown in Figure 5. The arch span is 6 m and its cross section measures 0.25 m $\times$ 1 m. The arch has a circular shape, with a mean radius of 3.75 m, and rests on two lateral piers with rectangular cross section of 0.8 m $\times$ 1 m and height 4 m. The structure is reinforced by means of two steel tie rods, with rectangular cross sections of 0.03 m $\times$ 0.03 m, fixed at the pier–arch nodes. An offset of 0.24 m has been considered between the axis of the arch and those of the piers. The structure, modelled by means of 1500 thick shell elements, [23], [36], is clamped at the piers’ base and forced to move in the $x$ – $z$ plane. Figure 5 also shows the elements’ local axes in red. Beam elements [23] have been used to model the tie rods.
Figure 2: The ratio $f_i/f_{i,1}$ for the first six natural frequencies of the beam vs. the lateral load.
Figure 3: The quantities $\text{MAC}-M(\phi_3^l, \phi_3)$ (blue line) and $\text{MAC}-M(\phi_3^l, \phi_4)$ (green line) vs. the lateral load.

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Figure 4: The ratio between the cracked area $A_f$ and total area $A$ along the beam at the last load increment. (lengths in cm).
Just as for the beam described in the previous subsection, we conducted a preliminary modal analysis by assuming the masonry structure to be made of a linear elastic material with Young’s modulus \( E = 3 \cdot 10^9 \) Pa, Poisson’s ratio \( \nu = 0.2 \) and mass density \( \rho = 1800 \) kg/m\(^3\), while for the tie rods we have assumed \( E = 2.1 \cdot 10^{11} \) Pa, Poisson’s ratio \( \nu = 0.3 \) and mass density \( \rho = 7850 \) kg/m\(^3\). The modal analysis allowed us to calculate the structure’s natural frequencies \( f_i \) and mode shapes \( \phi_i \). The loads have been applied incrementally, first the self-weight of the structure alone, and then a concentrated vertical load \( P \), whose value was increased through four increments from 150.61 \cdot 10^3 \) N to 154.61 \cdot 10^3 \) N. Seven analyses were performed, each time moving the load \( P \) to different positions along the arch span (see Figure 5). The natural frequencies \( f_i \) and mode shapes \( \phi_i \) of the damaged structure have thus been calculated for each analysis at each load increment.

Table 3 shows the structure’s first five natural frequencies vs. load increment for the concentrated load \( P \) in \( X_4 \), at about one fourth of the arch span. As the load \( P \) increases, the fundamental frequency value of the structure decreases from 6.54 Hz to 2.54 Hz, the others fall by up to fifty percent of their linear elastic value. This behaviour is also evident in Table 5 for \( P \) in the middle of the arch. Figures 6 to 10 show the first five frequencies of the structure vs. the load application point along the arch span, for the four load \( P \) values used during the analyses (\( P_1 = 150605 \) N, \( P_2 = 152605 \) N, \( P_3 = 153605 \) N, \( P_4 = 154605 \) N). These Figures show that all the structure’s frequencies reach their minimum values when load \( P \) is at the arch’s quarter points (between positions \( X_4 \) and \( X_5 \)). The minimum decrease with respect to the linear elastic case is exhibited by frequencies \( f_1, f_2, f_5 \) for \( P \) at \( X_7 \), while for frequencies \( f_3, f_4 \) it appears between positions \( X_5 \) and \( X_6 \). For a given load position \( X_i \), all frequency values decrease their values as the load increases.

The structure’s mode shapes are shown in Figures 11 and 12 for \( P \) at \( X_4 \) and \( X_7 \), respectively. The corresponding effective modal masses are shown vs. load increments in Table 3 and 5 respectively. The masses excited by the first five modes in the linear elastic case are about 80% of the total mass in the \( x \) direction and 10% in the \( z \) direction. The \( x \) direction, prevalent in modes 1, 2 and 5, is related to oscillations involving movement of the entire structure, while the \( z \) direction, prevalent in modes 3 and 4, relates to modes mainly involving local oscillations of the arch and piers. The values of the total excited masses remain stable during the nonlinear analyses, but their distribution among the modes tends to change: in particular, the masses
excited along $x$ tend to migrate from the first mode shape to the second, while the masses along $z$ pass from mode shape 3 to mode shape 4. This phenomenon is particularly evident when load $P$ is at $X_4$. Accordingly, the MAC–$M$ matrix plotted in Table 4 for $P$ at $X_4$ exhibits low values of the diagonal terms, while the off–diagonal terms reveal a high correlation at the end of the analysis between the first and second and third and fourth modes. Table 6 for $P$ at $X_7$, shows that the first, second and fifth modes substantially maintain their shape, while the third and fourth do not.

For each element we now introduce the Frobenius norm of the difference between the elemental stiffness matrices $\bar{K}_e$ and $K_e$, whose assemblages form matrices $\bar{K}$ and $K$,

$$d_e = \|\bar{K}_e - K_e\|. \quad (41)$$

Figures 13 and 14 show the behaviour of $d_e$, $e = 1, ..., 1500$ and highlight the elements in which the distance between the damaged and the linear elastic stiffness matrices attains the highest values. These elements substantially coincide with those characterized by the highest values of the fracture strain, as shown in Figures 15 and 16, where the strains $E_{\text{f}22} = g_2 \cdot \mathbf{E}' \cdot g_2$ at the intrados and extrados of the structure are shown for $P$ at $X_4$ and $X_7$, respectively.
Figure 5: Geometry of the structure made of up the arch on piers and the different positions along the arch span at which the concentrated load $P$ is applied (length in cm). The local axes of the finite-elements are drawn in red.
4.3. The bell tower of the Church of San Frediano in Lucca

The Basilica of San Frediano (Figure 17), dating back to the 11th century, is one of the most fascinating monuments in Lucca, much of its fascination being due to the marvellous mosaics that adorn its façade. In June 2015 the church’s 52 m high bell tower was fitted with four high-sensitivity triaxial seismometric stations, made available by the Arezzo Earthquake Observatory (Osservatorio Sismologico di Arezzo) and left active on the tower for three days (Figure 18). Data from the instruments, analyzed via OMA techniques [29], allowed us to determine the tower’s first five natural frequencies $f_i^{\text{exp}}$ and corresponding damping ratios [28]. Information on the mode shapes was also extracted, though the arrangement of the sensors, aligned along the vertical, did not allow for reliable identification of the torsional modes.

In the following the NOSA–ITACA code, together with model updating techniques, is employed in order to fit the experimental results in the linear elastic and the masonry–like (nonlinear) case. The finite–element model of the tower consists of 18495 thick shell elements [36]. The four steel tie rods fitted to the masonry vault under the bell chamber (between section 3 and section 4 in Figure 18) are modelled with beam elements [23], which have also been used to model the wooden trusses and rafters constituting the pavilion roof covering the tower. The mechanical properties of the tie rods and wooden elements are, respectively, $E = 2.1 \cdot 10^5$ MPa, $\nu = 0.3$, $\rho = 7850$ kg/m$^3$ and $E = 9500$ MPa, $\nu = 0.4$, $\rho = 800$ kg/m$^3$. With regard to the materials constituting the masonry tower, no experimental information is available to date. Visual inspection reveals that the external layers of the walls are made up of regular stone blocks at the base, while quite homogeneous brick masonry forms the upper part, except for the central part of the walls, where the masonry between the windows is made up of stone blocks. A first attempt to fit the experimental results through model updating procedures, conducted in [28], showed that good results can be achieved by assuming homogeneous values for the mechanical properties of all the materials making up the tower. Herein we proceed in the same way by assuming Young’s modulus $E$ and the mass density $\rho$ of masonry as parameters to be updated. In particular, for different values of $E$ and $\rho$ satisfying

$$3 \cdot 10^9 \text{ Pa} \leq E \leq 7 \cdot 10^9 \text{ Pa},$$
$$1800 \text{ kg/m}^3 \leq \rho \leq 2200 \text{ kg/m}^3,$$  \hspace{1cm} (42)
Figure 6: The structure’s frequency $f_1$ vs. the position of the concentrated load $P$ for different load increments.
Figure 7: The structure’s frequency $f_2$ vs. the position of the concentrated load $P$ for different load increments.
Figure 8: The structure’s frequency $f_3$ vs. the position of the concentrated load $P$ for different load increments.
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Figure 16: Fracture strain $E_{22}^f$ at the intrados (left) and the extrados (right) of the structure for load P at position $X_7$. 
Figure 17: The Basilica of San Frediano in Lucca (Italy). On the left, the San Frediano bell tower.
the first five natural frequencies of the tower are calculated for the linear elastic and nonlinear case, after having applied the structure’s self-weight alone. The optimal values of $E$ and $\rho$, are determined by minimizing the functions

$$e^l(E, \rho) = \sum_{i=1}^{5} (f^l_i(E, \rho) - f^{exp}_i)^2,$$

(43)

$$e(E, \rho) = \sum_{i=1}^{5} (f_i(E, \rho) - f^{exp}_i)^2,$$

(44)

respectively in the linear and nonlinear cases, for $E$ and $\rho$ satisfying (42).

Figures 19 and 20 respectively show the functions $e^l$ and $e$ vs. $E$ and $\rho$. They reach their minimum values at $E = 4 \cdot 10^9$ Pa and $\rho = 2000$ kg/m$^3$ for the linear elastic case and at $E = 5 \cdot 10^9$ Pa and $\rho = 2000$ kg/m$^3$ for the nonlinear case. Figure 21 plots the two functions vs. $E$ for $\rho = 2000$ kg/m$^3$. The first five natural frequencies of the tower are shown in Table 7, where the experimental values $f^{exp}_i$ are compared with the numerical values, in the linear elastic ($f^l_i$) and nonlinear ($f_i$) case. Columns $\Delta f^l_i$ and $\Delta f_i$ show the differences between the numerical and experimental values for the linear and nonlinear case, respectively. The numerical frequencies differ from the experimental values by no more than five percent. The nonlinear case is more accurate in evaluating the higher modes. Figure 22 shows the first five mode shapes, calculated in the nonlinear case for the optimal values ($E = 5 \cdot 10^9$ Pa, $\rho = 2000$ kg/m$^3$). They are substantially equal to the linear elastic modes, which are shown in 28. The Figure clearly shows that the third mode is torsional. Finally, Figure 23 shows the maximum principal fracture strains for the tower in equilibrium with its own weight in the inner and outer layers of the mesh for $E = 5 \cdot 10^9$ Pa and $\rho = 2000$ kg/m$^3$. The crack distribution and the low values of the fracture strains indicate a modest level of damage inside the structure: the fracture strains are concentrated around the windows and lintels, and in the masonry supporting the vault under the bell chamber. This confirms the observations made upon visual inspection. Crack strains are concentrated in regions of the structure quite far from those of the highest curvatures for the first mode shapes, which in fact remain very similar to those calculated in the linear elastic case. However, due to the nonlinearity of the constitutive equation adopted, the model updating procedure yields higher values of Young’s modulus, which
increases by about the twenty-five percent with respect to the linear elastic case.
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5. Conclusions

The paper deals with the modal analysis of masonry structures. In particular, a new procedure for calculating the natural frequencies and mode shapes of masonry buildings has been implemented within the NOSA–ITACA code, to take into account the inability of masonry to withstand tensile stresses. Some structures have been analyzed and the influence on their dynamic properties of the fracture strains induced by their self-weights and external loads investigated. In particular, the new algorithm has been applied to the finite-element model of the San Frediano bell tower in Lucca, and the results of the numerical simulation compared with those from an experimental campaign conducted on the tower in June, 2015. The new procedure provides for more realistic simulations of the dynamic behaviour of masonry structures in the presence of cracks and, when combined with experimental data, can be helpful in assessing damage. Moreover, the dependence of the natural frequencies on external loads (the case of thermal loads is under development) can be assessed, and used to interpret data from long-term monitoring protocols.

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| Lateral load [N] | \( f_1 [\text{Hz}] \) | Eff. modal mass [%] | \( f_2 [\text{Hz}] \) | Eff. modal mass [%] | \( f_3 [\text{Hz}] \) | Eff. modal mass [%] |
|-----------------|----------------|-------------------|----------------|----------------|----------------|----------------|
|                 | \( y \)  | \( z \)  | \( y \)  | \( z \)  | \( y \)  | \( z \)  |
| 0               | 6.47   | 80.73 | 0.00  | 25.48 | 0.00  | 53.79 | 0.00  | 81.06 |
| 9000            | 6.44   | 80.73 | 0.00  | 25.48 | 0.00  | 53.74 | 0.00  | 80.76 |
| 10125           | 6.13   | 80.32 | 0.00  | 25.27 | 0.00  | 52.63 | 2.95  | 54.97 |
| 11250           | 5.74   | 79.98 | 0.00  | 24.36 | 0.00  | 51.22 | 4.42  | 44.12 |
| 12375           | 5.23   | 79.53 | 0.00  | 23.12 | 0.00  | 48.97 | 6.05  | 32.33 |
| 13500           | 4.72   | 79.09 | 0.00  | 21.80 | 0.00  | 46.14 | 7.38  | 22.13 |
| 14625           | 4.21   | 78.77 | 0.00  | 19.95 | 0.00  | 43.75 | 8.10  | 16.40 |
| 15750           | 3.71   | 78.35 | 0.00  | 18.34 | 0.00  | 40.69 | 8.60  | 12.06 |

| Lateral load [N] | \( f_4 [\text{Hz}] \) | Eff. modal mass [%] | \( f_5 [\text{Hz}] \) | Eff. modal mass [%] | \( f_6 [\text{Hz}] \) | Eff. modal mass [%] |
|-----------------|----------------|-------------------|----------------|----------------|----------------|----------------|
|                 | \( y \)  | \( z \)  | \( y \)  | \( z \)  | \( y \)  | \( z \)  |
| 0               | 55.96  | 8.77  | 0.00  | 96.43 | 0.00  | 145.30 | 3.07  | 0.00 |
| 9000            | 55.71  | 8.77  | 0.00  | 96.37 | 0.00  | 144.79 | 3.07  | 0.00 |
| 10125           | 54.69  | 6.15  | 25.67 | 94.51 | 0.00  | 141.80 | 3.10  | 0.00 |
| 11250           | 54.30  | 4.90  | 36.06 | 90.15 | 0.00  | 137.51 | 3.11  | 0.00 |
| 12375           | 53.46  | 3.41  | 47.18 | 85.85 | 0.00  | 130.80 | 3.27  | 0.00 |
| 13500           | 52.31  | 2.07  | 56.67 | 82.42 | 0.00  | 122.50 | 3.49  | 0.00 |
| 14625           | 50.79  | 1.22  | 61.54 | 76.97 | 0.00  | 115.60 | 3.72  | 0.00 |
| 15750           | 49.37  | 0.66  | 64.93 | 71.94 | 0.00  | 108.10 | 3.92  | 0.00 |

Table 1: The beam’s natural frequencies \( f_i, i = 1, \ldots, 6 \) and the effective modal masses of the corresponding mode shapes versus load increments.

| MAC – M |          |          |          |          |          |          |
|---------|----------|----------|----------|----------|----------|----------|
| \( \phi_1 \) | \( \phi_2 \) | \( \phi_3 \) | \( \phi_4 \) | \( \phi_5 \) | \( \phi_6 \) |
| \( \phi_i^j \) | 0.99 | 0.00 | 0.06 | 0.03 | 0.00 | 0.00 |
| \( \phi_2 \) | 0.00 | 0.99 | 0.02 | 0.04 | 0.12 | 0.00 |
| \( \phi_3 \) | 0.05 | 0.04 | 0.41 | 0.90 | 0.04 | 0.01 |
| \( \phi_4 \) | 0.04 | 0.00 | 0.88 | 0.41 | 0.01 | 0.19 |
| \( \phi_5 \) | 0.00 | 0.12 | 0.00 | 0.04 | 0.95 | 0.00 |
| \( \phi_6 \) | 0.00 | 0.00 | 0.16 | 0.09 | 0.00 | 0.93 |

Table 2: MAC–M(\( \phi_i^j, \phi_j \)), for \( i,j = 1, 2, \ldots, 6 \) at the last load increment.
| Total vertical load [N] | \( f_1 \) [Hz] | Eff. modal mass [%] | \( f_2 \) [Hz] | Eff. modal mass [%] | \( f_3 \) [Hz] | Eff. modal mass [%] |
|------------------------|----------------|---------------------|----------------|---------------------|----------------|---------------------|
| 0                      | 6.54           | 59.43               | 18.81          | 8.75               | 22.49          | 0.00                |
| 144605                 | 6.53           | 59.29               | 18.50          | 8.83               | 22.01          | 0.00                |
| \( P_1 = 150605 \)     | 5.98           | 51.21               | 12.57          | 15.47              | 21.04          | 0.00                |
| \( P_2 = 152605 \)     | 4.79           | 28.54               | 9.00           | 37.27              | 18.79          | 0.00                |
| \( P_3 = 153605 \)     | 3.61           | 15.38               | 8.10           | 50.00              | 15.43          | 0.00                |
| \( P_4 = 154605 \)     | 2.54           | 9.91                | 7.74           | 77.40              | 12.06          | 0.00                |

Table 3: The structure’s natural frequencies \( f_i \), \( i = 1, \ldots, 5 \) and the effective modal masses of the corresponding mode shapes versus load increments. Load \( P \) at position \( X_4 \).

| Total vertical load [N] | \( f_4 \) [Hz] | Eff. modal mass [%] | \( f_5 \) [Hz] | Eff. modal mass [%] |
|------------------------|----------------|---------------------|----------------|---------------------|
| 0                      | 33.74          | 0.00                | 45.37          | 11.88               |
| 144605                 | 32.44          | 0.00                | 44.80          | 11.28               |
| \( P_1 = 150605 \)     | 30.34          | 0.00                | 38.48          | 6.74                |
| \( P_2 = 152605 \)     | 27.28          | 0.00                | 32.70          | 3.96                |
| \( P_3 = 153605 \)     | 25.45          | 0.00                | 28.19          | 2.98                |
| \( P_4 = 154605 \)     | 24.20          | 0.00                | 23.60          | 1.39                |

Table 4: MAC – \( M(\phi_i, \phi_j) \), for \( i,j = 1, 2, \ldots, 5 \) at the last load increment. Load \( P \) at position \( X_4 \).
Table 5: The structure’s natural frequencies $f_i$, $i = 1, \ldots, 5$, and the effective modal masses of the corresponding mode shapes versus load increments. Load P at position $X_7$.

| Total vertical load $[N]$ | $f_1[Hz]$ | Eff. modal mass [%] | $f_2[Hz]$ | Eff. modal mass [%] | $f_3[Hz]$ | Eff. modal mass [%] |
|--------------------------|-----------|---------------------|-----------|---------------------|-----------|---------------------|
|                          | x         | z                   | x         | z                   | x         | z                   |
| 0                        | 6.54      | 59.43               | 0.00      | 18.81               | 22.49     | 0.00                | 7.92       |
| 144605                   | 6.53      | 59.29               | 0.00      | 18.50               | 22.01     | 0.00                | 6.98       |
| $P_1=150605$             | 6.36      | 57.80               | 0.00      | 17.09               | 18.82     | 0.00                | 0.70       |
| $P_2=152605$             | 6.19      | 55.55               | 0.00      | 15.06               | 16.47     | 0.00                | 0.00       |
| $P_3=153605$             | 6.06      | 53.07               | 0.00      | 13.41               | 14.26     | 0.00                | 0.60       |
| $P_4=154605$             | 5.97      | 51.06               | 0.00      | 12.44               | 13.25     | 0.00                | 0.10       |

| Total vertical load $[N]$ | $f_4[Hz]$ | Eff. modal mass [%] | $f_5[Hz]$ | Eff. modal mass [%] |
|--------------------------|-----------|---------------------|-----------|---------------------|
|                          | x         | z                   | x         | z                   |
| 0                        | 33.74     | 0.00                | 45.37     | 11.88               | 0.00     |
| 144605                   | 32.44     | 0.00                | 44.80     | 11.28               | 0.00     |
| $P_1=150605$             | 26.00     | 0.00                | 40.18     | 0.00                | 5.55     |
| $P_2=152605$             | 24.37     | 0.00                | 39.31     | 0.00                | 3.69     |
| $P_3=153605$             | 23.34     | 0.00                | 37.13     | 2.74                | 0.00     |
| $P_4=154605$             | 22.96     | 0.00                | 35.52     | 2.09                | 0.00     |

Table 6: MAC–M($\phi_i, \phi_j$), for $i,j = 1, 2, \ldots, 5$ at the last load increment. Load P at position $X_7$.

| MAC – M            |
|--------------------|
| $\phi_1$ | $\phi_2$ | $\phi_3$ | $\phi_4$ | $\phi_5$ |
| $\phi_1$ | 0.98     | 0.17     | 0.00     | 0.00     | 0.00     |
| $\phi_2$ | 0.17     | 0.98     | 0.00     | 0.00     | 0.04     |
| $\phi_3$ | 0.00     | 0.00     | 0.46     | 0.89     | 0.00     |
| $\phi_4$ | 0.00     | 0.00     | 0.64     | 0.34     | 0.00     |
| $\phi_5$ | 0.00     | 0.03     | 0.00     | 0.00     | 0.75     |

Table 6: MAC–M($\phi_i, \phi_j$), for $i,j = 1, 2, \ldots, 5$ at the last load increment. Load P at position $X_7$. 49
Table 7: Natural frequencies [Hz] of the tower: experimental values $f_i^{\text{exp}}$, numerical values $f_i^l$ identified on the linear elastic model; numerical values $f_i$ identified on the masonry-like model.

| Mode Shape | $f_i^{\text{exp}}$ | $f_i^l$ | $\Delta f_i^l$ | $f_i$ | $\Delta f_i$ |
|------------|--------------------|--------|----------------|------|-------------|
| Mode Shape 1 | 1.14               | 1.15   | 0.01           | 1.18 | 0.04        |
| Mode Shape 2 | 1.38               | 1.38   | 0.00           | 1.41 | 0.02        |
| Mode Shape 3 | 3.44               | 3.60   | 0.05           | 3.39 | -0.01       |
| Mode Shape 4 | 4.60               | 4.41   | -0.04          | 4.61 | 0.00        |
| Mode Shape 5 | 5.34               | 5.35   | 0.00           | 5.44 | 0.02        |