Comparative study between the NLPI controller and the CPI controller

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ABSTRACT

Unquestionably, the classic CPI controller dominates the industry and has the advantage of being simple and easy to implement. Because its setting remains intuitive and more practical. On the other hand, these disadvantages lie in the fact that most of them reach a compromise in terms of speed of response and stability. Even worse, such an approach becomes insufficient at the increasingly demanding speeds demanded by the industry. In this context the NLPI controller is currently presented as an alternative. With its simple tuning method and robustness to process parameter variations, it stands out as a valuable addition to the toolbox of control engineering specialists. This paper aims to provide a simulation-based study using a MAS controlled by IFOC, comparing the PI controller system to the NLPI controller system. The results will be in favor of the last one.

Keywords:
NLPI (Non-linear PI)
CPI (Classical PI)
IFOC (Indirect Vector Control)
MAS (Asynchronous Machine)

1. INTRODUCTION

Currently, the NLPI controller currently presents itself as an alternative to both conventional CPI methods and modern model-based approaches to solving control problems subject to high variations and dynamics. The nonlinearities that occur at high speeds, means that the use of the CPI controller to control the speed of the asynchronous machine "MAS" is often characterized by an overrun in tracking mode and a bad rejection of load disturbance [1-3].

This is mainly due to the fact that controller gains cannot be granted to simultaneously resolve overflow and load disturbance rejection problems, as the overload removal setting will result in a bad charge rejection. The disruption of the load on the one hand and live on that on the other hand. However, the situation becomes more and more complicated for high speeds. It is from here that other types of control, otherwise technologically designed like the NLPI "non-linear PI", have to be used to manage par excellence and simultaneously the constraints relating to the overrun and to charge disturbance rejection [4-8].

The NLPI controller has therefore emerged as an alternative that combines easy applicability of conventional PI control methods with the power of modern model-based approaches. The NLPI controller foundation deals with real disturbances and modeling uncertainties together, so that only a very coarse process model is needed to design a control loop, making the NLPI controller an attractive choice for practitioners and promises good robustness against process variations. These current applications range from power electronics, motion control and superconducting radio frequency cavities to voltage and temperature control [8, 9].

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2. CONTROL TECHNIQUES

2.1. CPI Controller

Figure 1 illustrates the block diagram of MAS control by a CPI.

![Figure 1. Control of MAS by CPI](image)

Obviously, the CPI controller makes it possible to correct within a suitable time the difference that exists between a setpoint (speed, temperature, pressure, position) and its associated measurement return, by increasing or decreasing the output of the MAS and this, by the intermediate of two main parameters called gains (Proportional gain and Integral gain), which can accelerate, delay and stabilize this correction. The regulation function can be embodied in the form of a black box that can be represented by a comparator with 2 inputs (setpoint and return), the result of this comparator controls the output of the MAS [9, 11-15].

However, if the value of the feedback value is greater than the value of the setpoint, the difference is positive and the CPI controller will slow down the motor. Conversely, if the value of the measurement feedback is lower than the value of the setpoint, the difference is negative and the CPI controller will make sure to accelerate the motor. The response time of the system is adjustable via two gain parameters. In our case, the cruise control is used to determine the reference torque, in order to maintain the corresponding speed. The velocity dynamics is given by the following mechanical equation:

\[
F(s) = \frac{\omega}{C_{em}} = \frac{p}{T_s + s} \tag{1}
\]

For this purpose, the simplified block diagram of the control system based on a controller CPI is given in Figure 2.

![Figure 2. Block diagram of speed control by a CPI](image)

Then, the closed loop transfer function is written as follows:

\[
\frac{\omega}{\omega_r} = \frac{(kp_\omega s + k_i \omega)^p}{p(s)} \tag{2}
\]

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The characteristic equation \( \rho(s) \) is:

\[
\rho(s) = s^2 + \frac{f_0 + k_{pu}p}{j} s + \frac{k_{iu}p}{j} = 0
\]

(3)

By imposing two complex poles conjugated \( s_{1,2} = \rho_{\omega^*}(1 \mp j) \) in closed loop and by identification, one obtains the parameters of the controller PI:

\[
\begin{align*}
    k_{pu} &= \frac{2\rho_{\omega^*} - f_0}{p} \\
    k_{iu} &= \frac{2\rho_{\omega^*}}{p}
\end{align*}
\]

(4)

2.2. NLPI Controller

For the control of the MAS by NLPI, we adopt the same device illustrated by Figure 1, except change of the CPI controller by another NLPI. In addition, we also use NLPI regulators for current regulation of the IFOC command.

At speeds reaching higher and higher thresholds, in an industrial environment where the transient mode is characterized by a strong non-linearity, it is even more judicious to call on the NLPI controller, in order to improve the quality of control even more. By guaranteeing impeccable speed control, we say that NLPI control is considered one of the most efficient and simple methods for industrial applications [10], [11], [14]-[17]. This control can be expressed for any control structure of the following form:

\[
u(t) = k_p(\cdot)e(t) + k_i(\cdot) \int e(\tau)d\tau \quad (5)
\]

Where are control gains \( k_p(\cdot) \) et \( k_i(\cdot) \) varying over time, depending on the state of the system, the signal of the command and \( e(t) \) the error.

The action of the NLPI controller represented by the Figure 3:

![Figure 3. fal (e, \alpha, \delta) function characteristics](image)

This control action is given by:

\[
u = k_pfal(e, \alpha, \delta_p) + k_ifal(\int ed\tau, \alpha, \delta_i) \quad (6)
\]

With:

\[
fal(e, \alpha, \delta) = \begin{cases} 
|e|^\alpha \text{sign}(e) & |e| > \delta, \delta > 0 \\
\frac{e^{\delta}}{\delta^\alpha} & |e| \leq \delta
\end{cases}
\]

(7)
Where \( f_N (x, u, s) \) is a nonlinear function; \( k_p \) and \( k_i \) are respectively the proportional and integral gains of the NLPI controller; \( u \) is the command signal; the parameters \( \alpha_p \) and \( \alpha_i \) are constant, empirically chosen in the range 0 to 1. When \( \alpha_p = \alpha_i = 1 \), the controller becomes a linear PI. \( \phi \) is a constant, which can be empirically fixed to a small value.

The NLPI is thus designed by adapting its response based on the performance of the closed-loop control system. When the error between the controlled and actual values of the controlled variable is large, the gain amplifies the error substantially to generate a large correction to quickly drive the system out of its way. On the other hand, when the error decreases, the gain is reduced automatically to avoid excessive oscillations and large overshoots in the response. Thanks to this automatic gain adjustment, the NLPI controller has both a high initial gain and a fast response, followed by a small gain to avoid oscillatory behavior [17-25].

3. SIMULATION RESULTS AND ANALYSIS

A simulation series is performed for each of the two control cases described above, followed each time by a summary of the results and an in-depth analysis. All simulations take into account the dynamics of the torque, flux and current in the MAS subjected to the IFOC command.

To evaluate the tuning performance, we simulated each control case with speed controller parameters calculated by a pole location \( \rho \omega \), a set speed \( \omega_r = 150 \text{ rad/s} \), followed by an application and a charge removal (\( C_{ch} = 10 \text{ N.m} \)) at times \( t=1.5 \text{ s} \) and \( t=2.5 \text{ s} \), respectively. Then an application of a set point change at the moment \( t=4 \text{ s} \).

Table 1 below shows the control parameters for each case:

| Controller | Speed (rpm) |
|------------|-------------|
| CPI        |             |
| NLPI       |             |
| \( \alpha_p = 0.1 \), \( \alpha_i = 0.2 \), \( \alpha = 0.7 \), \( \alpha_i = 0.8 \), \( k_p = 10 \) | \( \phi = 0.1 \), \( k_i = 0.2 \), \( \phi = 0.7 \), \( k_i = 0.8 \), \( k_p = 1000 \) |
| \( \alpha_p = 0.1 \), \( \alpha_i = 0.2 \), \( \alpha = 0.7 \), \( \alpha_i = 0.8 \), \( k_p = 100 \) | \( \phi = 0.1 \), \( k_i = 0.2 \), \( \phi = 0.7 \), \( k_i = 0.8 \), \( k_p = 1000 \) |

3.1. Behavior of speed

The analysis of Figure 4 shows Behavior of the output speed \( y \) with respect to that of the input \( \omega_r \) (CPI case), allows us to deduce:

1. Registration of a transitional regime at start-up during 0,35 s;
2. During this phase the speed follows its reference value with an overrun of 10 rad/s (\( y = 154 \text{ rad/s et } \omega_r = 144 \text{ rad/s} \)) and this to \( t = 0.13 \text{s} \);
3. At the application of a load, at approximately \( t = 1.5 \text{s} \), we record a downward peak of a value of \( y = 142 \text{ rad/ s} \);
4. When lifting the load (\( t = 2,5 \text{s} \)), this time we record a peak at the top of a value \( y = 158 \text{ rad/ s} \).
5. We then say that the speed decreases with the increase of the load and then increases with its decrease.
6. At \( t = 4 \text{s} \), the direction of rotation of the MAS changes.
Overall, therefore, we record a good reference tracking by that of the output and a good rejection of disturbances during the application of the load.

The analysis of Figure 5, allows us to deduce:

1. Registration of a transitional regime at start-up during 0.33 s;
2. During this phase the speed follows its reference value with an overrun of 5 rad/s \( (y = 154 \text{ rad/s et } \omega_r = 149 \text{ rad/s}) \) and this to \( t = 0.13s \);
3. At the application of a load, at approximately \( t = 1.5s \), we record a downward peak of a value of \( y = 149.7 \text{ rad/s} \);
4. When lifting the load \( (t = 2.5s) \), this time we record a peak at the top of a value \( y = 150.2 \text{ rad/s} \);
5. We then say that the speed decreases with the increase of the load and then increases with its decrease;
6. At \( t = 4s \), the direction of rotation of the MAS changes.

Overall, therefore, this time we record a better reference tracking by that of the output and a better rejection of disturbances during the application of the load and this compared to the control case at CPI.
3.2. Behavior of current

Figure 6 shows current behavior $I_{ds}$ and $I_{qs}$ (CPI case).

![Figure 6. Current behavior $I_{ds}$ and $I_{qs}$ (CPI case)](image)

From the analysis figure 6 we can deduce:

a) Current $I_{ds}$:
   In general, during the entire period between 0 and 4s, the current $I_{ds}$ stabilizes at about 3.9 A.

b) Current $I_{qs}$:
   i. The rise time between 0 to 0.13 s has a peak of about 30 A.
   ii. Throughout the interval from $t = 1.5s$ to $t = 2.5s$ a rise (with bumps at the ends) of about 6A.
   iii. At $t = 4s$ a signal fluctuation indicates the change in the rotation of the MAS.

Figure 7 shows current behavior $I_{ds}$ and $I_{qs}$ (NLPI case)

![Figure 7. Current behavior $I_{ds}$ and $I_{qs}$ (NLPI case)](image)

From the analysis figure 6 we can deduce:

a) Current $I_{ds}$:
   In general, during the entire period between 0 and 4s, the current $I_{ds}$ stabilizes at about 3.1 A.

b) Current $I_{qs}$:
   i. The rise time between 0 to 0.13s records a peak of about 38 A.
   ii. Throughout the interval from $t = 1.5s$ to $t = 2.5s$ a virtually rectangular rise (without end fluctuations) of about 6A.
   iii. At $t = 4s$ a signal fluctuation indicates the change in the rotation of the MAS.

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3.3. Flux behavior

Figure 8 shows flux behavior $\phi_{dr}$ and $\phi_{qr}$ (CPI case).

Figure 8 shows that the fluxes and consequently the electromagnetic torque are maintained at their desired values. Obviously, we record:

i. After a short transient departure, the two flows have fluctuations (ripples) at startup:
   1. $\phi_{dr} = 1.6 \text{ Wb (Alternance positive)}$
   2. $\phi_{qr} = 0.7 \text{ Wb (Alternance positive)}$
   3. $\phi_{qr} = -0.3 \text{ Wb (Alternance négative)}$

ii. Very slight fluctuations at the moment of the lifting of the load at $t = 2.5s$;

iii. Slight fluctuations at the time of change of MAS rotation at $t = 4s$;

iv. During all the time, we can confirm the flow of the rotor knows only the reference flow ($\phi_{dr}= 1$ and $\phi_{qr}= 0$) and is therefore independent of speed and torque variations. This implies a good decoupling.

Figure 9 shows Flux behavior $\phi_{dr}$ and $\phi_{qr}$ (NLPI case).

Figure 9 shows that the fluxes and consequently the electromagnetic torque are maintained at their desired values. Obviously, we record:

i. After a short transient departure, the two flows have fluctuations (ripples) at startup:
1. $\Phi_{dr} = \begin{cases} 1.7 \text{ Wb (Alternance positive)} \\ 0 \text{ Wb (Alternance negative)} \end{cases}$
2. $\Phi_{dr} = \begin{cases} 1.7 \text{ Wb (Alternance positive)} \\ 0 \text{ Wb (Alternance negative)} \end{cases}$

ii. Slight fluctuations at the time of change of MAS rotation at $t = 4s$;
iii. During all the time, we can confirm the flow of the rotor knows only the reference flow ($\Phi_{dr} = 1$ and $\Phi_{qr} = 0$) and is therefore independent of speed and torque variations.

This implies a significantly improved decoupling compared to that of the CPI case.

4. CONCLUSION

Obviously, the constant development of research based on reflective and creative thinking for the continuous improvement of a speed controller putting an end to the constraints of the transient mode, led us to the NLPI controller thus ensuring a very good monitoring and an excellent rejection of disturbance is indeed the most effective way to ensure a better-quality treatment. The results of the simulations relating to a MAS commanded by IFOC and controlled by an NLPI controller are significantly improved, compared to the other ones of the CPI controller, that it is in terms of monitoring the speed of reference $\omega_r$ by that of the output $y$, the rejection of disturbance or even behavior of the currents $I_{as}$ and $I_{as}$ and flux behavior $\Phi_{dr}$ and $\Phi_{qr}$ involving decoupling improvement. We will end by saying that the main importance of NLPI controllers lies in their robustness and performance against variations.

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