Tribological Characteristics near Welding Limit for Petroleum Metal Pipelines

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INTRODUCTION
The number of petroleum transport pipelines that are operated in changing and harsh conditions increases each year [1]. The environmental conditions as well as the difference aggressive of raw extractive materials and many derivatives petroleum productions are responsible for pipelines damages due to fatigue, mechanical damages, wear and corrosion [2-4]. The internal surfaces of transport welded pipelines petroleum and its derivatives can be deformed and damaged due to one of wear types as a consequence of the movement of many kinds of solid particles caused by petroleum and its derivatives flows. Apart from typical deformations in the form of weight loss as well as cracks, these phenomena can cause a reduction in the cross-section of the welding regions. These factors make it necessary to use protective coating layers on the internal surfaces of transport-welded pipelines, which improve the corrosion and wear resistance [5-7]. These coatings can be deposited by thermal spraying during the production of welded pipelines [8].

In particular, for natural gas, petroleum and its derivatives transportation pipelines over long distances, the use of pipes fabricated from high strength carbon steel (HSCS) such as (A 106 grade B) is very attractive from a technical and economical point of view. The basic advantages of (A 106 grade B) are high strength and toughness, higher strength/weight ratio, good formability and good weldability [2]. Weldability, being a very important technological factor of petroleum and its derivatives transportation pipelines carbon steels, depends on the metallurgical purity and total content of carbon and alloying elements. These factors govern the crack sensitivity of welded joints during the solidification process of a weld and wear tack place.

In some applications, like wear resistance the compounds coatings can be used [9,10]. However, chemical compounds commonly deposited layers are manufactured by the use of ceramics and metals. The ceramic coatings Al₂O₃/TiO₂, ZrO₂/CaO and SnS can be applied by a thermal spraying process [11-13]. The research works are focused on the improvement
of wear and corrosion resistance of welded layers of pipeline. One of the methods to reduce the wear of welded regions could be the use of coating technique which is made in an air environment.

In petroleum and its derivatives transportation pipelines applications, the specimens made up of welded pipelines have been subjected to mechanical rubbing action, which creates wear action. Wear is one kind of materials removal from the contacting surfaces due to relative motions between the two mating parts [14, 15]. The wear that occurred in the components was mainly based on the applied load, the operating temperature, and speed, the hardness of the material, the time duration of test, the atmospheric action, and the presence of foreign matters.

Tribological behavior, namely the wear rate, the wear coefficient, and the coefficient of friction, has clarified the wear rate of materials, which have played a major role in material removal [16 - 20]. This indicates the transfer of material from one surface to another owing to the relative motion between the contacting surfaces. A dry sliding wear test is the best experimental technique for many structural applications to find out the wear behavior.

According to the conditions of the Pin – on disc machine the wear rate \( W_R \) and wear coefficient \( K \) are calculated according to the following equation [21]:

\[
W_R = \frac{\Delta W_V}{S_D} \\
K = \frac{W_V.H_V}{S_D.L}
\]  

Where: \( W_V \), is the wear volume loss (mm\(^3\)) of the specimen before and after the wear test, \( H_V \), Vickers hardness (N / mm\(^2\)), \( L \), load (N) and \( S_D \): is the sliding distance (m). The friction coefficient equation [22]:

\[
\mu = \frac{Q}{r \times L}
\]

Where: \( Q \), is friction torque (m.N), \( L \) (load) applied on the sample (Newton) and \( r \), is radius of counter face stainless steel disc in (m).

Wear is a foremost issue in the engineering field in which is one of the reasons of the failure during the usage of mechanical parts. Based on the literature, there was no report and detailed study on the dry sliding wear behavior of welded pipelines. Therefore, it is necessary to examine and study the wear behavior petroleum and its derivatives transportation pipelines. Therefore, the present research work was focused to optimize the essential wear parameters to determine the appropriate conditions and materials to be used for the petroleum and its derivatives transportation pipelines applications, which minimize the wear.

**MATERIALS AND METHODOLOGY**

The high strength carbon steel (HSCS) (A 106 grade B) was received from Ministry of Industry, Iraq. The chemical composition, the physical properties and the mechanical properties of as-received (HSCS) (A 106 grade B) are given in Table 1. Tin sulfide type (SnS) was synthesized using wet chemical method under standard conditions [23]. To get the (SnS) product, tin dichloride having the formula (SnCl\(_2\).2H\(_2\)O) as the first source with sodium sulfide of the formula (Na\(_2\)S) as the second source are used to produce tin and sulfide respectively, with the present of deionized water, which is used as solvent. The weight of (SnCl\(_2\).2H\(_2\)O) was (12) g and for (Na\(_2\)S) was (18) g according to appropriate molar ratio. The (SnCl\(_2\).2H\(_2\)O) were dissolved in deionized water using magnetic stirrer for 2 hours at room temperature and (Na\(_2\)S) solution was dropping into the solution of SnCl\(_2\).2H\(_2\)O). The precipitates (SnS) were wished with ethanol and dried at room temperature. According to this reaction (SnS) dark brown color nano powder were obtained.

Three models were prepared; the first as-received (A106 grade B) pipe was cut into several pieces of (ø40 × 10) mm as rings for doing the wear test. The second model is an ER6010 wire weld joint and the third model is the ER6010 wire weld joint with SnS nanoparticles added using the gas tungsten arc welding (GTAW) process after removing the cellulose cover of the wire. Each model was divided into three parts (R35x30x10) mm. Before the samples were cleaned with acetone and then the samples were polished as per the metallographic procedure. The samples were polished with different SiC grit papers (400, 600, 800, 1000, 1200, 2000 SiC grits/cm\(^2\)). Figure 1 show the specimen dimensions.
The ring – on – disc sliding wear test equipment was used to study the tribological behavior of high strength carbon steel (A 106 grade B) as pure, 6010 welded and SnS- 6010 welded) under the dry condition at ambient temperature. The work carried out by using a change loads from (70, 80, 90, 100, 110, 120 and 130) N for one hour with speed (120) rev/min and sliding distance (904.32) m. The cylindrical counter disc was made of hardened stainless steel having 10 mm thickness and 40 mm diameter. The counter disc was initially ground and cleaned with acetone to eliminate the presence of oxide layers, foreign particles, and moisture. The prepared specimens for wear test were weighed before and after the test by which the mass loss was determined. Hardness and density tests were employed to determine the wear rate and wear coefficients based on the mass losses using (1) and (2) formula. The microstructural images in as-received condition and the worn surfaces were examined using the SEM (scanning electron microscope), with the AFM (atomic force microscopy) (done at Iran) to show how could be the surfaces after tribological testes.

**RESULTS AND DISCUSSION**

The X – ray diffraction is shown in Figure 2. The size of the nanoparticles was estimated using Scherer’s formula based on the full width at half-maximum (FWHM) of the different diffraction peak with different values [24]:

\[
A = \frac{0.94\lambda}{\beta \cos \theta}
\]  

(4)

Where: A is crystallite size, \(\beta\) is the FWHM of the diffraction peak, \(\lambda\) is the wavelength of X-ray radiation and \(\theta\) is the angle of diffraction. The average Scherrer size of these NPs is (70) nm. The inter-planar spacing measured were in good agreement with the spacing for the (110) (120), (111), (040), (041), (200), (141), (151), (122), (231), (042), (202), (023) and (133) planes for the orthorhombic SnS structure with JCPDS file NO. 00.039-0354.

Figure 3 shows the SnS nanopowder and Figure 3b shows the AFM surface image of SnS.
below. There are shown the elements included in the used steel and their proportions. The results in Table 3 show an increase in the percentage of sulfur and tin.

### Table 1. Analytical chemical compositions of carbon steel A 106 GR B.

| Elements | C      | Mn      | Si      | P      | S      | Cr      | Ni      | Mo | V    | Cu | Fe     |
|----------|--------|---------|---------|--------|--------|---------|---------|----|------|----|--------|
| Value%   | 0.23   | 0.76    | 0.33    | 0.017  | 0.019  | 0.32    | 0.19    | 0.08| 0.04 | 0.24| Remaining |

### Table 2. Analytical chemical compositions of carbon steel ER 6010.

| Element | C      | Mn      | Si      | Cr      | Ni      | Mo | V | Sn | Fe    |
|---------|--------|---------|---------|---------|---------|----|---|----|------|
| Value%  | 0.18   | 0.28    | 0.14    | 0.02    | 0.02    | < 0.01| 0.01| 0.314| Remaining |

### Table 3. Analytical chemical compositions of carbon steel ER 6010 with SnS.

| Element | C      | Mn      | Si      | S      | Cr      | Ni      | Mo | V | Sn | Fe    |
|---------|--------|---------|---------|-------|---------|---------|----|---|----|------|
| Value%  | 0.18   | 0.28    | 0.14    | 0.034 | 0.02    | < 0.01 | 0.01| 0.314 | Remaining |

Tables 4, 5 and 6 illustrate the results of the wear tests for the three models as below, (the coefficient of friction, coefficient of wear and wear rate). The values were increased for all specimens with increase in the amount of load leads to deform and fracture the surface films and allows for true contact and the large area contact, thus increasing the mass removed.

### Table 4. Experimental results of sliding wear of A 106 Gr B.

| Exp. No. | Mass difference (g) | Load (N) | Wear rate (mm/m) | Coefficient of wear (k) *10^-3 | Coefficient of friction (µ) |
|----------|---------------------|----------|------------------|-------------------------------|--------------------------|
| 1        | 0.166               | 60       | 0.02379          | 0.4024                        | 0.2333                   |
| 2        | 0.209               | 70       | 0.02995          | 0.4515                        | 0.3019                   |
| 3        | 0.425               | 80       | 0.06092          | 0.7754                        | 0.6150                   |
| 4        | 0.563               | 90       | 0.08070          | 0.9140                        | 0.8155                   |
| 5        | 0.793               | 100      | 0.11367          | 1.1601                        | 1.1500                   |
| 6        | 0.868               | 110      | 0.12460          | 1.1548                        | 1.2593                   |
| 7        | 0.913               | 120      | 0.13090          | 1.1148                        | 1.3256                   |
| 8        | 1.017               | 130      | 0.14578          | 1.1464                        | 1.4775                   |

### Table 5. Experimental results of sliding wear for welding wire ER 6010.

| Exp. No. | Mass difference (g) | Load (N) | Wear rate (mm/m) | Coefficient of wear (k) *10^-3 | Coefficient of friction (µ) |
|----------|---------------------|----------|------------------|-------------------------------|--------------------------|
| 1        | 0.139               | 60       | 0.0198           | 0.3361                        | 0.1999                   |
| 2        | 0.222               | 70       | 0.0317           | 0.4611                        | 0.3200                   |
| 3        | 0.373               | 80       | 0.0533           | 0.6790                        | 0.5386                   |
| 4        | 0.458               | 90       | 0.0655           | 0.7422                        | 0.6621                   |
| 5        | 0.640               | 100      | 0.0915           | 0.9341                        | 0.9260                   |
| 6        | 0.753               | 110      | 0.1077           | 1.0014                        | 1.0905                   |
| 7        | 0.874               | 120      | 0.1250           | 1.0649                        | 1.2668                   |
| 8        | 1.121               | 130      | 0.1604           | 1.2612                        | 1.6253                   |

### Table 6. Experimental results of sliding wear for welding wire ER 6010 with SnS.

| Exp. No. | Mass difference (g) | Load (N) | Wear rate (mm/m) | Coefficient of wear (k) *10^-3 | Coefficient of friction (µ) |
|----------|---------------------|----------|------------------|-------------------------------|--------------------------|
| 1        | 0.123               | 60       | 0.0176           | 0.2967                        | 0.1764                   |
| 2        | 0.182               | 70       | 0.0260           | 0.3771                        | 0.2617                   |
| 3        | 0.269               | 80       | 0.0384           | 0.4791                        | 0.3878                   |
| 4        | 0.335               | 90       | 0.0479           | 0.5420                        | 0.483                    |
| 5        | 0.410               | 100      | 0.0530           | 0.5976                        | 0.5925                   |
| 6        | 0.507               | 110      | 0.0725           | 0.6718                        | 0.732                    |
| 7        | 0.602               | 120      | 0.8615           | 0.7318                        | 0.870                    |
| 8        | 0.694               | 130      | 0.0993           | 0.7793                        | 1.004                    |

From the Figures 4 and 5, it can be shown the relationship between the coefficient of friction and the coefficient of wear with the load imposed on the sample for the dependent models which is shown a decrease in the friction coefficient and the coefficient of wear of the (ER6010) alloy with the addition of SnS nanoparticles compared with the original metal and the ER6010 alloy which has the highest friction coefficient, and that because SnS plays as lubricant element.
Moreover, the resin for this result the SnS was increasing the durability and efficiency because wear and friction coefficient obtained decreasing in sample (ER6010WITH SnS).

Figure 6 illustrates the sample ER6010+SnS NPS show improvement in the wear rate compared with the pure metal and the ER 6010 alloy, this is due to the increase in the load because the wear rate proportional to the distance slid, which leads to an increase in the amount of erosion the wear rate for three curves is fairly constant.

Figure 7 represents the digital microscopy picture, sever wear occurs in load (7) Kg for pure metal and the ER6010 alloy and the module (ER6010 + SnS). It showed the formation of the oxide film as a result of mild wear, which leads to a decrease in the volume of the material removed and thus to a decrease in the wear and friction coefficient as well as in the wear rate.

In Figure 8, the digital microscopy images shown more sever wear occurs in load (10) Kg for the pure metal and the ER6010 alloy which it leads to more plowing depth, but the module ER6010 +SnS. It showed the less ploughing with superficial scratches as a result of the action of SnS which it leads to a decrease in the volume of the material removed and thus to a decrease in the wear and friction coefficient as well as in the wear rate.
The digital microscopy picture for more severe wear occurs in load (13) Kg for the pure metal and the ER6010 alloy are shown in Figure 9, which it leads to plastically deformation and more ploughing depth, but the module ER6010 + SnS showed less ploughing with superficial scratches plastically deformation as a result to severe wear. Figure 10 represents SEM images for pure metal and the ER6010 alloy, but the module ER6010 + SnS under load (130), which appears the samples nanostructures after wear tests.

CONCLUSIONS
The presence of normal load indeed affects the friction force, and wear rate considerably. The values of friction coefficient, coefficient of wear and the wear rates increase with the increase of normal load (but not always). Wear tests of three samples demonstrated that the SnS nanoparticles reduced the wear and friction coefficients as well as the wear rate despite the increased loads as they reduced the size of the material removed and also by reducing surface ploughing.

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