Design of a new railway wagon for intermodal transport with the adaptable loading platform

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Abstract. The aim of the article is to present the structural design of freight railway wagon with variable use of loading space with regard to the safe operation and assessment of the properties by the calculation methods of simulation analysis. Virtual model of wagon was created in a computer program PTC/Creo. After creating the construction, the calculation model of the wagon frame was subjected to the static and dynamic analysis in programs ANSYS and ADAMS/Rail. On the basis of computer aided simulation analysis was optimized the chassis frame of the wagon. This wagon will be able to offer even more capacity and utilize less resources and energy than current wagons for intermodal transport.

1 Introduction

Indeed nowadays about 1,5 billion t/km are transported in Europe by lorry at distances farther than 150 km, conversely only 0.4 billion t/km (20%) are transported by train, this entails important costs for fossil fuels. In the nearby future when transportation has to be more sustainable it seems quite clear that freight railways will win the mode choice more often [1].

For this to happen though, it is necessary that freight railways, apart from lowering their prices, significantly improve highly increase the quality of transportation. In that sense, quality standards such as reliability, flexibility, availability, cargo security and safety, punctuality, customisation, marketability, traceability, complementary servicing and time for transport among others have to be improved by railways as well.

Hence, rail freight has the challenge to become excellent and to gain in reputation. There are many actions to increase quality in rail freight transport; one of them is the optimisation of the current wagon fleet to improve availability, flexibility, marketability, commercial speed, cargo security and cost. This optimisation has to respond to the actual trends of transport demand and has to be in consonance with the required and feasible infrastructure upgrades.

The basic idea of new wagon is that in the future, longer loading surfaces without interruptions, as well as more capable platforms with higher axle loads and with lower loading heights will be necessary to increase the capacity of the freight railway transportation.

The time schedule is divided to several subtasks as shown in Fig. 1.

Fig. 1. The time schedule segmented into several subtasks.

2 Analysis of the problems and requirements of railway operators

Intermodal continental transports utilize a large amount of unit types (much more than hinterland (sea) transportation) and that increases the amount of possible loading cases for the trains. In that sense, there would be an optimal wagon for each case but this wagon could be sub-optimally utilized for other cases. This variety of cases makes difficult to know which wagon is the optimal for an averaged situation.

Wagons represent an important investment for companies (c.a. 100,000 € per wagon) and they should be extensively utilized during their whole life cycle (25-30 years) to achieve profitability. For this reason, wagons specialized in one kind of units are usually employed for other unit types even if they are not 100% efficient at it.

From individual wagons were created four trains, whose maximum length were 500 m and have been investigated the parameters listed in Fig. 2.
since it would enable better utilization of space (loading length) on trains than existing wagon technologies. The 80 ft wagons would be able to transport same of even more amount of units with the fewer axles and the less deadweight. Furthermore, the aerodynamics would improve (fewer gaps between containers, fewer bogies per meter) and the noise emissions would be reduced due fewer axles per train.

Longer loading lengths 85 ft and 90 ft could have an advantage too, but only if the 45 ft unit is widely introduced and if it dominates in intermodal traffics, which is not the actual case. A revision of this issue has to take place in the approximately 5 years.

The strategic procedure would be to design a 80 ft without pocket and try to make it as cheap as possible. By this it could be very competitive in its market segment.

### 3 Concept of wagon in the form of 3D model

The model presents a complete structure of the 4 – axle freight wagon (Fig. 3) for transportation of ISO containers and swap bodies which should meet prescriptions TSI-WAG, valid regulations UIC, agreement on the reciprocal use of freight wagons in international traffic AVV (RIV), recommendations ERRI and norms EN.

Chassis frame (Fig. 4) is welded the steel construction consisting of two side longitudinal beams (in the most of “I” profiles various sections) which through the cross beams create a support frame. On lower flange plate of main cross-beam there is rotating lead for upper of rotating pivot standard type and arms for sprung side bearers. The passage from the front to the center part of the wagon is branched because of better layout longitudinal forces. The center section of chassis frame contains the cross brace by reason optimizing the torsional stiffness of the wagon. The material thickness...
of individual structural parts has been optimized based on strength analysis simulations.

4 Computer aided simulation analyzes

Integral part of the development of rail vehicles are ride tests performed on test tracks and static tests on test stands. These tests are expensive and nowadays are using the modern computer technologies that can simulate riding a vehicle/train on the track and detect selected parameters \([2, 3]\).

For the proposed wagon were performed following simulation analyzes:
- Static evaluation:
  - wagon torsional stiffness,
  - max. vertical load forces,
  - wagon rising on one side,
  - all wagon rising,
  - longitudinal forces,
  - load combination.
- Dynamic evaluation:
  - modal analysis,
  - simulation of the ride through the S-curve,
  - geometric characteristics of wheelset and track,
  - acceleration, wheel forces, guiding forces and safety against derailment.

Investigated construction was subjected to static structural analysis (program ANSYS) based on the standards \([4, 5]\). On the basis of the 3D model of chassis frame was created calculation model which characteristics are shown in Fig. 5.

The task of structural analysis was to simulate behaviours of stresses and deformations in the proposed construction loaded maximum vertical forces.

The simulation analysis shows that the greatest deflection investigated structures (middle of the wagon) will be 29.92 mm. The report ERRI B12/RP17 states that the maximum deflection of the chassis frame must not exceed 3% of the distance between bogie pivots respectively axis of wheelsets. The results of the simulations show that the chassis frame satisfies strength conditions. For further development respectively production of this type of construction it is necessary to verify the results on the real construction on the test stand \([6, 7, 8]\).

The dynamics analysis was performed in accordance to the technical parameters for two track sections in Cerhenice railway test track circuit. In the first case was taken into account the track with 450 m radius and in

Fig. 4. Main parts of the chassis frame.

Fig. 5. Computational model – boundary conditions, loads and properties.
Fig. 6. Results of static simulation analysis – max. displacement and stress.
second case the right track and the velocity of 100 km/h.

The geometric characteristics [9, 10, 11] of railway wheelset and track (Equivalent conicity and Delta-R function) were evaluated for the profiles contact couples S1002 for wheel, UIC60 and R65 rail with inclinations of 1:20 and 1:40 (right track) and for the profiles contact couples S1002 for wheel, S49 rail with inclinations of 1:20 and 1:40 (radius R450 m).

The geometric characteristics analysis for both cases was completed with the Equivalent conicity and Delta R function characteristics (Fig. 7) set for different gauges values (in the interval of 1432-1448 mm).

For both analysis the accelerations, wheel vertical forces and leading lateral forces values, from them the derailment criterion (safety against derailment) was evaluated.

In both cases, the safety against derailment values, were in the interval of safe operation.

With this research and development subjects deals other publications too [12, 13, 14, 15].

5 Conclusion

It is estimated that this wagon will be a sustainable wagon that will be able to offer the same or even more capacity and utilize less resources and energy than current wagons for intermodal transport.

The analysis of the intermodal traffic has enabled to carry a simulation in which wagon capacity performance has been assessed. The primary results speak for an increase of capacity due to wagon use of about 10% in comparison to a realistic reference case. This capacity increase is mainly produced by a better arrangement of containers and by reduction of deadweight of the wagon. When it comes to energy consumption wagon could save up to 18% of the energy necessary to transport a units. This is mostly due to an improvement of the loading factor that enables a better compression of the containers, fewer gaps, a reduced number of axles and a lighter tare per units.

Main advantages over existing design are:
- lower loading plane = transport of containers and swap bodies all dimensions in kinematic gauge G1 (including HiCube containers and swap bodies with a wide of up to 2600 mm),
- distance between pivots to 18 000 mm = no special permissions.

Due to the reduced amount of axles, for same or even more units capacity, wagon will bring about important savings in maintenance and very importantly, it will reduce the noise emissions per train.

Fig. 7. Delta R function S1002/R65/1:40/gauge 1432 - 1448.
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