Experimental study on dynamic performance of pneumatic flexible manipulator with single degree of freedom

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Abstract. Aiming at the problem of insufficient transverse stiffness of flexible manipulator, a kind of pneumatic flexible manipulator with single degree of freedom is developed by using elongated pneumatic artificial muscle joint as the actuator. The structure and functional principle of the manipulator are introduced. The dynamic experiment of the manipulator was carried out by using the three-dimensional motion measurement system, and the dynamic characteristics of the manipulator under three different incentive signals (step, pulse and ramp) were analyzed, which provided a basis for the establishment of the manipulator control model in the later stage.

Keywords: pneumatic flexible manipulator with single degree of freedom, incentive signal, dynamic characteristic.

1. Introduction

In recent years, with the increasingly prominent problem of population aging, a large number of service robots are urgently needed in medical care, agriculture, family services and other industries [1, 2]. As a key executive component of robots, the mechanical structural characteristics, motion characteristics and driving ability of manipulator directly determine the robot's working ability [3]. At present, the structure, movement, control and other comprehensive flexibility of most manipulator are insufficient, so it is difficult to meet the safety, flexibility and adaptability requirements of service robots. At the same time, relevant theoretical research and product development lag behind, which seriously restricts the application and development of service robots [4, 5]. Compared with the rigid manipulator, the flexible manipulator has the characteristics of low energy consumption, light weight and high safety, which make the flexible manipulator widely used [6, 7].

In this paper, the dynamic performance experiments under different incentive signals are carried out on the experimental platform. The dynamic characteristics of the manipulator in three special deformation states are analyzed in order to adopt more practical control methods for different practical problems.

2. Structure and function of flexible manipulator

The flexible manipulator is composed of an upper and down cover, constrain components, latex tubes and a spring plate, which is shown in Fig. 1. Latex tube, constrain components and the plug form a good seal cavity called artificial muscle. The manipulator is equivalent to four artificial muscles in parallel leading to it with one degree of freedom so that it can swing left and right around spring plate. The restraint ring and the upper and lower covers are 3D printed. The restraint ring (Fig. 2) is provided with a 1mm convex platform, which is mainly used to increase the gap between the restraint rings and increase the deformation capacity of the manipulator.
When compressed gas is injected into the manipulator, the latex tube expands and deforms under pressure. Due to the external restraint ring limits the radial deformation of the latex tube, the upper cover is only subject to the axial force and resultant torque generated by the gas. Due to the large axial stiffness of spring plate, it can be assumed that the axial elongation is zero so that the manipulator only occur bending deformation. Therefore, when the 1 and 2 muscles are ventilated, the manipulator is bending in the direction of the 3 and 4. Similarly, when the 3 and 4 muscles are ventilated, the manipulator is bending in the direction of the 1 and 2 muscles (Fig. 3). It is obvious that we could control the bending direction and bending angle of the manipulator by adjusting the value of $P_1$.

**Fig. 1.** Three-dimensional model

**Fig. 2.** The structure of restraint ring

**Fig. 3.** The deformation mechanism of the manipulator

### 3. Dynamic experiments and analysis

The dynamic experiments of manipulator under different incentive signals were carried out by using the dynamic experimental device. The dynamic experimental device is composed of pneumatic control system, signal control system and 3d motion measurement system, which is shown in Fig. 4. During the experiment, the program of step, pulse and ramp was controlled by computer and PLC. Different air pressure was provided by the air pump to make the manipulator move. The 3D motion measurement system captured and tracked the mark points on the manipulator in order to get the position coordinates. The parameter of the joint is show in the Table 1.
Table 1. The parameter of the manipulator

| Parameter name                        | Value       |
|---------------------------------------|-------------|
| Total length of the manipulator       | 60 mm       |
| Mass of the manipulator               | 666.2 g     |
| Outer diameter of restrain ring       | 62 mm       |
| Thickness of restrain ring            | 3 mm        |
| Material of spring plate              | 65 MN       |
| Specification of spring plate         | 70×12×0.25 mm |
| Specification of latex tube           | Ø12×15 mm   |
| The elastic shear modulus of latex tube | 1.367 MPa  |
| The elastic shear modulus of spring   | 79 GPa      |

3.1. Response analysis under step signal

The step signal can be realized through on-off of the switching of the electromagnetic reversing valve. The time-domain response curve of step signal is shown in Fig. 5.

As can be seen from Fig. 5, the bending angle of the manipulator increases with the increase of the pressure. Due to the small damping and large inertia of the manipulator, the overshoot is large and the time required is also long when it reaches the equilibrium position.

3.2. Response analysis under pulse signal

The 1.0 s, 2.0 s and 4.0 s with different pulse width of incentive signals can be set by timer in
PLC respectively.

The time-domain response curve under different pulse width is shown in Fig. 6.

From Fig. 6(a) to Fig. 6(c), it can be seen that the stability of the manipulator is poor in the process of pressure relief and the response time to reach the steady state is long. The higher the pressure is, the more obvious the phenomenon is. The main reason is that the bending angle increases with the increase of pressure, and the inertia during pressure relief is relatively large, leading to large overshoot in the process of pressure relief of manipulator.

![Response curve under different pulse width](image)

**Fig. 6.** Response curve under different pulse width

### 3.3. Response analysis under ramp signal

By adjusting the electrical proportional valve, the ramp signals under different inflation rate can be realized. Fig. 7 is the response curve of different pressure with the same ramp signal. Fig. 8 is the response curve under different ramp signals with the same pressure. In Fig. 8, $T_0$ represents the time required for the pressure value to reach the specified value of 0.2 MPa at a certain inflation rate.

It can be seen from the comparison between Fig. 7 and Fig. 4 that the overshoot is smaller and the stability is better when the manipulator reaches the equilibrium position under the ramp signal. As can be seen from Fig. 8, the response time of the manipulator becomes shorter with the increase of the slope value when the steady state is reached.

### 3.4. Analysis of frequency domain response

The frequency domain characteristic curves of the manipulator under different pulse widths can be obtained by Fourier transform as shown in Fig. 9. The natural frequency data of the
manipulator under the same pressure under different pulse widths were averaged to obtain the curve of the manipulator's natural frequency changing with the pressure, which is shown in Fig. 10.

![Fig. 7. The response curve of different pressure with the same slope signal](image1)

![Fig. 8. The response curve of different slope signals with the same pressure](image2)

![Fig. 9. The frequency domain characteristic curves of the manipulator](image3)

It can be seen from Fig. 9 that the pulse width has a small effect on the inherent frequency of manipulator, and the air pressure has a large effect on it. The natural frequency of manipulator is mainly concentrated in the low frequency region, about 6 Hz. As can be seen from Fig. 10, with the increase of pressure, the rigidity of the overall structure becomes lower after the deformation of the manipulator, resulting in the decline of its inherent frequency.
4. Conclusions

A new kind of pneumatic flexible manipulator is developed, which has one degree of freedom and can only swing from side to side around the middle spring plate. The dynamic experiments of the manipulator were carried out by using the three-dimensional motion capture system, and the time-domain and frequency-domain responses of the manipulator under different incentive signals were obtained. After bending and deformation, the inherent frequency of the manipulator decreases with the increase of air pressure and it is mainly concentrated in the low-frequency region (about 6 Hz). By comparing the response curves under different signals, it can be concluded that the stability under ramp signals is better than the step signal and the response speed is adjustable, so it can be used as the actual control method of manipulator to reduce vibration impact.

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