Geometric Properties of Assur Graphs

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Abstract

In our previous paper, we presented the combinatorial theory for minimal isostatic pinned frameworks - Assur graphs - which arise in the analysis of mechanical linkages. In this paper we further explore the geometric properties of Assur graphs, with a focus on singular realizations which have static self-stresses. We provide a new geometric characterization of Assur graphs, based on special singular realizations. These singular positions are then related to dead-end positions in which an associated mechanism with an inserted driver will stop or jam.

1 Introduction

In our previous paper [13], we developed the combinatorial properties of a class of graphs which arise naturally in the analysis of minimal one-degree of freedom mechanisms in the plane with one driver, with one rigid piece designated at the ‘ground’. We defined an underlying isostatic graph (generically independent and rigid) formed by replacing the driver - the Assur graph, named after the Russian mechanical engineer who introduced and began the analysis of this class. Every other mechanism, which is independent (whose degree of freedom increases if we remove any bar) is formed by composing a partially ordered collection of \( k \) such Assur graphs (see Figure 5). The techniques of combinatorial rigidity provided an algorithm for decomposing an arbitrary linkage into these Assur components.

In this paper, we investigate the geometric properties (self-stresses and first-order motions) of such Assur graphs \( G \), when realized as a framework \( G(p) \) in special or singular positions \( p \). The properties of a full self-stress combined with a null motion relative to the ground, at selected singular positions provide an additional, geometric necessary and sufficient condition for Assur graphs (see §3).

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Geometric properties of Assur graphs are also important in terms of special positions reached by the mechanisms, when moving under forces applied through a driver (§4). These positions include configurations in which this driver is unable to force a continuing motion, because of the transmission difficulties, or reach ‘dead end’ positions in which the mechanism will ‘jam’ under the motion of the driver, and continued motion will require will need to be reversed.

As with the first paper, our initial motivation was to provide a more complete grounding and mathematical precision for some geometric observations and conjectures developed by Offer Shai, and presented at the Vienna workshop in April/May 2006. More generally, using the first-order and static theory of plane frameworks, we want to provide a careful mathematical description for the properties, observations and operations used by mechanical engineers in their practice. We also hope to develop new techniques and extensions in an ongoing collaboration between mathematicians and engineers.

2 Preliminaries

We will summarize some key results from the larger literature on rigidity [7, 21], and from our first paper [13]. Throughout this paper, we will assume that all frameworks are in the plane and we only consider rigidity in the plane.

2.1 Frameworks and the rigidity matrix

A plane framework is a graph \( G = (V, E) \) together with a configuration \( p \) of points for the vertices \( V \) in the Euclidean plane, with pairs of vertices sharing an edge in distinct positions. Together they are written \( G(p) \). A first-order motion of a framework \( G(p) \) is an assignment of plane vectors \( p' \) to the \( n = |V| \) vertices of \( G(p) \) such that, for each edge \((i, j)\) of \( G\):

\[
(p_i - p_j) \cdot (p'_i - p'_j) = 0. \tag{1}
\]

If the only first-order motions are trivial, that is, they arise from first-order translations or rotations of \( \mathbb{R}^2 \), then we say that the framework is infinitesimally rigid in the plane.

Equation (1) defines a system of linear equations, indexed by the edges \((i, j) \in E\), in the variables for the unknown velocities \( p'_i \) for the framework \( G(p) \). The rigidity matrix \( R_G(p) \) is the real \( E \) by \( 2n \) matrix of this system. As an example, we write out coordinates of \( p \) and of the rigidity matrix \( R_G(p) \), in the case \( n = 4 \) and the complete graph \( K_4 \).

\[
p = (p_1, p_2, p_3, p_4) = (p_{11}, p_{12}, p_{21}, p_{22}, p_{31}, p_{32}, p_{41}, p_{42}) = \left[
\begin{array}{cccccccc}
p_{11} - p_{21} & p_{12} - p_{22} & p_{21} - p_{11} & p_{22} - p_{12} & 0 & 0 & 0 & 0 \\
p_{11} - p_{31} & p_{12} - p_{32} & 0 & 0 & p_{31} - p_{11} & p_{32} - p_{12} & 0 & 0 \\
p_{11} - p_{41} & p_{12} - p_{42} & 0 & 0 & 0 & 0 & p_{41} - p_{11} & p_{42} - p_{12} \\
0 & 0 & p_{21} - p_{31} & p_{22} - p_{32} & p_{31} - p_{21} & p_{32} - p_{22} & 0 & 0 \\
0 & 0 & p_{21} - p_{41} & p_{22} - p_{42} & 0 & 0 & p_{41} - p_{21} & p_{42} - p_{22} \\
0 & 0 & 0 & 0 & p_{31} - p_{41} & p_{32} - p_{42} & p_{41} - p_{31} & p_{42} - p_{32}
\end{array}\right]
\]

2
A framework \((V,E,p)\), with at least one edge, is infinitesimally rigid (in dimension 2) if and only if the matrix of \(R_G(p)\) has rank \(2n - 3\). We say that the configuration on \(n\) vertices \(p\) is in generic position if the determinant of any submatrix of \(R_{K_i}(p)\) is zero only if it is identically equal to zero in the variables \(p_i\). For a generic configuration \(p\), linear dependence of the rows of \(R_G(p)\) is determined by the graph and the rigidity properties of a graph are the same for any generic embedding. A graph \(G\) on \(n\) vertices is generically rigid if the rank \(\rho\) of its rigidity matrix \(R_G(p)\) is \(2n - 3\), where \(R_G(p)\) is the submatrix of \(R(p)\) containing all rows corresponding to the edges of \(G\), for a generic configuration \(p\) for \(V\).

An first-order motion \(p'\) is a solution to the matrix equation \(R_G(p)p' = 0\), and first-order rigidity is studied through the column rank. We can also analyze the rigidity through the row rank of the rigidity matrix, or through the cokernel:

\[
[\Lambda ]_{R_G(p)} = 0
\]

Equivalently, these row dependencies \(\Lambda\) are assignments of scalars \(\lambda_{ij}\) to the edges such that at each vertex \(i\):

\[
\sum_{\{i,j\} \in E} \lambda_{ij}(p_i - p_j) = 0
\]  \(\quad (2)\)

These dependencies \(\Lambda\) are called static self-stresses, or self-stresses for short, in the language of structural engineering and mathematical rigidity, and the cokernel is the vector space of self-stresses. Equation (2) is also called the equilibrium condition, since the entries \(\lambda_{ij}(p_i - p_j)\) can be considered as forces applied to the vertex \(p_i\). Equation (2) then states that these forces are in local equilibrium at each vertex. Equivalently, a framework \(G(p)\) is independent if the only self-stress is the zero stress, and we see that framework with at least one bar is first-order rigid if and only if the space of self-stresses has dimension \(|E| - (2n - 3)| = 0\).

### 2.2 Isostatic graphs and rigidity circuits

A framework \(G(p)\) is isostatic if it is infinitesimally rigid and independent. There is a fundamental characterization of generically isostatic graphs (graphs that are isostatic when realized at generic configurations \(p\)).

**Theorem 1** (Laman [8]). A graph \(G = (V,E)\) has a realization \(p\) in the plane as an first-order rigid, independent framework \(G(p)\) if and only if \(G\) satisfies Laman’s conditions:

\[
|E| = 2|V| - 3
\]

\[
|F| \leq 2|V(F)| - 3 \text{ for all } F \subseteq E, F \neq \emptyset; \ldots (3)
\]

Such a graph is also generically rigid in the plane.

Minimally dependent sets, or circuits, are edge sets \(C\) satisfying \(|C| = 2|V(C)| - 2\) and every proper non-empty subset of \(C\) satisfies inequality (3). Note that these circuits, called rigidity circuits, always have an even number of edges.
If a rigidity circuit $C = (V, E)$ induces a planar graph, then a planar embedding of $C$ (with no crossing edges) has as many vertices as it has faces, which follows immediately from Euler’s formula for planar embeddings ($|V| - |E| + |F| = 2$). The geometric dual graph $C^d$ is also a rigidity circuit, see [12, 3]. (The vertices of $C^d$ are the faces of the embedded graph $C$ and two vertices of $C^d$ are adjacent if the corresponding faces of $C$ share an edge, see Figure 1).

![Diagram of a planar graph and its geometric dual](image)

Figure 1: The geometric dual $G^*$ (red) of a planar graph $G$ (black)

In our previous paper [13], we presented an overview theorem which presented an inductive construction of all rigidity circuits from $K_4$ by a simple sequence of steps, along with some other properties of circuits.

2.3 Reciprocal diagrams

In this paper, we will use a classical geometric method for analyzing self-stresses in planar frameworks (frameworks on planar graphs): the reciprocal diagram [4, 5]. This construction has a rich literature in structural engineering, beginning with the work of James Clerk Maxell [9] and continuing with the work of Cremona [6]. This technique has been revived in the last 25 years as a valuable technique for visualizing the behaviour of such frameworks [4, 5], in specific geometric realizations, as well as for the study of mechanisms [16, 17]. We describe this construction and some key properties in the remainder of this subsection. An example is worked out in Figure 2.

Given a framework $G(p)$ with a non-zero self-stress $\Lambda$ (Figure 2 (a)-(c)), there is a geometric way to verify the vertex equilibrium conditions of Equation 2. If the forces $\lambda_{ij}(p_i - p_j)$ are drawn end to end, as a polygonal path, then

$$\sum_{\{j \mid \{i,j\} \in E\}} \lambda_{ij}(p_i - p_j) = 0$$

if, and only if, the path closes to a polygon (classically called the polygon of forces, Figure 2 (d), (e)). If we start with a self-stress on a planar drawing of a planar graph $G$, then we can cycle clockwise through the edges at a vertex in the order of the edges, creating a polygon for the original vertex, and a vertex for each of the ‘faces’ (regions) of the drawing. When we create polygons for
each of the original vertices, we note that the two ends of each original bar use opposite vectors (Figure 2 (b), (c)): \( \lambda_{ij}(p_i - p_j) \) and \( \lambda_{ij}(p_j - p_i) \). We can then patch these polygons together at each dual vertex (original region of the drawing) (Figure 2 (e), (f)). Overall, there is a question whether all of these local polygons patch together into a global drawing of the dual graph of the planar drawing? If we started with a self-stress in a planar graph, the answer is yes [4, 5]! As just described, this is a reciprocal figure as studied by Cremona [6], with original edges parallel to the edges in the reciprocal drawing. Note that only the edges with a non-zero self-stress \( \lambda_{ij} \) will have non-zero length in the reciprocal. If the entire reciprocal is turned 90°, then we have the reciprocal figures as presented by Maxwell [9, 4, 5].

Conversely, we can start with a configuration for a planar graph \( G(p) \) and a configuration for the dual graph \( G_d(q) \) with edges in \( G(p) \) parallel to the edges in \( G_d(q) \): a reciprocal pair. We can then use the lengths of the dual edge to define the scalars for a self-stress, by solving for \( \lambda_{ij} \) in the equation:

\[
\lambda_{ij}(p_i - p_j) = (q_h - q_k)
\]

where edge \( hk \) is dual to the edge \( ij \). Overall, the existence of the reciprocal also implies the existence of a self-stress and this self-stress would recreate the
reciprocal. Note that, in this presentation each of the drawings is a reciprocal of the other. That is, each side corresponds to a self-stress of the other side of the pair.

The following theorem summarizes some key properties of frameworks and their reciprocals. We translate the results of [4, 5] into equivalent statements of the form we will use in §3.

**Theorem 2** (Crapo and Whiteley [5]). Given a planar framework $G(p)$ with a self-stress, and a reciprocal diagram $G^d(q)$, there are isomorphisms between:

1. the vector space of self stresses of the reciprocal framework $G^d(q)$;
2. the space of frameworks $G(p||)$ with this reciprocal (with one fixed vertex); and
3. the space of parallel drawings $G(p||)$ with one vertex fixed (equivalently, first-order motions with one vertex fixed) of the framework $G(p)$.

Reciprocal diagrams are particularly nice for rigidity circuits, as they exist for all generic realizations. They were studied in a much broader context in [4, 19] as well as in the context of linkages in [17]. We will return to them in the proof of Theorems 7 and 8.

### 2.4 Isostatic pinned frameworks

Given a framework, we are interested in its internal motions, not the trivial ones, so we ‘pin’ the framework by prescribing, for example, the coordinates of the endpoints of an edge, or in general by fixing the position of the vertices of some rigid subgraph. Alternatively, we take some rigid subgraph (a single bar or an isostatic block) which we make into the ‘ground’ and fix all its vertices which connect to the rest of the graph. We call these vertices with fixed positions pinned, the others unpinned, free, or inner. Edges between pinned vertices are irrelevant to the analysis of a pinned framework. We will denote a pinned graph by $G(I, P; E)$, where $I$ is the set of inner vertices, $P$ is the set of pinned vertices, and $E$ is the set of edges, and each edge has at least one endpoint in $I$.

A pinned graph $G(I, P; E)$ is said to satisfy the Pinned Framework Conditions if $|E| = 2|I|$ and for all subgraphs $G'(I', P'; E')$ the following conditions hold:

1. $|E'| \leq 2|I'|$ if $|P'| \geq 2$,
2. $|E'| = 2|I'| - 1$ if $|P'| = 1$,
3. $|E'| = 2|I'| - 3$ if $P' = \emptyset$.

We call a pinned graph $G(I, P; E)$ (pinned) isostatic if $\overline{G}(V, E \cup F)$ is isostatic as an unpinned graph, where $V \supset I \cup P$, no $F$ has any vertex from $I$ as endpoint and the restriction of $\overline{G}$ to $P = V \setminus I$ is rigid. In other words, we can “replace” the pinned vertex set by an isostatic graph containing the pins and
call \( G(I, P; E) \) isostatic, if this replacement graph on the pins produces an (unpinned) isostatic graph \( \tilde{G} \). Which isostatic framework we choose, and whether there are additional vertices there, is not relevant to either the combinatorics in the first paper or the geometry in this second paper. In [13] we proved the following result:

**Theorem 3.** Given a pinned graph \( G(I, P; E) \), the following are equivalent:

(i) There exists an isostatic realization of \( G \).

(ii) The Pinned Framework Conditions are satisfied.

(iii) For all placements \( P \) of \( P \) with at least two distinct locations and all generic positions of \( I \) the resulting pinned framework is isostatic.

2.5 Combinatorial characterizations of Assur graphs

In our previous paper [13], we proved the equivalence of a series of combinatorial properties which became the definition of an Assur graph.

**Theorem 4.** Assume \( G = (I, P; E) \) is a pinned isostatic graph. Then the following are equivalent:

(i) \( G = (I, P; E) \) is minimal as a pinned isostatic graph: that is for all proper subsets of vertices \( I' \cup P' \), \( I' \cup P' \) induces a pinned subgraph \( G' = (I' \cup P', E') \) with \( |E'| \leq 2|I'| - 1 \).

(ii) If the set \( P \) is contracted to a single vertex \( p^* \), inducing the unpinned graph \( G^* \) with edge set \( E \), then \( G^* \) is a rigidity circuit.

(iii) Either the graph has a single inner vertex of degree 2 or each time we delete a vertex, the resulting pinned graph has a motion of all inner vertices (in generic position).

(iv) Deletion of any edge from \( G \) results in a pinned graph that has a motion of all inner vertices (in generic position).

An Assur graph is a pinned graph satisfying one of these four equivalent conditions. Some examples of Assur graphs are drawn in Figure 3 and their corresponding rigidity circuits in Figure 4.

![Figure 3: Assur graphs](image)

We also demonstrated a decomposition theorem for all isostatic pinned frameworks.

**Theorem 5.** A pinned graph is isostatic if and only if it decomposes into Assur components. The decomposition into Assur components is unique.
The decomposition process described in the proof of Theorem 5 of [13] induces a partial order on the Assur components of an isostatic graph and this partial order in turn can be used to re-assemble the graph from its Assur components. This partial order can be represented in an ‘Assur scheme’ as in Figure 5.

Figure 5: An isostatics pinned framework a) has a unique decomposition into Assur graphs b) which is represented by a partial order or Assur scheme c).

3 Singular Realizations of Assur Graphs

We now show that these Assur graphs have an additional geometric property at selected special positions, and that this geometric property becomes another equivalent characterization. From the general analysis of frameworks, we know that the positions $p$ of such Assur graphs such that $G(p)$ is not isostatic are the solutions to a polynomial Pure Condition [19]. For an pinned isostatic framework, this pure condition is created by inserting distinct variables for the coordinates of the vertices (including the pinned vertices) and taking the determinant of the square $|E| \times 2|V|$ rigidity matrix. We are particularly interested in the typical solutions to the pure condition (the regular points of the associated algebraic variety). These properties are related to the behavior of the associated linkage when it reaches a ‘dead-end’ position and locks [10, 11] (see §4, Corollary 1 below).
3.1 A sufficient condition from stresses and motions

We first show that given a singular realization $G(p)$ with a special 1-dimensional space of self-stresses and 1-dimensional space of first-order motions, $G$ must be an Assur graph. This is based on an observation of Offer Shai.

**Theorem 6.** Assume a pinned graph $G$ has a realization $p$ such that

1. $G(p)$ has a unique (up to scalar) self-stress $\Lambda$ which is non-zero on all edges; and

2. $G(p)$ has a unique (up to scalar) first-order motion, and this is non-zero on all inner vertices;

then $G$ is an Assur graph.

**Proof.** First we show that $G$ is an isostatic pinned graph. For a pinned graph with inner vertices $V$, and any realization $p$:

$$|E| - 2|V| = \dim(\text{Stresses}[G(p)]) - \dim(\text{First-order Motions}[G(p)]).$$

Since $\dim(\text{Stresses}[G(p)]) - \dim(\text{First-order Motions}[G(p)]) = 1 - 1 = 0$, we know that $|E| = 2|V|$.

Similarly, assume there is some subgraph $G'$ (unpinned, or pinned with one vertex, or pinned with two vertices) which is generically dependent. This will always have a non-zero internal self-stress in $G(p)$ - which is zero outside of this subframework $G'(p)$. Therefore, this unique self-stress cannot be non-zero on all edges. This contradiction implies the overall graph $G$ is isostatic.

Now we assume that the graph $G$ is not an Assur graph. Therefore it can be decomposed into a base Assur graph $G_A$, and the rest of the vertices and edges $G_1$. With this decomposition, we can sort the vertices and edges of $G_A$, to the end of the indices for the rigidity matrix to give the rigidity matrix a block upper diagonal form. Since we have a first-order motion, non zero on all the inner vertices of $G_A$, we have the equation.

$$\begin{bmatrix} R_1(p) & R_{1A}(p) \\ 0 & R_A(p) \end{bmatrix} \begin{bmatrix} u_1 \\ u_A \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

which implies the equation for $G_A$: $[R_A(p)]u_A = 0$, with $[u_A] \neq 0$. Since $G_A$ is an Assur graph (generically isostatic), $[R_A(p)]$ has a row dependence. Equivalently, there is a self-stress $\Lambda_A[R_A(p)] = 0$. This is also a self-stress on the whole framework $G(p)$, which is zero on all edges in $G_1$. Since we assumed $G(p)$ had a 1-dimensional space of self-stresses, this contradicts the assumption that there is a self stress non-zero on all edges.

We conclude that $G$ is an Assur graph. $\square$

In the next two sections we prove that this condition is also necessary, completing this geometric characterization of the Assur graphs.
3.2 Stressed realizations of planar Assur graphs

Since this is a geometric theorem, we need to use some key geometric techniques for stresses and motions of frameworks $G(p)$. We begin with the special subclass of planar Assur graphs $G$, where we can use the techniques of reciprocal diagrams [4, 15].

**Theorem 7.** If we have a planar Assur graph $G$ then we have a configuration $p$, such that:

1. $G(p)$ has a one-dimensional space of self-stresses, and this self-stress is non-zero on all edges; and
2. there is a unique (up to scalar) non-trivial first-order motion of $G(p)$ and this is non-zero on all inner vertices.

![Figure 6: The sequence of steps for producing the configuration for a planar Assur graph which has both a non-zero self-stress and a non-zero motion: (a) take a generic realization of the underlying circuit and form its reciprocal; (b) Split the reciprocal face $K$ in order to generate a self-stress that will separate the ground vertex in the original into predescribed distinct ground vertices (c), still with a self-stress; (d) use a second self-stress to form a parallel drawing; (e) use this parallel drawing to create difference vectors; and (f) turn these difference vectors to create the first-order motion which is non-zero on all inner vertices.](image)

**Proof.** We will use property (i) from the combinatorial characterization Theorem 4: $G$ is a minimal isostatic pinned framework.

We assume that the graphs $G$ and $G^*$ (with the pinned vertices identified) are planar. Since $G^*$ is a planar circuit, it has a dual graph $G^{**d}$ which is also
a planar circuit (Figure 6 (a)). We take a generic realization of this dual graph \( G^d(q^*) \), which will have a non-zero self-stress \( \Lambda^* \) which is non-zero on all edges, and the graph will be first-order rigid. We have the corresponding reciprocal diagram \( G^*(p^*) \) which also is first-order rigid and has a self-stress non-zero on all edges, by the general theory of reciprocal diagrams (§2.3 and [4]).

![Diagram showing the process](image)

Figure 7: Given an Assur graph \( G \) we identify the pins to \( k \) which has a dual polygon \( K \) (a,b,c). We triangulate the ground (d) which gives a dual trivalent tree \( T \) (e). This tree is inserted into \( K \) (f) giving an additional self-stress whose reciprocal gives back the triangulated ground and and separates the pins (f,b).

We will now modify this pair to split up the indentified ‘ground vertex’ \( k \) while maintaining the self-stress and introducing a first-order motion \( p' \) which is non-zero on all vertices not in the ground. This process is illustrated in Figure 7.

For simplicity, we create this ground for the pinned vertices as an isostatic triangulation on the pinned vertices. In the extreme case where we have only two ground vertices, we are adding one edge - and this appears as a corresponding added dual edge \( T \) in the reciprocal. More generally, we take the original graph \( G \) with \( m \geq 3 \) pinned vertices, and topologically add an isostatic framework of triangles in place of the ‘ground’ to create the extended framework \( \hat{G} \), with the dual graph \( \hat{G}^d \). (For uniformity, we can take a path connecting the pinned vertices \( p_1, \ldots, p_m \) and then connect \( p_1 \) to each of the remaining vertices. This will be such a generically isostatic triangulation, see Figure 7 (d).)

If there were \( m \) pinned vertices, then we add \( 2m - 3 \) edges to create the triangulation, and create \( t = m - 2 \) triangles. In the dual \( \hat{G}^d \), this adds a 3-valent tree \( T \) with interior vertices for each of the triangles Figure 7 (e), and leaves attached to the vertices of the reciprocal polygon \( K \) (Figure 7 (f)). Transferring the counts to the reciprocal, we have added \( t \) vertices and \( 2t + 1 \) edges into the dual polygon \( K \).

Since we added \( 2t + 1 \) edges and \( t \) vertices to a generically rigid framework \( G^d(q^*) \), we have added an additional self-stress if all the vertices are in generic position \( q \). This added self-stress is non-zero on some of these added edges. Because the inserted graph is a 3-valent tree, if the self-stress is non-zero on one
edge, then resolution at any interior vertex in general position requires it to be non-zero on all edges at this vertex. In short, the added self-stress is non-zero on all edges in the tree.

This is now a realization of \( \hat{G}^d \) - the dual to the original pinned graph with an isostatic triangulated ground (Figure 6 (b)). In the two dimensional space of self-stresses in the dual, adding a small multiple of the new self-stress to the original \( \Lambda^* \) on \( \hat{G}^d(q^*) \) (with zero on the added edges) gives a self-stress \( \Lambda \) on \( \hat{G}^d(q) \) non-zero on all edges. The reciprocal of this self-stress is the desired realization \( \hat{G}(p) \) of the original pinned framework with a triangulated (isostatic) ground (Figure 6 (c)). Since the self-stress on \( \hat{G}^d(q) \) is non-zero on all edges, all edges are of non-zero length in \( \hat{G}(p) \) by the basic properties of reciprocals. Moreover, since all edges of \( \hat{G}^d(q) \) have non-zero length, all edges in \( \hat{G}(p) \) have non-zero self-stress. With the added sub-framework \( D \) replaced by the ground, this is the realization \( G(p) \) required for condition (i).

It remains to prove that this also satisfies condition (ii): there is a non-trivial first-order motion with all inner vertices having non-zero velocities while the ground has zero velocities.

If we add an additional small multiple of the non-zero self-stress \( \Lambda^* \) (extended with zeros on in the added tree in \( K \)) to \( \Lambda \), then we have a second self-stress \( \Lambda^v \) which is the same on the edges interior to \( K \) but different on all other edges. Taking a second drawing reciprocal to \( \Lambda^v \) will give a second drawing \( \hat{G}(p||) \) which is identical on the pinned vertices and the ground triangulation, but moves all other edges to new positions with different lengths than in \( \hat{G}(p) \) (because of the different self-stress on these edges) (Figure 6 (d)). This is a parallel drawing of \( \hat{G}(p) \). In particular, all of the edges of the reciprocal polygon \( K \) have different stresses, so the edges from inner vertices to the pinned vertices in \( \hat{G}(p||) \) all have different lengths (Figure 6 (d)). We can take the differences in positions \( (p|| - p) \) as parallel drawing vectors \( u \) (Figure 6 (e)). By general arguments, involving the 90 degree rotation of the ‘parallel drawing vectors’ [4,14], these parallel drawing vectors \( u \) convert the first-order motion \( v = u^\perp \) of \( \hat{G}(p) \) which is zero on the ground and non-zero on all the inner vertices (Figure 6 (e,f)). This completes the proof that \( G(p) \) satisfies condition (ii).

\[ \square \]

3.3 Extension to non-planar Assur graphs

We have the desired converse result for all planar Assur graphs. In order to extend this to singular realizations of all Assur graphs, we turn to another 19th century technique for converting a singular realization of a general framework \( G(p) \) into a singular realization of a related planar framework without actual crossings of edges, using the same locations for the original vertices plus the crossing points [10]. This technique is named after the American structural engineer Bow who introduced it to assist in the analysis of any plane drawing of a framework, in which the visible regions and the edges separating them became the pieces for the analysis of the framework via a reciprocal diagram.
Theorem 8. If we have an arbitrary Assur graph \( G \) then we have a configuration \( p \), such that

1. \( G(p) \) has a single self stress, and this self-stress is non-zero on all edges; and

2. there is a unique (up to scalar) non-trivial first-order motion of \( G(p) \) and this is non-zero on all inner vertices.

Figure 8: Given a non-planar graph (or drawing) (a) we can insert crossing points to create a planar graph (Bow’s Notation). Working on this planar graph we have a reciprocal (b) which also is a non-planar drawing of the planar reciprocal (c).

Proof. We already know the result for planar Assur graphs. The key step for non-planar graphs is the classical method called ‘Bow’s notation’. Given a non-planar framework realizing a graph \( G \), we select pairs of edges with transversal crossings, and insert those vertices, splitting the two edges, creating a new graph \( G_b \) (Figure 8). (Note that these ‘crossings’ do not have to be at internal points of the segments - just not at vertices of the segments. The ‘crossings are identified topologically, but the added vertices are geometrically on the points of intersection of the two infinite lines.) The general theorem is that the two frameworks have isomorphic spaces of self-stresses, and first-order motions.

With this technique in mind, we can sketch a plane drawing of the final graph we want, with the ground triangulation isolated with no crossings. This sketch identifies the crossing points to be added, within the identified circuit - the ‘Bowed framework’. Take a generic realization of the identified circuit \( G^* \). Add the crossing points as identified, to create a ‘planar graph’ needed for the reciprocal diagrams \( G^*_b(p) \). Create a reciprocal diagram \( G^*_b(q) \) for this planar framework. In this reciprocal, the identified framework, the duals of the ‘crossing points’ appear as parallelograms.

We now continue with the planar process, as outlined in the previous proof. With the addition of the vertices and edges to split the ‘dual face’ \( K \) for the
ground in the dual, we create a stressed framework, and an extended reciprocal which has the graph of the Bowed framework. Moreover, since the dual graph is realized with parallelograms dual to the vertices added in the Bowed framework, the crossings involve transversal crossings with the required $x$-appearance for later removal. This framework will have a self-stress which is non-zero on all edges and a non-trivial motion which is non-zero on all inner vertices. Moreover, this Bowed framework and the framework with the crossing points removed, have the isomorphic spaces of self-stresses and infinitesimal motions. In particular, a self-stress which is non-zero on all edges of the Bowed framework is non-zero on all edges of the original graph, and the first-order velocities of the original graph are exactly those velocities assigned at these vertices within the Bowed framework.

We have created the required configuration for the original (non-planar) graph with a self-stress non-zero on all edges, and a first-order motion which is zero on the ground and non-zero on all free vertices.

![Figure 9:](image)

Figure 9: Given the reciprocal pair, we can again split the face $K$ (b) and split the ground vertex in the original. This configuration of the Bowed graph has a non-zero stress, as does the non-planar original. The non-trivial parallel drawing of the Bowed graph is a non-trivial parallel drawing of the non-planar original (c) and induces the required first-order motion on the non-planar original (d,e).
3.4 Extensions to other singular realizations

With some special effort, and careful attention to some geometric details, it is possible to extend the previous result to show the existence of such a special configuration \( p \) which extends any initial configuration of the ground vertices as distinct points. Without giving all the details, the idea is to form an isostatic triangulation on the ground vertices as positioned, which in turn gives an appropriate dual tree \( t \) with dual edges for the triangles which are on the boundary of the ground, still as rays. The ‘ground’ polygon \( K \) is then placed on these rays, in general position. It remains to see that the rest of the dual graph \( G^{*d} \) can be realized with this initial polygon and a unique self-stress, non-zero on all edges. Because we started with a generic circuit, this can be accomplished by using some details about the ‘polynomial pure conditions’ of these graphs [19], and the occurrence of the remaining vertices in these polynomial conditions.

There is also a conjectured generalization of the result above to any realization of an Assur graph with a non-zero self-stress on all edges. The key seems to be not to insist that every vertex has a non-zero velocity, but to relax this to ask that every bar has at least one of its vertices with a non-zero velocity. The reader can review the proof in §3.1 to see that this is the condition we actually used in the sufficiency condition.

**Conjecture 1.** Assume we have an Assur graph \( G \) and a realization \( p \) such that there is a single self stress which is non-zero on all edges. Then there is a unique (up to scalar) non-trivial first-order motion and this is non-zero on at least one end of each bar.

4 Inserting Drivers into Assur Graphs

For simplicity, we will assume that our graph \( G \) is generically independent in the plane. In a 1 DOF linkage \( G(p) \) at an independent realization, a driver \( d \) is either

(i) a piston \( ab \) which changes the distance between the pair \( ab \) where \( ab \) this distance is changing during the 1 DOF motion;

(ii) an angle driver which changes the angle \( \angle abc \) between two bars \( ab, bc \) where this angle is changing during during the 1 DOF motion.

More generally, if we have a driver \( d \) in an independent 1 DOF linkage \( G(p) \), this driver will cause a finite motion in some independent realizations, and we continue to call this a piston or angle driver even in singular positions of the same 1 DOF graph as a linkage. We will discuss such singular positions in §3.3.

4.1 Replacing drivers

In the previous paper [13], we created the isostatic framework from a 1 DOF linkage by ‘replacing the driver’ to remove the degree of freedom. To return to
a 1 DOF linkage from an Assur graph, we can ‘insert a driver’. So far, we have used quotation marks here, because we find there are several alternatives for the process of replacing the driver, and converse operations to insert a driver because the processes are not yet defined - though mechanical engineering practice can guide us. We begin by defining one clear process for ‘replacing a driver’ with an added bar.

A simple method to remove the 1 DOF is to insert one bar in a way that blocks the single degree of freedom (Figure 10). Specifically, given a one-degree of freedom linkage $G(p)$ at a generic configuration:

1. to replace a piston $ab$, we insert the bar $ab$;
2. to replace an angle driver on $\angle abc$, at an internal vertex $b$ of degree $\geq 3$, we insert the bar $ac$;
3. to replace an angle driver on $\angle abc$, at an internal vertex $b$ of degree $2$, we insert the bar $ac$ and remove vertex $b$;
4. to replace an angle driver on an angle $\angle ap_ip_j$ where $p_ip_j$ are pinned vertices, we add $a$ as a pinned vertex.

This driver replacement creates a pinned graph $\overline{G}$ and a pinned framework $\overline{G}(p)$.

![Figure 10: There are four types of drivers: a) driving a distance $ab$ with a piston; b) driving an interior angle $\angle abc$; c) driving an angle at a pin $\angle ap_ip_j$, and the special type illustrated in Figure 11. Below these are the ways in which each of these drivers is replaced to create an isostatic graph by d) an added edge (for a piston); e) an added angle brace; or f) an added pinned vertex resulting from adding an angle brace to the ground.]

In the key example of our previous paper [13] Figures 1,2, we actually replaced the pistons in two steps:
1. we replaced the piston with a 2-valent vertex attached to the ends (which is mechanically equivalent); and

2. we contracted one of these edges to form the single edge which was inserted above.

A similar process could be applied to any angle driver, and the net effect would be the edge insertion presented above (Figure 11).

Figure 11: As an alternative to the simple insertions, we can replace each driver by a 2-valent vertex $o$ (a,b,c), and then contract $o$ to one of the end vertices, creating the same result (d,e,f) as the previous insertion.

The driver is active at a specific position $G(p)$ if it is possible, infinitesimally, to change the length of the bar we are adding without changing the lengths of any of the other bars. Specifically, in the rigidity matrix of $G$, with the added bar $d$ at the bottom: $R_{G}(p)p' = (0, \ldots, 0, s_d)^{tr}$ has a solution $p'$ for all possible values of the strain $s_d$ (instantaneous change in length) of $d$.

More generally, we claim $G(p)$ is isostatic, and $G$ is generically isostatic, provided that $G(p)$ is independent and the driver was active.

**Theorem 9.** Given an independent 1 DOF linkage $G$ with an active driver $d$, the driver replacement $\overline{G}$ is an isostatic pinned framework.

**Proof.** Consider an independent realization $G(q)$. There is a 1 dimensional vector space of non-trivial first-order motions $v'$, with the pinned vertices fixed
(which extends to a finite motion by general principles of algebraic geometry).

For a piston $ab$, the added bar $i,j$ is independent if, and only if, the added bar is on a pair $i,j$ with a non-zero strain:

$$(q_i - q_j) \cdot (v_i - v_j) \neq 0$$

The definitions of a piston driver the added bar has this required property, so inserting this bar blocks the motion, at first-order.

Similarly, for an angle driver $\angle abc$ at an interior vertex $b$, adding the bar $ac$ is also independent, generating an isostatic framework.

If we were replacing an angle driver at a 2-valent vertex, then with the added bar this vertex is attached to an isostatistic subframework with just two non-collinear bars. This vertex can be removed to leave an isostatic framework on the remaining vertices. This is done to prevent the appearance of an extra ‘Assur component’ in the derived isostatic graph and focus the analysis on the behaviour of the rest of the graph.

If we were replacing an angle driver $\angle ap_i P_j$ at the ground, then inserting the bar $ap_j$ will create an isostatic framework. It will also pin the vertex $a$ to the ground, artifically creating an extra Assur component. To assist the analysis of the original mechanism, we just pin the vertex $a$ and analyze the modified pinned framework.

We can speak of a 1 DOF graph $G$ with a driver $d$ as an Assur mechanism, if replacing the driver creates an Assur graph $\overline{G}$.

### 4.2 Inserting a Driver

Conversely, we can start with an Assur graph, and insert a driver using one of the three steps:

(i) Remove a bar $ab$ and insert a piston $ab$;

(ii) Remove an edge $ac$ which is in a triangle $abc$ and insert an angle driver on the angle $\angle abc$;

(iv) remove an edge $ac$ and insert a new 2-valent vertex $b$, with bars $ab, bc$ and an angle driver on the angle $\angle abc$;

(iv) if there are at least three pinned vertices $p_i, p_j, p_k$, make a pinned vertex $p_k$ into an inner vertex $a$, with a single bar to one of the other pinned vertices $p_i$ and an angle driver on the angle $\angle ap_i p_j$.

These operations are the reverse operations of the four ways of replacing a driver.

As a generic operation, we know that this driver insertion takes an isostatic framework to a 1 DOF framework. We now show that for an Assur graph, the driver insertion will create a 1 DOF framework with all inner vertices in motion relative to the ground.
Theorem 10. If we have an Assur graph $G$, realized as an independent framework $G(p)$ with the pins not all collinear, and we insert a driver as above, then the framework has 1 DOF, with all inner vertices in motion, and activating the driver will extend this to a continuous path.

Proof. The original framework is an independent pinned framework with $|E| = 2|V|$. For insertions (i) (ii), we have removed one bar, so there is 1 DOF. Since this is an Assur graph, the Characterization Theorem 4 (iii) guarantees all inner vertices have a non-zero velocity.

Moreover, the non-trivial first-order motion will have a non-zero strain on the pair of the removed bar. If we inserted a piston, this will change this length and drive the motion. If we inserted an angle driver on a triangle $abc$, then driving the angle will change the length $ac$ and thus drive the motion.

Finally, if we changed a pin to an interior vertex, then we can assume that the vertex being made an inner vertex is not collinear with two of the other pinned vertices. We can assume that is vertex is 2-valent in the isostatic ground framework, so that making it inner leaves an isostatic ground, and creates a new inner vertex $a$ attached to the ground by two edges $ap_i, ap_j$. Removing $ap_j$ gives a 1 DOF linkage, as required. In the resulting motion, $a$ will have a non-zero velocity, so driving the angle $\angle ap_i p_j$ will drive this 1 DOF. It remains to check that all other inner vertices are have non-zero velocities. If some inner vertex $h$ has the zero velocity, then it is attached to the remaining ground through an isostatic subframework. This would mean $h$ is contained in an isostatic pinned subframework which does not include $a$. This is a contradiction of our assumption that $G$ was Assur.

Driver insertion and driver replacement are inverses of one-another. That is, if we start with an Assur graph and insert a driver, then replacing the driver will return us to the same Assur graph. Conversely, if we start with a driver, and replace it, then we can choose to re-insert the same driver and return to the same 1 DOF linkage. (There is a choice in the insertion, one of which is corresponds to the original replacement. )

However, it is now clear that we have:

(i) as many ways to insert a piston as we have interior bars;
(ii) three times as many ways to insert an angle driver as we have interior triangles;
(iii) as many ways to insert a 2-valent angle driver as there are interior bars;
(iv) as many ways to insert a pinned angle driver as we have pins, provided there are at least two pins.

In short, there are a lot of 1 DOF linkage with drivers which come from the same underlying Assur graph. All of these will be Assur mechanisms.

Conversely, if we have a 1 DOF linkage, we can identify a number of pairs whose distances are change, and angles which are changing. Each of these could be used to insert a driver. However, different insertions will lead, after replacement, to different graphs $\bar{G}$. One may be an Assur graph while another may not. Figure [12] gives such an example.
4.3 Singular positions with a driver

The geometric question is: can we find positions at which this still has a self-stress? If we do, is it possible that some vertices must have zero velocities relative to the ground? Or more generally is it possible that the first-order motion does not continue in the same direction as the original motion?

We recall that if first-order motion $p'$ is a first-order motion, then $-p'$ is also a first-order motion. If both of these velocities extend to a finite motion, then we say the driver has a finite motion in both directions. As a contrapositive of the Theorem 10, we have the following corollary.

**Corollary 1.** If the linkage $G_d(p)$ with the driver does not have a finite motion in both directions, then the linkage with the driver insert has a self-stress.

Such configurations without a finite motion (continuing in both directions) are called ‘dead ends’ in the literature of linkages [13]. For example, the existence of a dead-end with an angle driver at the ground requires a self-stress with the driver joint pinned, so the original isostatic graph was realized in a singular position. As another example, if the driver is a piston, the driver edge is part of this singular position, so that its line applies the ‘ground force’ required for the self-stress of the isostatic graph. That an independent 1 DOF linkage can move under a driver to such a singular position (with the driver replacement), is one reason why we have investigated the occurrence of such singular positions with a self-stress on all members (including the edge used to replace the driver). That all the inner vertices have non-zero velocities at that singular position indicates that we could have moved into the self-stress position in the driven motion.

It is not true that every self-stress gives a dead end position, just that dead end positions require the self-stress. The study of such configurations is the subject of a recent paper of Rudi Penne [11]. That study focuses on centers of motion, rather than self-stresses, and it is well-understood in the literature that these are equivalent tools for many purposes, each giving its own insight into the geometry and the combinatorics of linkages.
5 Concluding comments

Working with several drivers. In the previous paper, we presented a decomposition of a general isostatic pinned framework into Assur components. With such a decomposition, we could insert one driver into all, or some, of the components, creating a larger mechanism with as many degrees of freedom as the drivers inserted. See, for example, the mechanism in Figures 1 and 2 of [13]. These drivers will be independent, in the sense that each of them could be given distinct instantaneous driving instructions without any interference or instability.

We could also insert several drivers into a single Assur graph extra analysis will be needed to ensure that these are independent. More generally, given a mechanism with a number of drivers, their ‘independence’ is equivalent to whether replacing all the drivers produces a graph which is isostatic.

Projective geometry for self-stresses. The instantaneous kinematics and statics of plane frameworks are projectively invariant. Thus the singular position of a graph $G(p)$ can be transferred to any projective image of the configuration $p$. In particular, we have seen that it is common in mechanical engineering to include pistons (also called ‘slide joints’ in structural engineering). These pistons are actually mechanically equivalent to ‘joints at infinity’ between the two ends of the slide [4] - and therefore are incorporated in the geometric (and combinatorial) theory we have described in these two papers.

We also note that spherical mechanisms (with joints built as pines pointed to the center of the sphere), share the same projective geometry as the plane mechanisms. As this suggests, all of the combinatorial and geometric methods and results presented in our two papers extend immediately from the plane to the spherical frameworks.

In animating a one degree of freedom mechanism in computer science, it is common to find that a single link cannot be taken as the ‘driver’ which completes a circle while preserving all of the edges of the mechanism. Somewhere along the path, the linkage will experience a self-stress. Some of these singular positions will ‘jam’, others will not. The analysis of the singular positions, helps clarify this situation. However the decision of which ‘new driver’ to pick to move points along the subject of another study.

Extensions to 3-D. In the conclusion of our earlier paper [13], we indicated that the combinatorial results do have appropriate generalizations in 3-space, at least for some ‘nice classes’ of structures, such as bar and body structures.

These will still have drivers and will have special geometry for their dependencies and for dead-end positions. The specific geometric theorems given are conjectured to also extend, but we will need new methods, because techniques such as ‘reciprocal diagrams’ are limited to plane structures.
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