Evaluation of Aerodynamic Forces over a Vertical Axis Wind Turbine Blade through CFD analysis

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Abstract

In the current age of global energy crisis, the production of energy through alternate energy resources has gained a significant attention. Wind as a source of energy is a very attractive due to the fact that fuel is free of cost in this case. This research is about the aerodynamic design of a VAWT blade using the analytical and CFD techniques for a small scale vertical axis wind turbine (VAWT), aiming 1 kW power output which may be use for domestic purposes to power a single room. The blade design parameters and dimensions are taken aiming the required power output and analytical models are developed to evaluate the aerodynamic forces like lift and drag over the surface of the blade. These forces which are very helpful for the evaluation of the structural integrity of the VAWT blade are then found to be in a close agreement with CFD results which are simulated using commercial software, ANSYS 13.0. The static CFD model is developed at a selected pitch angle during a complete 360° where the aerodynamic forces evaluated are comparable with the analytical values at the similar location.

Keywords: SB-VAWT; Aerodynamic forces; Design parameters

Introduction

VAWTs has certain advantage over HAWTs which makes it favorable for installation in urban areas, like it gives a better response to turbulent wind flows when installed on buildings. Improving the wind turbine performance through wind tunnel testing and theoretical studies is expensive and consumes too much time. Aerodynamics blade analysis problems can be addressed by CFD simulations software which offers inexpensive solutions. Two dimensional airfoil models have been analyzed using ANSYS-FLOUENT software and values of lift and drag coefficients obtained were compared with published calibrated values. The performance of small scale horizontal axis wind turbines have also been predicted using three dimensional models. The inherent unsteady aerodynamic behavior of VAWTs is because of variation in angle of attack with pitch angle, viz. also the angle of rotation (θ) and apparent velocity. This complexity of unsteady aerodynamics makes VAWTs very attractive to be analyzed by CFD models. The designed parameters including No. of blades, type of turbine, solidity and tip speed ratio have been evaluated for VAWTs using computational and experimental studies, in previous studies. K-ε turbulence and multiple reference frame (MRF) models have been chosen to perform the transient simulations. CFD and experimental analyses may help to improve the performance of VAWTs, studied the performance of VAWTs by changing design parameters with fixed pitch angle using CFD simulations and wind tunnel experiments. Due to the three dimensional operation of VAWT, its aerodynamic performance is complex simulate. Different prediction tools for complex shaped VAWT performance evaluation were presented like integrating CAD and CFD existing software for solutions. Also, the mathematical models such as double multiple streamtube (DMS) models were discussed. All these VAWT solutions that already exist in market are lacking of technology innovation which may be due to the high complex aerodynamic prediction of VAWTs. The dynamic stall for two dimensional flows around VAWT blades have also been investigated using CFD techniques and different turbulence models were compared with available data. Attempts have also been made to study the flow physics of hybrid three bladed Darrieus-Savonius wind turbines using FLUENT. SolidWorks and Star CCM+ have also been used for the development of geometry and carrying out the CFD analysis of wind turbines, respectively. The most effective airfoils were chosen on the basis of two dimensional and three dimensional analyses at different angles of attack and speeds in order to determine when flow separation takes place. VAWT’s performance and aerodynamic efficiency was confirmed using CFD analysis by designing a high efficiency 500W composite VAWT blade, applicable in low speed regions of Korea.

This research is about the development of solid and CFD models of blade for small scale VAWT using Pro Engineer and ANSYS 13.0 CFX, respectively.

The technique is used to evaluate the aerodynamic forces like lift and drag using CFD analysis. These values are compared with the analytical values calculated in earlier research on the same VAWT blade.

The only 2D analysis of this blade would not produce the same values as calculated using analytical modeling because during the calculation of total forces along and normal to the chord length, the value of span length plays an important role. Increasing the span length increases the total force along each direction. Therefore, a complete 3D model is required to be analyzed in order to verify the results obtained in analytical modeling.

Design constraints

The value of wind speed is taken as 8m/s for the purpose of analysis, which is an average value at the areas where this wind turbine will be installed. The values of design parameters and blade dimensions are summarized in the table 1.

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Received November 22, 2012; Accepted December 30, 2012; Published January 03, 2013

Citation: Hameed MS, Shahid F (2012) Evaluation of Aerodynamic Forces over a Vertical Axis Wind Turbine Blade through CFD analysis. J Appl Mech Eng 1:116

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Considering the importance of thickness of airfoil for a high value aspect ratio blade and the aerodynamic characteristics like life and drag coefficients, NACA 0015 was selected as the more suitable airfoil among the recommended family of airfoils, NACA 00XX.

### Evaluation of aerodynamic forces through CFD analysis

The results obtained from the analytical analysis of the aerodynamic forces [9], are verified in this section using computation fluid dynamic (CFD) analysis.

### Construction of blade geometry

A commercial software (Pro Engineer) is used for the construction of solid model of designed blade using the dimensions calculated in table 1. The key points as input are entered to generate cross section (NACA0015 airfoil) of the blade. These key points are joined through spline curve it will form the shape of perfect symmetric blade with well defines leading and trailing edge, the cross section is then extruded to the designed value of blade span length. The blade and other components of turbine including disc, strips, bolts and shaft are model and assembled. A view of this design blade and a complete solid model of the 3 straight bladed VAWT are shown in figure 1 and figure 2.

### Generation of mesh

The blade solid model is imported to ICEM (ANSYS 13.0) in .iges format, (Figure 3). The transformation of geometry is done by offsetting the points from blade geometry to develop the far field. These offset points are joined by lines and then surfaces are created between those lines (Figure 4). These surfaces are then extruded in up and down direction which completely enclosed the blade (Figure 5). The geometry is divided into parts i.e. inlet, outlet, blade left, blade right so it get easy to mesh the whole geometry properly and later to study each part result separately.

### Blocking

The geometry is joined to a series of blocks which make the actual volume so that the calculation will take place on those volumes or parts. Whole mesh geometry is selected for blocking (Figure 6).

The blocking allows turning off the certain surfaces which are not required in order to reduce the complexity of the mesh. The block is also spitted and associated with the blade surface in order to attach the mesh with the blade surface and reshape the mesh according to the contours of the blade.

### Table 1: Selection of Blade Design Parameters.

| Design Parameter            | Relationships used for Calculations | Given/Recommended Range for VAWT | Selected/Calculated Value |
|-----------------------------|-------------------------------------|----------------------------------|---------------------------|
| Total Number of Blades (n)  | -                                   | 3 or more                        | 3                         |
| Tip Speed Ratio (TSR)       | $\lambda_{tip} = \frac{r}{V}$      | 4 - 6                            |                           |
| Solidity                   | Solidity $= \frac{c}{D}$            | 0.1 - 0.25                       | 0.24                      |
| Power Coefficient (Cp)      | Solidity and TSR Curve              | 0.45 Max.                        | 0.43                      |
| Swept Area (s)              | $P = 0.5C_{pm}v^3$                  | -                                | 6.76 m²                   |
| Diameter of the Turbine (D)=Span Length (b) | Considering the Square Box | -                                | 2.58 m                    |
| Aspect Ratio (AR)           | -                                   | -                                | 12.5                      |
| Chord Length (c)            | AR = b/c                            | -                                | -                         |

### Figures

- **Figure 1**: Solid Model of VAWT Blade.
- **Figure 2**: Solid Model of the Designed VAWT.
- **Figure 3**: Blade model in ANSYS imported from Pro-E.
- **Figure 4**: Creation of far field around blade for CFD analysis.
Clustering

The clustering is performed to further refine the mesh close to the blade surfaces in order to achieve the improved results near the blade boundary/surface (Figure 10). This fine mesh is achieved by increasing the number of nodes (or distance between them).

Grid resolution

The grid quality is important and helpful for the reduction of computational errors. Bigeometric mesh law is considered to generate this grid. The size control for this grid shows that every line in the far field is divided into 25 parts and lines along blade cross section and span are divided in 20 parts. The total number of cells the complete grid are 863404.

Writing mesh for CFX PRE

After clustering, the unstructured mesh is saved as blocking. The mesh is reloaded from blocking and regenerated then it is written to an input file for CFX PRE. This CFX file is made readable in ICEM mesh. Now, this file is reloaded in CFX PRE and primitive regions are inserted defining inlet, outlet and blade regions for CFX PRE.

Numerical simulation and turbulence model: The simulations are performed in ANSYS CFX 13.0 using Shear Stress Transport (SST) k-ω turbulence model incorporating the Computational Finite Volume method for solution. Heat transfer by total energy is considered to make it a compressible fluid model in CFX-Pre.

Setting up the domain

In the airfoil domain basic settings, the material is set as Air Ideal Gas and reference pressure as 56.867kPa. For compressible analysis, the heat transfer option is set to total energy. Viscous turbulence effects are also including by including the viscous work term in the domain.

Boundary conditions

The outlet relative pressure of 0Pa is applied relative to the operating pressure of 56.867 kPa, as

\[ \text{Absolute Pressure} = \text{Reference Pressure} + \text{Relative Pressure} \]

The flow regime is subsonic and free stream velocity is taken as 8m/s
expressed as \( U_{\text{inf}} \) in software expressions. This free stream velocity is divided into rectangular components along x and y directions whereas, component of velocity along z axis is taken as zero. Static temperature is chosen as 298K. The boundary conditions are inserted as blade and boundary type is set as wall and following expressions are used. The analysis is carried out at a single value of angle of attack, i.e. 8.36 degree.

In the convergence criteria the residual type is selected as root mean square (RMS) and the residual target is set as 10-6

\[ \text{AOA} = 8.36[^\circ] = \text{Angle of attack} \]

\[ U_x = U_{\text{inf}} \cdot \cos(\text{AOA}) = \text{Flow velocity along x axis} \]

\[ U_y = U_{\text{inf}} \cdot \sin(\text{AOA}) = \text{Flow velocity along y axis} \]

\[ F_y = \text{force}_y(\text{@Blade}) = \text{Normal force on blade} \]

\[ F_x = \text{force}_x(\text{@Blade}) = \text{Axial force on blade} \]

\[ \text{Lift} = \cos(\text{AOA}) \cdot F_y - \sin(\text{AOA}) \cdot F_x = \text{Lift force} \]

\[ \text{Drag} = \cos(\text{AOA}) \cdot F_x + \sin(\text{AOA}) \cdot F_y = \text{Drag force} \]

\[ \text{Denom} = 0.5 \cdot \text{massFlowAve} \cdot \text{Density}(\text{Inlet}) \cdot U_{\text{inf}}^2 \cdot 0.206[^\text{m}] \cdot 2.58[^\text{m}] \]

\[ \text{CL} = \frac{\text{Lift}}{\text{Denom}} = \text{Coefficient of Lift} \]

\[ \text{CD} = \frac{\text{Drag}}{\text{Denom}} = \text{Coefficient of Drag} \]

Solver control

In the solver control tree, the maximum iterations are set to 2000 and timescale factor is 1 to 10 as, a large scale factor can accelerate the convergence but too large timescale may cause the solver to fail. The residual target is set to 1-6. The case is now ready to run for solution.

Findings of research

After, all the boundary conditions are applied and solution is completely, a result file is formed which can be open in post CFX for the evaluation of results.

1) Results obtained from CFD analysis are compared with the analytical solution of the same problem, which was done in the earlier research [9], the comparison of results is summarized in table 2.

2) The figure 11 shows higher values of velocity on the upper region of the blade, as the blade shown here is upside down (showing y axis downwards).

3) The higher values of velocity on upper regions make reduction in pressure in the same regions, which can be seen in figure 12. The higher pressure in lower region and lower pressure on upper region; the difference between both creates the lift force in the blade, which drives the turbine.

4) The 3D isometric view of the complete blade after analysis can also be seen in figure 13.

5) During the analytical analysis of the same blade the values of lift, drag, normal and tangential (axial) forces were calculated [9] at different angle of attacks during the complete rotation of the blade.

6) In numerical analysis, the results are evaluated at a random value of angle of attack and are compared with the analytical results.

7) At 8.36° angle of attack, the results for both analytical and numerical values are shown in table 2 and a good agreement between the results is found.

8) When CFD results are compared with analytical results, the maximum percentage error is in the case of drag where it is about 9.46%, which is comparable with previous research on the similar blades which shows that this error can rise up to 10% when compared with the experimental results [3]. As CFD technique is numerical approximation so this error can be further reduced by improving the grid resolution.

Conclusions

The CFD models can be generated for the evaluation of aerodynamic forces like lift and drag over VAWT blades. These models and numerical
methods can be used in order to avoid using expensive experimental setups like wind tunnels, for the evaluation of aerodynamic forces over these blades. In the process of analysis, the generation of mesh around the blade is a complex procedure and must be done carefully. If the mesh is not generated properly then there will be a large amount of error and even a small error in the mesh will cause the solution to never converge over the flow region. Solution takes time to converge and results may only be evaluated at a single value of angle of attack during a single run. To get more accurate results increase the discretization of the geometry. While blocking make sure that the required area get blocked otherwise it will affect final results.

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