Power-to-heat for renewable energy integration: A review of technologies, modeling approaches, and flexibility potentials

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HIGHLIGHTS

- We review model-based analyses of residential power-to-heat options.
- We compare and categorize research scopes, methods, and findings.
- We identify state-of-the-art analytical model formulations.
- Findings: fossil fuel substitution, renewable integration, decarbonization.
- Heat pumps and passive thermal storage are particularly favorable options.

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ABSTRACT

A flexible coupling of power and heat sectors can contribute to both renewable energy integration and decarbonization. We present a literature review of model-based analyses in this field, focusing on residential heating. We compare geographical and temporal research scopes and identify state-of-the-art analytical model formulations, particularly considering heat pumps and thermal storage. While numerical findings are idiosyncratic to specific assumptions, a synthesis of results indicates that power-to-heat technologies can cost-effectively contribute to fossil fuel substitution, renewable integration, and decarbonization. Heat pumps and passive thermal storage emerge as particularly favorable options.

1. Introduction

Not only since the 2015 Paris Agreement [1], there is widespread consensus that the use of renewable energy sources will play a major role in the global response to the threat of climate change. In particular, increasing shares of variable renewable energy sources such as wind and solar power have to be integrated in different end-use sectors. In this context, the coupling of power and heat sectors receives increasing attention of researchers and policymakers alike. Compared to other flexibility options and sector coupling strategies, linking the power and heat sectors is often considered to be particularly promising because both the costs of generating heat from electricity and the costs of heat storage are relatively low [2]. Flexibly using renewable electricity for heating purposes may (i) help to decarbonize the heat sector and (ii) contribute to the power system integration of variable renewables by providing additional flexibility.

In many industrialized countries, decarbonizing the heating sector is a precondition for achieving ambitious climate policy targets; in particular, space heating accounts for substantial fractions of final energy demand and greenhouse gas emissions [3]. Compared to the electricity sector, the utilization of renewable energy sources lags behind in the heat sectors in many countries. For example, Germany is often considered as an international front-runner with respect to the utilization of wind and solar energy [4]. By the end of 2016, the share of renewables in gross power consumption was nearly 32% in Germany, up from around 3% in the early 1990s. In contrast, the renewable share in final energy demand for heating and cooling was only around 13% in 2016, up from 2% in 1990 [5]. Comparable developments can be observed in other industrialized countries.

The integration of variable renewable energy sources requires additional flexibility in the power system as the feed-in patterns of wind and solar power are only partly correlated with electricity demand.
There are many ways of providing such flexibility, for example, flexible thermal generators, various forms of energy storage, \textsuperscript{1} demand-side measures, grid-connected electric vehicles, geographical balancing facilitated by transmission, as well as changes in design, siting, and dispatch of variable renewables \cite{10}. While generating heat from electricity was traditionally not a preferred option in fossil fuel-based power systems, the flexible use of electricity for heating purposes, often combined with heat storage, has recently received increasing attention as another – and particularly promising – source of system flexibility \cite{11}.

While the benefits and challenges of power-to-heat options in power systems with high shares of variable renewable energy sources are beginning to be understood, the literature is still heterogeneous: existing power system and market models have been extended, and new models have been developed. Applications focus on various geographical contexts, time horizons, and technologies. To consolidate the evidence at hand and lay out avenues for future research, we devise a structured account of model-based analyses of different power-to-heat options in the international peer-reviewed literature. In particular, we compare scopes, methodologies, and research questions and aim to synthesize some common findings.

In doing so, we focus on power system effects of power-to-heat technologies in the residential heating sector and largely exclude industrial heat applications. We further focus on power-to-heat options, that is, turning electric into thermal energy, and not on the combined generation of heat and power (CHP). We do not consider other sector coupling strategies, for example, interactions between electric vehicle batteries and the power system \cite{12}, or conversion paths like power-to-gas or power-to-liquids \cite{13–15}.

The remainder of this article is structured as follows. In Section 2, we categorize different power-to-heat options, that is, different approaches of using electricity in the residential heating sector. Section 3 introduces the methodology of our literature review. In Section 4, we discuss the research scope of model-based power-to-heat analyses in the international literature. Section 5 compares methodological approaches and introduces analytical model formulations of heat pumps and heat storage. In Section 6, we synthesize research questions and findings with respect to, among others, cost effectiveness, integration of variable renewables, and decarbonization. The final sections concludes, connects to some market trends, and hints toward future research directions.

2. Residential power-to-heat options

There are different means to convert electricity into heat. Fig. 1 categorizes the most important options for the residential heating sector.

Following the categorization provided in Fig. 1, we first distinguish between centralized and decentralized power-to-heat options. Under the centralized approach, electricity is converted into heat at a location that may be distant to the point of actual heat demand, and (district) heating networks are used to distribute the heat to where it is needed \cite{16}. In contrast, decentralized power-to-heat options make use of electricity right at, or very close to, the location of heat demand. In reality, the line between centralized and decentralized options is blurred as, for example, heat may be jointly provided for only a few flats or houses in local or neighborhood heating networks \cite{17}.

Second, some power-to-heat options involve thermal energy storage while others do not. Centralized options always come with some extent of storage because district heating networks have a certain thermal storage capacity \cite{18}. A heating network’s storage capability may be further increased with dedicated (central) thermal storage facilities which, depending on the storage size, may also allow for seasonal storage. Decentralized options, in contrast, may come without energy storage, which we refer to as direct heating. Other decentralized options are combined with thermal energy storage (TES), referred to as storage heating or TES-coupled heating. Such thermal energy storage may be either internal or external with respect to the actual power-to-heat element. An example for internal thermal energy storage are electric storage heaters, which store thermal energy in a well-insulated solid medium such as ceramic bricks. If such systems are equipped with advanced communication and control equipment, they are also referred to as “smart” electric thermal storage \cite{19}. An example for external thermal energy storage are hot water storage elements of standard water-based residential heating systems. Aside from such active thermal

\textsuperscript{1} A dedicated review of electricity storage requirements for renewable energy integration is provided by \cite{9}.
storage, which allows for controlled charging and discharging, there is also the option of passive thermal storage (not depicted in Fig. 1). Here, thermal energy is stored in the building mass or the interior and released in a non-controlled way \[20–22\].

Within these high-level categories, different technologies can be distinguished, among them various kinds of heat pumps\(^2\) and resistive heaters. Centralized power-to-heat approaches either draw on large-scale heat pumps that make use of geothermal (i.e. ground-sourced) energy, waste heat or brine, or on large electric boilers, often in the form of electrode boilers. In general, these options are also available in the group of decentralized heating options coupled with external thermal energy storage. Here, smaller-scale heat pumps are usually air- or ground-sourced. Resistive heating comes in the form of electric boilers or electric heating elements in boilers that are primarily fueled by some other energy carrier such as natural gas. An example for the latter, which is also referred to as hybrid heating, is a water-based residential heating system with a boiler that is primarily fueled with natural gas and has an additional electric heating element \[25\].

Particularly in the heat pump literature, a related classification distinguishes between monovalent, mono-energetic, and bivalent approaches \[24\]. For example, a monovalent system consists only of a heat pump that is designed to cover the full heating energy demand in all hours of the year. In a mono-energetic system, a heat pump may be complemented by an electric heating element, which allows for smaller heat pump dimensioning. Yet the energy source – i.e., electricity – does not change. In contrast, bivalent systems draw on two heating options with different energy carriers. An example for the latter is a heat pump in combination with a fossil-fueled backup boiler.

More indirect ways of electric heating, such as the conversion of electricity to hydrogen or methane which may then fuel a boiler, are not depicted in Fig. 1 and are also not considered further in this review. Likewise, Fig. 1 focuses on residential space heating and does not include details on domestic hot water provision. Yet most of the depicted options, with the exception of (smart) electric thermal storage, may also be used to provide hot water.

Fig. 2 illustrates the interconnection of different power-to-heat options with electricity and district heating networks. Centralized power-to-heat technologies draw electricity from the grid to generate heat, using either large-scale heat pumps or electric boilers. Heat energy is then transported to residential customers. In contrast, decentralized power-to-heat options do not make use of heating networks. Fig. 2 also indicates that most power-to-heat options involve some energy storage capability.\(^3\) From an energy system point of view, interactions between different kinds of heat storage and electricity storage technologies are of particular interest.

### 3. Methodology of this literature review

We conducted a systematic literature search of model-based analyses in leading peer-reviewed journals of applied and economic energy research. To this end, we first screened the journals *Applied Energy*, *Energy*, *Energy Economics*, *Energy Policy*, and *Renewable and Sustainable Energy Reviews* for the keywords *electric boiler*, *electric heating*, *electric thermal storage*, *heat pump*, and *power-to-heat*. A search in the Web of Science Database resulted in a total number of 721 articles that appeared between 2007 and 2016 in these journals. The keyword *heat pump* lead to the most hits, and *electric thermal storage* to the least hits. The number of articles featuring one or more of the keywords substantially increased between 2007 and 2016 as Fig. 3 illustrates. This may be interpreted as an increasing academic relevance of power-to-heat analyses. Yet the overall number of articles appearing in these journals also grew. The share of keyword articles in overall articles, thus, increased only moderately from 1.2% in 2007 to 2.6% in 2016.

Within the retrieved articles, we carried out both forward and backward searches to identify relevant papers that the articles cite and are cited by. Thus, the scope of journals broadened to include further articles.

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\(^2\) A much-cited review on heat pumps is provided by Chua et al. \[23\], a more recent one by Fischer and Madani \[24\].

\(^3\) In addition, there may be a passive thermal energy storage capacity related to the building mass. This is not depicted in Fig. 2.
with very high shares of renewables in electricity and heating sectors which will have adequately evolved instead of legacy systems shaped by currently installed capacities. Accordingly, many papers assume a renewables share of 40–60% or higher. However, a number of studies do either not explicitly state the share of renewable energy sources or – if they do – do not specify the renewable technologies.

The range of heating technologies considered in the analyses is broad. Centralized heating may be provided by CHP plants, heat pumps, resistive heating or any combination of these technologies, often combined with heat storage. The papers differ greatly with respect to the level of technological detail; many applications represent technologies in a rather stylized way. Decentralized heating is likewise analyzed for a broad range of different technologies. Many analyses implement some stylized model of heat pumps or resistive electric heaters.5 Hybrid heating technologies, explicitly considered in [38,66], and smart electric thermal storage (SETS), explicitly considered in [29,34,37,46], are not in the focus of most papers. A large share of models also employs some kind of decentralized heat storage to shift heating energy in time, often differentiated into passive and active storage.

Beyond electricity and heating, not many papers take further sectors into account; some applications explicitly model the mobility sector [29,35,48,52,57,67] or the cooling sector [35,57]. While interactions between the electricity and heating sectors can thus be conveniently focused on, broadening the scope to include further sectors, which are likewise subject to de-carbonization, could provide complementary insights.

Some model analyses comprise additional features such as the provision of control power [33,52,57] and an explicit consideration of behavioral incentives for households [65]. Also here, implementing more markets or objectives could render a more detailed picture of the dual challenge of decarbonizing the energy sector(s) and providing the necessary flexibility.

5. Research methods and modeling approaches

5.1. Overview

Most publications considered in this review are techno-economic

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4 Some of these appeared as journal articles in 2017.
| Paper | Geographic scope | Time horizon | RES shares | Centralized heat | Decentralized heat | Other sectors | Other features |
|-------|------------------|--------------|------------|------------------|-------------------|--------------|---------------|
| Arteconi et al. 2016 | [27] Belgium (stylized) | 2030 | 30% | Resistive / hybrid / SETS vehicles, cooling, H2 | Passive, DHW | – | – |
| Bach et al. 2016 | [28] Copenhagen | 2013, 2015 | n/s | ✓ | ✓ | Passive, DHW, SETS | Vehicles |
| Barton et al. 2013 | [29] UK | 2010–2050 | n/s | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | Passive, DHW, SETS | Vehicles |
| Bauer et al. 2014 | [30] Germany | 2010–2050 | n/s | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | Passive, DHW, active | – |
| Blärke 2012 | [31] West Denmark | 2003–2010 | 20% | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | – | – |
| Böttger et al. 2014 | [32] Germany | 2015–2030 | n/s | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | – | – |
| Böttger et al. 2015 | [33] Germany | 2012, 2025 | 23%, 54% | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | – | – |
| Bach et al. 2016 | [34] Copenhagen | 2009–2020 | 0, 40% | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | Passive, SETS | – |
| Connolly et al. 2016 | [35] EU–28 | 2050 | Up to 100% | – | ✓ ✓ ✓ ✓ | Passive, DHW, active | – |
| Cooper et al. 2016 | [36] UK | 2030s, 2030s, 2050s | 14, 25, 38% | – | ✓ ✓ ✓ ✓ | Passive, DHW | – |
| Dodds 2014 | [37] UK | 2010–2050 | n/s | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | – | – |
| Ehrlich et al. 2015 | [38] Germany | 2020 | n/s | – | ✓ ✓ ✓ ✓ | Active | – |
| Fehr and Et al. 2014 | [39] Ireland | 2030 | 6 GW wind | – | ✓ ✓ ✓ ✓ | Passive, DHW, active | – |
| Georges et al. 2017 | [40] Ireland | 2010 | 29% | – | ✓ ✓ ✓ ✓ | Passive, DHW, active | – |
| Hedegaard et al. 2012 | [41] Denmark | 2030 | 60% | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | Passive, DHW, active | – |
| Hedegaard, Balysk 2013 | [42] Denmark | 2030 | Up to 100% | – | ✓ ✓ ✓ ✓ | Passive, DHW, active | – |
| Hedegaard, Münster 2013 | [43] Denmark | 2030 | 50–60% | – | ✓ ✓ ✓ ✓ | Passive, (DHW), active | – |
| Henning, Palzer 2014 | [44,45] Germany | n/s | Up to 100% | – | ✓ ✓ ✓ ✓ | Passive, (DHW), active | – |
| Hughes 2010 | [46] Prince Edward Island, Canada | 2002–2003 | 5.15 MW wind power | – | ✓ ✓ ✓ ✓ | – | – |
| Kikerud et al. 2014 | [47] Norway, Sweden | 2010–2012 | Historical | – | ✓ ✓ ✓ ✓ | – | – |
| Kivlihoja, Melbom 2010 | [48] Finland | 2035 | 8–29% | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | – | Vehicules |
| Li et al. 2016 | [49] Stylized urban | n/s | n/s | – | ✓ ✓ ✓ ✓ | – | – |
| Liu et al. 2016 | [50] Beijing, Tianjin, Hebei | 2015 | n/s | – | ✓ ✓ ✓ ✓ | – | – |
| Lund et al. 2010 | [51] Denmark | 2020, 2040, 2060 | Up to 100% | ✓ ✓ ✓ ✓ | – | ✓ ✓ ✓ ✓ | Cooling, ind. heat, transp., H2 | – |
| Mathiesen, Lund 2009 | [52] Denmark | 2030 | Up to 100% | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | Active | – |
| Merkel et al. 2014 | [53] Germany | 2015–2050 | Endogenous | ✓ ✓ ✓ ✓ | ✓ ✓ ✓ ✓ | Active | – |
| Merkel et al. 2017 | [54] Germany | 2015–2050 | 60% (obs., heat) | ✓ ✓ ✓ ✓ | – | ✓ ✓ ✓ ✓ | Active | – |
| Münster et al. 2012 | [55] Denmark | 2025 | n/s | ✓ ✓ ✓ ✓ | – | ✓ ✓ ✓ ✓ | – | – |
| Nielsen et al. 2016 | [56] Copenhagen | Short-term scenarios | n/s | ✓ ✓ ✓ ✓ | – | ✓ ✓ ✓ ✓ | – | – |
| Östergaard et al. 2010 | [57] Aalborg | 2007, 2050 | 100% | – | ✓ ✓ ✓ ✓ | – | – |
| Östergaard, Andersen 2016 | [58] Denmark | 2014 | n/s | – | ✓ ✓ ✓ ✓ | – | – |
| Östergaard, Lund 2011 | [59] Frederikshavn “Long-term” | 2010 | 100% | – | ✓ ✓ ✓ ✓ | – | – |
| Papadopopulos et al. 2012 | [60] Germany | 2020, 2030 | 36, 47% | – | ✓ ✓ ✓ ✓ | Passive, DHW | – |
| Patteeuw et al. 2013 | [61] Belgium (stylized) | n/s | – | – | – | – | – |
| Patteeuw et al. 2015 | [62] Belgium (stylized) | 2030 | 40% | – | – | – | – |
| Patteeuw et al. 2015 | [63] Belgium (stylized) | 2030 | 40% | – | – | – | – |
| Patteeuw, Heisen 2016 | [64] Belgium | n/s | 10–100% | – | – | – | – |
| Patteeuw, Heisen 2016 | [65] Belgium | 2013 | 8–40% | – | – | – | – |
| Patteeuw et al. 2016 | [66] Belgium | n/s | 95% | – | – | – | – |
| Petrović, Karlson 2016 | [67] Denmark | 2010–2050 | 50% wind | – | ✓ ✓ ✓ ✓ | – | – |

(continued on next page)
Table 1 (continued)

| Heat storage | Other sectors | Other features |
|--------------|---------------|---------------|
| Centralized heat | Decentralized heat | Heat pumps /mCHP |
| || Heat pumps /mCHP |
| RES shares | Time horizon | Geographic scope |
| --- | --- | --- |
| Salpakari et al. 2016 | 2013-2015, 2050 | Helsinki 2013–2015, 2050 |
| Teng et al. 2016 | 2030, 2050 | United Kingdom 2030, 2050 |
| Waite, Modi 2014 | n/s a | New York (City) |

Notes:
Check marks indicate central consideration of the technology, parentheses a secondary consideration.
Abbreviations:
DHW: domestic hot water; mCHP: micro combined heat and power; n/s: not specified; SETS: smart electric thermal storage; transp.: transportation.

a [64] draw on input data of 2013–2016; [66] 1999–2002; [71] 2005–2016.

5.2. Formulations for modeling heat pumps

5.2.1. Coefficient of performance

Eq. (1a) is the basic approach to represent electrical heat pumps. It assumes a constant relation between power input $P_i$ and heat output $Q_{i,HP}$, the average coefficient of performance (COP) $COP_{avg}$, at any point in time $t$ [29,31,34,39,41,43,45,48,56,61,63,67]. This

\[ COP_{avg} = \frac{Q_{i,HP}}{P_i} \]
Table 2

Research method.

| Paper                  | General method      | Type of program | Model name             | Time resolution          | Endogenous investments | P2H Heat storage |
|-----------------------|---------------------|-----------------|------------------------|--------------------------|-------------------------|------------------|
| Artz et al. 2016      | Cost minimization   | MILP            | BALMOREL               | Hourly, one year         | –                       | (✓)              |
| Bach et al. 2016      | Cost minimization   | LP              | FESA                   | Hourly, one year         | –                       | –                |
| Barton et al. 2013    | Scenario assessment | n/a             | E2M2                   | Time steps (days, hours) | –                       | –                |
| Bauer et al. 2014     | Cost minimization   | Stochastic LPs  | HeatSYM                | Power, heat              | –                       | –                |
| Beller 2015           | Cost minimization   | MILP            | COMISE                  | Hourly, one year         | –                       | –                |
| Böttger et al. 2016   | Cost minimization   | MILP            | TIMES                   | Hourly, one year         | –                       | –                |
| Chen et al. 2015      | Minimization of residual demand variability | MILP | EnergyPLAN | Hourly, one year | – | – |
| Connolly et al. 2016  | Simulation          | n/a             | REMod-D                | Hourly, one year         | –                       | –                |
| Cooper et al. 2016    | Simulation          | n/a             | PowerFys               | Hourly, one year         | –                       | –                |
| Dodds 2014            | Iterative heuristic calibration | n/a | REMod-D | Hourly, one year | – | – |
| Ehrlich et al. 2015   | Simulation          | n/a             | REMod-C                | Hourly, one year         | –                       | –                |
| Feurenbach et al. 2014| Simulation          | n/a             | REMod-D                | Hourly, one year         | –                       | –                |
| Georgescu et al. 2014 | Flexibility maximization | MILP | E2M2                   | Power, heat              | –                       | –                |
| Hedegaard, 2012       | Simulation          | n/a             | EnergyPLAN             | Hourly, one year         | –                       | –                |
| Hedegaard, 2013       | Cost minimization   | LP              | BALMOREL               | Hourly, 4 weeks          | –                       | –                |
| Heinen, 2017          | Iterative heuristic calibration | n/a | REMod-D | Hourly, one year | – | – |
| Henning, 2017         | Simulation          | n/a             | REMod-D                | Hourly, one year         | –                       | –                |
| Hughes 2010           | Cost minimization   | LP              | BALMOREL               | 1768 time slices, 52 weeks | –                       | –                |
| Kivivuoma, 2010       | Simulation          | n/a             | TIMES                  | Hourly, 26 weeks         | –                       | –                |
| Li et al. 2016        | Cost minimization   | LP              | BALMOREL               | Hourly, one year         | –                       | –                |
| Liu et al. 2016       | Welfare maximization| LP              | TIMES                  | Hourly, one year         | –                       | –                |
| Lund et al. 2010      | Simulation          | n/a             | REMod-C                | Hourly, one year         | –                       | –                |
| Merkel et al. 2014    | Cost minimization   | MILP            | TIMES-HEAT-POWER       | Heat                     | –                       | –                |
| Münster et al. 2012   | Cost minimization   | LP              | BALMOREL               | Hourly, 4 weeks          | –                       | –                |
| Nielsen et al. 2016   | Cost minimization   | MILP            | PowerFys               | Hourly, one year         | –                       | –                |
| Östergaard et al. 2010| Simulation          | n/a             | REMod-D                | Hourly, 4 weeks          | –                       | –                |
| Östergaard, Andersen 2016 | Dispatch simulation  | n/a | REMod-D | Hourly, one year | – | – |
| Östergaard, Lund 2011 | Simulation          | n/a             | PowerFys               | Hourly, one year         | –                       | –                |
| Papamichalou et al. 2012 | Cost minimization | Stochastic     | PowerFys               | Hourly, one year         | –                       | –                |
| Patteeuw et al. 2015  | Cost minimization   | MILP            | –                      | Hourly, 48 h             | –                       | –                |
| Patteeuw et al. 2015  | Cost minimization   | MILP            | –                      | Hourly, 48 h             | –                       | –                |
| Patteeuw et al. 2015  | Cost minimization   | MILP            | –                      | Hourly, one week         | (✓)                     | –                |
| Patteeuw et al. 2015  | Cost minimization   | MILP            | –                      | Hourly, one week         | (✓)                     | –                |
| Penelli et al. 2014   | (Heuristic) Cost minimization | n/a | REMod-D | Hourly, 4 weeks Heat | – | – |
| Petrović, Karlsson 2016 | Cost minimization | LP | TIMES-DK | Hourly, 48 h | – | – |
| Salpikari et al. 2016 | Minimization of residual load | MILP | – | Hourly, 24 h | – | – |
| Schab et al. 2013     | Cost minimization   | LP              | REMod-D                | Hourly, 6 weeks          | –                       | –                |
| Teng et al. 2016      | Cost minimization   | MILP            | ASUC                   | Hourly, one year         | –                       | –                |
| Waite, Mod 2014       | Dispatch simulation | n/a             | REMod-D                | Hourly, one year         | (Pre-optimization)       | –                |

Notes: Parentheses indicate a secondary consideration. Abbreviations: LP: linear program; MILP: mixed integer linear program; NLP: non-linear program; n/a: not applicable; n/s: not specified; QP: quadratic program.

Explicit formulations

- **Formulation 1:** The COP of a heat pump is given by
  \[
  \text{COP}_{HP} = \frac{Q_{\text{out}}}{Q_{\text{in}}} \quad \forall \ t.
  \]  
  
- **Formulation 2:** However, in reality the COP strongly depends on the temperature levels of the energy source, \(T_{\text{source}}\), and sink, \(T_{\text{sink}}\), as expressed by the Carnot-COP, \(\text{COP}_{\text{carnot}} = \frac{T_{\text{sink}}}{T_{\text{source}} - T_{\text{sink}}}\). A low source temperature as well as high sink temperature lead to a lower COP, which can cause a pronounced increase of the required power input or operational problems for heat pumps; for instance, in cold winters when the energy source is environmental air or when high supply temperatures are required. This may apply to centralized heat pumps feeding into conventional district heating networks, conventional radiator-based domestic distribution systems, or domestic hot water supply, which requires a certain minimum temperature to prevent legionella bacteria.
Table 3 Nomenclature: most important abbreviations, variables and parameters for modeling heat pumps, electric boilers, and heat storage.

| Symbol | Unit | Technology | Explanation |
|--------|------|------------|-------------|
| DHW    | kW   | Heat pump  | Domestic hot water |
| EB     | kW   | Heat pump  | Electric boiler |
| HP     | kW   | Heat pump  | Heat pump |
| SH     | m³   | Space heating |

**Variables**

| P      | kW   | Heat pump | Electrical power input |
| Q      | kW   | Heat pump | Heat transfer rate |
| Q*     | kW   | Heat storage | Energy per period charged to storage |
| Q0     | kW   | Heat storage | Energy per period discharged from storage |
| QEB    | kW   | Electric boiler | Heat output |
| QHP    | kW   | Heat pump | Heat output |
| QSH    | kWh  | Heat storage | State of charge |

**Parameters**

| A      | m²   | Heat storage | Exposed surface |
| C      | kWh  | Heat storage | Heat capacity of storage medium (e.g. water) |
| e      | kWh  | Heat storage | Specific heat capacity of storage medium |
| COP   | ≥ 0 | Heat pump | Coefficient of performance |
| COP_dynamiscal | Heat storage | Storage losses (dynamic, static) |
| P     | kW   | Heat pump | Maximum electrical power input |
| QHP   | kW   | Electric boiler | Maximum heat output |
| QHPmax| kW   | Heat pump | Maximum heat output |
| QHPmin| kW   | Heat storage | Maximum charging restriction |
| QEB   | kW   | Heat storage | Maximum discharging restriction |
| DS    | kWh  | Heat storage | Maximum stored energy |
| T     | K    | | Temperature |
| T_A   | K    | | Ambient temperature |
| T_in  | K    | Heat storage | Temperature of the energy sink |
| T_source | K | Heat pump | Temperature of the energy source |
| A_T   | K    | | Temperature difference |
| U     | kW   | Heat storage | Heat transfer coefficient |
| T     | m³   | Heat storage | Physical storage volume |
| η     | [0;1] | | Efficiency, e.g. of electric resistance heaters |
| φ     | [0;1] | Heat pump | Quality grade as ratio of real COP to Carnot COP |
| ρ     | kg/m³ | Heat storage | Density of storage medium (e.g. water) |

Notes: Variables and parameters used only in specific modeling approaches are explained in the text upon appearance.

1 Blanks indicate relevance across technologies.

2 Can also be a variable in investment models.

3 Can also be a variable, depending on the specific context.

Formulation (1b) aims to capture this temperature dependence. Furthermore, a quality grade defined as $\phi = \frac{COP_{\text{real}}}{\text{COP}_\text{carnot}}$ can account for technical progress, where $COP_{\text{real}}$ expresses the theoretically achievable COP according to the temperature conditions and $COP_{\text{carnot}}$ the technically feasible COP. By 2014, such quality grades amounted to 0.24 to 0.45 [72].

$$COP_{\text{real}} = \phi \cdot COP_{\text{carnot}} = \phi \cdot \frac{P_{\text{sink}}}{P_{\text{sink}} - T_{\text{source}}} \quad \forall t$$  
(1b)

Several reviewed models [28,60,61,64] apply such formulations. Accounting for temperature dependence, relation (1a) transforms to (1c).

$$R = \frac{Q_{\text{HP}}}{COP_{\text{HP}}} = \frac{Q_{\text{EB}}}{\eta_{\text{EB}}} = \frac{Q_{\text{HP}}}{\eta_{\text{HP}}} + \frac{Q_{\text{EB}}}{\eta_{\text{EB}}} \quad \forall t$$  
(1c)

However, this formulation represents only an approximation; in reality, the supply temperature of a heating system, $T_{\text{sink}}$, is affected by the heat output $Q_{\text{HP}}$. Treating $COP_{\text{HP}}$ as an endogenous variable instead of an exogenous parameter leads to a non-linearity and, thus, to considerably higher computational effort. Verhelst et al. [73] suggest to pre-calculate $COP_{\text{pre-calc}}$ as a parameter by assuming an expected value for $T_{\text{sink}}$ and given values for $T_{\text{source}}^*$, as done in all cases found here applying a temperature dependent COP. Alternatively, Heinen et al. [25] suggest assuming a linear relationship between the hourly COP and the ambient temperature $T_a$ according to (1d), where empirical data determines the slope parameter $m$. Furthermore, they fit this relation to a fixed COP as input, $COP_{\text{input}}$, at a specific reference temperature of 7°C (280.15 K).

$$COR(T) = m\cdot(T_a - 280.15[K]) + COP_{\text{input}}$$  
(1d)

In order to capture higher efficiency in part-load mode and larger flexibility for load following of variable-speed heat pumps, Georges et al. [40] suggest a piece-wise linearization of the non-linear problem described by Verhelst et al. [73]. Salpakari et al. [68], who analyze the use of large-scale heat pumps in district heating networks, constrain heat pump use on the network’s exogenous supply temperature and allow operation only below 90°C. The COP is then treated as a constant parameter, which might be a justified assumption in the context of large-scale heat pumps with a stable heat energy source. Appendices A.1 and A.2 present detailed formulations for both suggestions.

5.2.2. Heat pumps with auxiliary electric boilers

Several articles provide formulations for the use of auxiliary electric boilers (EB). For instance, Patteeuw et al. [61] augment the standard approach (1a) to formulation (1e) and additionally distinguish between space heating (SH) and domestic hot water (DHW) applications. Heat output is limited through further constraints on maximum power consumptions of the heat pump and electric boiler.

$$R = \frac{Q_{\text{HP,SH}}}{\text{COP}_{\text{HP}}} + \frac{Q_{\text{EB,SH}}}{\eta_{\text{EB}}} + \frac{Q_{\text{HP,DHW}}}{\text{COP}_{\text{HP}}} + \frac{Q_{\text{EB,DHW}}}{\eta_{\text{EB}}} \quad \forall t$$  
(1e)

Waite and Modi [71] integrate the performance of an auxiliary electric boiler into the COP pre-calculation. Its operation is triggered by help of a minimum design temperature; Appendix A.3 provides further details. Also Hedegaard and Balyk [42] assume that the heating capacity of heat pumps is generally complemented with some electric boiler capacity for peak loads, which reduces investment costs. In their investment model, they specify a fixed ratio of heat pump capacity, $Q_{\text{HP}}$, to auxiliary electric boiler capacity, $Q_{\text{EB}}$, imposed by constraint (1f). Parameter $c_{\text{HP}}$ specifies this relationship; they suggest values between 0.72 and 0.82. For instance, if $c_{\text{HP}} = 0.8$, then for each 100 kW of heat pumps, 25 kW of auxiliary electric boilers must be installed.

$$Q_{\text{HP}} = c_{\text{HP}} \cdot Q_{\text{EB}}$$  
(1f)

5.3. Formulations for modeling heat storage

Technologies for heat storage are subject to different layouts, depending on their specific use. They may require different minimum temperature levels, which can have an impact on storage losses or on whether heat pumps can be connected to the storage. For instance, domestic hot water storage generally requires higher temperature levels, due to hygienic standards, than buffer tanks for conventional individual space heating. Floor heating systems require even lower temperatures.
5.3.1. Basic model

Most models implement such different kinds of storage according to a standard formulation (2), as found in [25,41,45,56,64,68]. It comprises a state-of-charge Eq. (2a), with storage energy level $S$ and periodically charged and discharged energy, $Q^C$ and $Q^D$, as well as a constraint on maximum storage energy $S$ (2b). Some analyses specify the maximum storage capacity using the storage volume $V$ in cubic meters as limiting parameter (2c), [25,45]), where $c, \rho$, and $\Delta T$ represent the specific heat capacity of water, its density, and temperature difference, respectively. Charging and discharging capacity constraints, (2d) and (2e), are rather found for larger heat storages for district heating. An alternative to energy levels is the use of temperature levels [38,45].

$$S_{t+1} = (1 - \text{loss}) - S_t + Q^C_t - Q^D_t \quad \forall t \quad (2a)$$

$$S_t \leq S \quad \forall t \quad (2b)$$

$$c \cdot \rho \cdot \Delta T \cdot V \quad \forall t \quad (2c)$$

$$Q^C_t \leq Q^C \quad \forall t \quad (2d)$$

$$Q^D_t \leq Q^D \quad \forall t \quad (2e)$$

There are different ways to account for storage losses; either only stationary losses $(\text{loss})$ are considered [25,38,41,62,64], or stationary as well as dynamic losses $(\text{dynamic})$ [45]. Patteeuw et al. [64] separate stationary heat losses in two parts: one proportional to the energy actually stored, the other one proportional to the storage size. In case of large heat storage devices for district heating, stationary losses are sometimes neglected [56,68].

Alternatively, Henning and Palzer [44,45] apply a differential equation for the storage energy content (3a) and its stationary losses (3b). The storage energy level is represented by the product of the storage’s constant total heat capacity $C^S$ and its temperature change over time $\frac{dT^S}{dt}$. Stationary heat losses $Q^\text{loss}$ depend on the temperature difference between storage and environment, $T^S - T^s$. Furthermore, they define a time lag $\tau$, which is rather long (180 days) for large centralized seasonal storage and small for decentralized short-term storage (72 h), to replace the constant heat loss coefficient $U \cdot A$ consisting of the heat transfer coefficient $U$ resulting from the storage’s insulation material and its exposed surface $A$.

$$C^S \cdot \frac{dT^S}{dt} = \text{dynamic} - Q^D - Q^\text{loss} \quad \forall t \quad (3a)$$

$$Q^\text{loss} = U \cdot A \cdot (T^S - T^s) = \sum_{\tau} \frac{E}{T^S - T^s} \quad \forall t \quad (3b)$$

5.3.2. Heat storage in district heating systems

For district heating, different network levels can be considered. Nielsen et al. [56] apply the standard formulation (2) for two types of heat accumulation tanks in district heating: one on the transmission, the other on the distribution level with lower supply temperature. The former is connected to CHP plants and electric boilers with charging and discharging constraints according to (2d) and (2e), the latter can be charged by the transmission network, heat pumps or electric boilers and has no charge or discharge constraints for storages. Salpakari et al. [68] additionally consider the storage capability of the network. They introduce a heat demand surplus, which allows for heat accumulation by increasing the storage level in the following time step.

5.3.3. Passive heat storage

Hedegaard et al. [41] apply the standard approach (2) to passive building mass storage and formulate the charging and discharging restrictions depending on a pre-specified temperature delta $\Delta T^\text{passive}$ for the building mass (4). This $\Delta T^\text{passive}$ captures the maximum temperature difference between the inside air temperature and the temperature of the building shell. Multiplied with the heat transfer coefficient $U$ and the exposed surface $A$, this renders the maximum storage charging and discharging capacity. Moreover, this maximum capacity is restrained by the state of charge: in case the storage state of charge of the previous time step is at its maximum level, $S_{t-1} = S$, the storage charging variable $Q^C_t$ is restricted to zero, which is achieved by constraint (4a). In this case, the discharging variable $Q^D_t$ is not further restricted, as constraint (4b) prescribes. Conversely, if the passive storage level is zero, $S_{t-1} = 0$, then no heat can be discharged, and charging is possible at the maximum rate. For instance, if $\frac{S_{t-1}}{S} = 0.9$, only this share of heat can be discharged in a time step, and only ten percent of the maximum capacity can be charged.

$$Q^C_t \leq U \cdot A \cdot \Delta T^\text{passive} \cdot \frac{1 - S_{t-1}}{S} \quad \forall t \quad (4a)$$

$$Q^D_t \leq U \cdot A \cdot \Delta T^\text{passive} \cdot S_{t-1} \quad \forall t \quad (4b)$$

Flexibility can also be provided by a room temperature target window. The formulation by Hedegaard et al. [42] comprises radiators and floor heating and accounts for another degree of flexibility provided by the passive energy storage capacity of the building mass. The temperature window is modeled by (5a) and (5b), where the actual room temperature $T_{1}^{R}$ is a variable that may deviate from a reference temperature $T_{1}^{ref}$ by an interval $[T_{1}^{ref}, T_{1}^{ref}]$. A fraction $\tau_{\text{inv}}$ of the residential building stock is equipped with control equipment to make use of this flexibility source. Eqs. (5c)-(5f) model the influence of exogenous shocks, such as heat from inhabitants or electrical appliances $Q^\text{inv}_{\text{ref}}$, ventilation $Q^\text{inv}_{\text{ref}}$, ambient temperature $T^a_{1}$, and endogenous decisions on heating technologies $Q^\text{HT}$.

The lower the ambient temperature $T^a_{1}$, the more heat $Q^\text{HT}$ escapes from the passive building storage (5e), thus reducing its temperature level $T_{1}^{S}$. In turn, more heat $Q^\text{HT}$ is transferred from the interior into the building mass (5d). Parameters $A$, $C^S$, and $U$ render the exposed surface, the heat capacity and the heat transfer coefficient for inner masses (1) and building mass (B).

$$T_{1}^{I} \geq T_{1}^{ref} + (T_{1}^{ref} - T_{1}^{ref}) \cdot \tau_{\text{inv}} \quad \forall t \quad (5a)$$

$$T_{1}^{I} \leq T_{1}^{ref} + (T_{1}^{ref} - T_{1}^{ref}) \cdot \tau_{\text{inv}} \quad \forall t \quad (5b)$$

$$T_{1}^{I} = T_{1}^{I} + Q^\text{HT} + Q^\text{inv}_{\text{ref}} + Q^\text{inv}_{\text{ref}} - Q^\text{inv}_{\text{ref}} - Q^\text{inv}_{\text{ref}} \cdot \frac{C^S \cdot A}{C^S \cdot A} \quad \forall t \quad (5c)$$

$$Q^\text{HT} = U \cdot \text{inv}_{\text{ref}} \cdot (T_{1}^{I} - T_{1}^{I}) \quad \forall t \quad (5d)$$

$$Q^\text{inv}_{\text{ref}} = U \cdot \text{inv}_{\text{ref}} \cdot (T_{1}^{I} - T_{1}^{I}) \quad \forall t \quad (5e)$$

$$T_{1}^{I} = T_{1}^{I} + Q^\text{HT} + Q^\text{inv}_{\text{ref}} \cdot \frac{C^S \cdot A}{C^S \cdot A} \quad \forall t \quad (5f)$$

Chen et al. [34], Patteeuw et al. [61], and Papaefthymiou et al. [60] also apply this concept. Specifically, they synthesize a virtual electric storage in a power system based on the aggregated building mass equipped with heat pumps. To this end, they derive room temperatures from a building simulation and use those as reference room temperatures $T_{1}^{ref}$ in an electricity market model without building mass storage. In a second step, they introduce a comfortable room temperature window (6b), taking into account passive building mass storage. The temperature delta $\Delta T$ between $T_{1}^{S}$ and $T_{1}$ in the second calculation is limited through a linear relationship with the power input $\Delta P$ in Eq. (6a) and its capacity constraint (6c). The demand response operation of heat pumps is thus modeled as equivalent energy storage. This two-step approach offers two advantages: (i) advanced models for thermal behavior of buildings can be applied as the reference operation is computed separately and (ii) linear models can be used to incorporate this type of demand response, which is defined as deviation from the reference case, allowing the use of typical dispatch.
5.3.4. State-space model

With varying co-authors, Patteeuw developed a state-space representation of residential heating demand that can be integrated into economic system optimization models [61–65]. The vector $\mathbf{T}_t$ captures the temperature state of the indoor air, floor, walls, and the roof for each building (class) $s$ in period $t$. Temperatures in the next period depend on the temperatures in the current period, where thermal conductances and capacities are summarized in matrix $A$. Moreover, vector $\mathbf{Q}_t$, contains all heat inputs to the system, consisting of solar irradiation, internal heat gains, outside and ground temperature as well as heat gains from heating devices, transmitted with factors summarized in matrix $B$. Heat dynamics are thus given by (7).

$$T_{t+1} = AT_t + BQ_t \quad \forall t$$

(7)

Together with constraints on the thermal comfort level $\mathbb{I} \leq T_t \leq \mathbb{T}$, where the indoor air temperature $T_{\text{int}}$ is an element of vector $\mathbf{T}_t$, this set of linear equations can be plugged into dispatch and investment optimization models of the energy system to capture relevant interactions.

6. Research questions and findings

6.1. Overview

The reviewed articles on power-to-heat not only differ with respect to technologies, research scopes, and methodologies, but also aim to answer distinct research questions. Most papers provide evidence on the potential of residential power-to-heat options for the system integration of variable renewable energy sources. A closely related research focus is the contribution to decarbonizing the energy sector. Cost effectiveness is another research focus and an important criterion to judge different technologies or policies; in fact, the majority of analyses are optimization exercises aiming to reduce overall system costs of operation and, partly, investments. It is difficult though to disentangle these three objectives because they mutually influence each other. For instance, cost effectiveness can be seen both as a primary goal under constraints on renewable energy use or decarbonization, and as a criterion to compare different scenarios of renewable energy integration. Beyond that, some analyses investigate the structure of heat supply, additional electricity demand of power-to-heat applications, and the impact on power prices. Moreover, some of the articles put emphasis on the development and presentation of a model or particular model features.

Table 4 summarizes the research questions. It indicates for all selected papers whether there is a distinct emphasis on particular research questions; parentheses indicate a secondary focus.

6.2. Cost effectiveness

A common finding of many reviewed articles is that power-to-heat applications have the potential to cost-effectively integrate high shares of variable renewable electricity into the energy system. Cost reductions are driven by (i) the substitution of fossil fuels [25,27,35,41,43,58,60,67,69], (ii) better use of capital invested in renewable assets by means of reduced curtailment [25,34,49,50,64,66,71], (iii) less need for costly auxiliary technologies such as peak-load capacity [43,63] or power storage [48], (iv) more efficient operation of thermal power plants because of less need for cycling and part-load operation [65,70], and (v) the use of existing district heating infrastructure [33,34,44,45,50,55].

For instance, Teng et al. [70] conclude from long-term analyses for the United Kingdom that flexible heat pumps could significantly lower the costs for integrating renewables and reducing carbon emission, particularly if heat pumps can provide frequency response. This is, among other factors, driven by improved efficiency of thermal generators due to smoother residual load and less need to run generators in part-load mode. Overall savings in renewable integration costs facilitated by flexible heat pumps are composed of lower costs for both backup and balancing (CAPEX and OPEX) and amount up to 6.5 Pound Sterling per MWh in a 2050 scenario. In an stylized Belgian setting, Patteeuw et al. [65] provide a related finding.

With different co-authors, Hedegaard [41,43] simulates the Danish energy system with complementary models. Here, heat pumps and passive heat storage are found to be low-cost options for reducing renewable curtailment and fossil fuel consumption. In 2030, decentralized heat pumps could save more than 10% of system costs compared to a reference without heat pumps [43]. Yet additional active heat storage decreases overall system costs not at all [41] or only to a minor extent [43] as this would incur relatively large capital costs. This finding is corroborated by Patteeuw and Helsen [64] for Belgium who find that high investment costs and losses make active thermal storage inefficient. Likewise, Papaefthymiou et al. [60] find significant cost reductions when large-scale deployment of decentralized heat pumps in Germany is combined with passive thermal storage in the building mass. The analysis shows a growing potential for system cost savings with higher renewables penetration, indicating a future key role for the flexibility provided by passive thermal storage.

Taking on a broader perspective, Conolly et al. [35] point to a general transformation of the cost structure of energy systems, that is, a shift from fuel to investment costs. Comparing scenarios with very high renewable shares up to 100% and more fossil-based reference scenarios, several analyses show that renewable-dominated systems that make use of power-to-heat may incur equal or only slightly higher total costs while achieving larger emission reductions [35,44,45,57]. Yet external costs of fossil fuel use are often not properly considered in such analyses.

Comparative analyses on sector coupling conclude that it is first and foremost cost-effective to couple the power and heat sectors [48,52,69]. Schaber et al. [69] argue that cost savings related to fuel consumption would be similar for coupling the power and hydrogen sectors; however, this would require higher investment costs compared to power-heat-coupling. Likewise, Mathiesen and Lund [52] concede that some types of electrolysers could supply larger operational flexibility than heat pumps, but are at the same time considerably less efficient and hence uneconomical. Kiviluoma and Meibom [48] find that power-to-heat options coupled with thermal storage can be more cost-effective than the grid integration of electric vehicles – yet both options combined would be even better.

6.3. Integration of variable renewable electricity

To be precise, we understand "renewable energy integration" as a higher utilization of renewable energy sources to meet final energy demand. Power-to-heat can contribute to such integration with respect to both a better utilization of existing assets and additional renewable capacity expansion. In particular, power-to-heat allows making better use of temporary renewable surplus generation. This may already apply (i) without the use of additional heat storage [63,71], particularly in case district heating systems can be used [32,33,43,50,51,57,68]. Utilization of renewable surpluses generally improves if (ii) additional active or passive heat storage is available [27,34,35,39,44,48,59,60,63,65,66,69], or if (iii) flexible fuel switching facilitated by hybrid heating systems enables "virtual" energy storage [25,38].
Patteeuw and co-authors highlight the value of additional flexibility for renewable energy integration in studies for Belgium: they find substantially reduced renewable curtailment in case of system-optimal use of decentralized heat pumps [64], in particular compared to inflexible heat pump use [63]. This allows reducing curtailment by around 50%; participating households, however, would have to receive appropriate incentives [65].

Hedegaard et al. [41] derive similar conclusions for a 2020 setting with a 50% wind power share. They identify reduced excess electricity production by 8% in case of a large-scale roll-out of decentralized heat pumps. While active heat storage only moderately increases this figure, passive thermal storage in buildings reduces curtailment by up to 19%. In two studies on municipal energy systems [57,59], Østergaard analogously derives a key role for centralized heat pumps in a renewable-dominated future system.

The literature provides similar findings for other energy systems: Kiviluoma and Meibom [48], for Finland, find that power-to-heat options with active storage lead to a higher optimal wind power capacity. Likewise, Waite and Modi [71], for New York City, argue that a mass roll-out of individual heat pumps enables greater expansion of wind power. While the overall utilization of wind power increases, the share of wind generation to cover conventional power demand decreases. In a study for Beijing in 2020, Chen et al. [34] determine an effective reduction of wind power curtailment through both the large-scale roll out of heat pumps and SETS. Under 20% wind penetration, passive heat pumps and SETS.

### Table 4

| Paper                        | Cost effectiveness | RES integration | Decarbonization | Power prices | Electricity demand for P2H | Structure of heat supply | Methodological contribution |
|------------------------------|--------------------|-----------------|-----------------|--------------|-----------------------------|--------------------------|-----------------------------|
| Artécon et al. 2016          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Bach et al. 2016             | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Barton et al. 2013           | ✓                  | ×               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Bauer et al. 2014            | ✓                  | ×               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Blanke 2012                  | ✓                  | ✓               | (✓)             | ✓            | ✓                           | ✓                        | ✓                           |
| Böttger et al. 2014          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Böttger et al. 2015          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Chen et al. 2014             | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Connolly et al. 2016         | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Cooper et al. 2016           | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Dods et al. 2013             | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Ehrlich et al. 2015          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Fehrenbach et al. 2014       | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Georges et al. 2014          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Hedegaard et al. 2012        | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Hedegaard et al. 2013        | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Heinen et al. 2016           | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Henning, Palzer 2014         | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Hughes 2016                  | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Kirkerud et al. 2014         | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Kiviluoma, Meibom 2010       | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Li et al. 2016               | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Liu et al. 2016              | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Lund et al. 2010             | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Mathiesen, Lund 2009         | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Merkel et al. 2014           | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Merkel et al. 2017           | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Münster et al. 2012          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Nielsen et al. 2016          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Østergaard et al. 2010       | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Østergaard, Andersen 2016    | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Østergaard, Lund 2011        | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Papathanasiou et al. 2012    | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Patteeuw et al. 2015         | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Patteeuw et al. 2015          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Patteeuw et al. 2016          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Patteeuw, Helsen 2016        | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Pensini et al. 2014          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Petrović, Karlsson 2016      | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Salpakari et al. 2016        | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Schaber et al. 2015          | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Teng et al. 2015              | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |
| Waite, Modi 2014             | ✓                  | ✓               | ✓               | ✓            | ✓                           | ✓                        | ✓                           |

**Note:** Parentheses indicate a secondary consideration.
storage suffices; with 40% wind, however, SETS are more effective to reduce wind curtailment due to additionally available active heat storage capacity. Also for the Beijing region, Liu et al. [50] determine a reduction in wind power curtailment from 7.5% to below 2% for an optimal roll-out of centralized electric boilers. Hughes [46] simulates the potential of SETS with different storage sizes for wind power integration in a Canadian island setting. Here, increasing numbers of SETS systems as well as increasing storage capacities of SETS reduce wind power surpluses, but may also diminish heating security due to (exogenously) limited wind power availability.

Two papers devise a framework of a fully renewable electricity supply: For Helsinki, Salpakari et al. [68] conclude on a significant reduction of excess electricity due to different centralized power-to-heat options – excess decreases from 40% to about 10%; the effect of additional thermal storage would be comparatively small. Pensini et al. [66] study the Pennsylvania-New Jersey-Maryland (PJM) market area in the US. Here, excess electricity could be reduced by almost 90% if decentralized resistive heaters with active storage were installed. A system with centralized heat pumps and active thermal storage could reduce curtailment by only 50%, however at considerably lower costs.

6.4. Decarbonization

Power-to-heat contributes to decarbonization if the substitution of fossil fuels for heating yields greater emission reductions than a potential emissions increase due to additional electricity demand [cp. also 74]. As reduced CO2 emissions are a corollary of a higher utilization of renewable energy sources, in principle, the same channels as laid out in Section 6.3 apply. Beyond that, some of the reviewed works explicitly derive figures. Specifically, cetesr paribus analyses that compare optimal system configurations (or paths) with and without particular power-to-heat technologies can shed light on decarbonization potentials.

For Beijing, Chen et al. [34] conclude on significant emission reductions, around 30–40%, in a scenario where heat pumps partly displace coal boilers. Similarly, Waite and Modi [71], for New York City, assess CO2 reductions up to about 10% when heat pumps replace 20% of natural gas-fired boilers. In an analysis for Finland in 2035, Kiviluoma and Melbom [48] conclude on a 30% emission reduction if heat pumps, electric boilers, and thermal energy storage are available compared to a baseline without these options. Likewise, studies for Belgium derive a substantial decarbonization potential for households. Here, decentralized heat pumps can save between about 10% and 75% of CO2 emissions compared to a baseline with natural gas-fired boilers, depending on the electricity mix [64]. Focusing on consumer behavior, it was found that emission savings are up to 7% higher if households do not behave myopically, but dispatch their heat pumps in a system-optimal way [65].

Several analyses find additional decarbonization benefits related to increased power-to-heat flexibility, facilitated in particular by heat storage. Patteeuw et al. [65] underline that CO2 emissions for residential heating would decrease by between 15% and 55% after replacing natural gas-fired boilers by heat pumps in a Belgian setting with 40% wind and PV; an additional flexible use of these heat pumps would contribute another 15% to decarbonization. Likewise, Papaefthymiou et al. [60] determine notable CO2 savings if a given residential heat pump fleet is operated flexibly, enabled by passive storage. In contrast, Hedegaard and Münster [43] see only minor carbon benefits from flexible operation of power-to-heat devices. In a study for Denmark in 2030, they find about 40% lower CO2 emissions from large-scale heat pump deployment compared to a case without heat pumps. Additional flexibility from active or passive heat storage would only add a minor 2% to this figure. A possible emission increase due to thermal losses of active heat storage is highlighted by Chen et al. [34] who analyze the roll-out of SETS in Beijing in 2020. Analogously, Patteeuw and Helsen [64] argue that thermal energy storage could increase CO2 emissions of buildings due to higher energy demand arising from standby losses; for space heating, the storage capacity of the building mass would be sufficient.

Focusing on system services, Böttger et al. [33], for Germany, determine lower CO2 emissions, by 0.4–0.9%, when centralized electric boilers participate in control power supply. Teng et al. [70] provide a related finding for the UK: they conclude that the flexibility provided by heat pumps enables substantial CO2 emission reductions, up to 30%, compared to a case with non-flexible heat pumps, particularly if heat pumps can provide frequency response.

6.5. Power prices and electricity demand for power-to-heat

Some papers explicitly consider the impact of power-to-heat applications on electricity prices. If electricity substitutes other fuels in residential heating, demand for electric power increases [27,29,35,36,47,54,62,63,69], and, in turn, power prices may rise [27,30,47]. However, the high efficiency of heat pumps and the flexibility of new loads may counteract this effect to some extent.

Arteconi et al. [27] and Patteeuw et al. [62] explicitly address this effect in studies for Belgium, loosely calibrated to 2030 parameter projections. They conclude that the additional electricity demand for residential heat can cause substantial price spikes if inflexible, that is, if it must be served within the hour of demand. If active and passive heat storage is available for a quarter to half of this load, price spikes largely vanish, also leading to a reduced consumer bill; the additional price effect of an entirely flexible demand is rather small.

For the United Kingdom, Cooper et al. [36] argue that peak demand, defined as the minute with the highest residual load, increases substantially in case of a large-scale roll-out of heat pumps. In 2030 scenarios without thermal storage except for the building mass, heat pumps in 80% of all buildings increase peak demand by about 30%, compared to a baseline without heat pumps. However, a more moderate heat pump roll-out, their flexible operation, and both active and passive thermal storage can mitigate the increase to between 7% and 16%. Likewise, Barton et al. [29] state that the additional peak load from heat pump electricity demand can be significantly reduced through flexible operation.

For Germany, Bauermann et al. [30] argue that heat pumps would only have a minor impact on power prices. Rather, they contribute to the power system’s flexibility and, thus, help smoothing prices. In a counterfactual analysis for Norway, whose power system is dominated by hydro power, Kirkerud et al. [47] come to a related conclusion: currently installed auxiliary electric boilers in district heating networks have a significant upward impact on power prices in a wet year but little impact in a dry year, where they are used only to a minor extent. Conversely, if heat pumps or electric boilers substitute fossil fuels in decentralized residential heating, price impacts in dry years would be substantial.

6.6. Structure of heat supply

Studies on the structure of heat supply shed light on trade-offs between power-to-heat technologies [28,52,57]. Specifically, several analyses endogenously determine an optimal capacity mix of heating technologies [25,30,37,39,48,54,55,64,66,67]. Most papers see a future central role for heat pumps in low-carbon energy systems.

Dodds [37] analyzes the residential heat supply in the United Kingdom until 2050 under an overall 80% decarbonization constraint. Until 2030, fossil fuel-fired boilers still dominate with 90% of the heat supply; by 2050, decentralized heat pumps prevail and supply around 60% of residential heat. Specifically, a range of boundary conditions restrict heat pump deployment below the economic optimum. These

6 For emission effects of other thermal energy storage applications, see [75].
comprise consumer preferences, space requirements, the presence of existing long-lived infrastructures, and regulatory inertia favoring incumbent technologies.

Likewise, Petrović and Karlsson [67] see a central role for ground-sourced heat pumps in Denmark by 2050, however also constrained by the available ground area. Other analyses for different Nordic countries also include a future switch from natural gas to heat pumps and, to a lesser extent, to electric boilers [48] for decentralized heat supply, alongside continued district heating [55]. Münster et al. [55] project that electric heating will have fully replaced natural gas and oil in individual heating by 2025 whereas electric heating will play only a minor role in district heating supply. In a study on the district heating system of Copenhagen in 2025, Bach et al. [28] stress a complementarity between heat pumps and CHP plants: while the former produce heat in hours with low electricity prices, the latter, conversely, do so in high-price hours. In any case, heat pumps would replace some CHP production. Moreover, the heat pumps’ full-load hours would be highest when connected to the heat distribution grid, since the lower supply temperatures compared to the transmission grid significantly increase their COP.

For Belgium, Patteeuw and Helsen [64] provide more differentiated results. Specifically, the optimal technology mix of decentralized heat technologies differs by house types. Especially rural and detached buildings rely on heat pumps with complementary electric resistive heaters. Active heat storage is never part of the optimal supply mix. Heinen et al. [25], in their study for Ireland in 2030, also stress that active heat storage is deployed to a larger extent only in a setting where the heating system is based on heat pumps and resistive heating to avoid price spikes. Otherwise, a mix of heat pumps or resistive heaters with natural gas boilers (and without active storage) would achieve the same goal efficiently. Moreover, the authors explicitly analyze the substitution between fuels for heating: in a hybrid system consisting of natural gas boilers and heat pumps, the latter would contribute around 50% of heat generation in a low-natural gas-price scenario. This share increases to 70% in a high-natural gas-price scenario. In combination with heat storages, these shares rise to 85–90%. In a hybrid natural gas-electric system, resistive electric heating achieves only a maximum share of 30% in heat generation. However, compared to a heating system only based on natural gas boilers, reductions in CO₂ emissions in the different hybrid systems are negligible.

Fehrenbach et al. [39] model the development of residential heat supply for Germany until 2050. Across all scenarios, oil-based heating is phased out until 2050. Likewise, the share of natural gas-fired boilers decreases. Both technologies are replaced by a renewable energy mix including heat pumps, which earn a considerable market share, especially in case of high fuel and CO₂ prices as well as ambitious renewable energy targets. Besides that, higher fuel prices favor energy efficiency measures. A more moderate fuel and CO₂ price development rather fosters the expansion of micro-CHP. In the most transformative scenario, assuming high fuel prices and decreased investment costs for heat pumps, the share of natural gas-based heat generation decreases from 46% in 2010 to 14% in 2050. In turn, heat pumps dominate heat supply with a share of 46%. In this case, the CO₂ emissions of residential heat supply decrease from around 140 million tons in 2010 to about 45 million tons in 2050. In all scenarios, heat pumps and, to a somewhat lesser extent, thermal storage also provide a significant load management potential. These findings are in line with those by Merkel et al. [54] who also identify high renewable energy targets as drivers for heat pumps as well as a strong impact of CO₂ and fuel prices on micro-CHP deployment. In a scenario with ambitious energy efficiency and renewable heat targets, oil- and natural gas-based heating is nearly phased out by 2030. Instead, heat pumps and electric boilers dominate heat supply by 2050, triggering a CO₂ emission reduction of 98% compared to 2015. In contrast, Bauermann et al. [30] conclude for Germany that, despite major differences in scenarios, natural gas heating systems would keep their leading market position.

6.7. Methodological contributions

Besides deriving results on the potentials of power-to-heat, several of the reviewed articles distinctly discuss model formulations [25,30,33,37,38,40,42,45,50,53,55,56,60–62,64,65,67,68]. Section 5 provides some specific examples in more detail.

Comparing results for heat pumps either modeled with a fixed or a temperature-dependent COP does not yield unanimous conclusions: Petrović and Karlsson [67] argue that the assumption of a fixed COP underestimates costs and CO₂ emissions substantially, especially for air-sourced heat pumps, due to underrated electricity demand at low ambient temperatures. On the contrary, Bach et al. [28] bring forward that the application of a variable or fixed COP for centralized heat pumps has little influence on model results. However, it is important to distinguish for centralized heat pumps to which level of the district heating network they supply. For the distribution level, they are subject to a lower supply (i.e., sink) temperature and, thus, exhibit a substantially higher COP than heat pumps on the transmission level.

Patteeuw et al. [65] compare different approaches of modeling demand-side behavior; specifically, household incentives to use their flexible heat pumps in a system-friendly manner. They conclude that time-of-use pricing yields poor incentives for system-friendly behavior because a large fleet of heat pumps does not anticipate its impact on dispatch. If price forecasts that take into account an integrated and optimized dispatch are communicated to households, their heat pump dispatch is much more system-friendly.

7. Conclusions

Achieving medium- and long-term climate targets calls for decarbonization not only of electricity generation, but also of the space heating sector. At the same time, the power system integration of variable wind and solar energy sources requires additional flexibility. A flexible coupling of power and heat sectors appears to be a promising strategy to address both of these challenges. Several power-to-heat technologies are available that may contribute to both decarbonizing heat supply and, if sufficiently flexible, integrating variable renewable electricity.

The reviewed literature provides a rich set of analytical approaches how to implement power-to-heat technologies in power system and market models. Numerical findings are generally idiosyncratic to geographical contexts, time horizons as well as assumptions on costs, policies, and technology availability. Yet some common findings can be synthesized. The consolidated evidence at hand suggests that power-to-heat can cost-effectively contribute to renewable energy integration. This is driven by the substitution of costly fossil fuels, reduced need for expensive peak load and power storage technologies, more efficient operation of thermal plants, synergies of using existing district heating infrastructures, and a better use of invested capital from reduced renewable curtailment. In turn, lower curtailment corresponds to a better integration and higher utilization of renewable energy sources. In a dynamic perspective, additional power demand for heating can also induce a further expansion of variable renewable generation capacity. Consequently, several analyses reviewed here indicate that power-to-heat decreases overall CO₂ emissions compared to scenarios without power-to-heat options. In contrast, the effect of additional heat storage on carbon emissions is ambiguous and depends on the power plant portfolio. If power is used for heating, electricity demand and power prices ceteris paribus rises. This does, however, not have to result in extreme price spikes if the heating sector is sufficiently flexible, for example enabled by flexible operation of power-to-heat options or thermal storage.

The reviewed articles analyze a range of different power-to-heat technologies. Specifically, many of the studies see a central role for heat pumps – be it decentralized or connected to district heating grids – because of their beneficial effects on system costs, renewable energy integration, and decarbonization (compare Sections 6.2, 6.3, 6.4).
Electric boilers are likewise identified as a relevant – and often supplementary – option. In contrast, smart electric thermal storage hardly plays a role in the reviewed analyses. Passive heat storage in well insulated buildings can help to tap additional low-cost flexibility potentials. Yet model formulations often appear to be rather stylized and may not take the residents’ behavior properly into account. The reviewed papers do not come to unanimous conclusions of the future relevance of active heat storage technologies most analyses conclude on a rather subordinate role.

While power-to-heat plays a major role in many articles reviewed here, it can still be considered a niche in most markets by 2017. Heating technologies not directly based on fossil fuels globally account for around a quarter of heat end energy use, over 95% of which are based on biomass and only around 1.5% on electricity [76]. Global statistics on power-to-heat technologies are largely missing. For Europe, market trends indicated increasing yearly sales volumes of heat pumps (largely air-to-air) between around 500,000 and 1 million units in recent years, totaling to a European stock of around 9.5 million units in 2016 [77]. When also including air conditioning devices whose primary use is cooling, the number accounts to more than 28 million units [78]. Yet this number is still small compared to the overall stock of dwellings in Europe [79]. As regards ground-sourced heat pumps, only around 50 GWth were installed worldwide in 2014, mostly in the US, China, and Europe [80]. Overall, heat pumps and other power-to-heat technologies are quantitatively still a small segment in Europe [79]. As regards ground-sourced heat pumps, only around a quarter of heat end energy use, over 95% of which are based on biomass and only around 1.5% on electricity [76]. Global statistics indicate increasing yearly sales volumes of heat pumps (largely air-to-air) between around 500,000 and 1 million units in recent years, totaling to a European stock of around 9.5 million units in 2016 [77]. Yet this number is still small compared to the overall stock of dwellings in Europe [79]. As regards ground-sourced heat pumps, only around 50 GWth were installed worldwide in 2014, mostly in the US, China, and Europe [80]. Overall, heat pumps and other power-to-heat technologies are quantitatively still a small segment – but also a growing one, in part driven by tighter regulations on building energy standards and partly also by direct support measures [76]. Accordingly, the power-to-heat industry would have to scale up substantially in order to achieve the high future deployment levels projected in many of the articles reviewed here.

Concerning avenues for future research, there is a growing body of evidence on the future role of power-to-heat technologies in low carbon energy systems, but there is much scope for further insights. First, our analysis of the reviewed articles’ scopes indicates that most papers focus on European case studies – complementary evidence on Asia, the Americas, and Africa, would be desirable. It would also be valuable to analyze power-to-heat options in specific settings of developing countries, where both heat and power system infrastructures are often less developed compared to most analyses reviewed here. Second, while some of the articles explicitly consider other sectors of the energy system, like mobility, broadening the scope of sector coupling could provide further insights on alternative or complementary decarbonization and flexibility potentials. Specifically, combined analyses of power-to-heat and other options referred to as power-to-x, for instance electrolytic hydrogen generation, may shed light on the comparative attractiveness of power-to-heat. In this context, it would further be desirable to include other, non-electric renewable heating options. Third, our methodological survey shows that most reviewed papers are based on optimization models. Future research could enrich the focus by explicitly considering behavior and incentives of involved parties, be it consumers, regulators or policymakers.

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Appendix A. Further formulations to model heat pumps

A.1. Formulation by Georges et al. [40]

Eqs. (8a)–(8c) are based on [81]. Eq. (8d) determines the maximum power consumption for time slice t; Eq. (8e) computes the actual power consumption in part-load mode. Note that part-load mode does not relate to the maximum installed capacity here, but to what is maximally possible under the temperature circumstances in a particular time step. Specific parameters c, d, and f are based on manufacturer data and n indicates “under nominal conditions.” Georges et al. [40] apply the same model for space heating and DHW heat pumps, with different supply temperatures.

\[ Q_{HP}^{\text{ff}} = (d_0 + d_1(T_{ff}^{\text{sink, n}} - T_{ff}^{\text{sink, n}}) + d_2(T_{ff}^{\text{sink, n}} - T_{ff}^{\text{sink, n}})) \cdot Q_{HP,n}^{\text{ff}} \forall t \]  
\[ \Delta T_{ff}^{\text{ff}} = \frac{T_{ff}^{\text{sink, n}} - T_{ff}^{\text{sink, n}}}{T_{ff}^{\text{sink, n}}} \forall t \]  
\[ \text{COP}_{ff}^{\text{ff}} = \frac{c_0 + c_1 \Delta T_{ff}^{\text{ff}} + c_2 \Delta T_{ff}^{\text{ff}}}{c_0} \forall t \]  
\[ P_t = \frac{Q_{HP}^{\text{ff}}}{\text{COP}_{ff}^{\text{ff}}} \forall t \]  
\[ P_t = \begin{cases} f_1 \frac{Q_{HP}^{\text{ff}}}{P_t} & \text{if } \frac{Q_{HP}^{\text{ff}}}{P_t} \leq 0.3 \\ f_2 \left( \frac{Q_{HP}^{\text{ff}}}{0.3} - 0.3 \right) + f_3 \frac{Q_{HP}^{\text{ff}}}{P_t} & \text{if } 0.3 < \frac{Q_{HP}^{\text{ff}}}{P_t} \leq 1 \end{cases} \forall t \]

A.2. Formulation by Salpakari et al. [68]

Using constraint (9a), Salpakari et al. [68] allow supply of heat pumps to the district heating network only below supply temperatures of 90 °C (with M as sufficiently big number). Likewise, heat accumulation in the piping network is only allowed for a temperature increase up to 15 K within each period (9b).
Waite and Modi [71] use data from heat pump manufacturers for $P_{HP}$ and $Q^{HP}$ at different ambient temperatures to compute a temperature dependent COP (10a). They augment their formulation to include an auxiliary electric boiler, whose use is triggered when the ambient temperature falls below a pre-determined design temperature $T_{design}$ according to Eq. (10b). $HD(T)$ renders the heating degree at temperature $T$ and $P_{electric}$ the boiler’s efficiency.

$$\text{COP}^{\text{Tdesign}} = \frac{Q^{\text{HP}}}{P_{\text{electric}}} + \frac{\eta_{EB}}{P_{\text{electric}}} \quad \text{if } T_{\text{source}} \geq T_{\text{design}}$$

$$Q^{\text{EB}}(T) = \begin{cases} 
0 & \text{if } T_{\text{source}} > T_{\text{design}} \\
\frac{\eta_{EB}}{P_{\text{electric}}} \sqrt{H_{\text{source}}^{\text{HP}}(T_{\text{source}}) - H_{\text{source}}^{\text{HP}}(T_{\text{design}})} & \text{if } T_{\text{source}} < T_{\text{design}} 
\end{cases}$$

$$P^{\text{EB}}(T) = \eta_{EB} Q^{\text{EB}}(T)$$

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