Design parameters analysis and verification of angular vibration sensor based on magnetohydrodynamics

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Abstract. The angular vibration is concerned in many fields such as satellite platform, manufacturing equipment for micro-electromechanical systems. However, the angular vibration with a frequency more than 15 Hz is difficult to be measured by traditional gyroscopes. The angular vibration sensor based on Magnetohydrodynamics can meet the requirements of both wide bandwidth and higher precision. In order to optimize the structure, a response of conducting fluid in the static magnetic field to the angular vibration is modeled in this paper. Based on this model, the sensitivity of the design parameters of magnetic field intensity, conducting fluids’ height and width are analyzed to get an optimized parameter for higher precision and bandwidth. A prototype was developed to verify the analysis and optimization. The experiment results showed that the model is accurate with 6.7% error in lower-cut-off frequency and 1.4% error in scale factor. It can meet the design requirement of 6–1000 Hz.

Keywords: Magnetohydrodynamics / angular vibration sensor / transfer function / parameter analysis / optimization

1 Introduction

Posture direction of the payload in a satellite platform has two basic requirements. The first is a very high precision with an ultra-low noise, and the second is the wide bandwidth to a frequency of 2000 Hz. The traditional gyroscope cannot meet these requirements any longer. At present, measurements of angular vibration with wide bandwidth rely on angular vibration sensors.

There have been several sensors widely used on satellite platforms to measure the micro-radian angular vibration with high precision and wide bandwidth. BEI 8301 Series of angular displacement sensor can measure angular displacement directly based on variable capacity, which was applied widely to land satellite mapping camera, GEOS (Geosynchronous Earth Orbit Satellite), GPS (Global Positioning System), etc. [1] Another type of sensor based on Magnetohydrodynamics (MHD) outputs the angular rate. The “magnetohydrodynamics” is first proposed by Hannes Alfvén and has been studied by many researchers [2–6]. Angular displacement is derived from the angular rate. It has also been employed in many missions such as American reflex tests of relay satellite, the most advanced meteorological satellite ALOS, etc. Magnetohydrodynamics is widely used in sensors, such as angular molecular-electronic Sensor [7], diagonal MHD accelerator [8], etc. The major advantages of the sensor based on MHD principle include simple structure, small size, light quality, low-power, insensitive to crossed-axis and linear acceleration, without moving parts, high reliability, impact resistance, well-adapted with temperature, and being able to realize broadband between 1 and 1000 Hz and high-precision measurements below 1 μrad [9–13]. He researched on micro-posture control algorithm of MHD angular velocity sensor combined with mechanical gyroscope end with simulation [14]. Huo studied the principle and designed the magnetic field of prototype [15,16]. Xu proposed a parallel magnetic field scheme to design and realize a prototype, and finished the corresponding testing [17], then improved its performance with new methods and structures in follow-up researches [18–21].

However, all these reports did not reveal the precise transfer function from the angular vibration to the voltage output. This transfer function is regarded as a coupled equation of electromagnetic field, conductive flow field and electrical network. The precise mathematic model should
be built to design and use the sensor well. This paper firstly 
abstracts the equivalent flow model. And then, the transfer 
function was derived based on the basic equations of fluid 
dynamics, Hartmann effect, Ohm’s law, and circuit 
equations. Next, the parameter sensitivity analysis was 
performed to optimize the design. Finally, a prototype was 
manufactured to verify the model and the design.

2 Mathematical models

The basic principle of MHD microradian angular vibration 
sensor is shown in Figure 1a. The vacant space of the 
annular barrel between the extern tube and the inner tube 
is filled with the conducting liquid (namely mercury ring), 
and the permanent magnetic circuit is designed to form 
outgoing radial magnetic field B in the mercury ring. When 
the sensor has a micro angular vibration \( \omega \) around Z axis, 
the conducting liquid will hold its position because of its 
great inertia and low friction between the tube surface and 
the liquid. In that case, the mercury ring incises magnetic 
flux to produce the induced electromotive force \( V_0 \) between 
the upper electrode and the lower electrode.

It is assumed that the conductive fluid’s width is one 
order of magnitude smaller than its length, then the 3D 
structure of the sensor can be simplified as a 2D equivalent 
flow model as shown in Figure 1b. The mercury ring’s Inner 
radius and extern radius are \( r \) and \( R \), respectively.

Furthermore, the 2D equivalent flow can be expanded 
along the circumference in order to deduce conveniently. In 
that case, a plate laminar flow is used to demonstrate the 
conductive fluid’s flow between the extern tube and inner 
tube as shown in Figure 2. The magnetic field, flow velocity, 
and electric field are orthogonal to each other governed by 
the right-hand rule. A micro flow unit is selected to analyze 
its dynamic model. Initial force \( F \), viscous force \( F_u \) and 
electromagnetic force \( F_{em} \) are applied on the unit.

According to the Navier-Stokes equation, the force per 
unit volume can be got as follow.

\[
F = \rho \frac{dv}{dt} = F_u + F_{em} \tag{1}
\]

where, \( \rho \) is the density of the conductive liquid, \( v \) is the 
relative velocity of the liquid to the tube. It is assumed that 
the flow between the two tubes in the perpendicular 
magnetic field is Hartmann flow. Then, the following 
equation can be achieved.

\[
\frac{F_{em}}{F_u} = M^2 \tag{2}
\]

where \( M \) is the Hartmann constant. Then equation (1) can 
be formulated.

\[
\rho \frac{dv}{dt} = \left(1 + \frac{1}{M^2}\right)F_{em} \tag{3}
\]

\( F_{em} \) can be calculated by the following.

\[
F_{em} = -BJz \tag{4}
\]
$J_z$ is the current density along $z$ axis. According to the Ohm’s law of the whole circuit, $J_z$ can be calculated.

$$J_z = \sigma(E_z - B \times V)$$  \hspace{1cm} (5)

where $\sigma$ is the conductivity, $E_z$ is the vortex electric field from the varying magnetic field induced by the varying induced current, and $B \times V$ is the induced electric field in the conductive fluid from the cutting magnetic wire. In general, $E_z$ is smaller than $B \times V$ because the induced eddy magnetic field is lower three orders of magnitudes than the permanent field $B$. The profile of the flow velocity $V$ relative to the magnetic field can be formulated as

$$V = v_y - v = \frac{y}{r} v_i - v$$  \hspace{1cm} (6)

where, when $y = r$, $v = v_i$ and when $y = R$, $v = v_i \times R/r$.

From equations (4)–(6), we can get

$$F_{em} = B^2 \sigma \left( \frac{y}{r} v_i - v \right)$$  \hspace{1cm} (7)

Substituting equation (7) into equation (3), the dynamic equation can be formulated.

$$\frac{\partial v}{\partial t} + \rho \frac{v}{r} v_i(s) \frac{1}{\rho s B^2 \sigma \left( 1 + \frac{M^2}{M^2} \right) + 1} = 1 + \frac{M^2}{M^2}$$  \hspace{1cm} (8)

where $v_i(r) = r \omega_i$.

After the Laplace Transformation, we get

$$\rho s v(s) = \left( 1 + \frac{M^2}{M^2} \right) B^2 \sigma \left( \frac{y}{r} v_i(s) - v(s) \right)$$  \hspace{1cm} (9)

So, we can get $v$

$$v(s) = \frac{y}{r} v_i(s) \frac{1}{\rho s B^2 \sigma \left( 1 + \frac{M^2}{M^2} \right) + 1} = 1 + \frac{M^2}{M^2}$$  \hspace{1cm} (10)

Then the output voltage $U_z$ can be formulated

$$U_z(s) = B \mathbf{L} V(s) = B \left( \frac{y}{r} v_i(s) - v(s) \right)$$

$$= B \left( \frac{i}{r} v_i(s) \frac{s}{s + \frac{B^2 \sigma}{\rho} \left( 1 + \frac{M^2}{M^2} \right)} \right)$$  \hspace{1cm} (11)

Because $v_i(s) = r \omega_i(s)$ and $M = B h \sqrt{\frac{R}{\rho}}$, the relation between $U_z(s)$ and $\omega_i(s)$ can be formulated. Here, $v$ is the dynamic viscosity coefficient of the liquid, $h$ is half of the width of hydrogarum loop as shown in Figure 2 [22].

$$\frac{U_z(s)}{\omega_i(s)} = B \mathbf{L} \left( \frac{s}{s + \frac{B^2 \sigma}{\rho} \left( 1 + \frac{M^2}{M^2} \right)} \right)$$  \hspace{1cm} (12)

where $y$ changes from $r$ to $R$, it can take the mean square root of $R$ and $r$. Then, the transfer function can also be formulated as

$$G(s) = \frac{U_z(s)}{\omega_i(s)} = B \tau_{RMS} \frac{s}{s + \frac{\sqrt{\tau_{RMS}^2 d_0}}{1 + \frac{M^2}{M^2}}}$$  \hspace{1cm} (13)

where $\tau_{RMS} = \sqrt{\frac{\tau}{R-r}} = \sqrt{\frac{(R^2-Br+r^2)}{3}}$.

### 3 Design and analysis

According to the schematic diagram as shown in Figure 1a, the mechanism is designed as shown in Figure 3. This is a 2D plane drawing, and the 3D model can be got by rotating around the axis. Magnetic flux from permanent 1 passes through magnetizer 1, magnetizer 2, hydrargyrum, magnetizer 3, magnetizer 4, in turn, and finally reaches to permanent 1. Another magnetic flux from permanent 2 with opposite magnetic field direction configuration also passes through the hydrargyrum in the same direction. The designed total height is 26 mm and the height of hydrargyrum is 15 mm. The radius is 13 mm, and the inner radius of the hydrargyrum loop is 9.6 mm and the thickness of the hydrargyrum loop is 1.4 mm. The design can get the radial magnetic field shown in Figure 1b.

Electrodes are installed between the top and the bottom of the hydrargyrum loop and connected through the conductive pillar along the axis as shown in Figure 3. The circuit of hydrargyrum loop, electrodes and conductive pillar can be seen as a primary coil of the transformer. Another coil is installed coaxially with the conductive pillar as the secondary coil of the transformer. This transformer is used to isolate and amplify the signal of the hydrargyrum loop.

The design prototype’s parameters are shown in Table 1.

Here, $\tau_{RMS} = 10.31$ mm and $M = 15.65$. From Table 1, the transfer function can be recalculated by equation (13).
as the following.

\[
G(s) = \frac{1.082 \times 10^{-4}s}{s + 37.624}
\]  \hspace{1cm} (14)

According to equation (14), the amplitude-frequency characteristic and the phase-frequency characteristic are shown in Figure 4a and b, respectively. It can be found that the sensor reveals a high pass feature. The corner frequency is about 6.0 Hz with \(-3\) dB attenuation. The output signal’s amplitude is about 108 \(\mu\)V/rad/s. Figure 4b shows that the phase changes from 90° to 0. The phase invert center is 45° at 6.0 Hz.

Several adjustable parameters related to the design are selected to explore the amplitude-frequency characteristic’s sensitivity. These parameters include \(B\), \(l\), \(r\), and \(R\). When one parameter is adjusted to be an alternation of \(\pm5\%\) or \(\pm10\%\), the other parameters are not changed, taking values according to Table 1. Each parameter’s influences to the frequency amplitude are shown in Figure 5.

It can be found that within \(\pm10\%\), the magnetic flux density \(B\) and the height of the hydrogarum loop \(l\), the inner radius of hydrogarum loop \(r\) and the outer radius of the hydrogarum loop \(R\) are all positively correlated with the amplitude. However, their influence on the amplitude is different. \(B\) and \(l\) have the strongest influence, \(r\) is less, and \(R\) is the least. In addition, \(B\) significantly influences the corner frequency. When \(B\) increase 10\%, the corner frequency increase 20.5\%. But \(l\), \(r\) and \(R\) have no relevance to the corner frequency. The detailed values are listed in Table 2.

### Table 1. The designed prototype’s parameters.

| Parameter                              | Symbol | Value | Unit   |
|----------------------------------------|--------|-------|--------|
| Magnetic flux density                  | \(B\)  | 0.7   | T      |
| Height of hydrogarum loop              | \(l\)  | 15    | mm     |
| Inner radius of hydrogarum loop        | \(r\)  | 9.6   | mm     |
| Outer radius of hydrogarum loop        | \(R\)  | 11    | mm     |
| Width of hydrogarum loop               | \(h\)  | 0.7   | mm     |
| Dynamic viscosity coefficient@20 °C    | \(v\)  | \(7.5 \times 10^{-8}\) | m\(^2\)/s |
| Hydrogarum density                     | \(\rho\) | \(13.6 \times 10^3\) | kg/m\(^3\) |
| Conductivity@20 °C                     | \(\sigma\) | \(1.04 \times 10^6\) | s/m |

**Fig. 4.** Frequency characteristics of MHD angular vibration sensor. (a) Amplitude-frequency responses. (b) Phase-frequency responses.

4 Experiments

The measuring circuit of the sensor is shown in Figure 6. The signal from the hydrargyrum loop is isolated and amplified through the transformer, then amplified again by an instrumentation amplifier, finally filtered by an active second-order low-pass filter with the cut-off frequency of 1200 Hz.
Fig. 5. The amplitude-frequency characteristic’s sensitivity analysis to micro adjustment of the designed parameters. (a) $B$ adjustment; (b) $l$ adjustment; (c) $r$ adjustment; (d) $R$ adjustment.

Table 2. Amplitude and corner frequency changing to parameter alteration.

| Characteristics      | Parameter | $-10\%$ | $-5\%$ | 0   | $5\%$ | $10\%$ |
|----------------------|-----------|---------|--------|-----|-------|--------|
| Amplitude ($\times 10^{-5}$ V/rad/s) | $B$       | 9.741   | 10.282 | 10.823 | 11.364 | 11.906 |
|                      | $l$       | 9.741   | 10.282 | 10.823 | 11.364 | 11.906 |
|                      | $r$       | 10.336  | 10.578 | 10.823 | 11.071 | 11.320 |
|                      | $R$       | 10.238  | 10.529 | 10.823 | 11.119 | 11.418 |
| Corner frequency (Hz) | $B$       | 4.85    | 5.41   | 6.00 | 6.59 | 7.23 |
|                      | $l$       | 6.00    | 6.00   | 6.00 | 6.00 | 6.00 |
|                      | $r$       | 6.00    | 6.00   | 6.00 | 6.00 | 6.00 |
|                      | $R$       | 6.00    | 6.00   | 6.00 | 6.00 | 6.00 |
The scale factor of the prototype is designed to be about 40 V/rad/s. As shown in Figure 4, the maximum output of the hydrargyrum loop is 108 μV/rad/s, so the total magnification of the measuring circuit is designed as 370 000. Then the maximum scale factor can be calculated by 108μ x 370 000 = 39.96 V/rad/s, close to the target. The prototype of the MHD angular vibration sensor has been made according to the above parameters and the structural design as shown in Figure 7a.

The prototype has been experimented under the frequency of 1–1000 Hz to test the performance of frequency characteristics as shown in Figure 7b. Since the angular vibration table used in the experiment can only measure under 1000 Hz, the experiment can be only implemented up to 1000 Hz.

The angular vibration used in the experiment is a sine wave with 0.174 rad/s (10°/s) Vpp. The original voltage signal of the sensor’s every frequency point is sine fitted by

\[
u = Cu\cos(2\pi ft + \phi_u) + Du\cos(2\pi ft) + Bu\sin(2\pi ft) + Du
\]

where \(u\) denotes the original voltage signal of the sensor, \(f\) is frequency, \(t\) is time, \(A_u, B_u, C_u, D_u\) are corresponding coefficients, then the output amplitude of the sensor \(Cu\) and the initial phase \(\phi_u\) are

\[
Cu = \sqrt{A^2 + B^2}, \quad \phi_u = \begin{cases} 
\tan^{-1}\left(-\frac{B}{A}\right), & A_u \geq 0 \\
\tan^{-1}\left(-\frac{B}{A}\right) + \pi, & A_u \leq 0
\end{cases}
\]

The scale factor \(S\) and phase delay \(\Delta\phi\) can be calculated by

\[
S = \frac{\dot{\theta}}{\dot{\theta}}, \quad \Delta\phi = \frac{180}{\pi} (\phi_u - \phi_\theta)
\]

where \(\dot{\theta}\) and \(\phi_\theta\) are the angular vibration amplitude and the initial phase of the angular vibration, respectively.

The frequency characteristics of this prototype can be drawn according to the experiment results on the angular vibration table. The sensor’s amplitude-frequency characteristic and phase-frequency characteristic curve are shown in Figure 8a and b, respectively. In addition, the theoretical amplitude-frequency curve of the model is the blue line in Figure 8a. It can be seen that the amplitude-frequency characteristic of the sensor prototype performs as a band-pass filter, and the maximum scale factor is about 40.2–40.5 V/rad/s. Thus, the lower-cut-off frequency (−3 dB) is about 6.4 Hz, the scale factor at 1000 Hz, which is still in the pass band.

The experiment shows that the lower-cut-off frequency of this MHD prototype is about 6.4 Hz, which has an 6.7% error rate. This delay is speculated due to the transformer. From Figure 8a, it can be seen that the scale factor has a significant downward trend at 1000 Hz, but still in the pass band. This is due to the active second-order low-pass filter. So, it can be estimated that the upper-cut-off frequency is not higher than the 1200 Hz, the cut-off frequency of the low-pass filter. The max scale factor of this prototype is 40.5 V/rad/s, which is about 1.4% error from the designed 39.96 V/rad/s. This error may be caused by various factors, such as noise, resistance accuracy in the circuit, etc.

5 Conclusion

This paper first derives the transfer function of the angular vibration response of the conductive fluid in the static magnetic field. Then, the magnetic field and electric circuit model of the sensor is designed. The model is designed as a high-pass filter with a cut-off frequency of 6 Hz. Based on this model, the sensitivity of the design parameters of magnetic field intensity, conducting fluids’ height and width are analyzed to get the optimized parameter for higher precision and width. After that, a prototype, including the measuring circuit, was developed to verify the analysis and optimization. Finally, the experiment is implemented to get the frequency characteristics of 1–1000 Hz. The result shows that the cut-off frequency of the prototype increases from 6 Hz of the model to 6.4 Hz, and the error is about 6.7%. The max error in scale factor is about 1.4%, which from 39.96 to 40.5 V/rad/s. These errors are within the acceptable, and the results can verify the model proposed in this paper. The experiment shows that the prototype meets the design requirement of 6–1000 Hz, and the model is accurate.

However, this MHD angular vibration sensor still needs some improvements. The current MHD prototype cannot measure low-frequency angular vibration signals below 6 Hz. The traditional gyroscope is needed to be used with...
this sensor prototype to complete angular vibration measurements within 0–1000 Hz. This is mainly because a larger magnetic flux density $B$ is used to improve the signal-to-noise ratio, the output of sensor and reduction circuit amplification, which leads to a higher cut-off frequency in the model. In addition, the transformer in the measuring circuit also increases the cut-off frequency. In the future research, a transformer with better low-frequency performance should be selected to decrease the cut-off frequency, and the measuring circuit should be modified to improve the detection and amplification of weak signals. As a result, a smaller magnetic flux density $B$ can be used to decrease the cut-off frequency of the sensor. In addition, noise and the accuracy of circuit components also should be concerned to decrease the error.

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