Research Article

Research on the Time-Domain Measurement Method of Low-Frequency Splitting for Hemispherical Resonator

Shaoliang Li, Hao Yang, Wanliang Zhao, Rui Qu, Jie Duan, Yijie Rong, Xin Jin, and Chaojiang Li

1Shanghai Aerospace Control Technology Institute, Shanghai 201109, China
2School of Mechanical Engineering, Nanjing University of Science and Technology, Nanjing 210094, China
3School of Mechanical Engineering, Beijing Institute of Technology, Beijing 100081, China

Correspondence should be addressed to Wanliang Zhao; zhaodada999@163.com

Received 3 February 2021; Revised 19 February 2021; Accepted 7 April 2021; Published 7 May 2021

Academic Editor: Vasilios N. Katsikis

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The measurement of resonator’s frequency splitting is a critical issue in vibratory gyroscopes, which would be elaborately treated in practical applications. The high-precision measurement of frequency splitting plays a significant role in frequency tuning control. A novel time-domain method of frequency splitting measurement for hemispherical resonator based on the standing wave swing effect was proposed. The frequency splitting value of the resonator can be directly obtained by taking the reciprocal of the one cycle time of standing wave swings, rather than through the frequency difference between two resonant modes. To begin with, the method was analyzed theoretically, and the measurement resolution and accuracy of the method were researched in detail. Simulation and experimental results showed that the frequency splitting value can be effectively obtained by measuring the period of the standing wave swings, improving the fine measurement resolution and high accuracy. The frequency splitting of lower than 0.007 Hz has to be effectively obtained in the experiment. It is found that the measurement error is a small proportional part of frequency splitting value, so the measurement accuracy is very high when the frequency splitting is very low. Therefore, this time-domain method would contribute to the measurement of ultralow-frequency splitting for high-Q resonators.

1. Introduction

Hemispherical resonant gyroscope (HRG) is a kind of vibrating gyroscope without high-speed rotor and movable parts, which is in operation based on the Coriolis effect. HRG offers many advantages which includes long running time, high accuracy, small structure, and short start-up time [1]. Therefore, it has a wide range of application prospects in high-precision and long-life inertial navigation such as aviation systems, precision guide missile, and other fields [2]. Due to the imperfect fabrication process, machining errors and manufacturing defects will inevitably occur on the resonator, which results in uneven circumferential distribution of mass, rigidity, and Q factor [3]. These problems are reflected in the vibration process of the harmonic oscillator, which will lead to the phenomenon of frequency splitting. Facts have proved that the primary error source of HRG is the drift caused by the frequency splitting [4–6]. The key issue in vibratory gyroscopes, therefore, will be the measurement of resonator’s frequency splitting, which would be elaborately treated by mechanical balancing [7] or electrical treatment [8–10] in practical applications. When the Q factor is high (typically up to $10^7$) and the frequency splitting is very low, it is a great challenge to obtain the measurement, because the frequency splitting value is sufficiently low to reduce the gyro drift.

Choi and Kim [11] estimated the frequency splitting amount of the natural frequencies based on the mathematical model of imperfect shell with multiple point masses. But no specific measurement method is proposed. Matveyev [7] proposed a method for determining the azimuth and frequency splitting of the inherent stiffness axis based on the amplitude frequency characteristics and phase angle characteristics. The resonator is excited by a sweeping voltage signal, and the vibration signal is recorded by two capacitive sensors at an...
angle of ±22.5°. The excitation voltage sweeps in steps of 0.001 Hz. The frequency splitting could be measured by analyzing the voltage amplitude and phase shift of the two capacitive sensors. This method has high measurement reliability and good stability which is wildly used in MEMS hemispherical resonator gyro [12–15] and MEMS gyros [16–19]. However, the disadvantage of this approach is that two precision capacitors need to be built around the resonator which is a challenging work to ensure the measurement accuracy. Besides of that, the measurement accuracy is limited by the step of the sweeping voltage. Cho et al. [20] mounted the resonator on a stacked PZT actuator and applied z-axis linear acceleration to the resonator. All the modes can be actuated due to the structural asymmetry in the resonator. And the resonant mode shapes and frequency spectrum are verified with a Polytec PSV 400 scanning laser vibrometer. Wang et al. [21] determined the resonant frequency through a PSV-500 and frequency-sweeping process. The frequency split of cylindrical axisymmetric-shell resonators was estimated based on FEM simulations and measured by getting the two natural frequencies based on resonator’s amplitude frequency response (AFR) characteristic [22, 23].

Measuring each resonant mode’s frequency separately was the main methods have been previously used. The resolution of frequency splitting suffers from the limitations of the Q factor of the resonators and the scanning step of exciting force. The measurement method of frequency splitting for a high-Q hemispherical resonator based on a standing wave swing effect was simply discussed previously [24].

In this paper, the time-domain measurement method of low-frequency splitting for a hemispherical resonator is demonstrated and analyzed in detail, which differs from our previous work in the analysis in detail and the further research of the measurement resolution and accuracy by the comparisons with the traditional AFR method and FFT analysis. A complete set of theoretical analysis and simulations of the time-domain measurement method based on the standing wave swing effect was investigated. The measurement resolution and accuracy of this method is researched by comparative experiments with the AFR method and FFT analysis. The experimental results show that the measurement method has a high-frequency resolution and accuracy. And the method is very suitable for the measurement of low frequency splitting for high-Q resonators.

2. Theoretical Analysis and Simulation

2.1. Theoretical Analysis. An ideal hemispherical resonator has a completely axisymmetric structure (as shown in Figure 1(a)), so the two working vibration mode \( n = 2 \) are degenerated and the natural frequencies are exactly the same, as shown in Figure 1(b).

Due to the processing shape and position deviations of the hemispherical resonator, such as circularity and coaxiality, and the anisotropy of the circumferential density and Young’s modulus of the resonator material, the two modes of the harmonic oscillator will develop into two 45° natural axes, the nature frequencies corresponding to the second-order bending modes of the two different natural axes reach the maximum and minimum, respectively (as shown in Figure 2). The difference between the two natural frequencies is called frequency splitting. And the two natural frequency axes are called natural stiffness axis. If the excitation of the harmonic oscillator is not along the natural axis direction, the frequency splitting will cause the standing wave of the resonator mode to slowly drift toward the natural axis until the vibration is along the natural axis direction, resulting in a gyro drift.

The two-dimensional dynamic model of an ideal axisymmetric hemispherical resonator could be presented as

\[
\dot{x} + \omega x - 2k\Omega \dot{y} = 0, \\
\dot{y} + \omega y + 2k\Omega \dot{x} = 0, 
\]

(1)

where \( x \) and \( y \) are the displacements of two vibration modes, \( \omega \) is the natural frequency of the vibration mode, \( k \) is the precession factor (\( k = 0.277 \) for the \( n = 2 \) mode of hemispherical resonator), and \( \Omega \) is the external angular velocity. For ideal axisymmetric hemispherical resonator, the natural frequency of two \( n = 2 \) modes is completely the same. When the external angular velocity is ignored, the vibration of two modes would be independently harmonic vibrations. The relationship of amplitudes and phases is denoted as

\[
x = x_0 \cos (\omega t + \alpha), \\
y = y_0 \cos (\omega t + \beta),
\]

(2)

where \( x_0, y_0 \) is the initial amplitude and \( \alpha, \beta \) is the initial phase.

Considering the damping issue of the vibration, the two-dimensional dynamic model of the hemispherical resonator can be expressed as

\[
\ddot{x} + \frac{\omega}{Q} k x + \omega^2 x - 4k\Omega \dot{y} = 0, \\
\ddot{y} + \frac{\omega}{Q} \dot{y} + \omega^2 y - 4k\Omega \dot{x} = 0,
\]

(3)

where \( Q \) is the Q factor of the damping vibration. And the vibration of two modes is as follows:

\[
x = x_0 e^{-(\omega Q)t} \cos (\omega t + \alpha), \\
y = y_0 e^{-(\omega Q)t} \cos (\omega t + \beta).
\]

(4)

When the external angular velocity is ignored, the vibration of two modes would be exponentially decayed.

For an incompletely axisymmetric hemispherical resonator, the two \( n = 2 \) vibration modes of the resonator are not degenerate. And the natural frequencies of the two modes are not equal due to the frequency splitting. The simplified dynamic model of the hemispherical resonator could be expressed by the following formula:

\[
\ddot{x} + \omega x - 2k\Omega \dot{y} = 0, \\
\ddot{y} + \omega y + 2k\Omega \dot{x} = 0,
\]

(5)
where $\omega_1$ and $\omega_2$ are the resonant frequencies of two vibration modes, respectively; $\omega_0 = (\omega_1 + \omega_2)/2$ when the $Q$ factor inconsistency of the two modes is ignored. The external angular velocity is also negligible when analyzing the frequency splitting. Therefore, the vibration of two modes is as follows:

\begin{align}
\dot{x} &= x_0 e^{(\omega_0/2) t} \cos \omega_1 t, \\
\dot{y} &= y_0 e^{(\omega_0/2) t} \cos \omega_2 t.
\end{align}

(5)

Thus, the vibration at the azimuth $\varphi$ of hemispherical resonator can be represented as

\begin{align}
w(\varphi, t) = x \cos 2\varphi + y \sin 2\varphi = x_0 \cos 2\varphi_0 e^{(\omega_0/2) t} \cos \omega_1 t \\
&+ y_0 \sin 2\varphi_0 e^{(\omega_0/2) t} \cos \omega_2 t.
\end{align}

(6)

2.1.1. Analyze the Vibration on the Effect of Frequency Splitting. It can be seen from the formula above, the $Q$ factor could be ignored when analyzing the vibration on the effect of frequency splitting.

The $n = 2$ vibration mode of the resonator is excited along by a certain azimuth through the specific excitation method. As a result, the description of resonator’s initial vibration is shown.

\begin{align}
w(\varphi, t) &= A \cos 2\varphi - \varphi_0 \cos \omega t, \\
&= A \cos 2\varphi \cos 2\varphi_0 \cos \omega_1 t + A \sin 2\varphi \sin 2\varphi_0 \cos \omega_2 t,
\end{align}

(8)

where $A$ is the initial vibrating amplitude, $\varphi_0$ is the azimuth between the standing wave and the normal axis, and $\varphi$ is the azimuth of the standing wave.

According to Matveev’s analysis, standing waves would be destroyed over time under the influence of frequency splitting. And the resonator’s vibration could be expressed as the superposition of two orthogonal vibration waves, which is

\begin{align}
w(\varphi, t) &= A \cos 2\varphi \cos 2\varphi_0 \cos \omega_1 t + A \sin 2\varphi \sin 2\varphi_0 \cos \omega_2 t, \\
&+ A \cos 2\varphi \cos 2\varphi_0 \cos \Delta_1 t \cos \omega t \\
&+ A \sin 2\varphi \sin 2\varphi_0 \cos \Delta_1 t \cos \omega t \\
&- A \sin 2\varphi \sin 2\varphi_0 \cos \Delta_2 t \sin \omega t.
\end{align}

(9)

where $\omega_1$ and $\omega_2$ are the resonant frequencies of two vibration modes, respectively. The frequency splitting value of hemispherical resonator is defined as

\begin{align}
\Delta f = \frac{1}{2\pi} (\Delta_1 + \Delta_2) = \frac{1}{2\pi} (\omega_2 - \omega_1),
\end{align}

(10)

where $\omega$ ($\omega_1 < \omega < \omega_2$) is a reference frequency; $\Delta_1 (\Delta_1 = \omega - \omega_1)$ and $\Delta_2 (\Delta_1 = \omega_2 - \omega)$ are two offset frequencies.

And the resonator’s vibration is rewritten as

\begin{align}
w(\varphi, t) &= A \cos 2\varphi \cos 2\varphi_0 \cos \Delta_1 t \cos \omega t \\
&+ A \sin 2\varphi \sin 2\varphi_0 \cos \Delta_1 t \cos \omega t \\
&+ A \cos 2\varphi \cos 2\varphi_0 \cos \Delta_2 t \sin \omega t \\
&- A \sin 2\varphi \sin 2\varphi_0 \cos \Delta_2 t \sin \omega t.
\end{align}

(11)
It can be derived that the vibrating energy at the position \( \varphi \) on resonator can be described as

\[
E(\varphi, t) = A^2 \cos^2 2\varphi_0 \cos^2 \varphi \cos^2 \Delta_1 t + A^2 \sin^2 2\varphi_0 \sin^2 2\varphi \cos^2 \Delta_2 t + 2A^2 \cos 2\varphi_0 \cos 2\varphi \sin 2\varphi_0 \sin 2\varphi \cos \Delta_1 t \cos \Delta_2 t + A^2 \cos^2 2\varphi_0 \cos^2 2\varphi \sin^2 \Delta_1 t + A^2 \sin^2 2\varphi_0 \sin^2 2\varphi \sin^2 \Delta_2 t - 2A^2 \cos 2\varphi_0 \cos 2\varphi \sin 2\varphi_0 \sin 2\varphi \Delta_1 t \sin \Delta_2 t = A^2 \cos^2 2\varphi_0 \cos^2 2\varphi + A^2 \sin^2 2\varphi_0 \sin^2 2\varphi + 2A^2 \cos 2\varphi_0 \cos 2\varphi \sin 2\varphi_0 \sin 2\varphi \cos (2\pi \Delta f). \]

(12)

It can be obtained from (12) that the energy of a certain position on resonator is demodulated by the frequency splitting value. And the demodulation period time is

\[
T = \frac{2\pi}{2\pi \Delta f} = \frac{1}{\Delta f}. \]

(13)

Thus, the frequency splitting of the hemispherical resonator can be calculated by measuring the period time of the resonator’s vibrating energy.

Take the partial differential of the vibrating energy, which is

\[
\frac{\partial E(\varphi, t)}{\partial \varphi} = -2A^2 \cos 4\varphi_0 \sin 4\varphi + 2A^2 \sin 4\varphi_0 \cos 4\varphi \cos (2\pi \Delta f t). \]

(14)

The azimuth of standing wave is presented as

\[
\tan 4\varphi = \tan 4\varphi_0 \cos (2\pi \Delta f) t. \]

(15)

It can be concluded from (15) that the azimuth of standing wave is demodulated by the frequency splitting value which results in wave swings under the influence of frequency splitting.

2.1.2. The Analysis of Effect of Q Factor on Frequency Splitting Measurement. Without external interference, the vibration of hemispherical resonator tends to ring down freely through the impact of vibration damping. It implies that the Q factor could affect the frequency splitting measurement.

Formula (9) could be reexpressed as the following formula under the influence of Q factor.

\[
\omega(\varphi, t) = A_0 e^{-i(\omega_0/2Q)t} \left[ \cos 2\varphi_0 \cos 2\varphi \cos \omega_1 t + \sin 2\varphi \sin 2\varphi_0 \cos \omega_1 t \right]. \]

(16)

From (16), the vibration would decay completely due to damping. The standing wave swing effect is rarely observed if the vibration attenuated to zero during the measurement process. The frequency splitting is hardly settled consequently. Thereby, the resolution of the frequency splitting requires that the vibration cannot be attenuated to the amplitude which is too small to be detected during the period time \( T \) shown in formula (13).

Formula (12) is represented as

\[
E(\varphi, t) = A^2 e^{-i(\omega_0/2Q)t} \left[ \cos^2 2\varphi_0 \cos^2 2\varphi + \sin^2 2\varphi_0 \sin^2 2\varphi + 2 \cos 2\varphi_0 \cos 2\varphi \sin 2\varphi_0 \sin 2\varphi \cos (2\pi \Delta f t) \right]. \]

(17)

From the formula above, the frequency splitting is much more accessible when the stand wave swing amplitude reaches a maximum. In this azimuth of the hemispherical resonator, the expression for vibration energy is

\[
E(t) = A^2 e^{-i(\omega_0/2Q)t} \left[ 1 + \cos (2\pi \Delta f) t \right]. \]

Let the initial time be zero, and the time will be

\[
2\pi \Delta f t = 2i\pi + \pi, i = 0, 1, 2 \cdots , \]

(19)

where \( t_i \) is defined as the situation when the vibration energy is one minimum value during the vibration. The formula above can be rewritten as

\[
t_i = \frac{2i + 1}{2\Delta f}. \]

(20)

The detection of effective vibration is limited by the resolution of the detection devices. It will be barely detected when the vibration signal is less than the noise of device.

The resolution of the detection device is recorded as \( \epsilon_k \) which is characterized by the ratio of the minimum vibration amplitude to the full range of the detection device. The vibration reaches to the full scale of the detection devices at time \( t_0 \). In order to distinguish one period time of the standing wave swing, it requires that

\[
\frac{E(t_1)}{E(t_1)} = e^{i(\omega_0/2Q)t} e^{-i(\omega_0/2Q)t} \geq (\epsilon_k)^2, \]

(21)

which can be reduced to

\[
\Delta f = -\frac{\omega_0}{2Q \ln \epsilon_k}. \]

(22)

One conclusion can be drawn that under a certain resolution of the vibration detection, the resolution of frequency splitting is higher as the Q factor increasing. The measurement method is very suitable for the high-Q resonators, because the resolution of frequency splitting is very high when Q factor is high enough.

2.2. Simulations. The time-domain measurement method of frequency splitting based on the standing wave swing effect is simulated in MATLAB based on the above analysis. In the simulations, the natural frequency of the resonator is 5000 Hz. The initial vibration amplitude is 5 \( \mu \)m. The frequency splitting value of the resonator is 0.015 Hz. And the Q factor is 1 500 000. The simulation results are shown in Figure 3. Lines 1 and 4 are the vibration attenuation curves when the standing wave coincident with stiffness principal
axis. Lines 2 and 3 are the vibration attenuation curves when the standing wave does not coincide with stiffness principal axis. It shows that a periodic oscillation signal can be obtained when the exciting and measuring direction is in the middle of two principal axes. And the frequency splitting value could be calculated by taking the reciprocal value of the signal’s period time.

2.3. Experimental Setup and Procedure. An experimental apparatus based on a laser Doppler vibrometer (LDV) is set up (as shown in Figure 4) in order to measure the frequency splitting of resonators based on the standing wave swing effect.

2.3.1. Experimental Setup. As shown in Figure 4, the hemispherical resonator is fixed on a turntable which could rotate within the range of 0°~360° in a vacuum chamber. The initial vibration of hemispherical resonator is excited by a striking hammer in a vacuum chamber. And the vibration information is recorded by a LDV.

2.3.2. Experimental Procedure. As a first step, adjust the exciting and measuring directions of the hemispherical resonator. Typically, the exciting point, measuring point, and center of resonator are in the same axial line for simplicity.

Evacuate the vacuum chamber to a certain pressure. The resonator’s Q factor varies with the pressure of the vacuum chamber. In the experiment, the pressure in the chamber should be smaller than 10^{-5} Pa to make sure the resonator get higher Q factor.

Strike the resonator in any initial orientation (set as 0°) to excite the initial vibration by the striking hammer in a vacuum chamber which is controlled by a pulse switch outside the chamber. Then, record the vibration signal of the resonator through a LDV.

The vibration signals are processed by filtering and fitting to get the energy exchange period time induced by the standing wave swing effect. The frequency splitting value could be obtained by taking the reciprocal value of the period time (T). As a matter of fact, the two natural frequencies and splitting value could also be obtained by the FFT processing the vibration signals.

3. Results and Discussions

In this section, the resonator’s frequency splitting is measured based on the method mentioned above. And the measurement resolution and accuracy of the method emphatically analyzed in detail. At last, this method is compared with the traditional AFR method and the FFT analysis.

3.1. The Measurement Results of Hemispherical Resonator’s Frequency Splitting. The frequency splitting of two typical
hemispherical resonators is measured using the method mentioned above. As reported in ref. [24], it is validated that the frequency splitting, as well as the $Q$ factor of the resonator could be obtained effectively by the measurement method. The results show that the frequency splitting value of 1# resonator is 0.0610 Hz and the $Q$ factor is 410000. And the frequency splitting value of 2# resonator is 0.3275 Hz and the $Q$ factor is 98000. The presented method is confirmed by comparing the simulation results and the experimental results. The parameters obtained in the experiment ($Q$, frequency, $\Delta f$) are substituted into the simulation model to obtain the simulation results under the resonator’s real parameters. And the comparison results are shown in Figure 5. The blue lines in the figure are the measured curves of the 1# and 2# resonators, and the red lines are the simulated curves under the parameters of measured data. It can be seen from the comparison that the simulation results are consistent with the experimental results. The difference phase between the two curves is caused by the selection of starting point in experimental data. The period time of vibrating signal between the simulation and experimental results is consistent which corroborate the correctness of this method.

3.2. The Measurement Resolution Analysis. According to the analysis of effect of $Q$ factor on frequency splitting
measurement and formula (22), it can be obtained that the frequency splitting measurement resolution based on the standing wave swing effect depends on the $Q$ factor of the resonator and the limit resolution of the detection devices. The higher the $Q$ value of the resonator and the higher the resolution of the detection devices, the higher the resolution of the frequency splitting measurement is of this method. It is simulated based on formula (22).

The simulation result is shown in Figure 6. The natural frequency of resonator is set to 5000 Hz. In the paper, the resolution of the detection device (LDV) is about $1E(-5)$ as shown by the orange line. Therefore, it is verified by measuring a resonator with lower frequency splitting as shown in Figure 7. The experimental results show that the frequency splitting of 0.007 Hz could be easily obtained while the $Q$ factor is more than 1.69 million. The result is basically consistent with the simulation results in Figure 6. Therefore, the resolution of frequency splitting value could reach to the level of $10^{-4}$ Hz if the $Q$ factor is more than 10 million.

### Table 1: Experimental results of the resonator.

| No. | $T_1$ (s) | $T_2$ (s) | Period time (s) | $\Delta f$ (Hz) |
|-----|----------|----------|----------------|---------------|
| 1#  | 2.967    | 19.350   | 16.383         | 0.0610        |
|     | 19.350   | 35.740   | 16.390         | 0.0610        |
|     | 35.740   | 52.390/  | 16.650         | 0.0601        |
|     | 1.603    | 4.656    | 3.053          | 0.3275        |
|     | 4.656    | 7.707    | 3.051          | 0.3278        |
| 2#  | 7.707    | 10.750   | 3.043          | 0.3286        |
|     | 10.750   | 13.810   | 3.060          | 0.3268        |

*The vibration signal is too small to detect; therefore, it induces errors.

### 3.3. The Measurement Error Analysis.

The measurement method based on the standing wave swing effect is essentially converting measuring the natural frequency of each mode into a period time based on the resonator’s inherent characteristics. As can be seen from the experimental results, the measurement error of this method is mainly on account of the reading error at the measurement time. The vibrating period time can be expressed by

$$T_m = T + kT |k| < 1,$$

where $T$ is the real period time without errors and $k$ is a small proportion estimation error. Therefore, the measured frequency splitting value is

$$\Delta f_m = \frac{1}{T_m} = (1-k+k^2+\cdots)\Delta f,$$

$$\Delta f_m \approx \Delta f + k\Delta f,$$

where $\Delta f$ is the real frequency splitting value of the resonator without errors.

It can be seen that the measurement error is a small proportion of the frequency splitting value. The smaller the frequency splitting, the higher the measurement accuracy is. In this paper, the frequency splitting value is calculated by selecting different swing period of 1# and 2# resonators as shown in Figure 5. The result is shown in Table 1. For 1# resonator, the period time of the third period has a little deviation compared with the first two periods, because the vibration signal has been greatly decayed and it is too small to distinguish which results in extra errors. In the measurement process, the first few periods with large amplitude
should be selected to avoid extra errors. Conclusion reached from the results that the lower the frequency splitting value, the lower the measurement error is. Therefore, this method presented is particularly suitable for the measurement of ultra-low frequency splitting for high-Q hemispherical resonators.

3.4. Comparison with AFR Method and FFT Analysis

3.4.1. AFR Method. As reported, the measurement method of frequency splitting based on AFR is widely used in many areas. However, its measurement accuracy is not enough owing to the resolution of the frequency is limited by the sweeping step of hardware. In the paper, the frequency splitting of hemispherical resonators 1# and 2# was measured based on AFR method for comparison, by the experimental setup reported in the previous article [25]. The hemispherical resonator is excited by a piezoelectric exciter in the sweeping frequency range of the resonator’s natural frequency. The sweeping frequency step is 0.002 Hz. The amplitude frequency response curves of the resonator in the orientation of 0° and 22.5° are obtained, respectively.

The experimental results are shown in Figure 8. For resonator 2#, the two resonant peaks are easily distinguished in the case of 0.326 Hz frequency splitting value. However, for resonator 1#, the two resonant peaks are hardly distinguished in the case of the frequency splitting value lower than 0.061 Hz. Although the measurement resolution of AFR would be a little better when the resonator’s Q factor is higher, a sweep frequency exciter should be placed inside the vacuum chamber, which made the setup more complicated and expensive. It would be still limited by the sweeping step of hardware and the sweeping process. In addition, the natural frequency is vulnerable by the impact of temperature changing which also make poor measurement repeatability base on AFR method. Therefore, this measurement method presented is more suitable for lower frequency splitting for high-Q resonators, while the previous method based on AFR is more suitable for larger frequency splitting.

3.4.2. FFT Analysis of the Vibration Signal. The vibration signal of the resonator could also be analyzed through fast Fourier transform (FFT), as shown in Figure 9. The results show that the natural frequencies of each mode and the frequency splitting value could be obtained through the FFT analysis of the vibration signal. However, the natural frequency peak of the FFT signal would be flattened which could result in measuring errors. There are two main reasons for that. To begin with, the frequency resolution is limited by the amount of the data for FFT analysis. And the natural frequency could be in the interval of two distinguishable frequencies. Moreover, the natural frequencies of vibrating modes fluctuate depend on the external environment during the process of the ring-down vibration which would lead to the broadening of the FFT frequency spectrum. For 1# resonator, as shown in Figure 9(a), the natural frequency of one mode is in the range of 7228.1069 Hz-7228.1128 Hz; and the natural frequency of the other mode is 7228.1680 Hz (the peak not flattened). Therefore, the frequency splitting value of 1# resonator is in the range of 0.0552 Hz-0.0611 Hz, while the measured frequency splitting value is 0.0610 Hz through time-domain analysis. For the 2# resonator, as shown in Figure 9(b), the natural frequency of one mode is in the range of 6129.8584 Hz-6129.8828 Hz; and the natural frequency of the other mode is 6130.2002 Hz-6130.2246 Hz. Therefore, the frequency splitting value of 2# resonator is in the range of 0.3174 Hz-0.3662 Hz, while the measured frequency splitting value is in the range of 0.3268 Hz-0.3278 Hz through time-domain analysis. The experimental results
show that, in the normal case of large frequency splitting value measurement, it could be obtained through the FFT analysis of the vibration signal. While in the case of the ultralow-frequency splitting measurement, the method presented in this paper has higher measuring accuracy by the time domain analysis.

4. Conclusions

The time-domain measurement method for low-frequency splitting of hemispherical resonators is researched in detail in the paper. The frequency splitting value of the resonator can be directly obtained without calculating the frequency difference between the two resonance modes. The experimental results reveal that the proposed method can effectively obtain a frequency lower than 0.007 Hz. And based on the proven evaluation, the resolution of frequency splitting value could reach to the level of $10^{-4}$ Hz if the Q factor is more than 10 million. Since the measurement error only makes up only a small fraction of the frequency splitting value, the measurement accuracy is much higher than the AFR method and FFT analysis. The time-domain method can not only greatly improve the measurement resolution of frequency splitting but also simplify the measurement procedure. Consequently, the measurement method holds great promise prospect in the field of high-precision inertial sensors.

Data Availability

The datasets used in the experiments and discussed in the paper will be available if required.

Conflicts of Interest

The authors declare that they have no conflicts of interest.
Acknowledgments

This work was supported by the National Key Research and Development Program of China (Grant No. 2017YFB1104700) and the Program of Shanghai Academic/Technology Research Leader under Project (Grant No. 18XD1421700).

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