Comparison and Interpretation Methods for Predictive Control of Mechanics

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Abstract: Objects that possess mass (e.g., automobiles, manufactured items, etc.) translationally accelerate in direct proportion to the force applied scaled by the object’s mass in accordance with Newton’s Law, while the rotational companion is Euler’s moment equations relating angular acceleration of objects that possess mass moments of inertia. Michel Chasles’s theorem allows us to simply invoke Newton and Euler’s equations to fully describe the six degrees of freedom of mechanical motion. Many options are available to control the motion of objects by controlling the applied force and moment. A long, distinguished list of references has matured the field of controlling a mechanical motion, which culminates in the burgeoning field of deterministic artificial intelligence as a natural progression of the laudable goal of adaptive and/or model predictive controllers that can be proven to be optimal subsequent to their development. Deterministic A.I. uses Chasle’s claim to assert Newton’s and Euler’s relations as deterministic self-awareness statements that are optimal with respect to state errors. Predictive controllers (both continuous and sampled-data) derived from the outset to be optimal by first solving an optimization problem with the governing dynamic equations of motion lead to several controllers (including a controller that twice invokes optimization to formulate robust, predictive control). These controllers are compared to each other with noise and modeling errors, and the many figures of merit are used: tracking error and rate error deviations and means, in addition to total mean cost. Robustness is evaluated using Monte Carlo analysis where plant parameters are randomly assumed to be incorrectly modeled. Six instances of controllers are compared against these methods and interpretations, which allow engineers to select a tailored control for their given circumstances. Novel versions of the ubiquitous classical proportional-derivative, “PD” controller, is developed from the optimization statement at the outset by using a novel re-parameterization of the optimal results from time-to-state parameterization. Furthermore, time-optimal controllers, continuous predictive controllers, and sampled-data predictive controllers, as well as combined feedforward plus feedback controllers, and the two degree of freedom controllers (i.e., 2DOF). The context of the term “feedforward” used in this study is the context of deterministic artificial intelligence, where analytic self-awareness statements are strictly determined by the governing physics (of mechanics in this case, e.g., Chasle, Newton, and Euler). When feedforward is combined with feedback per the previously mentioned method (provenance foremost in optimization), the combination is referred to as “2DOF” or two degrees of freedom to indicate the twice invocation of optimization at the genesis of the feedforward and the feedback, respectively. The feedforward plus feedback case is augmented by an online (real time) comparison to the optimal case. This manuscript compares these many optional control strategies against each other. Nominal plants are used, but the addition of plant noise reveals the robustness of each controller, even without optimally rejecting assumed-Gaussian noise (e.g., via the Kalman filter). In other words, noise terms are intentionally left unaddressed in the problem formulation to evaluate the robustness of the proposed method when the real-world noise is added. Lastly, mismodeled plants controlled by each strategy reveal relative performance. Well-anticipated results include the lowest cost, which is achieved by the optimal controller (with very poor robustness), while low mean errors and deviations are achieved by the classical controllers (at the highest cost). Both continuous predictive control and sampled-data
predictive control perform well at both cost as well as errors and deviations, while the 2DOF controller performance was the best overall.

**Keywords:** model predictive control; robust predictive control; computational algorithms for predictive control; optimization algorithms for predictive control; robustness of predictive control; data-driven predictive control; control quality assessment of predictive control; applications of predictive control; adaptive control; control of satellite systems

1. Introduction

Michel Chasles’s Theorem [1] allows us to simply invoke Euler’s [2] and Newton’s [3] equations to fully describe the six degrees of freedom of mechanical motion. Controlling mechanical motion has an even longer pedigree. The issue of automatic control of mechanical motion has occupied engineers for hundreds of years. Consider the Greeks and Arabs as early as 300 BC to about 1200 AD. They used float regulators for water clocks [4]. Imagine the float in a toilet reservoir. When the reservoir tank levels reached the desired levels, floats would actuate releases to keep the water at that level. This guaranteed a constant flow rate from one tank to a second tank. The time it takes to fill the second tank at this constant flow rate was a constant clock. Perhaps feedback was born in this way if not earlier.

The period prior to 1900 is often referred to as the primitive period of automatic controls. The period from 1900 to the 1960s can be considered the classical period, while the 1960s through present time is often referred to as the modern period of automatic controls [5]. By the 1840s, automatic controls were examined mathematically through differential equations (e.g., British astronomer G.B. Airy’s feedback pointed telescope [6] was developed using mathematical stability analysis). By the 1920s and 1930s, the difficulty in dealing with differential equations led to the development of frequency domain techniques driving the analysis of automatic control systems via such tools as gain and phase margins measured on frequency response plots [6]. While some work was done with stochastic analysis (e.g., Wiener’s 1949 statistically optimal filter [7]), it was in the 1960s that time-domain differential equations re-emerged in strength. By 1957, Bellman had solved optimum control problems backwards in time [8]. Meanwhile Pontryagin used Euler’s calculus of variations to develop his maximum principle to solve optimization problems [9,10]. Around this same time, Kalman developed equations for an estimator [11] that proved equivalent to the linear quadratic regulator (LQR) [12]. Kalman demonstrated that time-domain approaches are more applicable for linear and non-linear systems (especially when dealing with multi-input/output systems). Kalman formalized the optimality of control by minimizing a general quadratic cost function. Subsequent to the establishment of optimal benchmarks, adaptive [13–20] and learning controls [21–32] arose with a subsequent effort to combine the two, which makes adaptive controllers optimal, where optimization occurs after first establishing the control equation, such as by using ‘approximate’ dynamic programming [33]. Assuming distributed models, Reference [34] and Reference [35] used neighbor-based optimization to develop predictive controls. Other researchers [36] sought extensions into nonlinear systems with actuator failures, while Reference [37] tried to use neural networks to find the model and response in a predictive topology. Inspired by the adaptive lineages [13–22,38] and learning lineages [23–32], both Reference [39] and Reference [40] break with the usual design paradigm and, instead, begin with the solution of a nonlinear optimization problem as the first step of control design, and that an alternative paradigm is adopted in this study, which results in controllers’ instances leading to a novel proposal: a two degree of freedom controller that results from the original optimization statement including predictive control and robust feedback control achieving non-trivial results in the face of noise modeling errors not expressly written in the original problem formulation. The achievement of good results despite a dramatic simplification of methods is original and refreshing.
Over 25 years ago, in 1993, Lee et al. formulated a nonlinear model predictive control algorithm based on successive linearization, where the extended Kalman filter uses a multi-step future state prediction [41]. Almost 10 years later, in 2002, Wang et al. [42] proposed an optimal model predictive control for high order plants modeled first-order linear systems with time-delay and random disturbance including non-gaussian noise, while the Kalman filter was used for present and future state estimation. By 2007, model predictive control (MPC) with Kalman filters was appearing in undergraduate theses [43]. Published in 2012, Reference [44] deals with the closed-loop robust stability of nonlinear model predictive control (NMPC) coupled with an extended Kalman filter (EKF) where optimization was used for target setting used in the objective function in nonlinear model predictive control, while, unfortunately, stability deteriorates due to the estimation error. In 2013, predictive control was used for the constrained Hammerstein-Wiener model of nonlinear chemical processes (e.g., continuous stirred tank reactors) with a Kalman filter for output feedback of a linear subsystem elaborated from a finite horizon through the receding horizon [45]. In that same year, Reference [46] focused on state estimation, comparing the Kalman filter and the Particle Filter, while state estimation with finite state models was also briefly considered. The Bayesian approach was emphasized.

In 2016, Binette et al. sought to improve economic performance of batch processes using the Nonlinear Model Predictive Control (NMPC) without parameter adaptation, where the measured states serve as new initial conditions for the re-optimization problem with a diminishing horizon. Model mismatch of the plant preclude achievement of optimality. However, neglecting process noise and assuming small parametric variations, re-optimization always improves performance assuming the trajectory is feasible and, thus, Binette et al. assert that this illustrates robustness. This manuscript will re-examine this goal instead of seeking to achieve robustness by including feedback that directly derives from the original optimization problem (whose results are codified in time-parameters rather than state parameters). [47] Cao et al. represented uncertainties using scenarios from a robust nonlinear model’s predictive control strategy, where large-scale stochastic programming is used at each sampling instance solved by the parallel Schur complement method. The method merely exceeds the robustness of an open-loop optimal control, and, furthermore, processing time remains a limitation while achieving an optimal update only twice per minute. [48] Genesh et al. used a model predictive control system developed to control furnace exit temperature to prevent parts from overheating in a quench hardening process. Knowing only a proportional state and time, a mere 5.3% efficiency increase was achievable [49].

In 2017, Jost et al. used nonlinear model predictive control combined with parameter and state estimation techniques to provide dynamic process feedback to counter uncertainty. To counter uncertainty, ubiquitous dual methods provide excitation with the control while minimizing a given objective. Jost et al. proposed to sequentially solve robust optimal control, optimal experimental design, and state and parameter estimation problems separating the control and experimental design problems, and permitting independent analysis of sampling. This proved very useful when fast feedback was needed. One such case is the Lotka-Volterra fishing benchmark where the Jost algorithm indicates a 36% reduction of parameter [50]. Xu et al. used the Discrete Mechanics and Optimal Control (DMOC) method to formulate a nonlinear model predictive control, using numerical simulations to compare with the standard nonlinear model predictive control (NMPC) formulation based on the method of multiple-shooting. Using the Real-Time Iteration scheme, the real-time application of the NMPC algorithm extended the DMOC based NMPC applied to the swing-up and stabilization task of a double pendulum on a cart with limited rail length to show its capability for controlling nonlinear dynamical systems with input and state constraints [51]. Suwartadi et al. presented a sensitivity-based predictor-corrector path-following algorithm for fast nonlinear model predictive control (NMPC) and demonstrated it on a large case study with an economic cost function solving a sequence of quadratic programs to trace the optimal nonlinear model’s predictive control solution along a parameter change incorporating strongly-active inequality constraints included as equality constraints in the quadratic programs, while the weakly-active constraints are left as inequalities [52].
Vaccari et al. designed an economic model’s predictive control to asymptotically achieve optimality in the presence of plant-model mismatch using an offset-free disturbance model and to modify the target optimization problem with a correction term that is iteratively computed to enforce the necessary conditions of optimality in the presence of a plant-model mismatch [53].

Last year, in 2018, Kheradmandi et al. addressed a data-driven model-based economic model and predictive control capable of stabilizing a system at an unstable equilibrium point [54]. Wong et al. applied model predictive control to pharmaceutical manufacturing operations using physics-based models akin to the technique adopted in this study from deterministic artificial intelligence [37]. Durand et al. developed a nonlinear systems framework for understanding cyberattack-resilience of process and control designs and indicates through an analysis of three control designs for resiliency [55]. Singh Sidhu utilized model predictive control for hydraulic fracturing based on the physics-based assumptions that the rock mechanical properties, such as the Young’s modulus, are known and spatially homogenous, where optimality is forsaken if there is an uncertainty in the rock mechanical properties by proposing an approximate dynamic programming-based approach. This is a model-based control technique, which combines a high-fidelity simulation and function approximate to alleviate the “curse-of-dimensionality” associated with the traditional dynamic programming approach [33]. Xue et al. addressed the problem of fault-tolerant stabilization of nonlinear processes subject to input constraints, control actuator faults, and limited sensor–controller communication by using the fault-tolerant Lyapunov-based model [36]. Tian et al. applied model predictive control design for a mineral column flotation process modeled by a set of nonlinear coupled hetero-directional hyperbolic partial differential equations (PDEs) and ordinary differential equations (ODEs), which accounts for the interconnection of well-stirred regions represented by continuous stirred tank reactors (CSTRs) and transport systems given by hetero-directional hyperbolic PDEs, with these two regions combined through the PDEs boundaries. Cayley–Tustin time discretization transformation was used to develop a discrete model’s predictive control. In this article, discretization delays are implemented alone without specified treatment to evaluate their deleterious effects on the control’s accuracy and robustness [56]. Gao et al. considered a distributed model predictive controller on a class of large-scale systems, which is composed of many interacting subsystems. Each of them is controlled by an individual controller where, by integrating the steady-state calculation, the designed controller proved able to guarantee the recursive feasibility and asymptotic stability of the closed-loop system in the cases of both tracking the set point and stabilizing the system to zeroes [34]. Wu et al. utilized an economic model’s predictive control on a nonlinear system class with input constraints for chemical seeking to first guarantee stability, and then ensure optimality (this is the opposite approach taken in this manuscript) [57]. Liu et al. proposed an economic model’s predictive control with zone tracking based on economic performance and reachability of the optimal steady-state by decoupling dynamic zone tracking and economic objectives to simplify parameter tuning [58]. Zhang et al. sought to augment dissipative theory to propose a novel distributed control framework to simultaneously optimize conflicting cost statements in demand-side management schemes (e.g., building air conditioning), while seeking to mitigate power fluctuations. Their proposed method achieved the objective of renewable intermittency mitigation through proper coordination of distributed controllers, and the solution proved to be scalable and computationally efficient [35]. Bonfitto et al. presented a linear offset-free model predictive controller applied to active magnetic bearings, which exploits the performance and stability advantages of a classical model-based control while overcoming the effects of the plant-model mismatch on reference tracking by incorporating a disturbance observer to augment the plant model equations. Effectiveness of the method was demonstrated in terms of the reference tracking performance, cancellation of plant-model mismatch effects, and low-frequency disturbance estimation [59]. Godina et al. compared model predictive control to bang-bang control and proportional-integral-derivative control of home air conditioning seeking to minimize energy consumption assuming six time-of-use electricity cost rates during an entire summer, while tariff (taxation) rates were evaluated for efficacy [60].
Lastly, this year (in 2019), Khan et al. described a model predictive control based on the technique of a finite control set applied to uninterrupted power supply systems, where the system’s mathematical model is used to predict voltages of each switch at each sampling time applied to switching states of parallel inverters. They achieved a low computational burden, good steady state performance, fast transient response, and robust results against parameter disturbances as compared to linear control [61]. Since variable switching frequency negatively impacts output filters, Yoo et al. implemented constant switching frequency in a finite control-set model predictive controller based on the gradient descent method to find optimal voltage vectors. They found lower total harmonic distortion in output current with improved computation times [62]. Zhang et al. utilized sequential model predictive control for power electronic AC converter, and their research sought to improve a complicated weighting factor design and also computational burden applied to three-phase direct matrix conversion. The performance akin nominal model’s predictive control was revealed [63]. Baždarić et al. introduced approaches to the predictive control of a DC-DC boost converter in a comparison of the controllers that consider all of the current objectives and minimize the complexity of the online processing, which results in an alternative way of building the process model based on the fuzzy identification that contributes to the final objective. This includes the applicability of the predictive methods for fast processes [64]. Seeking to optimize the control strategy of the traction line-side converter (LSC) in high-speed railways, Wang et al. applied a model-based predictive current control and the results indicate a significant influence on the performance of traditional transient direct current control. Both the oscillation pattern and the oscillation peak under traditional transient direct current control can be easily influenced when these parameters change [65].

In 2019, Maxim et al. [66] investigated how, when, and why to choose model predictive control over proportional-integral-derivative control. With this backdrop, the goal of the research presented in this manuscript is to develop and then examine several variations of classical and modern era optimal controllers and compare their performance against each other.

A key contribution is the design of predictive controllers that are designed using optimization as the very first step, including a formulation of state feedback for robustness in the same optimization while a subsequent step converts the optimal solution from time-parameterization to state-parameterization allowing proportional-derivative gains to be expressed as exact functions of the optimal solution. Thus, feedback errors are de facto expressed exactly in terms of the solutions to the original optimization problem and errors are, thereby, optimally rejected. This notion permits the reader to use only this predictive, optimal feedback controller by itself and also together with the optimal feedforward. Lastly, comparing the optimal feedforward to the predictive optimal feedback control permits expression of a proposed controller called “2DOF” to imply the twice-invocation of the original optimization problem. This proposed 2DOF topology achieves near-machine precision target tracking errors while using near-minimal costs.

Despite the previously mentioned lengthy lineage of smoothing estimators to eliminate negative effects of noise (e.g., control chattering), a distinguishing feature presented in this case is the production of state estimates as an output of the original optimization problem, which alleviates the need for state estimation, which is often performed in the literature by various realizations of Kalman filters. In this case, the noisy states provided by sensors will not be smoothed by optimal estimators (e.g., Kalman filters), instead left in full-effect as agitators of deleterious effects in order to critically compare the robustness of each approach compared. The same is true of modeling errors: the effects were purposefully left out of the original problem formulation, yet applied in the validating simulations to critically expose the strengths and weaknesses of each approach compared.

2. Materials and Methods

This section derives and describes the implementation of the controllers in this study including Optimum Open Loop, Continuous Predictive Optimum, Sampled Predictive Optimum,
proportional-derivative (PD), Feedforward/Feedback PD, and 2DOF control. The context of the already commonly defined term “feedforward” used here (differently and more restrictively) is the context of deterministic artificial intelligence, where analytical self-awareness statements are strictly determined by the governing physics (of mechanics in this case, e.g. Chasle, Newton, and Euler). When the feedforward is combined with proportional-derivative, “PD” feedback control (developed from the optimization statement at the outset using a novel re-parameterization of the time-optimal results from time-to-state parameterization), the resulting controller is referred to here as “2DOF” or two degrees-of-freedom to indicate the twice invocation of optimization at the genesis of the feedforward and the feedback, respectively.

First, a brief derivation of the optimum control for our double integrator assumes a quadratic cost function. Using a simple double-integrator plant (modeling translational motion in an inertial reference frame described by Newton and Chasle), controllers were developed in accordance with the new design techniques to travel from the initial conditions \( \theta(0) = 0, \omega(0) = 0 \) to the final conditions \( \theta(1) = 1, \omega(1) = 0 \), where \( \theta \) is the translational motion state and \( \omega \) is the translational velocity state. The controllers were then applied to a mismodeled plant, which contained a \( 0.005[I] \{\omega^2\} \) dependence (where \( I \) is the mass inertia and \( \omega \) is angular velocity). Additionally, the inertia of a body is difficult to know exactly what adds “inertia noise,” which is also referred to as modeling errors. After witnessing the deleterious impact of noise on an open loop controller’s robustness, the remaining controllers studied in this research were closed loop. Feedback was instituted to deal with noise. Figures of merit address two key issues of control: achieving desired end conditions and minimizing the required control effort. Accordingly, controllers in this research report will be compared via nominal error norms (for \( \theta, \omega \), and cost) and via statistical analysis of random variation (mean error, deviation, and mean cost).

Quadratic cost was evaluated for each controller, where every controller’s model coded in MATLAB/SIMULINK is depicted in the Appendix A to aide repeatability for the reader. End conditions were compared via error norms (specifically \( \ell^2 \), also known as least squares). The controllers were designed to both meet the end conditions and minimize the cost function (often opposing goals) from the outset. Designs were iterated to get as close as possible to the end conditions while keeping cost low (a ubiquitous trade-off). In the main body of the manuscript, each controller is designed with descriptions to address the key issues of control in addition to how well they perform in a Monte Carlo analysis.

It will be shown that the optimum control for our double integrator is \( u = at + b \), and this is simply a generalized form that can be narrowed to \( u = K_p (1 - t) \theta + K_v (1 - t) \omega \) (looks somewhat like a classical PD controller). This research implements the optimum control in several manners: Open loop, Continuous-predictive, Sampled-Predictive, Feedforward/feedback, and 2 Degree-of-freedom (2DOF). Thus, the predictive controllers are derived from optimality rather than the opposite.

2.1. Open Loop Optimum Controller

This section derives the optimum control from first principles: standard articulation of an optimization problem. While, subsequent derivation of innovative control architectures uses the optimum control developed in this section, this explanation will not be repeated in subsequent discussions. The optimum controller uses optimization theory to address the key control issue of cost. The optimum controller is defined as the one that minimizes the quadratic control cost function. To find an optimum controller, we choose to optimize the double integrator cost \( J \) and the square of the control as a function of all variables annotated (\( \cdot \)) as follows.

\[
\text{Minimize } J[\theta(\cdot), \omega(\cdot), u(\cdot)] = \frac{1}{2} \int_{t_0}^{t_f} u^2 dt.
\]

Subject to \( \dot{\theta} = \omega \)
\[ \dot{\theta} = \omega \]  \hspace{1cm} (4)

\[ (\theta_0, \omega_0) = (0, 0) \]  \hspace{1cm} (5)

\[ (\theta_f - 1, \omega_f - 1, t_f - 1) = (0, 0, 0) \]  \hspace{1cm} (6)

Make some important observations. First, notice the notation. \( \theta(\cdot), \omega(\cdot), u(\cdot) \) live in infinite dimensional functional space, while \( \theta(t), \omega(t), u(t) \) live in finite dimensional space. In addition, the smooth terminal manifold is the following: \( e(\theta_f, t_f) = 0 \). We intend to minimize the control Hamiltonian.

1. Write the control Hamiltonian.

\[ H(\lambda, \theta, u, t) = F(\theta, u, t) + \lambda^T f(\theta, u, t) \]  \hspace{1cm} (7)

2. Implement the Hamiltonian Minimization Condition for the static problem of Equation (8).

\[ \min H(...; u) \text{ where } u \in \mathbb{R} \text{, that is } \frac{\partial H}{\partial u} = 0 \]  \hspace{1cm} (8)

This is a constrained minimization problem, so use Equations (9) and (10) where \( \lambda_\omega \) is the Lagrange multiplier associated with the co-state.

\[ \frac{\partial H(...; u)}{\partial u} = u + \lambda_\omega = 0 \]  \hspace{1cm} (9)

\[ u(\lambda_\omega) = -\lambda_\omega \]  \hspace{1cm} (10)

Confirm optimality by verifying the convexity condition in Equation (11).

\[ \frac{\partial^2 H(...; u)}{\partial u^2} = 1 > 0 \]  \hspace{1cm} (11)

The result: once we find the co-state, we will have optimum control. Notice Karush-Kuhn-Tucker conditions often used with inequality constraints are not necessary here.

3. Apply the Adjoint Equations per Equations (12) and (14), which result in Equations (13) and (15), respectively, and are plotted in Figure 1b.

\[ -\dot{\lambda}_\theta = \frac{\partial H}{\partial \theta} = 0 \]  \hspace{1cm} (12)

\[ \lambda_\theta = a \]  \hspace{1cm} (13)

\[ -\dot{\lambda}_\omega = \frac{\partial H}{\partial \omega} = \lambda_\theta = a \]  \hspace{1cm} (14)

Integrating (14)

\[ \lambda_\omega = -at + b \]  \hspace{1cm} (15)

4. Rewrite the Hamiltonian Minimization Condition in Equation (8) by substituting Equation (9).

\[ \frac{\partial H}{\partial u} = u + \lambda_\omega = 0 \rightarrow u = -\lambda_\omega \]  \hspace{1cm} (16)
5. Substituting Equation (15) into Equation (16) produces Equation (17).

\[ \text{Optimal control, } u^* \equiv u = at + b \] (17)

6. Implement in SIMULINK as displayed in Figure 2 using Equations (3), (18)–(20), which are simulated per Figure 2 and whose results are plotted in Figure 1a.

\[ u = \dot{\theta} = \dot{\omega} = at + b \] (18)

\[ \omega = \dot{\theta} = \int \dot{\theta} \, dt = \int \omega \, dt = \frac{a}{2}t^2 + bt + c \] (19)

\[ \theta = \int \dot{\theta} \, dt = \int (at^2 + bt + c) \, dt = \frac{a}{6}t^3 + \frac{b}{2}t^2 + ct + d \] (20)

Equation (17) will be the enforced form of the control in subsequent derivations substantiating the novel paradigm contribution for each subsequent form. Using the initial conditions \( \theta(0) = 0, \omega(0) = 0 \) and the final conditions \( \theta(1) = 1, \omega(1) = 0 \) yields the values for constants: \( a = -12, b = 6, c = 0, \) and \( d = 0. \) Thus, the optimum open loop controller is Equation (21) implemented in SIMULINK, as depicted in Figure 2.

\[ u^*_{\text{openloop}} = -12t + 6 \] (21)

It is important to remember that \( u = -12t + 6 \) is not the global optimum for the simple double integrator. We established the values \( a = -12, b = 6, \) using the boundary values stated, which makes this a local optimum control from \( 0 \leq t \leq 1. \) The more general \( u = at + b \) is the global optimum control.

**ASIDE:** We’ll see in the next section how to implement the more general optimum control and solve for constants \( a \) and \( b \) as time, \( t \) progresses. That controller is referred to as the continuous predictive optimal closed-loop controller.

Notice in Figure 1 the behavior of the extremal states, co-states (in shadow space), and control in the figures below are limited to \( 0 \leq t \leq 1. \) Observe that the state \( \theta \) is nonlinear in normal space, while the co-state is linear and sloped. Similarly, the state \( \omega \) is nonlinear in normal space, while the co-state is linear and sloped. Additionally, notice the figure displayed in Figure 2 is provided in full form in the Appendix A. In addition to the simple double integrator implementation of the optimum control, there are subsystems to calculate the resultant cost as well as \( \int \) and subsystems to display the data numerically and via plots. This format has been maintained throughout analysis of all the controllers considered in this study.

![Figure 1](image-url)

**Figure 1.** External states, co-states, and control for optimally controlled double integrator plant with time on the abscissa and values on the ordinate. (a) State trajectories, \( \theta(t) \) and \( \omega(t) \) from Equations (18)–(20). (b) Co-state trajectories in shadow space \( \lambda_{\theta}(t), \lambda_{\omega}(t) \), and control \( u(t) \) solutions to Equations (12)–(15), and (17).
2.2. Continuous Predictive Closed Loop Optimum Controller

This section briefly describes the implementation of a feedback optimum controller where the constants of integration are continuously calculated by the controller instantaneously. The basis of the derivation begins with the previously mentioned solution to the optimization problem, substantiating a novel contribution. Recall from the previous section, Equation (21) $u = -12t + 6$ is not the global optimum controller. As time progresses, the boundary conditions used to solve Equation (17) $u = at + b$ change. Additionally, the starting position changes due to movement, noise, and uncertainties. This and subsequent controllers implement the optimum control using feedback to deal with changing boundary conditions. Rather than command the controller using the local optimum solution in Equation (21), instead use the global optimal solution in Equation (17) with full state feedback. Using the present state as the new initial boundary value, we solve for the constants of integration of the general optimum control in Equation (17). Recall that $a = -12$ and $b = 6$ resulted from the specified initial and final conditions. As time progresses, this continuous predictive (closed loop) optimum controller will calculate new $a$'s and $b$'s using its present time as the initial condition.

1. Recall Equations (18)–(20) where $\theta(1) = 1$, $\omega(1) = 0$ yields:

$$\theta(1) = \frac{a}{6}(1) + \frac{b}{2}(1) + c(1) + d = \frac{a}{6} + \frac{b}{2} + c + d = 1 \tag{22}$$

$$\omega(1) = \frac{a}{2}(1) + b(1) + c = \frac{a}{2} + b + c = 0 \tag{23}$$

2. Set up a matrix equation in the form $[A][x] = [B]$ simulated in Figure 3.

$$[A] = \begin{bmatrix} \frac{a}{6} & \frac{b}{2} & l_0 & 1 \\ 1 & 1 & 1 & 1 \\ \frac{a}{6} & \frac{b}{2} & l_0 & 1 \\ 1 & 1 & c & d \end{bmatrix}, \quad [B] = \begin{bmatrix} a \\ b \\ c \\ d \end{bmatrix}, \quad \{x'\} = [A]^+ [B] \rightarrow \begin{bmatrix} x(1) \\ x(2) \\ x(3) \\ x(4) \end{bmatrix} \tag{24}$$

This matrix expression in Equation (24) can be solved in real-time as the simulation progresses in several ways. Two methods are depicted in Figure 3. There is a complication to continuously updating constants $a$ and $b$. As time approaches the end time ($t = 1$), the $[A]$ matrix becomes rank deficient, because the rows are linearly dependent. This will cause the computer program to become unstable. The integration constants in Figure 4 vary as time progresses. As $t$ approaches $t_f = 1$, the values of the constants oscillate unstably to very large values. To account for this shrinking horizon prediction control problem, an optimum open loop controller is inserted and used at these times. We disregard...
the unstable values that come out of the continuous predictive control logic. Intuition: This should increase cost slightly from the optimum case.

The LU inverse approach to dealing with matrix inverses in Figure 3 required open loop command to occur as early as $t = 0.9$, while the LU solver approach allows for smaller blind time. This is desirable, because this blind time foreshadows errors in meeting our end conditions. This is a good time to point out that it is not the value of $[A]^{-1}$. The value of $[A]^{-1}$ is an intermediate calculation step to get $[a b c d]$. Calculating the matrix inverse can be problematic, so avoid the unnecessary pitfalls by not calculating $[A]^{-1}$. Instead, use the LU solver (or QR solver, etc.) or the LU inverse.

This control is somewhat unrealistic, since a body’s onboard computer may not have access to information quickly enough to generate correct integration constants. There is a lag time associated with sensing, and onboard computers may be limited. Accordingly, the next section develops a sampled-data version of this controller, where the sensor data has delayed delivery to the control logic.

Figure 3. SIMULINK systems to solve for constants of integration in real-time online.
2.3. Sampled-Data Predictive Optimum Controller

To simulate a real-world sensor delay, a delay was inserted before the control logic in SIMULINK and the delay time was iterated considering acceptable system performance. Using a time delay of 0.05 s allowed this controller to achieve performance in the neighborhood of the other controllers in this study, which establishes the bounds of permissible time lags for performance. Poor Monte Carlo performance required decreasing the time delay to 0.025, while establishing the bounds of permissible time lags for robustness. The letters q and w in the SIMULINK model in Figure 5 are θ and ω, respectively. This is consistent throughout this report.

![Figure 5. Time-delay inserted before control logic.](image)

2.4. Proportional Plus Derivative (PD) Controller Derived Foremost from an Optimization Problem

Defining \( a = -(K_\theta + K_\omega \omega) \) and \( b = K_\theta + K_\omega \omega \), the optimum control \( u = at + b \) as a function of the time may be written as a function of states: \( u = K_\theta(1-t) + K_\omega(1-t)\omega \) where the K's are feedback gains that are functions of the θ and ω error.

The gains could be calculated online, but typically static gains are chosen by the control designer to meet some stated design criteria. This makes the controller performance somewhat arbitrary. For this reason, we can expect non-optimum performance from this controller, unless the stated design criteria comprised only optimization of an identical cost function. In this case, constant PD controller gains were iterated to address the trade-off between low cost and minimum error of stated end conditions. Since gains were iterated via a MATLAB program over a specified range, an applicable descriptive adjective is elusive. The range was finite, so iteration can be done differently by achieving gains (in a range not iterated here).
2.5. Feedforward/Feedback PD Controller

This controller feeds forward (where this topological variant justifies the use of the term “feedforward”) the time-parameterized optimal solution reparametrized as optimal states and then compares the optimum states to the current state to generate the error signal fed to the PD controller, as depicted in Figure 6. The states are taken from a double integrator with the optimum control calculated in Figure 7 corresponding to Equations (19) and (20), as the control logic input seen in Figure 8.

![Figure 6. PD Controller Implemented in SIMULINK.](image)

![Figure 7. Calculation of optimum states (Ramp = -12t + 6).](image)

![Figure 8. Feedforward/feedback implementation in SIMULINK.](image)

2.6. Two-DOF Controller: Optimal Control Augmented with Feedback Errors Calculated with Optimal States

Very similar to the previous controller, this controller uses the feedforward/feedback method and further adds another comparison to the optimum case. Once again, allowing this topological variant to justify the use of the term “feedforward,” this controller (again) feeds forward the time-parameterized optimal solution reparametrized as optimal states and then compares the optimum states to the current state to generate the PD feedback portion of the control. Next, the output of the PD controller is also compared to the optimum control in an outer loop context depicted in Figure 9, which permits the optimal (time-parameterized) control to insure good target tracking performance. The optimal control (reparametrized as states) is used for feedback to yield robustness despite not accounting for the deleterious disturbance in a more-complicated problem formulation. The utilization of the results of the optimization problem lead to the terminology “two degree of freedom” or “2DOF” to indicate the two opportunities for the control designer.
3. Results

This section is divided into two portions. The first portion summarizes the simulation analysis of the deterministic plant (the double integrator). The second portion summarizes the simulation analysis of the mismodeled plant (the double integrator with an inertia-velocity control dependence). After succinct display of the results, a detailed discussion follows that summarizes some of the main lessons learned.

3.1. Monte Carlo Analysis on a Deterministic Plant with Noise

To critically evaluate robustness, random noise was added to the deterministic plant, and identical experiments were simulated from the initial position \( \theta(0) = 0 \) to the final position \( \theta(1) = 1 \) while \( \omega(0) = \omega(1) = 0 \). Comparison of the controller’s performance is made based on the mean and standard deviations of meeting both the final conditions and also the mean cost displayed in Table 1.

![Monte Carlo Analysis Diagram]

Figure 9. 2DOF Implementation in SIMULINK.

### Table 1. Statistical analysis via Monte Carlo simulations

| Controller                  | Deviation, \( \sigma_\theta \) | Mean Error, \( \mu_\theta \) | Deviation, \( \sigma_\omega \) | Mean Error, \( \mu_\omega \) | Mean Cost, \( J \) |
|-----------------------------|-------------------------------|-------------------------------|-------------------------------|-------------------------------|-------------------|
| Optimal open loop           | 0.0328                        | 0.0439                        | 0.03                          | 0.045                         | 5.9787            |
| Continuous predictive       | 0.0015                        | 0.002                         | 0.0192                        | 0.0251                        | 6.0117            |
| Sampled predictive          | 0.0031                        | 0.0074                        | 0.0249                        | 0.0321                        | 6.1137            |
| Proportional-derivative (PD)| \( 9.66 \times 10^{-4} \)     | \( 1.41 \times 10^{-2} \)    | \( 3.14 \times 10^{-16} \)   | \( 1.02 \times 10^{-15} \)   | 59.6885           |
| Feedforward + feedback      | \( 7.86 \times 10^{-16} \)    | \( 9.83 \times 10^{-16} \)   | 0.0058                        | 0.0087                        | 8.4507            |
| 2DOF                        | \( 7.07 \times 10^{-16} \)    | \( 1.02 \times 10^{-15} \)   | \( 1.89 \times 10^{-15} \)   | \( 2.51 \times 10^{-15} \)   | 8.619             |

\(^1\) note highest and lowest values in bold font.

3.2. Open Loop Optimal Controller

First, note the open loop optimum controller. Quickly checking the Appendix A, we see that it optimally achieved the exact end conditions when no noise was present. Everything seemed perfect until we added noise to the plant: +5\% inertia noise, +0.1° \( \theta \) measurement noise, and +0.1 degrees/second \( \omega \) measurement noise. In the Appendix A, we immediately noticed missed end conditions. The 3\( \sigma \) Monte Carlo analysis allowed the \( \theta \), \( \omega \) and inertia parameters to vary by 0.1, 0.1, and 0.05, respectively. This further antagonized the open loop controller. Note the very large deviation and mean error. Note the cost was below the optimal scalar value of 6. Squashing the enthusiasm seemingly exceeded the mathematically optimal cost. The end conditions were not met precisely. Considering the errors, open loop is not the way to implement optimum control in this case.

3.3. Continuous-Update Optimal Controller

The Continuous Update Optimum Controller seems to have exactly met the stated conditions and minimized the cost without noise. This controller also handles the measurement and inertia noise very well. End conditions are nearly met at a very low cost (nearly optimum). However, the Monte Carlo analysis reveals susceptibility to random errors (especially angular velocity conditions). Mean error and deviation for angular velocity were the third highest of all the controllers recalling the handicap of a blind time. Since a continuously updated optimum controller that calculates its measurement matrix
at every instance of time is an idealization, implementation uses a sampled-data computer system. Therefore, the next section examines the Sample Predictive Optimum Controller.

3.4. Sample-Predictive Optimum Controller

The angular position and velocity were sampled at a rate simulating delays in processing and commanding onboard a real body, and the Sampled Predictive Controller performed very well. Selecting a time delay of 0.05 s with no noise, the controller performed very close to optimum levels, and performed modestly well (at least as good as our PD controller to be discussed shortly) in the presence of noise. Monte Carlo analysis revealed the same vulnerability to random conditions seen with the Continuously Updated Controller (particularly susceptible to random angular velocity conditions). The delay time was reduced to 0.025 s to regain a nominal performance. However, its performance mimics the continuously updated case so that we could consider real-world implementation of this controller, if the controller’s application can accept the high angular velocity error and deviation.

3.5. PD Control Derived Foremost from an Optimization Problem

The PD controller does a modest job at achieving the stated end conditions even in the presence of noise, but the cost is huge. As the controller was tuned to more exactly meet the end conditions, cost rose in illustration of the ubiquitous trade-off. The relative immunity to noise foreshadowed its very good performance in the Monte Carlo analysis. The PD controller was not the absolute best controllers regarding \( \theta \) deviation and mean error. However, it was the best controller when considering \( \omega \) deviation and mean error. The PD controller does not reveal a susceptibility to random angular velocity conditions that we noted earlier with other controllers. The downside is the cost, which is the highest cost by far. The PD controller design was adjusted to get the nominal cost down to near 30, but the cost sky-rocketed to more than 50 during Monte Carlo analysis. In the next section, the optimum feedforward control will be added to augment the PD controller.

3.6. Feedforward/Feedback PD Controller

Applying the optimum control states to the control logic of our PD controller lead to the Feedforward/Feedback PD Controller, which performed very well. We nearly exactly met the stated end conditions with very low cost. Some susceptibility to noise leads to slightly increased missing of the end conditions and a slightly increased cost (yet remaining far superior to the PD controller). This foreshadows the good performance evident in the random analysis displayed in Figure 10e. The feedforward/feedback controller had the second-best \( \theta \) deviation and mean error of all the controllers. It showed some modest vulnerability to random errors in \( \omega \) deviation and error. The significant part is the vastly improved cost. This controller did a very good job meeting end conditions while simultaneously keeping cost pretty low. The next section examines the addition of the optimum control in addition to the optimum states via the 2DOF Controller.

3.7. Two-DOF Controller: Optimal Control Augmented with Feedback Errors Calculated with Optimal States

The 2DOF controller is another very good controller in the presence of noise (noting a similar increase in cost with the feedforward/feedback controller). In Table 1, the 2DOF controller eliminated the \( \omega \) deviation and mean error vulnerability to random noise that we saw with the other optimum controllers. The cost was (very slightly) higher than we saw with the feedforward/feedback controller. Using an applicable label from the sport of boxing, this controller may be viewed as the “pound-for-pound” best controller. It essentially matched the #1 performance of the Feedforward/Feedback controller for \( \theta \) deviation and mean error. Simultaneously, it nearly matched the #1 performance of the PD controller for \( \omega \) deviation and mean error. All the while, it was also superior regarding cost. If the control requirements can accommodate the slightly higher \( \omega \) deviation and mean error, the feedforward/feedback controller is best based on lower cost. Otherwise, the 2DOF controller is clearly superior.
were not correct). The controllers experienced slightly degraded performance overall, that, in most

Applying these controllers to a mismodeled plant, notice in Table 2 that many of the controllers still

perform very well (surprising, since the controls were based in optimization of analytic dynamics that

were not correct). The controllers experienced slightly degraded performance overall, that, in most
cases, was well within expected tolerances for most simple applications.

Table 2. Statistical Analysis or mis-modeled plant ¹.

| Controller                  | Deviation, \( \sigma_\theta \) | Mean Error, \( \mu_\theta \) | Deviation, \( \sigma_\omega \) | Mean Error, \( \mu_\omega \) | Mean Cost, J |
|-----------------------------|---------------------------------|-------------------------------|-------------------------------|-------------------------------|--------------|
| Optimal open loop           | 0.034                           | 0.044                         | 0.0283                        | 0.0484                        | 6.004        |
| Continuous predictive       | 0.0015                          | 0.0019                        | 0.02                           | 0.025                         | 6.0137       |
| Sampled predictive          | 0.0029                          | 0.007                         | 0.0222                        | 0.0307                        | 6.0183       |
| Proportional-derivative (PD)| \(9.89 \times 10^{-4}\)         | 0.014                         | \(3.12 \times 10^{-16}\)     | \(1.02 \times 10^{-15}\)     | 59.3244      |
| Feedforward + feedback      | \(7.36 \times 10^{-16}\)        | \(9.69 \times 10^{-16}\)     | 0.0058                        | 0.0086                        | 8.9637       |
| 2DOF                        | \(6.42 \times 10^{-16}\)        | \(9.02 \times 10^{-16}\)     | 1.90 \times 10^{-15}         | 2.56 \times 10^{-15}         | 7.8284       |

¹ note highest and lowest values in bold font.

If the situation demands precise attainment of the final conditions with mismodeled plants, we can
deduce four tiers of controllers: (1) 2 DOF PD controller, (2) PD controller and Feedforward/Feedback
PD controller, and (3) Continuous, and Sampled Predictive controllers, and (4) Optimal Open Loop
Controller. The first tier (the 2DOF PD controller) met the stated end conditions within a miniscule
error and maintained a very low-cost number (while not optimum). The second tier (PD and
Feedforward/Feedback PD) each did well at meeting one of the stated end conditions, but showed
increased error in the other end condition. The third tier (Continuous/Sampled Optimal Controllers) had increased error in both stated end conditions, while the final tier (the open loop controller) was inappropriate for application due to high errors and deviation in end conditions.

If the requirements on stated end conditions can be relaxed sufficiently to emphasize minimum cost, the tiers of results would change. The optimum open loop controller would join the first tier due to its minimum cost, but such a re-ranking will not be done in this case due to the large error achieving end conditions. The first tier would include the continuous and the sample optimum controllers. They achieved minimum cost while also closely meeting the end conditions. Next, the second tier would include feedback/feedforward and 2DOF PD controllers. They had slightly higher costs while very precisely meeting the stated end conditions. Third, the PD controller had very high cost, but also did a good job meeting the end conditions.

4. Discussion

Predictive controllers (both continuous and sampled-data) derived from the outset to be optimal by first solving an optimization problem with the governing dynamic equations of motion lead to several controllers (including a controller that twice invokes optimization to formulate robust, predictive control, which was referred to as a two degree-of-freedom controller). These controllers were compared to each other with noise and modeling errors in Table 3. Many figures of merit are used: tracking error and rate error deviations and means, in addition to total mean cost. Robustness was evaluated using Monte Carlo analysis where plant parameters were randomly assumed to be incorrectly modeled. Six instances of controllers were compared against these methods and interpretations allow engineers to select a tailored control for their given circumstances. Optimal control, predictive control, discrete (sampled-data) control, proportional-derivative control, feedforward + feedback control, and a proposed 2DOF control were all derived with the articulation of an individual optimization problem. Control structures were then designed to enforce optimality, rather than the ubiquitous methodology of first asserting the control structure, then subsequently seeking to optimize the imposed structure. The proposed comparison paradigm utilized six disparate types of control that were compared against each other’s ability to meet commanded end conditions at minimum cost in the face of both noise and modeling errors.

| Controller                        | Benefits                  | Weakness                          |
|-----------------------------------|---------------------------|-----------------------------------|
| Optimal open loop                 | Establishes optimal case  | Not realistically implementable    |
| Continuous predictive *           | Good cost and position    | High rate error and deviation     |
| Sampled predictive *              | Good cost and position    | High rate error and deviation     |
| Proportional-derivative (PD) *    | Best rate control         | Worst cost                        |
| Feedforward + feedback            | Best position control     | Slight rate error and deviation   |
| 2DOF *                            | All-around good           | Slightly higher cost than optimal |

Table 3. Summary 1 of the deterministic plant.

1 Large generalizations; * Not the ubiquitous PD topology.

The optimal open loop controller established the baseline cost (being the mathematical minimum possible), but was not realistically capable of implementation due to lacking feedback. Continuous and discrete (sampled-data) predictive controls proved very good at achieving the commanded final position with very low cost, but revealed a tolerable, but non-trivial, susceptibility to noise and mismodeling only with regard to difficulties achieving the final commanded velocity state. Very much akin the classical PD controller, the proportional-derivative controller developed under the new design paradigm was very good at state control (achieving the best rate control), but had the highest cost by a very wide margin compared to all of the other control options. Akin to classical feedforward + feedback controllers, the controller designed under the new paradigm achieved the very best position control at very good cost. However, it has some slight susceptibility to noise and
modeling errors with regard to the rate control. The proposed 2DOF controller was the all-around best controller, achieving near-perfect state controls with very low (suboptimal) cost. The controller had no performance degradation due to noise or modeling errors.

*Proposed 2DOF control designed foremost as an optimization problem: The open-loop optimal control is used as a feedforward, while the optimal states derived from a time-parameterized optimal control are compared to the feedback signal to generate the error fed to the feedback controller whose gains are a reparameterization of the optimal solution.*

5. Future Works

Noise was not treated in the problem formulation, but the proposed 2DOF controller achieved near-precision accuracy with near-minimal costs in the face of noise. It remains of interest to investigate whether further performance improvement is possible by augmenting the problem formulation with smooth filters (e.g., Kalman filters). Since target tracking accuracy was near machine-precision even with modeling errors and noise, the investigation of smoothing filters should seek to further reduce the already low-cost value.

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**Appendix A**

The appendix contains detailed explanations of experimental details that would disrupt the flow of the main text, but, nonetheless, remain crucial to understanding and reproducing the research shown. It contains figures of replicates for simulation experiments of which representative data is shown in the main text. For each of the controllers implemented, figures of the SIMULINK models used to generate the results are provided and summarized in this research report. Each system was tested with no noise and then with a +1% angle and rate sensor noise and 5% body inertia noise. Each system was compared to the optimum open loop controller (with no noise) via l2 error norms.

1. **Optimum Open Loop no noise (Figure A1)**
2. **Optimum Open Loop with noise (Figure A2)**
3. **PD Controller no noise (Figure A3)**
4. **PD Controller with noise (Figure A4)**
5. **Continuous Predictive Optimum no noise (Figure A5)**
6. **Continuous Predictive Optimum with noise (Figure A6)**
7. **Feedforward/Feedback PD no noise (Figure A7)**
8. **Feedforward/Feedback PD with noise (Figure A8)**
9. **2DOF PD Controller no noise (Figure A9)**
10. **2DOF PD Controller with noise (Figure A10)**
11. **Sampled Predictive Controller without Noise (Figure A11)**
12. **Sampled Predictive Controller with Noise (Figure A12)**
13. **Mis-modeled Plant (Figure A13)**
14. **Error Norms (Figure A14)**
Appendix A.1. Optimum Open Loop Controller-No Noise

![Simulation Model](image)

**Figure A1.** Optimum open loop controller simulation model and results.

Appendix A.2. Optimum Open Loop Controller-with Noise

![Simulation Model with Noise](image)

**Figure A2.** Optimum open loop controller simulation model with noise and results.
Appendix A.3. PD Controller No Noise

(a) Proportional-derivative (PD) controller simulation model.

(b) $\theta(t) \text{ vs. } t$

(c) $\omega(t) \text{ vs. } t$

(d) Cost $J(t)$ vs. $t$

(e) $\|\theta_a(t) - \theta(t)\| \text{ vs. } t$

(f) $\|\omega_a(t) - \omega(t)\| \text{ vs. } t$

(g) $\|u(t)\| \text{ vs. } t$

Figure A3. Proportional-derivative (PD) controller ($K_w = -13.3$, $K_q = 40$) simulation model.
Appendix A.4. PD Controller with Noise

(a) Proportional-derivative (PD) controller simulation model.

(b) Proportional-derivative (PD) controlled simulation plant-only model.

Figure A4. Proportional-derivative (PD) controller ($K_w = -13.3$, $K_q = 40$) simulation model with noise.
Appendix A.5. Continuous Predictive Optimum Controller-No Noise

Figure A5. Continuous predictive optimum controlled simulation model.
Appendix A.6. Continuous Predictive Optimum Controller-with Noise

(a) Continuous predictive optimal controlled simulation model with noise.

(b) Continuous predictive optimal controlled simulation model plant-only with noise.

Figure A6. Continuous predictive optimum controlled simulation model with noise.
Appendix A.7. Feedforward/Feedback PD Controller-No Noise

Figure A7. Feedforward/feedback PD controlled simulation.
Appendix A.8. Feedforward/Feedback PD Controller-with Noise

(a) Feedforward/feedback-controlled plant-only simulation model with noise.

(b) Feedforward/feedback-controlled simulation model with noise.

Figure A8. Feedforward/feedback PD controlled simulation with noise.
Appendix A.9. 2DOF PD Controller-No Noise

(a) 2DOF PD controlled plant-only simulation model.

(b) $\theta(t)$ vs. $t$

(c) $\omega(t)$ vs. $t$

(d) $J(t)$ vs. $t$

(e) $\|\theta_d(t) - \theta(t)\| vs. t$

(f) $\|\omega_d(t) - \omega(t)\| vs. t$

(g) $\|u(t)\| vs. t$

Figure A9. 2DOF PD controlled simulation.
Appendix A.10. 2DOF PD Controller-with Noise

(a) 2DOF PD controlled plant-only simulation plant-only model with noise.

(b) 2DOF PD controlled plant-only simulation model with noise.

Figure A10. 2DOF PD controlled simulation with noise.
Appendix A.11. Sampled Predictive Controller-without Noise

(a) Sampled predictive controlled simulation model.

(b) $\theta(t)$ vs. $t$
(c) $\omega(t)$ vs. $t$
(d) $f(t)$ vs. $t$

(e) $||\theta_d(t) - \theta(t)||$ vs. $t$
(f) $||\omega_d(t) - \omega(t)||$ vs. $t$
(g) $||u(t)||$ vs. $t$

Figure A11. Sampled predictive controlled simulation.
Appendix A.12. Sampled Predictive Controller-with Noise

(a) Sampled predictive controlled simulation model with noise.

(b) $\theta(t)$ vs. $t$

(c) $\omega(t)$ vs. $t$

(d) $J(t)$ vs. $t$

(e) $||\theta_d(t) - \theta(t)||$ vs. $t$

(f) $||\omega_d(t) - \omega(t)||$ vs. $t$

(g) $\|u(t)\|$ vs. $t$

Figure A12. Sampled predictive controlled simulation with noise.
Appendix A.13. Mismodeled Plant

Figure A13. Mismodeled plant inserted into previous programming.

Appendix A.14. Error Norms

Figure A14. Error norm for optimum controller ($\mathcal{L}_2 = 0$ when compared to itself).

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