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Whys and Wherefores of Transmissibility

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1. Introduction

The present chapter draws a general overview on the concept of transmissibility and on its potentialities, virtues, limitations and possible applications. The notion of transmissibility has, for a long time, been limited to the single degree-of-freedom (SDOF) system; it is only in the last ten years that the concept has evolved in a consistent manner to a generalized definition applicable to a multiple degree-of-freedom (MDOF) system. Such a generalization can be and has been not only developed in terms of a relation between two sets of harmonic responses for a given loading, but also between applied harmonic forces and corresponding reactions. Extensions to comply with random motions and random forces have also been achieved. From the establishment of the various formulations it was possible to deduce and understand several important properties, which allow for diverse applications that have been envisaged, such as evaluation of unmeasured frequency response functions (FRFs), estimation of reaction forces and detection of damage in a structure. All these aspects are reviewed and described in a logical sequence along this chapter.

The notion of transmissibility is presented in every classic textbook on vibrations, associated to the single degree-of-freedom system, when its basis is moving harmonically; it is defined as the ratio between the modulus of the response amplitude and the modulus of the imposed amplitude of motion. Its study enhances some interesting aspects, namely the fact that beyond a certain imposed frequency there is an attenuation in the response amplitude, compared to the input one, i.e., one enters into an isolated region of the spectrum. This enables the design of modifications on the dynamic properties so that the system becomes “more isolated” than before, as its transmissibility has decreased.

Usually, the transmissibility of forces, defined as the ratio between the modulus of the transmitted force magnitude to the ground and the modulus of the imposed force magnitude, is also deduced and the conclusion is that the mathematical formula of the transmissibility of forces is exactly the same as for the transmissibility of displacements. As it will be explained, this is not the case for multiple degree of freedom systems.

The question that arises is how to extend the idea of transmissibility to a system with N degrees-of-freedom, i.e., how to relate a set of unknown responses to another set of known responses, for a given set of applied forces, or how to evaluate a set of reaction forces from a set of applied ones. Some initial attempts were given by Vakakis et al. (Paipetis & Vakakis, 1985; Vakakis, 1985; Vakakis & Paipetis, 1985; 1986), although that generalization was still
limited to a very particular type of N degree-of-freedom system, one where a set constituted by a mass, stiffness and damper is repeated several times in the vertical direction. The works of (Liu & Ewins, 1998), (Liu, 2000) and (Varoto & McConnell, 1998) also extend the initial concept to N degrees-of-freedom systems, but again in a limited way, the former using a definition that makes the calculations dependent on the path taken between the considered co-ordinates involved, the latter by making the set of co-ordinates where the displacements are known coincident to the set of applied forces.

An application where the transmissibility seems of great interest is when in field service one cannot measure the response at some co-ordinates of the structure. If the transmissibility could be evaluated in the laboratory or theoretically (numerically) beforehand, then by measuring in service some responses one would be able to estimate the responses at the inaccessible co-ordinates.

To the best knowledge of the authors, the first time that a general answer to the problem has been given was in 1998, by (Ribeiro, 1998). Surprisingly enough, as the solution is very simple indeed. In what follows, a chronological description of the evolution of the studies on this subject is presented.

2. Transmissibility of motion

In this section and next sub-sections the main definitions, properties and applications will be presented.

2.1 Fundamental formulation

The fundamental deduction (Ribeiro, 1998), based on harmonically applied forces (easy to generalize to periodic ones), begins with the relationships between responses and forces in terms of receptance: if one has a vector \( F_A \) of magnitudes of the applied forces at co-ordinates \( A \), a vector \( X_U \) of unknown response amplitudes at co-ordinates \( U \) and a vector \( X_K \) of known response amplitudes at co-ordinates \( K \), as shown in Fig. 1.

One may establish the following relationships:

\[
X_U = H_{UA} F_A
\]

Fig. 1. System with co-ordinates \( A, U, K \)
\[ X_K = H_{KA} F_A \]  

(2)

where \( H_{UA} \) and \( H_{KA} \) are the receptance frequency response matrices relating co-ordinates \( U \) and \( A \), and \( K \) and \( A \), respectively. Eliminating \( F_A \) between (1) and (2), it follows that

\[ X_U = H_{UA} H_{KA}^\dagger X_K \]  

(3)

or

\[ X_U = T_{UK}^{(A)} X_K \]  

(4)

where \( H_{KA}^\dagger \) is the pseudo-inverse of \( H_{KA} \). Thus, the transmissibility matrix is defined as:

\[ T_{UK}^{(A)} = H_{UA} H_{KA}^\dagger \]  

(5)

Note that the set of co-ordinates where the forces are (or may be) applied \((A)\) need not coincide with the set of known responses \((K)\). The only restriction is that – for the pseudo-inverse to exist – the number of \( K \) co-ordinates must be greater or equal than the number of \( A \) co-ordinates.

An important property of the transmissibility matrix is that it does not depend on the magnitude of the forces, one simply has to know or to choose the co-ordinates where the forces are going to be applied (or not, as one can even choose more co-ordinates \( A \) if one is not sure whether or not there will be some forces there and, later on, one states that those forces are zero) and measure the necessary frequency-response-functions.

2.2 Alternative formulation

An alternative approach, developed by (Ribeiro et al., 2005) evaluates the transmissibility matrix from the dynamic stiffness matrices, where the spatial properties (mass, stiffness, etc.) are explicitly included.

The dynamic behaviour of an MDOF system can be described in the frequency domain by the following equation (assuming harmonic loading):

\[ Z X = F \]  

(6)

where \( Z \) represents the dynamic stiffness matrix, \( X \) is the vector of the amplitudes of the dynamic responses and \( F \) represents the vector of the amplitudes of the dynamic loads applied to the system.

From the set of dynamic responses, as defined before, it is possible to distinguish between two subsets of co-ordinates \( K \) and \( U \); from the set of dynamic loads it is also possible to distinguish between two subsets, \( A \) and \( B \), where \( A \) is the subset where dynamic loads may be applied and \( B \) is the set formed of the remaining co-ordinates, where dynamic loads are never applied. One can write \( X \) and \( F \) as:

\[ X = \begin{bmatrix} X_K \\ X_U \end{bmatrix}, \quad F = \begin{bmatrix} F_A \\ F_B \end{bmatrix} \]  

(7)

With these subsets, Eq. (6) can be partitioned accordingly:
Taking into account that co-ordinates $B$ represent the ones where the dynamic loads are never applied, and considering that the number of these co-ordinates is greater or equal to the number of co-ordinates $U$, from Eq. (8) it is possible to obtain the unknown response vector:

$$F_B = 0, \quad \# B \geq \# U$$

$$X_{UL} = -Z_{BU}^T Z_{BK} X_K$$

where $Z_{BU}^T$ is the pseudo-inverse of $Z_{BU}$. Therefore, this means that the transmissibility matrix can also be defined as

$$T_{UK}^{(4)} = -Z_{BU}^T Z_{BK}$$

Eq. (10) is an alternative definition of transmissibility, based on the dynamic stiffness matrices of the structure. Therefore,

$$T_{UK}^{(4)} = H_{UA} H_{KA} = -Z_{BU}^T Z_{BK}$$

(11)

Taking into account that the dynamic stiffness matrix for an undamped system is described in terms of the stiffness and mass matrices, $Z = K - \omega^2 M$, one can now relate the transmissibility functions to the spatial properties of the system. To make this possible, one must bear in mind that it is mandatory that both conditions regarding the number of co-ordinates be valid, i.e.,

$$\# B \geq \# U \quad \text{and} \quad \# K \geq \# A$$

(12)

### 2.3 Numerical example

An MDOF mass-spring system, presented in Fig. 2, will be used to illustrate the principal differences observed between the transmissibilities and FRFs curves. This is a six mass-spring system (designated as original system), possessing the characteristics described in Table 1.

$$X_K = [X_2 \quad X_4 \quad X_6]^T \quad \text{and} \quad X_U = [X_1 \quad X_3 \quad X_5]^T$$

(13)

The number of loads can be grouped in the sub-set $F_A$ (even if some of them are, in certain cases, null) and in subset $F_B$.

$$F_A = [F_4 \quad F_5 \quad F_6]^T \quad \text{and} \quad F_B = [F_1 \quad F_2 \quad F_3]^T$$

(14)

The subsets of known and unknown responses are assumed as:

According to Eq. (11), and considering the above-defined subsets, the transmissibility matrix is given by:
|                   | Original System |
|------------------|-----------------|
| kg               | mₙ              |
| m₁               | 7               |
| m₂               | 7               |
| m₃               | 4               |
| m₄               | 3               |
| m₅               | 6               |
| m₆               | 8               |
| N/m              | kᵢ             |
| k₁               | 10⁻⁶           |
| k₂               | 10⁻⁶           |
| k₃               | 4.0x10⁻⁶       |
| k₄               | 5.0x10⁻⁶       |
| k₅               | 7.0x10⁻⁶       |
| k₆               | 2.0x10⁻⁶       |
| k₇               | 8.0x10⁻⁶       |
| k₈               | 3.0x10⁻⁶       |
| k₉               | 6.0x10⁻⁶       |
| k₁₀              | 3.0x10⁻⁶       |
| k₁₁              | 5.0x10⁻⁶       |

Table 1. Characteristics of the original system

![Mass-spring MDOF system](image-url)

Fig. 2. Mass-spring MDOF system
The characteristics of the system of Fig. 2 are presented in Table 1. It may be noted from Fig. 3 that the maxima of the transmissibility curves occur all at the same frequencies. It can also be observed that the maxima and minima of the transmissibility curves do not coincide with the maxima and minima of the FRF curves. No simple relationships (if any) can be established between the picks and anti-picks of the transmissibilities and FRFs. Transmissibilities have a local nature and therefore they do not reflect the existence of the global properties of the system (natural frequencies and damping ratios).

\[
\begin{bmatrix}
T_{12}^{(A)} & T_{14}^{(A)} & T_{16}^{(A)} \\
T_{32}^{(A)} & T_{34}^{(A)} & T_{36}^{(A)} \\
T_{52}^{(A)} & T_{54}^{(A)} & T_{56}^{(A)} \\
\end{bmatrix}
= 
\begin{bmatrix}
H_{14} & H_{15} & H_{16} \\
H_{34} & H_{35} & H_{36} \\
H_{54} & H_{55} & H_{56} \\
\end{bmatrix}
\begin{bmatrix}
H_{24} & H_{25} & H_{26} \\
H_{44} & H_{45} & H_{46} \\
H_{64} & H_{65} & H_{66} \\
\end{bmatrix}
\begin{bmatrix}
Z_{11} & Z_{13} & Z_{15} \\
Z_{21} & Z_{23} & Z_{25} \\
Z_{31} & Z_{33} & Z_{35} \\
\end{bmatrix}
\begin{bmatrix}
Z_{12} & Z_{14} & Z_{16} \\
Z_{22} & Z_{24} & Z_{26} \\
Z_{32} & Z_{34} & Z_{36} \\
\end{bmatrix}
\]

(15)

Fig. 3. Some transmissibilities and FRFs curves of the original system

2.4 Transmissibility properties

The formulation presented in section 2.1 allows us to extract some important properties for the transmissibility matrix \( T_{UK}^{(A)} \). From Eq. (3) and (5) it is possible to conclude that the transmissibility matrix is independent from the force vector \( F_A \) (Note that \( F_A \) is eliminated between eqs. (1) and (2)). This means that any change verified in one of the force values, acting along with co-ordinates of set \( A \), will not affect \( T_{UK}^{(A)} \). This change can be due, for instance, to the alteration of mass values associated to co-ordinates \( A \) or stiffness values of springs interconnecting those co-ordinates.

Additionally, to highlight that characteristic of matrix \( T_{UK}^{(A)} \), it can be verified from Eq. (10) that there is no part of matrix \( Z \) involving co-ordinates of set \( A \) (neither \( Z_{AK} \) nor \( Z_{AU} \)). This statement reinforces the previous conclusion extracted from Eq. (5) and will lead to the formulation of two properties, as follows:

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Property 1. The transmissibility matrix does not change if some modification is made on the mass values of the system where the loads can be applied – subset A.

Property 2. The transmissibility matrix does not change if some modification is made on the stiffness values of springs interconnecting co-ordinates of subset A – (where the loads can be applied).

In fact, any changes in the mass values associated to co-ordinates A and/or any changes in the stiffness values of springs interconnecting co-ordinates A, will affect the inertia forces and elastic forces, respectively, acting along those co-ordinates and thus belonging to $F_A$.

The same MDOF system presented in Fig. 2 will be used to illustrate the transmissibility properties. In Table 2 four different modifications are made in the original system. Situations I and II correspond to modifications on the original masses; situations III and IV correspond to modifications on stiffness. Situations I and III only involve co-ordinates A, whereas situations II and IV involve co-ordinates pertaining to both sets A and B.

Choosing, for instance, the transmissibility function $T_{12}^{(A)}$, one obtains the results presented in Figs. 4 and 5, where one can see that $T_{12}^{(A)}$ and $T_{14}^{(A)}$ remain the same only when changes are made at co-ordinates A, where the forces are applied.

### 2.5 Evaluation of the transmissibility from measurement responses

In 1999, (Ribeiro et al., 1999) and (Maia et al., 1999) showed how the transmissibility matrix could be evaluated directly from the measurement of the responses, rather than measuring the frequency response functions. In Eq. (4), the problem is to evaluate the $U \times K$ values of $T_{UK}^{(i)}$ knowing $X_U$ and $X_K$. This can be achieved by applying various sets of forces, at a time, on co-ordinates $A$. Let $F_A^{(i)}$ be the first set of applied forces (amplitudes). Then,

$$X_U^{(i)} = T_{UK}^{(i)} X_K^{(i)} \quad (16)$$

---

| Original System | Situation I | Situation II | Situation III | Situation IV |
|-----------------|-------------|--------------|---------------|--------------|
| kg              | m1 7        | 14           | 14            | --           |
|                 | m2 7        | --           | --            | --           |
|                 | m3 4        | --           | --            | --           |
|                 | m4 3        | 13           | --            | --           |
|                 | m5 6        | 12           | --            | --           |
|                 | m6 8        | --           | 13            | 13           |
| N/m             | k1 $10^5$   | --           | --            | --           |
|                 | k2 $10^5$   | --           | --            | --           |
|                 | k3 $4.0 \times 10^5$ | -- | -- | -- |
|                 | k4 $5.0 \times 10^5$ | -- | -- | $10.0 \times 10^5$ |
|                 | k5 $7.0 \times 10^5$ | -- | -- | -- |
|                 | k6 $2.0 \times 10^5$ | -- | -- | -- |
|                 | k7 $8.0 \times 10^5$ | -- | -- | -- |
|                 | k8 $3.0 \times 10^5$ | -- | -- | -- |
|                 | k9 $6.0 \times 10^5$ | -- | -- | -- |
|                 | k10 $3.0 \times 10^5$ | -- | -- | $9.0 \times 10^5$ |
|                 | k11 $5.0 \times 10^5$ | -- | -- | $9.0 \times 10^6$ |

-- unchanged value

Table 2. Characteristics of the modifications made in the original system
However, when forces change, the transmissibility matrix does not, although the responses themselves do. Therefore, if another test is performed, with a set of forces linearly independent of the first one – though applied at the same co-ordinates – $F^{(2)}_A$, a new set of $U$ equations can be obtained, which are linearly independent of the first ones. If $K$ tests are undertaken on the structure, with linearly independent forces always applied at the $A$ set of co-ordinates, one can obtain a system of $U \times K$ equations to solve for the same number of $T^{(4)}_{UK}$ unknowns:

$$\begin{bmatrix} X^{(1)}_U & X^{(2)}_U & \cdots & X^{(K)}_U \end{bmatrix} = T^{(4)}_{UK} \begin{bmatrix} X^{(1)}_K & X^{(2)}_K & \cdots & X^{(K)}_K \end{bmatrix}$$

(17)

From which

$$T^{(4)}_{UK} = \begin{bmatrix} \begin{bmatrix} X^{(1)}_U & X^{(2)}_U & \cdots & X^{(K)}_U \end{bmatrix} & \begin{bmatrix} X^{(1)}_U & X^{(2)}_U & \cdots & X^{(K)}_U \end{bmatrix} \end{bmatrix}^{-1}$$

(18)

Fig. 4. Transmissibility $T^{(4)}_{A1}$, for the original system and the modified systems I e II

Fig. 5. Transmissibility $T^{(4)}_{A1}$, for the original system and the modified systems III e IV
Note: an easy way to obtain the \( K \) sets of linearly independent forces is to apply a single force at each of the \( A \) locations at a time.

### 2.6 The distributed forces case

In (Ribeiro, Maia, & Silva, 2000b) the authors discussed the transmissibility between the two sets of co-ordinates \( U \) and \( K \) when a distributed force is applied to the structure. Such a force should be discretized to form the \( A \) set of applied forces. The issue was then to study what happened if one only took a subset of those co-ordinates \( A \), which implies the reduction or condensation of the applied forces to such a subset and to ask the question: “how to study the transmissibility of responses from a set of condensed forces?” Let the set \( A \) be composed by the set \( C \) to where one wishes to condense the forces and the set \( D \) of the remaining co-ordinates, so that:

\[
F_A = \begin{bmatrix} F_C \\ F_D \end{bmatrix}
\]  
(19)

If one wishes to condense \( F_A \) to \( F_C \), one needs to assume some relationship between \( F_D \) and \( F_C \), i.e., one cannot contemplate the case where all the applied forces are completely independent from each other. However, this is not a big restriction, as it seems reasonable to expect that the applied forces exhibit a more or less fixed spatial pattern along the structure. Therefore, let us assume a linear relationship between the sets of forces \( F_D \) and \( F_C \), through the matrix \( P_{DC} \):

\[
F_D = P_{DC} F_C
\]  
(20)

If matrices \( H_{UA} \) and \( H_{KA} \) from eqs. (1) and (2) are partitioned into

\[
H_{UA} = \begin{bmatrix} H_{UC} & H_{UD} \end{bmatrix}
\]  
(21)

\[
H_{KA} = \begin{bmatrix} H_{KC} & H_{KD} \end{bmatrix}
\]  
(22)

and one has Eq. (19) into account, then eqs. (1) and (2) become

\[
X_U = H_{UC} F_C + H_{UD} F_D
\]  
(23)

\[
X_K = H_{KC} F_C + H_{KD} F_D
\]  
(24)

Substituting Eq. (20) in eqs. (23) and (24) and eliminating \( F_C \), it follows that

\[
X_U = (H_{UC} + H_{UD} P_{DC})(H_{KC} + H_{KD} P_{DC})^T X_K
\]  
(25)

Therefore, one has now, instead of Eq. (4), another one relating \( X_U \) and \( X_K \), through a new transmissibility matrix referred to the new subset of co-ordinates \( C \):

\[
X_U = T_{UK}^{(C)} X_K
\]  
(26)
where

\[
T_{UK}^{(C)} = (H_{UC} + H_{UD} P_{DC})(H_{KC} + H_{KD} P_{DC})^T
\]  

(27)

Note: for the pseudo-inverse to exist, the number of \( K \) co-ordinates must be higher than the number of \( C \) co-ordinates. This is obviously verified, as \( \#K > \#A \) and \( \#A > \#C \). It should also be stressed that although referred to a reduced set of co-ordinates \( C \) where the forces are applied, the transmissibility matrix still relates the responses \( U \) and \( K \) and so it keeps the same size.

In (Ribeiro, Maia, & Silva, 2000a), (Ribeiro, Maia, & Silva, 2000b) and (Maia et al., 2001) the authors summarize some of the previous works and suggest some other possible applications for the transmissibility concept, namely in the area of damage detection. In this area, there has been some activity trying to use the transmissibility as defined, as well as some other variations of it, with limited results in [(Sampaio et al., 1999; 2000; 2001)], but with some promising evolution in [(Maia et al., 2007)].

An example is presented in order to illustrate the above discussion: a cantilevered beam is subjected to the loading shown in Fig. 6.

Fig. 6. Example of a loaded beam

It is assumed that forces are applied at co-ordinates 1 to 6 but the forces at co-ordinates 2 to 5 can be related to those applied at 1 and 6 through the expression:

\[
F_D = P_{DC} F_C \quad \Rightarrow \quad \begin{bmatrix} f_2 \\ f_3 \\ f_4 \\ f_5 \end{bmatrix} = \begin{bmatrix} 4 & 1 \\ 3 & 2 \\ 2 & 3 \\ 1 & 4 \end{bmatrix} \begin{bmatrix} f_1 \\ f_6 \end{bmatrix}
\]  

(28)

One can further assume that the responses at co-ordinates 1 and 2 can be measured and those at co-ordinates 4 and 5 can be computed through the transmissibility, i.e.,

\[
X_U = T_{UK}^{(C)} X_K, \text{ with } X_U = \begin{bmatrix} X_4 \\ X_5 \end{bmatrix}, X_K = \begin{bmatrix} X_1 \\ X_2 \end{bmatrix}
\]  

and
2.7 The random forces case

Often one has to deal with random forces, for instance when a structure is submitted to environmental loads. The cases that have been addressed so far were limited to harmonic or periodic forces. The generalization to random forces has been derived in \((\text{Ribeiro et al., 2002; Fontul et al., 2004})\), now in terms of power spectral densities, rather than in terms of response amplitudes. Let \(S_{XX}\) denote the auto-spectral density of the responses \(X_X\) and \(S_{XU}\) the cross-spectral density between responses \(X_X\) and \(U_X\). Then, it can be shown (see \[(\text{Fontul, 2005})\] for specific details) that both are related through the same transmissibility matrix as before (using Eq. (5), for instance):

\[
T_{UK}^{(C)} = (H_{DC} + H_{DD} P_{DC})(H_{KC} + H_{KD} P_{DC})^* \Rightarrow
\]

\[
T_{UK}^{(C)} = \begin{bmatrix}
H_{41} & H_{46} \\
H_{51} & H_{56}
\end{bmatrix} + \frac{1}{5} \begin{bmatrix}
H_{42} & H_{43} & H_{44} & H_{45} \\
H_{52} & H_{53} & H_{54} & H_{55}
\end{bmatrix} \begin{bmatrix}
4 & 1 \\
3 & 2 \\
2 & 3 \\
1 & 4
\end{bmatrix}
\]

(29)

2.8 Some possible applications

2.8.1 Transmissibility of motion in structural coupling

This topic has been addressed in (Devriendt, 2004; Ribeiro et al., 2004; Devriendt & Fontul, 2005). Let us consider a main structure, to which an additional structure is coupled through some coupling co-ordinates, i.e., the additional structure applies a set of forces (and moments) to the main structure. As the transmissibility between two sets of responses on the main structure does not depend on the magnitude of those forces, the transmissibility matrix of the main structure is equal to the transmissibility matrix of the total structure (main + additional). In other words, under certain conditions, the transmissibility matrix of the main structure remains unchanged, even if an additional structure is coupled to the main one. To make this property valid it is necessary to consider a sufficient number of coupled co-ordinates. Although it might be argued that a reduced number of coupling co-ordinates would hamper the results since it would not include information about some modes, it has been shown in (Devriendt, 2004; Ribeiro et al., 2004; Devriendt & Fontul, 2005) that, as long as there is enough information regarding the modes included in the frequency range of interest the minimum number of coupling co-ordinates can be reduced without deterioration of the results.

2.8.2 Evaluation of unmeasured frequency response functions

Recent papers [(Maia et al., 2008; Urgueira et al., 2008)] have explored some invariance properties of the transmissibility, namely when modifications are made in terms of masses

\[
S_{KK}^T = T_{UK}^{(A)} S_{UK}^T
\]

(30)
and/or stiffnesses at the co-ordinates where the forces are applied, to be able to estimate the new FRFs in locations that become no longer accessible. For instance, if one calculates the transmissibility matrix at some stage between two sets of responses for a given set of applied forces and later on there are some modifications at the force co-ordinates due to some added masses, the FRFs will change but the transmissibility remains the same. This allows the estimation of the new FRFs. So, initially one has \[ \mathbf{X}_U(1) = \mathbf{H}_{UA}^{(1)} \mathbf{K}_{A}^{(1)} \mathbf{X}_K^{(1)} \] and later on one has \[ \mathbf{X}_U(2) = \mathbf{H}_{UA}^{(2)} \mathbf{K}_{KA}^{(2)} \mathbf{X}_K^{(2)}. \] As the transmissibility remains unchanged, one has

\[
T_{UK}^{(a)} = H_{UA}^{(1)} H_{KA}^{(1)} = H_{UA}^{(2)} H_{KA}^{(2)}
\]

and one can calculate, for instance, \( H_{UA}^{(2)} \), given by:

\[
H_{UA}^{(2)} = T_{UK}^{(a)} H_{KA}^{(2)}
\]

### 2.9 Direct transmissibility

From the definition given before one has,

\[
\mathbf{X}_U = \mathbf{H}_{UA} \mathbf{H}_{KA} \mathbf{X}_K = T_{UK}^{(a)} \mathbf{X}_K
\]

which, as explained, is a generalisation from the one degree of freedom system. However, in some cases it might be useful to divide two responses directly. In strict sense that is a transmissibility only if a single force is applied. Otherwise, one has to name it differently, like pseudo-transmissibility (e.g. (Sampaio et al., 1999)), scalar transmissibility (Devriendt et al., 2010) or direct transmissibility, which is the one we shall adopt.

Direct transmissibilities will depend on the force magnitudes (as well as location, of course). For example, dividing Eq. (33) by one of the amplitudes \( \mathbf{X}_K \), say \( X_s \), one has:

\[
\mathbf{X}_U/X_s = T_{UK}^{(a)} X_s/X_s \quad \text{or} \quad \tau_{UL} = T_{UK}^{(a)} \tau_{Ks}
\]

It is easier to understand the implications of both definitions through an example: let \( \mathbf{X}_U \), \( \mathbf{X}_K \), and \( \mathbf{F}_A \) be given respectively by

\[
\mathbf{X}_U = \begin{bmatrix} X_1 \\ X_2 \end{bmatrix}, \quad \mathbf{X}_K = \begin{bmatrix} X_3 \\ X_4 \end{bmatrix}, \quad \mathbf{F}_A = \begin{bmatrix} F_3 \\ F_4 \end{bmatrix}
\]

The relation between \( \mathbf{X}_U \) and \( \mathbf{X}_K \) would be:

\[
\begin{bmatrix} X_1 \\ X_2 \end{bmatrix} = \begin{bmatrix} T_{13} & T_{14} \\ T_{23} & T_{24} \end{bmatrix} \begin{bmatrix} X_3 \\ X_4 \end{bmatrix} \Rightarrow X_1 = T_{13} X_3 + T_{14} X_4 \\
X_2 = T_{23} X_3 + T_{24} X_4
\]

Dividing Eq. (36) by, say, \( X_3 \), it follows that

\[
X_1/X_3 = T_{13} + T_{14} X_4/X_3, \quad X_2/X_3 = T_{23} + T_{24} X_4/X_3
\]

or

\[
\tau_{13} = T_{13} + T_{14} \tau_{43} \quad \tau_{23} = T_{23} + T_{24} \tau_{43}
\]
One can also write:

\[
\begin{align*}
X_1 &= H_{15}F_5 + H_{16}F_6 \\
X_3 &= H_{35}F_5 + H_{36}F_6
\end{align*}
\]

\[\Rightarrow \quad r_{13} = \frac{X_1}{X_3} = \frac{H_{15}F_5 + H_{16}F_6}{H_{35}F_5 + H_{36}F_6} \tag{38}\]

From Eq. (38) it is clear that the direct transmissibility \( r_{13} \) depends on the magnitudes of \( F_5 \) and \( F_6 \), unless the relation \( F_5/F_6 \) remains constant. Only in the case where there is just a single force one has a coincidence between both types of transmissibility.

Both kinds of definitions can be useful. For instance, concerning now the direct transmissibilities, one can see that from Eq. (37) one can calculate \( r_{23} \) and \( r_{43} \) from \( r_{43} \), and one can eliminate \( r_{43} \) between both equations and establish a relationship between \( r_{13} \) and \( r_{23} \), therefore allowing the evaluation of one of them from the other. Moreover and similarly to what was mentioned in section 2.5, the direct transmissibilities allow the calculation of the other ones.

To illustrate the main differences between the curves of general and the direct transmissibilities, the MDOF system presented in Fig. 2 has been used. In Fig. 7 some direct transmissibility curves are presented.

Fig. 7. Some direct transmissibility curves from the system of Fig. 2

By comparing the results of the transmissibilities of Fig. 3 with the curves obtained with the direct transmissibilities, Fig. 7, one can see that both look like FRFs, though it may be noted that in the case of the transmissibilities all the maxima occur at the same frequencies; the same is not true with the direct transmissibilities, where each curve presents distinct maxima.

### 2.10 Other applications

Other works have presented the possibility of using the transmissibility concept for model updating (Steenackers et al., 2007) and to identify the dynamic properties of a structure (Devriendt & Guillaume, 2007; Devriendt, De Sitter, et al., 2009; Devriendt et al., 2010).
Other recent studies have applied the transmissibility to the problem of transfer path analysis in vibro-acoustics (Tcherniak & Schuhmacher, 2009) and for damage detection (Canales et al., 2009; Devriendt, Vanbrabant, et al., 2009; Urgueira et al., 2011).

3. Transmissibility of forces

3.1 In terms of frequency response functions

Another important topic may be the prediction of the dynamic forces transmitted to the ground when a machine is working. For a single degree of freedom, the solution is well known and the transmissibility is defined as the ratio between the transmitted load (the ground reaction) and the applied one, for harmonic excitation. For an MDOF system, one has to relate the known applied loads \( F_k \) to the unknown reactions \( F_u \), Fig. 8.

The displacements at the co-ordinates of one set (the set of the reactions) are constrained, so they must also be known (possibly zero). The inverse problem may also be of interest, i.e., to estimate the loads applied to a structure (wind, traffic, earthquakes, etc.) from the measured reaction loads. Once the load transmissibility matrix is established between the appropriate sets, the measurement of the reactions is expected to allow for the estimation of the external loads.

\[
\begin{bmatrix}
X_k \\
X_U \\
X_C
\end{bmatrix} =
\begin{bmatrix}
H_{kk} & H_{kU} \\
H_{UK} & H_{UU} \\
H_{CK} & H_{CU}
\end{bmatrix}
\begin{bmatrix}
F_k \\
F_U \\
F_C
\end{bmatrix}
\]  

(39)

Assuming the responses at the reactions co-ordinates as zero, i.e., \( X_U = 0 \), it follows that:

\[
F_U = -H_{UU}^{-1}H_{UK} F_k
\]  

(40)
Therefore, the force transmissibility is defined as:

$$T_{UK} = -H^{-1}_{UK}$$  

(41)

If the displacements at the co-ordinates of the reactions are not zero, or in the more general case when the two sets of loads are not the applied loads and the reactions, but any disjoint sets that encompass all the loads applied to the structure, it is easy to show (Maia et al., 2006) that:

$$F_U = T_{UK} F_K + H^{-1}_{UUL} X_U$$  

(42)

### 3.2 In terms of dynamic stiffness

Instead of Eq. (39) one has now:

$$\begin{bmatrix} F_K \\ F_U \end{bmatrix} = \begin{bmatrix} Z_{KK} & Z_{KU} & Z_{KC} \\ Z_{UK} & Z_{UU} & Z_{UC} \end{bmatrix} \begin{bmatrix} X_K \\ X_U \\ X_C \end{bmatrix}$$

(43)

Assuming fictitious loads $F_C$ at the remaining co-ordinates and rearranging, one obtains:

$$\begin{bmatrix} F_K \\ F_U \\ F_C \end{bmatrix} = \begin{bmatrix} Z_{KK} & Z_{KC} & Z_{KU} \\ Z_{CK} & Z_{CC} & Z_{CU} \\ Z_{UK} & Z_{UC} & Z_{UU} \end{bmatrix} \begin{bmatrix} X_K \\ X_C \\ X_U \end{bmatrix}$$

(44)

Defining $X_E = (X_K X_C)^T$ and $F_E = (F_K F_C)^T$ and assuming, as before, that at the reaction co-ordinates there is no motion ($X_{ij} = 0$), one can write:

$$F_E = Z_{EE} X_E$$

$$F_U = Z_{UE} X_E$$

(45)

Eliminating $X_E$ between eqs.(45), it follows that

$$F_U = Z_{UE} Z_{EE}^{-1} F_E$$

(46)

and the force transmissibility becomes now:

$$T_{UE} = Z_{UE} Z_{EE}^{-1}$$

(47)

Note that because $F_C = 0$, $F_E = (F_K \theta)^T$, and thus only the columns of $T_{UE}$ corresponding to $F_K$ are relevant to the transmissibility between the two sets of loads, the sub-matrix $T_{UK}$. One should also note that, in contrast with the SDOF system, the transmissibility of forces is different from the transmissibility of displacements.

Simply to illustrate the application of the concept, a numerical example is presented. The model is shown in Fig. 9, similar to the one of Fig. 3, where the displacements at co-ordinates 1 and 2 are now zero, i.e., $X_1 = X_2 = 0$. External forces are applied at co-ordinates 5 and 6 and the reactions happen at co-ordinates 1 and 2.
The force transmissibility between the two sets of loads – forces at 5 and 6 being known (set K) and forces at 1 and 2 being unknown (set U) – was computed using both described methods.

\[ \mathbf{F}_K = \begin{bmatrix} F_5 \\ F_6 \end{bmatrix}, \quad \mathbf{F}_U = \begin{bmatrix} F_1 \\ F_2 \end{bmatrix}, \quad \mathbf{F}_E = \begin{bmatrix} F_3 \\ 0 \\ 0 \end{bmatrix}, \quad \mathbf{X}_K = \begin{bmatrix} X_5 \\ X_6 \end{bmatrix}, \quad \mathbf{X}_U = \begin{bmatrix} X_1 \\ X_2 \end{bmatrix}, \quad \mathbf{X}_E = \begin{bmatrix} X_3 \\ X_4 \end{bmatrix} \]  

(48)

Equation (41) becomes:

\[ \mathbf{T}_{U|K} = \begin{bmatrix} H_{11} & H_{12} \\ H_{21} & H_{22} \end{bmatrix}^{-1} \begin{bmatrix} T_{H_{11}} & T_{H_{12}} \\ T_{H_{21}} & T_{H_{22}} \end{bmatrix} \]  

(49)

where the subscript H means that the transmissibility has been computed using FRFs.

Equation (47) becomes:

\[ \mathbf{T}_{U|E} = \begin{bmatrix} Z_{15} & Z_{16} & Z_{13} & Z_{14} \\ Z_{25} & Z_{26} & Z_{23} & Z_{24} \\ Z_{35} & Z_{36} & Z_{33} & Z_{34} \\ Z_{45} & Z_{46} & Z_{43} & Z_{44} \end{bmatrix}^{-1} \begin{bmatrix} T_{Z_{15}} & T_{Z_{16}} & T_{Z_{13}} & T_{Z_{14}} \\ T_{Z_{25}} & T_{Z_{26}} & T_{Z_{23}} & T_{Z_{24}} \\ T_{Z_{35}} & T_{Z_{36}} & T_{Z_{33}} & T_{Z_{34}} \\ T_{Z_{45}} & T_{Z_{46}} & T_{Z_{43}} & T_{Z_{44}} \end{bmatrix} \]  

(50)

from which
where the subscript $Z$ means that the transmissibility has been computed using dynamic stiffness matrices. The results obtained by using equations (49) and (51) superimpose perfectly, as expected. Two of the four transmissibilities are presented in Fig. 10 to illustrate this fact.

\[
T_{UK} = \begin{bmatrix}
T_{Z11} & T_{Z22} \\
T_{Z21} & T_{Z22}
\end{bmatrix}
\]  
(51)

Fig. 10. Comparison between corresponding force transmissibility terms computed from FRFs ($T_{H11}$ and $T_{H22}$) and dynamic stiffness matrices ($T_{Z11}$ and $T_{Z22}$).

It may be noted from Fig. 10 that the maxima of the force transmissibility curves also occur all at the same frequencies.

4. Conclusions

The transmissibility concept for multiple degree-of-freedom systems has been developed and applied for the last ten years and the interest in this matter is continuously growing. In this paper a general overview has been given, concerning the main achievements so far and it has been shown that the various ways in which transmissibility can be defined and applied opens various possibilities for research in different domains, like system identification, structural modification, coupling analysis, damage detection, model updating, vibro-acoustic applications, isolation and vibration attenuation.

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