Nanoscale torsional optomechanics

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Optomechanical transduction is demonstrated for nanoscale torsional resonators evanescently coupled to optical microdisk whispering gallery mode resonators. The on-chip, integrated devices are measured using a fully fiber-based system, including a tapered and dimpled optical fiber probe. With a thermomechanically calibrated optomechanical noise floor down to 7 fm/√Hz, these devices open the door for a wide range of physical measurements involving extremely small torques, as little as 4 x 10^{-20} N·m. © 2013 American Institute of Physics.

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The simplicity of the angular form of Hooke’s law, \( \tau = \kappa \theta \), belies its outstanding versatility and usefulness. Mechanical devices with small torsion constants (\( \kappa \)) have been designed to respond to torques associated with, and therefore sensitively measure gravity, charge, magnetism, the Casimir Force, shear modulus, superfluids, the de Hass-van Alphen effect, superconductors, persistent currents, the angular momentum of light, and more. In most cases such measurements can be improved by scaling down the size of the torsional devices to reduce the torsion constant. Often this scale-down presents opportunities for new physics associated with these smaller length-scales, such as mesoscopic phenomena. Hence there is a natural progression towards construction of nanoscale torsional resonators. But nanomechanical resonators suffer from their own set of difficulties, namely, in sensitive transduction of their mechanical displacements using techniques that do not scale down well. In this manuscript, we present the first nanoscale torsional resonators to take advantage of optomechanical detection, which has broken through this limitation.

We demonstrate nanoscale torsional devices with a displacement noise floor down to 7 fm/√Hz, corresponding to an angular displacement of 4 rad/√Hz and a torque sensitivity of 4 x 10^{-20} N·m/√Hz. For comparison, this torque sensitivity is just barely surpassed by optical and magnetic tweezers (\( \sim 10^{-21} N·m \)) that are capable of single-molecule torque measurements. In a modulated magnetic field (\( H_{ac} \)) of 1 kA/m, a torque (\( \tau = M \times H_{ac} \)) of 4 x 10^{-20} N·m/√Hz would correspond to a magnetic moment (\( M \)) sensitivity of only 3 x 10^{-9} Hz/√Hz spins for a pure torsional mode. Straightforward modifications to the torsional resonator geometry within this scenario pave the way to orders of magnitude improvement in torque sensitivity over the results presented here.

The first step in optomechanical transduction is the formation of an optical cavity, whether it be a Fabry-Perot cavity, whispering gallery mode resonator, or photonic crystal cavity. Resonances occur in the optical cavity based on the geometry and effective index of refraction occupied by the optical modes. Introduction of material with higher index than vacuum, such as a nanomechanical resonator, into the evanescent field of the optical mode shifts the effective index of refraction and hence the frequency of the resonance. Monitoring the time dependence of the optical resonance is, therefore, a readout of the motion of the nanomechanical device. This can be extremely sensitive as the optical resonances’ quality factor (\( Q_{opt} \)) can be quite high, the optomechanical coupling can be designed to be large, and optical detection is free from many of the problems associated with high frequency electronic measurements.

FIG. 1. (a) The schematic of the experimental setup. (b) Optical image of the dimpled, tapered fiber coupling into the (10 μm) optical microdisk of a device with a 100 nm gap to the torsion paddles. (c) Top down SEM image of the same device as in (b) overlaid with a 2D simulation of the optical mode. (d) SEM image of the same device tilted to 84°.
The layout of our optomechanical devices consists of an optical microdisk whispering gallery mode resonator side-coupled to a torsional mechanical resonator (Fig. 1(c)). The torsional resonator has a torsion rod of length \( \frac{24}{5} \) \( l \) and width \( \frac{24}{200} \) \( nm \), and a pair of paddles that are \( \frac{1.5}{l} \) \( mn \) each side. The left paddle follows the radius of curvature of the microdisk so as to maximize coupling to the evanescent field. Nine devices are compared for which two parameters, the optical microdisk size and the gap between the optical resonator and the mechanical resonator, are varied. This allows experimental determination of the conditions that result in the highest optomechanical coupling and therefore lowest noise floor.

The devices are fabricated from silicon-on-insulator, with a single-crystal silicon device layer of 215 nm—measured by SEM. The device structure is patterned into the silicon by high resolution optical lithography and subsequent etching via a commercial foundry. The wafer is then diced into individual chips, which are released using a buffered oxide etch of the 2 \( \mu m \) silicon dioxide layer. Upon subsequent drying, an in-plane distortion of the torsional resonator towards the optical microdisk is introduced, apparent in the SEM image of Fig. 1(c).

Additionally, tilted SEM imaging reveals that the side-walls of the both the optical and mechanical resonators are not vertical but instead angled by approximately 10° (Fig. 1(d)). This has important consequences. For symmetric and aligned evanescent field and mechanical resonator the optomechanical coupling is second order in the displacement out of plane. Asymmetry in the mechanical resonator\(^{35}\) and the evanescent field\(^{34}\) independently introduce first order coupling and account for the effectiveness of our optomechanical transduction of the torsional and out-of-plane mechanical modes.

A schematic of our fiber-based setup to measure the devices is shown in Fig. 1(a). Light from a tunable diode laser (New Focus TLB-6330) is fiber-coupled into a custom-built vacuum chamber (at \( \frac{10^{-6}}{6} \) Torr) after passing through a variable optical attenuator (VOA) and fiber polarization controller (FPC). Inside the chamber a dimpled, tapered optical fiber\(^{22,36}\) is held fixed on a goniometer and accurately aligned with the optomechanical torsional devices, which are mounted on three axes of vacuum-compatible nanopositioning stages. The chamber has a top viewport to allow observation of the fiber while positioning the dimple to either hover above, or, to reduce mechanical instability, touch the optical whispering gallery mode resonator (Fig. 1(b)). The transmission through the optical fiber is detected using a photodiode (PD: New Focus 1811) that splits the alternating (ac) and direct (dc) signals. The dc transmission is used to monitor the amount of light coupling into the optical microdisk, whereas the ac signal, which encodes the motion of the torsional resonator, is sent to a high frequency lock-in amplifier (Zurich Instruments HF2LI). The optical power is kept low (\( \leq 20 \mu W \)) to reduce optical nonlinearities.

As the wavelength of the tunable diode laser is swept, optical resonances appear as dips in the dc transmission (Fig. 2(a)). Simultaneously, the demodulated ac signal from the lock-in, with bandwidth of 100 Hz, is monitored to determine those optical resonances with the largest optomechanical coupling, which is dictated by the optical mode shape, the coupling between the fiber and the microdisk, and the \( Q_{opt} \) of the microdisk. This lock-in technique also allows us to directly verify the position within the optical resonance with

![FIG. 2. (a) Coarse wavelength scan of the tunable diode laser showing the optical resonances (orange) of the device in Fig. 1, with simultaneous peak thermo-mechanical torsion amplitude (black) recorded using a high frequency lock-in amplifier. (b) Same as in (a) but over a shorter wavelength range with finer resolution. In addition we show in blue the absolute value of the scaled and smoothed numerical derivative of the optical resonance. In both (a) and (b) the mechanical data has been smoothed— as the entire wavelength sweep is performed quickly (~60 s). (c) Mechanical resonances detected optomechanically, with their corresponding mode shapes as determined using FEM modeling (enhanced online) [URL: http://dx.doi.org/10.1063/1.4789442.1] [URL: http://dx.doi.org/10.1063/1.4789442.2].](image-url)
the strongest optomechanical signal. As can be seen in Fig. 2(b), this corresponds to the largest slope of the optical resonance (which confirms the basis of the optomechanical transduction scenario described above). A direct comparison of the peak mechanical signal with the absolute value of the numerical derivative of the optical resonance (arbitrarily scaled) shows very good qualitative agreement.

Once the wavelength of maximum optomechanical sensitivity is determined, the mechanical resonances can be further explored using the lock-in as a spectrum analyzer. A typical frequency spectrum is shown in Fig. 2(c) for the device with the 10 μm diameter microdisk and 100 nm gap. Two peaks are seen, which can be compared to finite element method (FEM) modeling of the mechanical structure using as-fabricated dimensions extracted from SEM images (Figs. 3(a)–3(c)). Eigenfrequencies are simulated for each of the three mechanical device paddle shapes (Figs. 3(d)–3(f)) which are dictated by the diameter of the optical microdisk. We have also explored how the frequencies depend on the mechanical device geometry, for example, by an overall scaling factor to the lateral dimensions or by adjusting the thickness of the silicon layer, but we find best agreement with the measured modes using the dimensions measured by SEM. The results are shown in Fig. 3(g) as a function of the change in moment of inertia of the torsional resonator, ΔI, calculated within the FEM model for the three paddle shapes. The devices with a 20 μm diameter optical microdisk, as in Figs. 3(c) and 3(f), have the smallest moment of inertia and are set as zero ΔI. In FEM simulations the lower frequency mode always corresponds to the torsional mode, and the higher frequency always corresponds to the out-of-plane mode (Fig. 3(g)). Yet both of these modes are substantially hybridized, in that the lower frequency torsion mode has some out-of-plane character and the higher frequency out-of-plane mode has significant twist, as can be seen in the simulated mode shapes in the inset to Fig. 2(c). To large part, this is due to the device undercut, which makes the effective length of the out-of-plane mode longer and lowers its frequency until it is nearly degenerate with the torsional mode. The total out-of-plane displacement in the lower frequency mode is small, and therefore we neglect this in our calibration of the torque sensitivity. As the paddle shape changes, altering the moment of inertia, the frequencies of the simulated lower frequency torsion modes are noticeably shifted. In contrast, the frequencies of the simulated higher frequency out-of-plane modes show little dependence on ΔI. This validates the identification of the measured lower frequency mode as the predominantly torsional mode, since it shows a larger dependence on ΔI than the higher frequency mode. Further confirmation of correct mode identification comes from the fact that the higher frequency out-of-plane mode is more efficiently actuated with an external piezoelectric buzzer (that vibrates the chip out-of-plane) than the lower frequency torsional mode (not shown) — consistent with previous observations. Finally, we note that we observe a resonance at ~12 MHz, which we identify as an in-plane resonance but do not concern ourselves further with this mode here.

For all of our devices thermomechanical motion of the mechanical modes is easily resolved and external actuation is unnecessary. Therefore calibration of the absolute displacement of the paddles is possible, enabling determination of the absolute noise floor—an important metric of transduction efficiency. To do so a known torque is applied to the FEM simulated device, resulting in a calculated displacement and hence the torsion constant of the mode. Once this torsion constant is known, calibration is straightforward. Fitting to the power spectral density gives the quality factor of the mechanical mode, $Q_{\text{mech}}$, the resonance frequency, the conversion from voltage to displacement, and the noise floor. A calibrated power spectral density, $S_\beta (S_f)$, is shown in Fig. 4(a) for our best device.

Comparing the calibrated displacement noise floor across the nine different devices is useful for understanding how to optimize torsional optomechanical coupling. For example, shown in Fig. 4(b), the best sensitivity generally occurs for 10 μm diameter optical microdisks. Because of competing effects, this is not obvious a priori. The optomechanical coupling is expected to increase as the intrinsic (unloaded) $Q_{\text{opt}}$ increases, which suggests that the largest microdisks should be best as they are least perturbed by surface effects. The loaded (nanomechanical resonator present) $Q_{\text{opt}}$ of the resonances used for optomechanical detection are approximately $5 \times 10^3$, $1 \times 10^4$, and $5 \times 10^4$ for the 5, 10, and 20 μm microdisks, respectively. On the other hand, small microdisks have smaller optical mode

![FIG. 3. SEM images of the paddles of the fabricated torsional devices with (a) 5 μm, (b) 10 μm, and (c) 20 μm optical microdisks. (d)–(f) Top-down geometries used to model the eigenfrequencies of the respective devices. (g) Predicted frequencies calculated through FEM modeling (dashed lines), and linear fits to the measured frequencies (solid lines), of devices with varying sizes of optical microdisks plotted against the change in moment of inertia calculated from the modeled geometries (d)–(f).]
volumes and therefore larger per photon optical energy densities, as well as larger overlaps between the optical field and paddle, both of which will result in increased optomechanical coupling. The balance between these competing effects is revealed in Fig. 4(b). However, the trend with gap between the optical microdisk and torsional resonator (Fig. 4(c)) is straightforward. As the gap is decreased there is an increase in the optomechanical transduction efficiency and decrease in the displacement noise floor, due to increasing the overlap between the mechanical resonator and the evanescent field.

Our most sensitive device, with a 10 μm optical microdisk diameter and a gap to the torsion paddle of 100 nm, has a displacement noise floor of 7 fm/√Hz. This surpasses previous experiments on macroscopic silicon torsional resonators \(^{40,41}\) and corresponds to an angular displacement noise floor 4 nrad/√Hz. In a reasonable ac magnetic field strength of 1 kA/m (root mean squared), for torque actuation, this device would be sensitive to \(3 \times 10^{14} \mu g/\sqrt{Hz}\) two orders of magnitude better than recent experiments with a nanoscale torsional resonator \(^{5}\) that had an approximately twenty times smaller torsion constant (\(5 \times 10^{-13} N\cdot m/\text{rad}\)) than our current device (\(1 \times 10^{-10} N\cdot m/\text{rad}\)). Therefore by simply altering our mechanical device geometry to match this torsion constant (by making the torsion rod thinner and longer for example) the \(10^6 \mu g\) level could be reached. Even further improvements to the torsional device geometry are well within the limits of current fabrication technology but have not been actively pursued up to this point, as previous measurements were detection limited and not device limited. With the introduction of optomechanical transduction this system is now device limited.

Finally, while the \(Q_{\text{mech}}\) of the devices in a vacuum of \(\approx 10^{-6}\) Torr average \(\sim 4000\), at ambient pressure this drops to \(\sim 150\). Nonetheless, thermomechanical motion of the devices can be detected in air, as shown in the inset of Fig. 4(a).

Nano-optomechanical torsional resonators have been fabricated and measured. With a displacement noise floor down to 7 fm/√Hz, this opens up the door for more sensitive torque measurements than have previously been possible using nanoscale torsional resonators. In addition, straightforward optimization of the torsional resonator geometry will enable considerable improvement, based on softening of the torsion constant. Nano-optomechanical torsional devices such as these will be useful for a number of future studies, such as exploration of nanoscale magnetic materials, or for low temperature experiments exploring quantum effects.\(^{42}\)

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