Supplementary Information

Monolithic Composite “Pressure + Acceleration + Temperature + Infrared” Sensor Using a Versatile Single-Sided “SiN/Poly-Si/Al” Process-Module. *Sensors* 2013, 13, 1085-1101

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1. Theoretical Design and Finite Element Analysis on Absolute-Pressure Sensor

1.1. Structure Design

Figure S1(a) illustrates the pressure-sensing SiN diaphragm in the absolute-pressure sensor. The diaphragm length, width and thickness are symbolized as $2b$, $2a$ and $h$, respectively. Assuming $2b \gg 2a \gg h$, the deflection and stress along the $x$-direction are [1]:

$$w_{\text{SiN}}(x) = \frac{P}{2E_{\text{eff}}h^3}x^2(a-x)^2 = \frac{P}{2E_{\text{eff}}h^3}x^2(a-x)^2$$ \hspace{1cm} \text{S(1.1)}

$$\sigma_{\text{SiN}}(x) = \frac{P}{h^2}(3x^2-a^2)$$ \hspace{1cm} \text{S(1.2)}

$$\sigma_{\text{SiN}}(x) = \nu_{\text{SiN}} \frac{P}{h^2}(3x^2-a^2)$$ \hspace{1cm} \text{S(1.3)}

where $x$ varies in $(-a, a)$, $P$ is the applied pressure, $\nu_{\text{SiN}}$ is the Poisson’s ratio, $E_{\text{eff}}$ is the effective Young’s module:

$$E_{\text{eff}} = \frac{E_{\text{SiN}}}{1-\nu_{\text{SiN}}^2}$$ \hspace{1cm} \text{S(1.4)}

The stress distribution is shown in Figure S1(b).
Figure S1. (a) Schematic of the rectangular pressure-sensing diaphragm and piezoresistors of the absolute-pressure sensor. (b) Stress distribution in the diaphragm and the piezoresistors’ longitudinal-parts placement design.

To realize a Wheatstone-bridge piezoresistive sensing, four meander-shape poly-Si piezoresistors $R_1 \sim R_4$ are designed, with their longitudinal parts placed along the $x$-direction at the high-stress area for large sensitivity, as shown Figure S1(b). In detail, the length of tensile-stress area is $0.42a$. Thus the longitudinal parts of $R_1$ and $R_4$ are laid at the tensile-stress areas of $(-a, -0.58a)$ and $(0.58a, a)$. Note that the length of longitudinal part $l$ is bigger than $0.42a$, therefore a minor part is laid outside the $(-a, a)$ region. In contrast, both longitudinal parts of $R_2$ and $R_3$ are placed at the compressive-stress area of $(-l/2, l/2)$. Each piezoresistor has $n$ longitudinal segments, and $(1-n)$ turning parts in which the current flows transversally. The total resistance, longitudinal resistance and effective transverse resistance of one piezoresistor are symbolized as $R$, $R_L$ and $R_T$, respectively. Thus, $R = R_L + R_T$.

Since the strain of poly-Si strips are nearly the same as the strain of SiN, i.e.,

$$
\varepsilon_{si} = \frac{\sigma_{si}(x)}{E_{si}/(1-\nu_{si}^2)} = \varepsilon_{SiN} = \frac{\sigma_{SiN}(x)}{E_{SiN}/(1-\nu_{SiN}^2)}
$$

S(1.5)

We obtain

$$
\sigma_{si}(x) = \sigma_{SiN}(x) \frac{E_{si}/(1-\nu_{si}^2)}{E_{SiN}/(1-\nu_{SiN}^2)}
$$

S(1.6)

Thus the stress in the poly-Si piezoresistor will be
\[ \sigma_{\text{Si}}^x(x) = \frac{P}{R} \left( 3x^2 - a^2 \right) \frac{E_{\text{Si}}}{E_{\text{SiN}}} \left( 1 - \nu_{\text{Si}}^2 \right) \]  
S(1.7)

\[ \sigma_{\text{Si}}^y(x) = \nu_{\text{Si}}^x \frac{P}{R} \left( 3x^2 - a^2 \right) \frac{E_{\text{Si}}}{E_{\text{SiN}}} \left( 1 - \nu_{\text{Si}}^2 \right) \]  
S(1.8)

1.2. Performance Analysis

From the above discussion, we can calculate the pressure-induced resistance change of the piezoresistors. For \( R_1 \) and \( R_4 \),

\[
\Delta R_i = \Delta R_4 \left( \frac{0.42a}{l} \right) R_L \left[ \int_{-\frac{a}{2}}^{\frac{a}{2}} \sigma_{\text{Si}}^x(x) \, dx \right]^{-\nu_{\text{Si}}^x} \left[ \int_{-\frac{a}{2}}^{\frac{a}{2}} \sigma_{\text{Si}}^y(x) \, dx \right] + \nu_{\text{Si}}^x \left( \frac{n/2}{1-n} \right) R_L \left( \frac{W_{RT}}{a} \right)^2 \left( \frac{1.74W_{RT}}{a} + 0.0092 \right) R_L \left( \nu_{\text{SiN}} \pi_L + \pi_t \right)
\]

\[
\frac{p a^2}{h^2} \frac{E_{\text{Si}}}{\left( 1 - \nu_{\text{Si}}^2 \right)}
\]

For \( R_2 \) and \( R_3 \),

\[
\Delta R_i = \Delta R_3 \left( \frac{0.42a}{l} \right) R_L \left[ \int_{-\frac{a}{2}}^{\frac{a}{2}} \sigma_{\text{Si}}^y(x) \, dx \right]^{-\nu_{\text{Si}}^x} \left[ \int_{-\frac{a}{2}}^{\frac{a}{2}} \sigma_{\text{Si}}^x(x) \, dx \right] \]

\[
+ R_L \left[ \int_{-\frac{a}{2}}^{\frac{a}{2}} T_{\text{Si}}(x) \, dx \right]^{-\nu_{\text{Si}}^x} \left[ \int_{-\frac{a}{2}}^{\frac{a}{2}} T_{\text{Si}}(x) \, dx \right] \]

\[
= \left[ 1 - \frac{1}{2} \right] R_L \left( \nu_{\text{SiN}} \pi_L + \pi_t \right) = \left[ 1 - \frac{1}{2} \right] R_L \left( \nu_{\text{SiN}} \pi_L + \pi_t \right)
\]

Thus the Wheatstone-bridge output voltage turns to be

\[
V_{\text{out}} = \frac{V_{\text{in}}}{2} \left[ \frac{R_L + \Delta R_4}{R_L + \Delta R_1 + \Delta R_4 + \Delta R_3} - \frac{R_L + \Delta R_3}{R_L + \Delta R_1 + \Delta R_3 + \Delta R_4} \right] = \frac{V_{\text{in}}}{2 \left( R_L + R_4 \right)} \left[ \alpha R_L \left( \nu_{\text{SiN}} \pi_L + \pi_t \right) - \nu_{\text{Si}} R_L \left( \nu_{\text{SiN}} \pi_L + \pi_t \right) \right]
\]

\[
\text{S(1.11)}
\]
where
\[
\alpha = 1 + 0.39a - \frac{t^2}{4a^2},
\]
\[
\beta = \left(\frac{n/2}{1-n}\right)\left(\frac{W_{a}^2}{a^4} - \frac{1.74W_{a}}{a} + 0.0092\right) - \left(1 - \frac{3W_{a}^2}{2a^4} - \frac{3t^2}{4a^2} - \frac{W_{a}}{a^2}\right)
\]

Then the sensitivity is calculated as
\[
S = \frac{V_{\text{out}}}{P} = \frac{1}{2} \frac{a^2}{h^2} \frac{E_{a}}{1 - V_{a}^2} V_{\text{in}} \left[\alpha R_L \left(\pi_1 + V_{a} \pi_3\right) - \beta R_L \left(V_{a} \pi_1 + \pi_7\right)\right] \frac{R_L + R_T}{R_L + R_T}
\]

Based on the design parameters in this paper, we obtain
\[
S = \frac{a^2}{h^2} \frac{E_{a}}{1 - V_{a}^2} V_{\text{in}} \left[0.83R_L \left(\pi_1 + V_{a} \pi_3\right) - 0.17R_L \left(V_{a} \pi_1 + \pi_7\right)\right] \frac{R_L + R_T}{R_L + R_T}
\]

The calculated sensitivity is 112 mV/MPa (\(V_{\text{in}} = 3.3\) V). Equation (1.14) gives a design guideline on the rectangular-diafragma based pressure sensor, in which the effects of various structure and material parameters on the sensor performance are revealed.

Due to the residual in-plane tensile stress of the deposited LS-SiN diaphragm (about 100 MPa), the stiffness of the diaphragm will be increased and therefore the real sensitivity will be reduced. Then the ANSYS numerical analysis is implemented with considering the effect of residual tensile stress.

Figure S2 illustrates the simulated stress distribution of \(\sigma_{\text{SiN}}(x)\) (with deflected shape), with 1 MPa pressure perpendicular to the diaphragm and an in-plane 100 MPa tensile stress. The \(\sigma_{\text{SiN}}(x)\) curve along \(x\)-direction is shown in Figure S3(a). In contrast, the simulated \(\sigma_{\text{SiN}}(x)\) distribution without considering the residual tensile stress is also shown in Figure S3(b).

**Figure S2.** Simulated stress distribution of pressure-sensing diaphragm under 1 MPa applied pressure.
**Figure S3.** Simulated stress distributions at the pressure-sensing diaphragm surface. (a) Residual tensile stress considered. (b) Residual tensile stress neglected.

![Image](a)

![Image](b)

From the simulated stress data in Figure S3, the resistance changes ($\Delta R_1$ and $\Delta R_2$) are calculated according to (1.9) and (1.10). Note that the stress data in Figure S3(a) is firstly subtracted by 100 MPa before the calculation. Then we obtain the simulated sensitivity of the pressure sensor. With the supplied voltage ($V_{in}$) of 3.3 V, the sensitivity of the device without residual tensile stress is 71 mV/MPa. By considering the residual stress, the simulated sensitivity decreases to 55 mV/MPa.

### 2. Theoretical Analysis and Finite Element Analysis on Accelerometer

#### 2.1. Structure Design

Figure S4(a) illustrates the structure design of the accelerometer. The length, width and thickness of the force-sensing beam are symbolized as $L$, $b$ and $h$, respectively. With considering the residual stress, the deflection and stress in the beam along the $x$-direction are [1]:

$$w_{\text{SiN}}(x) = \frac{Ma}{2E_{\text{SiN}}bh^2}x^2\left(1 + \frac{\sigma_0L^2}{E_{\text{SiN}}h^2}\right)$$  \hspace{1cm} S(2.1)

$$\sigma_{\text{SiN}}^x(x) = \frac{3Ma}{2bh^7}x^2\left(1 + \frac{\sigma_0L^2}{E_{\text{SiN}}h^2}\right)$$  \hspace{1cm} S(2.2)

where $x$ varies in ($0, L$), $a$ is the applied acceleration, $\nu_{\text{SiN}}$ is the Poisson’s ratio, $E_{\text{SiN}}$ is the Young’s module of SiN, $\gamma_1 (=0.295)$ is a coefficient expressing the additional contribution of the axial force to the beam stiffness, $\sigma_0$ is the residual tensile stress in the LS-SiN beam. As the beam is free along the $y$-direction, $\sigma_{\text{SiN}}^y(x)$ is ignored. The stress distribution is shown in Figure S4(b).
To realize a Wheatstone-bridge piezoresistive sensing, four poly-Si piezoresistors $R_1 \sim R_4$ are designed, with their longitudinal parts placed along the $x$-direction at the high-stress area for large sensitivity, as shown Figure S4(b). In detail, the length of tensile-stress area is $0.5L$. Thus the longitudinal parts of $R_1$ and $R_4$ are laid at the tensile-stress areas of $(0, l)$. In contrast, both longitudinal parts of $R_2$ and $R_3$ are placed at the compressive-stress area of $(L-l, L)$. Each piezoresistor has two longitudinal segments, as well as one square turning part (size = $W_{RT}$) in which the current flows transversally. The total resistance, longitudinal resistance and transverse resistance of one piezoresistor are symbolized as $R$, $R_L$ and $R_T$, respectively. Thus, $R = R_L + R_T$.

With the similar analysis in Part 1, we obtain the strain in poly-Si strips as

$$\sigma_{Si}^x(x) = \frac{3Ma}{2bh^2} \left( \frac{1}{2} L - x \right) \frac{E_{Si}/(1-\nu_{si}^2)}{E_{Si}/(1-\nu_{si}^2)} \left( 1 + \gamma \frac{\sigma_x L^2}{E_{Si}h^2} \right)^{-1} \tag{S2.3}$$

### 2.2. Performance Analysis

From the above discussion, we can calculate the pressure-induced resistance change of the piezoresistors. For $R_1$ and $R_4$, the performance analysis is as follows.
\[ \Delta R_i = \Delta R_l = R_l \pi_l \frac{\int_0^l \sigma_{Si}^i(x) \, dx}{l} + R_t \pi_t \frac{\int_0^{l-W_{RT}} \sigma_{Si}^i(x) \, dx}{W_{RT}} \]

\[ = \left[ \left( \frac{L-l}{2} \right) R_l \pi_l + \left( \frac{L-2l-W_{RT}}{2} \right) R_t \pi_t \right] \]

\[ = \frac{3Ma}{2bh^2} \frac{E_{Si}/(1-v_{Si}^2)}{E_{Si}/(1-v_{Si}^2) \left( 1 + \chi \frac{\sigma_0 L^2}{E_{Si} h^2} \right)} \]

For \( R_2 \) and \( R_3 \),

\[ \Delta R_2 = \Delta R_3 = R_l \pi_l \frac{\int_0^{L-l} \sigma_{Si}^i(x) \, dx}{l} \]

\[ = \left[ \left( \frac{L-l}{2} \right) R_l \pi_l \cdot \frac{3Ma}{2bh^2} \frac{E_{Si}/(1-v_{Si}^2)}{E_{Si}/(1-v_{Si}^2) \left( 1 + \chi \frac{\sigma_0 L^2}{E_{Si} h^2} \right)} \right] \]

Thus the sensitivity is calculated as

\[ S = \frac{V_{out}}{a} = \frac{V_{in} \Delta R_i - \Delta R_l}{2R} = V_{in} \left[ \frac{(L-l) R_l \pi_l + (0.5L - l - 0.5W_{RT}) R_t \pi_t}{R_l + R_t} \right] \frac{3M}{4bh^2} \frac{E_{Si}/(1-v_{Si}^2)}{E_{Si}/(1-v_{Si}^2) \left( 1 + \chi \frac{\sigma_0 L^2}{E_{Si} h^2} \right)} \]

Based on the design parameters in this paper, we obtain

\[ S = \frac{V_{out}}{a} = \frac{ML}{bh^2} \frac{E_{Si}/(1-v_{Si}^2)}{E_{Si}/(1-v_{Si}^2)} V_{in} \left[ \frac{0.6R_l \pi_l + 0.2R_t \pi_t}{R_l + R_t} \right] \left( 1 + \chi \frac{\sigma_0 L^2}{E_{Si} h^2} \right) \]

For the residual stress, ANSYS numerical analysis is conducted, as show in Figure S5. It shows that the rotating structure design releases the deposition stress significantly, and the residual stress \( \sigma_0 \) at piezoresistor location is lowered to only about 5 MPa. Then the calculated sensitivity of accelerometer is 69 \( \mu V/g \).

**Figure S5.** Simulated residual stress distribution in the beam of accelerometer.
3. Finite Element Analysis on Infrared Detector

Numerical simulation by using ANSYS is conducted to simulate the temperature distribution over the structure of infrared detector under radiation. With the applied radiation-density of 6.5 mW/cm², the temperature distribution at one-quarter of the structure are shown in Figures S6 and S7. Then we obtain that the temperature difference $\Delta T_{HC}$ between the hot and cold ends is about 0.12 °C.

**Figure S6.** Simulated temperature distribution over one-quarter of the infrared detector.

![Figure S6](image)

**Figure S7.** Simulated temperature distribution from the absorbing membrane center to the cold-end.

![Figure S7](image)

Reference

1. Bao, M. *Analysis and Design Principles of MEMS Devices*; Elsevier: Amsterdam, The Netherlands, 2005.

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