Structure Optimization of a High-Temperature Oxygen-Membrane Module Using Finite Element Analysis

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Abstract: The oxygen transport membrane (OTM) is a high-density ion-conducting ceramic membrane that selectively transfers oxygen ions and electrons through the pressure differential across its layers. It can operate at more than 800 °C and serves as an economical method for gas separation. However, it is difficult to predict the material properties of the OTM through experiments or analyses because its structure contains pores and depends on the characteristics of the ceramic composite. In addition, the transmittance of porous ceramic materials fluctuates strongly owing to their irregular structure and arbitrary shape, making it difficult to design such materials using conventional methods. This study analyzes the structural weakness of an OTM using CAE software (ANSYS Inc., Pittsburgh, PA, USA). To enhance the structural strength, a structurally optimized design of the OTM was proposed by identifying the relevant geometric parameters.

Keywords: ceramic composite materials; finite element method; material properties; membrane

1. Introduction

Oxygen Transport Membrane

An oxygen transport membrane (OTM) is an ion-conducting membrane that selectively transports oxygen via the pressure difference between its two sides. The OTM is an economical component used in gas separation methods [1–8]. Ceramic membranes with mixed conductivity transport both ions and electrons. These membranes can separate oxygen without requiring external energy or applied voltages, as the electrons travels in or against the direction of the oxygen ions [9–16].

Oxygen ions migrate through the porous lattice when the lattice vibrates at more than 800 °C. OTMs exhibit feeble lattice vibrations at low temperatures. Pure OTMs operate at high temperatures and pressures; therefore, they necessitate mechanical, thermal, and structural stability. When a ceramic membrane is exposed to high temperatures and pressures, it disintegrates. To protect such membranes and maximize their efficiency, researchers have developed various types of laminated forms and module systems.

In laminated forms, each layer must have a specific functional characteristic. For example, the membrane layer must have structural stability to endure external pressures and osmosis; the porous layer must have a microstructure to transport oxygen ions; the catalyst, chemical stability; and the support layer must have mechanical, thermal, and chemical stability. Additionally, if the structure shown in Figure 1 acts as the membrane assembly, then it would require sealing methods to allay external particles, and it must possess mechanical durability [17].
Module systems including the plate-, hollow [18], and honeycomb-type membrane modules (Figure 2) have been developed [19]. These systems must have a high effective cross-sectional area, relative to the membrane-response area, for ease of manufacturing and to ensure complete sealing.

In the late 1980s, the US, Japan, Europe, and other countries acknowledged the importance of high-oxygen separation techniques [20] and invested substantially in related studies. The Korea Institute of Energy Research has been conducting research on ion-conductive membrane processes. OTM conductors can be fabricated by selectively separating high concentrations of oxygen, regardless of the size. Therefore, these conductors are gradually being expanded to steelmaking, automotive, and manufacturing industries [21].

The research on composite ceramic or porous ceramic OTMs and OTM modules can be classified into six subjects as shown in Figure 3:

1. Membrane material, which covers the characteristics of the membrane along with its structure, composition, and production process [1,17–50].
2. Composite ceramic characteristics, structures, and mechanical behaviors [4,32,34,51–54].
3. OTM systems, which include the lifetime and fatigue failure predictions at high temperatures and pressures [55,56].
4. Pure material experiments [5,57], which theoretically and experimentally investigate the material properties at high temperatures and pressures, structure of mixed materials, and mixed compositions [51,58].

5. Computer simulation, which is used to analyze the weak points in the membrane, predict the stresses in the OTM module system, and optimize or verify production [53,57,59–76]. Although a few researchers have studied hollow and tubular modules, limited research has been devoted toward plate-type modules.

6. The latest research trends and applications of composite ceramics and OTM systems [77–84].

Figure 3. Research on OTMs and modules (the numbers in the figure represent the approximate number of available articles related to the present research).

2. Structural Analysis of OTM Modules

2.1. Model Description

OTMs should possess mechanical, thermal, and structural stability because the OTM modules must selectively permeate oxygen via ion conduction, which is only possible at high temperatures (more than 800 °C) and pressures (more than 10 bar). The membrane is only 10–100 µm thick, and therefore it is vulnerable to external conditions. However, enhancing the membrane’s structural stability by simply increasing the membrane thickness would make the penetration ratio impractically low. Therefore, researchers are experimenting with various module shapes that can compensate for the thickness, to ensure that the membrane is protected without losing its features. The addition of porous ceramics that permeate oxygen around the membrane to stabilize the structure opens the possibility of a variety of module systems. As mentioned earlier, the OTM must be impermeable, thermally and structurally stable, and easy to produce, while possessing a high cross-sectional area of effective penetration relative to the response volume.

The plate-type module used in previous studies [45,47] was modified and used as the basis for the OTM module shape in this study (Figure 4).
The area and diffusion rate of the ceramic membrane cannot be disclosed as it is a proprietary technology of the Korea Institute of Energy Research. However, the thickness of the layer and the stacking order have been disclosed.

In collaboration with the institution, the membrane was designed with a high penetration ratio and structural stability. Table 1 provides information on the material composition and the thickness of each layer. Figures 5 and 6 illustrate the components of the OTM module. As seen in the figure, each unit is a vertically symmetrical layered structure.

**Table 1.** Material and thickness of the OTM module components.

| ID | Name                  | Material       | Thickness (μm) |
|----|-----------------------|----------------|----------------|
| 1  | Membrane              | G8L2 (Dense)   | 120            |
| 2  | Porous support layer  | G5L5 (Porous)  | 30             |
| 3  | Gas-channel layer     | G8L2 (Dense)   | 200            |

Figure 4. Fully sealed OTM module.
The structure of the OTM module was analyzed using commercial finite element analysis software and modeled on Catia V5.0 (ANSYS Workbench Design Modeler, ANSYS Inc., Pittsburgh, PA, USA) after defining the composition of each layer as a set, based on the specifications mentioned earlier. This module was prepared as a set concept through the sintering process after each layer was prepared, and all the layers were laminated.

Each layer in ANSYS Workbench consisted of a solid, and the above models were assembled to form parts. The symmetry condition was used to reduce the computational time, which allowed expressing only half of the OTM module set in ANSYS. The material properties of each layer were obtained using the data from the material experiments, as detailed in Table 1 of [85].

The boundary conditions of the model were as follows:

- An internal pressure of 10 kPa was applied because the 1 MPa gas-channel layer of the external pressure can extract oxygen via the pressure difference between the internal and external surfaces.
- Each layer was laminated to constitute a unit. Thus, symmetric conditions were applied to the top face of the membrane layer and the bottom face of the gas-channel layer via the entire structure of all the laminated units.
- In view of the half model, the surroundings of the side with the hole were assigned the left-symmetry right-symmetry condition.

Part of the high-stress concentration appears between the porous and gas-channel layers, which represents the pressure differential across the OTM module.
It was essential to choose the element formation because the thickness of the generated element is in the micrometer range, owing to the thin membrane of the OTM and the difference in the stress values in accordance with the formation of the membrane and the size of the lattice for the element generating.

Element generation or meshing is an important step in the finite element method. Omitting this step or using the auto-mesh function can produce critical errors in the analysis. To enhance the accuracy, the type, number, and distribution of finite elements must be meticulously selected according to the analysis model and type. The area and space of the detailed structure and the region of stress concentration and gradient fluctuations require detailed grid separation and additional mesh convergence tests after the analysis.

The number of grid layers according to each layer and the element size of the total OTM model were used as the parameters for the mesh convergence test (Figures 7 and 8).

For the grids, using the eight-node hexahedral element would have led to a local stress concentration and the accuracy of the stress values with the form of the gas-channel layer. As the number of stacked elements per OTM module layer increases, the computational time and the number of elements increase. To precisely express the stress gradient section due to the material difference of each layer and to increase the accuracy of the analysis of the thin-membrane OTM module, we chose the three-dimensional 20-node hexahedral element.

**Figure 7.** Mesh convergence test (number of mesh layers vs. maximum von Mises stress).
The OTM unit model had an element size of 100 μm. The mesh layer with five layers toward the thickness of each layer forms the grid model. There were approximately 500,000 elements and 2,000,000 nodes.

2.3. Result

Two vulnerable regions were identified on the OTM module: the neighborhood of the support structures between the porous layer and the gas-channel layer with large membrane stress and pressure differences; and a processing hole. Figures 9–11 illustrate the results of the structural analysis for each layer. The membrane layer had a stress distribution less than the breaking strength (58.5 MPa). The maximum stress (35 MPa) around the processing hole was also less than the breaking strength. In the porous layer, the stress (95 MPa) was significantly higher than the breaking strength (44.6 MPa). This region of high stress was in contact with the gas-channel support layer with a significant pressure difference.
Table 2. Structural analysis results for each layer of the half model.

| Layer               | Maximum von Mises Stress [MPa] |
|---------------------|---------------------------------|
| Membrane layer      | 35                              |
| Porous layer        | 95                              |
| Gas-channel layer   | 46                              |

Figure 9. Structural analysis for membrane layer of the half model.

The detailed structural analysis revealed that the membrane layer of the section model had a lower maximum stress than the half model. The stress contour was reconstructed, considering that the machining hole was removed. Subsequently, the region except the machining hole was confirmed as having the same stress distribution as the half model. The same stresses as in the half model were observed in the porous layer sections. The region of high stress was adjacent to a supporting layer of the gas-channel layer with the pressure difference, which was the same as in the half model. Additionally, the stress of the gas-channel layer was higher in the peripheral region supporting the porous layer in the section model, as compared to that in the half model.

Figure 10. Structural analysis for porous layer of the half model.

To summarize, the design of the applied area must be modified to account for the high stress observed in the area with the pressure gradient, i.e., the point where the porous layer and gas-channel layer come in contact, and to prevent failure. This stress exceeded the breaking strength. However, the membrane layer, which had a stress lower than the breaking strength (58.5 MPa), did not require a design change because the structure would not fail and was safer than the other layers.

The structural analyses of the half model and the section model revealed the fragile and high-stress regions in the membrane, respectively. The boundary between the porous and the gas-channel layers, where the pressure gradient was significant, was the fragile region. A more detailed observation of the section model confirmed that the weak point was the area connecting the gas-channel support and the porous layer.
Therefore, the peripheral region of the porous layer must be designed as a support structure to account for the high stress around the region adjacent to the high-stress region on the porous layer of the gas-channel layer.

A detailed model must be constructed instead of a half model to perform a complete structural analysis and observe the weak points in the structure. Therefore, we defined a section model (Figure 12) by slicing a few sections from the half model.

The section model was assigned the same boundary conditions and grid compositions as the half model. In the analysis, a symmetric boundary condition was applied to the top and bottom of the model because a few sections were extracted as shown in Figure 12. The results of the structural analysis of each layer in the section model are presented in Figures 13–15 and Tables 2–4.

Table 2. Structural analysis results for each layer of the half model.

| Layer             | Maximum von Mises Stress [MPa] |
|-------------------|--------------------------------|
| Membrane layer    | 35                             |
| Porous layer      | 95                             |
| Gas-channel layer | 46                             |
To compensate for the weak point, the stress must be reduced by modifying the geometric design of the honeycomb base support of the gas-channel layer and the width of the gas channel. A design supplement must be recommended through additional structural analyses by setting the parameters with these causes. The parametric and structural optimization studies are explained in Section 3.

According to previous research, the most optimal method of reducing stress for structural stability is to increase each layer’s thickness. However, as mentioned earlier, this approach would reduce the penetration ratio. Therefore, to retain the original penetration ratio of the OTM module, we omit thickness as a parameter in the parametric and optimization studies. Further details of this tradeoff are provided in the Limitations section of Section 3.

**Figure 12.** Definition and extraction method for section models.

**Figure 13.** Structural analysis for membrane layer of the section model.
Figure 14. Structural analysis results for porous layer of the section model.

Table 3. Structural analysis results for each layer of the section model.

| Layer             | Maximum von Mises Stress [MPa] |
|-------------------|--------------------------------|
| Membrane layer    | 28                             |
| Porous layer      | 95                             |
| Gas-channel layer | 58                             |

Table 4. Comparison of maximum stress values between half model and unit model.

| Layer             | Maximum von Mises Stress [Half Model] | Maximum von Mises Stress [Section Model] |
|-------------------|--------------------------------------|------------------------------------------|
| Membrane layer    | 35 MPa                               | 28 MPa                                   |
| Porous layer      | 95 MPa                               | 95 MPa                                   |
| Gas-channel layer | 46 MPa                               | 58 MPa                                   |
Figure 15. Structural analysis results for gas channel layer of the section model.

The detailed structural analysis revealed that the membrane layer of the section model had a lower maximum stress than the half model. The stress contour was reconstructed, considering that the machining hole was removed. Subsequently, the region except the machining hole was confirmed as having the same stress distribution as the half model. The same stresses as in the half model were observed in the porous layer sections. The region of high stress was adjacent to a supporting layer of the gas-channel layer with the pressure difference, which was the same as in the half model. Additionally, the stress of the gas-channel layer was higher in the peripheral region supporting the porous layer in the section model, as compared to that in the half model.

To summarize, the design of the applied area must be modified to account for the high stress observed in the area with the pressure gradient, i.e., the point where the porous layer and gas-channel layer come in contact, and to prevent failure. This stress exceeded the breaking strength. However, the membrane layer, which had a stress lower than the breaking strength (58.5 MPa), did not require a design change because the structure would not fail and was safer than the other layers.
The structural analyses of the half model and the section model revealed the fragile and high-stress regions in the membrane, respectively. The boundary between the porous and the gas-channel layers, where the pressure gradient was significant, was the fragile region.

A more detailed observation of the section model confirmed that the weak point was the area connecting the gas-channel support and the porous layer.

To compensate for the weak point, the stress must be reduced by modifying the geometric design of the honeycomb base support of the gas-channel layer and the width of the gas channel. A design supplement must be recommended through additional structural analyses by setting the parameters with these causes. The parametric and structural optimization studies are explained in Section 3.

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3. Structure Optimization of OTM Module

3.1. Model Description

The structural analysis results of the OTM module suggest using the shape of the gas-channel layer support (the length and apex round of the honeycomb structure) and the gas-flow channel as alternative parameters for the design supplement of the OTM module.

However, using the applied variables in the model and analyzing the structure of every geometric shape in each case is computationally taxing.

Therefore, we conducted the parametric study by reducing the computational time as a design supplement and using the ANSYS Workbench optimization program to suggest the optimal specifications of the OTM module.

For the structural optimization, the response surface of each parameter was established and the design variables and objective function were set. To expedite the process, a unit model was chosen by compressing the models. The length (L) and apex round of the honeycomb (R) and the width of the flow channel (W) were chosen as the input design variables. The equivalent stresses of the membrane, porous layers, and entire module were selected as the objective functions.

The response-surface method (or the response-surface analytical method) was used for the optimization. This method is used to optimize the level of a variable by helping to expect the result values of the unselective level of the total area of interest, and thus achieve the desired value.

This makes the changing estimation by a variable appear in a two- or three-dimensional space, which is a dot on a flat or curved surface. It must be performed to determine and optimize the maximum and minimum on the surface. The line or surface on which the optimal conditions pass is defined as the response surface. Theoretically, higher the number of parameter cases, greater is the analytical load in the response-surface method. However, the computational load is limited by the computer hardware. This will subsequently reduce the accuracy of the response curve surfaces or lines. Therefore, the response-surface method, which is an experimental method for the selection of parametric variables, was used for the analysis of the minimum.

3.2. Simulation Setup

We used the following variables in the ANSYS Workbench Design Modeler: Honeycomb structure length = L, Honeycomb vertex round = R, and Gas channel width = W.

The design variable was set as an input variable, which is expressed as a unit model in Figure 16. The material data from Table 1 of ref [85] were input as the material properties.
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We established the symmetric conditions as the boundary conditions in all directions of the unit model (Figure 17).

The maximum equivalent stress of the membrane layer, which governs the function of an OTM module; the porous layer designated as a vulnerable area based on the structural analysis; and the maximum equivalent stress of the entire module, including the region where a stress change was expected along with a shape change, according to the geometric design safety parameters, were selected as the objective functions. Additionally, induction variables (variables generated using the input variables or design variables and numerical combinations of the objective variables) were adopted. The effective area through which oxygen ions permeate is defined as the active area. The total area of the OTM module is defined as the total area. The ratio of the effective area to the total module area is called the effective area ratio. These are the induction variables. The conditions for increasing the cross-sectional area of effective permeability per reaction volume of the membrane module were selected by introducing the concept of the final objective-function variable being the equivalent stress/effective area ratio of the membrane layer (Figures 19 and 20).
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**Figure 18.** Symmetry conditions of unit model.

**Figure 19.** Boundary conditions of unit model.

**Figure 20.** Concept of Effective Area Ratio.
The structure was optimized based on these conditions. In general, the parametric study is performed based on a method chosen to determine the total number of parameter cases in the experimental design stage.

In this parametric study, we chose 100 cases by introducing an additional sampling technique because the central synthesis method used three design variables and only 15 cases among the selected experimental design methods.

Despite the increased computational time, the method offers several advantages. For example, it allows for the generation of a reaction surface with a more accurate slope as it has a greater number of parameter design points, and the relationship between the response variables becomes narrower. Moreover, it reduces both the errors in the solution of the optimized variables and the result of the actual structural analysis. The structure was optimized to analyze the impact of each parameter, create a response surface, and propose an optimal OTM module design that satisfies the objective function.

The final objectives are:
- Minimization of the maximum equivalent stress of the membrane layer/effective area ratio
- Minimization of the maximum equivalent stress of the membrane layer
- Maximization of the effective area ratio
- Minimization of the maximum equivalent stress of the porous layer
- Minimization of the maximum von Mises stress in the entire OTM module.

### 3.3. Result

This simulation result can be compared to the width of the flow path, which had the most significant effect on the stress in the OTM module. However, when the width was reduced to lessen the stress, it reduced the effective area of oxygen permeation. Therefore, these objective functions are not conducive to the optimization.

Figures 21–25 illustrate the influences of the parameters and the response surface curvatures for the 100 cases with the chosen experimental design method and the additional sampling technique. The minimum points of the honeycomb width and apex round were observed. However, as the width of the channel decreased, the stress also decreased, and vice versa. To set the width of the flow path as the optimal design variable, the final objective function was optimized by applying 1.5 times the weight to the maximization of the effective area ratio.

![Figure 21. Analysis of parameter influence on honeycomb width (L).](image-url)
Figure 21. Analysis of parameter influence on honeycomb width (L).

Figure 22. Analysis of parameter influence on the supporting vertex round value (R).

Figure 23. Analysis of the parameter influence on the width of the flow path (W).

Figure 24. Local sensitivity of the parameters (L, R, W).
Figure 23. Analysis of the parameter influence on the width of the flow path (W).

Figure 24. Local sensitivity of the parameters (L, R, W).

The structural analysis enabled the selection of the minimum values on the reaction surface curves per parameter. The membrane stress (21 MPa) was less than the allowable stress (58.5 MPa); however, the stress (78.3 MPa) at the porous support–gas layer interface was confirmed to be higher than the minimum allowable stress (32 MPa). Therefore, the optimization analysis was repeated by adding the fracture stress of each layer to the objective function, as follows:

- Minimization of the maximum equivalent stress of the membrane layer to less than 58.5 MPa.
- Minimization of the maximum equivalent stress of the porous layer to less than 32 MPa.
- Minimization of the maximum equivalent stress/effective area ratio of the membrane layer.
- Maximization of the effective area ratio.
- Minimization of the maximum von Mises stress in the entire OTM module.

Figures 26 and 27 illustrate the results of the second optimization analysis based on the reset objective function. The results show that the brittle stress of the porous support layer mentioned in Section 2 was 30 MPa, which was less than the breaking stress (32 MPa). The brittle stress of the membrane layer was 16.2 MPa, which was less than the minimum breaking stress of 58.5 MPa.

Based on these results, we proposed a design method to stabilize the structure of the plate-type OTM module. The porous layer was confirmed to be the weakest part of the OTM module, which agrees with the observations of a previous study [67]. To structurally stabilize the OTM module, the geometry of the gas-channel layer and the width of the gas channel were changed without increasing the thickness of each layer of the OTM module. This could reduce the stress of the porous layer. Based on the structural stability, this study proposed a design method to achieve an effective permeable cross-sectional area ratio of the OTM module.

Figure 24. Local sensitivity of the parameters (L, R, W).

Response surface curve of the parameters L, R.

Figure 25. Response surface curve of parameters L and R.

Figure 23 depicts the local sensitivity of each parameter. The design parameter with the greatest effect on the maximum stress of the OTM module is the width of the flow path. The effective permeable area is determined by the widths of the honeycomb and the
flow path. However, considering the membrane’s maximum stress/effective permeability area ratio as the final objective function of this optimization, L was identified as the most sensitive variable.

The structural analysis enabled the selection of the minimum values on the reaction surface curves per parameter. The membrane stress (21 MPa) was less than the allowable stress (58.5 MPa); however, the stress (78.3 MPa) at the porous support–gas layer interface was confirmed to be higher than the minimum allowable stress (32 MPa).

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- Maximization of the effective area ratio.
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Figure 26. Response surface curve of the parameters L, W.
4. Discussion

4.1. General Discussion

The application of porous composite ceramics has been expanded to various fields. The material properties of these ceramics must be obtained and their characteristics must be predicted because their mechanical properties change with the pore size, porosity, and microstructure.

Based on these results, we proposed a design method to stabilize the structure of the plate-type OTM module. The porous layer was confirmed to be the weakest part of the OTM module, which agrees with the observations of a previous study [67]. To structurally stabilize the OTM module, the geometry of the gas-channel layer and the width of the gas channel were changed without increasing the thickness of each layer of the OTM module. This could reduce the stress of the porous layer. Based on the structural stability, this study proposed a design method to achieve an effective permeable cross-sectional area ratio of the OTM module.

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4.1. General Discussion

The application of porous composite ceramics has been expanded to various fields. The material properties of these ceramics must be obtained and their characteristics must be predicted because their mechanical properties change with the pore size, porosity, and microstructure.
In addition, the material properties of specimens are significantly different from the stiffness and strength distributions of other materials. This implies that there exists a significant difference between the material properties of each specimen. Several studies have proven that this is due to differences in the pore microstructures [85].

We confirmed that the point in the OTM module at which the maximum stress is generated lies on the porous layer, i.e., this layer is the weakest point in the module. Studies have suggested increasing the thickness of the porous layer to compensate for its structural weakness. This approach could overcome the structural weakness, but it eventually returns to its vulnerable state because of the relatively low oxygen permeability in thick layers, which warrants the application of higher external pressures.

Therefore, we selected the geometric shape parameters of the support structure of the gas-channel layer (and not the porous layer) and the width of the gas channel as the design variables and optimized it without increasing the thickness of the porous layer.

The objective function of a typical optimization procedure is to minimize the maximum stress value among the fragile structural parts or in the overall model. However, if the design variables and input variables for the optimization are proportional to the objective function, or if the input variables are also selected via objective function minimization, the results may not converge or the process may yield incorrect results. Therefore, the correct objective function must be selected by observing the influence of the design variables and the response curves before performing the optimization.

In this study, increasing the effective permeable cross-section relative to the reaction area of the membrane module, which was the priority of the OTM module design, was selected as the final objective function. Moreover, the proposed approach eliminated the need for further parametric studies proportional to the input and objective function. The results demonstrated an effective oxygen permeable area ratio and paved the way for an optimal, structurally stable OTM module design.

4.2. Limitations

The structural analysis results presented in Section 3 do not include the effect of flow analysis results. Simulation studies on OTM membranes reveal the porous layer or porous support as the vulnerable component of the membrane. The studies suggested increasing the thickness of the porous layer to enhance the structural strength and stabilize the structure. However, this reduces the oxygen transmission rate. Further verification of oxygen permeability is required to solve this problem. Studies have attempted to replace the verification with flow analysis. In this study, the structural strength was enhanced through structural analysis, without increasing the thickness and affecting the oxygen permeability. Additionally, the optimal structural design of an OTM membrane module was determined by selecting the parameters that influence the geometrical shape of the weak region to stabilize the structure.

The current structural optimization design technique determines the optimal values of design variables that satisfy the objective functions and design constraints defined for a given physical condition, using a structural analysis program based on mathematical theory. Recent studies have designed optimal structures by treating not only the dimensions and geometric shapes but also the topology and material compositions as design variables. Several studies have developed optimization programs that couple multiphysics. These programs require a significant amount of time to analyze each physical system and implement the results in a complex system, in addition to verifications. Therefore, to propose an optimal design for the OTM module, we replaced the flow effect with structural optimization analysis through the constraint function, which maintains oxygen permeability.

However, the reliability of the OTM module can be further improved. Recent multiphysics optimization programs that effectively interpret multiphysics systems are now bundled with commercial software. Structural optimization analyses that consider the flow–structure interaction could identify problems not covered in this study. The results could help modify and supplement the proposed design, albeit at a longer computational time.
5. Conclusions

We analyzed the structure of a modified plate-type OTM module composed of a composite ceramic material with a porous layer. A whole model, a half model, and a section model were analyzed to obtain a comprehensive profile. The results confirmed the porous layer to be the most vulnerable part of the module because of the external–internal pressure gradient and the difference of material properties between the porous layer and the gas-channel layer. To retain the oxygen permeability and oxygen permeation area ratio of the OTM module, the porous layer thickness and the oxygen permeation area of the gas-channel layer were not changed. To enhance the structural stability, the structures were optimized by introducing a design parameter that changed the geometrical shape around the region with the maximum stress. Based on the results, the final design of the OTM module was proposed.

The results of this study can be used to supplement the design of the region around the porous layer and promote structural stability.

Future studies must consider the effect of flow to confirm the inverse relationship between the thickness and transmittance of the porous layer, which has the most considerable influence on the actual transmittance. Therefore, the permeability and structural stability of the porous layer must be enhanced through an optimization program that couples a multiphysics system with the flow simulation of the OTM module. The final design recommended in this study must be experimentally validated.

Author Contributions: D.G.L. and S.-H.K. made a concept of device and simulation; D.G.L. conceived and designed the simulations; D.G.L. performed the simulations; D.G.L., J.W.N. and S.W.C. analyzed the data; D.G.L., J.W.N. and S.W.C. wrote the paper. All authors have read and agreed to the published version of the manuscript.

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