Measurement of Linear Springs’ Stiffness Factor Using Ultrasonic Sensing

Zhongwei Zhang *, Xiyan Zhang, Bohui Ma, Mengyao Ding, Bowen Zhu and Dezheng Tong

Department of Optical Engineering, College of Optical, Mechanical and Electrical, Zhejiang A & F University, Hangzhou 311300, China
* Correspondence: zzw3535@zafu.edu.cn

Abstract: We designed an ultrasonic testing instrument that consisted of a single-chip microcomputer module, a digital display module, and an ultrasonic sensor module, which conveniently eliminated the troubles faced by the traditional Jolly’s scale. For comparison purpose, three linear springs’ stiffness factors were measured by Jolly’s scale and by our ultrasonic testing instrument. We found that our instrument could more conveniently and in real time display the distance values between the ultrasonic ranging module and the horizontal bottom plate when loading different weights. By processing these distance data, we found that our instrument was more convenient for obtaining the linear springs’ stiffness factors and that the results were more accurate than those of Jolly’s scale. This study verified that our instrument can accurately realize the performance of Jolly’s scale under diverse temperatures and humidity levels with high data reliability and perfect stability.

Keywords: ultrasonic wave; Jolly’s scale; Hooke’s law; stiffness factor; shape variable

1. Introduction

The linear spring stiffness factor $k$, also known as the stubbornness coefficient, is an important property that indicates the ability of the linear spring to resist deformation [1–3]. Being a kind of elastic object that stores energy, linear springs are widely used in many fields, such as furniture, architecture, machinery, and electronics [4–8]. Depending on the purpose for their usage, different types of linear springs should have different stiffness factors. Generally, $k$ is measured by Jolly’s scale [9] and then calculated based on Hooke’s law [10–12]. However, when using the Jolly’s scale, it is necessary to first “align the three lines” (note: the first line $L_1$ is on the surface of the glass tube, the second line $L_2$ is on the surface of the mirror, and the third line $L_3$ is a reflection image of $L_1$ in the mirror). Next, the readout on the Vernier Caliper must be manually identified after each addition of equal weight, and the measurement results must be evaluated. These will incur artificial accidental errors and lead to inaccurate measurement results.

The term ultrasonic wave refers to sound waves with frequencies higher than 20 kHz, and common ultrasonic frequencies range from tens of kHz to tens of MHz. It is a type of mechanical wave that has the advantages of a strong penetrating force, good directionality, and easy access to concentrated sound energy [13–16]. Due to its advantages of non-contact measurement, low cost, easy operation, and rapid measurement, it has been widely used in industrial measurements, safety warnings, scientific obstacle avoidance by robots, and ultrasonic motors [17–23]. Particularly in distance measurement, as researchers continue to explore and further expand the ultrasonic ranging technology [24–28], measurement accuracy and the stability of ultrasonic ranging are more suitable for industrial control and high-level instruments.

After searching on the Web of Science, we found very few studies on the measurement of the linear spring stiffness factor using ultrasonic sensing, and there was no one instrument using ultrasonic sensing to measure the linear spring stiffness factor. In this study, we used
an ultrasonic testing instrument that consisted of a single-chip microcomputer module, a
digital display module, and an ultrasonic sensor module to measure displacement. For
comparison purposes, three linear springs’ stiffness factors were measured by Jolly’s scale
and by the ultrasonic testing instrument. We found that our ultrasonic testing instrument
could more conveniently eliminate the troubles faced by the traditional Jolly’s scale and
displayed the distance values in real time between the ultrasonic ranging module and
horizontal bottom plate when loading different weights. By processing these distance
data, we found that our ultrasonic testing instrument was more convenient for obtaining
the linear springs’ stiffness factors and that the measurement results were more accurate
than those of Jolly’s scale. This study verified that our instrument can accurately realize
the performance of Jolly’s scale under diverse temperatures and humidity levels and also
excelled in other quantitative ranging measurements.

2. Experimental Method

2.1. Linear Spring Stiffness Factor \( k \)

The linear spring stiffness factor \( k \) complies with Hooke’s law as follows:

\[ F = k \cdot \Delta x \] (1)

In Equation (1), \( F \) is the elastic force received by the linear spring, and \( \Delta x \) is the
deformation quantity of the spring once it is deformed.

In general, we can find the linear spring stiffness factor \( k \) by following three steps with
our ultrasonic testing instrument. First, the weights of different masses are applied to the
oscillator to produce a series of forces of known magnitude. Next, the ultrasonic sensor
module measures the corresponding deformations of the spring. Finally, in line with the
successive differentiation method, the linear spring stiffness factor \( k \) can easily be obtained
from Equation (1).

2.2. Device Composition

Our ultrasonic testing instrument consisted of a single-chip microcomputer (MCU)
module (Model: 51 HC-SR04), a digital display module (digital tube), and an integrated
ultrasonic sensor module (fixed together with the vibration module during use; referred to
as the ultrasonic ranging module in context). The vibration change of the vibration module
was measured by the ultrasonic sensor module, and the values of the measured distance
parameter were displayed on the digital tube in real time. The photo of our ultrasonic
testing instrument is shown in Figure 1.

![Figure 1. Photo of our ultrasonic testing instrument (ultrasonic sensor module was fixed together
with the vibration module during use).](image-url)
2.2.1. Single-Chip MCU Module

The single-chip MCU module is shown in Figure 2. The function and name of each component are marked in the figure.

![Figure 2. Interface of the single-chip MCU module.](image)

2.2.2. Digital Display Module

As shown in Figure 3, the digital display module involved an 8-digit tube. It displayed the measurement data in real time. Using programming techniques, effective data with different numbers of digits can be implemented in the digital display module.

![Figure 3. Photo of digital display module with an 8-digit tube.](image)

2.2.3. Ultrasonic Sensor Module

As shown in Figure 4, the ultrasonic sensor module used an I2C or TTL serial port interface to communicate with the host and output the distance value in the millimeter order of magnitude. The range detection with real-time temperature compensation had high detection accuracy. Moreover, it could transmit multi-range detection instructions to meet the requirements of short-distance detection, along with quickly and precisely measuring the temperature and ultrasonic intensity.

2.3. Physical Principle of Ultrasonic Ranging

Ultrasonic waves are generated by the vibration of a transducer wafer under voltage excitation. When an ultrasonic wave hits an impurity or the interface of an obstacle, either it will produce a significant reflection to form an echo or the Doppler effect will occur if the wave meets a moving object.
The front face of KS103.

(a) The front face of KS103.

(b) The back face of KS103.

Figure 4. Photo of ultrasonic sensor module KS103.

Our instrument adopted dual probes: one was the transmitter, and the other was the receiver, as shown in Figures 4a and 5. The distance between the transmitter and receiver is \( d \). The relationship between the measured distance \( L_M \) and actual distance \( L_A \) is 
\[
L_M = \sqrt{L_A^2 + \frac{d^2}{4}}.
\]
When the value of \( L_A \) changes, the value of \( L_M \) can be displayed in real time on the digital display module through the measurement of the Doppler frequency shift by the ultrasonic ranging module.

Figure 5. Physical principle of ultrasonic ranging.

3. Measuring Linear Springs’ Stiffness Factors

As shown in Figures 6 and 7, we selected three linear springs with different stiffness factors and measured them with the traditional Jolly’s scale and our ultrasonic testing instrument, respectively. (Note: The three legs of our instrument were strictly locked to prevent sliding.) In the experiments, we added a fixed mass (5.05 g) to the lower end of each spring each time in turn, and we recorded the readouts of the two instruments when their states were stable. The readout \( l_i \) (\( i = 0, 1, 2, 3, 4, 5 \)) on the Jolly’s scale is a series of indicators of the vernier caliper when the scale realizes “aligning the three lines”, while the readout \( l_i \) (\( i = 0, 1, 2, 3, 4, 5 \)) on the ultrasonic sensor is a series of distance values between the ranging module and the horizontal bottom plate after the vibration module is stable and stationary. Our instrument automatically showed these values with the digital display module, which ruled out artificial accidental errors. 
\[
\overline{\Delta l} = \frac{|(l_3 - l_0) + (l_4 - l_1) + (l_5 - l_2)|}{3}
\]
is the average difference of spring’s elongation. The measured and processed results are listed in Table 1. Among them, 
\[
k = \frac{\Delta m \Delta l}{M} = \frac{(15.15 - 0.00) \times 10^{-3} \times 9.8}{M \times 10^{-2}}
\]
and the standard deviation of the deformation is 
\[
\sqrt{\frac{(l_3 - l_0)^2 + (l_4 - l_1)^2 + (l_5 - l_2)^2}{3}}.
\]
The relative error of \( k \) is \( |k - k_0|/k_0 \times 100\% \). True value \( k_0 \) is the linear spring’s stiffness factor provided by the factory and measured with a dynamometer, according to Hooke’s law. We bought springs with different stiffness factors from the factory, and the manufacturer marked the values of the stiffness factors on the outer package. We referred to these as the true
values. In Chinese university-level physics experiment textbooks, the term “true value” is a default regulation.

![Image of a spring testing device]

**Figure 6.** The stiffness factor $k$ of the spring is measured with the Jolly’s scale.

![Image of an ultrasonic testing instrument]

**Figure 7.** The stiffness factor $k$ of the spring is measured by our ultrasonic testing instrument.

For the expression of $k$, the combinations of $l_3$ and $l_0$, $l_4$ and $l_1$, and $l_5$ and $l_2$ correspond to 15.15 g and 0.00 g, 20.20 g and 5.05 g, and 25.25 g and 10.10 g, respectively. Thus, $\Delta m = 15.15 - 0.00 = 20.20 - 5.05 = 25.25 - 10.10 = 15.15$ (g), which means that it concerns a series of measures with the same weight.

In general, the smaller the value of the standard deviation is, the smaller the deviation of the measurement value is, and the more accurate the measurement is. The smaller the value of the relative error of $k$ is, the closer the measurement value is to the true one.

Table 1 shows that the stiffness factors of the same group of three springs measured by our ultrasonic testing instrument and Jolly’s scale were very close, and the standard deviations of the deformation indications were very small. Clearly, the standard deviations of the deformation measured by our ultrasonic testing instrument were much smaller than those of Jolly’s scale, demonstrating the accuracy of our ultrasonic testing instrument’s
results. In addition, we know their true values. The relative errors between the measured results by our ultrasonic testing instrument and the true values were 0.26%, 0.00%, and 0.00%, respectively, better than those of Jolly’s scale, which were 1.8%, 0.67%, and 1.6%, respectively. Using the software Origin Pro 2015 to map, Figure 8 shows the comparison between the measurement results from our instrument and Jolly’s scale accompanied by their respective fitting straight lines. It is known from the software that the standard deviations of the slopes of the fitting straight lines obtained are all within 0.004, exhibiting perfect linearity in the fitting diagram and high accuracy in the measurement results. In summary, the results in Table 1 and Figure 8 show clearly that our ultrasonic testing instrument can accurately measure the stiffness factor of springs, that it can fully realize the performance of the traditional Jolly’s scale, and that its measurement results are more accurate than those of Jolly’s scale.

Table 1. Comparison table of the measurement results of the three springs’ stiffness factors.

| Mass of Weights (g) | 0.00 | 5.05 | 10.10 | 15.15 | 20.20 | 25.25 |
|---------------------|------|------|-------|-------|-------|-------|
| Group               |      |      |       |       |       |       |
| Spring 1            |      |      |       |       |       |       |
| Jolly’s scale       | 1.37 | 2.67 | 4.05  | 5.33  | 6.54  | 7.73  |
| Ultrasonic sensor   | 35.45| 34.16| 32.94 | 31.61 | 30.26 | 28.92 |
| True value k₀       | /    | /    | /     | /     | /     | /     |
| Spring 2            |      |      |       |       |       |       |
| Jolly’s scale       | 6.16 | 9.50 | 12.76 | 16.03 | 19.27 | 22.61 |
| Ultrasonic sensor   | 45.92| 42.64| 39.37 | 36.04 | 32.69 | 29.47 |
| True value k₀       | /    | /    | /     | /     | /     | /     |
| Spring 3            |      |      |       |       |       |       |
| Jolly’s scale       | 1.98 | 3.90 | 5.91  | 7.84  | 9.78  | 11.72 |
| Ultrasonic sensor   | 38.26| 36.27| 34.38 | 32.32 | 30.34 | 28.46 |
| True value k₀       | /    | /    | /     | /     | /     | /     |

In general, both temperature and humidity have effects on ultrasonic waves [29,30]. To verify the sensitivity and stability of our ultrasonic testing instrument, we measured the stiffness factors of each spring at different times on the same day (under different
temperatures and humidity levels). The experiment and processed results are shown in Tables 2 and 3, respectively.

Table 2. Measurement results of each spring using our ultrasonic testing instrument at different times on the same day under different temperatures and humidity levels.

| Date       | Time | Temperature (°C) | Humidity (%) | l₀ (cm) | l₁ (cm) | l₂ (cm) | l₃ (cm) | l₄ (cm) | l₅ (cm) | kᵢ (N/m) |
|------------|------|------------------|--------------|---------|---------|---------|---------|---------|---------|-----------|
| On the same day | 9:30 | 17.2 | 74        | 37.51   | 36.16   | 34.83   | 33.55   | 32.27   | 30.97   | 3.794220  |
|            | 13:30| 18.1 | 71        | 37.29   | 35.96   | 34.71   | 33.34   | 32.1    | 30.77   | 3.791127  |
|            | 17:30| 16.4 | 79        | 38.68   | 37.47   | 36.15   | 34.73   | 33.49   | 32.33   | 3.791928  |
|            | 21:30| 15.2 | 80        | 37.41   | 36.14   | 34.82   | 33.44   | 32.21   | 30.97   | 3.791342  |

True value k₀ / / / / / / 3.80

Table 3. Statistics of the stiffness factors of the same spring under different temperatures and humidity levels.

| Temperature (°C) | Humidity (%) | kᵢ (N/m) | Sᵢ (N/m) | ReAD (%) | Eᵪ (%) | SE (N/m) |
|------------------|--------------|----------|----------|----------|--------|----------|
| 17.2             | 74           | 3.794220 | 0.04     | 0.03     | 0.03   |          |
| 18.1             | 71           | 3.791127 | 0.05     | 0.03     | 0.03   |          |
| 16.4             | 79           | 3.791928 | 3.79     | 0.07     | 1.6    | 0.26     |
| 15.2             | 80           | 3.791342 | 0.07     | 0.04     | 0.04   |          |

True value k₀ 3.80 / / / / / /

Note: Each stiffness factor of each spring at different times on the same day is kᵢ = \( \frac{\Delta m}{\Sigma (l_i - l₀)} \) = \[ \frac{\Delta m \times 10^{-3} \times 9.8}{(l₀ - l₁) \times 10^{-2}} + \frac{\Delta m \times 10^{-3} \times 9.8}{(l₁ - l₂) \times 10^{-2}} + \frac{\Delta m \times 10^{-3} \times 9.8}{(l₂ - l₃) \times 10^{-2}} \] \( \times 3; \Delta m = 15.15 \) (g). The mean value of kᵢ is \( \bar{k} = \frac{1}{n} \sum_{i=1}^{n} kᵢ \). The standard deviation of kᵢ is \( Sᵢ = \sqrt{\frac{1}{n-1} \sum_{i=1}^{n} (kᵢ - \bar{k})^2} \). The average standard deviation of kᵢ is ReAD = \( \frac{Sᵢ}{\bar{k}} \). The relative error of kᵢ is \( Eᵪ = \frac{\mid \bar{k} - kᵢ \mid}{\bar{k}} \times 100\% \). The standard error of kᵢ is SE = \( Sᵢ / \sqrt{3} \).

Table 2 shows the comparison of each spring’s stiffness factor with our ultrasonic testing instrument under different temperatures and humidity levels. To express these data intuitively, we converted them into Figure 9, which shows perfect linearity in the fitting diagram and high accuracy in the measurement results. The slopes of these four fitting lines are exactly equal, and they almost coincide with the exception of the line representing Time 17:30. For the line representing Time 17:30, we consciously increased the initial height of the \( l₀ \) to test the sensitivity of our instrument. As expected, the slope of the fitting line formed by subsequent measurements was perfectly consistent with the slope of the other three. This showed that our instrument was sensitive to the initial value.

Using the values of kᵢ and the stiffness factor of each spring at different times on the same day, obtained from a series of measures with the same weight, we performed the uncertainty analysis based on the data in Tables 2 and 3 as follows: Our instrument’s instrument error value was \( \Delta q = k₀ - kᵢ, \min = 3.80 - 3.791127 = 0.008873 \) (N/m). The overall uncertainty is \( \sigma = \sqrt{\Delta q^2 + \sum \frac{1}{4} \left( \frac{\sigma_i}{\bar{k} - kᵢ} \right)^2} = \sqrt{0.008873^2 + 0.072^2} \approx 0.07 \) (N/m). Considering the overall uncertainty, \( kᵢ = \bar{k} \pm \sigma = 3.79 \pm 0.07 \) (N/m). This meant that the probability of \( kᵢ \) was high in the interval (3.72, 3.86), and the probability of true value being outside the interval was small.

Table 3 showed that the mean value of the measured stiffness factors of each spring using our ultrasonic testing instrument under different temperatures and humidity levels was 3.79 N/m. We know the true value of this spring is 3.80 N/m, and the following data are calculated on the true value. The average standard deviation of the measured
stiffness factors was 0.06 N/m, and the relative average standard deviation was 1.6% (<5.0%, confidence interval). The relative error of $k_i$ was 0.26%, and the values of the standard errors are all within the range of 0.00–0.05 N/m. All these data indicate that the difference extent between the stiffness factors measured by our instrument and the true value 3.80 N/m was negligible and reflected the high precision of the measurement results. In summary, our ultrasonic testing instrument measured the linear springs’ stiffness factors under diverse temperatures and humidity levels, with high data reliability and perfect stability.

Figure 9. (Color online) Measurement results of each spring using our instrument at different times on the same day under different temperatures and humidity levels, accompanied by their respective fitting lines.

4. Conclusions

We designed an ultrasonic testing instrument that used ultrasonic waves to measure the stiffness factors of linear springs. We measured the stiffness factors of three different linear springs with Jolly’s scale and our instrument. By comparison, we found that our instrument was more convenient to operate than Jolly’s scale. We verified that our instrument had the advantages of accurate results with little error, perfect sensitivity, and stable performance in measuring the linear springs’ stiffness factors under diverse temperatures and humidity levels, which precisely surpassed the performance of Jolly’s scale. In addition, our instrument can also be exploited to measure the surface tension and surface tension coefficients of liquids, Young’s modulus of a wire, tensile strength of a paper, tiny deformation of objects, flexibility of a wool wire, and other quantitative measurements.

Author Contributions: Z.Z. conceived and designed the analysis, collected the data, performed the analysis, wrote the paper, and acquired funding. X.Z. contributed data and analysis tools, wrote the paper, and revised the paper. B.M., M.D., B.Z. and D.T. conceived and designed the electronic circuit for ultrasonic experiments and contributed data and analysis tools. All authors have read and agreed to the published version of the manuscript.
Funding: This research was funded by [the Project of Science and Technology Innovation of University Students in Zhejiang Province & Project of New Talent, China] grant number [2020R412031 and 2021R412033]; [the Project of National College Students’ Innovation and Entrepreneurship Training Program, China] grant number [202110341068X]; [the Second Batch of Industry-Education Cooperation and Collaborative Education Project of the Ministry of Education, China] grant number [202102227068 and 202102227069]; [Zhejiang Agriculture and Forestry University Project] grant number [KC2021058]. And The APC was funded by [2020R412031, 2021R412033, 202110341068X, 202102227068, 202102227069, KC2021058].

Institutional Review Board Statement: This study did not require ethical approval for it not involving humans or animals.

Informed Consent Statement: This study did not involve humans.

Data Availability Statement: We provide detailed data supporting reported results can be found in all the Tables and Figures in this paper.

Acknowledgments: This work was supported by the Project of Science and Technology Innovation of University Students in Zhejiang Province & Project of New Talent, China (Nos. 2020R412031 and 2021R412033); the Project of National College Students’ Innovation and Entrepreneurship Training Program, China (No. 202110341068X); the Second Batch of Industry-Education Cooperation and Collaborative Education Project of the Ministry of Education, China (Nos. 202102227068 and 202102227069); the Zhejiang Agriculture and Forestry University Project (No. KC2021058). The authors would like to acknowledge these valuable supporters. The authors also thanks Liang Mao, Kai Ren, Long Ying, and Wenyu Feng for their help in experiments.

Conflicts of Interest: The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this study.

References

1. Tan, B.; Feyzullahoglu, E. The investigation of effects of environmental conditions on fatigue strength and spring stiffness of rubber materials. J. Elastom. Plast. 2021, 53, 632–652. [CrossRef]
2. Liu, M.; Zhan, J.; Zhu, B.; Zhang, X. Topology optimization of compliant mechanism considering actual output displacement using adaptive output spring stiffness. Mech. Mach. Theory 2020, 146, 103728. [CrossRef]
3. Wu, J.; Zhang, L.; Wan, L. A mode-III crack under adhesion studied by non-uniform linear spring models. Acta Mech. 2017, 228, 1621–1629. [CrossRef]
4. Chung, D.D.L.; Shi, K. Sensing the stress in steel by capacitance measurement. Sens. Actuators A Phys. 2018, 274, 244–251. [CrossRef]
5. Yi, H. Simulation of Shape Memory Alloy (SMA)-Bias Spring Actuation for Self-Shaping Architecture: Investigation of Parametric Sensitivity. Materials 2020, 13, 2485. [CrossRef]
6. Rantz, R.; Halim, M.A.; Xue, T.; Zhang, Q.; Gu, L.; Yang, K.; Roundy, S. Architectures for wrist-worn energy harvesting. Smart Mater. Struct. 2018, 27, 0440014. [CrossRef]
7. Jiang, Q.; Qiao, Y.; Zhao, F.; Pan, Z.; Wu, X.; Wu, L.; Fu, H. Composite helical spring with skin-core structure: Structural design and compression property evaluation. Polym. Compos. 2020, 42, 1292–1304. [CrossRef]
8. Xue, Z.; Sun, M.; Dong, T.; Tang, Z.; Zhao, Y.; Wang, J.; Wei, X.; Yu, L.; Chen, Q.; Xu, J.; et al. Deterministic Line-Shape Programming of Silicon Nanowires for Extremely Stretchable Springs and Electronics. Nano. Lett. 2017, 17, 7638–7646. [CrossRef]
9. Chunling, Z.; Qian, S.; Xiumei, C.; Lei, T. Discussion on the experiment of measuring the effective mass of spring by Jolly’s scale. Coll. Phys. 2017, 36, 31–35.
10. Graves, J.; Kuang, Y.; Zhu, M. Pendulum energy harvester with torsion spring mechanical energy storage regulator. Sens. Actuators A Phys. 2022, 339, 113505. [CrossRef]
11. Martynov, N.I. Complex Form of Hooke’s Law of Anisotropic Elastic Body. Mech. Solids 2020, 55, 514–535. [CrossRef]
12. Fahy, S.; O’Riordan, J.; O’Sullivan, C.; Twomey, P. How reflected wave fronts dynamically establish Hooke’s law in a spring. Eur. J. Phys. 2012, 33, 417–426. [CrossRef]
13. Zhu, X.; Huang, Z.; Wang, G.; Li, W.; Zou, D.; Li, C. Ultrasonic detection based on polarization-dependent optical reflection. Opt. Lett. 2017, 42, 439–441. [CrossRef] [PubMed]
14. Kim, G.; Seo, M.; Choi, N.; Baek, K.S.; Kim, K. Application of KLM Model for an Ultrasonic Through-Transmission Method. Int. J. Precis Eng. Man. 2019, 20, 383–393. [CrossRef]
15. Kang, H.S.; Song, C.G.; Lee, K.I. Simultaneous measurement of acoustic pressure and temperature at the focus of high intensity focused ultrasound in a tissue-mimicking phantom by using a fiber-optic hydrophone. Ipn. J. Appl. Phys. 2019, 58, 0909089. [CrossRef]
16. Cao, Y.; Chen, Q.; Zheng, H.; Lu, L.; Wang, Y.; Zhu, J. Study on the Mechanism of Ultrasonic Power Measurement Sensor based on Pyroelectric Effect. *Acoust. Phys.* 2018, 64, 789–795. [CrossRef]
17. Bakhtiari-Nejad, M. Multi-focal transmission acoustic phase holograms in contactless ultrasonic power transfer systems. *Sens. Actuators A Phys.* 2022, 340, 113551. [CrossRef]
18. Kang, L.; Feeney, A.; Dixon, S. The High Frequency Flexural Ultrasonic Transducer for Transmitting and Receiving Ultrasound in Air. *IEEE Sens. J.* 2020, 20, 7653–7660. [CrossRef]
19. Zhong, H.; Duan, T.; Lan, H.; Zhou, M.; Gao, F. Review of Low-Cost Photoacoustic Sensing and Imaging Based on Laser Diode and Light-Emitting Diode. *Sensors* 2018, 18, 2264. [CrossRef]
20. Harvey, G.; Gachagan, A.; Mutasa, T. Review of High-Power Ultrasound-Industrial Applications and Measurement Methods. *IEEE T. Ultrason. Ferr.* 2014, 61, 481–495. [CrossRef]
21. De Simone, M.C.; Rivera, Z.B.; Guida, D. Obstacle Avoidance System for Unmanned Ground Vehicles by Using Ultrasonic Sensors. *Machines* 2018, 6, 18. [CrossRef]
22. Li, S.; Feng, C.; Liang, X.; Qin, H.; Li, H.; Shi, L. A Guided Vehicle under Fire Conditions Based on a Modified Ultrasonic Obstacle Avoidance Technology. *Sensors* 2018, 18, 4366. [CrossRef] [PubMed]
23. Li, H.; Liu, Y.; Deng, J.; Chen, W.; Li, K. Design philosophy for ultrasonic motors using the bending hybrid modes. *Sens. Actuators A Phys.* 2021, 331, 113029. [CrossRef]
24. Fitzka, M.; Rennhofer, H.; Catoor, D.; Reiterer, M.; Lichtenegger, H.; Checchia, S.; di Michiel, M.; Irrasch, D.; Gruenewald, T.A.; Mayer, H. High Speed In Situ Synchrotron Observation of Cyclic Deformation and Phase Transformation of Superelastic Nitinol at Ultrasonic Frequency. *Exp. Mech.* 2020, 60, 317–328. [CrossRef]
25. Robichaud, A.; Cicek, P.; Deslandes, D.; Nabki, F. Frequency Tuning Technique of Piezoelectric Ultrasonic Transducers for Ranging Applications. *J. Microelectromech. S.* 2018, 27, 570–579. [CrossRef]
26. Wang, Y.; Qiao, Y.; Zhang, H.; Gao, Y.; Zhang, M.; Tan, H.; Wang, D.; Jin, B. Transformerless Ultrasonic Ranging System with the Feature of Intrinsic Safety for Explosive Environment. *Sensors* 2018, 18, 4397. [CrossRef]
27. Liu, L.; Guo, R.; Wu, J. Design of a fast echo matching algorithm to reduce crosstalk with Doppler shifts in ultrasonic ranging. *Meas. Sci. Technol.* 2017, 28, 025103. [CrossRef]
28. Zhang, B.; Wei, Y.; Liu, W.; Zhang, Y.; Yao, Z.; Zhang, L.; Xiong, J. A Novel Ultrasonic Method for Liquid Level Measurement Based on the Balance of Echo Energy. *Sensors* 2017, 17, 706. [CrossRef]
29. Mujahid, A.; Afzal, A.; Dickert, F.L. An Overview of High Frequency Acoustic Sensors-QCMs, SAWs and FBARs-Chemical and Biochemical Applications. *Sensors* 2019, 19, 4395. [CrossRef]
30. Guntekin, E.; Aydin, T.Y.; Niemz, P. Prediction of Compression Properties in Three Orthotropic Directions for Some Important Turkish Wood Species Using Ultrasound. *Bioresources* 2015, 10, 7252–7262. [CrossRef]