High-resolution nanomechanical analysis of suspended electrospun silk fibers with the torsional harmonic atomic force microscope

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Abstract
Atomic force microscopes have become indispensable tools for mechanical characterization of nanoscale and submicron structures. However, materials with complex geometries, such as electrospun fiber networks used for tissue scaffolds, still pose challenges due to the influence of tension and bending modulus on the response of the suspended structures. Here we report mechanical measurements on electrospun silk fibers with various treatments that allow discriminating among the different mechanisms that determine the mechanical behavior of these complex structures. In particular we were able to identify the role of tension and boundary conditions (pinned versus clamped) in determining the mechanical response of electrospun silk fibers. Our findings show that high-resolution mechanical imaging with torsional harmonic atomic force microscopy provides a reliable method to investigate the mechanics of materials with complex geometries.

Introduction
Dynamic atomic force microscopy (AFM) methods provide opportunities for high-resolution compositional mapping of heterogeneous samples [1]. Recent developments in dynamic AFM methods offer the possibility of relating the measured vibration signals to the particular physical properties of the samples, such as elastic modulus, viscosity, adhesion, and chemical affinity [2-16]. These developments are accomplished by employing multiple excitation and detection frequencies during dynamic AFM imaging [17-26]. A critical element of these mechanical measurements is the physical model being used to relate the force–distance curves to parameters representing the material properties. Although contact-mechanics models can be used for a wide variety of polymer composites, block-copolymers, and biomaterials [27-34], these models are
not applicable to materials with complex geometries. For example, the use of local interaction models provides limited information in the case of suspended structures where bending modulus, geometry, and mechanical tension are the key determinants of tip–sample interactions. The interpretation of measurements on these kinds of samples requires simultaneous analysis of mechanical measurements and topography, as well as comparison of various mechanical models.

In this work we investigate the mechanical behavior of electrospun silk fibers, which are used for making scaffolds for bone-tissue engineering [35]. Mechanical characteristics of these structures are important because their primary purpose is to mimic extracellular-matrix conditions, including their rigidity. Bulk properties, while important, are not sufficient to predict the mechanical behavior of the electrospun fibers. Geometry of the network of fibers, fiber diameter, mechanical boundary conditions at the nodes of the fiber network (pinned versus clamped), and the presence of mechanical tension within the fibers can influence their mechanical behavior. We have carried out experiments to determine the relative influences of these parameters on the mechanics of electrospun silk fibers.

Results and Discussion
Electrospun silk fibers form mesh-like networks with nodes and branches. Diameters of these fibers are typically in the submicron range. Separation between the nodes, defined by intersections between two or more fibers, can be on the order of one to ten micrometers depending on the electrospinning process. This size scale is readily accessible by atomic force microscopy for topographical and mechanical characterization. When several fiber layers are deposited to form fibrous tissue scaffolds, these branches form suspended structures. We have limited our experiments to samples that are formed by two to three layers of fibers so that we can readily identify individual fibers. Although some of the branches in the first few layers appear to rest on the substrate, some branches still form suspended fibers sufficient for our study (Figure 1).

We have used torsional harmonic AFM to determine the surface topography and local mechanical response with high spatial resolution [20,31]. This mode uses a T-shaped cantilever with an offset tip. When used in dynamic AFM, the cantilever vibrates up and down, similar to conventional cantilevers. In addition to the vertical motion, tip–sample-interaction forces twist the cantilever by a detectable amount. The high bandwidth of torsional motion allows accessing higher harmonics of the tip–sample-interaction forces to reconstruct tip–sample-force waveforms. This process involves calibration of the frequency response of the torsional mode by measuring its resonance frequency and quality factor (either by frequency sweeps or from the thermal peak in the noise spectrum). The gain of the torsional mode, defined as the photodetector signal corresponding to a unit amount of a quasi-static force acting on the tip, is determined by independently measuring the quasi-static force from vertical deflections while monitoring the torsional deflection signal. Note that the same force acts on both vertical and torsional modes. Therefore, after calibrating the vertical spring constant, the gain of the torsional mode can be determined by comparing time-average detector signals in vertical and horizontal channels during a tapping-mode AFM experiment. To minimize contributions of drift in quasi-static deflection signals, we previously developed a procedure that takes advantage of the transitions between attractive and repulsive modes [36]. The calibrated frequency response and gain of the torsional mode allows the reconstruction of the tip–sample-force waveforms. A computer program carries out these calculations in real time during the tapping-mode imaging process. The program also corrects for nonlinearities of the position-sensitive diode and for crosstalk from large vertical signals into torsional vibration signals. Once the tip–sample-force waveform is determined, the program constructs force–distance curves using the distance information in the vertical deflection signals [20]. It is possible to analyze these force–distance curves according to various physical models to obtain parameters describing the mechanical response of the sample. In the case of electrospun silk fibers, we have calculated both the local elastic modulus and the local spring constant values. The elastic modulus is calculated according to the Derjaguin–Muller–Toporov (DMT) model and the spring constant (stiffness) is calculated by fitting the unloading portion of the force–distance curve with a straight line. For the DMT model, we used a tip radius of 7 nm, which is characterized by blind reconstruction from a sample with sharp edges. Our calcu-
Figures 2: Simultaneously measured topography (a), elastic modulus (b), and stiffness (c) maps obtained from electrospun silk fibers. Color bars in (a–c) correspond to the ranges in height (0–1.8 µm), elastic modulus (10 MPa to 10 GPa, mapped logarithmically), and stiffness (0–5 N/m). The horizontal fiber appears to be suspended above the glass substrate. A 3-D rendering of the topography image is given in (d). The fiber is suspended between positions indicated by arrows in (d). This image is colored according to the local spring constant. Both the elastic modulus and stiffness maps show gradual variations across the suspended fiber. Line profiles of elastic modulus and stiffness across the dashed line in (b) are given in (e) and (f), respectively. While the local elastic modulus of the silk fiber is likely to be constant across the length of the fiber, the values in (e) show significant variation. This is because the elastic modulus values in (b,e) are calculated by the DMT contact-mechanics model, which does not take the suspended geometry of the fiber into account. Therefore, the regions of the elastic modulus image corresponding to suspended fibers are not reliable. These regions are better analyzed in the light of mechanical models describing the entire suspended structure by using the stiffness values in (c,f).
resonance frequency of the cantilever, about 1 MHz. One can estimate the resonance frequency of the suspended fiber structure using Euler–Bernoulli analysis [37]:

\[ f = \frac{\beta^2}{8\pi} \sqrt{\frac{E D}{\rho L^2}}. \]  

Here the constant \( \beta^2 \) is equal to 22.373 for the clamped-end boundary condition. \( E \) and \( \rho \) are the elastic modulus and mass density, and \( D \) and \( L \) are the diameter and length of the silk fiber. Using \( E = 10 \) GPa, \( \rho = 1.3 \) g/cm\(^3\), \( D = 2R = 0.52 \) µm, we obtain the resonance frequency \( f = 129.4 \) MHz, which is far above the torsional resonance frequency of the AFM cantilever. Therefore, we neglect the effects of the inertia of the fibers in our experiments.

While gradual changes in stiffness of the suspended fiber are not surprising, the precise mechanism that determines the mechanical response of the suspended fiber is not immediately clear. We identified three scenarios that can qualitatively explain the observed results. The suspended fiber can be viewed as a cantilever structure pinned at both ends, clamped at both ends, or as a string that is under tension. Graphical depictions of these three cases are given in Figure 3a–c. All three scenarios would result in variations in the local stiffness of the fiber as probed by the AFM tip. However, the stiffness values predicted by these models would have different spatial dependencies. It is worth noting that all these models assume that the displacement of the fiber at the nodes is zero, which would result in an effectively infinite spring constant at these locations. In our experiments, the spring constant at the nodes are finite and determined by both the tip–fiber contact-mechanics and the spring constant associated with fiber–fiber interactions at the nodes. To take these effects into account, we assumed a simple model depicted in Figure 3d, which we refer to as the suspended-rigid-rod model. The variables required by all four models and the equations describing local spring constants based on these models are listed in Table 1. Note that the effective spring constant originating from the suspended-rigid-rod model acts in series with the other three models. Additionally, a more sophisticated model could include the fiber–tip spring constant, which acts in series with the spring constants due to fiber–fiber interactions at the nodes. The two models have to give the same total spring constant at the nodes, but the model that takes the fiber–tip spring constant into account results in a nonlinear dependency on the distance.

To determine if any of the three models in Figure 3a–c, in combination with the rigid-rod model in Figure 3d, can explain the observed variations in spring constant, we attempted to fit the data in Figure 2f with the total spring constant \( K(x) \) based on equations listed in Table 1. For the pinned-end and clamped-end models, we used \( E \) and \( L \) as variables for the fitting. For the tension model, we used \( T \) and \( L \) as variables. For all three models, \( K_S \) and \( K_T \) are assumed to be constant and equal to 26.2 N/m and 18.4 N/m, respectively. In addition we used \( R = 0.26 \) µm. The values for \( K_S \) and \( K_T \) are determined from the peak spring constant values in the data plotted in Figure 2f. The value for \( R \) is determined from the topography measurements in Figure 2a. The values of the parameters used for fitting are also listed in Table 1 and the resulting curves are plotted in Figure 4.

From the results of the fitting procedures we see that all three models could reproduce qualitative trends similar to the
Table 1: Description of the variables and equations for spring constants based on the four mechanical models depicted in Figure 3. The values of variables calculated by curve fitting are given in the last column. Standard errors are given in parenthesis with the same units.

| Model description       | Variables                  | Constants          | Equation                                                                 | Best fit (standard error) |
|-------------------------|----------------------------|--------------------|--------------------------------------------------------------------------|---------------------------|
| Pinned end              | $E$: elastic modulus       | $R$: fiber radius  | $K_P(x) = \frac{3\pi ER^4}{4x^2(L-x)^2}$                               | $E = 35.47 \text{ GPa (0.29)}$ |
|                         | $L$: branch length         |                    |                                                                           | $L = 13.59 \text{ µm (0.014)}$ |
|                         | $x$: position              |                    |                                                                           |                           |
| Clamped end             | $E$: elastic modulus       | $R$: fiber radius  | $K_C(x) = \frac{3\pi ER^4}{4x^3(L-x)^3}$                               | $E = 10.16 \text{ GPa (0.13)}$ |
|                         | $L$: branch length         |                    |                                                                           | $L = 14.94 \text{ µm (0.021)}$ |
|                         | $x$: position              |                    |                                                                           |                           |
| Tension                 | $T$: tension               |                    | $K_T(x) = \frac{7L}{x(L-x)}$                                            | $T = 16.73 \text{ µN (0.42)}$ |
|                         | $L$: branch length         |                    |                                                                           | $L = 13.47 \text{ µm (0.003)}$ |
|                         | $x$: position              |                    |                                                                           |                           |
| Suspended rigid rod     | $L$: branch length         | $K_{left}$: left spring constant | $K_S(x) = K_{left} \frac{(L-x)}{L} + K_{right} \frac{x}{L}$          |                           |
|                         | $x$: position              | $K_{right}$: right spring constant |                                                               |                           |

Figure 4: Curves described by equations for pinned end (a), clamped end (b), and tension (c) models fitted to the data. Values of the variables used for fitting are listed in Table 1.

measured spring-constant profile. However, the tension model did not produce a good overall fit to the data. The tension value calculated as 16.7 µN translates to a tensile stress of ≈78.6 MPa, which is a relatively high value, but silk could potentially sustain such large stresses. The other two models provided a better overall fit to the data as seen in Figure 4a and Figure 4b. However, quantitatively the pinned-end model required $E$ to be 35.5 GPa, which is higher than even in native silk fibers ($E \approx 14 \text{ GPa}$ [38]). The clamped-end model predicts $E$ to be around 10.2 GPa and therefore it is more likely that this model provides a better description of suspended silk fibers. Note that the numerical estimates depend on the fourth power of the fiber radius; however, even with 10% increase in radius, the pinned end model cannot provide an elastic modulus value that falls in a plausible range (<15 GPa).

Conclusion

In this paper we have investigated the mechanical behavior of electrospun silk fibers whose geometry does not allow the straightforward use of contact-mechanics models. We used elastic-modulus and stiffness maps determined by torsional harmonic AFM and fitted the data obtained from suspended silk fibers with models that could potentially explain the observed variations in stiffness. This analysis revealed that a clamped-end model, in which the displacements and bending of a fiber are restricted at nodes, successfully describes the observed characteristics. We expect that the applications of the general methodology used in this paper could also be extended to characterization of cytoskeletal protein networks and microelectromechanical (MEMS) devices where suspended structures are commonly encountered.

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