COGENERATION UNIT NOISE REDUCTION BY ITS CASE

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This paper deals with cogeneration unit noise measurement by the acoustic camera. Noise is not only measured as the final number of sound power levels, but also its original location is determined with the use of the beamforming algorithm. The properties of the used microphone array are considered and numerically calculated as every different microscope array layout will measure with another resolution. From the frequency spectrum, the possible technical source is determined. The results of noise source visualization show the cogeneration unit case noise decreasing effect while also offering the possibilities for design improvements.

KEYWORDS

Cogeneration unit, noise, acoustic camera, beamforming, noise measurement

1 INTRODUCTION

Localisation of noise sources is a crucial practice in the design and maintenance of machines because the requirement for modern tools and devices is to operate precisely and quietly. Noise is neither healthy nor ergonomic, and vibrating parts lead to malfunctions. The noise limits for different applications are decreasing every year. Another reason for the importance of acoustic noise diagnostics is that the noise spectra can be used as a tool for malfunction detection or life prediction. Vibrating mechanical parts, impacts, imbalance, malfunctions, or flow of air or other fluids are usually the sources of noise. For complex structures, the problem can arise from too many possible sources.

Exterior noise from the devices can usually be decreased by separating the excitation parts (e.g. an engine) by silent blocks, covering the noise area, or implementing active or passive vibration damping [Wrona, 2019]. For complex machines, it is a challenging task to perform a noise diagnostic without proper feedback.

The ideal tool for best understanding the condition of the device is the visualisation of noise. For this purpose, we can use acoustic cameras which allow us to see the sound power in the real image. However, for interpreting the results, it is mandatory to understand the method and behaviour on the measured frequency area as every camera will perform differently on various frequency bands. In this article, we use an acoustic camera to evaluate the construction of new cogeneration unit chassis and localise the noisiest sources of remaining noise for future innovations [Tuma, 2012].

2 ACOUSTIC CAMERA METHODOLOGY

An acoustic image or video consists of combining a pressure contribution or an acoustic intensity map with a digital picture. The idea is based on simultaneously measuring many acoustic signals, calculating the acoustic intensity power at specified points, and combining this intensity power map with a picture or recorded video. The sound power loss due to distance must be considered, when estimating the correct sound power. Otherwise, we are working with the acoustic pressure or intensity contribution at a measured distance.

The beamforming filter enables the calculation of the acoustic contribution at the specified point for far-field measurements. Near-field holography measurement methods use the information regarding the shape of the wave near the noise emitter. This method is not used for the diagnostics in this article. For every measured signal and every measured point, the aim is to estimate acoustic intensity, the times it takes to arrive from the measured point to the microphones are calculated. Then we delay the signals by these times to get the synchronised signals in the measured point. Lastly, we calculate the arithmetic average. Signals from the direction of the point are close to the original level, while all other directions are dampened. The algorithm is simple but demands many resources. For lower precision error due to discrete delay, a higher sample rate, signal resampling, and interpolation can be used [Gerges, 2009; Miljko, 2011].

The frequency range, in which we are still able to recognise the correct noise sources, depends on the microphone array pattern. Choosing the right criterion to compare different patterns can be difficult. For this article, we are using the Brüel&Kjær WA1764-W-001 acoustic camera [Brüel & Kjær 2020]. The camera uses their patented optimising method by lowering the MSL (maximum sidelobe) where the mainlobe is in the direction of the real noise and the sidelobes are in wrong directions [Hald, 2004a]. The camera’s 30 microphones are specially designed, being built into the microphone array enabling good results for near-field and far-field localisation, while keeping the dimension of the camera small. Randomised patterns show favourable results with far-field localisation while organised patterns (e.g. grids) perform well with near-field localisation. The design of this microphone grid combines these two approaches by dividing the camera into the same segments while optimising the microphone positions in the segment numerically by minimising the MSL.

For correct interpretation of the measurement, it is mandatory to understand the method limits of the camera used as every microphone array will perform differently for contrasting frequency bands. We were not able to use beamforming for low frequencies up to 1.5 kHz due to the results being unclear, making it hard to find the correct location of the noise. However, for these low frequencies, the acoustic camera can be used with the near-field holography SONAH algorithm. The directional characteristic for 1.5 kHz can be seen in Figure 1.

With a higher frequency, the resolution improves, as can be seen from Figure 2; however, more side lobes appear. These results were obtained by numerical simulation of a simple beamforming algorithm with the given microphone pattern. The performance (MSL) then increases with the frequency of measured noise (more and stronger side lobes appear).
Bruel&Kjær provide the MSL dependence on frequency in their product information (see Figure 3). Higher frequencies are hard to localise as MSL decreases, creating ghost images from erroneous locations. Practically, this array can localise noise sources up to 12 kHz. The MSL can be decreased with a higher sample rate or interpolation between samples, so the time delay can be calculated more precisely.

Figure 4. Cogeneration unit

Figure 5 includes a description of the cogeneration unit inner components, which are involved in the acoustic noise of the complete unit.

Not only are the mechanical components significant noise sources but also the exhaust pipeline, although they were equipped with two exhaust silencers to reduce noise. The noise can also get through the unit’s bottom section, through which the cables and the gas hose are routed. The operating range of the cogeneration unit engine speed is from 3200 RPM to 6000 RPM, which corresponds to the output shaft speed from 1350 RPM to 2500 RPM.

4 MEASUREMENT

The main goal of the measurement was to compare the effect of reducing the sound pressure level of noise made by the designed cogeneration unit case. Due to the cogeneration unit system connections the noise measurement was taken from the inner
space with the limitation of noise reflection. The conditions were the same for all measurements; thus, these results are compared. We had three basic types of monitored operating conditions: idle speed, increased speed of 3500 RPM without load, and increased speed of 3500 RPM with load. All these measurements were repeated with and without a running second fan installed inside and all these written combinations were measured with an open and closed case to compare the differences and its efficiency of reducing noise. The measurement was taken by an acoustic camera with an outer diameter of 300 mm using 30 channels at a distance 1.2 m from the cogeneration unit. As seen in Figure 6, the measurement was carried out from two positions, one after another. We made multiple measurements with different conditions and then calculated the noise difference based on altered properties (opened/closed case, load, fans).

The measured signals are weighted by an A-type acoustic filter, which reduces the lower frequency to correspond to the human hearing. As explained before, the measured range for the used camera used for far-field localisation is above 1500Hz. Therefore, this filtering does not significantly affect the results.

5 RESULTS

From the measured data, it is clear to see that there is almost zero effect of turning the fan on and off. The noise from the combustion engine exceeds the fan noise. Furthermore, it was noticed that adding the load at 3500 RPM takes effect in increasing the noise power covered by the camera image by approximately 1,5 dB. A change of RPM naturally raises the noise by 7,4 dB on average. The most researched factor – cogeneration unit case noise damping efficiency – is confirmed by the absorption of 14 dB. Differences in acoustic camera position during the experiment (measurements from the front and the side) are not significant. All results are recorded in Table 1.

![Image](image.png)

**Figure 6. Measurement scheme**

| Conditions                      | AVG ΔL<sub>W</sub> [dB] |
|--------------------------------|------------------------|
| Fan added                      | 0,38                   |
| Load added                     | 1,48                   |
| Increasing of working speed    | 7,38                   |
| Case open                      | 14,10                  |
| Measuring from the front vs from side | 1,36                |

**Table 1. Measurement results**

The average ΔL<sub>W</sub> represents the average sound power level covered by the camera. The average is calculated from six different measurements. The loudest noise was obtained by measuring 3500 RPM with an open case, load and working fan. For this case, we measured a sound power of 85 dB. With the case closed, it reaches approximately 70 dB.

The strongest source of the noise is the gearbox, which is usually the field of research for noise reduction, as seen in Figure 7. The reason is the impact of the gears’ contacts. The second expected source of noise is the combustion engine. However, as seen from the measurements, the noise from the ignitions are lower than the gearbox. Other components are silent compared to these two dominant sound sources.

![Image](image.png)

**Figure 7. Visualisation of noise sources of the cogeneration unit using the acoustic camera – opened case, front view.**

The newly developed case for this cogeneration unit is much more powerful in terms of noise reduction and represents a measured difference of 14 dB in the sound power covered by the camera image. The weak points are the areas which are not enclosed and are used for air ventilation. To be more specific, the remaining noise comes from the bottom of the unit and the ventilation hole as can be seen in Figures 8 and 9.

![Image](image.png)

**Figure 8. Visualisation of noise sources of the cogeneration unit using the acoustic camera – closed case, front view.**
Although noise leaking through the ventilation hole is difficult to reduce due to the airflow needed, there are still possibilities to reduce the noise by properly designing the bottom part of the unit case while also considering all connections.

Even with a higher frequency (Figure 10d), we are still able to see the source (gearbox), but the image contains many ghost images which are created not by a reflection but from the very low MSL. These acoustic camera properties are always needed to be considered when the evaluation is done.

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