In recent years, researchers have shown interest in the study of vacuum glazing technology. Innovative designs were developed and researched. Among them are; Bao, Liu (2014) who have introduced a novel hybrid vacuum/triple glazing system in which a vacuum glazing unit is enclosed by two glass panels to form a triple glazing unit system. To eliminate additional loads due to pressure difference which can cause unwanted breakage, they designed an equalized air pressure arrangement on both sides of the vacuum glazing unit. Fang, Hyde (2015) have performed simulation model using a finite volume model on thermal performance analysis of a hybrid thin film Photovoltaic (PV) Vacuum Glazing.

In this study, we have investigated a hybrid thin film PV vacuum glazing called: ‘PV VG-4L’. The glazing involves an integration between a thin film PV glazing with a double vacuum glazing (both manufactured independently), and an additional layer of self-cleaning coated glass which totalling four layers of glass. The mathematical model of the PV VG-4L designs were developed and numerically solved in MATLAB. To evaluate the performance of the PV VG-4L, the prototype was manufactured and investigated at lab-scale and also under real conditions. Lab-scale experiments were conducted at steady state conditions using a TEC driven calibrated hot box at the Sustainable Energy Research Lab, University of Nottingham, UK. Meanwhile, outdoors, the prototype was tested at a research house at the University of Nottingham, UK. Under the influence of solar irradiance, the electrical performance of the PV-VG and the temperature difference between the surfaces of the glazing were analysed. However, the measurement of U-value under real conditions is not reliable due to the influence of solar irradiance on the heat flux sensor and also due to the absorbed solar irradiance by the thin film PV layer. Nevertheless, during low to zero solar irradiance, the U-value of the prototype can be estimated. The developed model was then validated against the experimental results by direct comparison to the trend of the experimental and theoretical curves obtained, and also by conducting error analysis using root mean squared percentage deviation (RMSPD) method. Testing using the calibrated hot box, adhering closely to ISO 12567 standards, resulted in an average measured total U-value of 0.6 W/m²K which is when compared to a single thin film PV glazing with a typical U-value of 5 W/m²K; the U-value is higher by almost 90%. From the analysis, the computed RMSPD value for the glazing surface temperature and the U-value are 4.02% and 0.92% respectively. Meanwhile, field testing under real conditions with a 0.4 m x 0.4 m PV VG-4L prototype found that 14 W/m² power can be generated by the PV VG-4L at average solar irradiance of ~600 W/m². RMSPD computed glazing surface temperatures, electrical power generated under real conditions and U-value are 2.90%, 8.70% and 2.89% respectively. The theoretical and experimental results are concluded to be in good agreement. This study has significant contributions to the knowledge of building integrated photovoltaic PV technology. The mathematical model that has been developed can be used for PV VG-4L design optimisation and also to simulate the performance of PV VG-4L under various conditions. At building efficiency level, the PV VG-4L not only can produce power, but it also has high insulating properties. The promising U-value implies its range of potential applications which can be improved depending on the energy needs and applications, such as for BIPV solar façade (PV curtain walling) in commercial buildings, greenhouses, skylight and conservatory.

Keywords: PV; vacuum glazing; U-value; calibrated hot box; innovative technology
performance of triple vacuum glazing (TVG) sealed using indium alloy, and 4 mm thick glass panes. In their validated computer simulation work, the emissivity values of the glass sheets were varied from 0.18 to 0.03. The simulated centre U-values are reduced from 0.41 Wm⁻²K⁻¹ to 0.22 Wm⁻²K⁻¹ for a 0.4 m by 0.4 m size triple vacuum glazing. Memon (2017) introduced a triple vacuum glazing using Cerasolzer CS186 alloy as the edge sealant. From the 3D finite element model, the centre of glass U-value of 0.33 Wm⁻²K⁻¹ is found achievable. Meanwhile, if single glazing windows are to be replaced with the designed triple vacuum glazing, a heat loss reduction of 12.92% and 2.69% are achievable. Research on vacuum glazing technology has also led to integrating the technology with other advanced glazing technology. Fang, Hyde (2010) have introduced an integration between electrochromic (EC) with vacuum glazing (VG) technology. The EC component was arranged facing the outdoor environment. For an incident solar irradiance of 300 W m⁻², simulations show that the EC layer is opaque and the inside temperature of the glass pane is higher than the indoor temperature. Meanwhile, for solar irradiance of 1000 W m⁻², the outdoor glass pane temperature exceeds the indoor glass pane temperature, of which as a consequence heat is transferred from outdoors to indoors. Ghosh, Norton (2017) integrated suspended particle device (SPD) with vacuum glazing. In their study, a strong correlation between the measures of the clearness of the atmosphere or clearness index was evaluated for south facing vertical glazing. For clearness index below 0.5, the SPD-vacuum glazing transmission was 17% and 1.1% for transparent and opaque states respectively. A group of researchers from China Zhang, Lu and Chen (2017) proposed an integration between a thin film PV glazing with a vacuum glazing. The glazing prototype was designed and manufactured in such a way that a lamination layer was sandwiched between the thin film PV glazing unit and a vacuum glazing unit, both are independently manufactured. According to the researchers, although semi-transparent photovoltaic windows can generate electricity in situ, they also increase the cooling load of buildings significantly due to the waste heat as a by-product. Experiment has been conducted and the results have indicated that the prototype can not only generate electricity, but also help reduce the cooling load as well as improve the indoor thermal comfort. Recently, researchers Ghosh, Sundaram (2018) investigated a combined semi-transparent multi-crystalline PV vacuum glazing. In their research, both thermal and electrical performance have been investigated. They have found that the combination of multi-crystalline PV cells with vacuum glazing provide low overall heat transfer coefficient, reduces solar heat gain, generates clean electricity and allows comfortable daylight. This paper introduces a design of thin film PV glazing combined with a vacuum glazing. The novelty of the glazing lies in the designs, manufacturing and application of the PV VG-4L. The heat transfer analysis based on the analytical analysis is discussed and solved numerically in MATLAB. Experimental works are presented and the model was then validated against indoor and outdoor experimental results.

2. Design concept and heat transfer analysis of thin film PV vacuum insulated glazing (PV VG-4L)

The design of PV VG-4L is illustrated in Figure 1a. PV VG-4L involves an integration between a thin film PV glazing with a double vacuum glazing (both manufactured independently) and an additional layer of self-cleaning coated glass which totalling 4 layers of glass units which gives the total thickness of 14 mm. The layers were combined together via Ethylene Vinyl Acetate (EVA) film using
an autoclave facility set at its optimum temperature and pressure.

The thermal resistance network for the PV VG-4L is shown in Figure 1b. Both the width and the length of the PV VG-4L have the same length and denoted as W. The heat transfer was analysed based on the following temperature nodes which are; the average glass temperature facing indoors denoted by \( T_g \), the average internal glass temperature denoted by \( T_{i_g} \), the average temperature of the thin film PV glazing denoted by \( T_{TF} \), and the average temperature of the self-cleaning coated glass denoted by \( T_{g_c} \). The lateral heat transfer ends at the point right at the edge sealing and the temperature was denoted as \( T_{edge} \). Due to high thermal conductivity of the edge sealing, it acts as a short circuit for the heat transfer at the edges of the glazing. Meanwhile, due to high thermal conductivity between \( g_2 \) and \( g_4 \), the lateral heat conduction in the glass slab \( g_2 \) and thin film PV glazing \( (TF) \) are assumed negligible. The temperature of the position in \( g_4 \), of which the edge heat transfer takes place is denoted as \( T_{edge flattened}. \) For the uninsulated edges, there will be heat transferred from the internal ambient to the area around the edges.

The thermal resistance network implies that, at certain level of solar irradiance and ambient temperature, the internal surface temperature is in general lower than the external surface temperature. This can be explained as follows. When the PV panel absorbed solar irradiance, a fraction of the absorbed solar irradiance will be converted into electricity meanwhile a fraction will be wasted in the form of heat. This heat released by the PV panel is insulated from the building via the vacuum layer of which, the main heat transfer occurs in the gap are mainly due to radiation and conduction through the support pillars. In the summer, this can be considered as an advantage since the installation of PV glazing would normally cause additional heating during summer of which in return, increase the cooling load of the building. Based on the thermal nodal networks, the energy balance equations were developed. In order to simplify the analysis, the following assumptions have been made:

1) The heat transfer involved is assumed symmetrical. Therefore, we only consider a quarter of the PV VG-4L area in the heat transfer analysis.
2) In this study we have considered the lateral heat transfer through the glass slab in \( g_1 \). This is due to the fact that, the vacuum glazing is highly thermally insulated which makes the lateral heat transfer to become prominent.
3) The edge boundaries of the PV VG-4L are well insulated and hence the edge losses are assumed negligible.
4) Due to high thermal conductivity of the edge sealing, the edge seal is assumed as a thermal short circuit.
5) Meanwhile, due to high thermal conductivity between \( g_2 \) and \( g_4 \), the lateral heat conduction in the glass slab \( g_2 \) and thin film PV glazing \( (TF) \) are assumed negligible.

To simulate the performance of the PV VG-4L, the following energy balance equations were developed for each of the temperature nodes.

For \( g_1 \):

\[
S_{g_1} + h_{g_1-g_1} (T_{g_1} - T_{edge1}) = h_{g_1-g_2} (T_{g_1} - T_{g_2}) + h_{edge-edge1} (T_{edge1} - T_{edge2})
\]

\[
= h_{edge-edge1} (T_{edge1} - T_{edge2})
\]

The heat transfer terms are defined as follows:

1: The rate of the solar energy received by the glass cover or surface facing indoors after transmission through different glass layers per unit area; 2: The rate of heat transfer from the indoor ambient to \( g_1 \) per unit area; 3: The rate of heat transfer from \( g_1 \) to \( g_2 \) per unit area. The heat transfer includes heat conduction through the glass slabs of \( g_1 \), which then followed by heat transfer in the vacuum gap which are due to the heat conduction of the gas particles, heat transfer via radiation and heat conduction through the support pillars, and then followed by heat conduction through the glass slab \( g_4 \); 4: The rate of lateral heat conducted along the x-direction and y-direction of the PV VG-4L from the centre of the PV VG-4L.

For \( g_2 \):

\[
S_{g_2} + h_{g_1-g_2} (T_{g_1} - T_{g_2}) + S_{g_2} + h_{g_2-TF} (T_{g_2} - T_{TF})
\]

The heat transfer terms are defined as follows:

5: The rate of heat transfer from the indoor ambient to the edge area per unit glazing area (if not insulated); 6: The rate of heat transfer from edge 1 to edge 2 through \( g_2 \), \( TF \) and EVA layer per unit area.

For \( TF \):

\[
S_{TF} + h_{g_2-TF} (T_{g_2} - T_{TF}) + S_{g_4} + h_{g_4-TF} (T_{TF} - T_{g_4})
\]

The heat transfer terms are defined as follows:

7: The rate of the solar energy received by \( g_2 \) after transmission through different glass layers per unit area; 8: The rate of heat transfer from \( g_2 \) to \( TF \) through the EVA layer per unit area.

For EVA:

\[
S_{EVA} + h_{g_4-EVA} (T_{g_4} - T_{EVA})
\]

The heat transfer terms are defined as follows:

9: The rate of the solar energy received by \( TF \) after transmission through different glass layers per unit area; 10: The rate of heat transfer from \( TF \) to \( g_4 \) through the EVA layer per unit area.
The heat transfer terms are defined as follows:

11: The rate of lateral heat conducted along the x-direction and y-direction of the PV VG-4L from the centre of the PV VG-4L.

For g4:

\[
\frac{h_{TF,g4}}{10} \left( T_{TF} - T_{g4} \right) + \frac{h_{k,edge}}{11} \left( T_{edge2} - T_{edge4} \right) + S_{g4} = \frac{h_{k}}{14} \left( T_{g4} - T_{a,o} \right)
\]

(6)

The heat transfer terms are defined as follows:

12: The rate of the solar energy received by g4 after transmission through different glass layers per unit area;
14: The rate of heat transfer from g4 to \( T_{a,o} \) per unit area.

The thin film PV VG-4L introduced in this study is new and has never been discussed in existing research. Therefore, in order to theoretically predict the glazing’s electrical performance, the equation that has been widely used in the study of PV/Thermal solar collector is utilized. The following correlation developed by Schott (1985) and Evans (1981) is the most common correlation implemented by researchers who study the PV/T –type solar collector such as Sarhaddi, Farahat (2010) and Tonui and Tripanagnostopoulos (2007):

\[
\eta_{eff} = \eta_{ref} (1 - \beta_{ref} (T_{TF} - T_{ref}))
\]

(7)

Where \( \beta_{ref} \) is the temperature coefficient of the thin film PV cells and \( \eta_{ref} \) is the electrical efficiency at reference temperature (25°C). In this study, by referring to (Virtuani, Pavanello and Friesen 2010), for the a-Si thin film PV layer, the value of \( \beta_{ref} \) is –0.36%, meanwhile, the \( \eta_{ref} \) is 4% by referring to the preliminary experimental results conducted by the current authors. It is important to note that, under real solar condition, the amount of the irradiance reaching the thin-film PV glass component will be influenced by the optical properties of the glazing which depends on the solar angle incidence that varies with the time of the day, and also the day of the year.

From the simulation, the values of the average temperature of the glass sheets (layers) were used to compute the heat transfer coefficients summarised in Table 1. By referring to Figure 1b the centre thermal resistance \( R_{centre} \) and the edge thermal resistance \( R_{edge} \) can be computed as in equation (8) and (9) respectively.

\[
R_{centre} = \frac{1}{h_{a,g1} + \frac{1}{h_{g2} + \frac{1}{h_{TF,g4} + \frac{1}{h_{o}}}}}
\]

(8)

\[
R_{edge, tot} = \frac{1}{h_{edge} + \frac{1}{h_{k, edge} + \frac{1}{h_{edge}}}}
\]

(9)

Hence, the centre U-value \( U_{centre} \) and the total U-value \( U_{total} \) may be computed using equation 10 and 11 respectively:

\[
U_{centre} = \frac{1}{R_{centre}}
\]

(10)

\[
U_{total} = \gamma_{c} U_{centre} + \gamma_{edge} U_{edge}
\]

(11)

In this study, to solve the energy balance equations, we have used the inverse matrix method. MATLAB is used to

Table 1: The definition of the symbol used.

| Symbol   | Definition                                           | Symbol   | Definition                                           |
|----------|------------------------------------------------------|----------|------------------------------------------------------|
| \( \alpha \) | Solar absorptance                                   | \( \tau \) | Solar transmittance                                   |
| \( Y_{edge} \) | Correction factor due to the total area of edge sealing to the centre area | \( G_{rad} \) | Total global solar irradiance incident upon the glazing structure. |
| \( \delta_{edge} \) | The edge sealing thickness                          | \( h_{edge} \) | Conductive heat transfer coefficient due to the edge sealing |
| \( h_{a} \) | Heat transfer coefficient through the support pillars | \( h_{edge} \) | Total heat transfer coefficient from the indoor ambient air to the uninsulated edge sealing area |
| \( h_{k,edge} \) | The lateral heat transfer to the glass edge         | \( h_{g1,g2} \) | The total heat transfer coefficient from g1 to g2 of vacuum glazing |
| \( h_{a,g1} \) | Total heat transfer coefficient from the internal ambient to g1 | \( h_{g2,TF} \) | The total coefficient of heat transfer by conduction from g2 to the thin film PV glazing layer TF. |
| \( h_{TF,g4} \) | The total heat transfer coefficient by conduction from the thin film PV glazing layer TF to g4. | \( k_{edge} \) | Thermal conductivity through the edge sealing |
| \( h_{o} \) | Total heat transfer coefficient from the external glass g4 to the ambient | \( T_{a,o} \) | Indoor ambient temperature |
| \( T_{a,o} \) | Outdoor ambient temperature                          |          |                                                       |
carry out the iteration process. Newton-Raphson iteration technique is used to estimate the temperature and hence the temperature-dependant heat transfer coefficients of the variables.

3. Experimental validation of the theoretical model

3.1. U-value measurement using a TEC driven calibrated hot box

A PV VG-4L prototype using an amorphous silicon (α-Si) solar cell as shown in Figure 2 was manufactured. The U-value of the prototype was evaluated using the TEC-driven calibrated hot box built at the University of Nottingham. Interested readers may refer to Jarimi (2018) for further details. As can be seen in Figure 3, by following closely ISO 12567 standards, the sample was installed at the specimen area of the calibrated hot box. It was tested under three different air temperature conditions summarised in Table 2. However, the air speed in the hot and cold side were fixed at 0.3 m/s and 1.5 m/s respectively. Using the calibrated hot box, we could estimate the total heat transfer coefficient from the hot and cold surface of the PV VG-4L prototype. The values were then used as the input parameters for the computer simulation. To derive the absolute error, the Kline–McCintock second power law as given in NCEES (National Council of Examiners for Engineering and Surveying) (2001) is used. These errors were represented by the error bars of the associated curves. Additionally, the guideline in ISO 12567 was also being referred to evaluate the error from indoor testing.

The mathematical model validation method is performed by comparing the results obtained experimentally and theoretically based on the trends shown on the related graphs. In this study, the mathematical model has been validated against the indoor experimental data with the input parameters recorded in the experiment were used in the computer simulation for all the three different conditions. In addition to the direct comparison between the simulation and theoretical curves, the validation of the mathematical model is further justified using root mean square percentage deviation (RMSPD). As shown in Figure 4, and summarised in Table 3, the evaluated glazing surface temperatures and U-value are found to be in good agreement such that the trend of the theoretical curves are consistent with the experimental curves and the computed RMSPD for the temperatures and U-value are 4.02% and 0.92% respectively.

3.2. Performance analysis under real conditions

The developed mathematical model has been validated against indoor experimental analysis. Nonetheless, the true performance of the PV VG-4L under real sky conditions still needs to be investigated in order to further justify the validity of the mathematical model especially that the electrical performance of the thin film PV glazing could not be evaluated indoors. That said, this section first discusses the performance of the PV VG-4L under real conditions. To carry out the testing, the prototype was installed at E.ON 2016 research house at the University of Nottingham, United Kingdom with latitude of 52.9438°N, and longitude of 1.1934°W. It is worth emphasising that the outdoor monitoring of the PV VG-4L has been conducted under two conditions; during the day and during the night (at zero solar irradiance). The thermal and electrical characteristics of the PV VG-4L under real conditions were monitored using the sensors as summarised in Table 2.

![Figure 2: The PV VG-4L prototype.](image2)

![Figure 3: The calibrated hot box with the installed sample.](image3)

| Cond. | Text (°C) | Tint (°C) | Tint (Exp) (°C) | Text (Exp) (°C) | Tint (Theo) (°C) | Text (Theo) (°C) | U-value total (Exp) | U-value total (Theo) |
|-------|-----------|-----------|----------------|----------------|----------------|----------------|-------------------|---------------------|
| 1     | 12.7      | 32.70     | 30.34          | 13.50          | 29.90          | 13.37          | 0.56              | 0.57                |
| 2     | 17.5      | 27.5      | 27.40          | 18.65          | 26.42          | 17.84          | 0.57              | 0.57                |
| 3     | 7.6       | 27.83     | 25.19          | 9.13           | 25.67          | 8.29           | 0.56              | 0.56                |
During the day, the reading given by the heat flux sensor for the thermal transmittance of the PV VG-4L is not reliable due to the error from the solar irradiance to the heat flux sensor and also due to the absorbed solar irradiance by the PV layer. Therefore, during the day, the only parameters that are being considered are the surface temperature difference of the PV VG-4L and its electrical power produced. The typical monitoring period is between 9:30 a.m. to 6:00 p.m. in a typical day of May and June. Meanwhile, the focus of the experiment during the night is the thermal transmittance measurement or U-value. A 1kW radiator was used as the heat source for indoors.

Figure 5 shows the solar irradiance, surface temperatures of the PV VG-4L, and both internal and external; glazing surface temperature during the day and night. It should be noted that for the data taken on the 23rd to 24th of May 2019, the internal heat source was switched off during the day meanwhile, the average heating temperature was set at 30°C during the night (i.e. from 8 p.m. to 4 a.m. the next day) to heat the ambient room at 24 to 25°C. On the day of testing, the sky was in a clear sky condition and hence clear pattern of solar irradiance curve with the time of the day was obtained. Another set of experiment was conducted to evaluate the performance of the PV VG-4L on the 20th to 21st of June 2019.

Table 3: Measured parameters and uncertainties.

| Parameters                          | Sensor                                      | Uncertainties          |
|-------------------------------------|---------------------------------------------|------------------------|
| Temperatures                        | K-type Thermocouples                        | ±1.5°C                 |
| Heat flux                           | Hukesflux Thermal Sensors                   | ±1.9 × 10⁻⁶ V/(W/m²)   |
| Solar irradiance                    | Pyranometer                                 | ±5%                    |
| Maximum power produced by the PV VG-4L | RO4 with Keysight 34972A                    | Electric current (I) (±1.5 µA) |
|                                     |                                             | Voltage (V) (±190 µV)  |

However, as can be seen on Figure 6, during the day of testing, the sky turned cloudy and hence periods of fluctuating solar irradiance were recorded. During the aforementioned period, steady state condition of the PV VG cannot be well justified. This is attributed to the response time of the glass (time constant) which is caused by the thermal mass of the PV VG components. Furthermore, the influence of the thermal time response of the PV VG to the sudden drop in solar irradiance is clearly shown by the change in the recorded temperature profiles from 1:45 p.m. to 2:15 p.m. During the aforementioned period, the average intensity of the incident solar irradiance dropped for a step change from 400 W/m² to 60 W/m² and the temperature profiles clearly follows an exponential decay lagging behind the step change. Additionally, by carefully examining Figure 6, at points in which there was a sudden drop in the value of solar irradiance due to the movement of clouds, the temperature profiles are also observed to lag behind the values of solar irradiance.

It is worth noting that, in both graphs, at low–zero solar irradiance, the indoor surface temperature of the PV VG-4L is in general higher than its external surface temperature. However, as the incident solar irradiance increases, the external surface temperature increases. The trend of the graph is explained as follows; when the PV component
of the PV VG-4L absorbed the incident solar irradiance, a fraction was converted into electrical energy meanwhile the rest was wasted in the form of heat. However, the vacuum layer behind the PV component of the PV VG-4L acts as the insulation layer or barrier to the wasted heat from being transferred indoors. As a result, the external surface temperature of the PV VG-4L became higher compared to its internal surface temperature. In the summer, this will be an advantage in comparison to the typical installation of BIPV in double glazed configuration. For the
data analysis, the temperatures, solar irradiance and heat flux were recorded for every 1 s meanwhile, the electrical parameters were recorded for every 10 s.

The outdoor experimental results obtained from the 23rd to 24th of May, as discussed previously, were compared with the theoretical results using the developed mathematical model. Due to the varying condition of the ambient climate, the electrical performance and thermal characteristics of PV VG-4L in a steady state condition are analysed as per time constant of the PV VG-4L. From our analysis, it is concluded that the computed time constant for the PV VG-4L is approximately 30 minutes. The experimental results were compared based on the PV VG-4L surface temperature difference and electrical performance at quasi-steady state during the day and thermal transmittance or U-value during the night. The comparisons between the outdoor experimental and theoretical results are represented in Figures 7 and 8 for the reading during the day and night respectively. The trend given by both outdoor experimental and theoretical curves are in good agreement. Figure 7 shows that the maximum power produced at average solar irradiance of 600 W/m² is approximately 14W/m² for 0.4 m × 0.4 m PV VG-4L made of amorphous silicon solar cells. It is worth emphasizing that, a different power produced is expected when a different type of thin film PV is used as the prototype. For example, using the validated mathematical model, it is predicted that, if amorphous/microcrystalline silicon solar cells at 20% transparency is used as the thin film layer, the power output at the same average solar irradiance can achieve as high as 32 W/m². In order to further justify the validity of the mathematical model, error analysis using RMSPD analysis was performed. The average RMSPD for the glazing surface temperatures, the power produced and U-value are 2.90%, 8.7% and 2.89% respectively. It is worth noting that the derived absolute errors for the power produced are too small to be included in the plotted curves.

4. Parametric studies of PV VG-4L

Using the validated mathematical model, the U-value of the PV VG-4L was evaluated at different pairs of width and length as summarised in Table 4. The U-value improves
due to the lateral heat conduction across the glass due to the edge seal. The finding is in agreement with the findings in Fang (2015).

At 1.0 m × 1.0 m in size, the influence of solar irradiance $G_{rad}$ to the increase in PV temperature and hence the PV performance of the thin film PV glass was evaluated. The thin film PV performance was simulated with the variation in solar irradiance at fixed outdoor temperature of 15°C. The size of the window was fixed at 1 m by 1 m. The

| $L_w$ (Glazing width) (m) | $L_w$ (Glazing length) (m) | U-value (W/m²K) |
|--------------------------|---------------------------|-----------------|
| 0.3                      | 0.3                       | 0.66            |
| 0.5                      | 0.5                       | 0.55            |
| 1.0                      | 1.0                       | 0.51            |
| 1.5                      | 1.5                       | 0.50            |

(designes with the increase in the glazing size) due to the lateral heat conduction across the glass due to the edge seal. The finding is in agreement with the findings in Fang (2015).

Figure 8: Top: Outdoor Experimental and Theoretical Curves for the Glazing surface temperature and bottom: U-value for 0.4 m × 0.4 m prototype as per time constant of the PV VG-4L with time of the day as (data was taken on the 23rd of May 2019 to 24th of May 2019 at low-zero solar irradiance).

Table 4: The parameters and the computed U-values.
Figure 9: The variation in the electrical efficiency $\eta_{ele}$ with the increase in solar irradiance at fixed indoor and ambient temperature for PV VG-4L at high and low vacuum level.

Figure 10: The variation in temperature of the PV cells, with the increase in solar irradiance at fixed indoor and ambient temperature for PV VG-4L at high and low vacuum level.

Figure 11: The variation in the electrical efficiency with the change in outdoor ambient temperature at fixed solar irradiance and indoor temperature.
aim is to keep the indoor temperature at 23°C. Figure 9 shows that, due to the electricity generation, the by-product of waste heat leads to the increase in the average temperature of the glazing which influences the electrical efficiency of the PV VG. Figures 9 and 10 shows that, the influence of the absorbed solar irradiance to the increase on PV temperature and hence the electrical efficiency of the PV VG was only obvious at solar irradiance above 300 W/m². On average, the drop in the electrical efficiency due to the presence of vacuum level is only 0.1%.

Additionally, the temperature of the PV VG was also predicted with the change in the outdoor ambient temperature ranging from −10°C to the extreme of 40°C at fixed solar irradiance and indoor ambient condition of 700 W/m² and 23°C respectively. Figure 11 shows that the electrical efficiency of the PV VG decreases by 0.032% with the increase in each degree of the outdoor ambient temperature. From Figures 9 to 11, we may conclude that influence of the excellent heat insulating properties of the thin film PV VG to the electrical efficiency of the thin film PV glass is significantly small and can be neglected.

5. Discussions and Conclusions
An innovative Thin Film Photovoltaic Glazing with Vacuum Insulated Layer (PV VG-4L) is presented in this paper. A mathematical model was developed by taking into account all the parameters related to the individual component of a typical vacuum glazing unit which is the dominant in the design and also the fraction the solar irradiance absorbed by the different layers of PV VG-4L. To validate the mathematical model, a lab-scale prototype was manufactured and tested indoors using a calibrated hot box, and outdoors by installing the sample at a research house. Under controlled conditions, the overall U-value of the PV VG-4L was measured to be as low as 0.6 W/m²K. When investigated under real conditions, an obvious trend in glazing surface temperature variation with solar irradiance was obtained. During low to zero solar irradiance, the internal glazing surface temperature is on average higher than the external glazing surface temperature. However, as the solar irradiance increases, the by-product of the absorbed heat by the thin film PV glazing layer, has led to an increase in the external glazing surface temperature as the heat is hindered from being transferred indoors by the vacuum layer. At average solar irradiance of 600 W/m² the PV VG-4L can produced in total of 14W of power per m² of panel. Meanwhile, at low to zero solar irradiance (i.e. during the night), at outdoor ambient temperature of 14°C, the average U-value of the typical PV VG-4L was found to be as low as 0.6 W/m²K while maintaining the indoor ambient temperature at 30°C. Please note that high internal ambient temperature was obtained due to the use of heater at its maximum setting. For a conventional thin film PV glazing, to improve the thermal performance of the thin film PV glazing, a combination with a double-glazing unit is possible with the estimated U-value of 2.5–2.8 W/m²K depending on the type of gas used to provide the insulation in the air gap. Clearly, the vacuum layer introduced in the PV VG-4L design presented in this paper is better in performance with the slim configuration of the glazing unit as an additional benefit. The results also show that the PV VG-4L not only can produce power but also has high insulation properties when compared to a single thin film PV glazing with a typical U-value of 5 W/m²K; the U-value is higher by almost 90%. The promising U-value implies its range of potential applications can be improved depending on the energy needs and applications, such as for BIPV solar façade (PV curtain walling) in commercial buildings, greenhouses, skylight and conservatory.

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Saffa Riffat is the Editor in Chief of the journal and was removed from all editorial processing for this paper.

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